

Investigation and Considerations about Option #2
(install SuperFGD in ND280 w/o HA-TPC's:
Install SFGD in ND280 – uninstall it – install Bottom
TPC – install SFGD in ND280 again)

Option #2: considerations from WG1

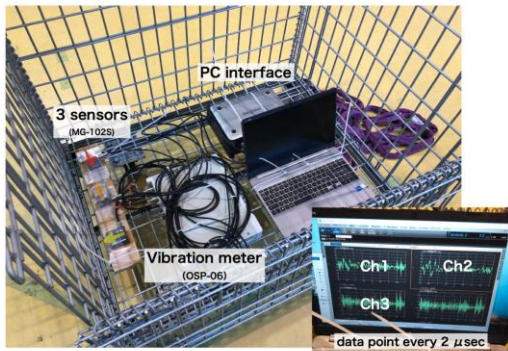
Comments from Franck

- To be considered the cabling/uncabling work
 - not a big issue for SFGD (C.Mauger - Electronics WG)
 - more complicated for ToF detector because it's more fragile and has a lot of cables to be plugged (F.Cadoux)
- It may be considered to install SFGD w/ partial ToF installation, i.e. that does not need to be removed later for Bottom-TPC installation
- Be careful to avoid shocks - Typical risk during detector installation

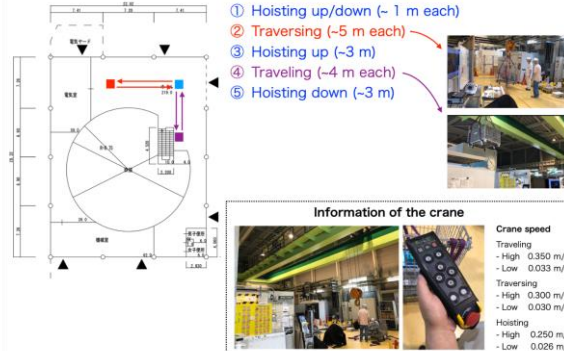
Summary: Risk assessment of the crane acceleration in the lifting

- Acceleration measurement of the crane at the NM building were performed to evaluate the risk in the lifting detector.

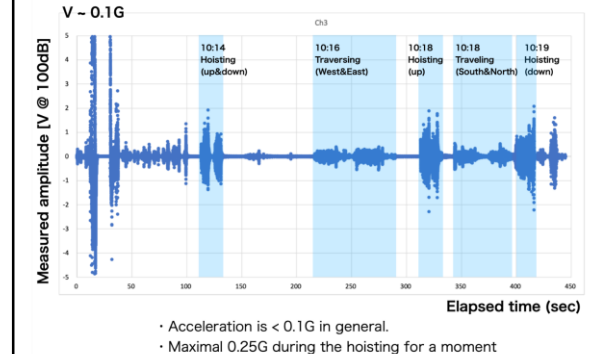
Accelerometer



Crane motions



Ch3 (Vertical direction)



- Comparable with the general standard value of 0.1G in every direction.
- Lower acceleration than the seismic standard of 0.65G is confirmed.
→ Relatively small risk w.r.t. the requirement for an earthquake.
- Other risk mitigations under discussion
 - A guide to the basket during the detector installation
 - Transportation between buildings (NA → NM)

BOX FEA RESULTS

- Deformations at the Box caused by using crane are negligible.
- Vibrations induced from the crane are for sure not worse than vibrations caused by an earthquake

C: Static with G and Nominal Cubes Weight
 Directional Deformation
 Type: Directional Deformation(Z Axis)
 Unit: mm
 Global Coordinate System
 Time: 1
 Custom
 Max: 0.0082694
 Min: -3.3205

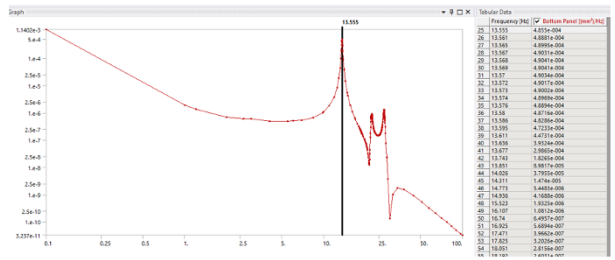
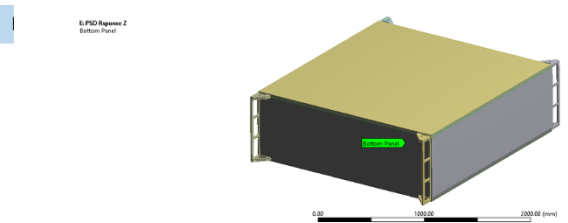
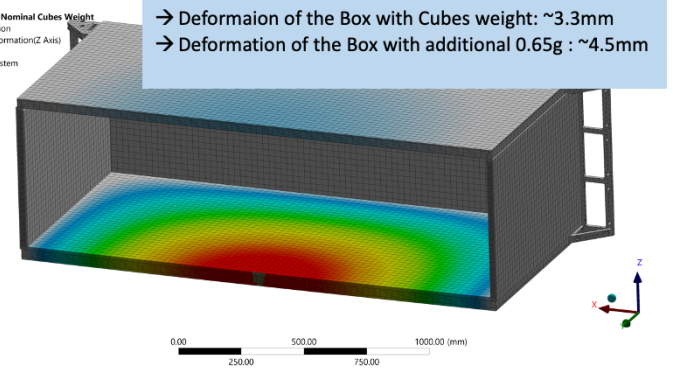


Figure 6-59 Scenario 1 – Vibration in Z direction – Results at the Bottom Panel

A peak is found at 13,555 [Hz] with a PSD of 4.855e⁻⁴ [(mm²)/Hz]

Indicatively vibrational amplitude derived of the Bottom Panel → A=0.081 mm

Crane Acceleration considered for Lifting Device Design:
 Additional 0.1g in every direction

→ Semi-Static FEA were done assuming 0.65g in every direction

EUROPEAN STANDARD **EN 13155**
 NORME EUROPÉENNE
 EUROPÄISCHE NORM
 December 2020
 BS EN 13155:2020
 Supersedes EN 13155:2001+A2:2008

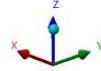
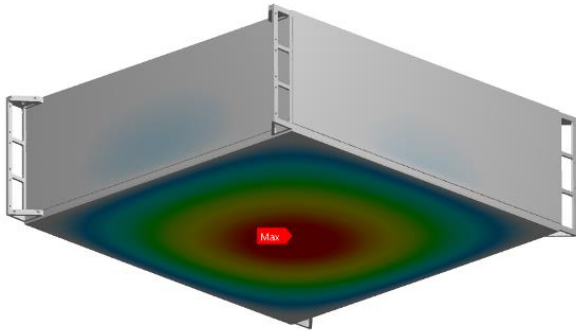
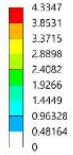
English Version
 Crane - Safety - Non-loaded load lifting attachments
 Approve di legge a legge regionale, Veneto: Attribuzione di legge autonoma
 Norm: Schindler, Lenz,Leinfelder,Reinhardt
 This European Standard was approved for EN on 17 January 2020.
 CEN members are bound to comply with the CEN/CENELEC Internal Regulations which stipulate the conditions for giving this European Standard the status of a national standard without any alteration. Upon authorization of the responsible national authority, CEN members are permitted to make limited deviations to this European Standard in order to take local conditions into account.
 This European Standard applies to those lifting devices (English, French, German, Italian) which are used for lifting loads for machines on the one hand and for lifting devices on the other hand and are not included in the CEN/CENELEC Management Centre for the same category as the official version.
 CEN members on the national standards bodies of Austria, Belgium, Bulgaria, Croatia, Cyprus, Czech Republic, Denmark, Estonia, Finland, France, Germany, Greece, Hungary, Iceland, Ireland, Italy, Latvia, Lithuania, Luxembourg, Malta, Netherlands, Norway, Poland, Portugal, Republic of Moldova, Romania, Slovakia, Slovenia, Switzerland, Sweden, Switzerland, Turkey and United Kingdom.



Considerations from Mechanics WG

A.Gendotti

D: Static Earthquake Load Bottom Panel
Total Deformation
Type: Total Deformation
Unit: mm
Time: 1
Max: 4.3347
Min: 0



Static FEA analysis (worst-case scenario): all cubes move together along the same direction. Very unlikely

1 G along Z (gravity)

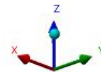
- Max deformation ~2.6mm bottom panel, smooth
- 1.65G along Z (gravity + earthquake)

- Max deformation ~4.3mm bottom panel, smooth

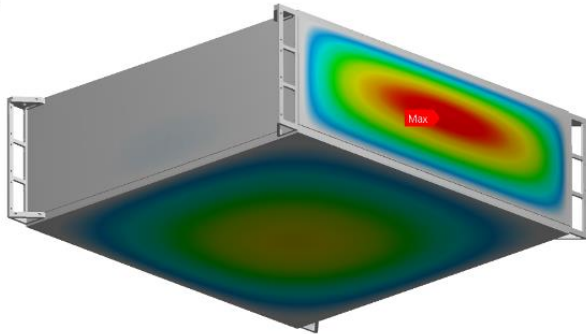
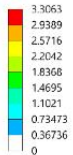
Near the box holes the deformation is almost 0

Still 1G along Z (gravity) and 0.65G along Y

- No change on bottom panel deformation.
- Max ~3mm side panel deformation but smooth
- Near the box holes the deformation is almost 0



E: S
Total Deformation
Type: Total Deformation
Unit: mm
Time: 1
Max: 3.3063
Min: 0



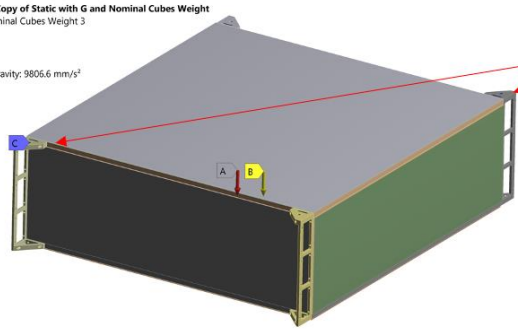
Important consideration: the FEA static shows the worst-case scenario to be considered as a safety margin that the target (5mm max deformation) is achieved

Stress at the Bracket during Lifting

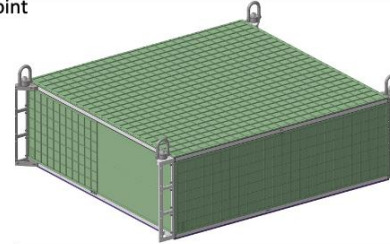
3

O: Copy of Copy of Copy of Static with G and Nominal Cubes Weight
Static with G and Nominal Cubes Weight 3
Time: 1. s

- A Force: 37224 N
- B Standard Earth Gravity: 9806.6 mm/s²
- C Fixed Support



- Force 37'224 N → Cubes weight + 0.65g
- Only 2 Anchoring Point

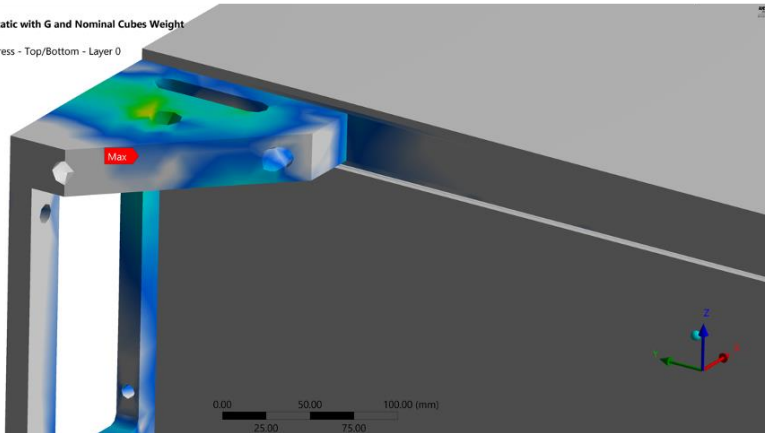
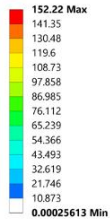


Max Stress at the Bracket:
152Mpa



MoS= 0.577

O: Copy of Copy of Copy of Static with G and Nominal Cubes Weight
Equivalent Stress
Type: Equivalent (von-Mises) Stress - Top/Bottom - Layer 0
Unit: MPa
Time: 1



The static FEA were done by applying the acceleration also to the side panels of 0.65G. Irrelevant the impact of the slightly different boundary conditions between lifting and basket (e.g. 2 vs 8 anchoring points), also given the fact that the static FEA is the very worst-case scenario, very unlikely to happen

Conclusion

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Last Verifications:

- Threads calculations at the aluminum Frame

Threads have a relatively big safety marge in the ALU frame → SF=8 for M6 and SF=3 for M8

- Stress Calculation with finer mesh at the Alu Frames

Stress calculated with finer mesh shows always a consistent low stress of 54 MPa

- Stress Calculation at Brackets during Lifting

By calculating the stress during lifting using only 2 lifting point, the stress is always inside the Margin of Safety
MoS = 0.577 (MoS includes all the safety Factors i.e Material safety factor and load safety factor). If MoS is > or = to zero, it can be considered as safe.

Considerations from Mechanics WG

- No major problems about mechanics and sagging
- Hard to say what happens about fibers inside the box as it can't be modeled. In general, people don't very worried about breaking fibers during the lifting
 - very low crane acceleration
 - conservative assumptions made about cubes in the FEA model (cubes are strong and when pressed it will be a rigid body)
 - However, it's not possible to make a precise model of cubes+fibers inside the box

Overall, the box design was done to be safe for earthquake with minimal deformation. However, it's not possible to model fiber+cube inside box, so there could be other boundary conditions (displacement of cubes from vibrations, variation of size and hole position of the cubes, etc.) that can't be taken into account precisely.

Run plan based on discussions so far

- **March beam time is important because lets ND280 check new beam, so**

A. if March beam time is given, we request

- **1+ α cycles in Jan. for beamline commissioning(w/o ND280),**
- **1 cycle in Mar. w/ ND280**
- **1 cycle in Jun. w/ ND280 partially upgrade, and**
- **3 cycles after October w/ ND280 full upgraded**

B. if March beamtime is not given, we request

- **1+ α cycles in Jan. for beamline commissioning(w/o ND280),**
- **2 cycle in Apr.-Jun. w/ ND280 (possibly w/o upgrade), and**
- **3 cycles after October w/ ND280 full upgraded**