

C2Or1B-01: Development of a Helium Recondensing Cryostat

P.K.Muley, **S.L.Bapat** & M.D.Atrey

Mechanical Engineering Department
Indian Institute of Technology Bombay, Mumbai, India
email: slbapat@iitb.ac.in



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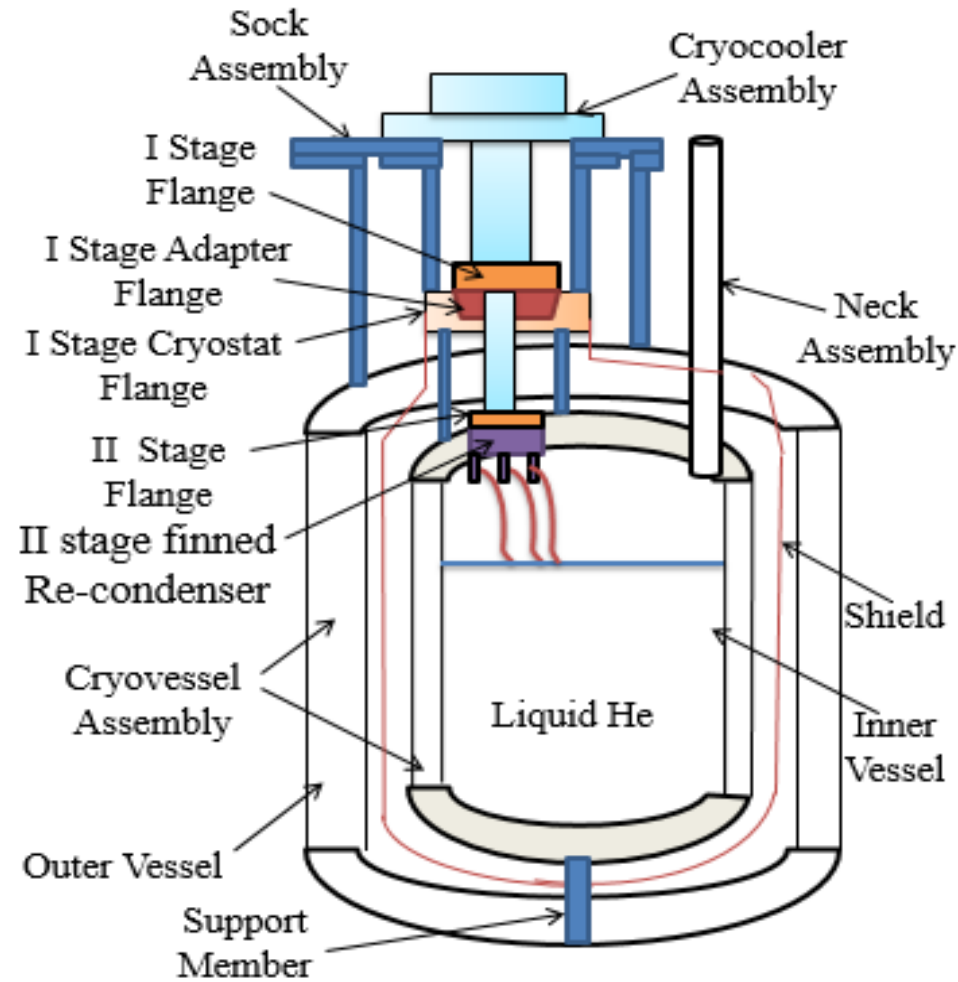
- **Demerits of conventional Helium cryostats:**
- Disruption in operation and loss of liquid Helium due to periodic refilling.
- Require Helium recovery system, liquefaction plant.
- **Present work:** A step towards conservation of liquid Helium with an objective to develop an “**in-situ**” recondensing Helium cryostat.



**Conventional
Cryostat**

**Mother
Dewar**

- Mechanical and thermal design:**

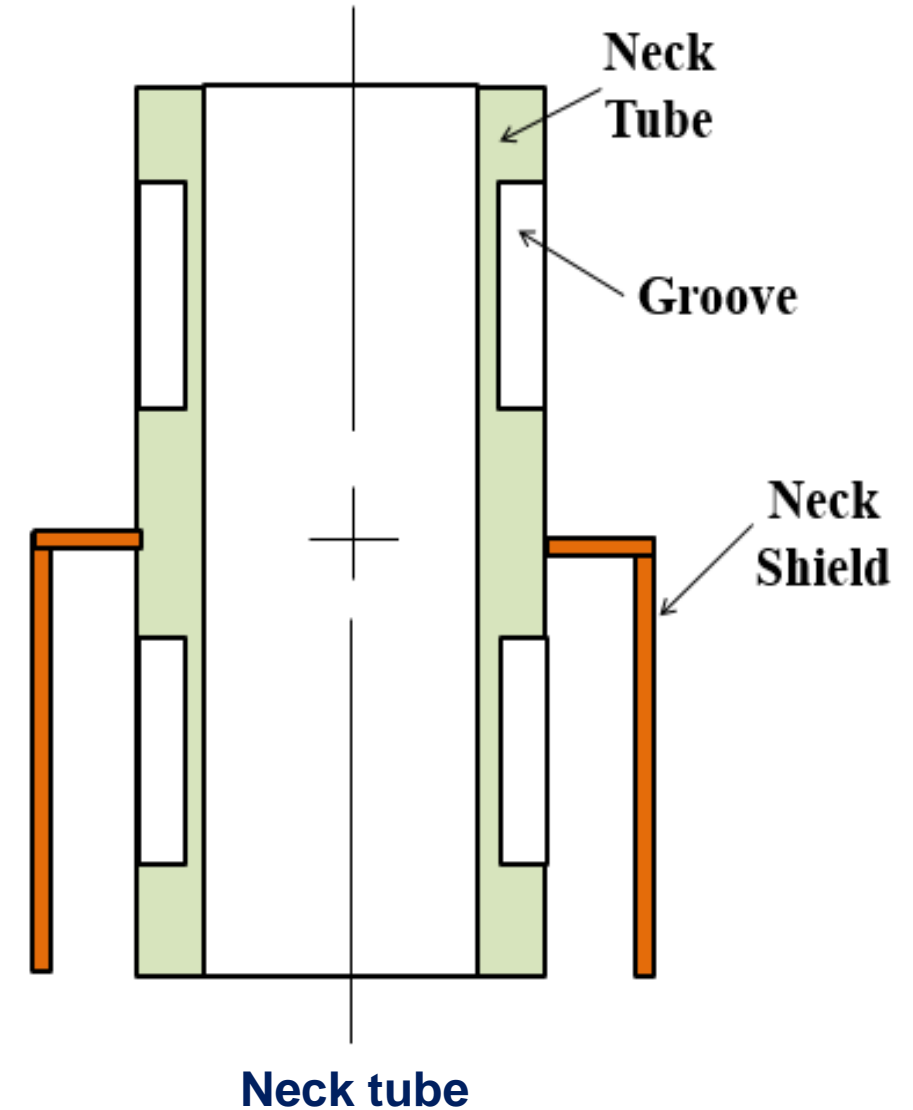


Design specifications of Helium recondensing cryostat

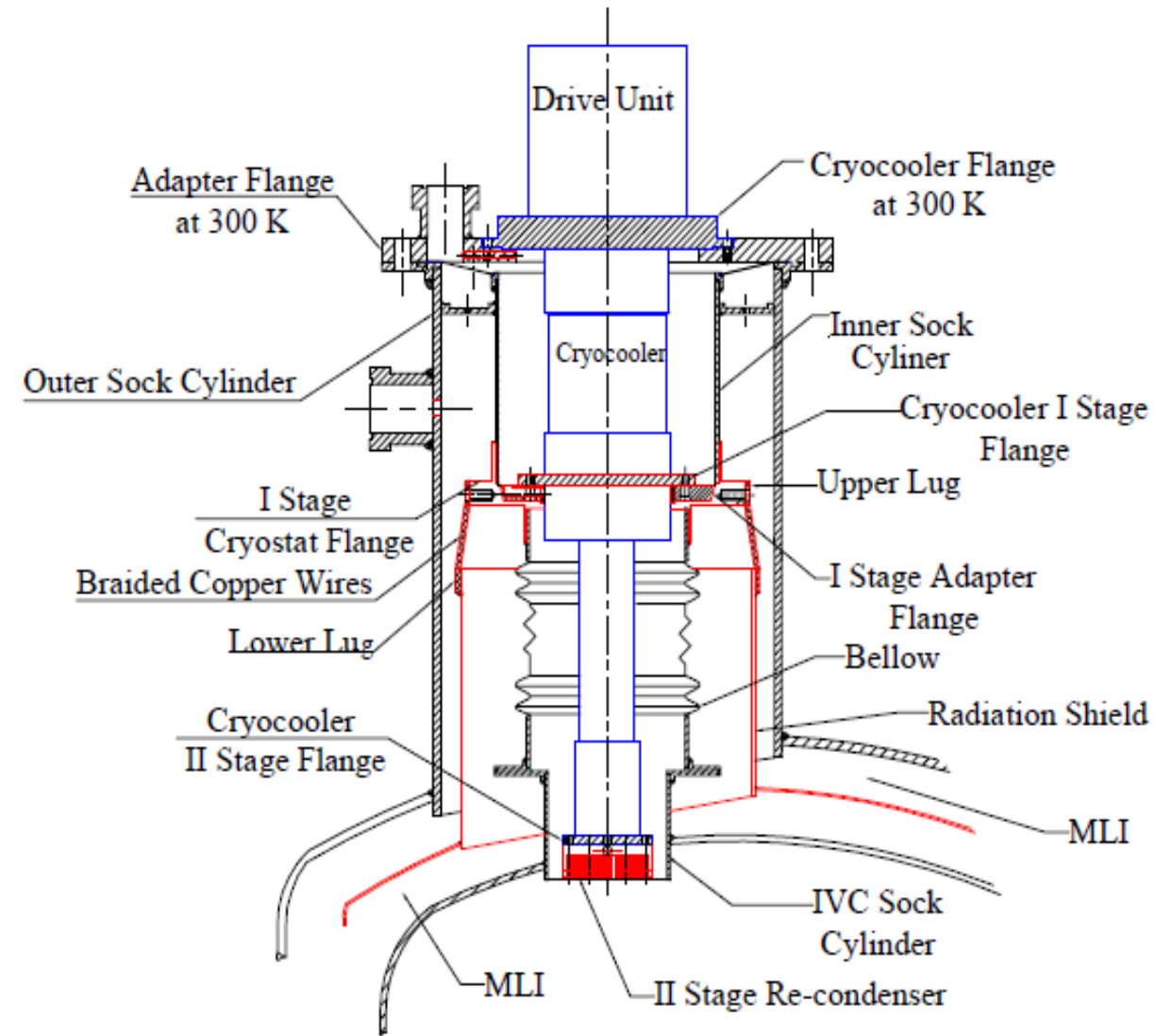
Design Parameter	Design Specification
LHe storage capacity	100 litre
Ullage volume	6% of liquid volume
Design pressure	6 bar
Length to inside diameter ratio (l/d)	0.9
Normal permissible boil-off rate	1% per day
GM Cryocooler	40 W at 43 K – I stage 1 W at 4.2 K- II stage

“In-situ” Helium recondensing cryostat

- **Mechanical and thermal Design:**
- Radiation shield- Perforated, 1.5 mm thick, Material- ETP Copper,
- Multilayer insulation- 0.006 mm Aluminum foil and 0.15 mm fiberglass paper.
- Neck Tube – SS 304.
- Radiation shield brazed to neck tube.
- Neck thermal load transferred to I stage of cryocooler.

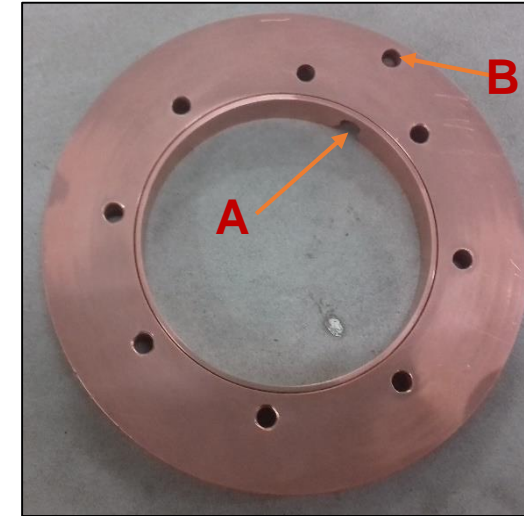


- **Sock assembly:** For smooth insertion and proper positioning of cryocooler for “in-situ” recondensation .
- Encloses adapter flange and cryostat flange of I stage for transfer of cooling effect at I stage of cryocooler to radiation shield through braided copper wires.



Sock assembly

- **I Stage adapter flange** of OFHC Copper is mounted on integral cryocooler flange at I stage.
- Internal cross hole **(A)** and hole **(B)** are provided in it for passage of lead wires of Silicon diode and heater mounted at II stage re-condenser.
- **I Stage cryostat flange** of ETP Copper is connected to shielded through braided copper wires.
- **H6h6** precision location rotational sliding fit.
 - Perfect thermal coupling.
 - Minimum contact thermal resistance.
 - Minimum loss during transfer of cold generated at I stage of cryocooler to radiation shield.



I stage Adapter flange



I stage cryostat flange

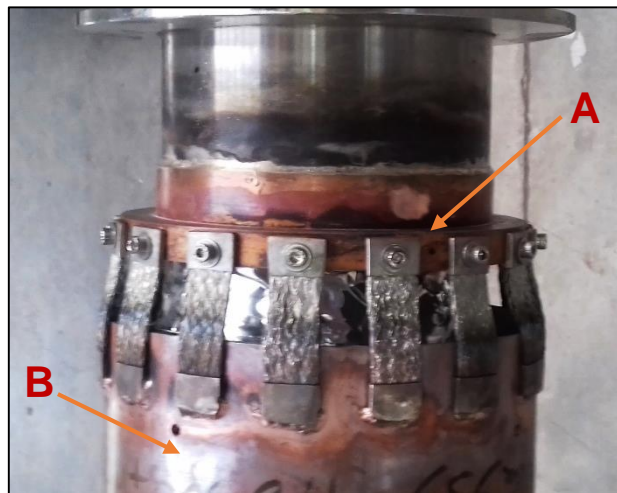


Fitment of both I stage flanges

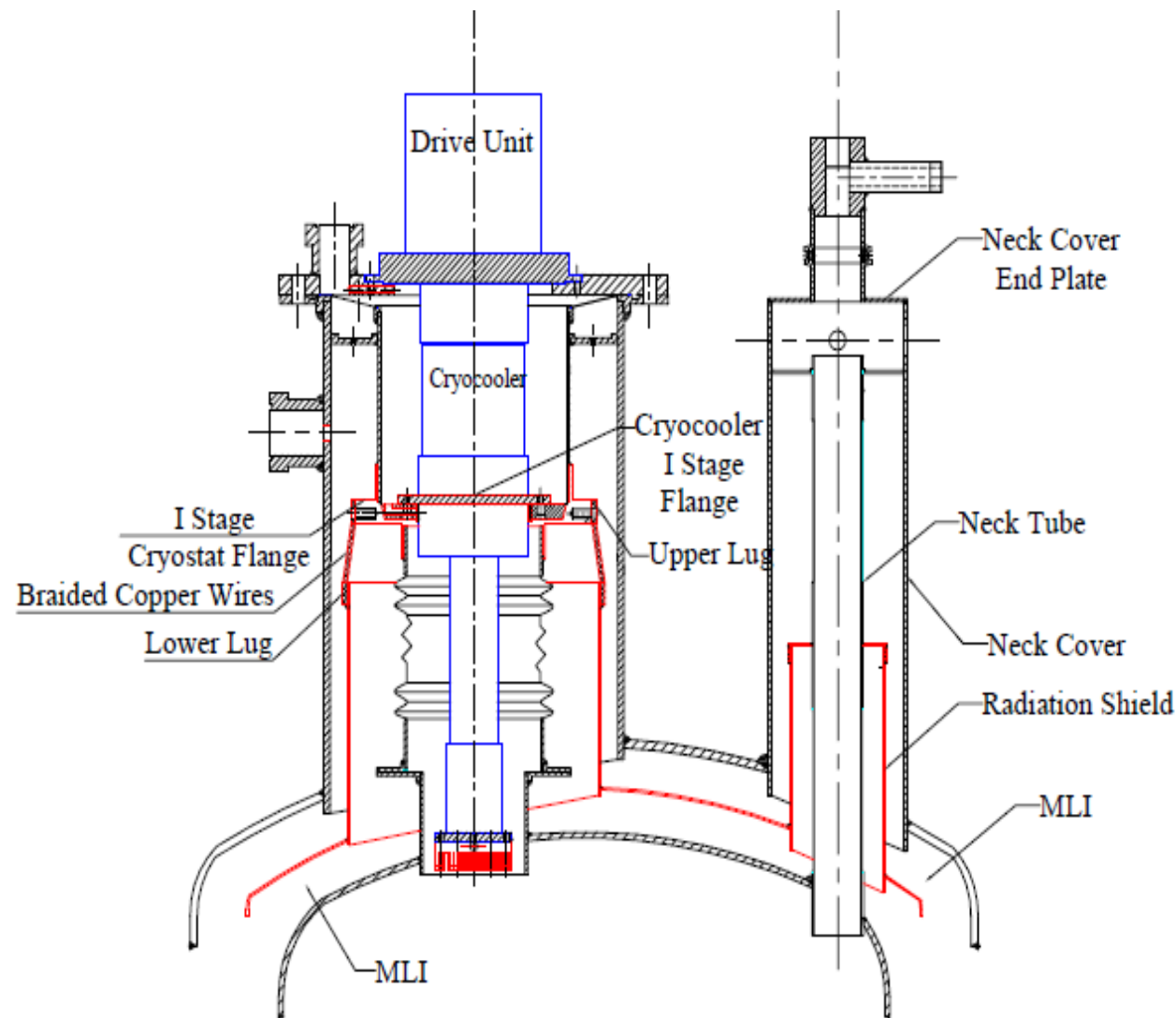
- **Braided Copper wires with lugs:** 16 sets of 80 mm long, 32 x 30 x 0.2/1.0 mm, each with 02 No. ETP Copper lugs provide thermal linkage for transfer of cooling effect from I stage cryostat flange (A) to radiation shield (B).



Braided Copper wires with lugs

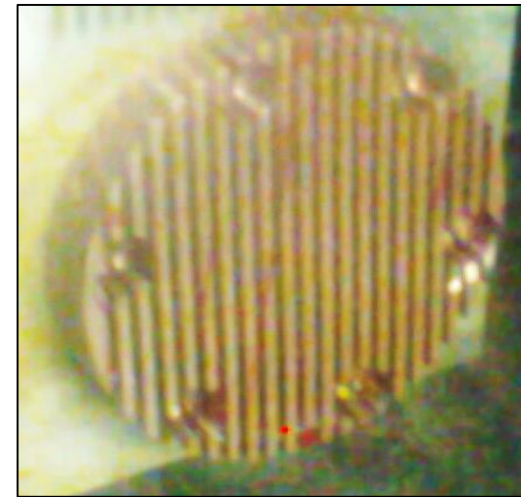


Thermal linkage of I stage cryostat flange to shield

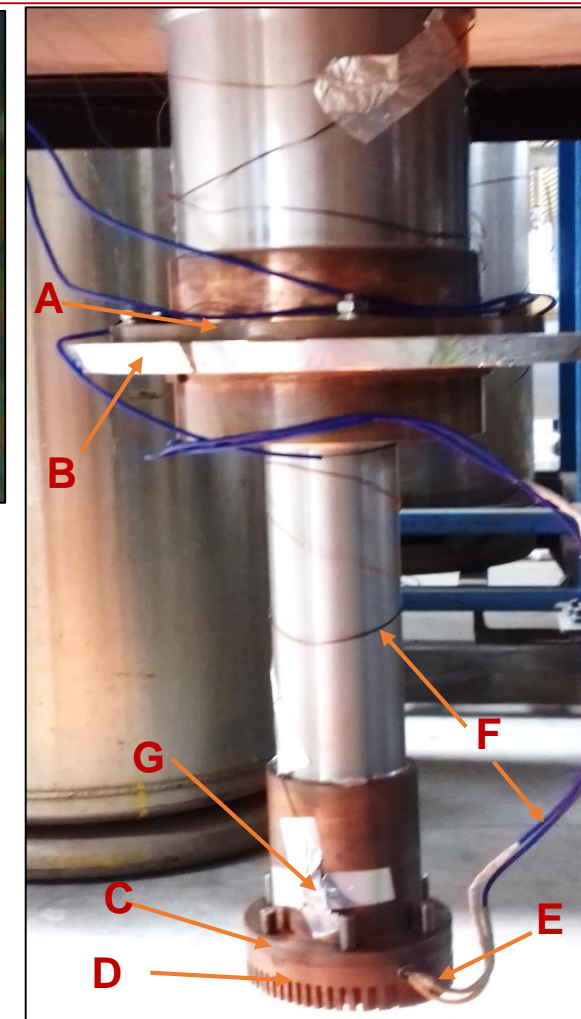


Thermal linkage of neck tube to cryocooler

- **II stage Re-condenser (D)** made of OFHC Copper is mounted on integral II stage flange **(C)** of cryocooler for “**in-situ**” recondensation.
- 19 No. straight rectangular integral fins each of thickness 1.5 mm, height 10 mm with 1.5 mm gap between two fins are cut on an **inclined blank** of varying thickness of 20.7 mm at one edge and 17.5 mm at diametrically opposite edge.
- This ensures that condensed liquid Helium will collate at one end of fin and drop from there.



Finned re-condenser (C)



I Stage adapter flange B and II stage re-condenser D mounted on integral cryocooler flanges A and C; Heater E, Lead wires F, Silicon diode G at II Stage

- Manufacturing processes involved:** Forming, Machining, Joining, MLI application.



Inner vessel



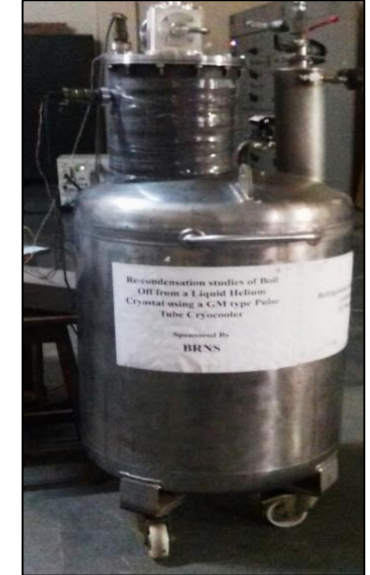
Integration of inner vessel with middle Copper shield



Application of MLI on Copper shield assembly



Integration with outer vessel



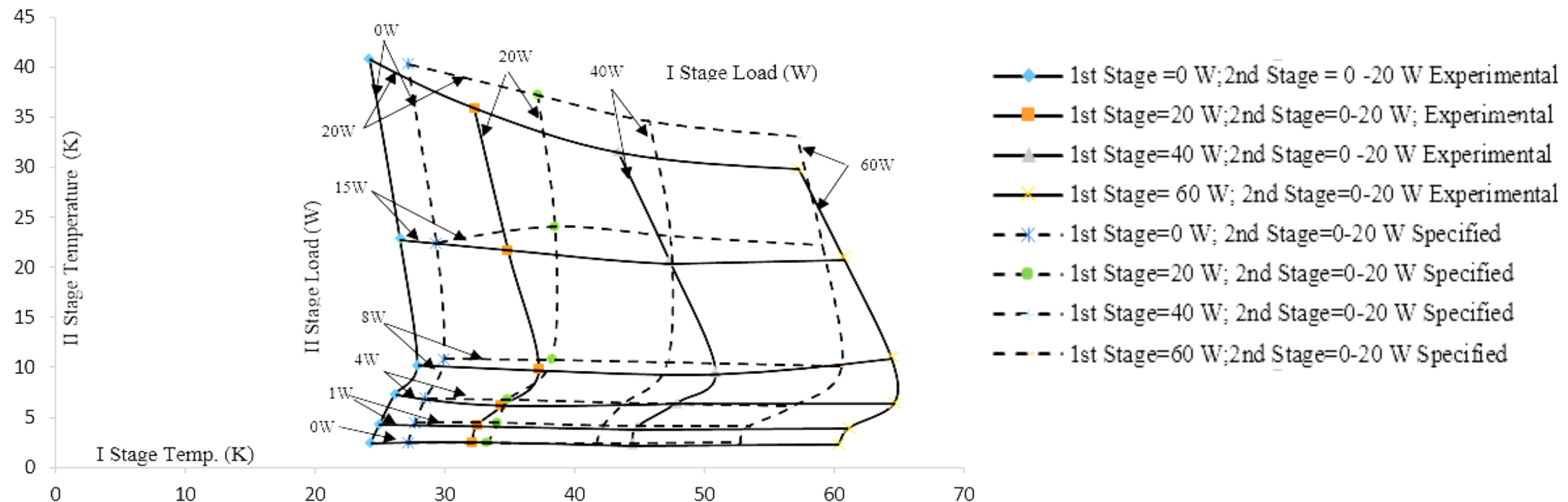
Cryocooler mounted on re-condensing cryostat

- Quality tests conducted:** Pneumatic and hydrostatic pressure tests, MSLD test using Helium gas.
As per ASME Code, Section VIII, Division I.



- Continuous trials conducted separately on Sumitomo Heavy Industries (SHI), Japan make two stage GM Cryocooler in a specially fabricated vacuum jacket.
- **Sensors and Instruments:**
 - Silicon diode and Heater-1 No. each at I stage cryostat flange.
 - Silicon diode and Heater-1 No. each at II stage re-condenser.
 - Silicon diode- 1 No. at shield bottom.
 - Temperature indicators- 2 No. for I stage, shield and II stage temperatures.
 - 0- 60 W D.C. Power source - 1 No. for I and II stage heaters.
- **Results : 40 W at 40.3 K - I Stage , 1 W at 4.02 K - II Stage.**
- **Cryocooler capacity : 40 W at 43 K - I Stage , 1 W at 4.2 K - II Stage.**

- Thermal load map of cryocooler obtained experimentally at various electrical heater loads applied to both stages and compared with thermal load map specified by SHI.



Comparison of thermal load map of GM cryocooler for experimental results with specifications by SHI

- Conclusion:** Performance of GM cryocooler found **better** than specifications by manufacturer.

Parts of sock assembly causing problems in fitment of cryocooler in cryostat

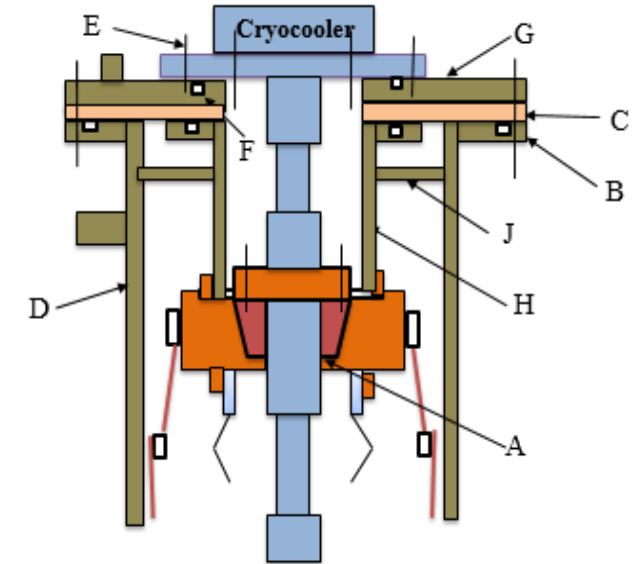
Part No./Name	Reason of problem	Corrective Measures
A- Face of I stage cryostat flange.	Waviness and gap at face (A) .	Lapping at face (A) using tool developed 'in-house'.
B - Outer sock cylinder top flange	Deformation in outer sock cylinder top flange (B) during its welding with outer sock cylinder (D) resulted in gap upto 3 mm with mating face of 7 mm thick intermediate flange (C) .	Intermediate Flange (C) discarded from sock assembly. A ring of varying thickness of 7 mm at one edge and 4 mm at diametrically opposite edge fused with outer sock cylinder (D) .

M-5 x 9 – 08 No. through holes **(E)** - Blinded by about 1 mm from bottom.

O-Ring **(F)** in outer sock cylinder top flange **(B)** - Changed from Φ 3.5 to Φ 5 mm.

Vertical alignment of cryostat- **Ensured**.

Result: Cryocooler **fitted properly** in sock assembly of cryostat.



Parts of sock assembly causing problems in cryocooler fitment;
G- 300 K Cryocooler flange,
H- Inner sock cylinder



Lapping tool

- First trial of cryostat performance test in vacuum:
- Results: **Unsatisfactory. Heavy condensation** of ambient water vapour on outer surface of outer sock cylinder D.

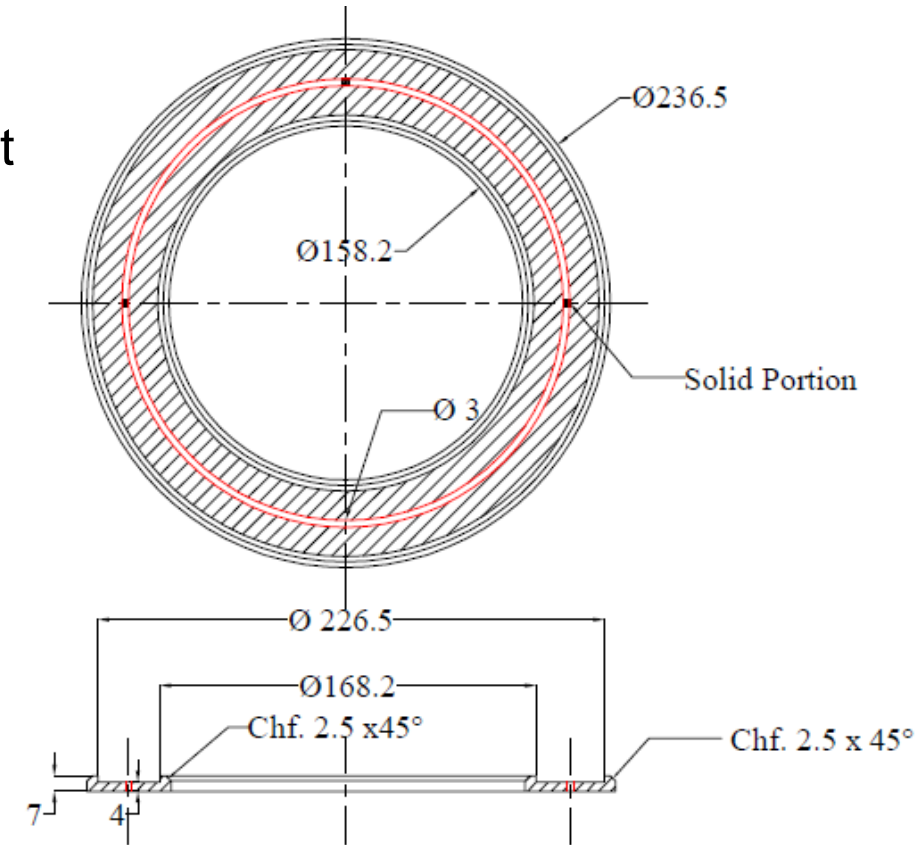
High temperatures of **82.60 K** and **6.47 K** for **I** and **II stage** respectively at no electrical heater load.

Reason: High radial conduction heat transfer from ambient to I stage due to **large thickness** of vacuum arresting plate **(J)**.

Corrective action: ϕ 3 mm **through slot** provided in 4 mm thickness of vacuum arresting plate **(J)**, **retaining 3 mm solid portion** at **3 locations**, 120° apart.

O- Ring : Φ 5 mm neoprene rubber, Sealant: Vacuum clay.

Vacuum in inner vessel volume and insulation space $< 10^{-6}$ mbar.

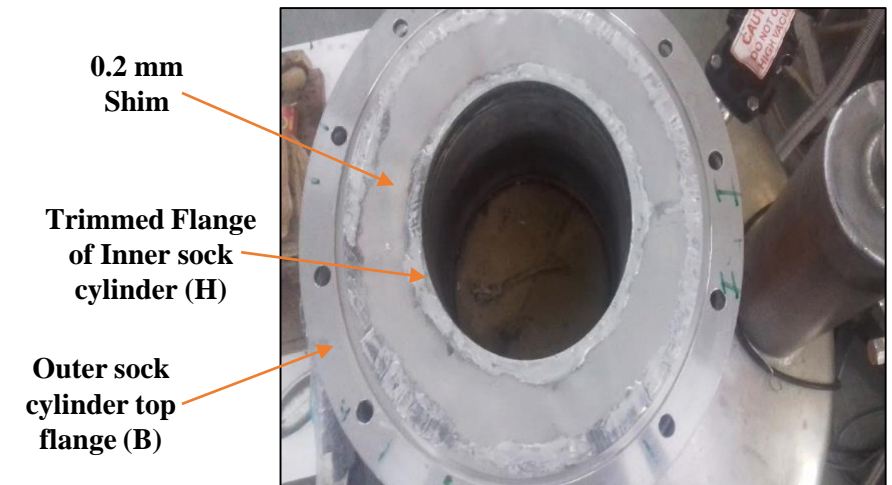
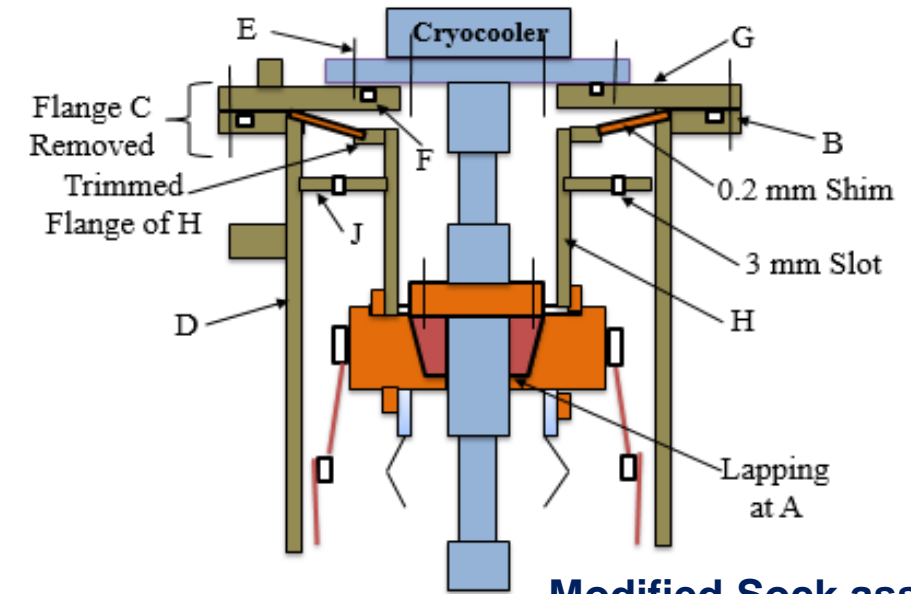


Modified vacuum arresting plate

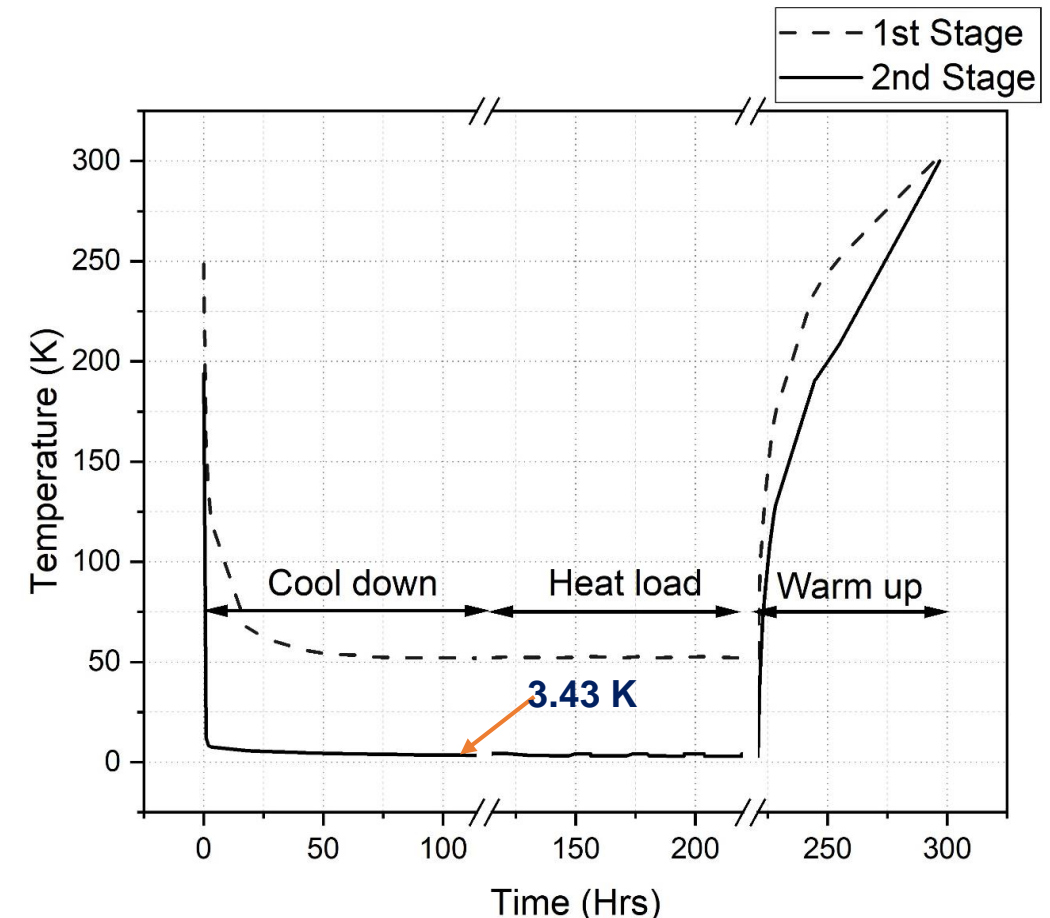
- **Results: Unsatisfactory.** With no electrical heater load, **I stage : 51 K, II stage : 5 to 6 K.**
Reason: Cracking of vacuum clay due to **very low temperature. Loss of vacuum** on insulation side.
Corrective action: Sealant applied: **Araldite.**
Cryocooler trial with cryostat in vacuum conducted for **08 Days.**
- **Results: Steady temperatures** of **50.75 K** and **3.80 K** at **I** and **II stage** respectively **after cooldown.**
Satisfactory results of **51.33 K** at **I stage** and **0.336 W** at **4.21 K** for **II stage** obtained on applying electrical heat load to II stage only.
Cracking of araldite and loss of vacuum after the trial, prior to LN2 purging of cryostat.
Reason: Exposure of araldite to **very low temperatures** and **high vacuum** for **long period** during trial.
Modification: Used 0.2 mm thick SS-304 vacuum arresting shim in sock assembly.
Discarded O-ring in slot of modified vacuum arresting plate.

Effect of modifications in mechanical design of sock assembly on its thermal load due to conduction for I stage

Arrangement in sock assembly	Solid conduction heat load (W)		
	Radial	Axial	Total
Thick vacuum arresting plate J only	66.63	22.62	89.25
Vacuum arresting plate with ϕ 3 mm slot fitted with O-ring and vacuum clay around it.	16.06 (Includes 0.967 W)	22.40	38.46
Vacuum arresting plate with ϕ 3mm slot, No O-ring, Vacuum arresting shim, 0.2 mm thick SS-304, brazed in sock assembly.	5.35 (Vacuum arresting plate: 0.967 W + Vacuum arresting shim: 4.38 W)	22.40	27.75



- **Second trial of cryostat performance test in vacuum:** Carried out after all modifications in sock assembly.
- **Sensors and Instruments:**
 - Silicon diode and Heater-1 No. each at I stage cryostat flange.
 - Silicon diode and Heater-1 No. each at II stage re-condenser.
 - Pt 100 – 1 No. each at shield bottom and neck tube joint.
 - Temperature indicator- 1 No. for I and II stage temperatures.
 - 0- 60 W D.C. Power source - 1 No. for I and II stage heaters.
 - Ohm meter- 1 No. for shield Pt 100 resistances measurement.
- **Results of cooldown:** Almost **steady state temperatures** of **51.95 K** and **3.43 K** for **I** and **II** stage respectively at no electrical heater load.



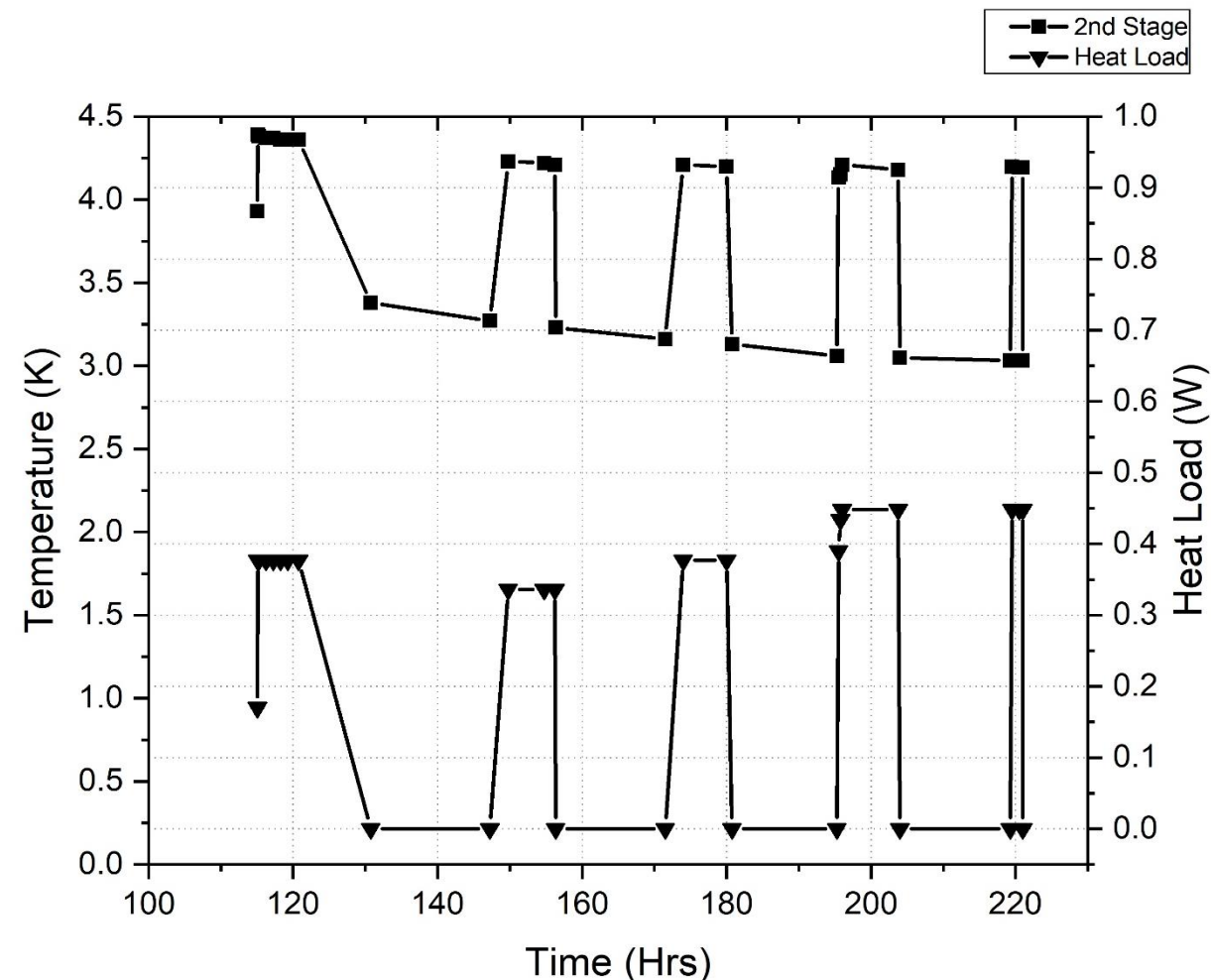
Cool down curve of Helium recondensing cryostat performance test in vacuum

- Results of electrical heater load application:
II stage : 0.448 W at 4.21 K,
I stage : 52.47 K with no electrical heater load.



I and II Stage temperatures with applied electrical heater load for cryostat performance test in vacuum

- Test Results: Satisfactory



Variation of II Stage temperatures and applied electrical heater load for cryostat performance test in vacuum



Results of thermal load estimation due to cryostat assembly (after modifications)

Thermal Load (W)	Cryovessel		Neck		Sock	
Stage:	I	II	I	II	I	II
1.Solid Conduction	2.73	0.076	1.74	0.08	27.75	0.045
2.Radiation	0.67	2.23×10^{-4}	0.119	2.17×10^{-5}	0.337	0.056
3.Residual Gas Conduction	0.33	0.038	0.011	2.86×10^{-4}	0.039	1.5×10^{-3}
Total (1+2+3)	3.73	0.114	1.87	0.08	28.13	0.103
Stage:	I				II	
Thermal load due to cryovessel, sock and neck assembly:	33.72 W				0.298W	
Conduction load due to sensors/instruments lead wires:	6.67×10^{-3}				4.98×10^{-3}	
Total thermal load of cryostat assembly:	33.73 W				0.303 W	
Cryocooler Specifications:	40 W at 43 K				1 W at 4.2 K	
Cryocooler Performance: (our results of second test)	0 W at 52.47 K				0.448 W at 4.21K	

- **Conclusions:** Excess of thermal load over and above normal boil-off load = **0.4176 W**
= (0.448 - 0.0304)
- Any experiment with **heat load < 0.4176 W** can be conducted using present cryostat, with **“in-situ”** recondensation for any length of time, without any increase in pressure of cryostat from 1 bar, without any need of refilling of LHe in cryostat.
- **Objective** of present work to develop an **“in-situ”** recondensing Helium cryostat is thus **fulfilled**.
- The cryocooler needs to be tested one more time in cryostat filled with LHe.
- **Acknowledgements:** The authors acknowledge financial support from **Board of Research in Nuclear Sciences (BRNS), India** for present research work in area of **“in-situ”** Helium recondensation cryostat.



Thank you !