The CLIC Two-Beam Accelerator Module & Requirements

Andrea Latina

for the CLIC Group

This presentation has been prepared with the help of

G. Riddone, W. Wuensch, I. Syratchev, A. Gridiev, M. Modena, H. Mainaud Durand

Outline

Two-Beam Module Description

Module Design Process

Module Subsystems: Status, Specifications and Requirements

Introduction

- CLIC is based on the two-beam acceleration concept
- To facilitate the matching of the two beams, all components are assembled in 2m long modules of few different types. The modules make use of girders to meet the pre-alignment requirements (smoothing of adjacent girders)
- Since each linac length is about 21 km, more than 10,000 modules are needed per linac
- The Module defines boundary conditions for technical solutions for important systems: AS, PETS, RF distribution network, stabilization, alignment, vacuum, ... and has to cope with the challenging requirements from each of them
- It's a cost driver for CLIC, as well as a performance driver



Two-Beam Module Design

- Longitudinal Requirements
 - 1. Physics -> Beam Dynamics <-> Acc. Structures Design
 - 2. PETS Design and Optimization
 - 3. Module Layout
- Transverse Requirements
 - 1. Stabilization
 - Instrumentation
 - 3. Cooling
 - 4. Vacuum, ...

Two-Beam Module Design Chart

Luminosity per linac input power

$$\frac{L}{P_l} = \frac{L_{bx}N_bf_{rep}}{eE_cNN_bf_{rep}} = \frac{1}{eE_{cm}} \cdot \frac{L_{bx}}{N} \eta$$



PETS design

- Drive beam current and energy
- RF power production needs
- Beam transport and stability
- Module layout and fabrication technology
- Power extraction and transfer

BD <-> AC design

Beam dynamics (BD) constraints based on the simulation of the main linac, BDS and beam-beam collision at the IP:

- N bunch population depends on $\langle a \rangle/\lambda$, $\Delta a/\langle a \rangle$, f and $\langle E_a \rangle$ because of short-range wakes
- N_s bunch separation depends on the long-range dipole wake

RF breakdown and pulsed surface heating constraints



AS cells



Two-Beam Module Layout



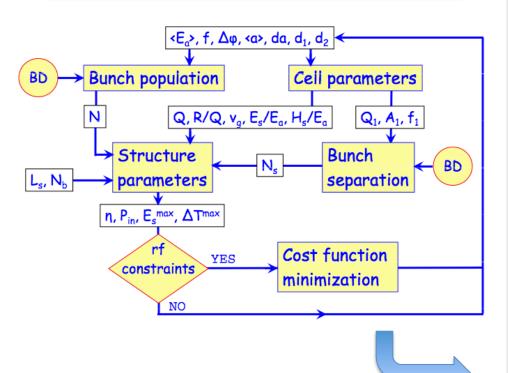
Pre-assembled PETS

RF power extractors

(1) Accelerating Structure Optimization

CLIC Main Linac Parameters

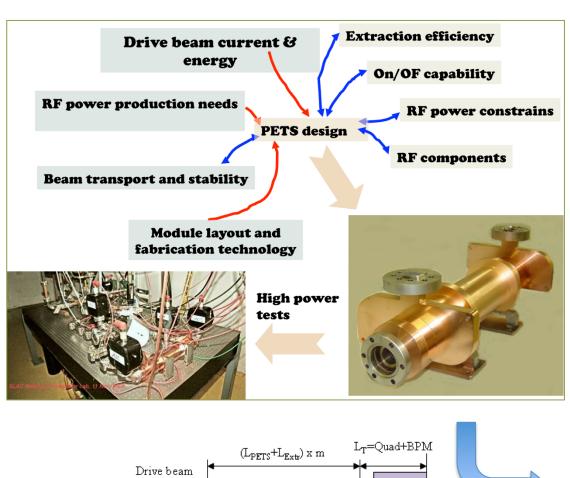
Parameter	Symbol	Value	Units
Energy CM	E _{cm}	3000	GeV
Luminosity	L _{99%}	2	10 ³⁴ cm ⁻² s ⁻¹
RF freq	f _{RF}	11.994	GHz
Repetition Rate	f _{rep}	50	Hz
Bunch population	N _b	3.72	10 ⁹

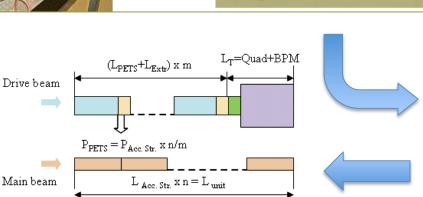


Accelerating Structure Parameters

Structure	CLIC_G
Frequency: f [GHz]	12
Gradient [MV/m]	100
Average iris radius/wavelength: <a>/λ	0.11
Input/Output iris radii: a _{1,2} [mm]	3.15, 2.35
Input/Output iris thickness: d _{1,2} [mm]	1.67, 1.00
Group velocity: v _g ^(1,2) /c [%]	1.66, 0.83
N. of reg. cells, str. length: N _c , I [mm]	24, 229
Bunch separation: N _s [rf cycles]	6
Number of bunches in a train: N _b	312
Pulse length, rise time: τ_p , τ_r [ns]	240.8, 22.4
Input power: P _{in} [MW]	63.8
Max. surface field: E _{surf} ^{max} [MV/m]	245
Max. temperature rise: ΔT ^{max} [K]	53
Efficiency: η [%]	27.7
Luminosity per bunch X-ing: L _{b×} [m ⁻²]	1.22×10 ³⁴
Bunch population: N	3.72×10 ⁹
Figure of merit: ηL _{b×} /N [a.u.]	9.1

(2) PETS Design Development Chart





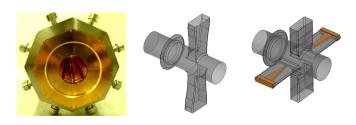


Table 3: The 12 GHz CLIC PETS parameters.

23
2.0
6.253
90
4.283
2290
0.453
7200
0.213 (34 cells)
241
101
133.7
56
0.08
1.8
1×10 ⁻⁷

Courtesy of I. Syratchev

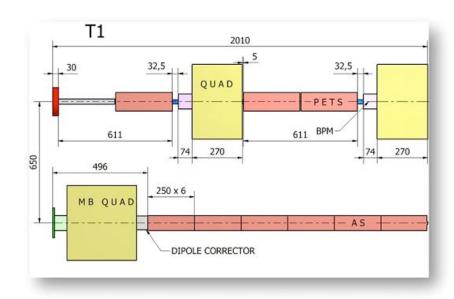
Module Parameters & Layout

Each module has a length multiple of two sets of PETS and 1 drive beam (DB) quad. Each PETS feeds 2 AS. As a result of this, the length of the CLIC Module is obtained as

$$L_{\text{module}} = L_{\text{acc}} \times 8 + L_{\text{int}} = (250 \times 8 + 10) \text{ mm} = 2010 \text{ mm}$$

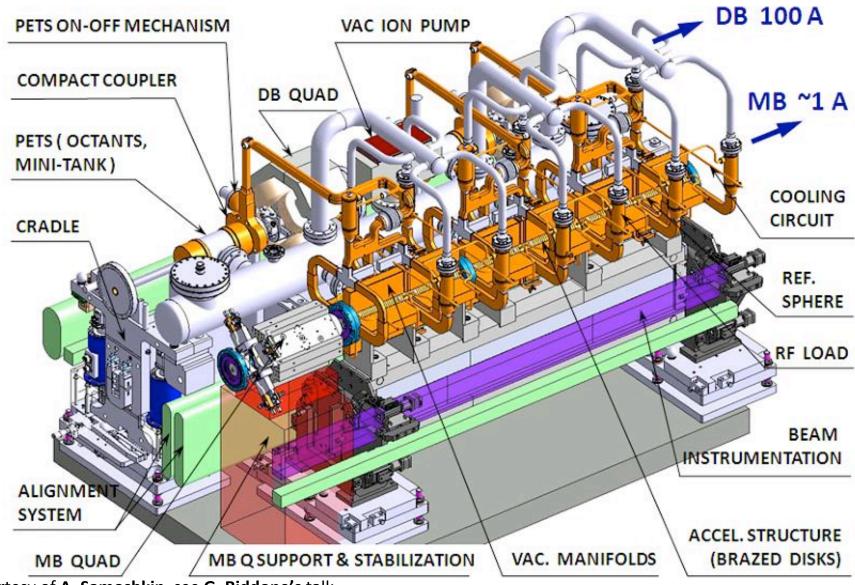
	Act. Length [mm]	Aperture [mm]	Gradient [MV/m]	Power [MW]
PETS	213	23	6.5	133.7
AS	229	5	100	63.8

Parameter	3 TeV	Unit
Module length	2010	mm
Length of PETS w/couplers	308	mm
PETS tolerance (1σ)	31	μm
Number of AS (in Module Type-I)	6	-
Length of AS	250	mm



There are five Types of Modules: "Type-?" depending on the number of MB quadrupoles

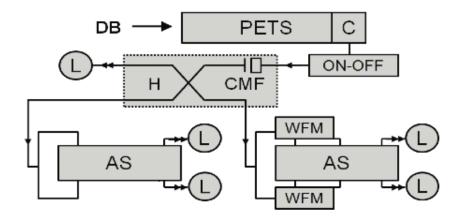
Two-Beam Module Type-I



Courtesy of A. Samoshkin, see G. Riddone's talk

Module RF Network

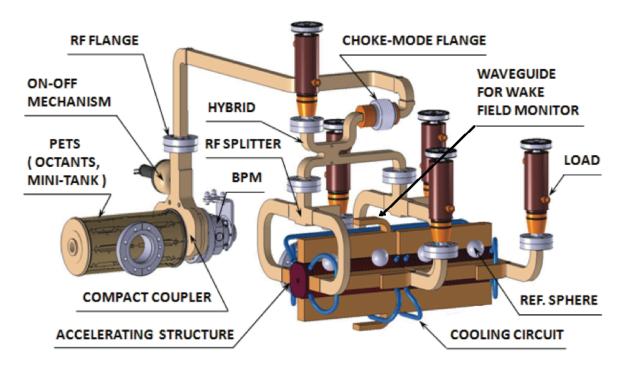
• The RF system comprises the connection between the PETS and ACS and then from the ACS to the high power loads.



CMF - choke mode flange, H - compact E-hybrid,C - compact coupler, L - rf load, WFM - wake field monitor

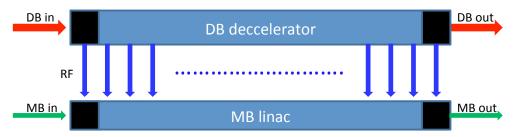
- The two-beam RF network includes the X-band rectangular wave-guides providing connection between PETS, AC and other supplementary devices.
- It's necessary to join the PETS outgoing waves by one channel. Because of the limited longitudinal space a compact coupler is needed.
- In case of a breakdown in a AS it is necessary to interrupt the power produced by the corresponding PETS within 20 ms. "On-Off" mechanism

Module RF Network



- Another requirement is to guarantee transverse alignment flexibility between the two beams and thus to allow for the power transmission without electrical contact
- Another necessity is to have a split of power between two AS without any reflection to the feeding PETS in a broad frequency range
- The power delivered from PETS to the two fed AS must be synchronized in phase

Main Beam RF Diagnostics

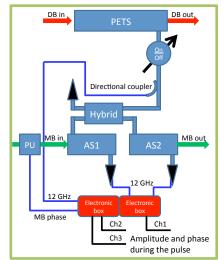


General layout in a DB Sector (~800m)

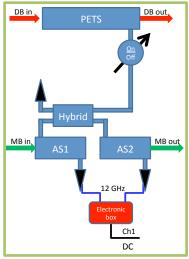
Two types of PETS+2AS units will be installed:

- 1. Reference **(black)** (2 units at the beginning and 2 units at the end of a DB sector)
 - It will have more signals and with higher resolution
 - The signals will be time resolved: dt ~ 0.5 ns (pulse shape)
- Regular (blue) (all the rest)
 - It will have 1 or 2 signals
 - Integral over the pulse (1 or 2 numbers per pulse)

PETS+2AS unit: Reference



PETS+2AS unit: Regular



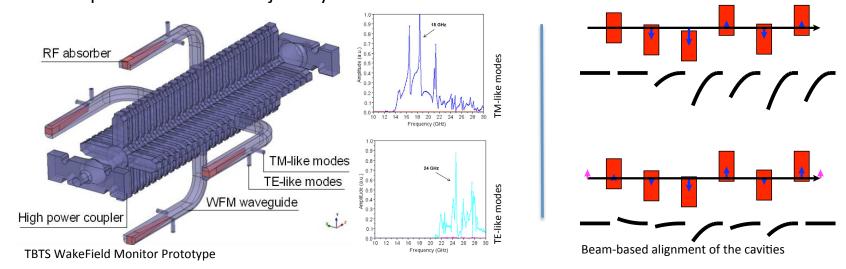
- RF diagnostic
 - 1. RF breakdown in PETS and AS
 - PETS on/off failure
 - 3. Provide references for the regular PETS+2AS units
- Beam control
 - RF power production
 - 2. Energy measurement and beam loading transient compensation

- 1. RF breakdown
- PETS on/off failure

WakeField Monitors

Position monitors called Wakefield Monitors (WFM) are integrated to the structure for beam-based alignment (cost saving solution). There will be 1 WFM for each super-structure.

To achieve the target luminosity, the accelerating structures must be aligned to an accuracy of **3.5** µm with respect to the beam trajectory.



The alignment of the accelerating structures in the CLIC main linac is necessary to remove the wakefield effects on the beam.

Simulations moving the articulation points showed that the emittance growth can be very well improved by aligning the accelerating structures to an accuracy of 3.5 μ m

For our tolerance σ_{WFM} = 3.5 μm , we found $\Delta \epsilon_{v} \approx 0.5$ nm.

Supporting Girders & Active-Pre Alignment

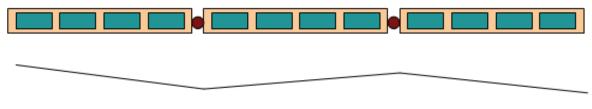
The pre-alignment of CLIC will have to guarantee a tolerance on the transverse positions of the girders of \pm 10 μ m (1 σ) on a sliding window of 200m.

A simplification of the problem is to pre-align a number of components of each linac on girders,
 i.e. align the girders instead of aligning each of the components



Adjacent girders linked by a common articulation point

 A simplification of the alignment by linking all the adjacent girders by a common articulation point. The alignment then consists in aligning each articulation point according to three degrees of freedom



Adjacent girders linked by a common articulation point

Supporting Girders

- The module design is based on a common girder that supports the components of each different system: such as AS, vacuum manifolds, RF components, PETS with their compact couplers, "On-Off" mechanism, vacuum connections as well as the DB quadrupoles equipped with BPM
- The girders should provide a sufficient mechanical stability and assure continuity for the chain of components of one beam
- The main components of each beam must be properly aligned to cope with beam and RF requirements
- The MB Q must stay independent of other components and such require the selfgoverning supports, which should combine both alignment and stabilization functions. Five degrees of freedom

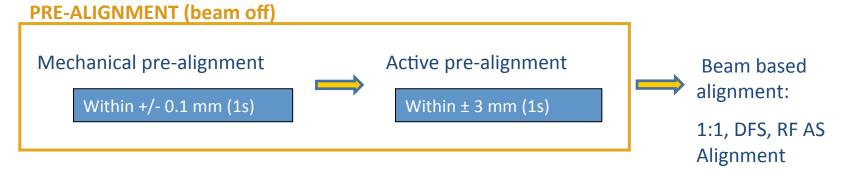
Pre-Alignment & Stabilization Requirements

Main alignment requirements in the vertical plane

Component	Tolerance (1σ)
Accelerating Structures	14 μm
PETS	31 μm
Main beam quadrupoles	17 μm
BPM for main beam quadrupole	14 μm
Relative position of MB quad. and BPM	5 μm

- The AS are the most sensitive components
- The stabilization system is required for the MB quadrupoles only
- The quadrupole support has five degrees of freedom, to be adjusted coherently with the other
 MB components
- A high mechanical stability is required for the MB quadrupoles: the requirement is not to exceed 1 nm in the vertical plane and 5 nm in the horizontal plane, for frequencies higher than 1 Hz. Two solutions are being studied at this purpose: active stabilization with nano-alignment between pulses; another is a combination of passive and active isolation
- The alignment of the AS is achieved through WF monitors

Active Pre-alignment

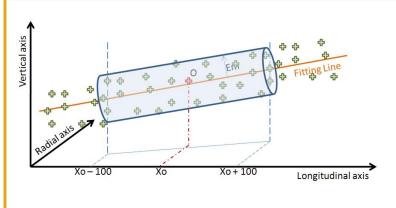


Active pre-alignment

Determination of the position of the complete the complete of the position of the complete of

+

Re-adjustment thanks to actuators



After computation, for a sliding window of 200 m, the standard deviations of the transverse position of the zero of each component w.r.t a straight fitting line will be included in a cylinder with a radius of a few microns:

- → 14 µm (RF structures & MB quad BPM)
- → 17 µm (MB quad)

Adjustment: step size of the order of 1 μ m

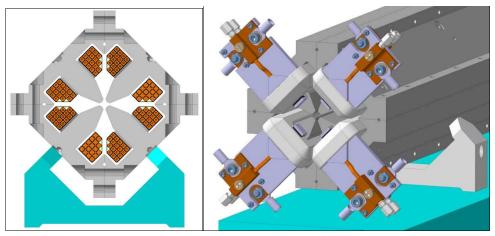
Main Linac Mover Tolerances

- Coarse mechanical motion
 - girders, MB quadrupoles supports
 - range: ≈ 1 mm
 - resolution: Δ ≈ 1 μm
 - precision: ≈ 0.5 μm
 - speed: may take a few pulses, but controlled
- Fine quadrupole motion
 - resolution: Δ ≈ 5 nm
 - range: ≈ 20 μm
 - precision: ≈ 2 nm
 - speed: from pulse to pulse
- Very fine quadrupole motion (stabilization system)
 - resolution: Δ order of ≈ 0.1 nm
 - range and precision: site dependent
 - speed: works in interval between pulses

Main Beam Quadrupoles

There are 4 types of MB quadrupoles. They are different only for the active (magnetic) length, keeping all the cross-section dimensions and operational parameters identical.

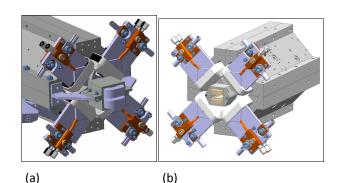
Types	Mag. Length	Units
Type-1	35 cm	308
Type-2	85 cm	1268
Type-3	135 cm	953
Type-4	185 cm	1462



Cross-section and Extremities view of the MB Quadrupoles

Some of the quadrupoles are 2 m long and weight about 450 kg.

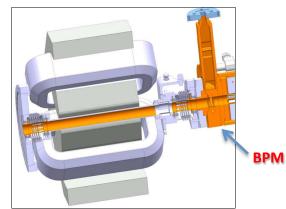
- A **beam steering** capability is required along the Main Beams
- The required steering capability is asked in one plane per magnet alternatively.
- Two main solutions are being considered: a MBQ built-in correction scheme with extra-coils in each magnet (a) or an "ad hoc" small dipole corrector to be added to each MBQ (b).



View of the Beam Steering Corrector integration for the MBQ Type-4 (a) and Type-1 (b)

Drive Beam Quadrupoles

- DB quadrupoles are present all along the beam in a FODO configuration with one F and one D quadrupole on each Module.
- Decelerators are so characterized by the presence of 20740 quadrupoles on each linac.
- Specification: one quadrupole per meter gives beta function (for most decelerated particles) of $<\beta>=1.25$ m
 - Deemed necessary for robust mitigation of dipole wake
 - Gives r = 3.3 mm (out of a_0 = 11.5 mm) for ideal beam
- Powered magnets is the baseline
 - failure tolerant serial powering scheme is a necessity



Drive Beam Quadrupole prototype and BPM

Tuneable permanent magnets option investigated: it must cover all operational scenarios

Quadrupole specifications			
Total nb. of quads	$N_{ m tot}$	~42000	-
Inner radius of vacuum chamber	a_0	11.5	mm
Max. integrated gradient	\hat{G}	12.2	T
Tunabiliy		See operational scenarios	
Magnet design accuracy rms	$\sigma(Gl)/(Gl)$	1×10^{-3}	-
Resulting magnet design tolerance	$\Delta(Gl)/(Gl)$	$\sqrt{3} \times 10^{-3}$	-
Max. dodecapole component at 11 mm	Int B6 / Int B2	3×10^{-4}	-
Power supply accuracy rms	$\sigma(I)/I$	5×10^{-4}	-

Note that DB correctors will be the girder's articulation points (baseline).

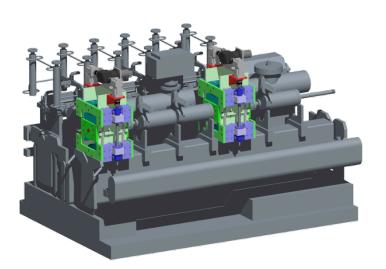
Drive Beam Quadrupoles

Daresbury Lab was asked to look at Permanent Magnet options and also to assess new techniques for building ~50 quads/day

- The CLIC drive beam needs a quadrupole every meter (~42,000)
- The electromagnet option will consume ~400W per magnet
- Want to maintain heat load in tunnel to <150W/m

Why Permanent Magnets?

- No direct power consumption
- No heatload in the tunnel
- Low running costs
- Higher integrated gradient (potentially)



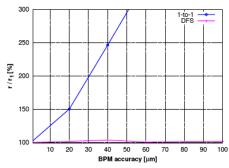
Possible issues: Radiation Damage? Is large tuneability feasible? Is required motion control precision feasible? Sensitivity to material errors & temperature? Sufficient magnet quality?

Parameter	Value		
Inscribed radius	14 mm		
PM size	18 x 100 mm		
PM angle	40°		
Magnet length	230 mm		
Maximum stroke	64 mm		
Gradient	81 T/m (max)	8.1 T/m (min)	
Integrated gradient	15.0 T (max)	3.6 T (min)	
Relative to nominal	123% 30%		
Magnetic length	241 mm		
Good gradient region	±7.0 mm		

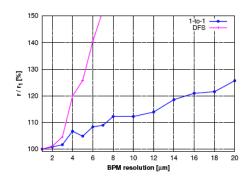
Drive Beam Beam Position Monitors

Beam-based correction performance drive the BPM specifications.

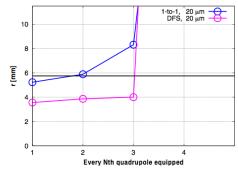
Target: negligible envelope growth due to quadrupole kicks.



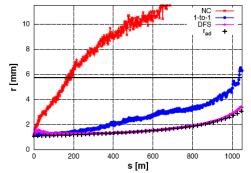
Effect of BPM accuracy



Effect of BPM resolution



Effect of # of BPMs

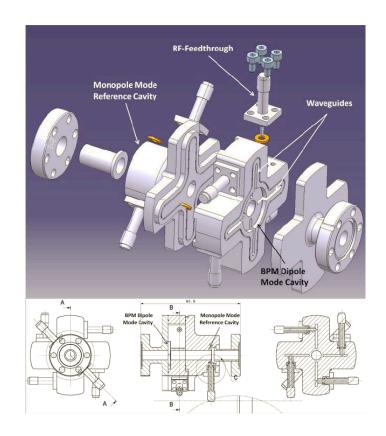


Results with baseline parameters

Quantity	Symbol	Value	Unit
# of quadrupoles per BPMs	N	$\sim 2/3$	-
Total number of BPMS	N_{tot}	~ 28000	-
Production beam			
BPM accuracy	$\sigma_{ m acc}$	20	$\mu\mathrm{m}$
BPM resolution	$\sigma_{ m res}$	2	$\mu\mathrm{m}$
Time resolution	$t_{ m res}$	60	ns
Pilot beam			
BPM accuracy	$\sigma_{ m acc}$	20	$\mu\mathrm{m}$
BPM resolution	$\sigma_{ m res}$	4	$\mu\mathrm{m}$
Time resolution	$t_{ m res}$	60	ns

Main Beam Beam Position Monitors

- The BPM are required for measurement of beam trajectory
- The time resolution was set to less than 50 ns and the spatial one to~50~nm for the MB. The corresponding accuracy must be better than 5 μm
- A precision of 20 μm and resolution of about
 2 μm are necessary for the DB
- The pre-alignment system will be needed for making possible the "first" beams pass through the linac.

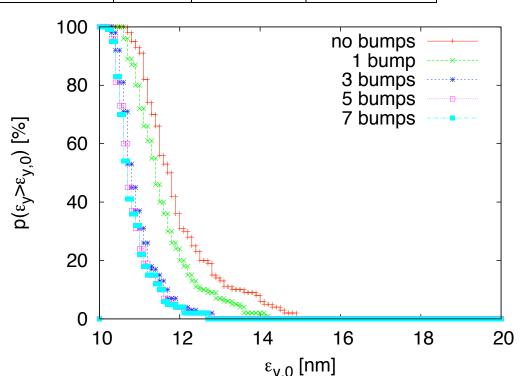


- Then, beam-based alignment will be applied using 1:1 correction, Dispersion Free Steering and accelerating cavity alignment

Main Beam Transverse Tolerances

imperfection	with respect to	symbol	value	emitt. growth
BPM offset	wire reference	σ_{BPM}	14 $\mu\mathrm{m}$	$0.367\mathrm{nm}$
BPM resolution		σ_{res}	0.1 $\mu\mathrm{m}$	$0.04\mathrm{nm}$
accelerating structure offset	girder axis	σ_4	10 $\mu\mathrm{m}$	$0.03\mathrm{nm}$
accelerating structure tilt	girder axis	σ_t	140 μ radian	$0.38\mathrm{nm}$
articulation point offset	wire reference	σ_5	12 $\mu\mathrm{m}$	$0.1\mathrm{nm}$
girder end point	articulation point	σ_6	$5\mu\mathrm{m}$	$0.02\mathrm{nm}$
wake monitor	structure centre	σ_7	3.5 $\mu { m m}$	$0.54\mathrm{nm}$
quadrupole roll	longitudinal axis	σ_r	100 μ radian	$\approx 0.12\mathrm{nm}$

- Selected a good DFS implementation
 - trade-offs are possible
- Multi-bunch wakefield misalignments of $10\,\mu\mathrm{m}$ lead to $\Delta\epsilon_y \approx 0.13\,\mathrm{nm}$
- Performance of local prealignment is acceptable



Main to Drive Beam Phase Tolerance

- Integrated simulations have been performed with PLACET and GUINEA-PIG of main linac, BDS and beam-beam
 - system is assumed to be perfectly aligned (to determine BDS bandwidth effect)
 - assuming target emittance at BDS
- Resulting luminosity loss is about 2% for

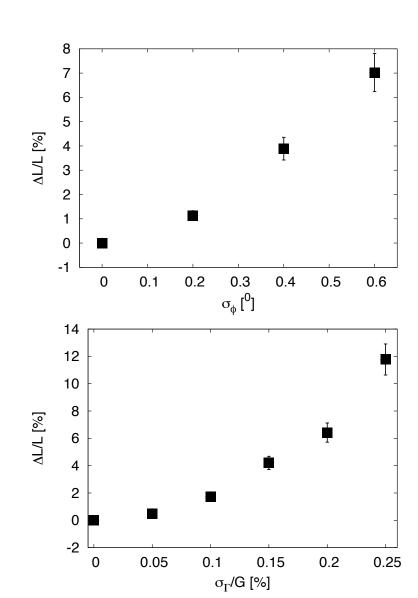
$$\frac{\sigma_G}{G} \approx 1 \times 10^{-3}$$

and

$$\sigma_{\phi} \approx 0.3^{\circ}$$

$$\frac{\Delta \mathcal{L}}{\mathcal{L}} \approx 0.01 \left[\left(\frac{\sigma_{\phi,coh}}{0.2^{\circ}} \right)^{2} + \left(\frac{\sigma_{\phi,inc}}{0.8^{\circ}} \right)^{2} + \left(\frac{\sigma_{G,coh}}{0.75 \cdot 10^{-3} G} \right)^{2} + \left(\frac{\sigma_{G,inc}}{2.2 \cdot 10^{-3} G} \right)^{2} \right]$$

• Main beam current needs to be stable to $\approx 0.1 - 0.2\%$



Cooling System

- A significant part of the RF input power is transferred into heat in the RF structures.
- The cooling system consists of 21 km long linear accelerator circuits with about 20K temperature difference for each module
- Functions of the cooling system are:
 - Cooling the system
 - Providing micro-meter thermal stability to the accelerator tunnel

Sustaining the required alignment of 14 μm for ACS within 1 σ requires a special development of the cooling system under dynamic heat load conditions

- Ensuring mechanical stability of components, alignment and vacuum systems require evacuating the heat efficiently by specially designed water circuits
- Any vibration induced by coolant flow can result in loss of machine performance

Estimated Power Dissipation

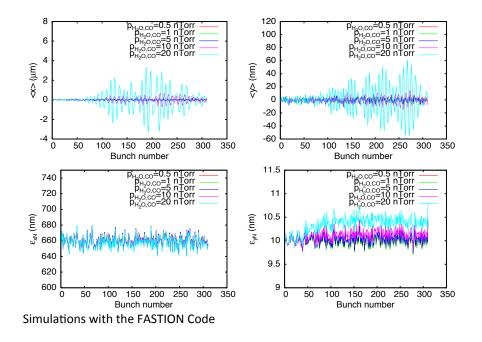
- Per Accelerating Structure:
 - loaded = 336 W
 - unloaded = 410 W

• Per Module:

Components	Type 0	Type I	Type 2	Type 3	Type 4
Ac. structures [W]	3285	2464	1642	821	
PETS [W]	352	264	176	88	
DB quadrupoles [W]	342	342	342	342	342
MB quadrupole [W]		890	1780	2600	3831
Loads [W]	2861	2146	1430	715	
WGs [W]	4 5	34	23	11	
Total per module [W]	6885	6139	5393	4578	4173

Vacuum: Requirements from Beam Dynamics

Proceedings of IPAC10, J.-B. Jeanneret, G. Rumolo, D. Schulte



New fast ion instability simulations show that the influence of field ionization for a threshold electric field value between 18 and 30 GV/m, which covers the common gas species H2, CO, N2 and H2O, makes the **beam unstable for partial pressures above 5 nTorr**.

A dynamic pressure of **1 nTorr** must be specified in order to ensure beam stability. In the future, we plan to simulate the fast ion instability for a more realistic composition of the residual gas

Concluding...

 The layout of the two-beam acceleration modules has been presented, and each subsystem has been introduced

 Answers to how to cope with the presented requirements and technical issues will be given in the following talks...

 I wish to thank all the people that have helped me during the preparation of this talk.