

## Forum on Tracking Detector Mechanics - Universität Tübingen

# CO<sub>2</sub> cooling system prototype for the ATLAS ITk Pixel Endcap detector

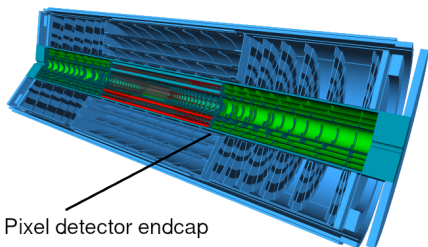
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University of Manchester - INFN Milano - University of Milano - University of Sheffield

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# Introduction

- In preparation for **high-luminosity LHC**  
→ ATLAS detector will be upgraded with a **new silicon inner tracker ITk**
- ITk will rely on **evaporative CO<sub>2</sub>** based cooling system
- In order to test the key aspects of the cooling, a **prototype for the ITk Pixel Endcap Layer 4 half-shell cooling system** was built in Milan
- Prototype tested at the **CERN CO<sub>2</sub> BabyDEMO cooling plant**  
→ facility is able to provide a flow up to 150 g/s with a temperature as low as **-45 °C**



# Outline of the talk

- Introduction to **ITk Pixel Endcap cooling system**
- **Layer 4 prototype**
  - ▶ mechanical construction
  - ▶ capillary sizing
  - ▶ sensors
  - ▶ thermal load
- **Measurements performed at BabyDEMO plant**
  - ▶ nominal detector operations conditions
  - ▶ transient and extreme conditions
- **Conclusions**

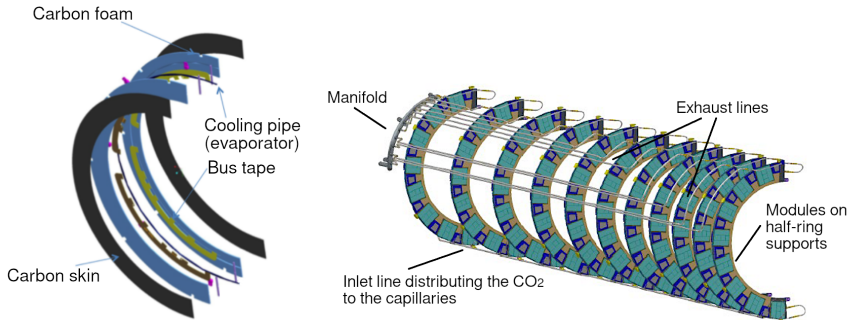


# ITk Pixel Endcap cooling system

# ITk Pixel Endcap: cooling system overview

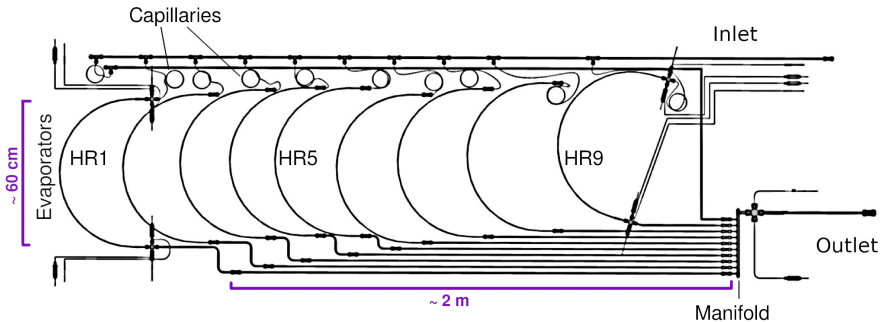
Pixel modules mounted on half-ring local supports.

Injecting liquid  $\text{CO}_2$  in the system, the  $\text{CO}_2$  reaches the **two-phase state** over the detector local supports evaporator tubes:



Layer 2 CAD by Fred Gannaway (QMUL)

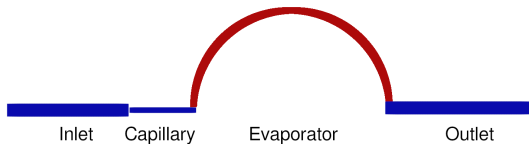
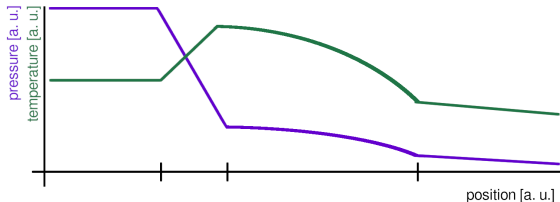
## Cooling system and flat Layer 4 prototype overview



- Common **inlet line**
- **Capillaries** distributing CO<sub>2</sub> to all the **half-ring evaporators**  
→ capillary crucial both to trigger CO<sub>2</sub> evaporation and to equalize pressure drops in the system
- **Exhaust line** collecting two-phase CO<sub>2</sub> and bringing it to the **manifold**
- Prototype is flat to simplify thermal insulation

# Evaporative CO<sub>2</sub>

Capillary ensures that the evaporation **starts at the beginning of the evaporator**: liquid very close to saturation, it starts bubbling as soon as it enters the evaporator

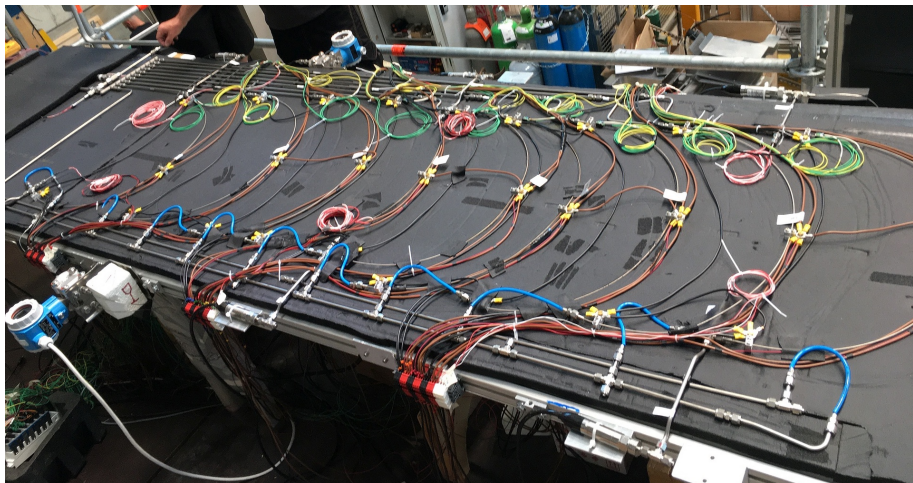


## Specifications for the ITk cooling system:

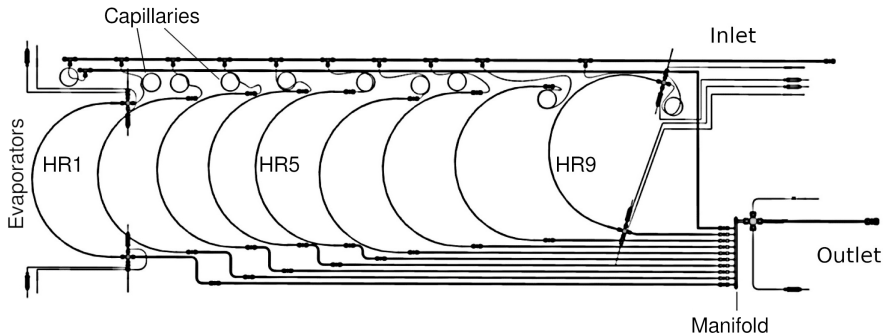
- vapor quality of 30% in the evaporator
- temperature drop within 5°C
- pressure drop in capillary: 8 bar
- pressure drop in evaporator + exhaust line within 2 bar

# Mechanical construction of the prototype





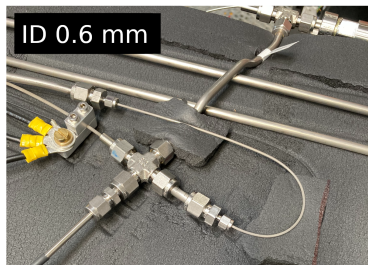
# Overview of the Layer 4 prototype



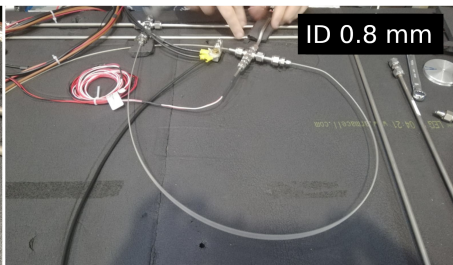
- All tubes are **grade 2 titanium**
- Inlet and outlet lines connected to **BabyDEMO plant**
- **Capillaries** measured and sized at CERN
- 3D printed flat **manifold**
- **Heating system** to simulated detector thermal load
- Temperature and pressure **sensors**

# Capillaries

- **Two types of capillaries tested**, both with OD 1/16":  
ID 0.6 mm and ID 0.8 mm.
- Preliminary FLUDY and COBRA simulations to predict length, but dedicated measurements to find correct lengths at BabyDEMO plant to achieve the 8 bar pressure drop



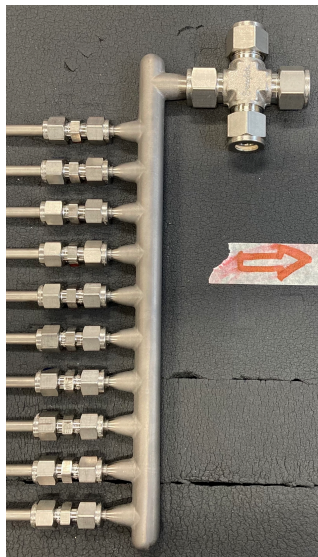
Capillary length: 22.5 cm



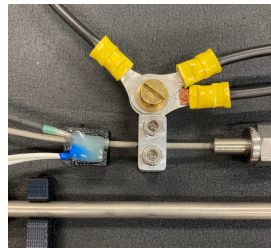
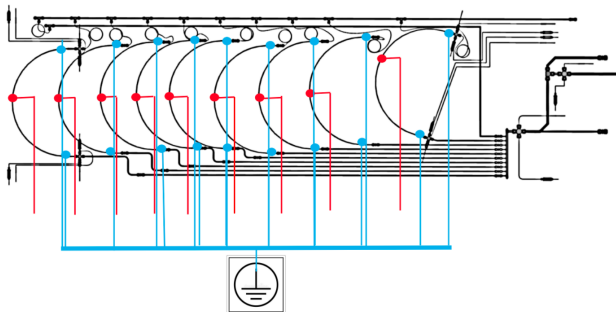
Capillary length: 90 cm

# Manifold

- Flat **manifold 3D printed in titanium** at LAMA Labs in Udine
- To prevent debris:
  - ▶ thermal treatment
  - ▶ cleaning in ultrasonic bath
  - ▶ hirtisation foreseen for real detector manifold

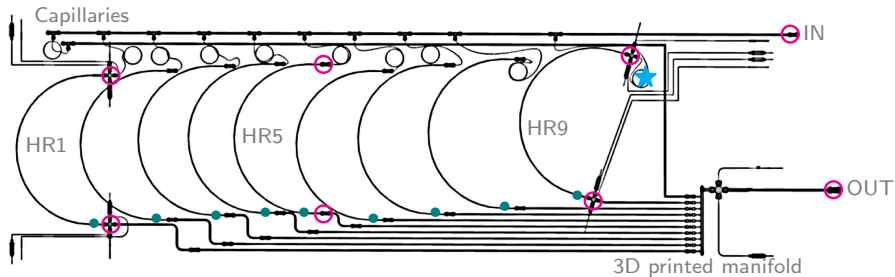


## Heating system



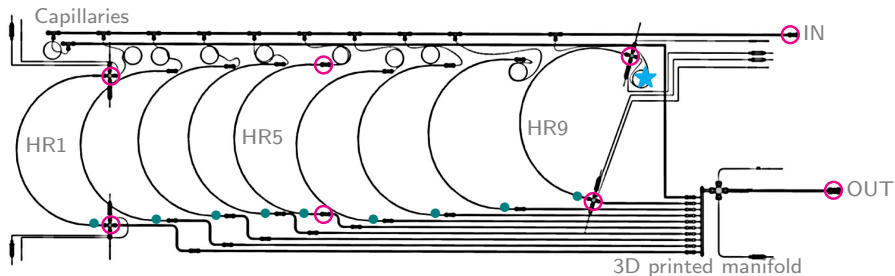
- In order to simulate the **detector thermal load**, HRs connected to power-supplies and treated as resistors
- Nominal power applied to each HR: 300 W ( $5\text{ V} \times 60\text{ A}$ ), maximum power to be tested estimated to be  $\sim 400\text{ W}$
- **Red lines**: current injection points
- **Blue lines**: connection to ground

## In-flow sensors

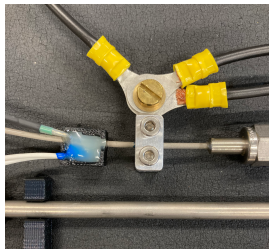


- **In-flow** pressure and temperature sensors (○)
  - Global IN and OUT
  - HRs 1, 5 and 9 IN and OUT
- PT100 temperature sensors
- EPSAS pressure transmitters with  $\pm 32$  mbar accuracy
- **DeltaBar** (★) for pressure measurement of the capillary of HR9, with  $\pm 12$  mbar accuracy

## External sensors



- **External wall temperature** and interlock temperature sensors (●) glued to each evaporator pipe
- All sensors (except interlock ones) read through LabView



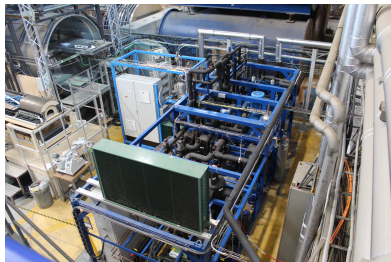
# Measurements at BabyDEMO



# Summary of measurements [1]

Measurements performed at the BabyDEMO plant:

- First measurements with HR9 evaporator loop only, to **size the capillaries** to give a pressure drop of  $\sim 8$  bar with 3g/s  $\text{CO}_2$  flow
- Full system measurements at **ITk nominal operation conditions**:
  - ▶  $\text{CO}_2$  flows 30 g/s and  $-40^\circ\text{C}$
  - ▶ thermal loads of 300 W for each evaporator
  - ▶ with two different sets of capillaries, ID 0.6 mm and 0.8 mm

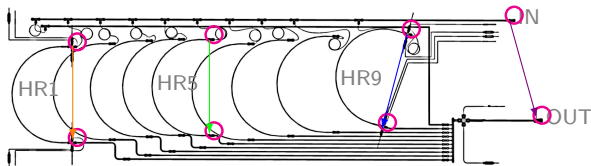


## Summary of measurements [2]

- Measurements in **stable conditions** with the full system, at different operating conditions:
  - ▶ CO<sub>2</sub> flows [30 g/s and 40 g/s]
  - ▶ CO<sub>2</sub> temperatures [-20 °C, -35 °C, -40 °C]
  - ▶ thermal loads [0 W, 100 W, 200 W, 300 W and 400 W for each evaporator]
  - ▶ two sets of capillaries [ID 0.6 mm and 0.8 mm]
- Measurements of various situation with **transitions**, changing the power to single (or pairs of) evaporators
- Extreme thermal load conditions, up to 700 W for each evaporator
- Finally, measurements decreasing the flow to detect **dry-out** conditions, using nominal power, at 0 °C, -20 °C and -40 °C.

→ Only a selection of the results is shown in the following

## Pressure drop after sizing the capillaries ID 0.6 mm

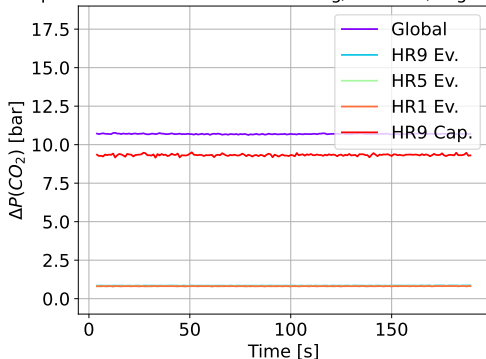


In-flow CO<sub>2</sub>  $\Delta P$  as a function of time between:

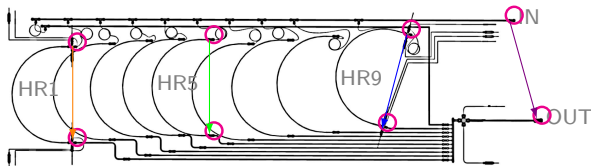
- Global IN and OUT
- HR9, HR5 and HR1 evaporator IN and OUT
- HR9 capillary with DeltaBar

- Nominal conditions, CO<sub>2</sub>  
T = -40°C and 30 g/s,  
using 300 W/evaporator
- $\Delta P$  around 9.5 bar over  
capillary, length of  
0.225 m
- System stable over time
- Same behaviour for the  
HR evaporators

Cap.ID=0.6mm T=-40C Flow=30g/s Power/ring=300W



## Pressure drop after sizing the capillaries ID 0.8 mm

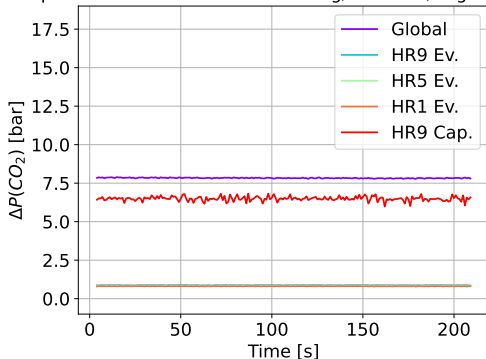


In-flow CO<sub>2</sub>  $\Delta P$  as a function of time between:

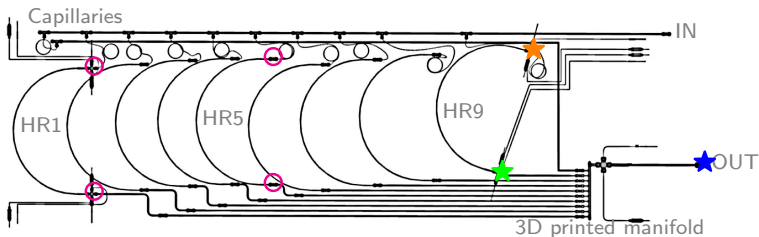
- Global IN and OUT
- HR9, HR5 and HR1 evaporator IN and OUT
- HR9 capillary with DeltaBar

- Nominal conditions, CO<sub>2</sub> T = -40 °C and 30 g/s, using 300 W/evaporator
- $\Delta P$  around 6.5 bar over capillary, length of 0.9 m
- System stable over time
- Same behaviour for the HR evaporators

Cap.ID=0.8mm T=-40C Flow=30g/s Power/ring=300W

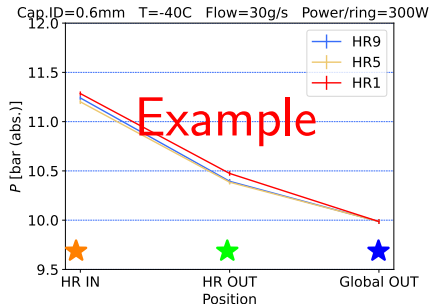


# Pressure drop and temperature in HR and exhaust lines



Showing the in-flow  $\text{CO}_2$  pressure drop and temperature as a function of the position along the prototype:

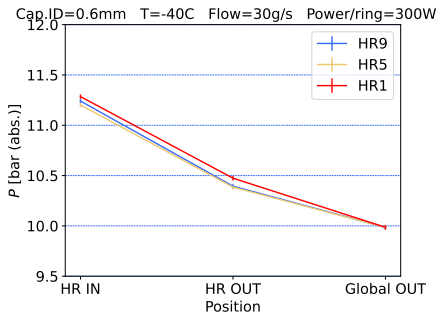
- from HR IN to HR OUT  
→ evaporator
- from HR OUT to Global OUT  
→ exhaust line



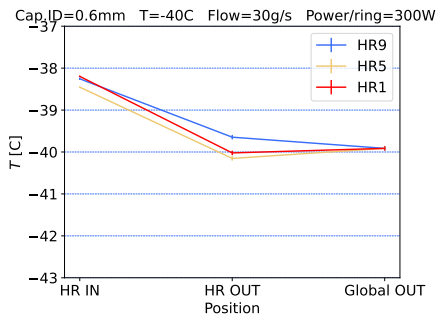
# $\Delta P$ and T in evaporators and exh. lines - Cap. ID 0.6 mm

CO<sub>2</sub> T = -40°C and 30 g/s, using 300 W/evaporator

## Pressure



## Temperature



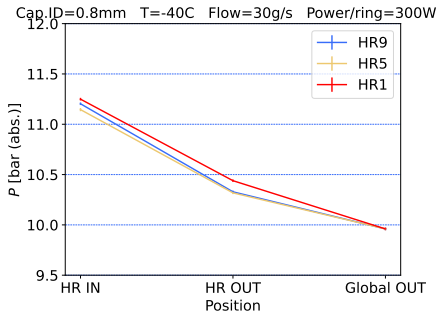
Pressure drop is 1.5 bar, within the 2 bar specification

Temperature difference is 2°C, within the 5°C specification

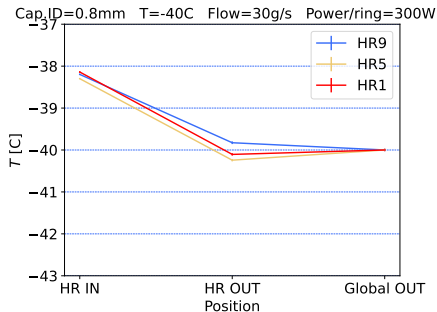
# $\Delta P$ and T in evaporators and exh. lines - Cap. ID 0.8 mm

CO<sub>2</sub> T = -40°C and 30 g/s, using 300 W/evaporator

## Pressure



## Temperature

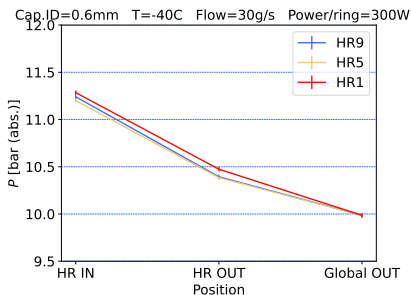


Pressure drop is 1.5 bar, within the 2 bar specification

Temperature difference is 2°C, within the 5°C specification

# Pressure drop in the exhaust lines

## Pressure drop at nominal conditions



- At the nominal operating conditions, the pressure drop in the longest exhaust line (HR1) is 0.5 bar
- ID 4.55 mm pipes were used for the exhaust line, since the ID 4 mm ones foreseen for the real system were not available yet
- Need to correct pressure drop for the smaller ID: the correction is a factor 2, computed from analytic formula and FLUDY simulation

Pressure drop is 1 bar in the exhaust line after the correction and 1 bar in the evaporator → total pressure drop is 2 bar



# Conclusions

# Conclusions

- ITk Pixel Layer 4 cooling system prototype **successfully tested** at the BabyDEMO plant
  - We thank the CERN cooling group and Joao Noite for their support
- Only selected set of measurements shown today, but results show that **the system is operational in all the tested condition**
  - pressure and temperature drops are **within the specification**
  - the **system is stable, even in extreme conditions**, like power variation and temperature/flow condition different from the nominal ones (some reported in the backup slides)
- Group is now moving to the **production of the real detector cooling system**

# Outlooks

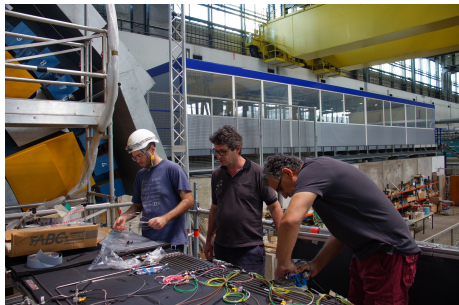
Preparation for **production of the real detector cooling system** for all the layers, challenges:

- 3D **bending** of the pipes
- orbital **welding** of Ti pipes
- preparation of the **capillaries** and brazing to the inlet line

Currently working on 3D prototype for Layer 2 of the system to test the full production process



Prototype of Layer 2 exhaust line and manifold on integration tool



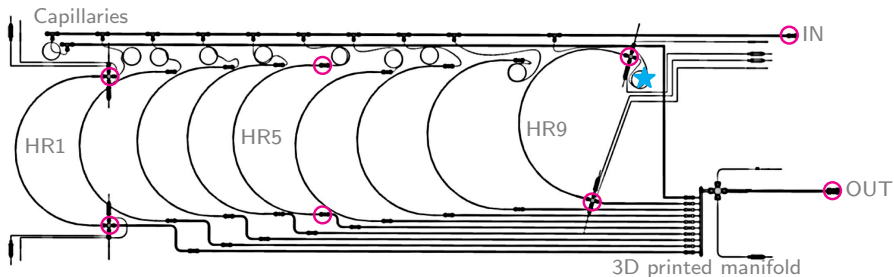
# Backup

# Mechanics and sensors

# Power-supplies

- Each power-supply connected to two HRs, except HR9 that has its own power supply.
  - This ensures the electrical isolation of all the HRs, power consumption only over HRs.
- Using sensing connections: performing measurement of voltage and giving feedback to the power supply itself.
  - This ensure that the voltage and current, read on the power supply, are exactly the ones applied to the HRs.
- Power-supplies used:
  - ▶ three Delta Electronic SM 30V-200A power-supplies for HR1-HR2, HR3-HR4 and HR5-HR6,
  - ▶ two EA 80V-120A power-supplies connected in series for HR7-HR8,
  - ▶ one Delta Electronic SM 15V-100A power-supply for HR9.

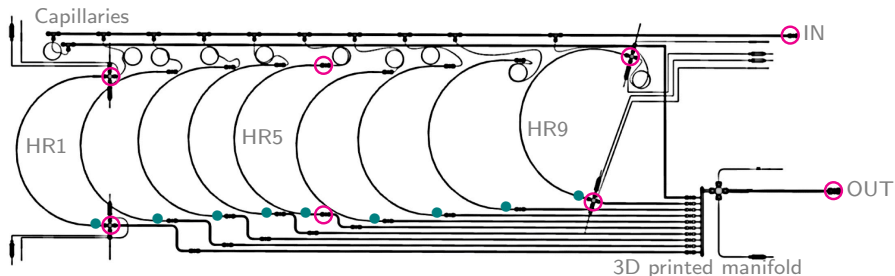
## Details on pressure sensors



- 8 EPSAS **pressure transmitters**,  
output 4 mA–20 mA, calibration 0 bar–80 bar ABS  
accuracy: 0.04 %, FS = 80 bar  $\rightarrow$   $\pm 32$  mbar
- 1 **DeltaBar** PMD75 ENDRES+HAUSER,  
output 4 mA–20 mA, calibration  $-16$  bar/ $16$  bar  
accuracy: max. measurement error 0.075 %  $\rightarrow$   $\pm 12$  mbar



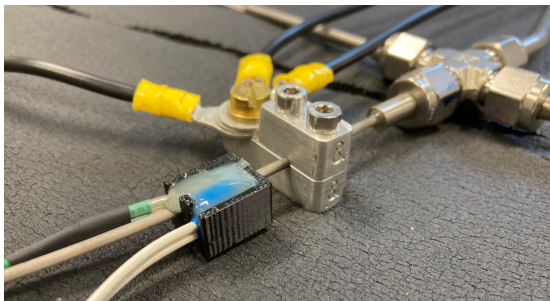
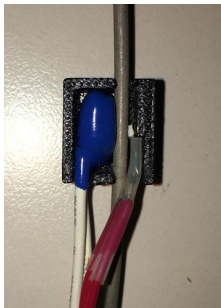
## Details on temperature sensors



- 8 **temperature transmitters**, PT100 with 4 wires
- external sensors glued to the HRs:
  - ▶ sensors for **dry-out** and pipe temperature: 9 RTD PT100  $\varnothing$  2 mm
  - ▶ sensors for power-supplies **interlock**: 9 thermal switches T11A, 80 °C

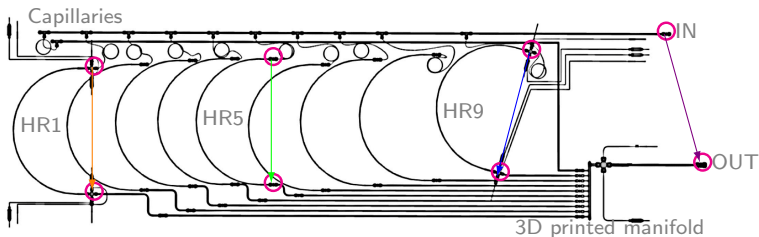
## Sensor gluing

- Small support, 3D printed, to position the dry-out and interlock sensors around the pipe.
  - ▶ Helps the gluing procedure and protects from torsions that could detach the sensors from the pipes.
- First use Loctite 315, to glue the sensor to the pipe.
- Then fill the support with Araldite 2012.



# $\Delta P$ and T plots

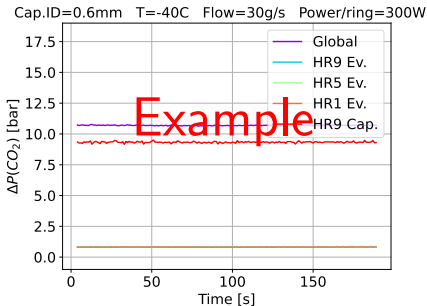
## Pressure after sizing the capillaries



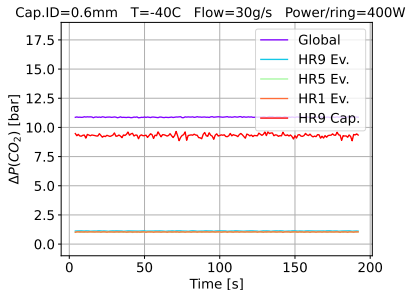
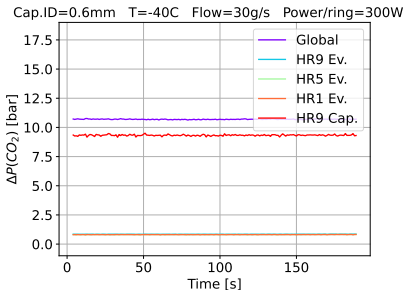
Shown in the following in-flow CO<sub>2</sub>  $\Delta P$  as a function of time between:

- Global IN and OUT,
- HR9, HR5 and HR1 IN and OUT
- HR9 capillary with DeltaBar

The plot title shows the capillary ID, the CO<sub>2</sub> temperature, the flow and the power.



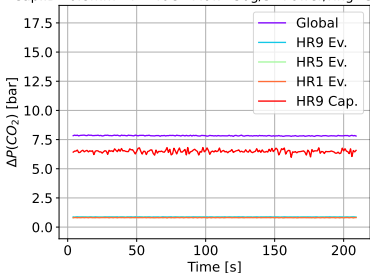
## After sizing the capillaries - ID 0.6 mm



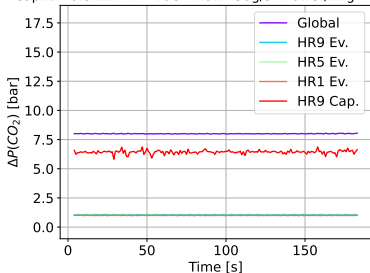
- Plots for  $T_{ev} = -40^\circ\text{C}$  and 30 g/s, using 300 W and 400 W.
- $\Delta P$  around 9.5 bar over capillary with ID 0.6 mm and length of 0.225 m.
  - Final ID 0.6 mm length should be around 20 cm.
- $\Delta P$  for HR1, HR5 and HR9 is the same, despite different capillary shapes (due to space).

## After sizing the capillaries - ID 0.8 mm

Cap.ID=0.8mm T=-40C Flow=30g/s Power/ring=300W

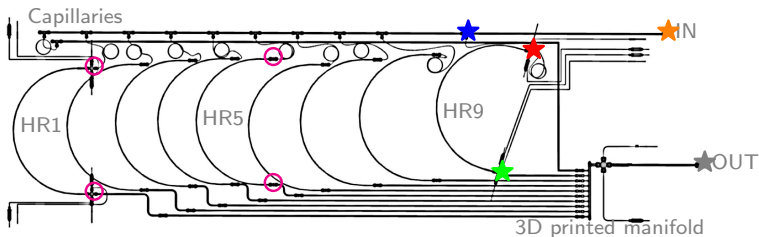


Cap.ID=0.8mm T=-40C Flow=30g/s Power/ring=400W



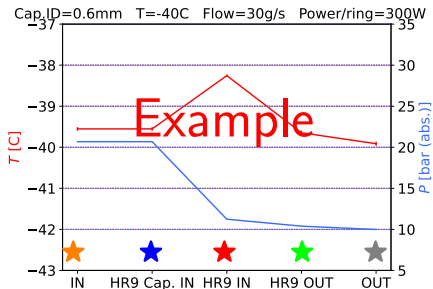
- Plots for  $T_{ev} = -40^\circ\text{C}$  and 30 g/s, using 300 W and 400 W.
- $\Delta P$  around 6.5 bar over capillary with ID 0.8 mm and length of 0.9 m.
  - ▶ Final ID 0.8 mm length should be slightly more than 90 cm.
- $\Delta P$  for HR1, HR5 and HR9 is the same, despite different capillary shapes (due to space).

## Pressure and temperature along the loops

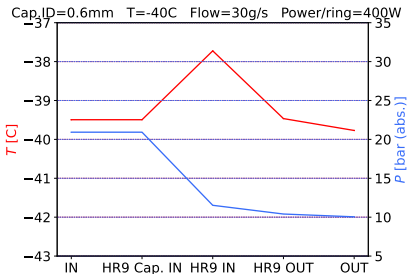
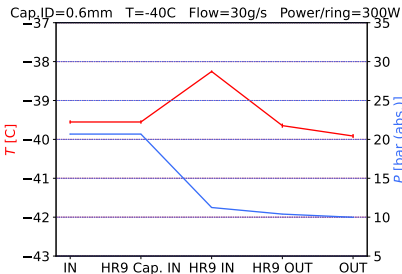
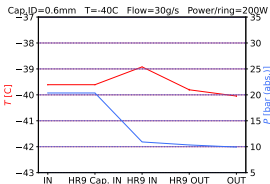
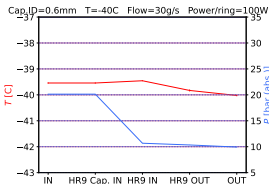
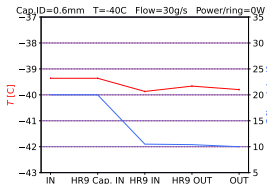


Shown in the following in-flow  $\text{CO}_2$   
 $P$  and  $T$  as a function of the position  
 along the mockup:

- $T$ , following the scale on the  $y$ -axis on the left,
- $P$ , following the scale on the  $y$ -axis on the right.



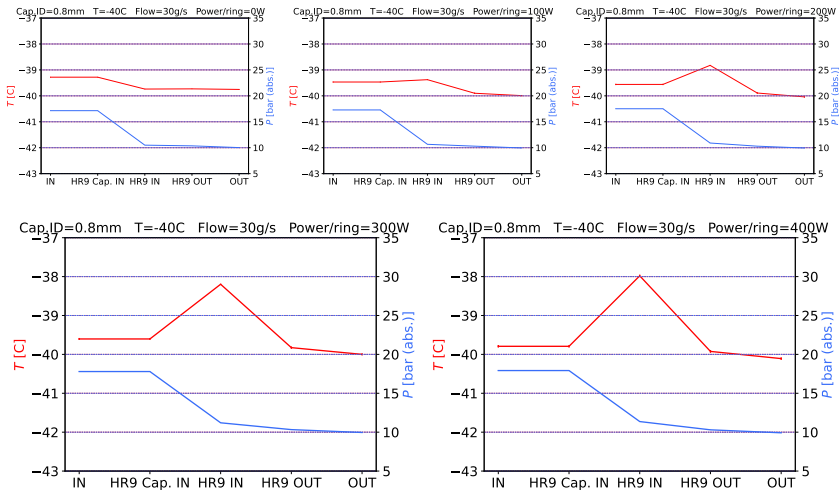
# Pressure and temperature - ID 0.6 mm



Note: data points are obtained as the average of the measurements over time. The uncertainty is present and it's computed as the standard deviation of the measurements.



# Pressure and temperature - ID 0.8 mm

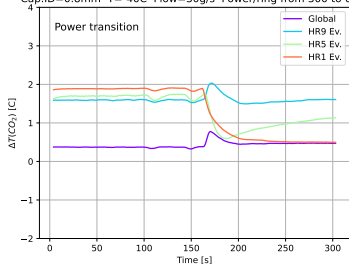


Note: data points are obtained as the average of the measurements over time. The uncertainty is present and it's computed as the standard deviation of the measurements.

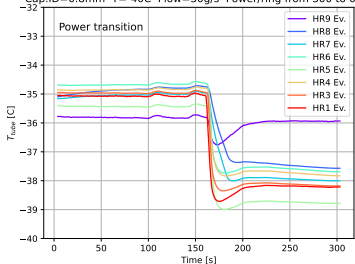
# Power transitions - Capillaries ID 0.8 mm

# Power transition from 300 W to 0 W - ID 0.8 mm

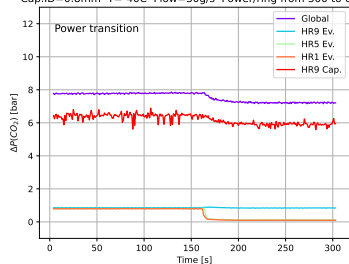
Cap.ID=0.8mm T=-40C Flow=30g/s Power/ring from 300 to 0W



Cap.ID=0.8mm T=-40C Flow=30g/s Power/ring from 300 to 0W



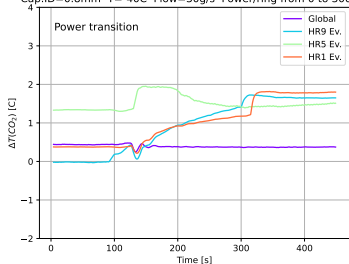
Cap.ID=0.8mm T=-40C Flow=30g/s Power/ring from 300 to 0W



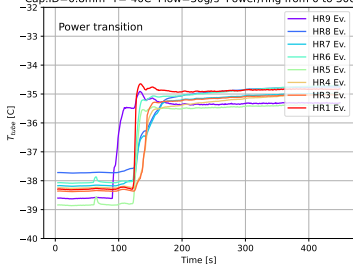
- $T_{PP1}$   $-40^{\circ}\text{C}$  and flow 30 g/s
- Transitioning from 300 W to 0 W for all evaporators
- $\Delta T$  variation within  $2^{\circ}\text{C}$
- $\Delta P$  variation within 1 bar

# Power transition, from 0 W to 300 W - ID 0.8 mm

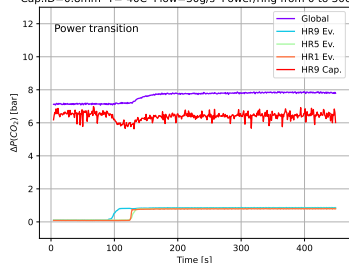
Cap.ID=0.8mm T=-40C Flow=30g/s Power/ring from 0 to 300W



Cap.ID=0.8mm T=-40C Flow=30g/s Power/ring from 0 to 300W



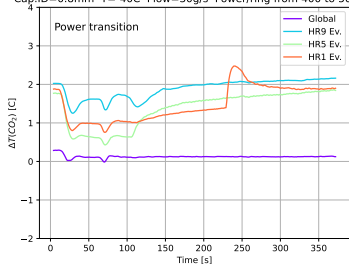
Cap.ID=0.8mm T=-40C Flow=30g/s Power/ring from 0 to 300W



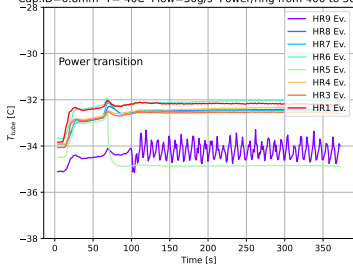
- $T_{PP1}$  -40 °C and flow 30 g/s
- Transitioning from 0 W to 300 W
- $\Delta T$  variation within 2 °C
- $\Delta P$  variation within 1 bar

# Power transition, from 400 W to 500 W - ID 0.8 mm

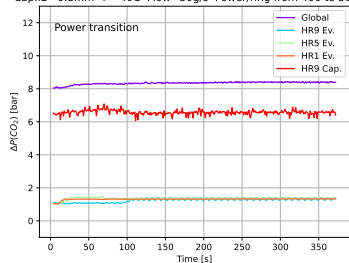
Cap.ID=0.8mm T=-40C Flow=30g/s Power/ring from 400 to 500W



Cap.ID=0.8mm T=-40C Flow=30g/s Power/ring from 400 to 500W



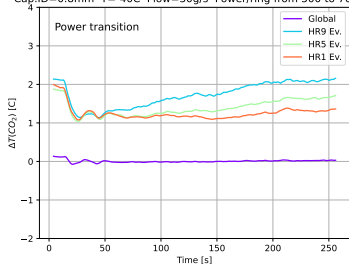
Cap.ID=0.8mm T=-40C Flow=30g/s Power/ring from 400 to 500W



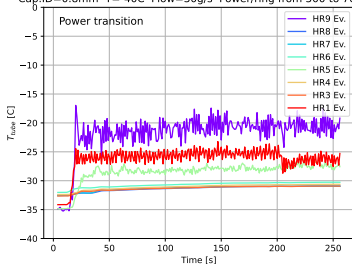
- $T_{PP1}$   $-40^{\circ}\text{C}$  and flow 30 g/s
- Transitioning from 400 W to 500 W
- $\Delta T$  variation within  $2^{\circ}\text{C}$
- $\Delta P$  variation within 1 bar

# Power transition, from 500 W to 700 W - ID 0.8 mm

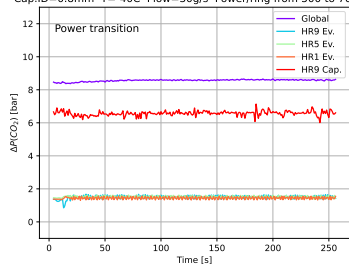
Cap.ID=0.8mm T=-40C Flow=30g/s Power/ring from 500 to 700W



Cap.ID=0.8mm T=-40C Flow=30g/s Power/ring from 500 to 700W



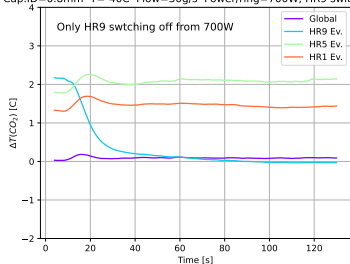
Cap.ID=0.8mm T=-40C Flow=30g/s Power/ring from 500 to 700W



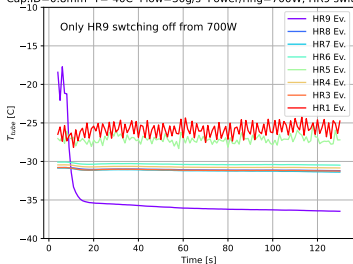
- $T_{PP1}$   $-40^{\circ}\text{C}$  and flow 30 g/s
- Transitioning from 500 W to 700 W
- $\Delta T$  variation within  $1.5^{\circ}\text{C}$
- $\Delta P$  variation within 1 bar

# Power transition for HR9 only - ID 0.8 mm

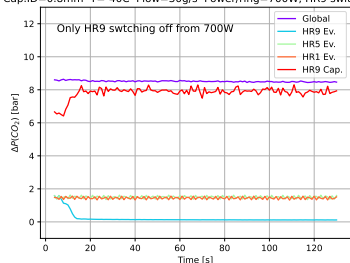
Cap.ID=0.8mm T=-40C Flow=30g/s Power/ring=700W, HR9 switch-off



Cap.ID=0.8mm T=-40C Flow=30g/s Power/ring=700W, HR9 switch-off



Cap.ID=0.8mm T=-40C Flow=30g/s Power/ring=700W, HR9 switch-off



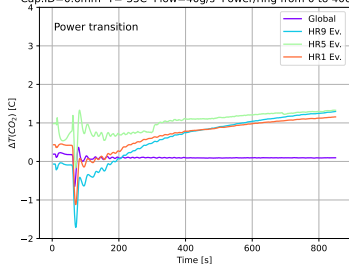
- $T_{PP1}$   $-40^{\circ}\text{C}$  and flow 30 g/s
- Transitioning from 700 W to 0 W for HR9 only, other evaporators at 700 W
- $\Delta T$  variation within  $2.5^{\circ}\text{C}$
- $\Delta P$  variation within 2 bar

# Power transitions - Capillaries ID 0.6 mm

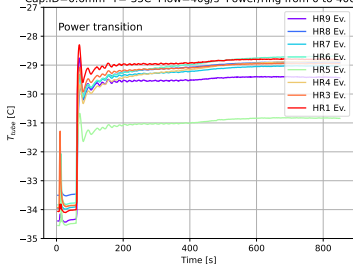


# Power transition from 0 W to 400 W - ID 0.6 mm

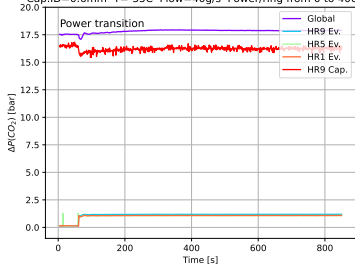
Cap.ID=0.6mm T=-35C Flow=40g/s Power/ring from 0 to 400W



Cap.ID=0.6mm T=-35C Flow=40g/s Power/ring from 0 to 400W

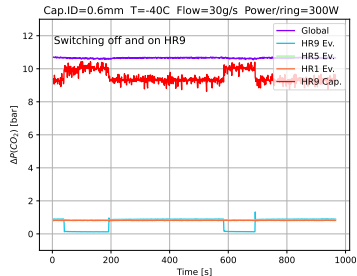
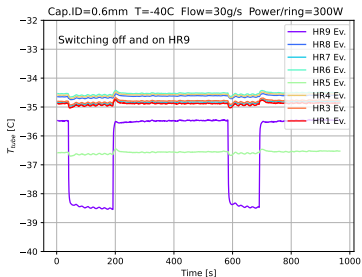
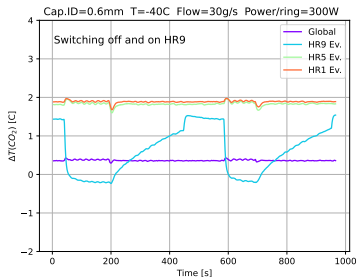


Cap.ID=0.6mm T=-35C Flow=40g/s Power/ring from 0 to 400W



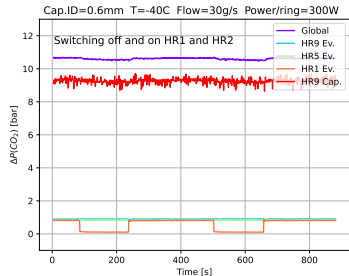
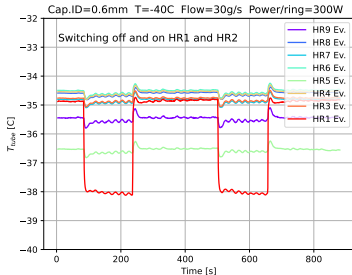
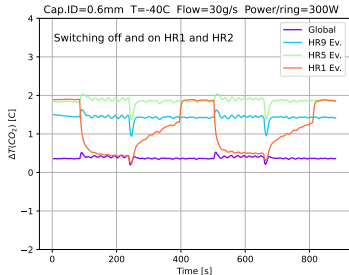
- $T_{PP1}$  -35 °C and flow 40 g/s
- Transitioning from 0 W to 400 W
- $\Delta T$  variation within 2.5 °C
- $\Delta P$  variation within 1 bar

# Power switch on HR9 only - ID 0.6 mm



- $T_{PP1}$  -40 °C and flow 30 g/s
- Switching off and on HR9 only, while other evaporators are at 300 W
- $\Delta T$  variation within 2 °C
- $\Delta P$  variation within 1 bar

# Power switch on HR1 and HR2 - ID 0.6 mm



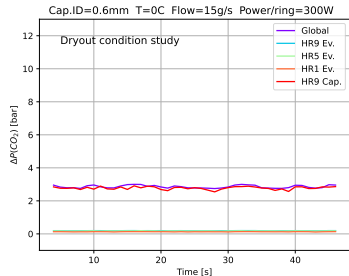
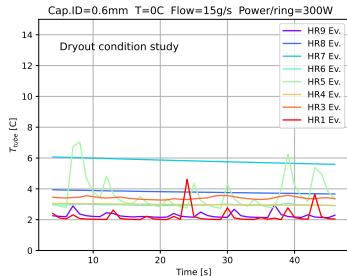
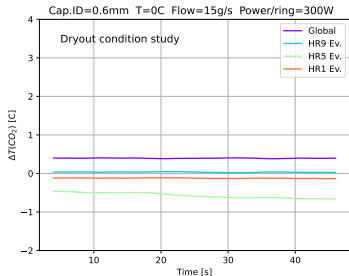
- $T_{PP1}$   $-40^\circ\text{C}$  and flow 30 g/s
- Switching off and on HR1 and HR2 only, while other evaporators are at 300 W
- $T_{\text{tube}}$  for HR2 not available
- $\Delta T$  variation within  $2^\circ\text{C}$
- $\Delta P$  variation within 1 bar

# Dry-out studies - Capillaries ID 0.6 mm

## Dry-out studies

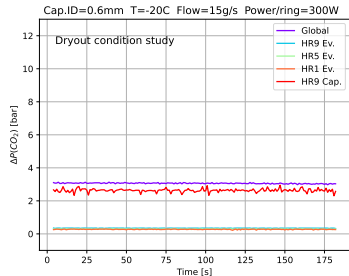
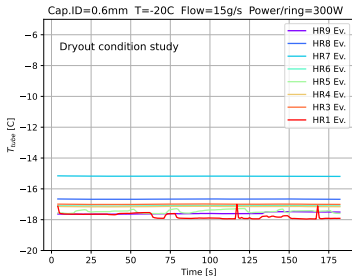
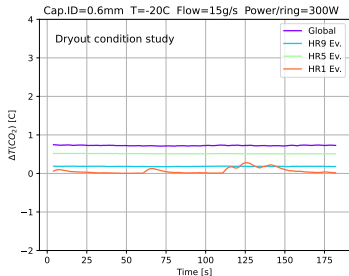
- Measurements decreasing the flow to detect dry-out conditions
- Using nominal power, 300W/HR
- PP1 temperature of 0 °C, -20 °C and -40 °C.
- Flow transition from 30 g/s to 15 g/s (10 g/s in one case), which was found to be the value that triggers the dry-out in this condition
- Plots for  $\Delta P$ ,  $\Delta T$  and  $T_{tube}$  at the 15 g/s (or 10 g/s) condition are reported, for each temperature

# Dry-out condition at 0 °C - ID 0.6 mm



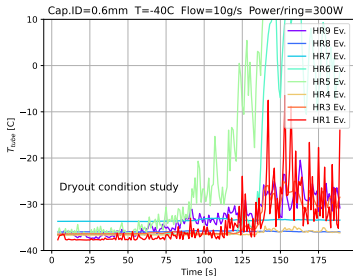
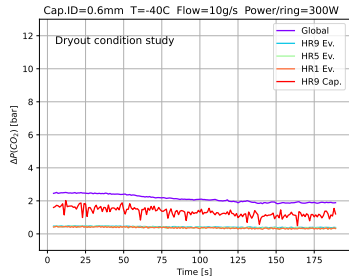
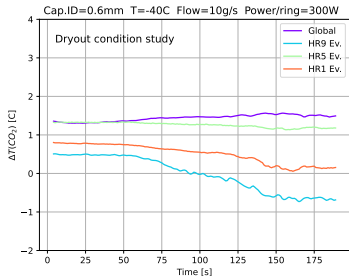
- $T_{PP1}$  0 °C and 300 W power applied to each evaporator
- Dry-out reached around 15 g/s

# Dry-out condition at $-20^{\circ}\text{C}$ - ID 0.6 mm



- $T_{PP1}$   $-20^{\circ}\text{C}$  and 300 W power applied to each evaporator
- Dry-out reached around 15 g/s

# Dry-out condition at $-40^{\circ}\text{C}$ - ID 0.6 mm



- $T_{PP1}$   $-40^{\circ}\text{C}$  and 300 W power applied to each evaporator
- Dry-out starting around 19 g/s for few HR (HR5 and HR9), decreasing flow until 10 g/s