

# Development and Testing of Composite Vacuum Tubes for Einstein Telescope

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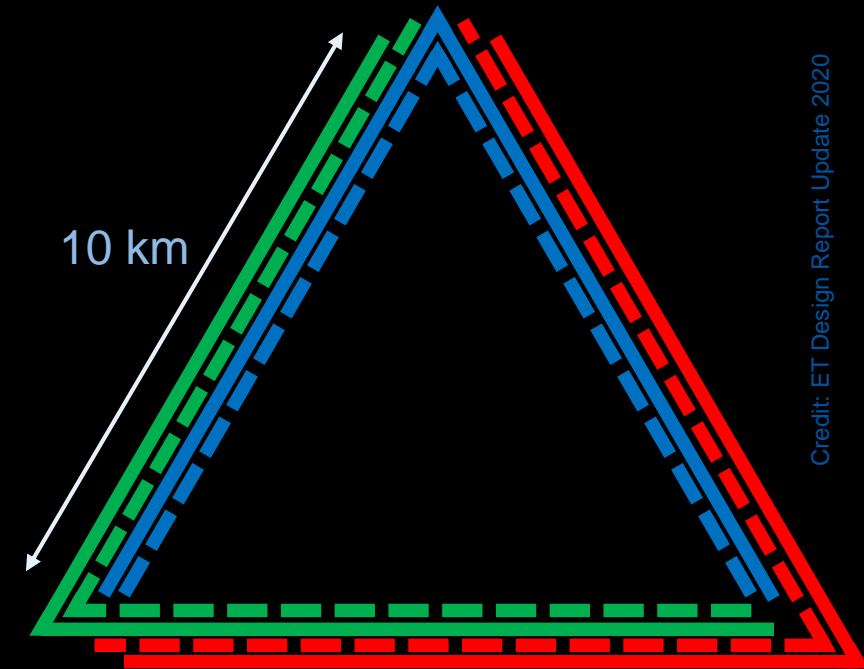
# Vacuum Tubes for ET

- Total vacuum pipe requirement :
  - 3 detectors
  - 2 interferometers per detector
  - 2 arms per interferometer
  - Each arm is 10 km

$$\Rightarrow 3 \times 2 \times 2 \times 10 \text{ km} = 120 \text{ km}$$

- Ultra High Vacuum (UHV) is required,

- Reduce the noise due to excess gas density fluctuations along the beam path
- Reduction of test mass motion and thermal isolation
- Preserving cleanliness of optical elements



Credit: ET Design Report Update 2020

# Vacuum Tubes of Current Gravitational Wave Detectors

Credits: Caltech/MIT/LIGO Laboratory



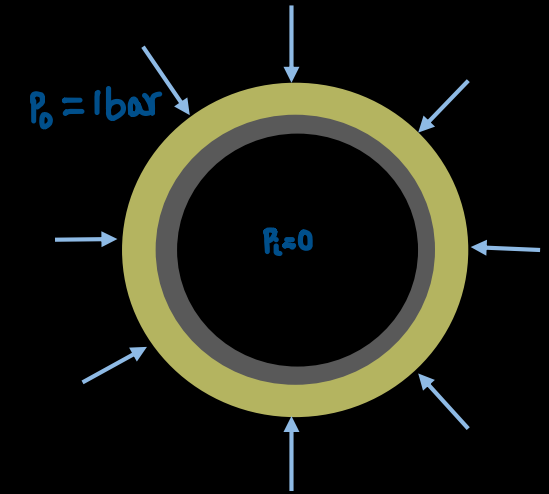
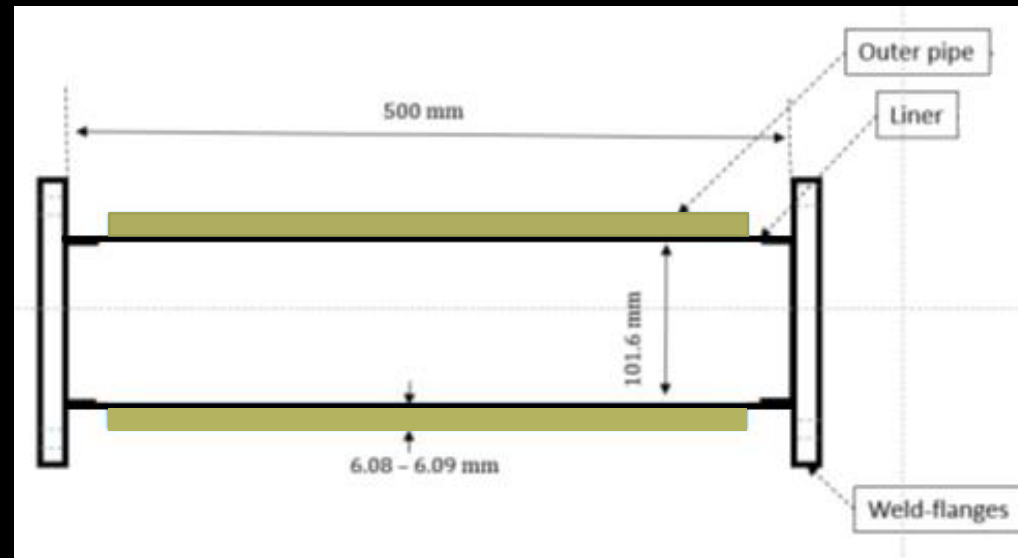
Credits: The Virgo Collaboration



- Stainless steel is the standard due to,
  - UHV compatibility and mechanical integrity
- With diameter up to 1 m and 120 km of vacuum pipes, Einstein Telescope requires,
  - Total volume  $\sim 100,000 \text{ m}^3$  ( $\sim 10$  times greater than LIGO)
  - Total surface area  $\sim 800,000 \text{ m}^2$
  - 4 mm thick stainless steel required  $\sim 100 \text{ kg/m}$  (12000 tons in total)
- Decreasing of stainless steel would help in,
  - Reduction of vacuum firing cost (reduce  $\text{H}_2$  outgassing)
  - Help in logistics of moving and assembly underground

# Composite Vacuum Tube - First Prototype

- Composite Vacuum Tubes : metal liner + fiber composite have the potential of
  - Stainless steel reduction by a factor 10 ( e.g., 4 to 0.4 mm)
  - Lighter tubes which help in logistics of moving and assembly underground
  - On-site production

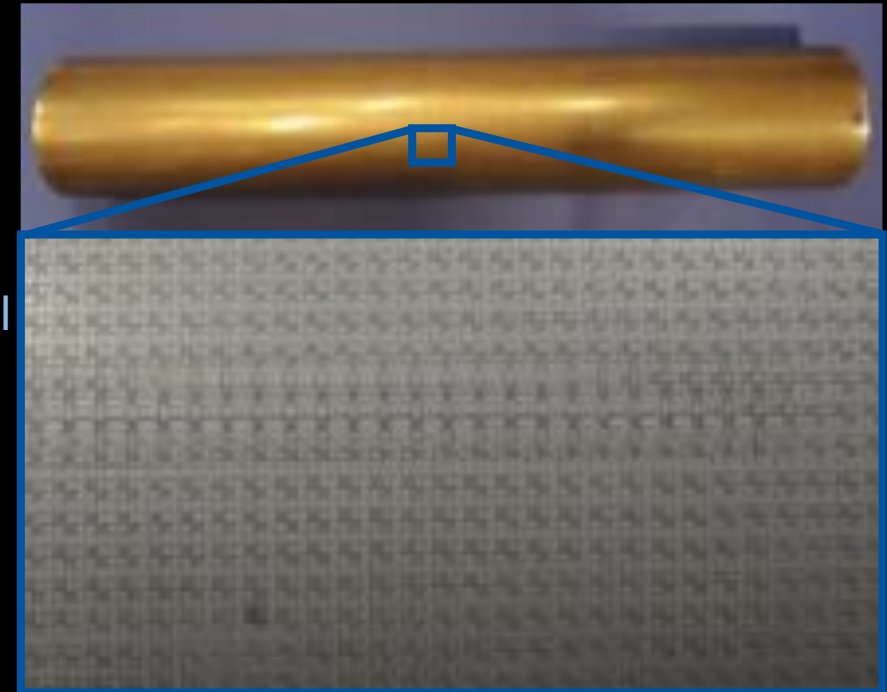
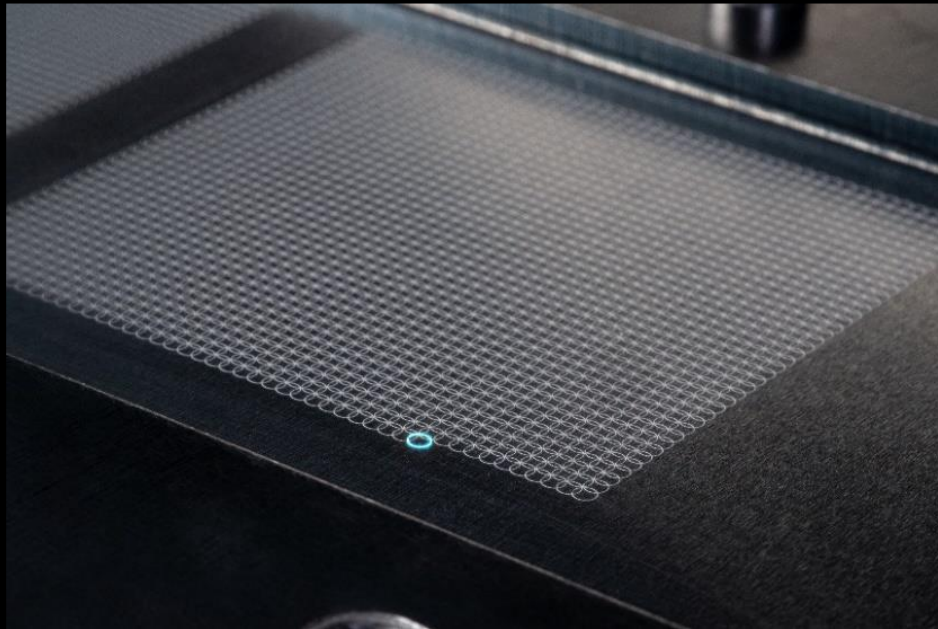


- Metal liner thickness - 0.8 mm
- Composite - 6 mm



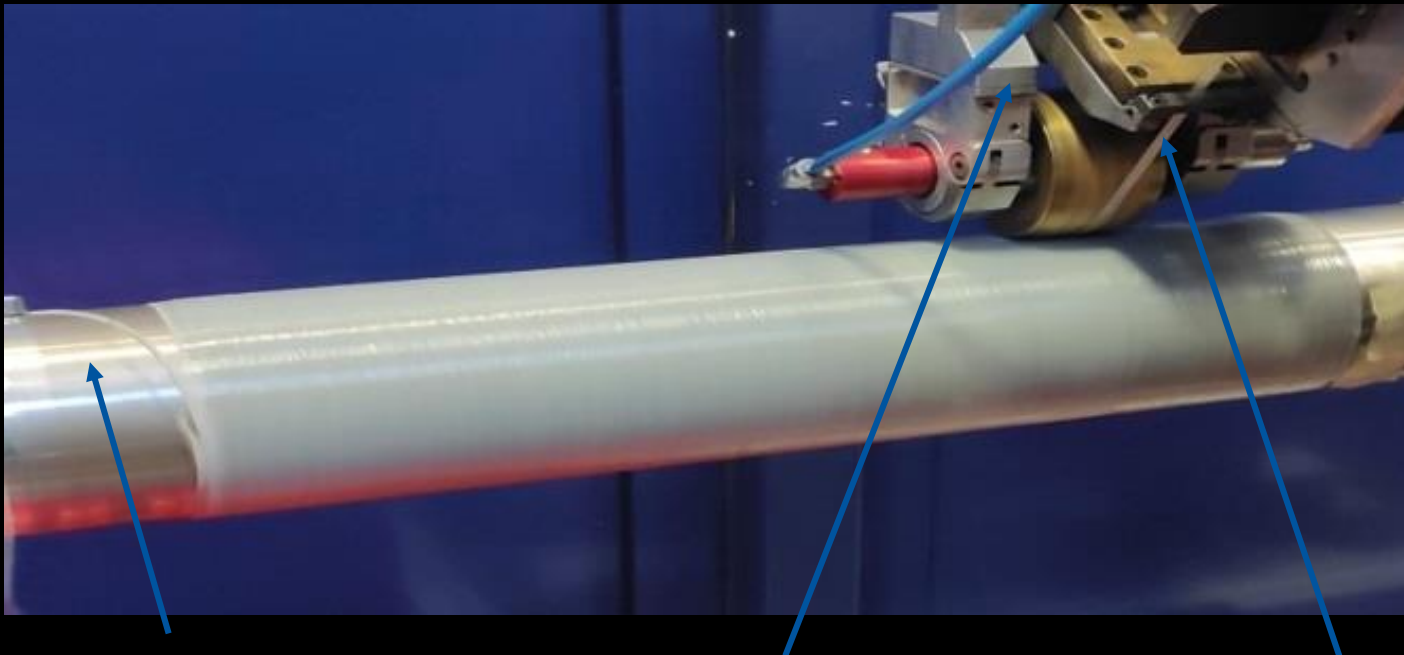
# Manufacturing of the First Prototype

- Stainless Steel Liner (304L,  $t = 0.8$  mm)
  - Laser structuring of the liner : 20  $\mu\text{m}$  deep grooves
  - To improve the bonding between the liner and the composite material



# Manufacturing of the First Prototype

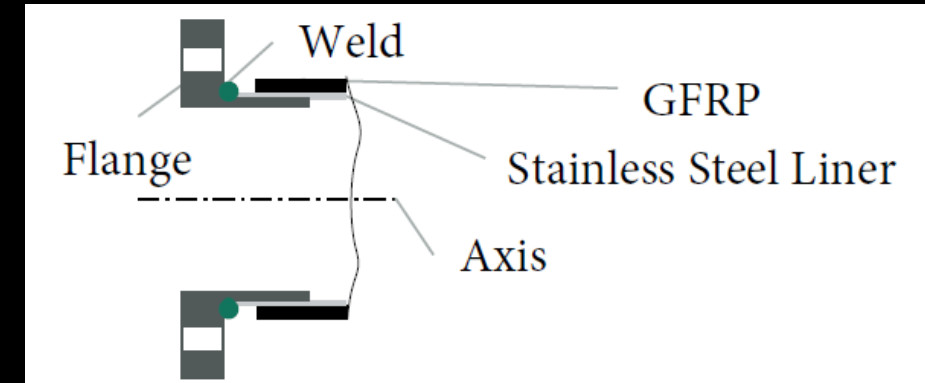
- Composite material - Glass Fiber Reinforced Plastic (GFRP)
  - Glass fiber with epoxy matrix (circumferential winding)
  - Curing Temperature : 120° C (limits the bake-out temp)



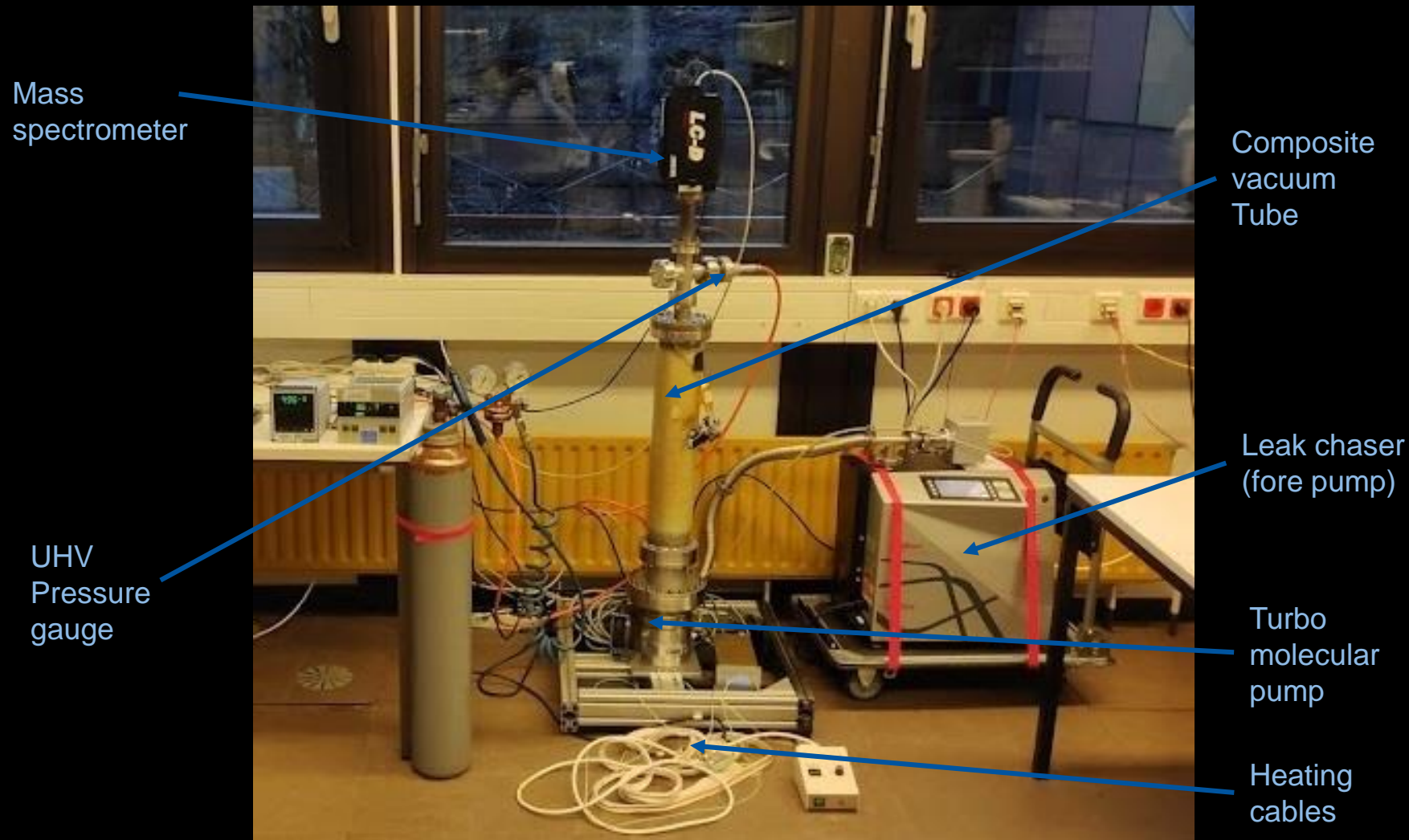
stainless steel liner

winding robot

glass fiber with epoxy matrix



# Vacuum System and Testing

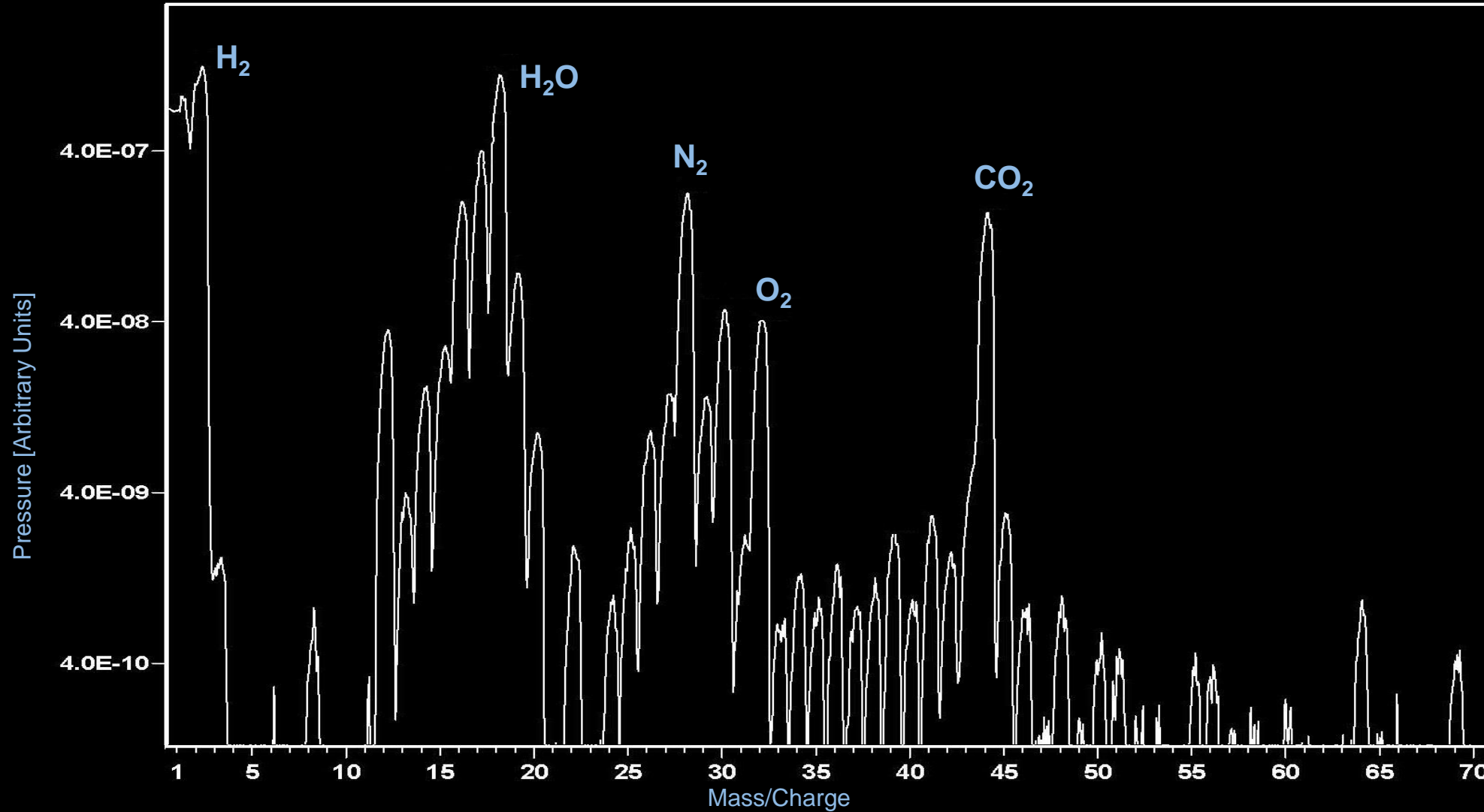


The vacuum test :

- Helium leak test
- With turbo molecular pump after baking the system for 7 days at 100° C

$$P = 7 \times 10^{-10} \text{ mbar}$$

# Residual Gas Spectrum

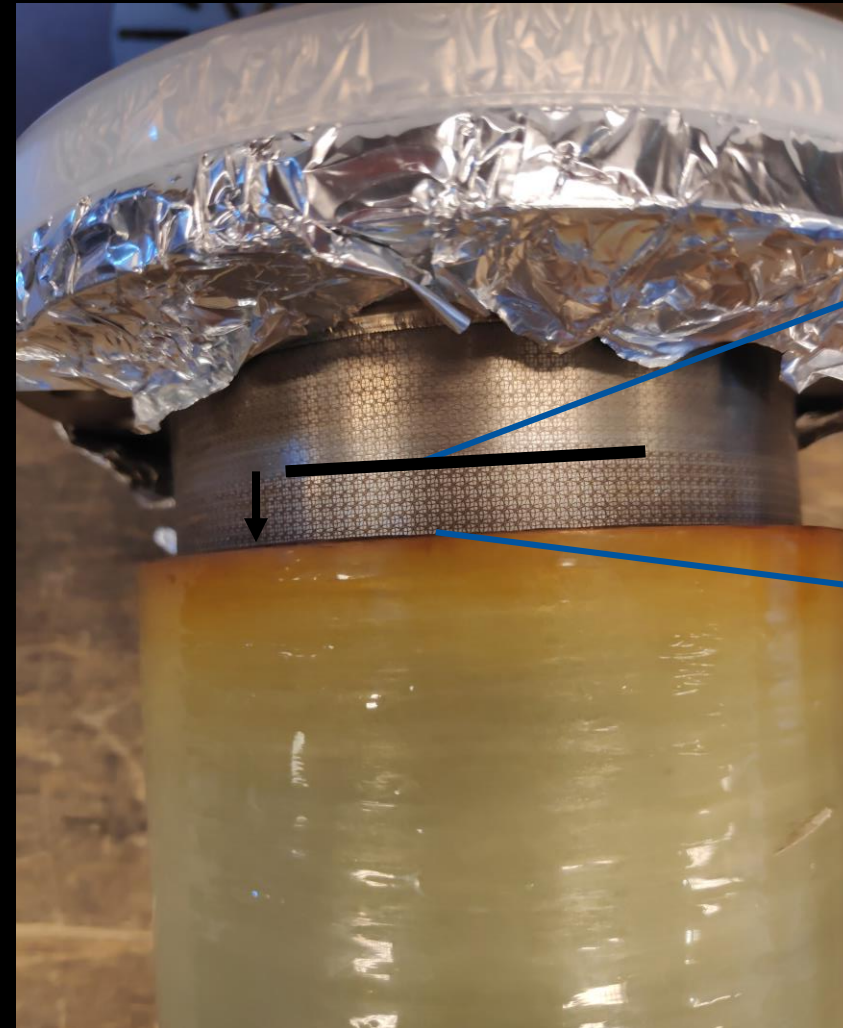


- H<sub>2</sub> and H<sub>2</sub>O limited
- High CO<sub>2</sub> peak



# Delamination of the First Prototype

- Operation for more than 6 months under vacuum and survived multiple bake-outs (100 °C)
- The liner got delaminated from fiber composite tube
- Partial destruction of laser structures while rolling the stainless-steel sheet into a tube



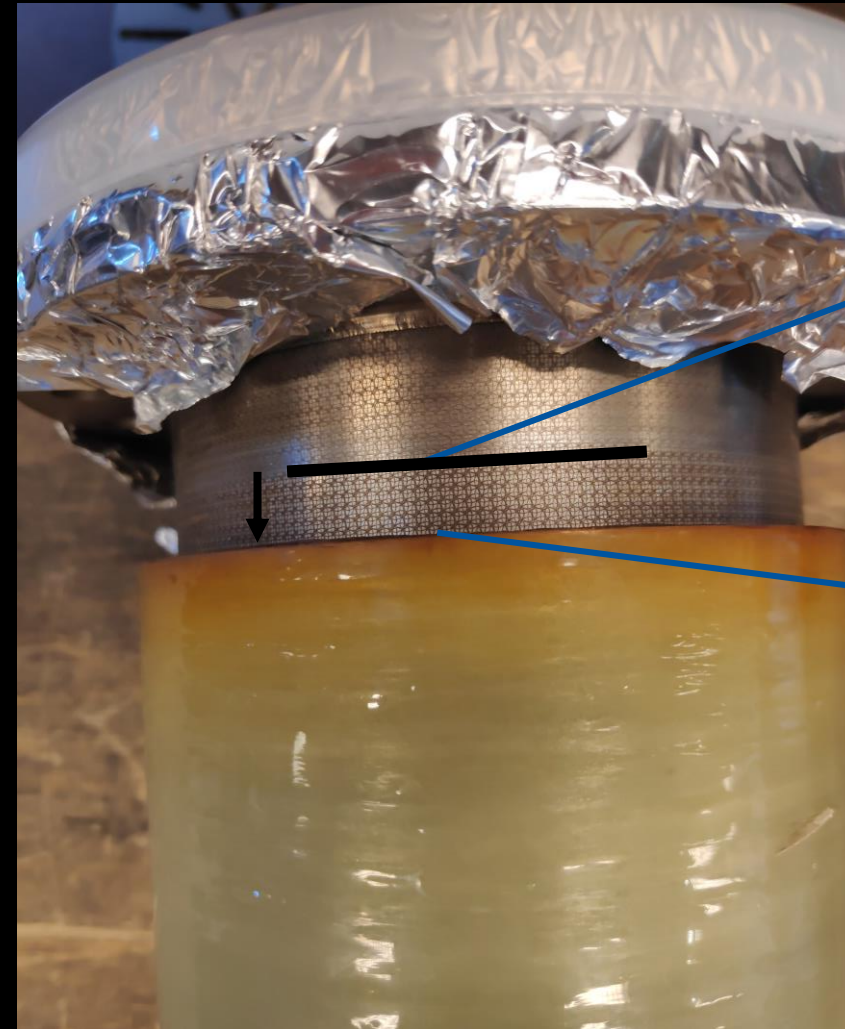
Initial position  
of the GFRP  
tube

Position after  
delamination

# Possible Reasons for the Delamination

- ✦ Mismatch of coefficient of thermal expansion (CTE) of SS304L and fiber composite

Material	Typical CTE ( $\mu\text{m} / \text{m} \text{ } ^\circ\text{C}$ )
SS304L	17.3
Fiber Composite	7.1 (along fibers) 31.3 (transverse to fibers)

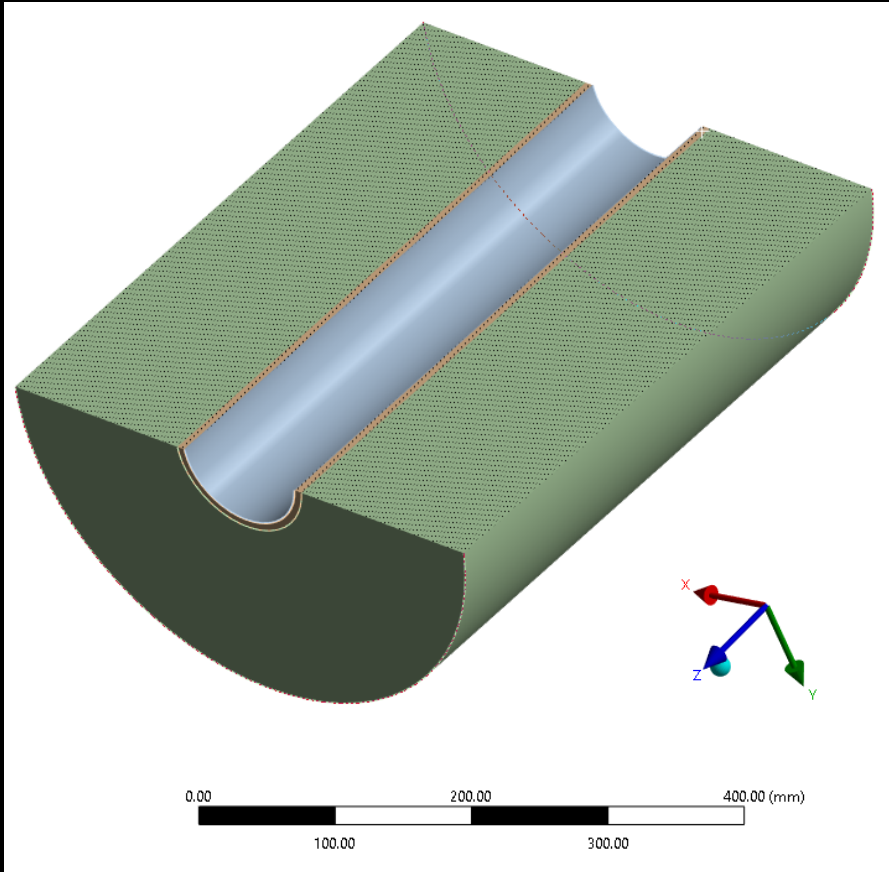


Initial position of the GFRP tube

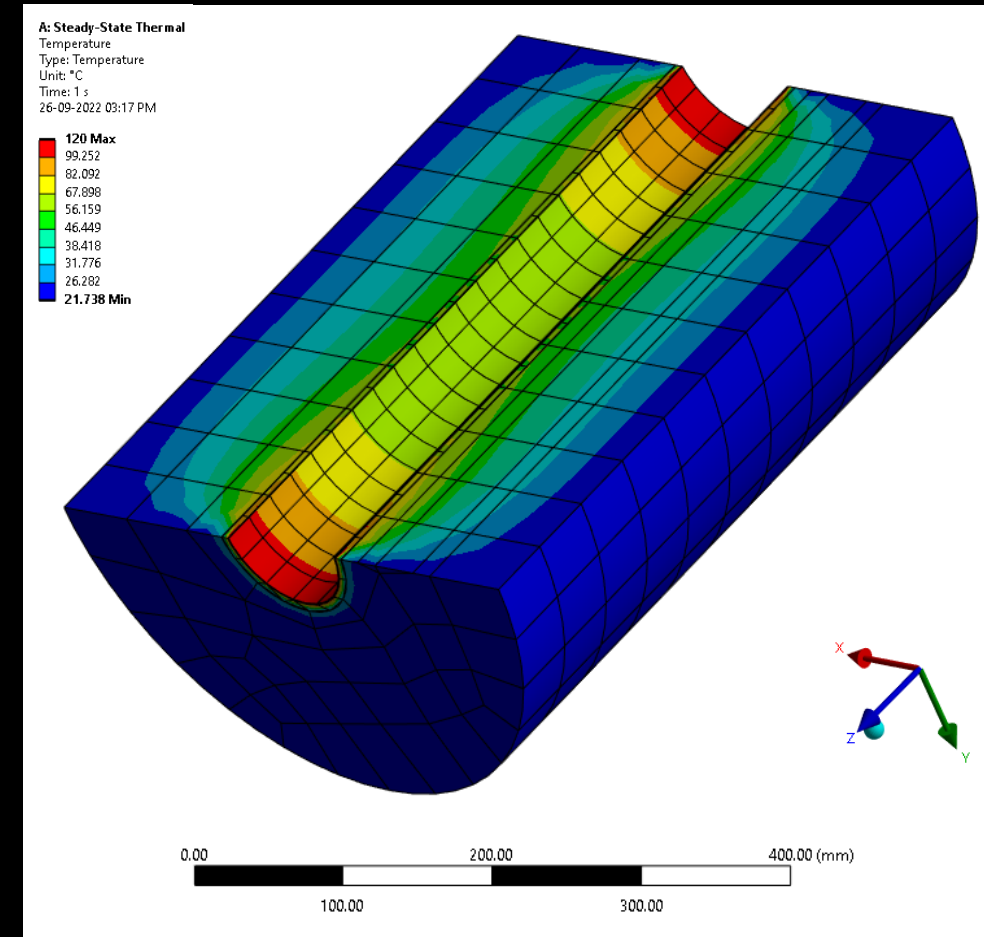
Position after delamination

# Possible Reasons for the Delamination

- Differential thermal expansion due to temperature gradient along and across the composite vacuum tube



- Temperature of 120 °C at the stainless steel ends
- Insulation using mineral wool of 15 cm
- Convection boundary condition with coefficient of  $5 \text{ Wm}^{-2} \text{ °C}^{-1}$



# Conclusions

- The first prototype was under vacuum for more than 6 months
- The fiber composite got delaminated from the stainless-steel liner
- Choice of composite :
  - Epoxy with randomly oriented fibers to make it isotropic and matching CTE with metal
- Simulation :
  - Minimize the material budget
  - Scaling of composite vacuum tube for ET dimensions
- Tests :
  - Shearing tests of chosen composites along with heat cycling test
  - UHV compatibility of composite tubes
  - Overpressure tests (safety margin)
- Design a heating strategy



Buckled Tube



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Thank You!

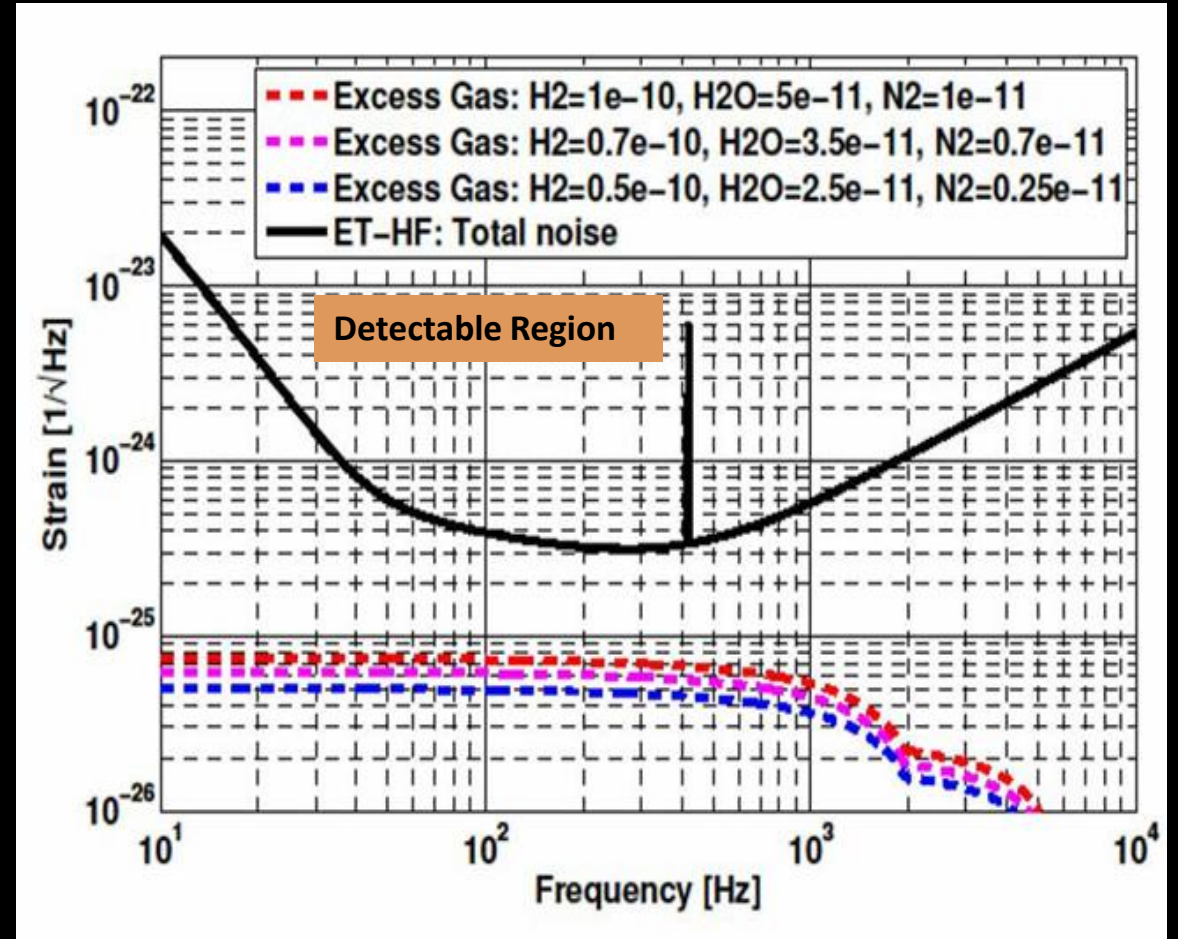
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# Appendix

# Ultra High Vacuum(UHV) Requirement

- Reduce the noise due to excess gas density fluctuations along the laser beam path
- Reduction of test mass motion and thermal isolation
- Preserving cleanliness of optical elements
- Hydrogen ( $1 \times 10^{-10}$  mbar) :
  - From the bulk during stainless steel production
- Water ( $< 5 \times 10^{-11}$  mbar) :
  - Moisture from air
  - Removed using heating/bake-out
- Hydrocarbons ( $< 10^{-14}$  mbar) :
  - Deposited on the surface e.g., oil during machining
  - Careful cleaning using detergents and assembly



1 order sensitivity increase requires 1 order improvement in vacuum

# Vacuum Tubes of Current Gravitational Wave Detectors

## LIGO, USA



Credits: Caltech/MIT/LIGO Laboratory

- ✦ Material : Stainless Steel 304L
- ✦ Diameter : 1.2 m
- ✦ Arm length : 4 km each
- ✦ Achieved pressure :  $2 \times 10^{-9}$  mbar

## Virgo, Italy



Credits: The Virgo Collaboration

- ✦ Material : Stainless Steel 304L
- ✦ Diameter : 1.2 m
- ✦ Arm length : 3 km each
- ✦ Achieved pressure :  $1 \times 10^{-9}$  mbar

## KAGRA, Japan



Credits: KAGRA Observatory, ICRR, The University of Tokyo

- ✦ Material : Stainless Steel 304L
- ✦ Diameter : 0.8 m
- ✦ Arm length : 3 km each
- ✦ Achieved pressure :  $1 \times 10^{-7}$  mbar



# History of Composite Vacuum Tubes at CERN

## New material development for transparent vacuum chambers – Structural composite

Case of an external thin leak tight aluminium envelop with internal C/C reinforcement

- ☺ Good vacuum performance:
  - Outgassing rate ~ 10-12 mbar.l.s-1.cm-2 after bakeout
  - Compatible with NEG coating

	Activation 200 °C for 24h	Activation 250 °C for 6h
<b>H<sub>2</sub> Pumping Speed [l/s]</b>	310	530
<b>Sticking probability [-]</b>	5.9*10 <sup>-3</sup>	1*10 <sup>-2</sup>

Pumping speed with NEG coating

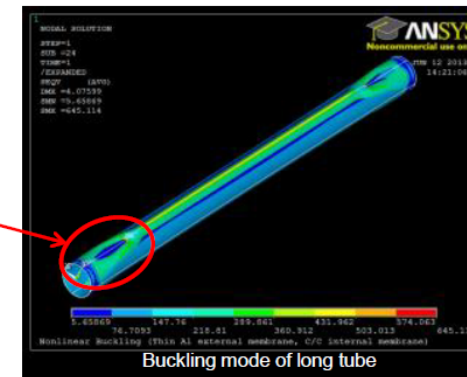


Internal C/C tube

- ☹ Issue with the mechanical behaviour:
  - Differential thermal expansion → buckling of the thin aluminium envelop

- Need to have significant envelop thickness
- Either not reliable or not interesting

Strain  
concentration



(Source :<https://indico.cern.ch/event/278903/contributions/631198/>)



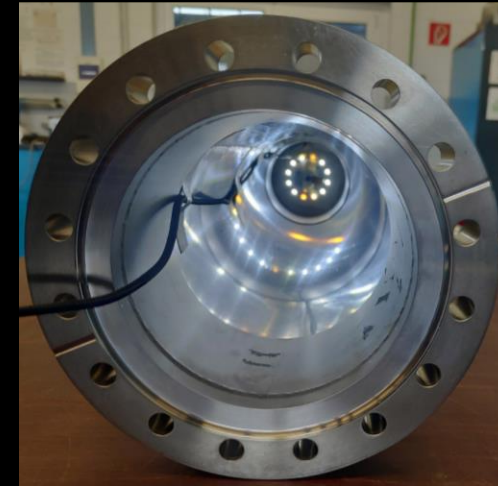
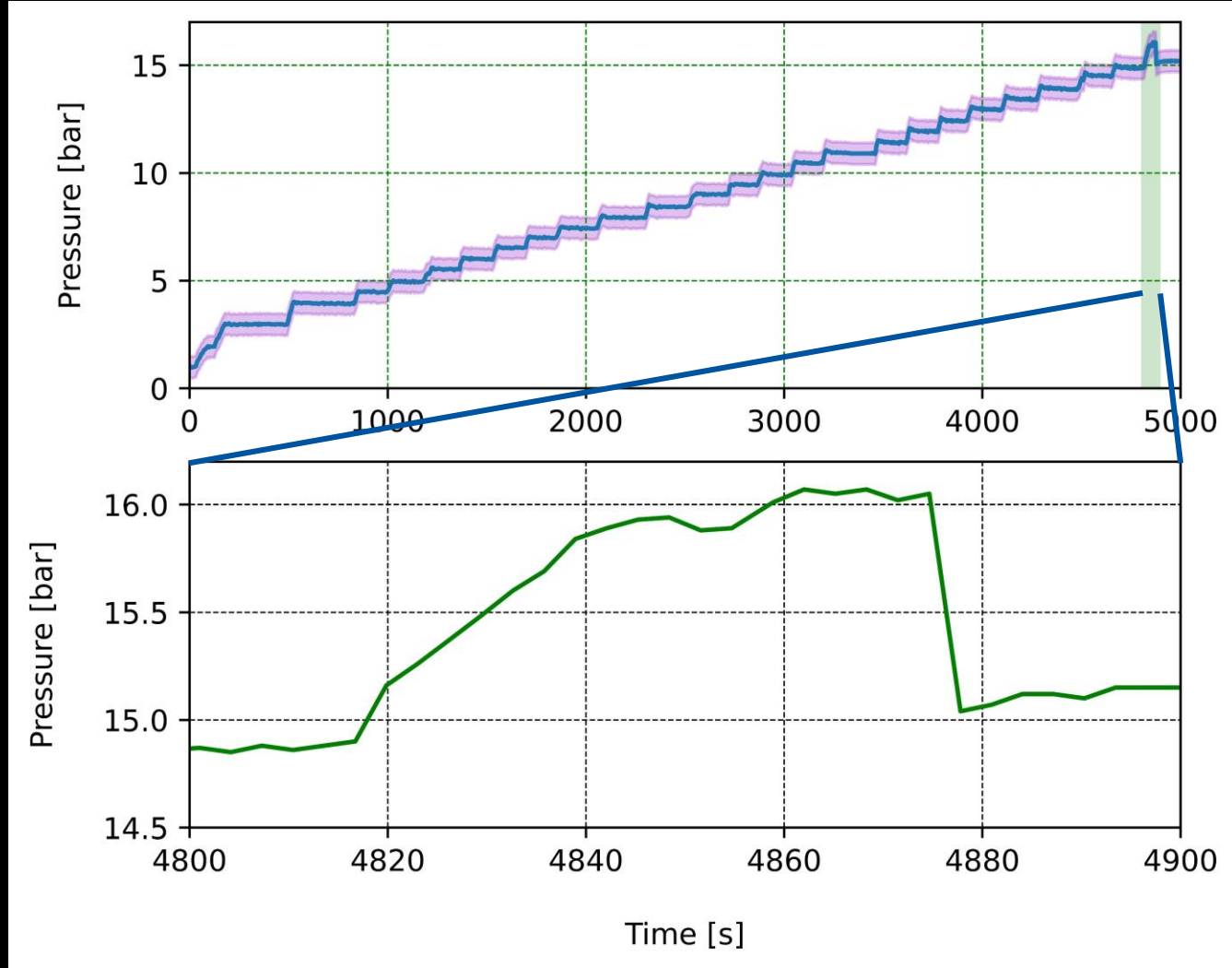
Vacuum, Surfaces & Coatings Group  
Technology Department

LHeC Workshop  
20-21 January 2014

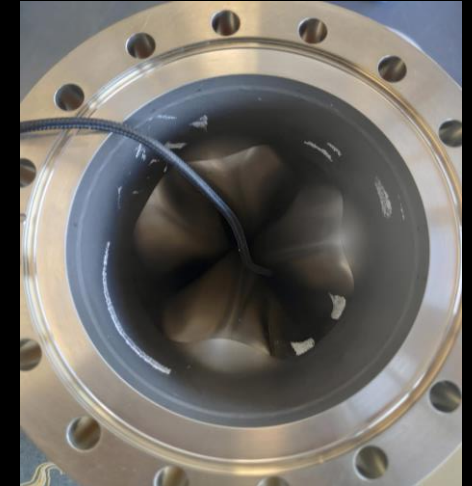
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# Critical Buckling Pressure ( $16.0 \pm 0.5$ bar)

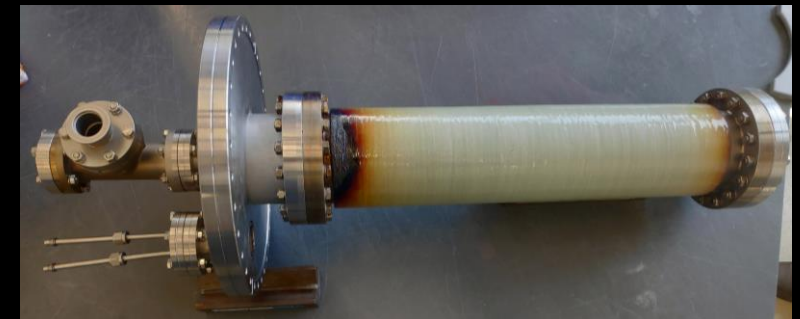


Original Tube

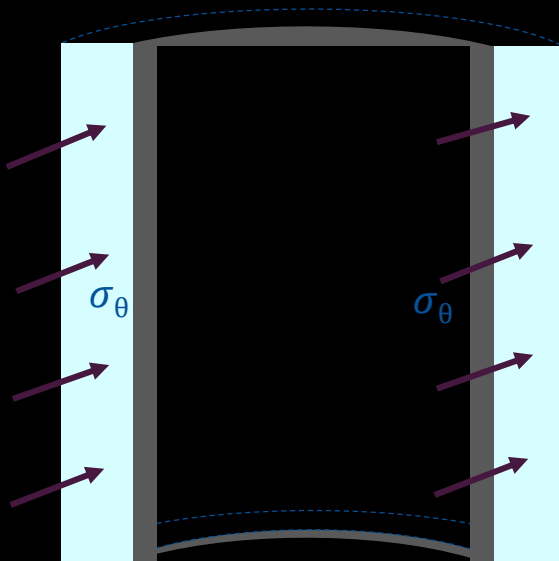


Buckled Tube

- Only the SS304 liner buckled and the GFRP tube stayed intact

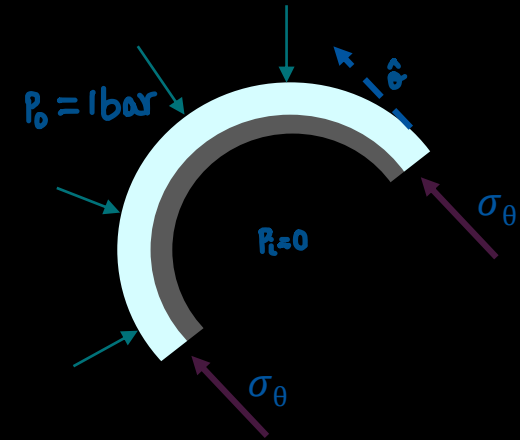


# Hoop/Circumferential Stress in a Cylinder



- Hoop/Tangential stress :  $\sigma_{\theta}(r) = \frac{p_0 r_0^2}{r_0^2 - r_i^2} \left( \frac{r_i^2}{r^2} + 1 \right)$  as  $d/t < 20$  (thick tube)
- Yield Stress = 205 MPa

Design	Thickness (mm)	Hoop Stress max at atmospheric pressure (MPa)
Prototype 1.0	0.8 + 6.08	- 0.88



- As the hoop stresses are compressive, epoxy matrix should support the load and prevent the E-glass fibers from buckling
- Longitudinal Compressive strength of E-glass Epoxy ( $V_f = 0.6$ ) = 620 MPa
- Typical Compressive strength of an unmodified epoxy = 70 MPa

# Overpressure Test

- Overpressure test was performed in order to
  - Verify/set a baseline for the simulation results
  - Effect of the delamination

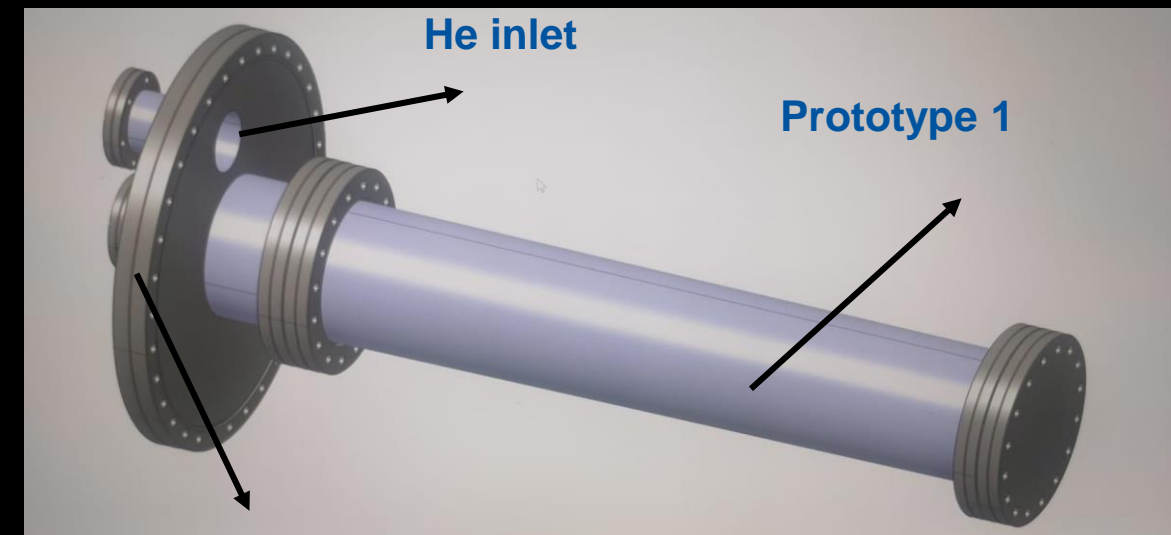
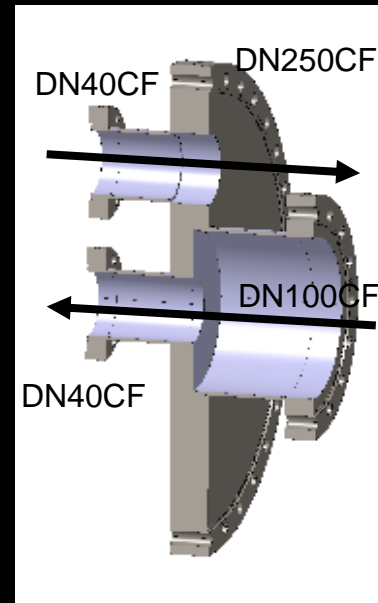


1300 mm

Thickness 2 mm

He 0-16 bar

Leak chaser



DN250CF outer flange