

## Introduction

- Large dewars employ support systems to hold the inner and outer vessels
- Supports should be mechanically strong and allow minimal heat inleak through them
- Design of supports become an optimization problem

## Objectives

- To design an optimized dewar support system
- To study the stress distribution in, and heat inleak through a dewar support member

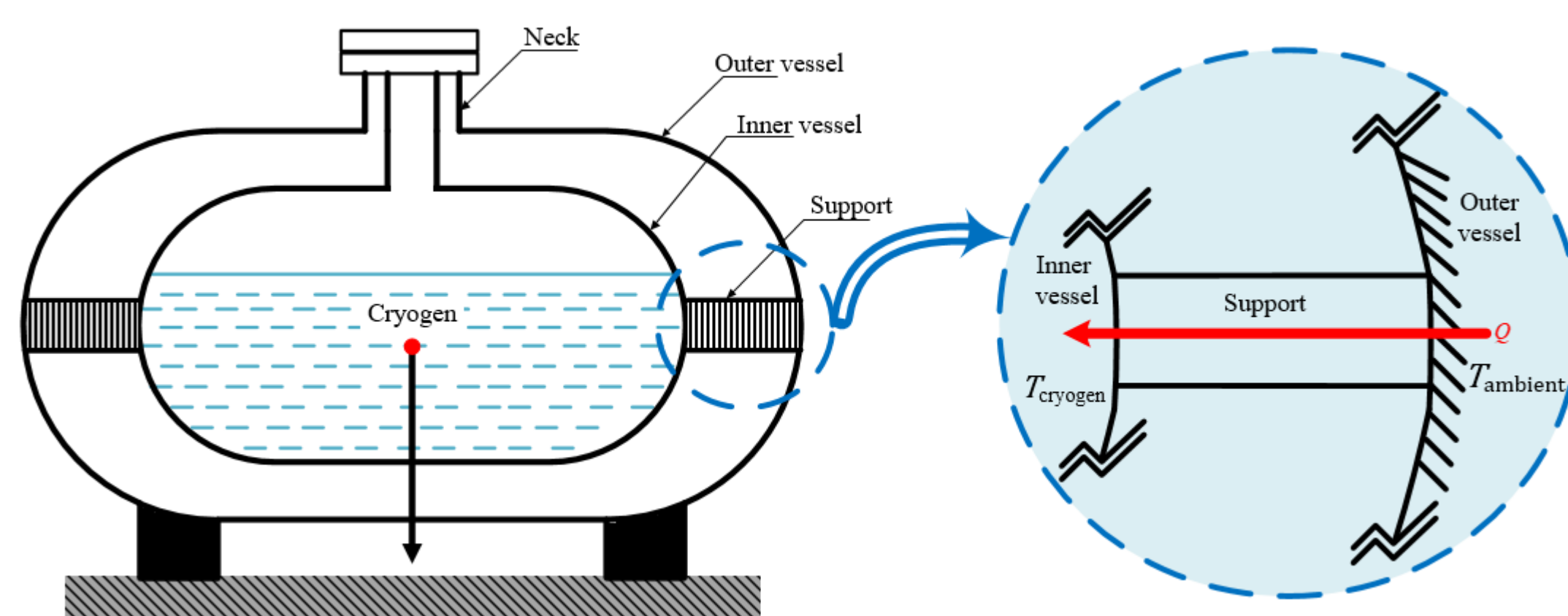


Figure 1: Typical support system in a dewar

## Model Assumptions

- Steady state
- Material of construction of the support is isotropic and homogeneous
- Heat inleak to the inner vessel is due to only conductive heat transfer through the support member
- Weight of the support member is negligible compared to the loads acting on it

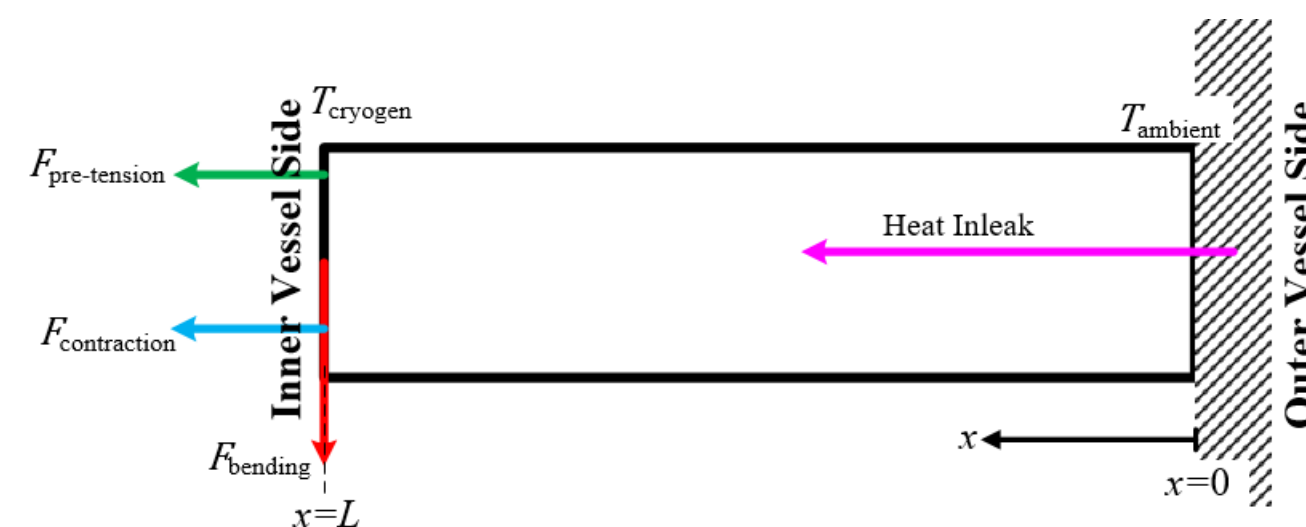


Figure 2: Forces acting on, and heat inleak through a support member

## Model Equations

### Stress Distribution

$$\sigma_{\text{bending}} = \frac{My}{I} \quad (1)$$

$$\sigma_{\text{pre-tension}} = \frac{F_{\text{pre-tension}}}{A} \quad (2)$$

$$\sigma_{\text{contraction}} = E\alpha (\Delta T) \quad (3)$$

$$\sigma = \sigma_{\text{bending}} + \sigma_{\text{pre-tension}} + \sigma_{\text{contraction}} = \frac{My}{I} + \frac{F_{\text{pre-tension}}}{A} + E\alpha (\Delta T) \quad (4)$$

$$\Delta T = T(x, t) - T(x, 0) \quad (5)$$

### Energy Balance

$$\frac{d}{dx} \left( k(T) \frac{dT}{dx} \right) = 0; \text{B.C. : } T|_{x=0} = T_{\text{ambient}} \text{ and } T|_{x=L} = T_{\text{cryogenic}} \quad (6)$$

### Heat inleak

$$\dot{q} = -k \frac{dT}{dx} \bigg|_{x=L} \quad (7)$$

## Solution Methodology

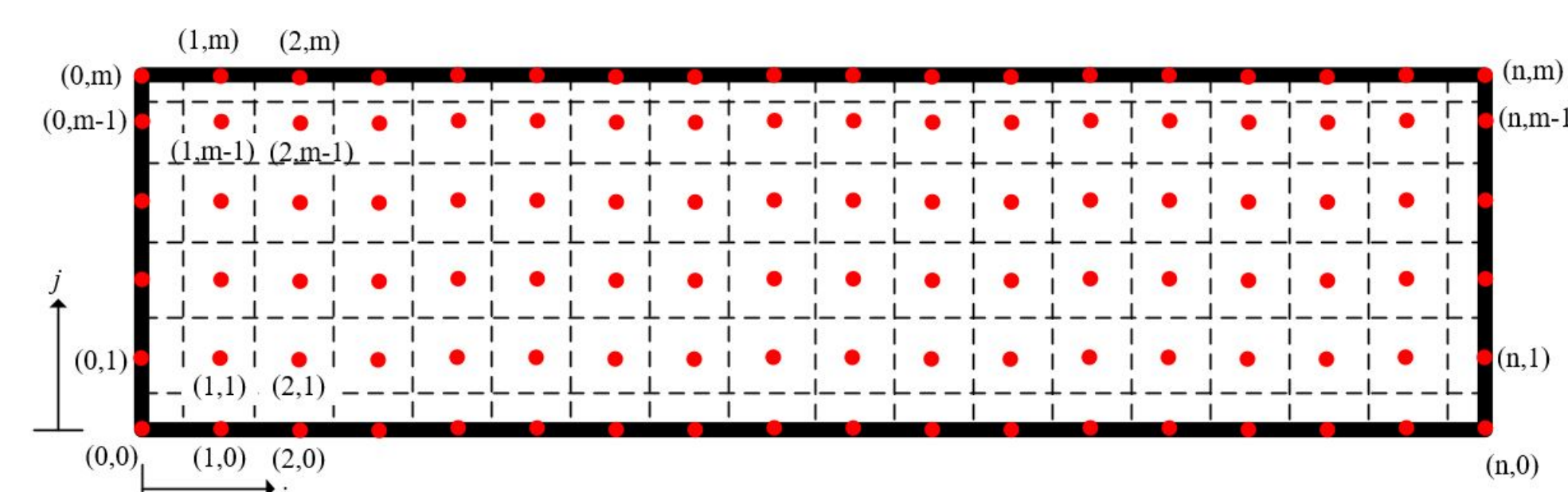


Figure 3: Meshing of the support member

### Determine stress distribution using Equation 4

### Determine temperature distribution using Equation 6

### Topology optimization (TO)

#### Initialize removal rate, RR

#### Specify evolution rate, ER

#### Compute stress ratio, $SR = \frac{\sigma}{\sigma_{\text{max}}}$ at all mesh points

#### For $\sigma_{\text{max}} < \sigma_{\text{ultimate}}$ , compare SR and RR at each discretized control volume. Remove material if $SR < RR$

#### If $\sigma_{\text{max}} = \sigma_{\text{min}}$ then update RR as $RR^{(l)} = RR^{(l-1)} + ER$

#### Continue until $\sigma_{\text{max}} > \sigma_{\text{ultimate}}$

### Determine heat inleak and mass of support member initially and after topology optimization

## Results and Discussion

### Operating Conditions

Length of the support (mm)	300	Pre-tension load (N)	10000
Diameter of the support (mm)	54	Bending load (N)	10000
Ambient temperature (K)	300	Material of construction	SS 304
Temperature of the cryogen (K)	70	Density of SS 304 (kg/m <sup>3</sup> )	8000

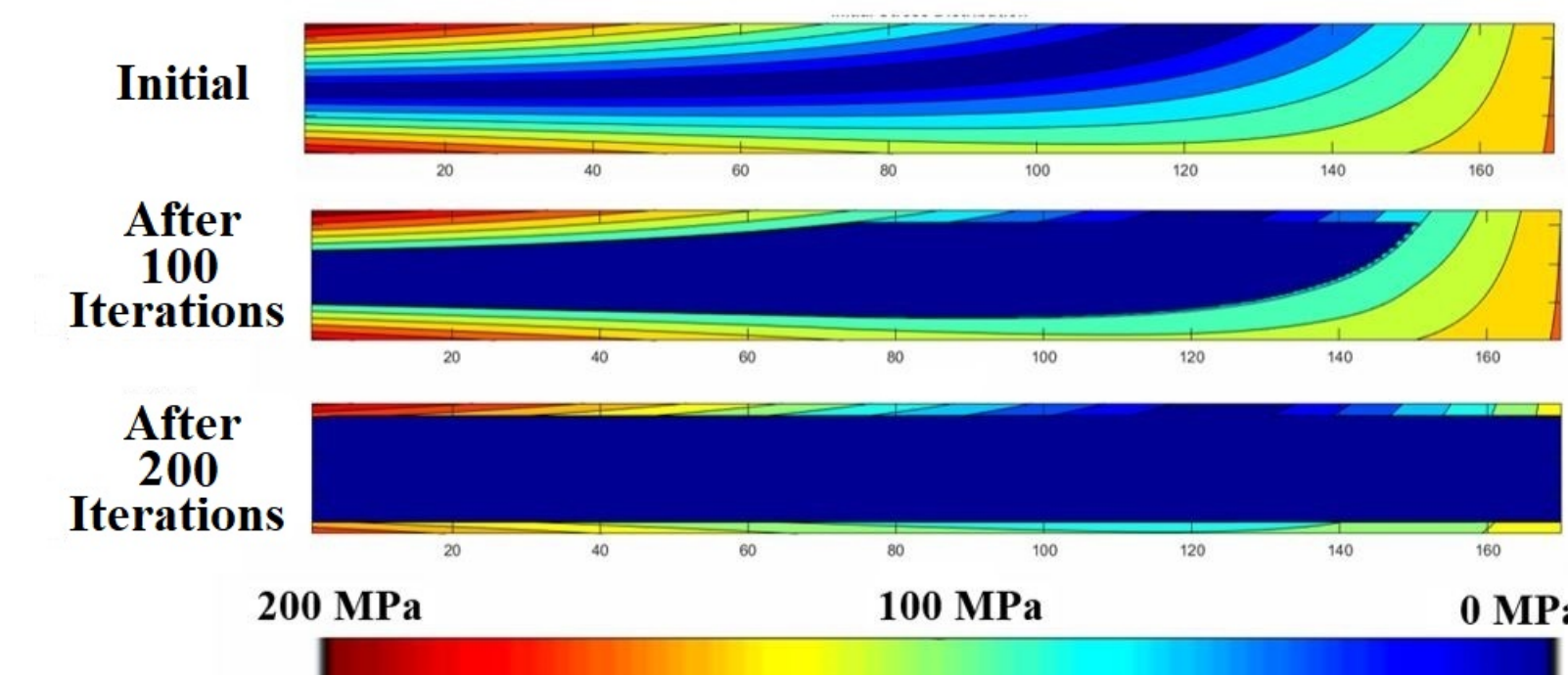


Figure 4: Development of stress field in the support member

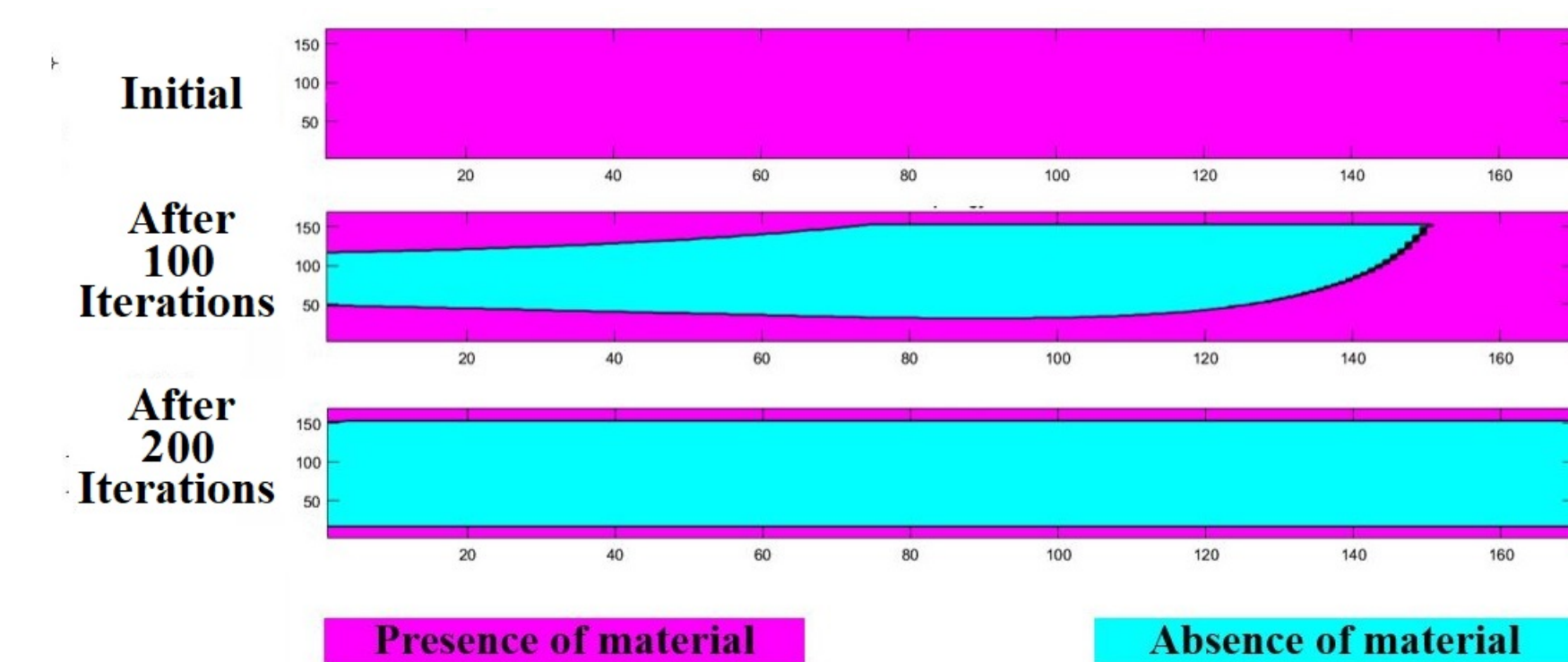


Figure 5: Evolution of topology of the support member

### Optimized values

	Mass	Heat inleak
Before TO	5.49 kg	6.532 mW
After TO	1.92 kg	2.353 mW

## Conclusions

- Topology optimization has been used to arrive at an optimized geometry for a dewar support system
- A reduction of 65% in mass and 64% in heat inleak in the optimized shape has been obtained

## Selected References

- [1] B Nitin and Pavitra Sandilya. Structural and thermal analysis of dewar supports for low boil-off long duration storage of cryogenic liquids. *Indian Journal of Cryogenics*, 43(1):33–39, 2018.
- [2] Denghong Xiao, Xiandong Liu, Wenhua Du, Junyuan Wang, and Tian He. Application of topology optimization to design an electric bicycle main frame. *Structural and Multidisciplinary Optimization*, 46(6):913–929, 2012.

## Nomenclature

$\sigma_{\text{bending}}$	Stress due to bending	$M$	Moment	$I$	Moment of Inertia	$x, y$	Directions
$\sigma_{\text{pre-tension}}$	Stress due to pre-tension	$A$	Area of cross section	$E$	Young's Modulus	$\alpha$	Thermal expansion coefficient
$\sigma_{\text{contraction}}$	Stress due to thermal contraction	$\sigma$	Total Stress	$\Delta T$	Temperature difference	$k$	Thermal conductivity
$\sigma_{\text{ultimate}}$	Ultimate strength of the material	$\sigma_{\text{max}}$	Maximum stress	$\sigma_{\text{min}}$	Minimum Stress	$l$	Iteration count

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