

Hydrostatics and Hydrodynamics of the Tank

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Contents of the Presentation

- Hydrostatic Analysis
- Modal Analysis
- Mathematical Modelling of Sloshing



Static Analysis

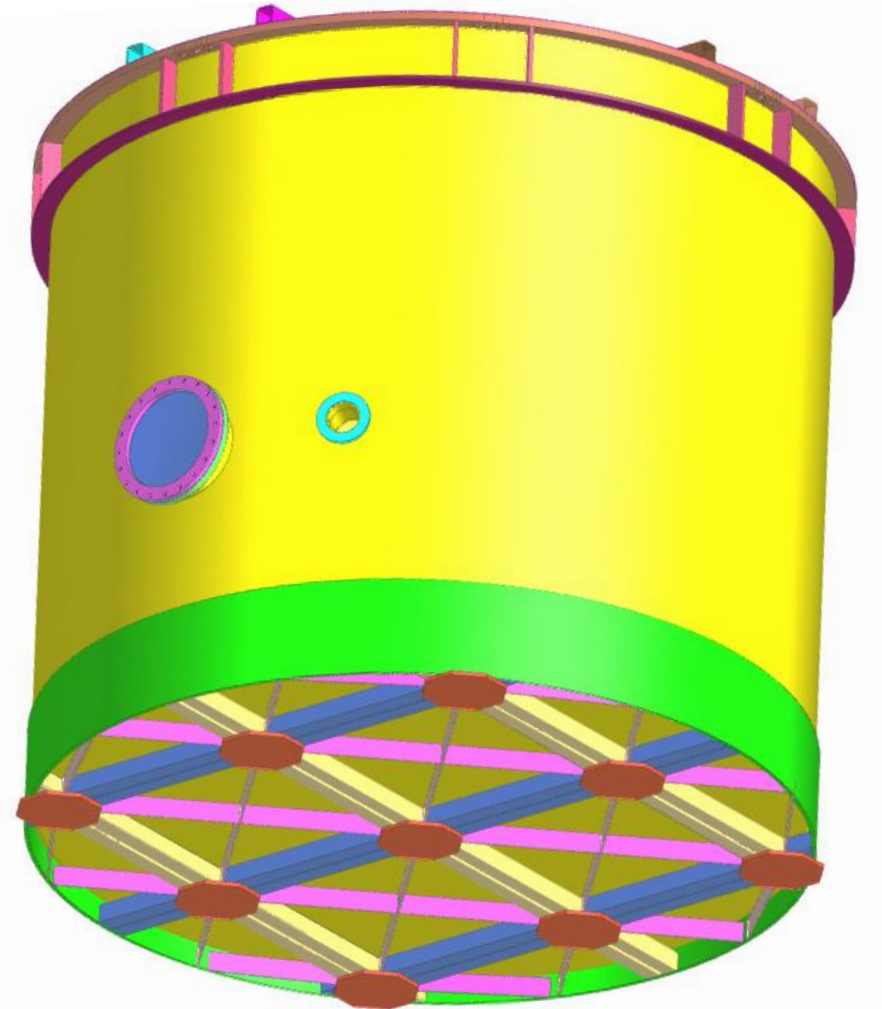
Boundary Condition

The bottom plates are restricted to all dofs.

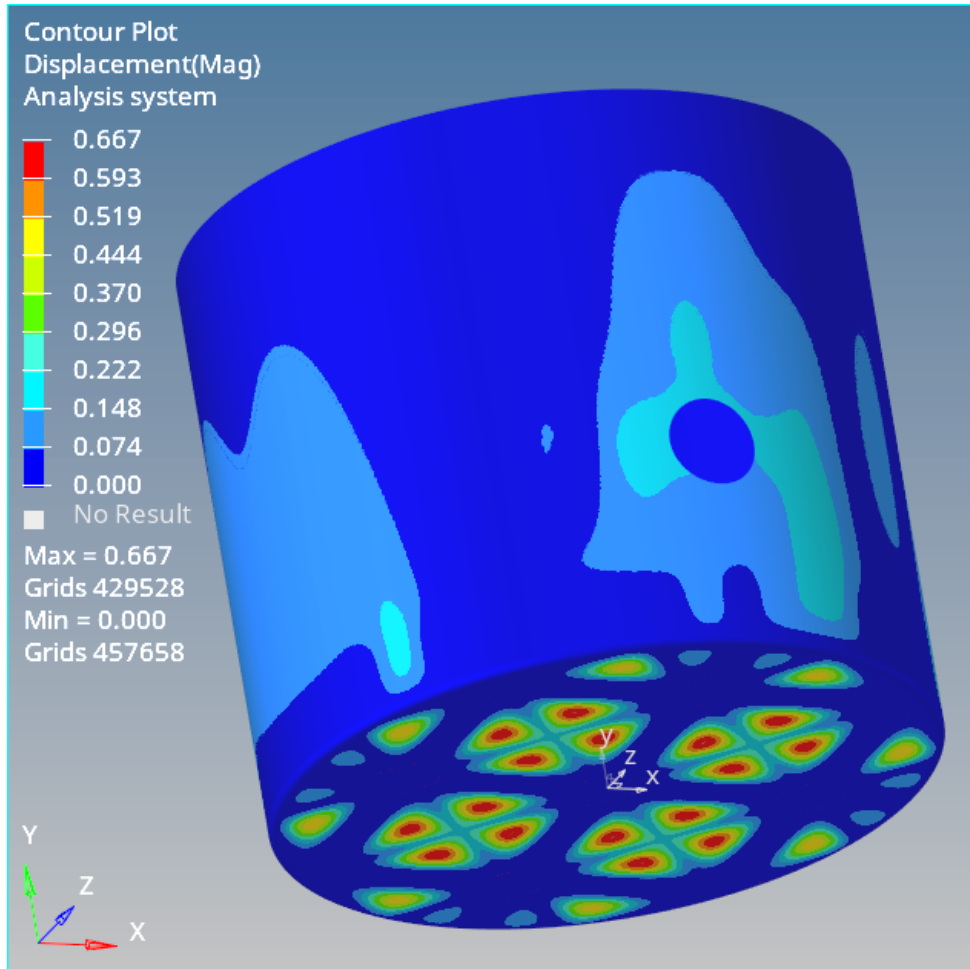
Loads

Self wt.(Gravity)+ Hydrostatic Pressure on tank walls
(ht. of water =3.4m) +100 kg (CDS) on top rafters

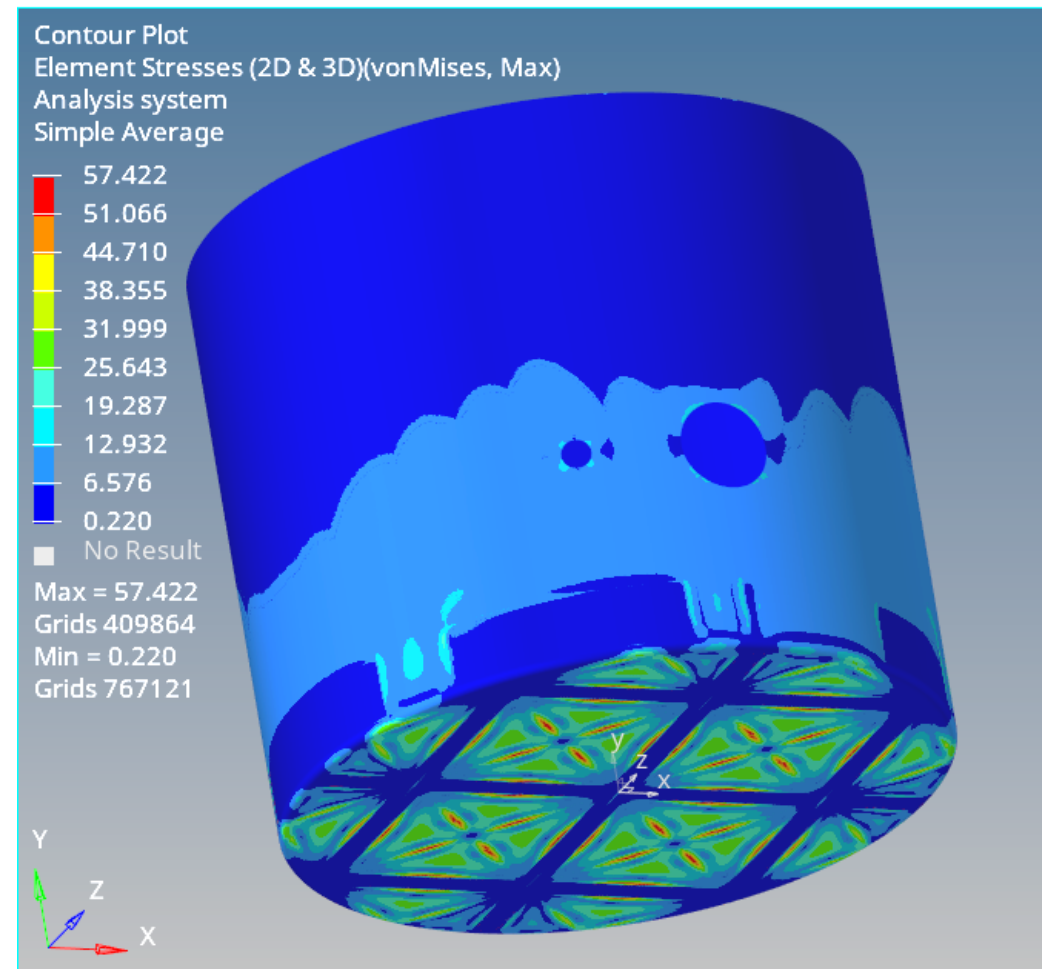
- Wt. of Water = 38.50 tons
- Ht. of Water level = 3400mm
- Tank Wall Thickness = 6mm



Tank Results

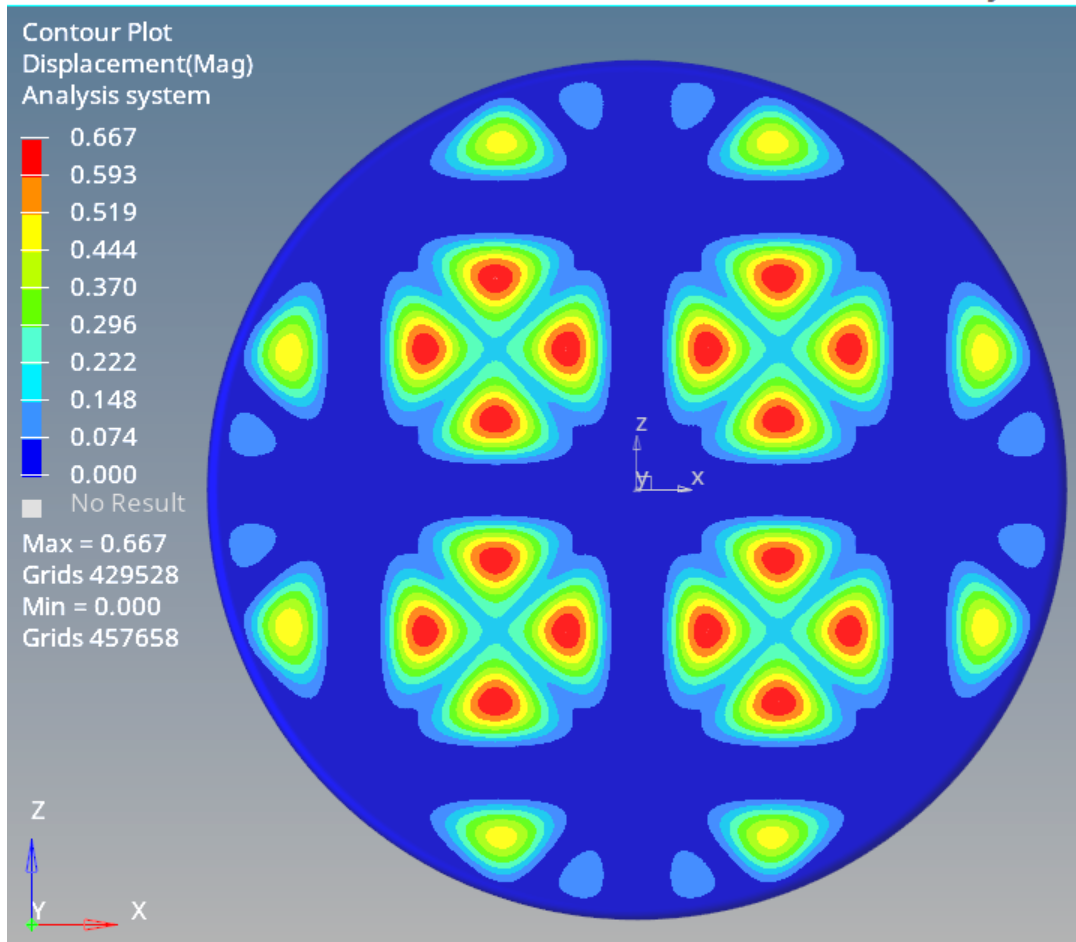


Max. Displacement = 0.67 mm

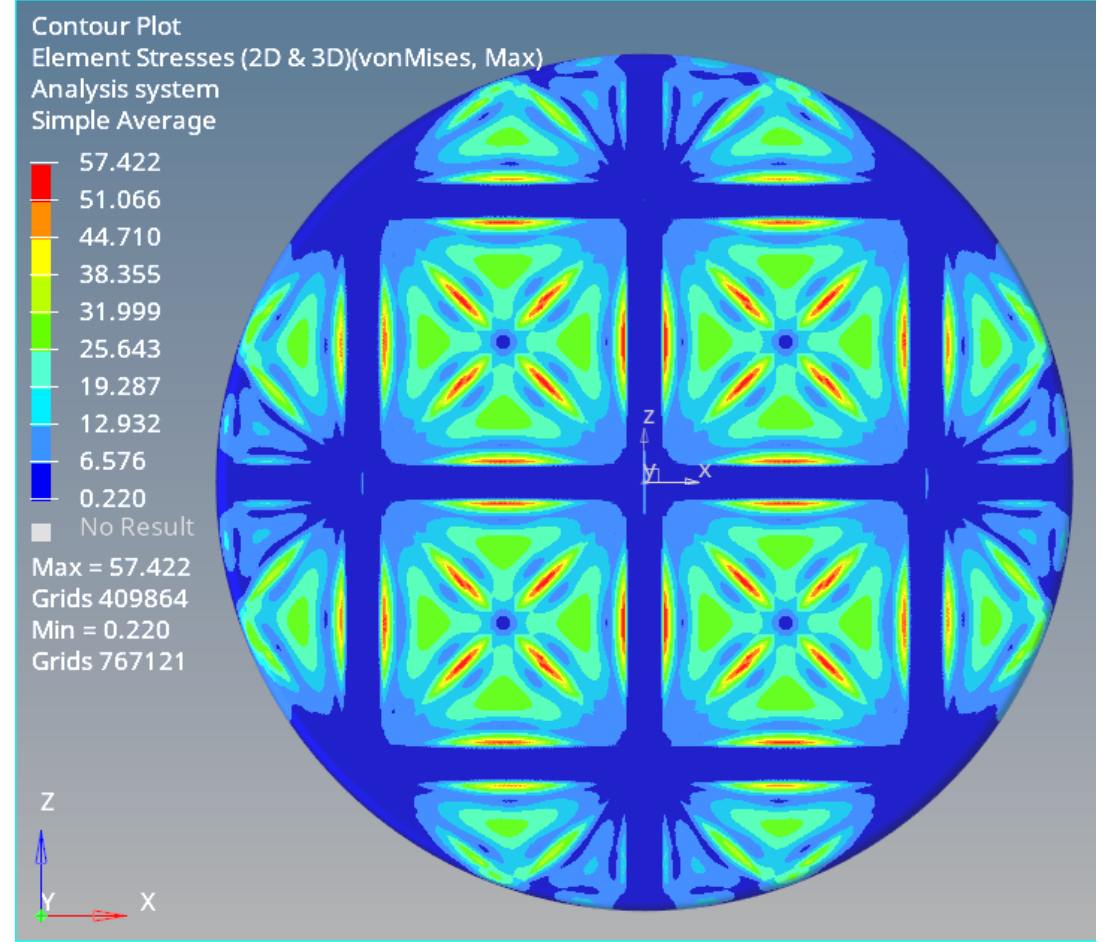


Max. Stress = 57.42 MPa

Tank Results

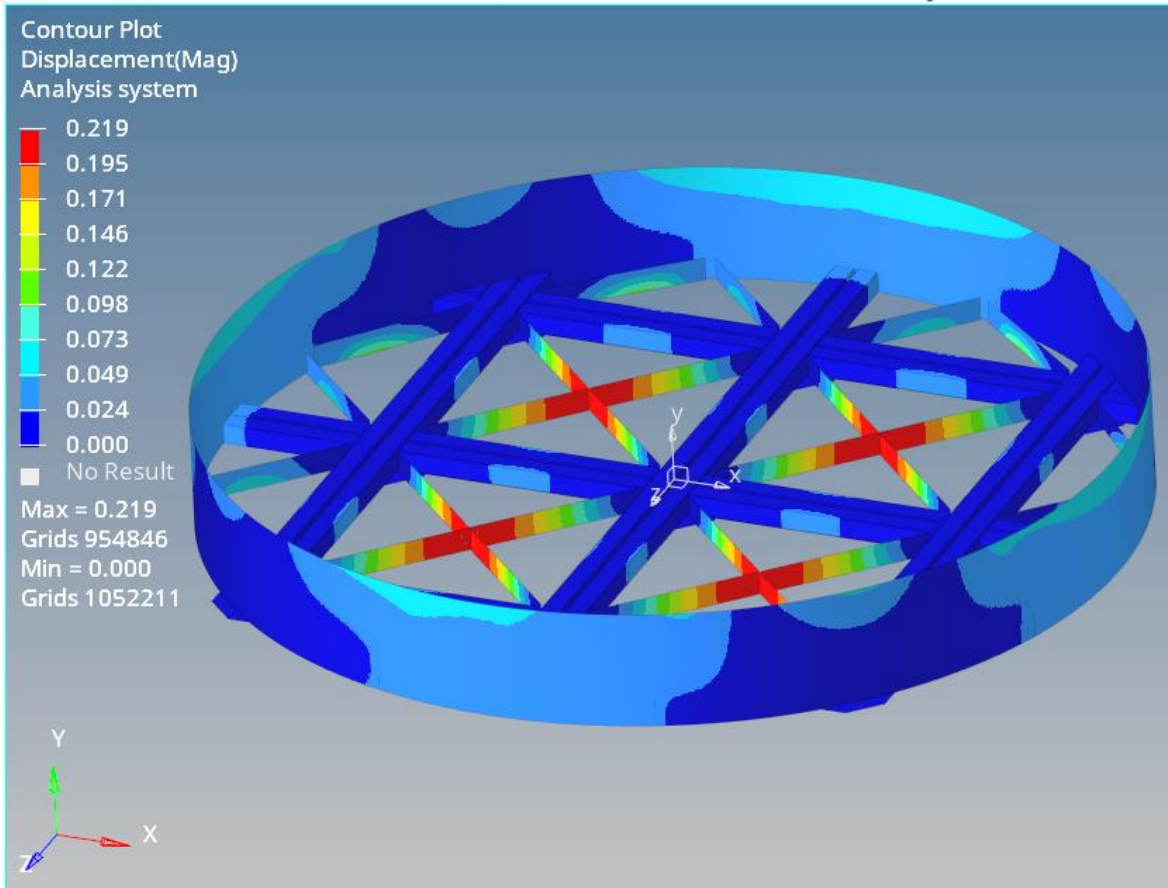


Max. Displacement = 0.67 mm

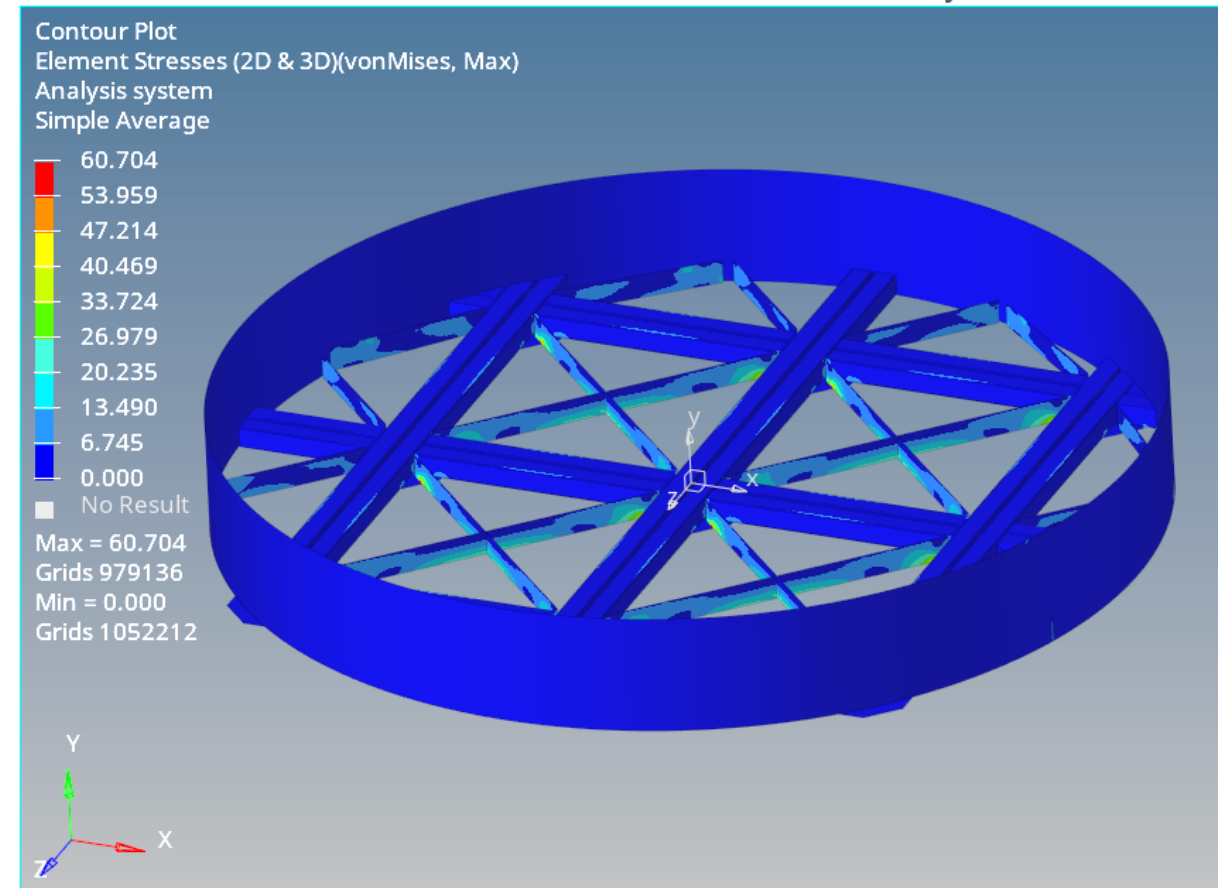


Max. Stress = 57.42 MPa

Base Results

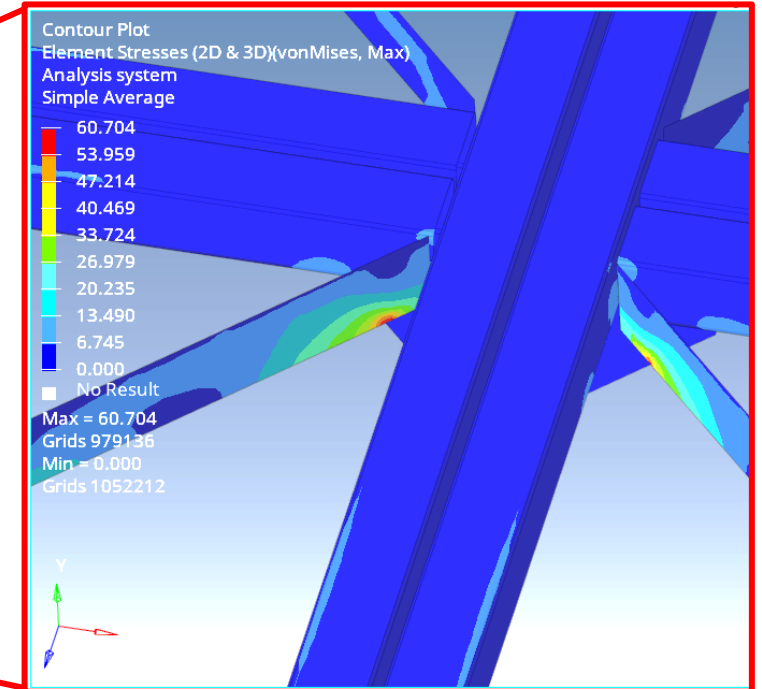
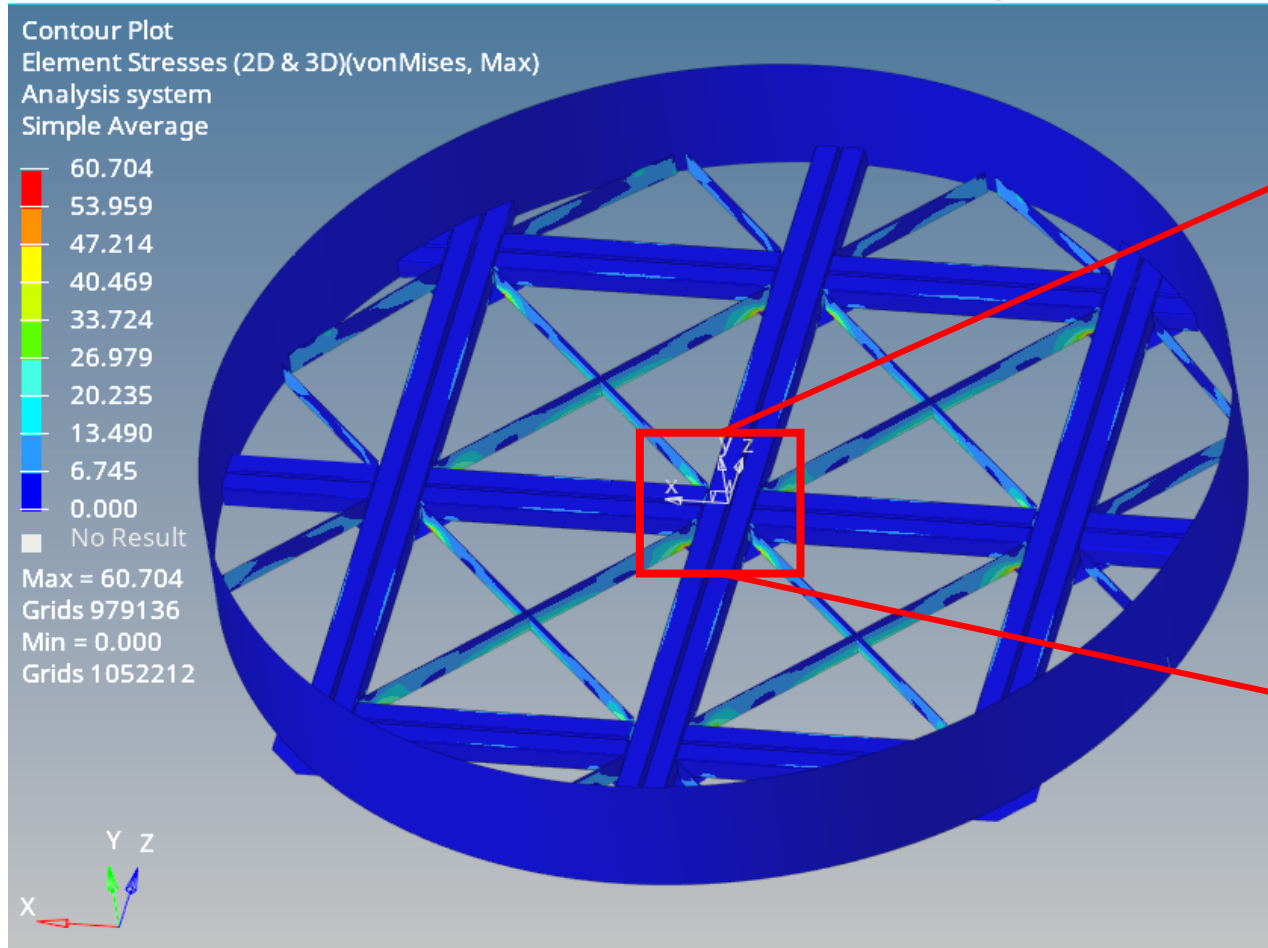


Max. Displacement = 0.22 mm



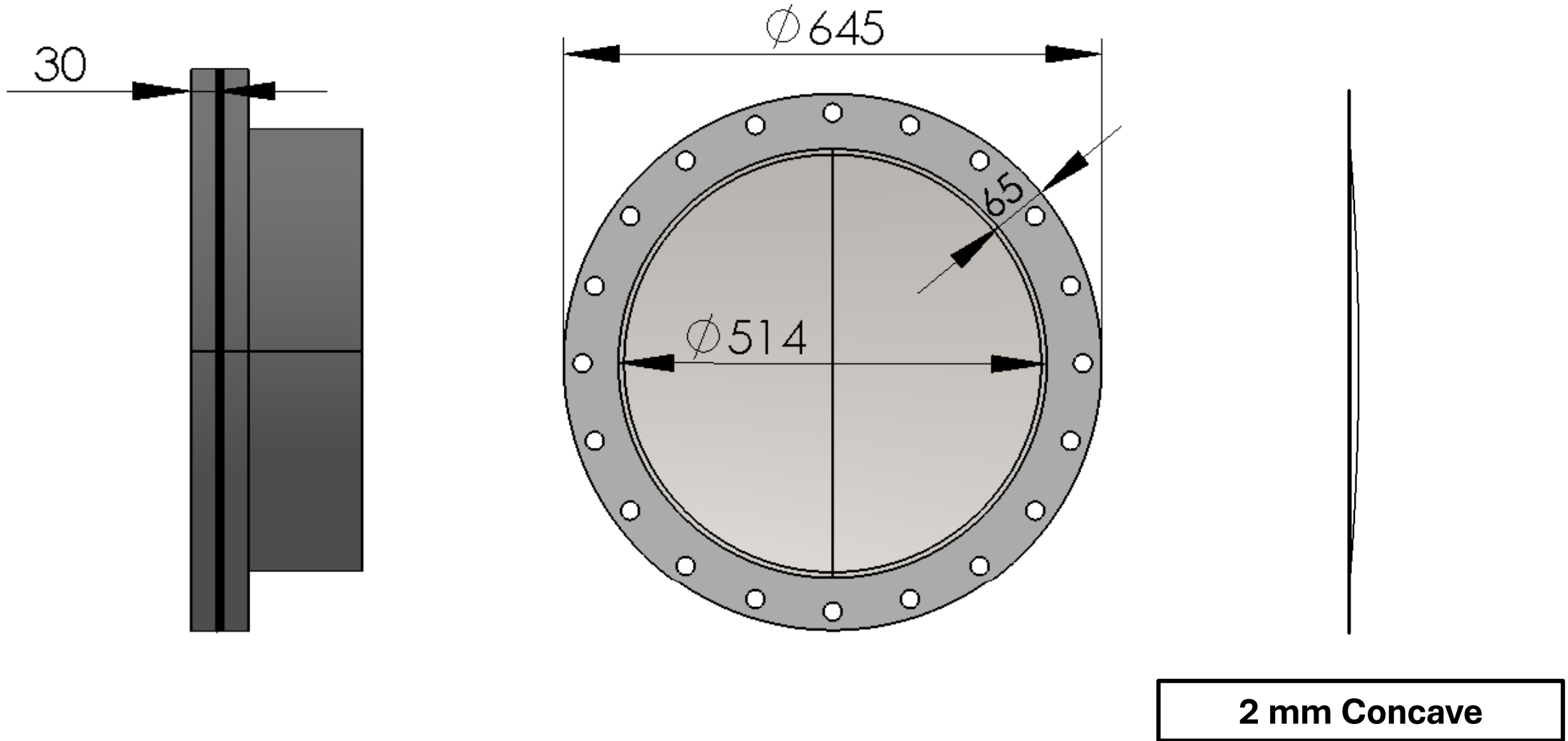
Max. Stress = 60.70 MPa

Base Results

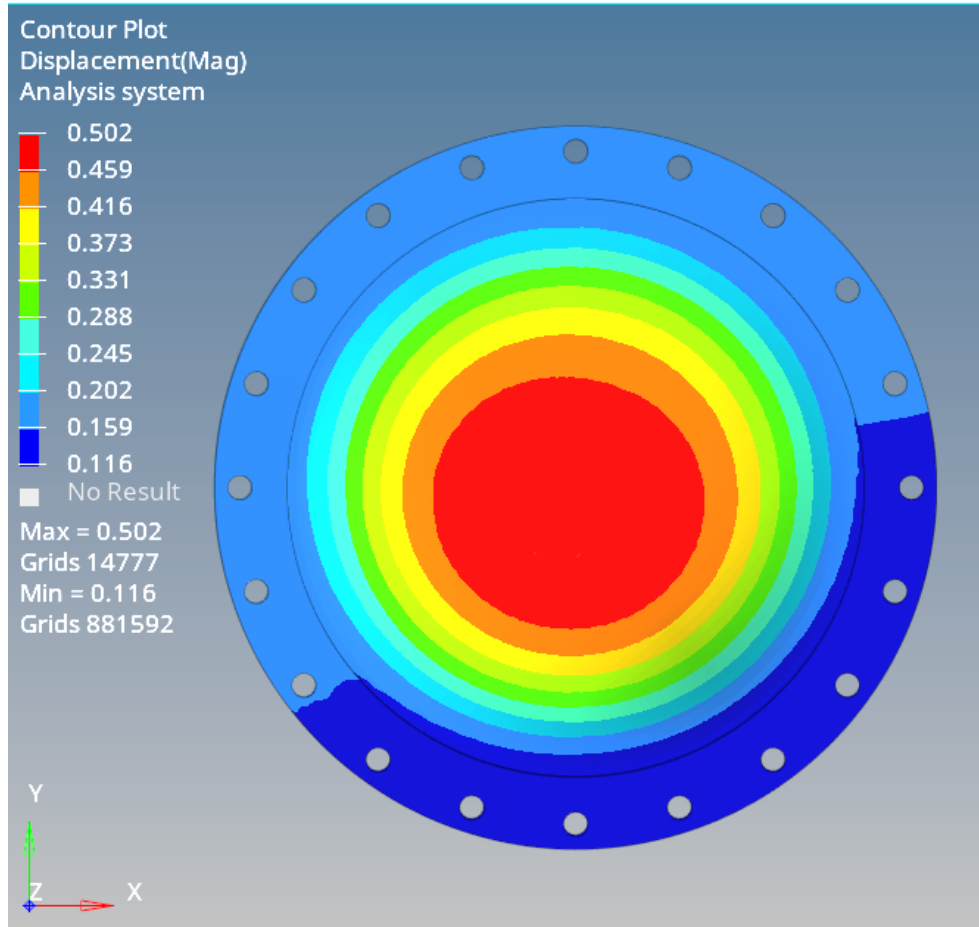


Max. Stress = 60.70 MPa

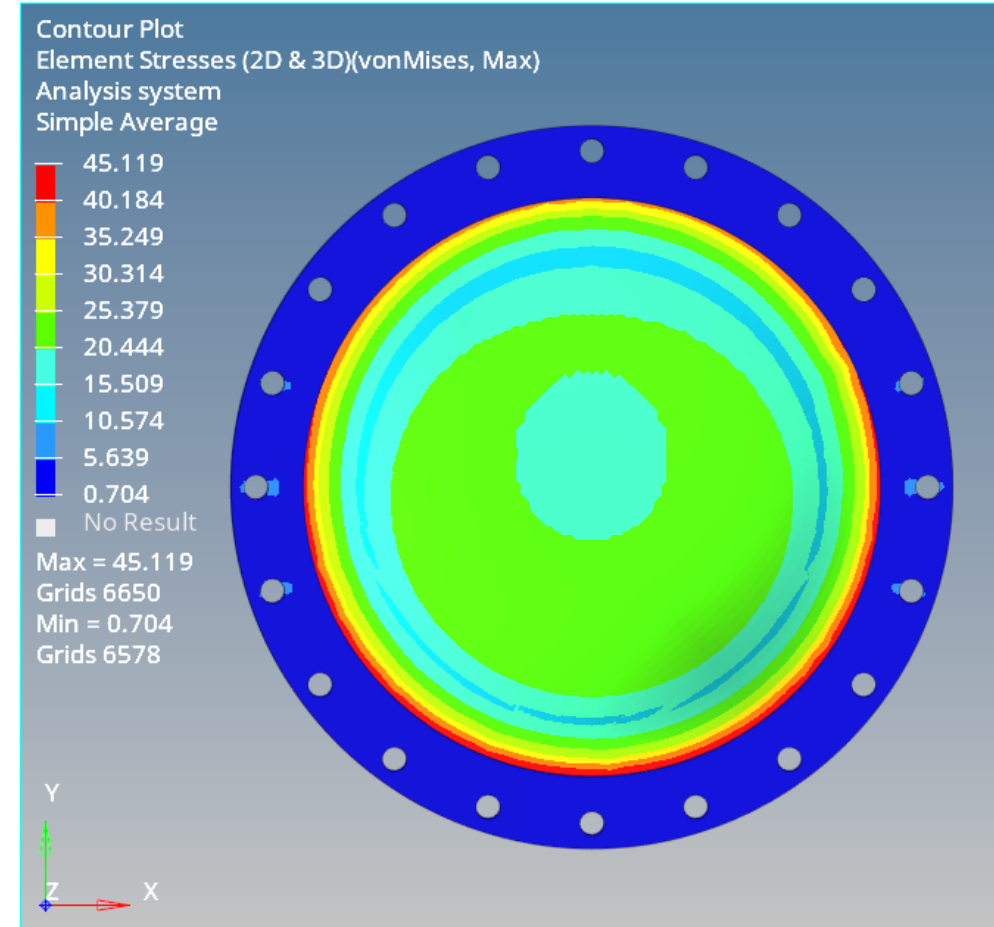
Beam Window Details



Tertiary Beam Window



Max. Displacement = 0.50 mm



Max. Stress = 45.20 MPa

Conclusion

From the results we can infer that

- The maximum stress and displacement due to Hydrostatic Pressure on the Tank assembly occurs on the Tank base of **60.70 MPa** .
- The maximum displacement is obtained on the tertiary beam i.e., **0.50mm**
- In the study, the model appears to be safe by the factor of safety **3.46** (w.r.t to yield stress of 207 MPa)

Tank		Tertiary Beam Window		Base Mesh		Factor of Safety
Max. VonMises stress (MPa)	Max. Displacement (mm)	Max. VonMises stress (MPa)	Max. Displacement (mm)	Max. VonMises stress (MPa)	Max. Displacement (mm)	
57.42	0.67	45.20	0.50	60.70	0.22	3.46

Modal Analysis

Boundary Condition

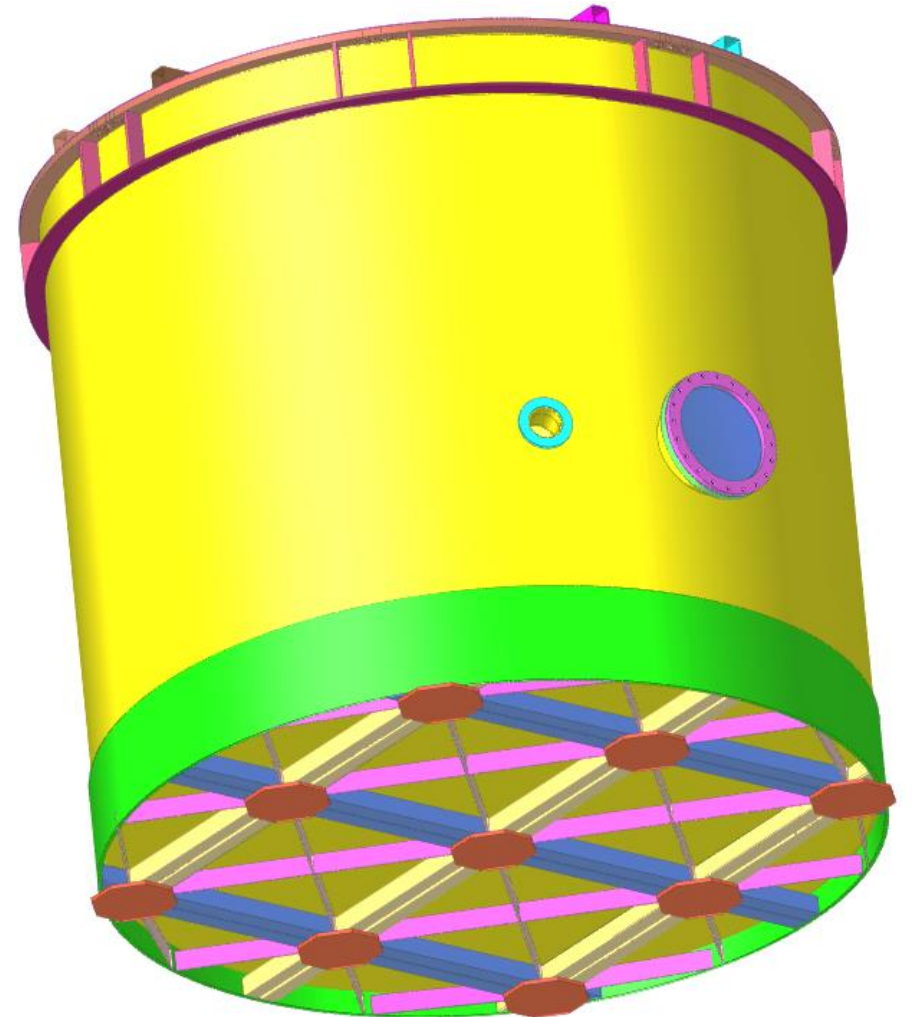
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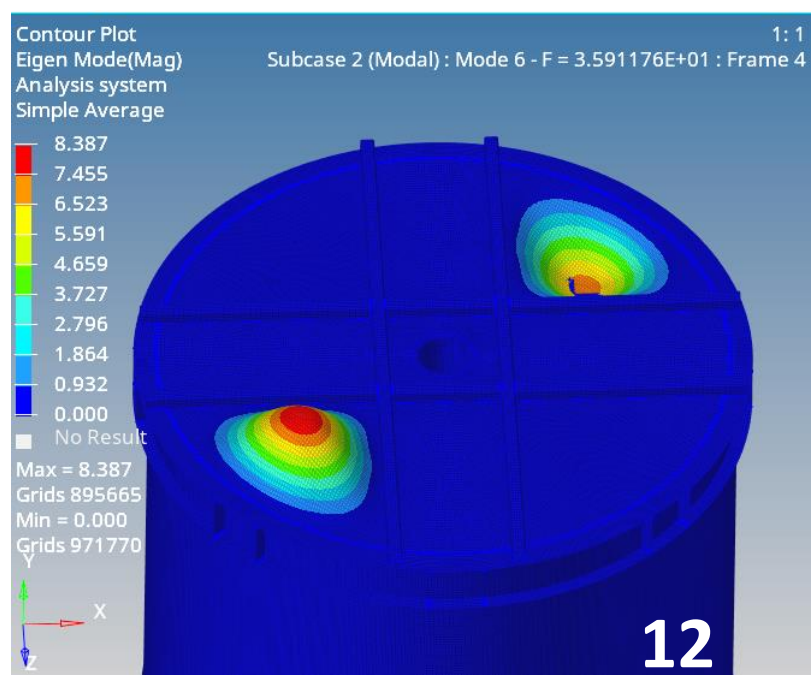
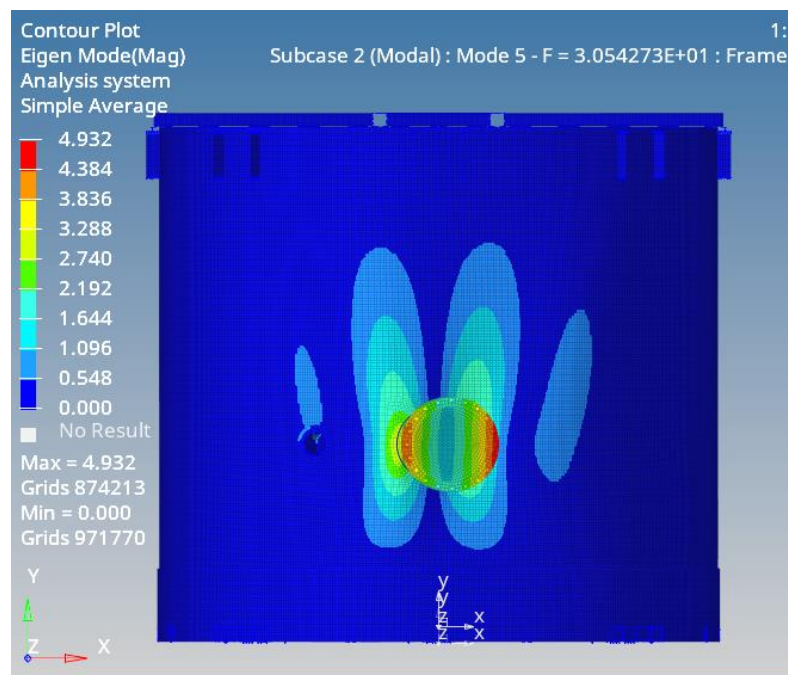
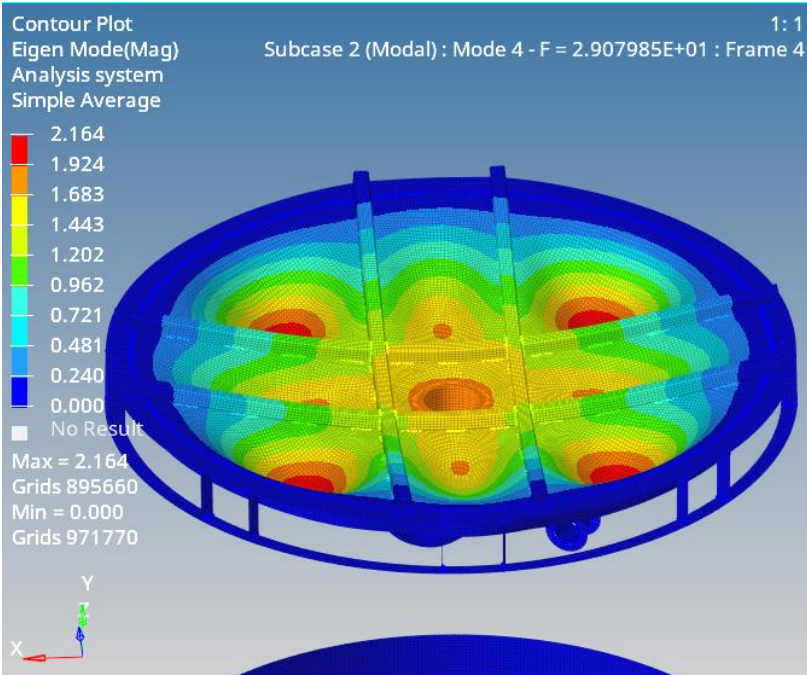
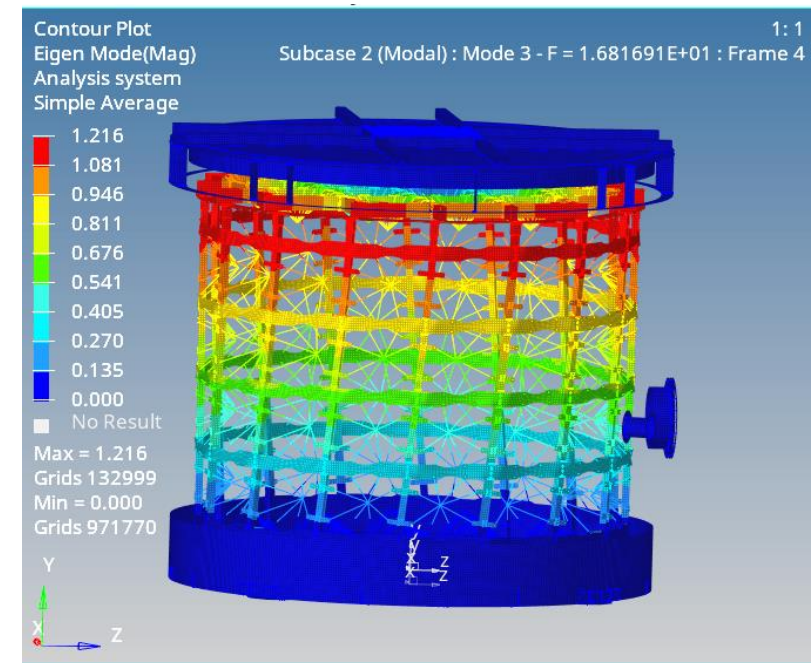
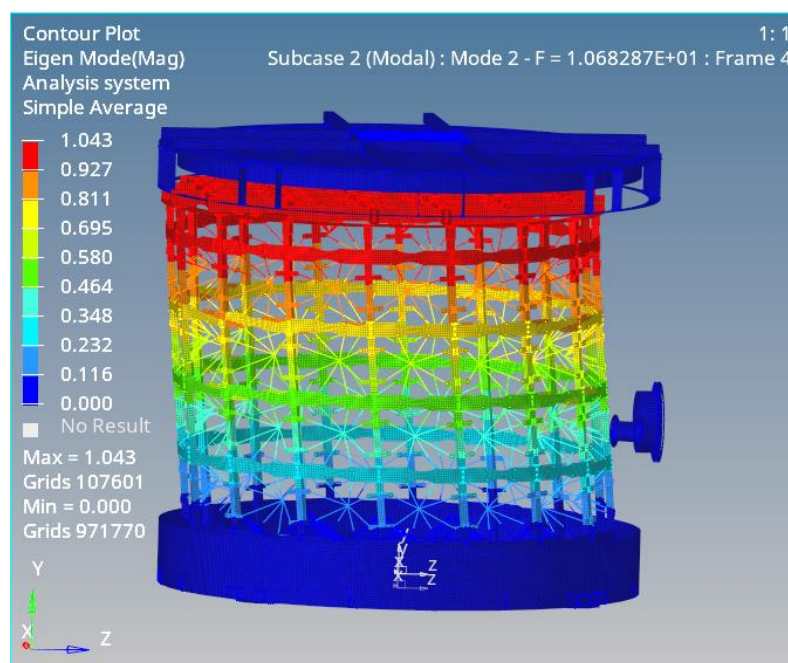
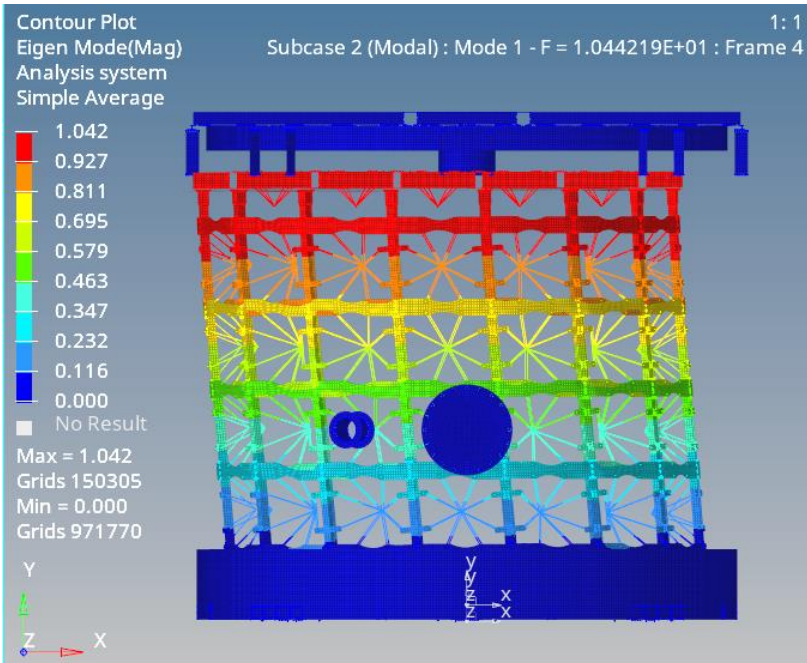
Loads

Self wt.(Gravity)+ Hydrostatic Pressure on tank walls
(free surface ht. =3.4m) +100 kg (CDS) on top rafters

Analysis up to 50 Natural Modes

Structure and Tank are not coupled in this Study





Results

MODAL PARTICIPATION FACTORS FOR SUBCASE 3
 RIGID BODY MODES BASED ON REFERENCE POINT AT ORIGIN OF BASIC COORDINATE SYSTEM

Mode	Frequency	X-TRANS	Y-TRANS	Z-TRANS	X-ROTAT	Y-ROTAT	Z-ROTAT
1	1.044E+01	-1.077E+00	3.128E-07	5.367E-02	1.464E+02	1.731E-01	2.888E+03
2	1.068E+01	-5.391E-02	9.424E-05	-1.068E+00	-2.878E+03	-1.190E-01	1.463E+02
3	1.682E+01	7.458E-05	2.006E-03	-4.195E-05	-1.119E-01	1.656E+03	-2.466E-01
4	2.872E+01	-9.483E-02	-3.873E-02	3.779E-02	5.492E+01	-2.873E+02	5.444E+01
5	2.908E+01	5.511E-03	-8.250E-01	9.631E-03	2.172E+01	1.516E+01	-3.505E+00
6	3.505E+01	2.656E-02	-1.340E-02	2.890E-01	6.916E+02	1.880E+01	-5.038E+01
7	3.591E+01	-2.413E-02	2.247E-03	4.619E-04	2.521E+02	-1.622E+00	3.417E+02
8	3.607E+01	1.763E-02	3.327E-04	1.722E-03	2.932E+02	-3.510E-01	-2.911E+02
9	3.613E+01	-5.311E-03	-4.411E-03	-3.133E-04	7.264E-01	-8.039E-01	7.323E+01
10	3.669E+01	1.778E-03	-3.551E-03	-3.214E-04	4.105E-01	-1.148E+01	-8.333E-02

Critical frequency of tank with structure is:

X dir. = 10.44 Hz

Y dir. = 29.08 Hz

Z dir. = 10.68 Hz

These frequencies will be required to perform the water sloshing analysis

Mathematical Model for Water Sloshing

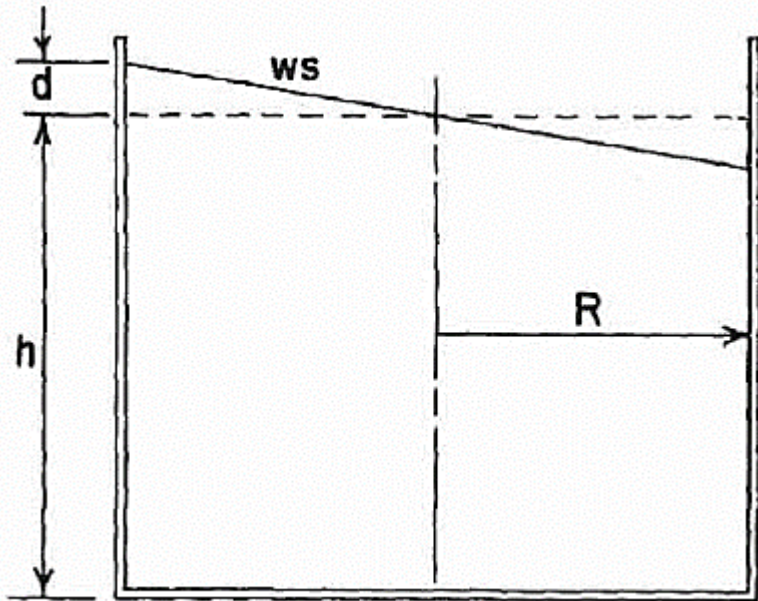


FIG. 1a

$$A_1 = v * \frac{T}{2\pi}$$

$$d = \frac{0.63 A_1 \left(\frac{k_1 R}{M_1 g} \right)}{1 - 0.85 \frac{A_1}{R} \left(\frac{k_1 R}{M_1 g} \right)^2}$$

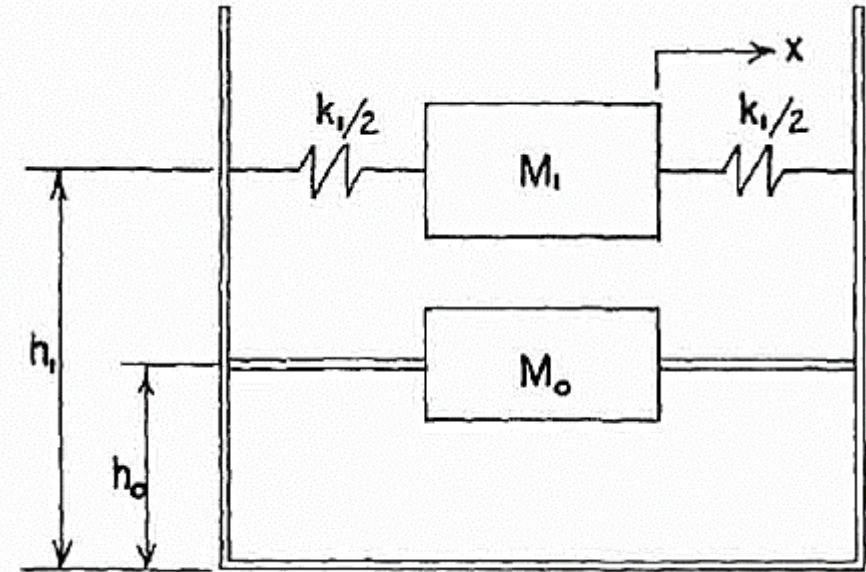


FIG. 1b

$$M_1 = M(0.6) \frac{\tanh 1.8 h/R}{1.8 h/R}$$

$$k_1 = 5.4 \frac{M_1^2 g h}{M R^2}$$

$$T_w = 2\pi \sqrt{\frac{M_1}{k_1}} = \text{a period of vibration}$$

M = total mass of water in the tank

$$M_1 = 7130 \text{ Kg}$$

$$k_1 = 65.74 \text{ kN/m}$$

$$T_w = 2.06 \text{ s}^*$$

$$\text{Natural slosh frequency } (f_w) = 0.5 \text{ Hz}^*$$

*For first mode of vibration

Wave Height vs Velocity

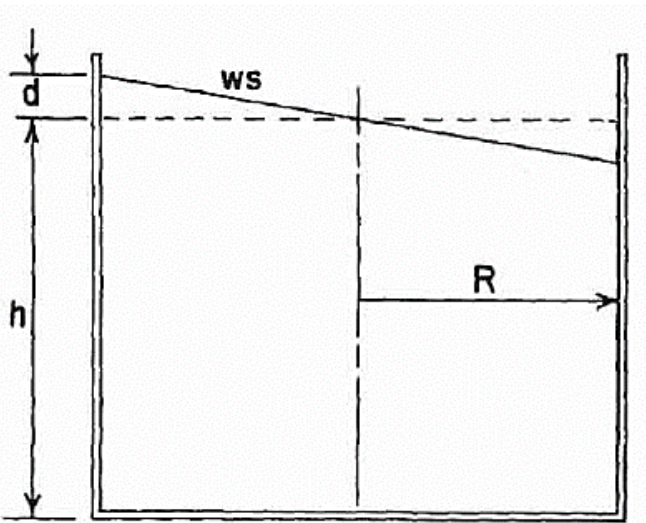
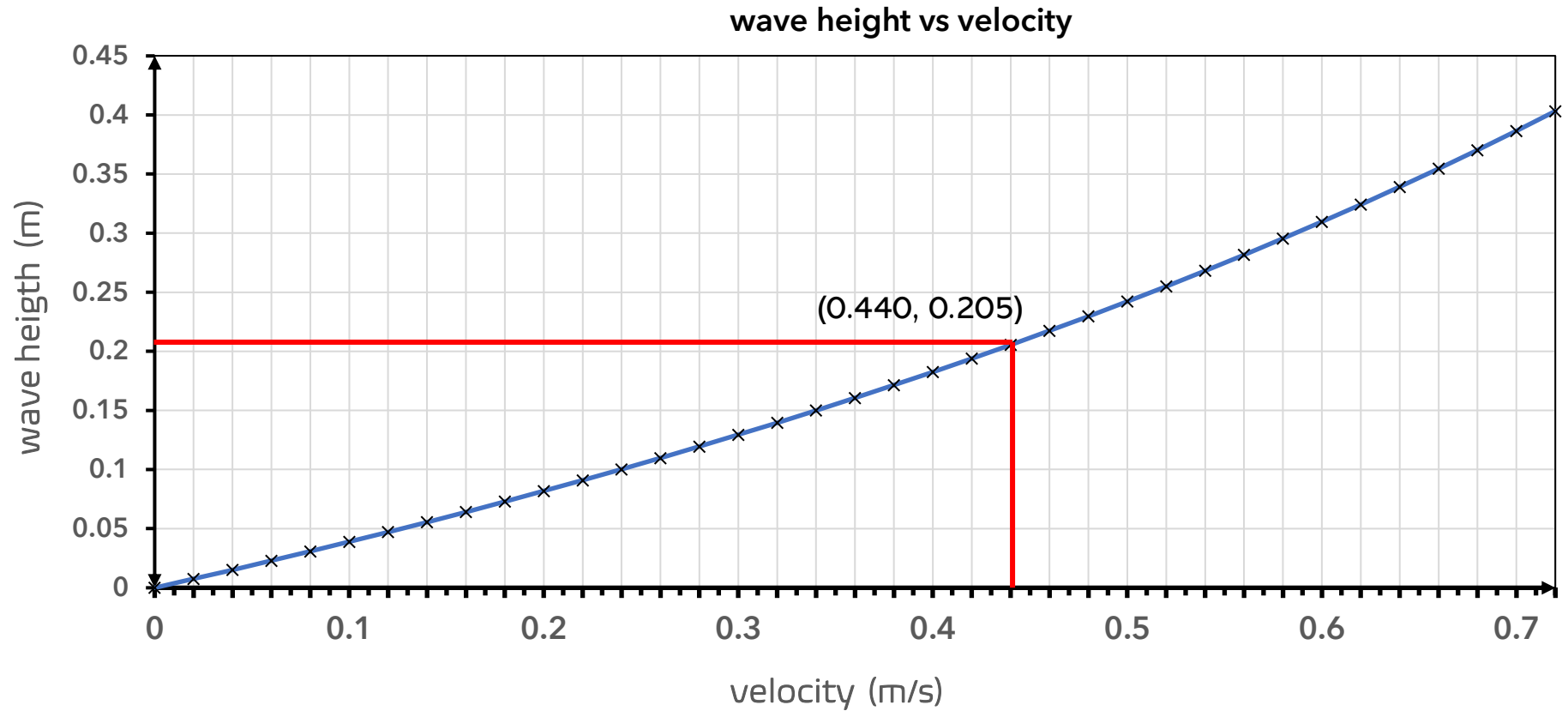
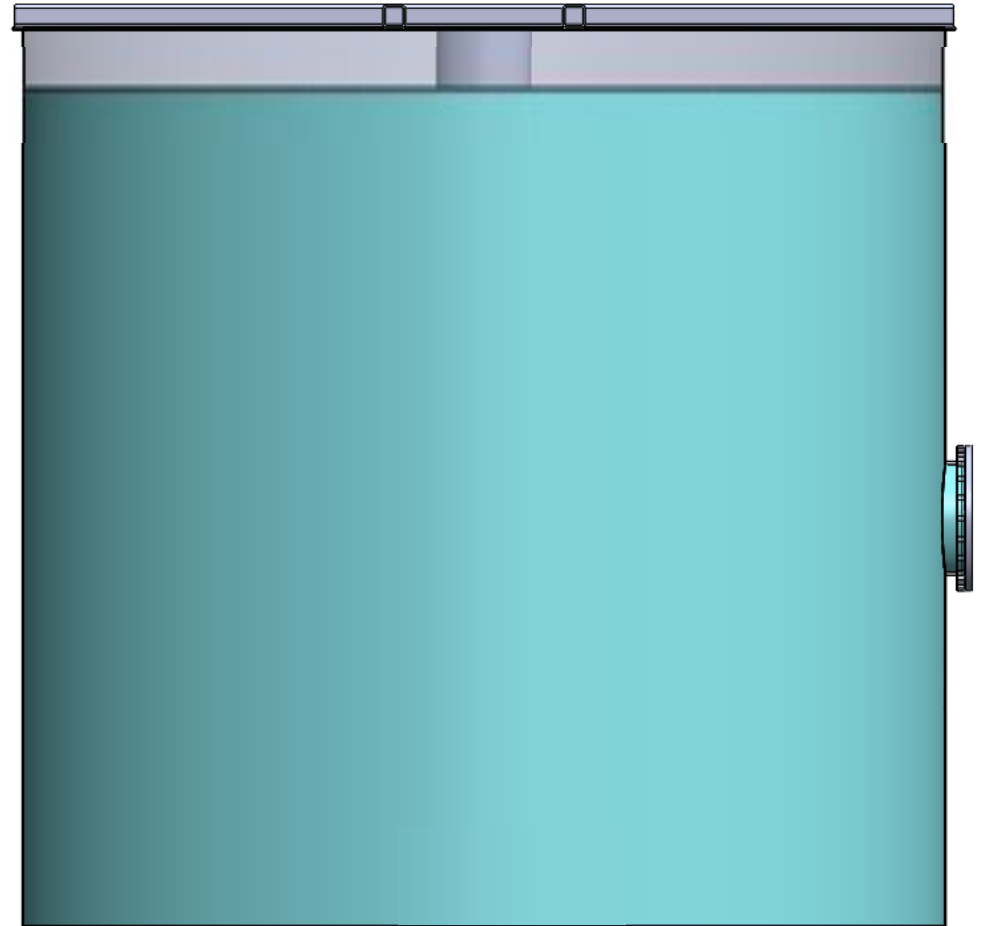


FIG. 1a



Conclusion

- The natural frequency of the water sloshing of the designed tank is **0.5 Hz**.
- The velocity of water corresponding to wave height of 200mm is **0.44 m/s**.
- The initial acceleration of the tank to attain velocity of water 0.44 m/s is under study.



Thank You
