

The Physics Beyond Colliders Study Group

4th Technology Workshop

CERN

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Effective passive thermal links for cryocooling

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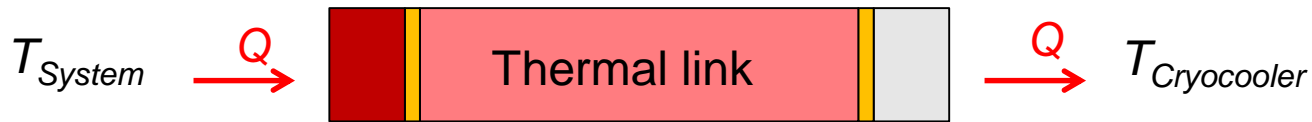
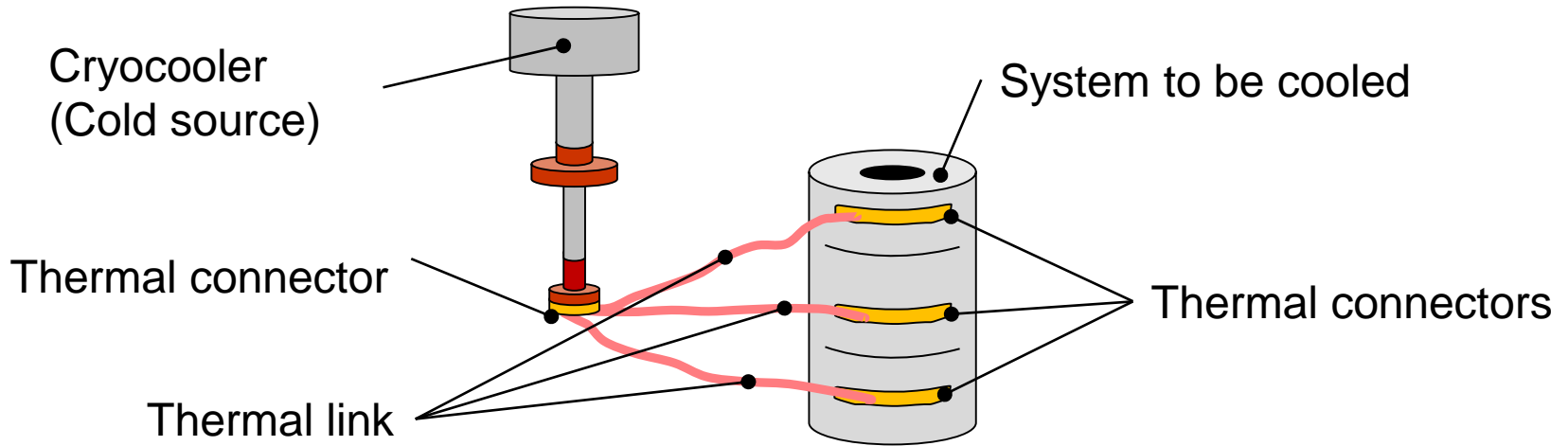
bertrand.baudouy@cea.fr

Outline

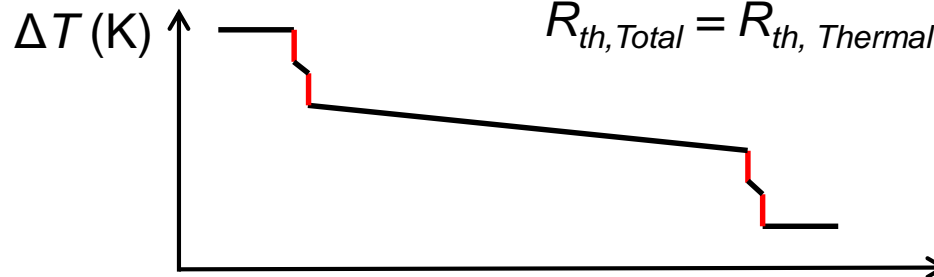
- Cryocooling
- Possible thermal links for cryocooling
- Gravity assisted circulation loop
- The example of the WAVE magnet
- Pulsating heat pipe
- The example of the HTS magnet cooled by a neon PHP project

Cooling with a cryocooler (1/4)

- Indirect cooling method
 - Thermal contact resistances between the different solid component of the thermal link



$$\Delta T \text{ (K)} \uparrow \quad R_{th, Total} = R_{th, Thermal link} + R_{th, Contact}$$



Cooling with a cryocooler (2/4)

- Limited cooling power

Typical max power for Gifford-McMahon, Pulse tube, Stirling,... cryo-cooler

Temperature (K)	4.2	10	20	77
Power (W)	2	5-10	10-30	500

- Cryogenic design must be accurate (accurate heat load estimation)
- If higher heat load than designed, working temperature (and pressure if two-phase thermal link) higher
- A small cooling area
 - Cooling cross-section between 10 cm² to 85 cm²
 - Cooling power distribution needed : thermal links
 - Active and passive thermal links
- Passive thermal links are preferable
 - Automation of the cooling scheme
 - Cryogen-free system
 - Possibility to use cryogen



Gifford-McMahon type
Cryocooler



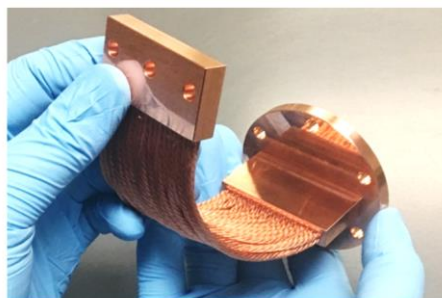
Pulsed Tube type
Cryocooler

Cooling with a cryocooler (3/4)

- Cooling without a cryogen (Cryogen-free cooling)
- Solid thermal links
 - Conduction between cold source and the system
 - Thermal contact and heavy (Cu, Al,...)
 - Small heat load especially at low temperature
 - Slow (diffusion) and low thermal perturbations
 - Easy implementation



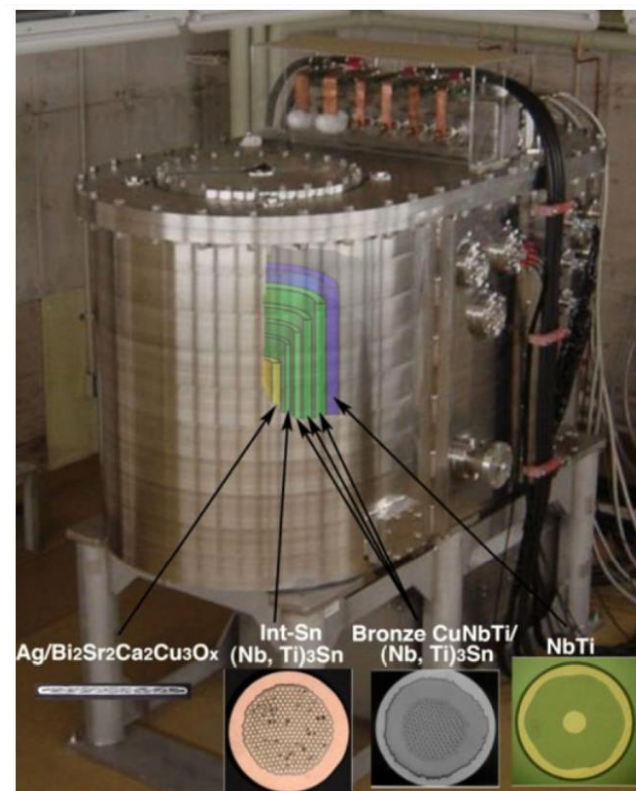
<https://thermal-space.com>



<https://www.techapps.com>

- Examples
 - Detector for space
 - Superconducting magnet with cold bore
 - Instrumentation for medical imaging
 - Physics instruments
 - HTc current leads

18T cryogen-free
superconducting magnet

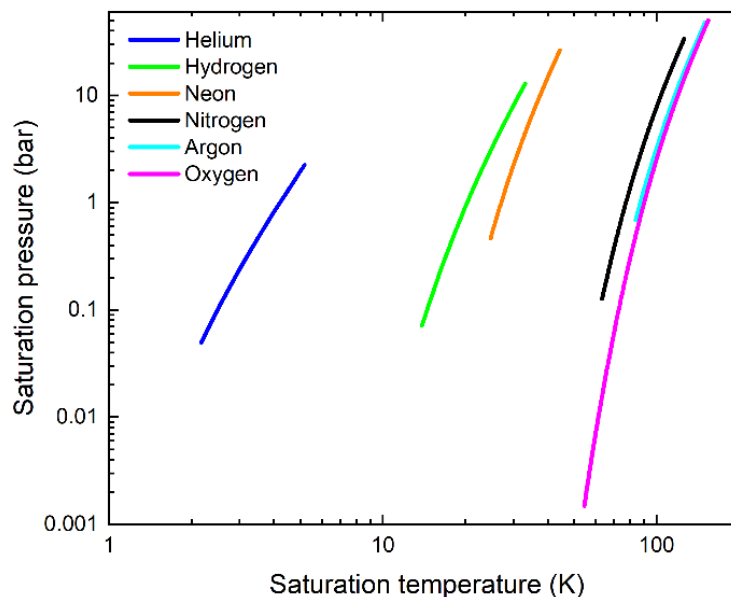


K. Watanabe, *J Supercond Nov Magn* (2011) 24: 993–997,
<https://doi.org/10.1007/s10948-010-0871-0>

Cooling with a cryocooler (4/4)

- Cooling with a cryogen
- Thermal links with gas, liquid or two-phase mixture with or without a forced circulation
 - Heat transfer by phase change (closed to saturation)
 - Heat transfer by advection (pump..)
 - Lighter and higher heat transfer rates (phase change + advection...) than for the solid thermal links
 - Limited temperature and pressure ranges for two-phase thermal links (at saturation)
- Examples
 - Small superconducting magnet
 - Liquid targets for nuclear physics

Superconducting magnet cooled by a helium gravity assisted circulation loop

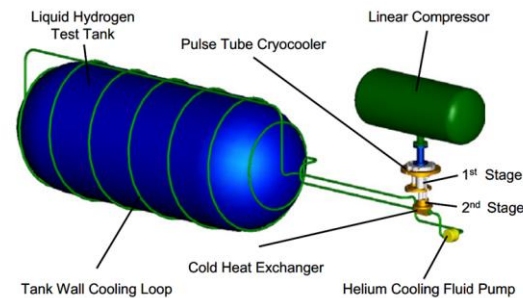
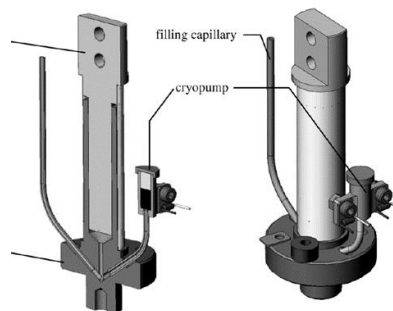


C. Berriaud et al., *IEEE Transactions on Applied Superconductivity*, vol. 26, no. 4, pp. 1-4, June 2016, Art no. 4102804, doi: [10.1109/TASC.2016.2517338](https://doi.org/10.1109/TASC.2016.2517338)
T. Robillard et al. *Journal of Neutron Research* 22 (2020) 379–391 379, <https://content.iospress.com/articles/journal-of-neutron-research/jnr200167>

Two-phase thermal links

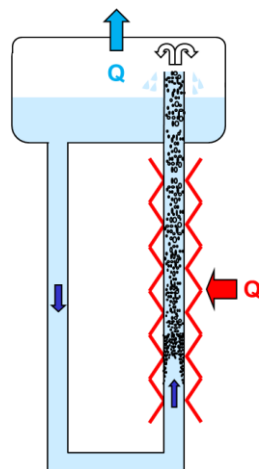
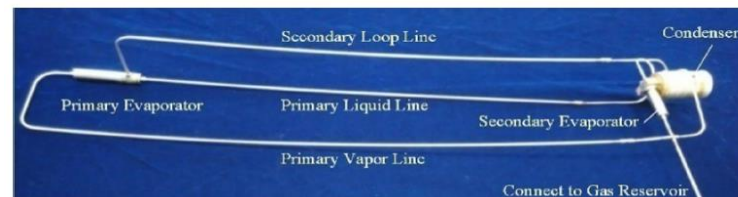
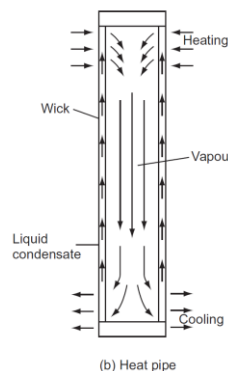
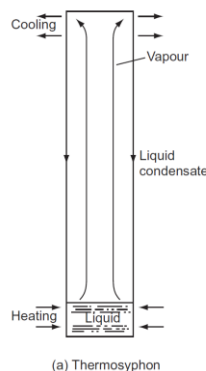
- **Active thermal links**

- Forced flow circulation loop
- Heat switch (mechanical, gas, magnetic,...)



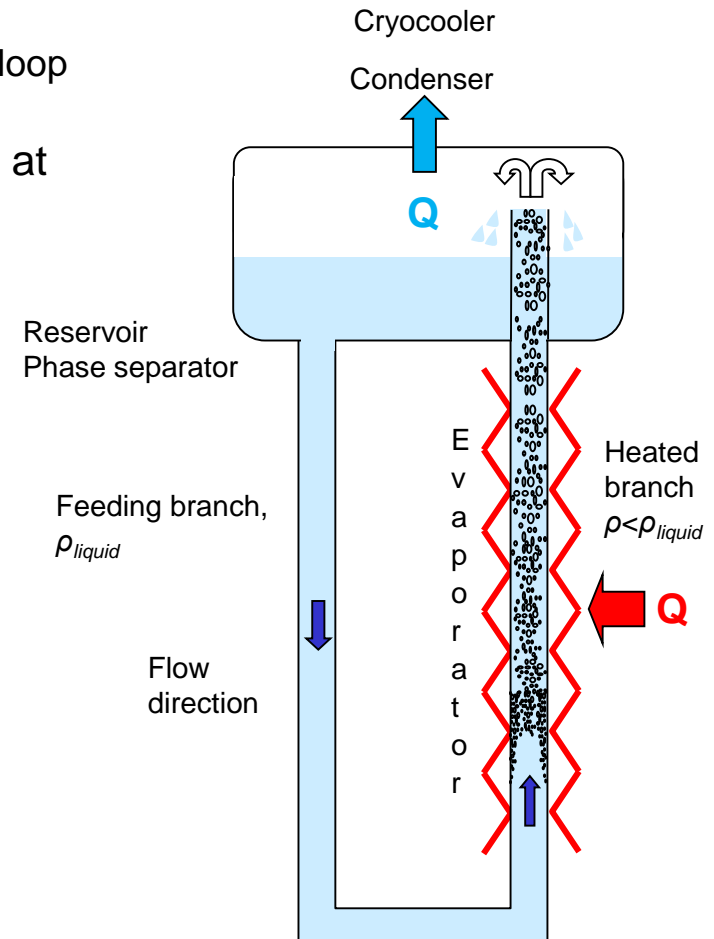
- **Passive thermal links**

- Thermosiphon
- Heat pipe
- Loop heat pipe
- Gravity assisted circulation loop
- Pulsating heat pipe



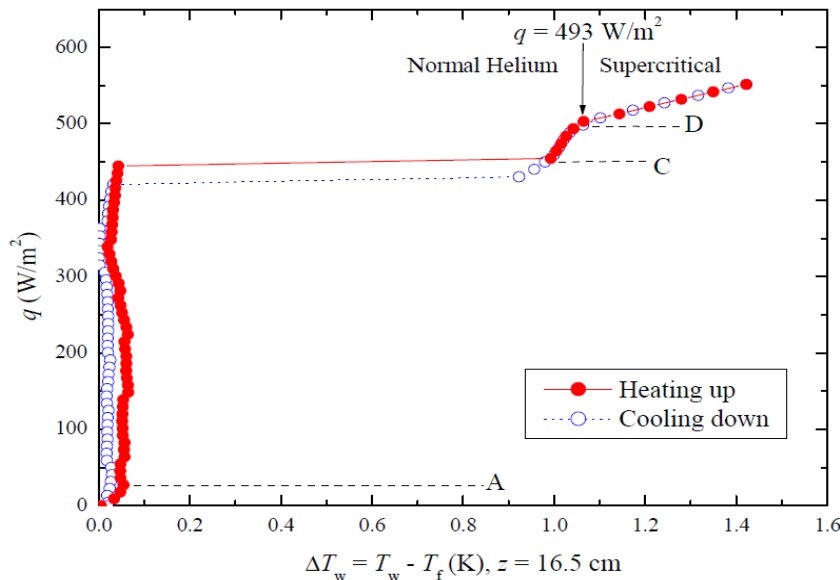
Two-phase thermal links | Gravity loop (1/4)

- Gravity assisted circulation loop
 - Heat exchange due to advection and boiling
 - Evaporator : one of the branch of the loop
 - Condenser : cryocooler + heat exchanger
 - Evaporator and condenser inserted in a vertically oriented loop
 - Flow created due to phase change or density decrease at evaporator
 - Lighter evaporator branch ($\rho < \rho_{\text{liquid}}$) than feeding branch
 - Circulation due to loop configuration
 - Co-current flow of liquid and vapor in the evaporator
- Advantages
 - Easy construction and implementation
 - No pumping system
 - Self-adjusting flow rate
- Disadvantages
 - Gravity dependent
 - Lower heat transfer than pool boiling



Two-phase thermal links | Gravity loop (2/4)

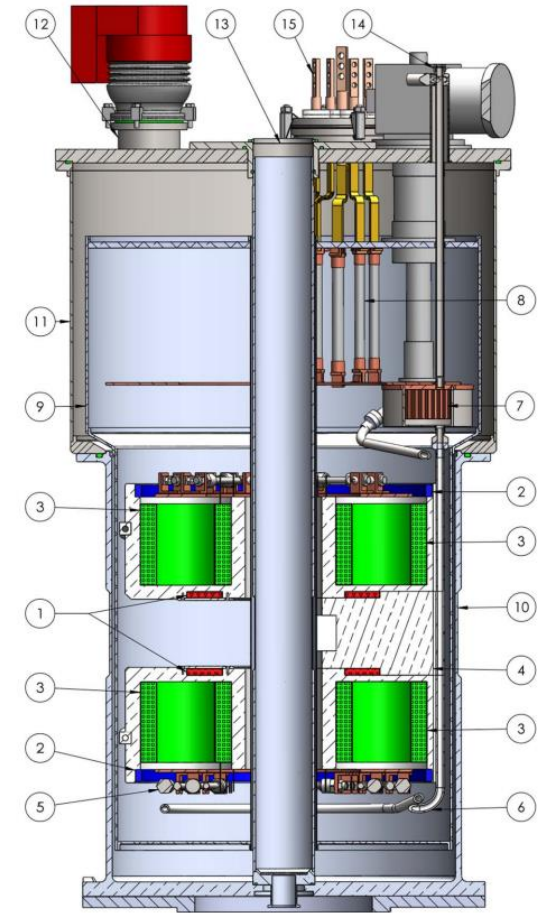
- 0.5 m autonomous loop in helium
 - Two-stage GM Cryocooler: 1.5 W at 4.2 K
 - 1st stage
 - Radiation shield
 - Precooling of the gas with a heat exchanger
 - 2nd stage
 - Copper finned tube exchanger in the reservoir
- Test in helium around saturation for a Ø4 mm tube
 - $h \sim 5000 \text{ W/m}^2\text{K}$ et $q_c \approx 500 \text{ W/m}^2$



Y. Song et al., *International Journal of Heat and Mass Transfer*, Volume 66, November 2013, Pages 64-71, <https://doi.org/10.1016/j.ijheatmasstransfer.2013.07.002>

Two-phase thermal links | Gravity loop (3/4)

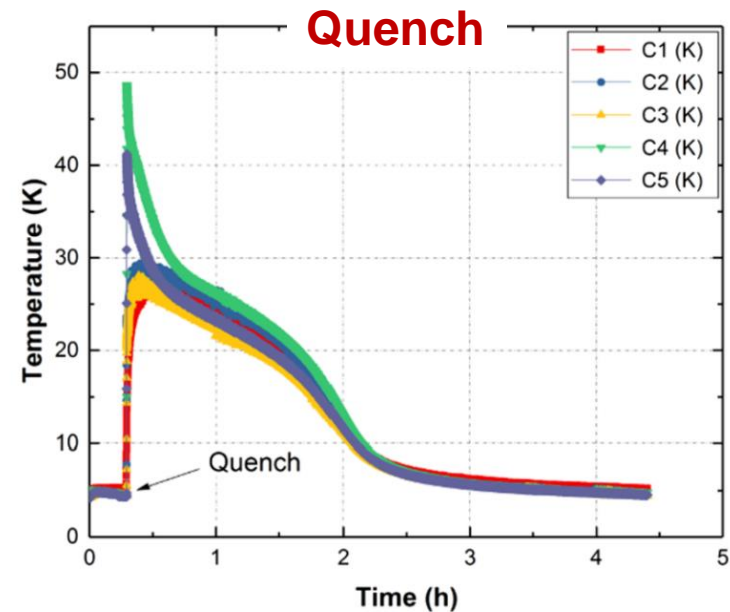
- Example of the superconducting magnet WAVE (Wide Aperture Vector Magnet)
 - 1 T superconducting vector magnet for neutron scattering experiment
- Cold sources
 - Single-stage 100 W at 50 K cryocooler
 - Pre-cooling and Radiation shield
 - Two-stage 35 W @ 50 K and 1.5 W @ 4.2 K cryocooler
 - Radiation shield and circulation loop
- Gravity circulation loop in helium at 4.5 K
 - Copper tank (phase separator) of 1.5 liters with heat exchanger on the 2nd stage of the cryocooler
 - Aluminum magnet cold mass (cryo-stability)
 - Heat exchanger by aluminum tubes ($\text{\O}10\text{ mm}$) with square section ($16 \times 16\text{ mm}^2$) glued on the cold mass in spiral
 - Helium tank of 200 liters and 6 bar max
 - Initial charge of helium gas for condensation
 - Helium recovery in case of quench



C. Berriaud et al., *IEEE Transactions on Applied Superconductivity*, vol. 26, no. 4, pp. 1-4, June 2016, Art no. 4102804, doi: [10.1109/TASC.2016.2517338](https://doi.org/10.1109/TASC.2016.2517338)
T. Robillard et al. *Journal of Neutron Research* 22 (2020) 379–391 379, <https://content.iospress.com/articles/journal-of-neutron-research/jnr200167>

Two-phase thermal links | Gravity loop (4/4)

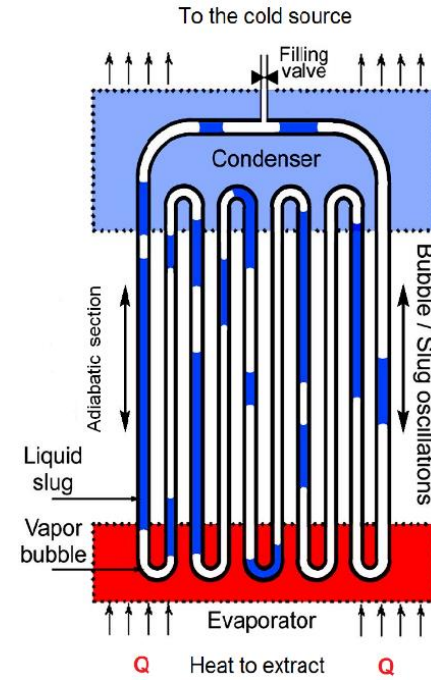
- Superconducting magnet WAVE (Wide Aperture Vector Magnet)
- Total heat input of 0.240 W on the superconducting magnet
 - Conduction in the supports
 - Current leads (conduction and resistive heat loads)
 - Radiation
- Magnet cooling in 8 days in total autonomy (cryocooled only!)
 - Helium tank loaded with helium gas
- Quench of the magnetic system
 - Temperature increase
 - Vaporization of liquid helium
 - Gas recovery in the helium tank
 - 4 hours to re-condense the vapors and recover the operating temperature



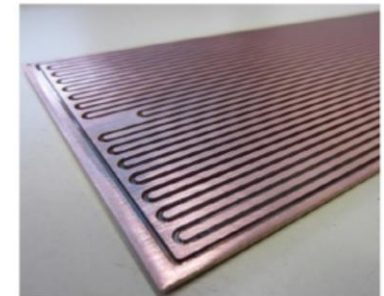
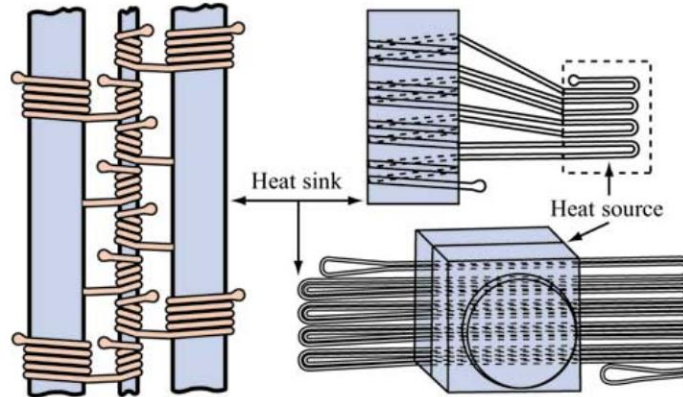
C. Berriaud et al., *IEEE Transactions on Applied Superconductivity*, vol. 26, no. 4, pp. 1-4, June 2016, Art no. 4102804, doi: [10.1109/TASC.2016.2517338](https://doi.org/10.1109/TASC.2016.2517338)
T. Robillard et al. *Journal of Neutron Research* 22 (2020) 379–391 379, <https://content.iospress.com/articles/journal-of-neutron-research/jnr200167>

Two-phase thermal links | Pulsating Heat Pipe (1/5)

- Heat pipe with a capillary oscillating flow
- Passive heat pipe constructed with a single capillary tube
 - No extra structure on the wall of the tube
 - Tube winding in turns between an evaporator and a condenser separated by an adiabatic section
- Tube partially filled with a two-phase mixture close to saturation
 - Random distribution of vapor bubbles and liquid plugs
 - Oscillation of vapor bubbles and liquid plugs between the evaporator and condenser



- Advantages
 - Easy construction
 - Geometry independant
 - Gravity independent
- Disadvantages
 - Start-up scenario (long PHP)
 - No full understanding / rules for design



Two-phase thermal links | Pulsating Heat Pipe (2/5)

- Vapor bubbles and liquid plugs oscillations
 - Capillary force creates liquid and vapor structures
 - Δp due to expansion and contraction of phase transitions
 - Vaporization at evaporator creates overpressure
 - Movement of vapor bubbles lubricated by a wall liquid film
 - Vapor liquefaction at the condenser
- Liquid/vapor fraction important (Filling Ratio = $V_{\text{liquid}}/V_{\text{total}}$)
- Maximal diameter to create the “slug” flow

$$D_{crit} \leq 2 \sqrt{\frac{\sigma}{g(\rho_l - \rho_v)}}$$

Surface tension

Gravity force

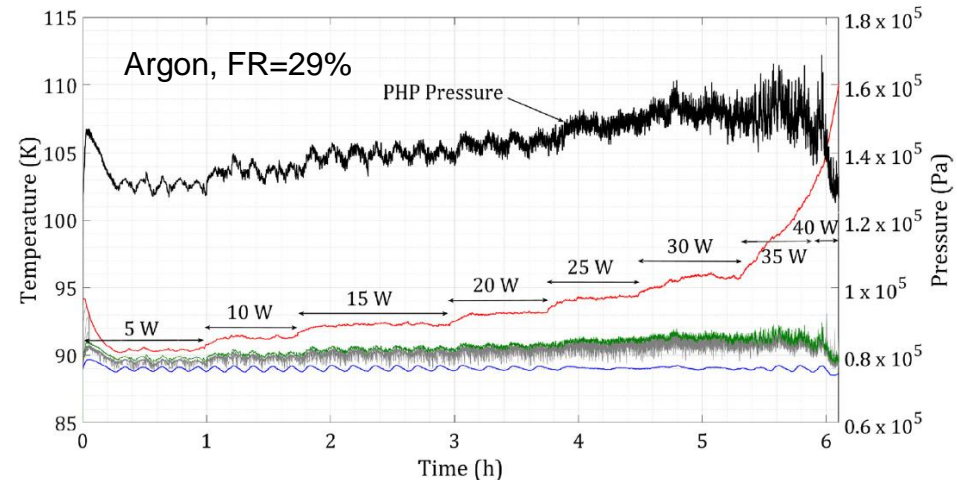
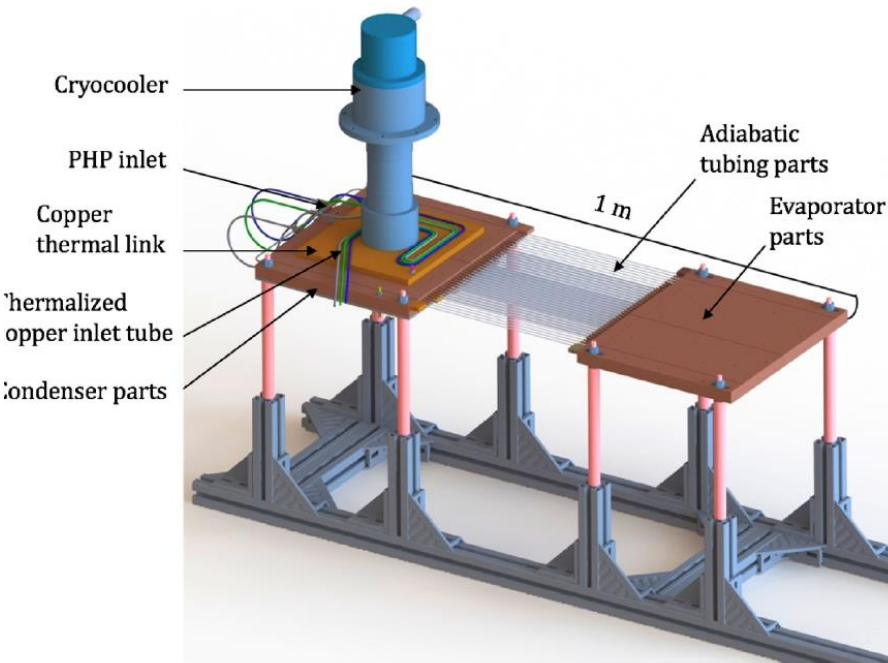
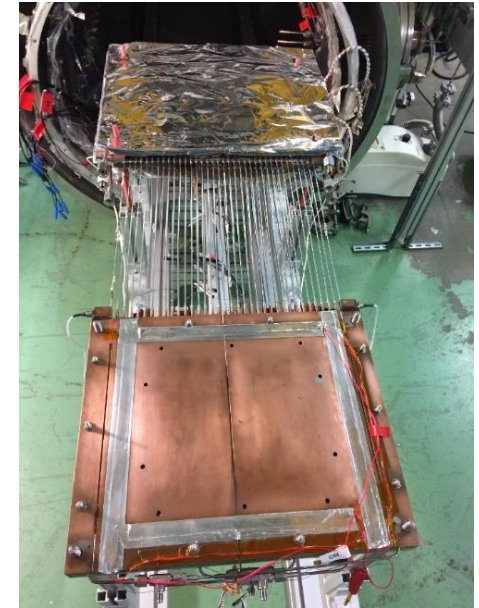
- High heat transfer
 - Phase change and advection heat transfer combination
 - Advection: Sensible heat essentially carried by the liquid
 - Phase change: latent heat due to phase changes within the mixture and with the solid (at evaporator and condenser)



*K Sameer et al. Applied Thermal Engineering
Volume 23, Issue 16, November 2003, Pages 2021-
2033* [https://doi.org/10.1016/S1359-4311\(03\)00168-6](https://doi.org/10.1016/S1359-4311(03)00168-6) (2003)

Two-phase thermal links | Pulsating Heat Pipe (3/5)

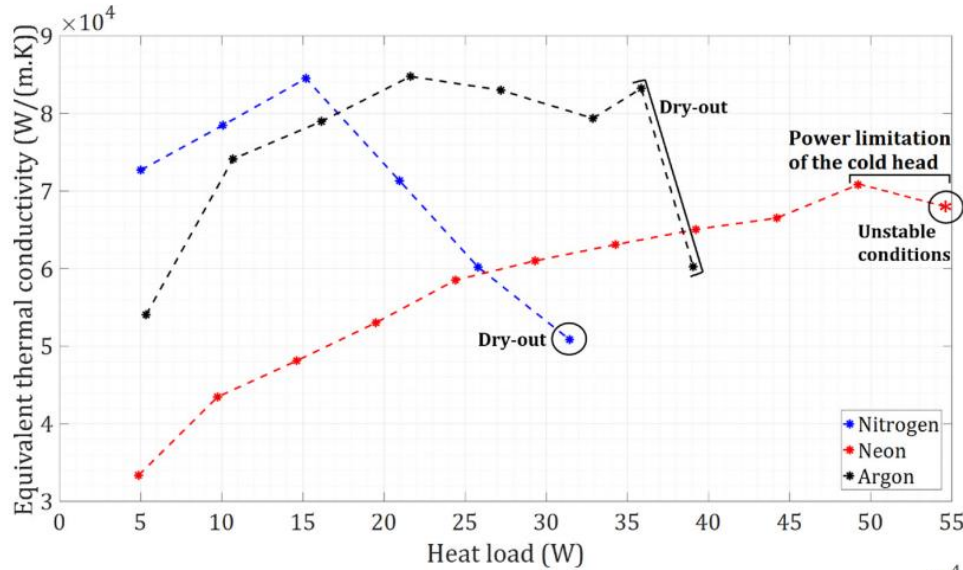
- 1 m long horizontal PHP
 - SS tube with $\text{\O}1.5$ mm and 36 turns
 - Condenser and evaporator: 330 mm long
 - Adiabatic section : 300 mm long
- Fluids : Nitrogen (77 K), neon(27 K) and argon (87 K)



Maria Barba et al. *International Journal of Heat and Mass Transfer*, Volume 187, 2022, 122458, <https://doi.org/10.1016/j.ijheatmasstransfer.2021.122458>
 Maria Barba et al. *Applied Thermal Engineering*, Volume 194, 2021, 117072, <https://doi.org/10.1016/j.applthermaleng.2021.117072>.

Two-phase thermal links | Pulsating Heat Pipe (4/5)

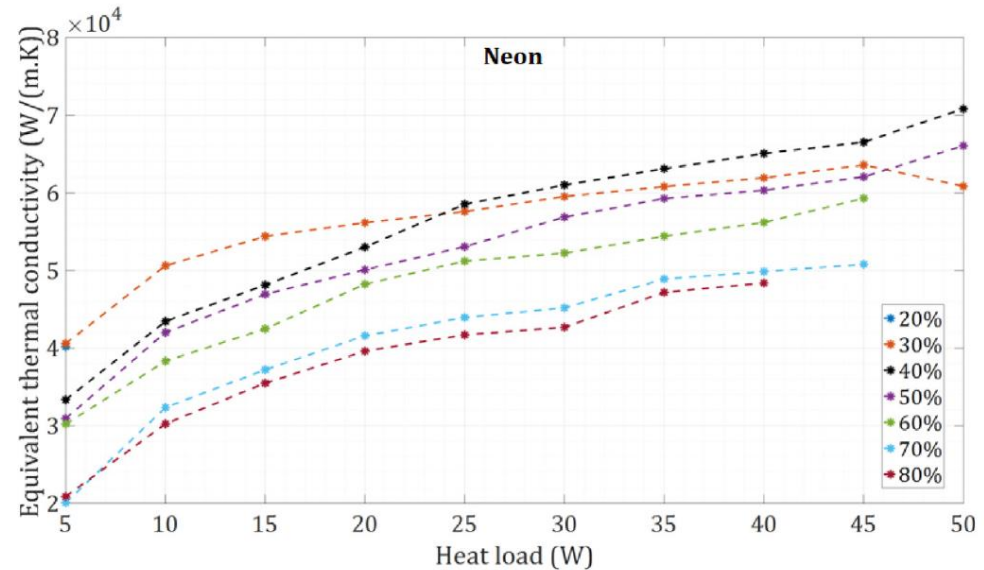
- 1 m long horizontal PHP with Ø1.5 mm and 36 turns



Fluids difference

$$k_{eff} = \frac{QL}{A\Delta T}$$

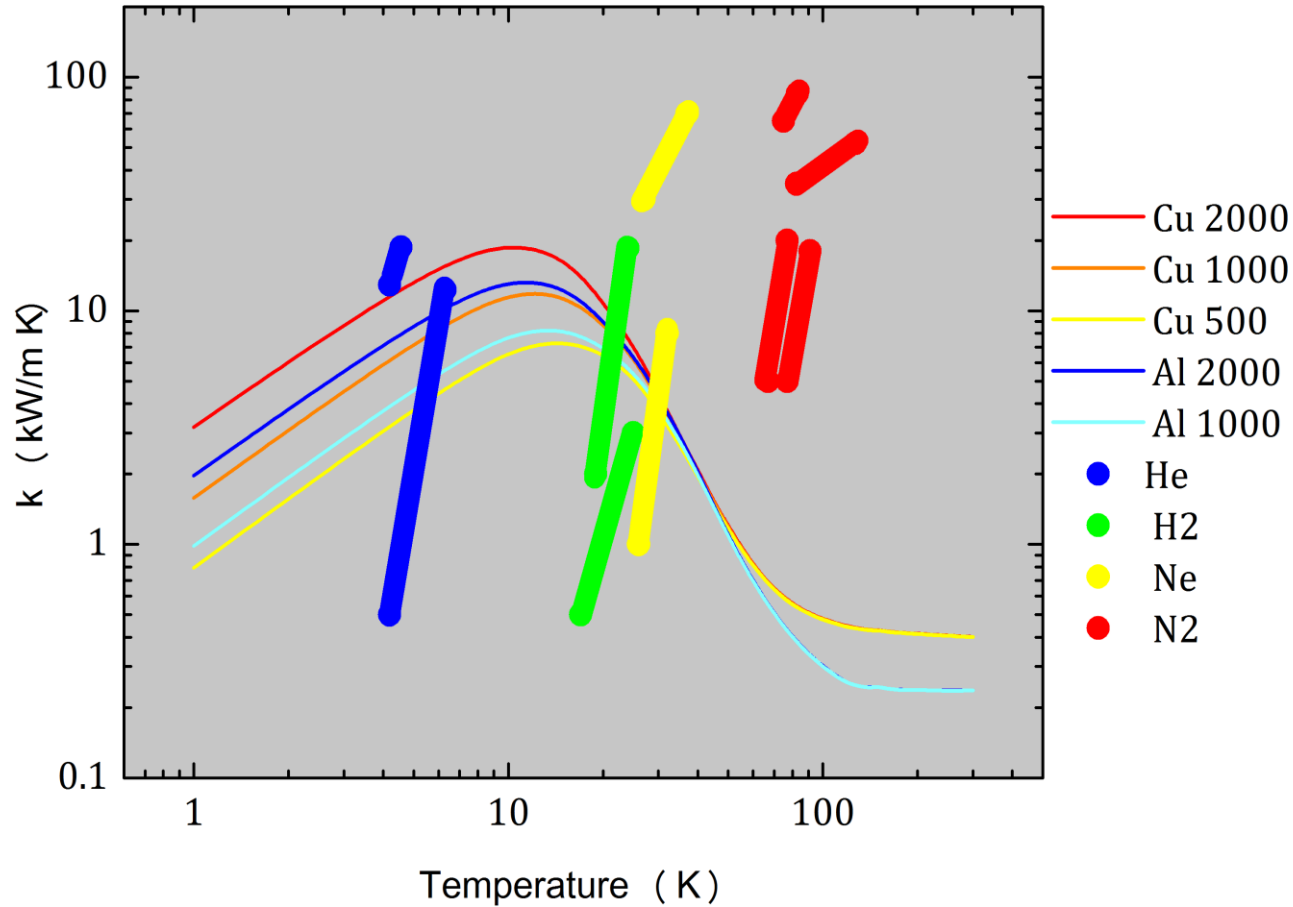
FR effect



Maria Barba et al. *International Journal of Heat and Mass Transfer*, Volume 187, 2022, 122458, <https://doi.org/10.1016/j.ijheatmasstransfer.2021.122458>
 Maria Barba, *Applied Thermal Engineering*, Volume 194, 2021, 117072, <https://doi.org/10.1016/j.applthermaleng.2021.117072>.

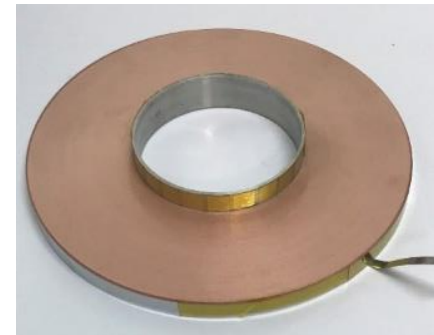
Two-phase thermal links | Pulsating Heat Pipe (5/5)

$$k_{eff} = \frac{QL}{A\Delta T}$$



Two-phase thermal links | Neon PHP for magnet (1/5)

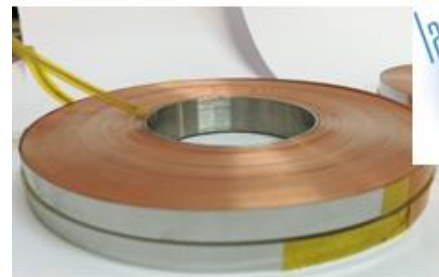
- 10 T class HTS magnet cooled by an PHP at Neon temperature
- MI for Metal-as-Insulation technique
 - A resistive metallic tape is co-wound with RebCo tape
- Magnet like the NOUGAT magnet project (2015-2018, CNRS-CEA ANR)
- MI-HTS magnet at intermediate temperature (27 K)
- Cryogenic scheme based on a PHP link
- Working conditions
- Long-duration stability of cryogenic PHP



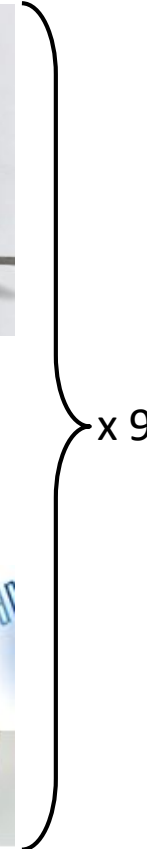
Pancake

Internal
join

x 2



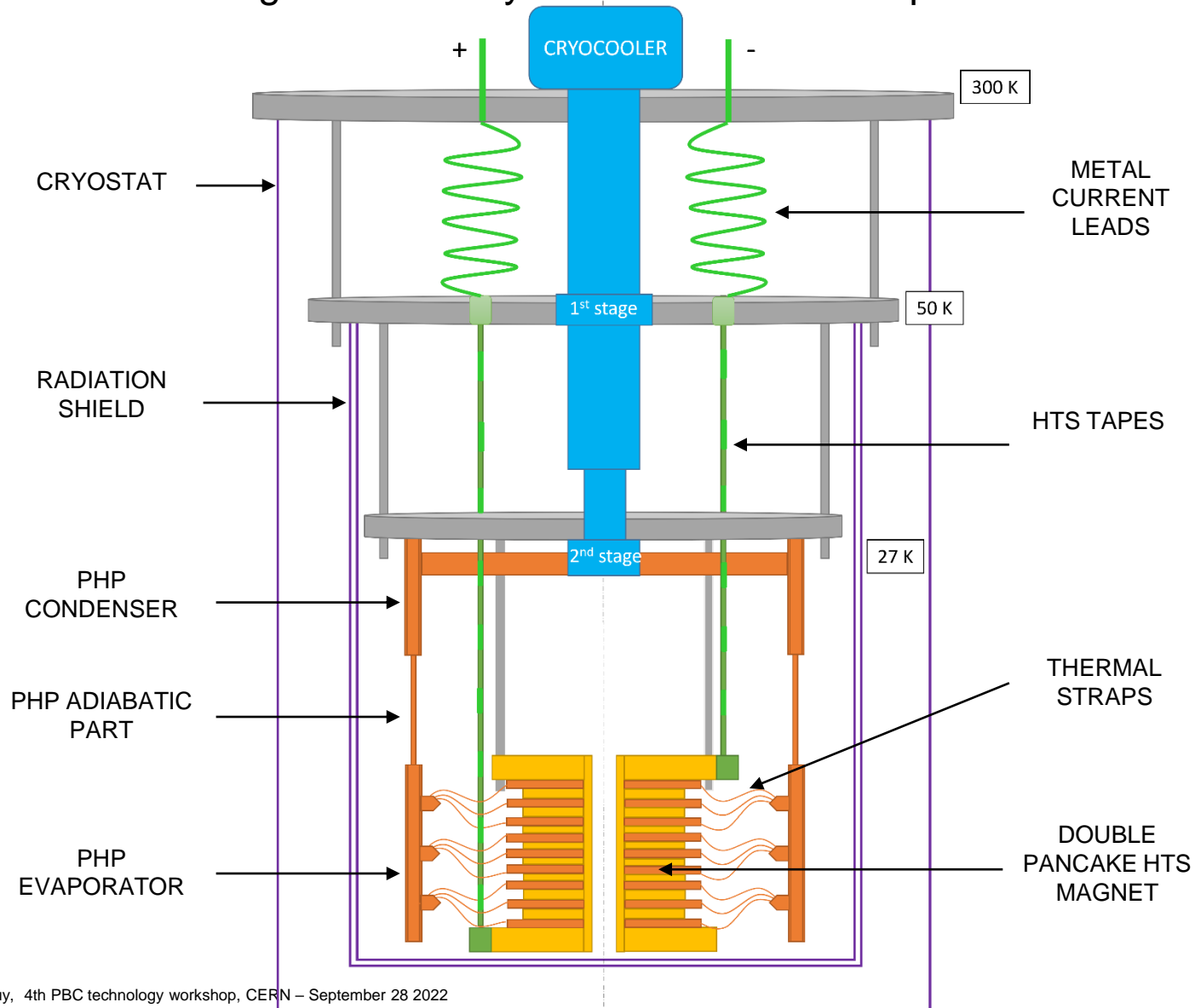
Double pancake



Fazilleau, P., Chaud, X., Debray, F., Lécresse, T., & Song, J. B. (2020). 38 mm diameter cold bore metal-as-insulation HTS insert reached 32.5 T in a background magnetic field generated by resistive magnet. *Cryogenics*, 106, 103053.

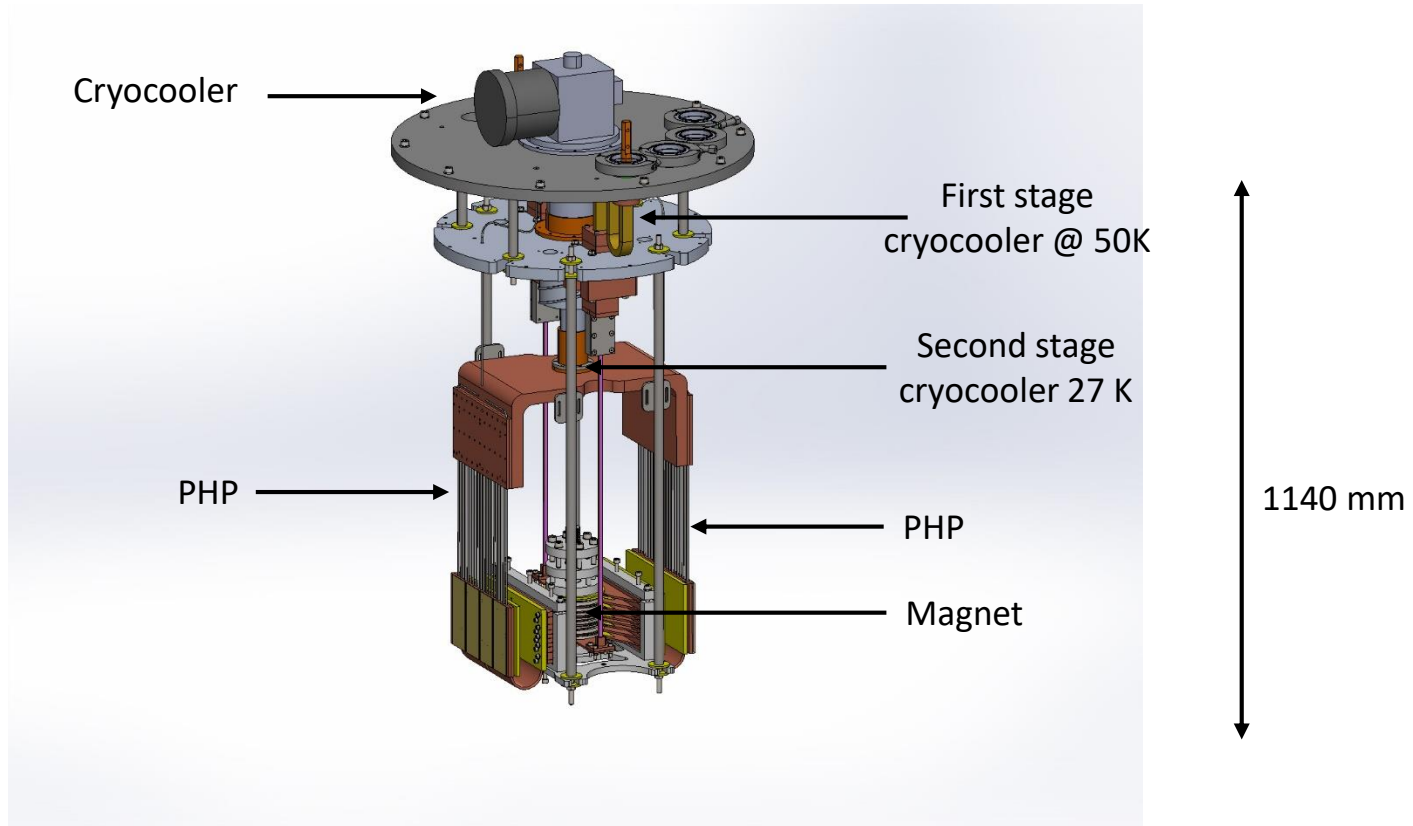
Two-phase thermal links | Neon PHP for magnet (2/5)

- 10 T class HTS magnet cooled by an PHP at Neon temperature



Two-phase thermal links | Neon PHP for magnet (3/5)

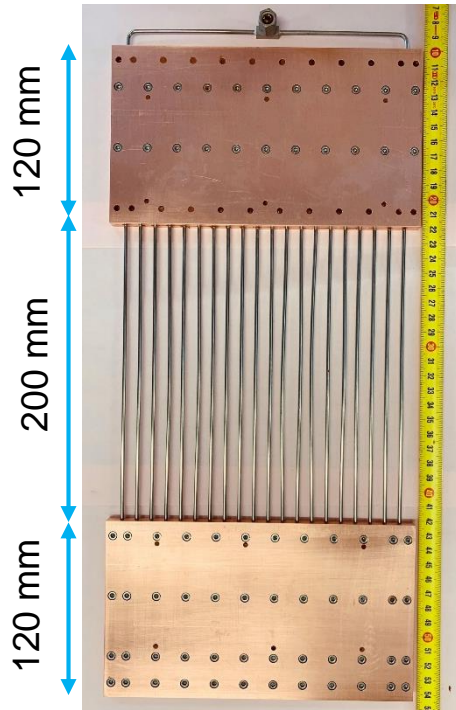
- 10 T class HTS magnet cooled by an PHP at Neon temperature



Two-phase thermal links | Neon PHP for magnet (4/5)

- 10 T class HTS magnet cooled by an PHP at Neon temperature

No brazing at
condenser/evaporator



Chosen PHP parameters
Material = SS316L
Inner diameter = 1.0 mm
Outer diameter = 2.5 mm
Number of tubes = 20
Length = 440 mm

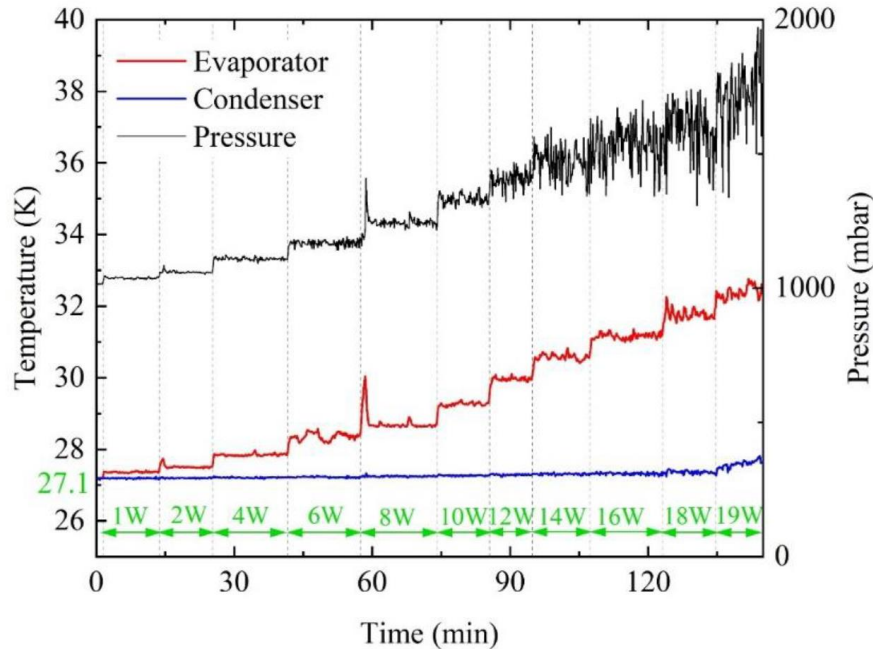


Two-phase thermal links | Neon PHP for magnet (5/5)

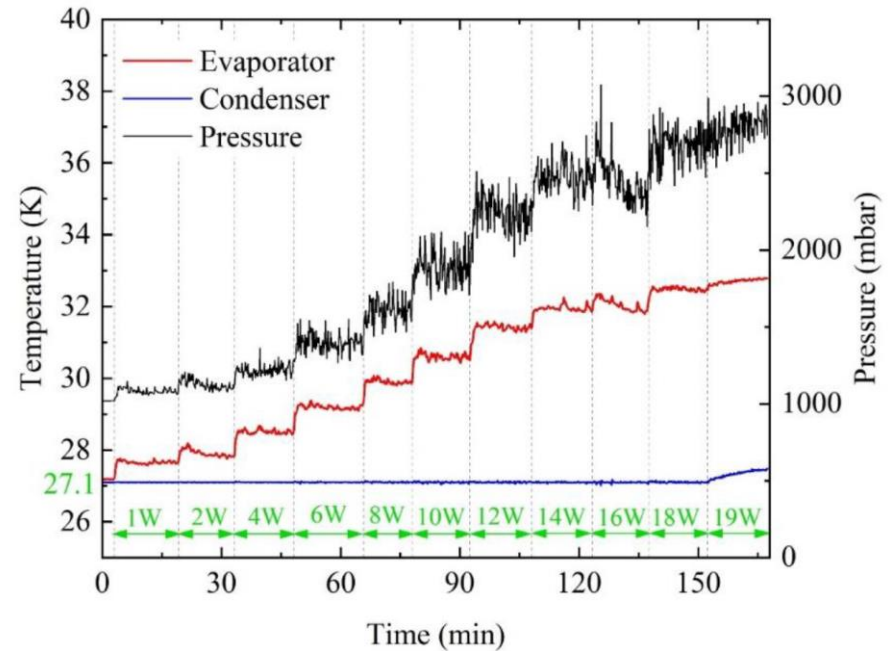
- 10 T class HTS magnet cooled by an PHP at Neon temperature

Vertical tests

FR=20.9%



FR=73.5%



- Maximum capability is reached at 18 W
 - $\Delta T = 4.4$ K for FR = 73.5 %
 - $\Delta T = 5.4$ K for FR = 20.9 %