



# Muon Collider Magnet Moving For Neutrino Mitigation

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# OUTLINE

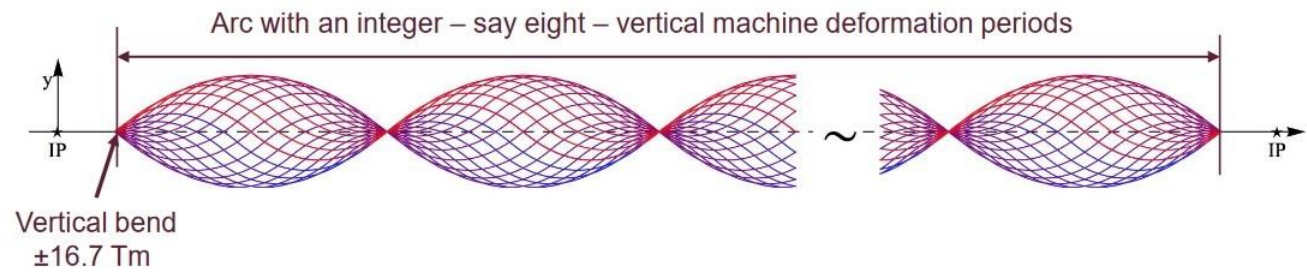
- **Collider ring neutrino radiation mitigation**
- **Assumptions**
- **Magnet alignment and lay-out change jack system – Solution 1**
- **Combined jack – Solution 2**
- **Tunnel space**
- **He supply**
- **Changing magnet lay-out**
- **Conclusions**

# COLLIDER RING NEUTRINO RADIATION MITIGATION

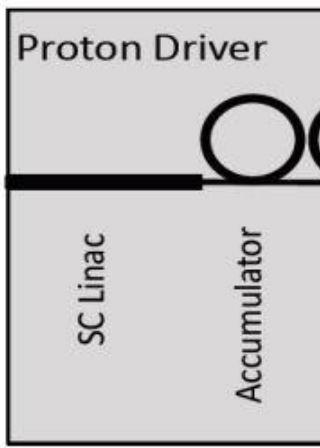
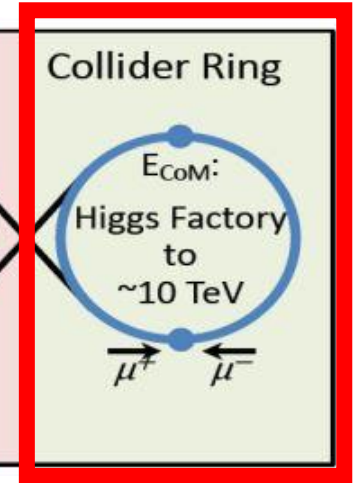


## Mitigation by "Wobbling"

- Wobbling of machine in vertical direction – part of MAP proposal?
  - ◆ Time-dependent mechanical deformation of ring around arc (including chromatic compensation, matching section and FMC arc cells)
  - ◆ High precision movement system
  - ◆ Impact on optics?
- For 10 TeV com collider with 10 km circumference and say 4.8 km arcs



- ◆ Combination of pieces of parabola – two pieces with opposite curvature one period
  - ◆ Say 8 periods 660 m long periods generating angles between -1 mrad and + 1 mrad
  - ◆ Magnetic field (average) bending in vertical  $\pm 0.11$  T
  - ◆ Excursion (maximum total)  $\pm 150$  mm
  - ◆ Replaces vertical Gaussian angle distribution with rms opening of  $\approx 0.0086$  mrad by about rectangular distribution within  $\pm 1$  mrad
- => About two order of magnitude reduction of peak dose rates



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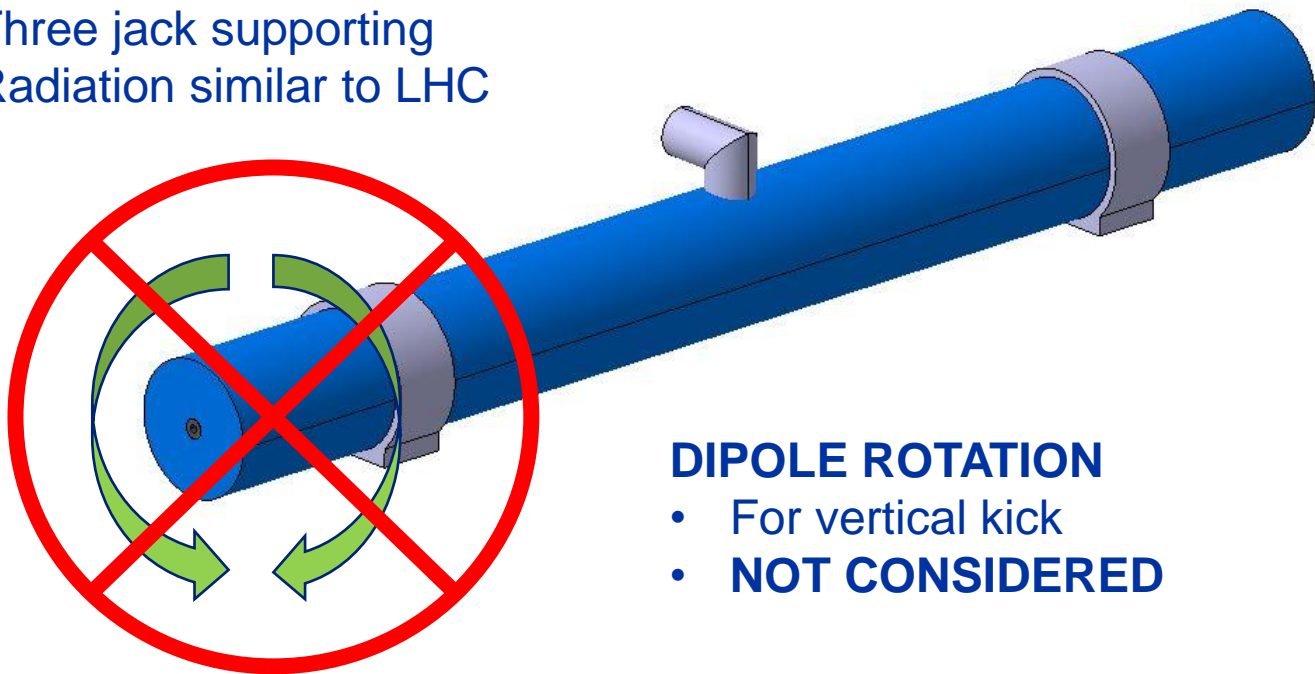
Courtesy of Christian Carli

Neutrino flux requires change of beam trajectory every 12 hours

# ASSUMPTIONS

## DIPOLE MAGNET MODEL

- Inspired by LHC magnets
- $L = 10\text{ m}$ ,  $D = 1\text{ m}$
- Mass: 24.5 T approximately (based on LHC D2 magnet and W-shield model)
- Length of interconnection = 500 - 800 mm (coldmass – coldmass)
- Cold bore  $\varnothing 50\text{ mm}$
- Three jack supporting
- Radiation similar to LHC

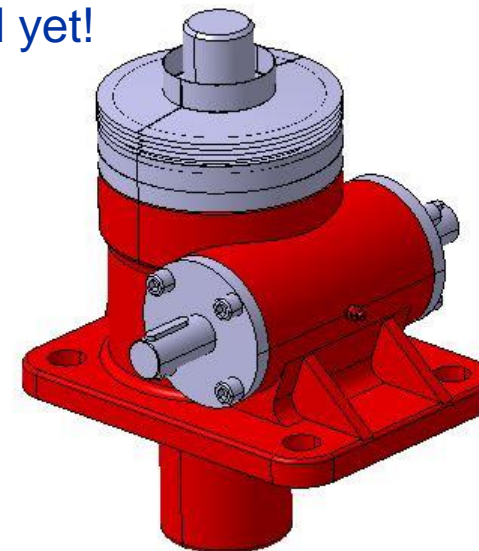


### DIPOLE ROTATION

- For vertical kick
- **NOT CONSIDERED**

## LATERAL FORCES:

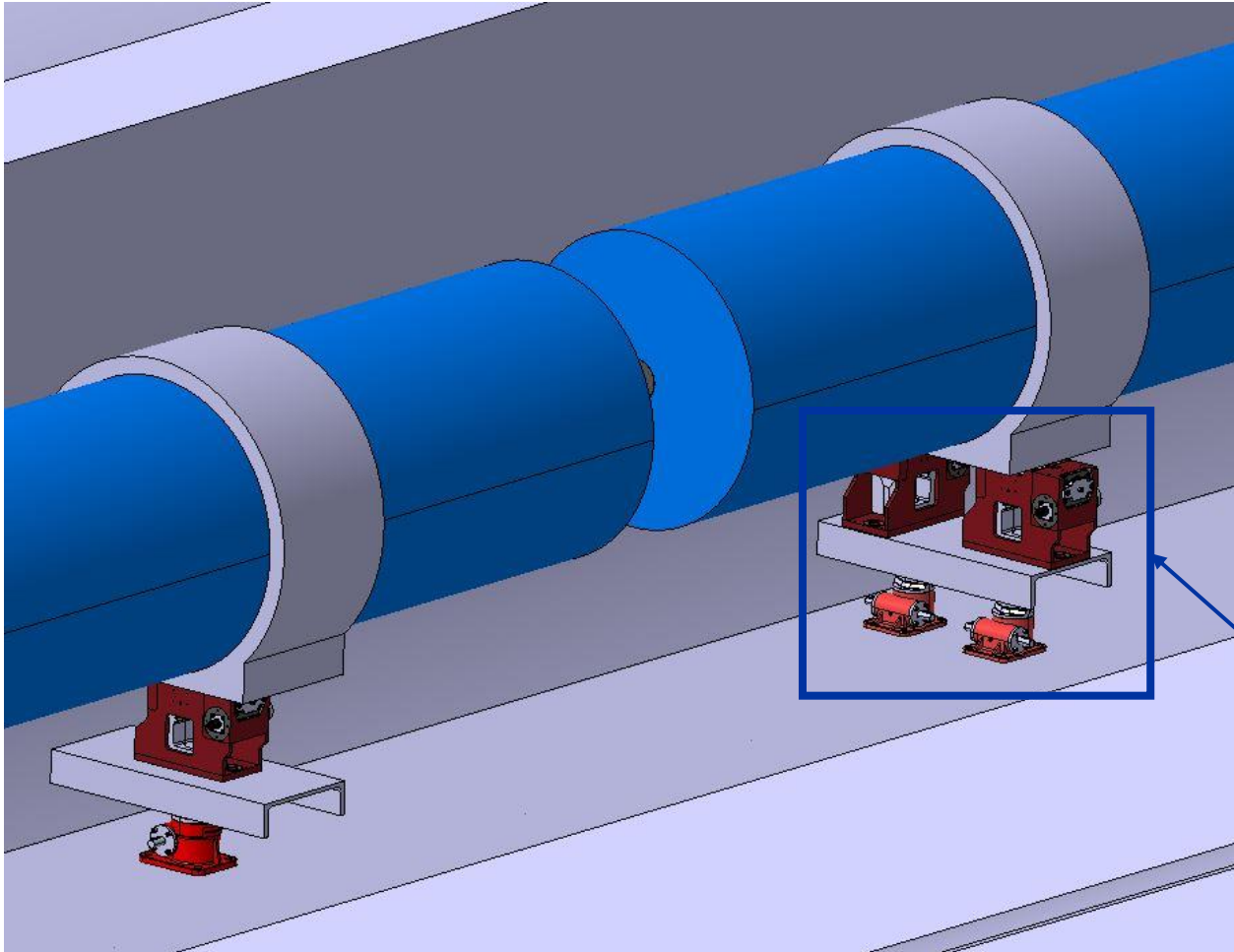
- On dipole only from gravity (LHC tunnel angle of 1.4%)
- On quadrupole also from vacuum barrier and He lines as well – not studied yet!



### COMMERCIAL JACK

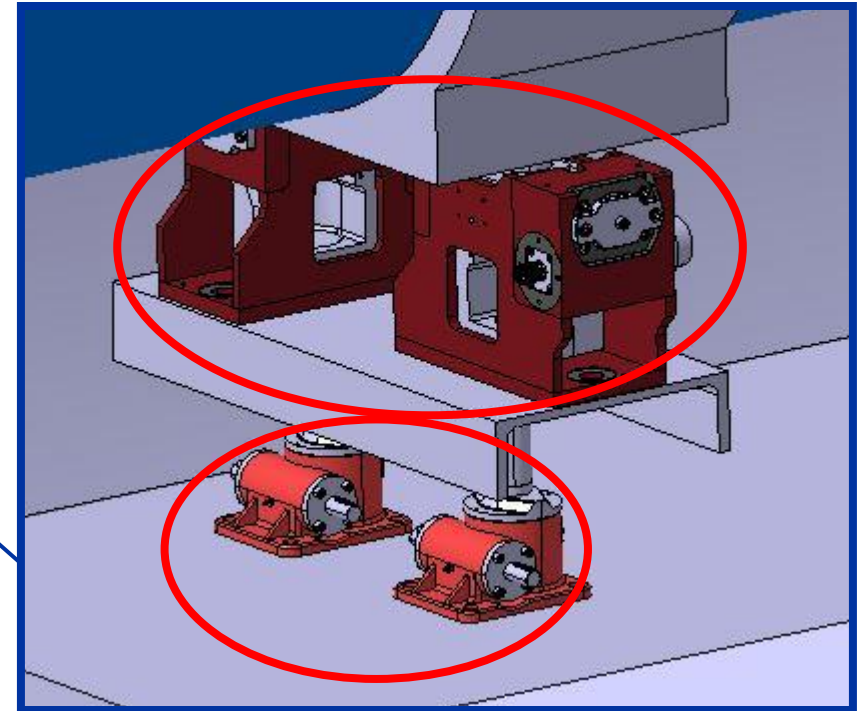
- Precision:  $\pm 0.5\text{ mm}$
- Mechanically driven
- Self-locking mechanism
- Lateral forces appr. 3-5%

# TWO JACKING SYSTEMS – SOLUTION 1



## ALIGNMENT JACKS

- Magnet alignment
- HL-LHC Jack



## MOTORIZED JACKS

- Only vertical movement
- Commercial jack



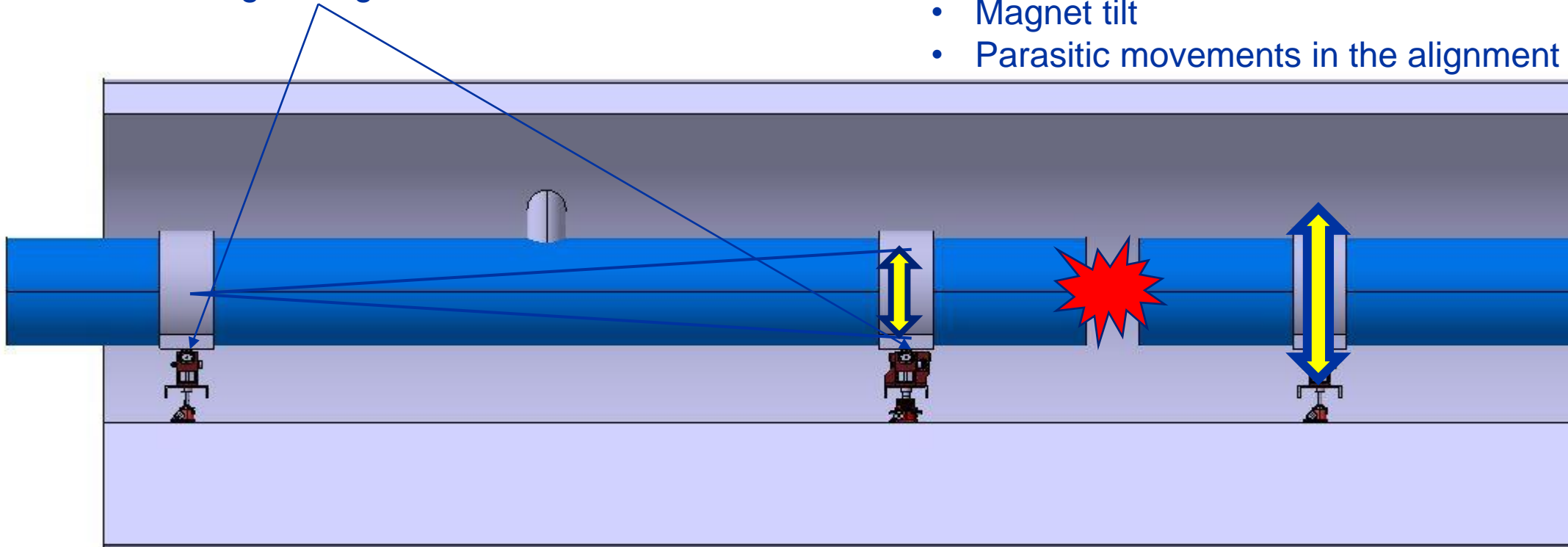
# ALIGNMENT AND LAY-OUT CHANGE

## ALIGNMENT JACKS

- Magnet alignment

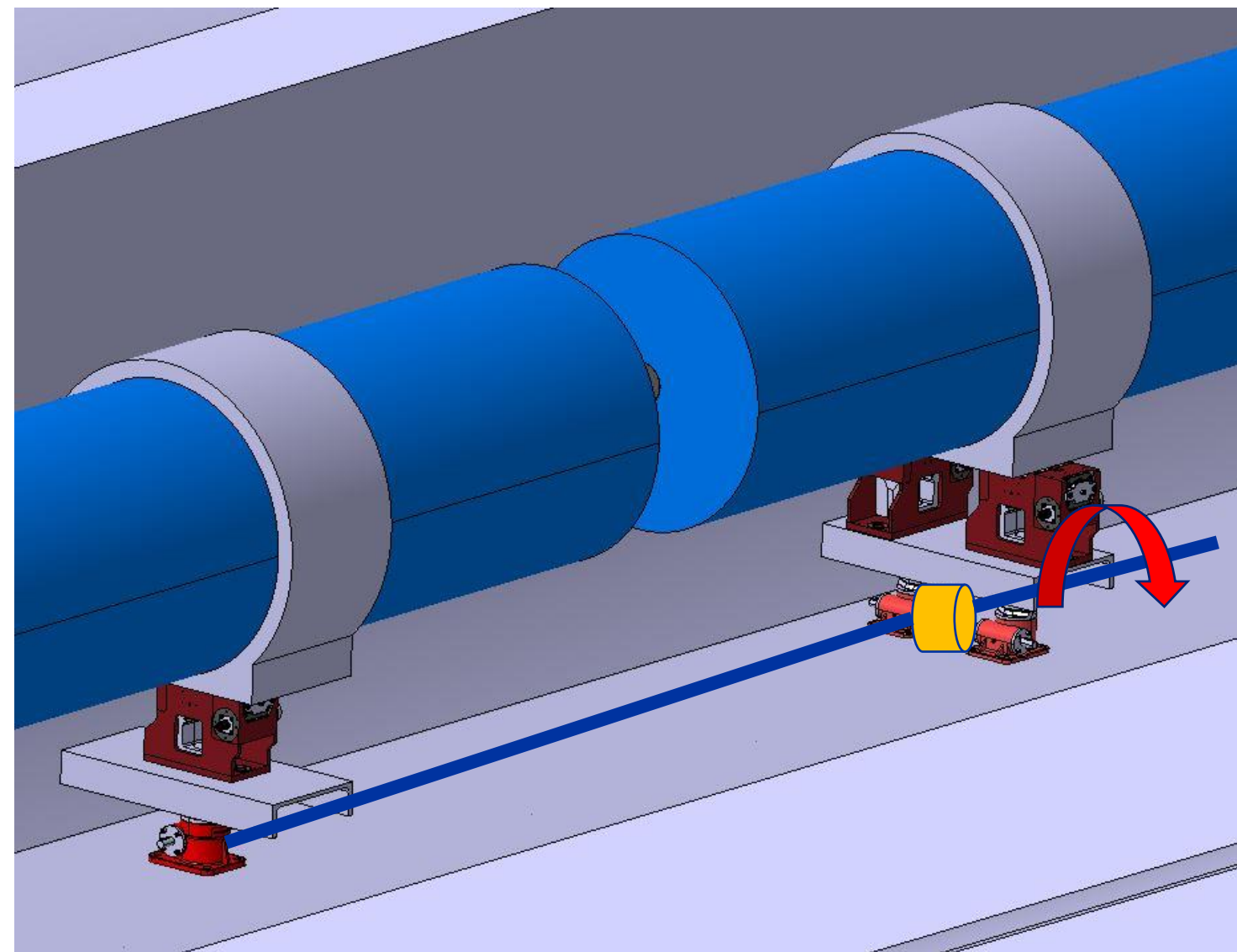
## MOTORIZED JACKS

- Vertical movement  $\pm 50$  mm
- Magnet tilt
- Parasitic movements in the alignment jack ( $< 3\mu\text{m}$ )



Maximum stroke in opposite direction over the interconnection will destroy the bellows!  
SAFE OPERATION SHALL NOT RELY ON THE CONTROL SYSTEM!

# TWO JACKING SYSTEMS – SOLUTION 1a



## PARALLEL OPERATION OF JACKS ACROSS THE INTERCONNECTION

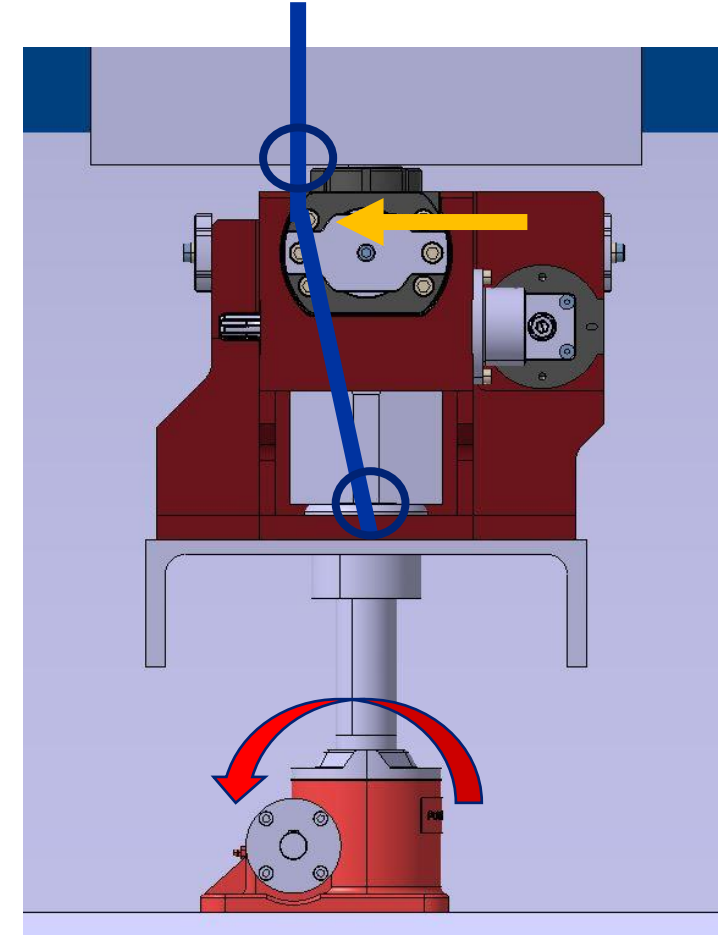
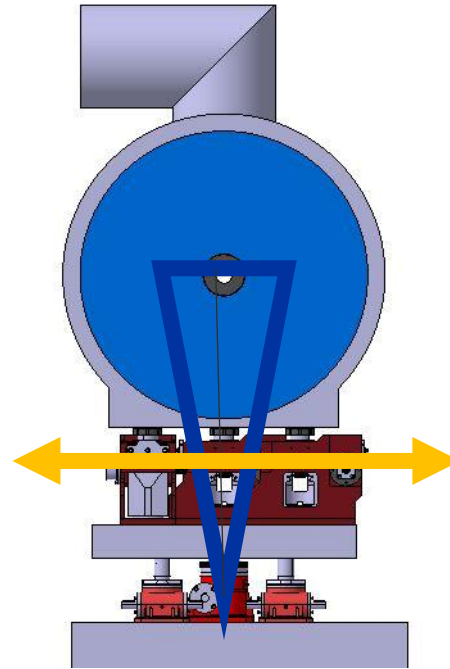
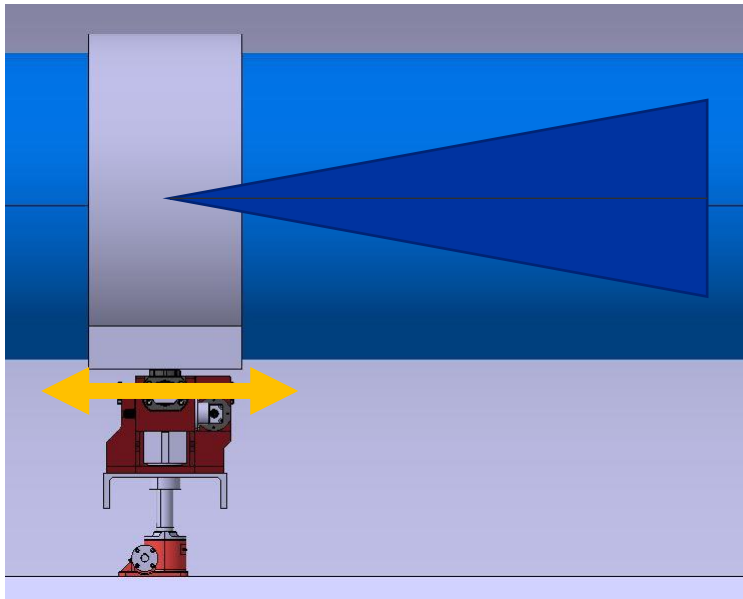
- Connect jacks mechanically
- 4-way gearbox
- Shafts
- Single motor on the gearbox
- IN STUDY – redundant system with two motors, requires two gearboxes

- **BELLOWS OFFSET BLOCKED**
- **ANGLE BETWEEN MAGNETS ALLOWED**

# SUPPORTING OF MOTORIZED JACKS

## BENDING MOMENT ON THE MOTORIZED JACK

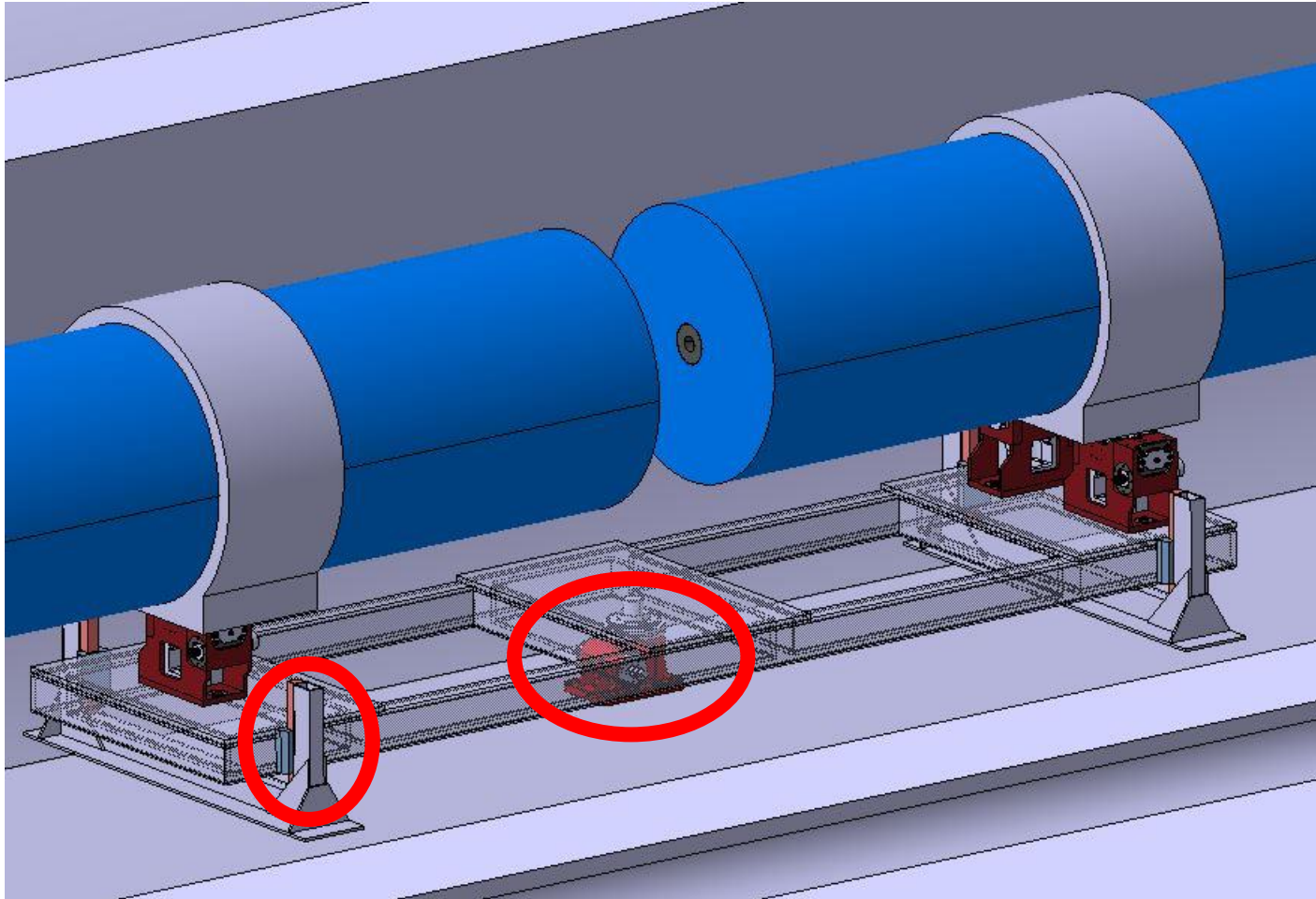
- From magnet tilt or roll due to the tunnel
- From maximum lateral stroke of the alignment jack
- To be calculated for different motorized jack options
- Compare with allowed moments for the motorized jacks
- Higher the stroke – higher the moment



**MOTORIZED JACKS MAY NEED SUPPORTING GUIDES TO PROTECT THEM FROM BENDING MOMENTS!**



# TWO JACKING SYSTEMS – SOLUTION 1b



## ALIGNMENT JACKS ACROSS THE INTERCONNECTION ON COMMON PLATFORM

- One central jack for the height change
- Platform must have linear guides to avoid bending moment on the jack
- One motor per interconnection = magnet
- Motor for redundancy required?

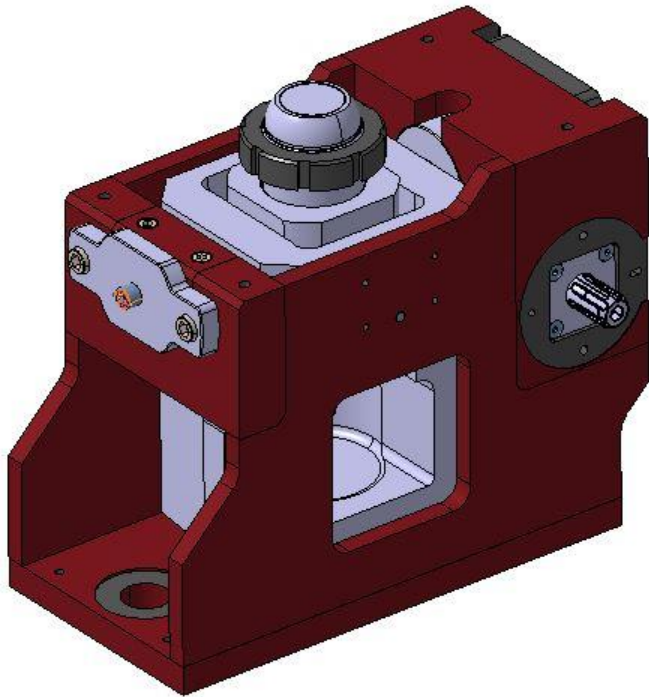
- **BELLOWS OFFSET BLOCKED**
- **ANGLE BETWEEN MAGNETS ALLOWED**

# COMBINED JACK - SOLUTION 2

*Combine the functions of the two jacks – alignment and lay-out change!*

## HL-LHC JACK

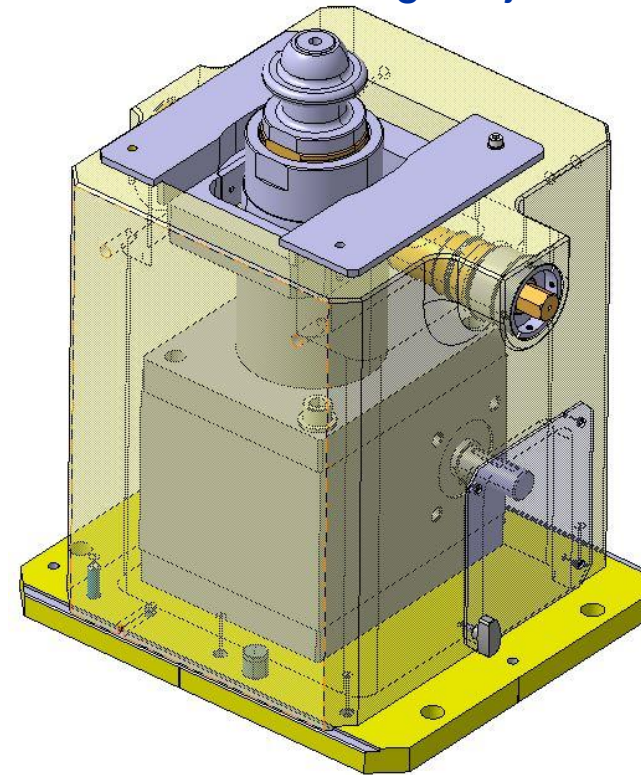
- Max load 17 T
- Vertical stroke  $\pm 20$  mm



Both require three jacks together to create an isostatic support system!

## L4 JACK – inspired by LHC jack

- Max load 5 T
- Vertical stroke  $\pm 15$  mm
- Commercial wormgear jack

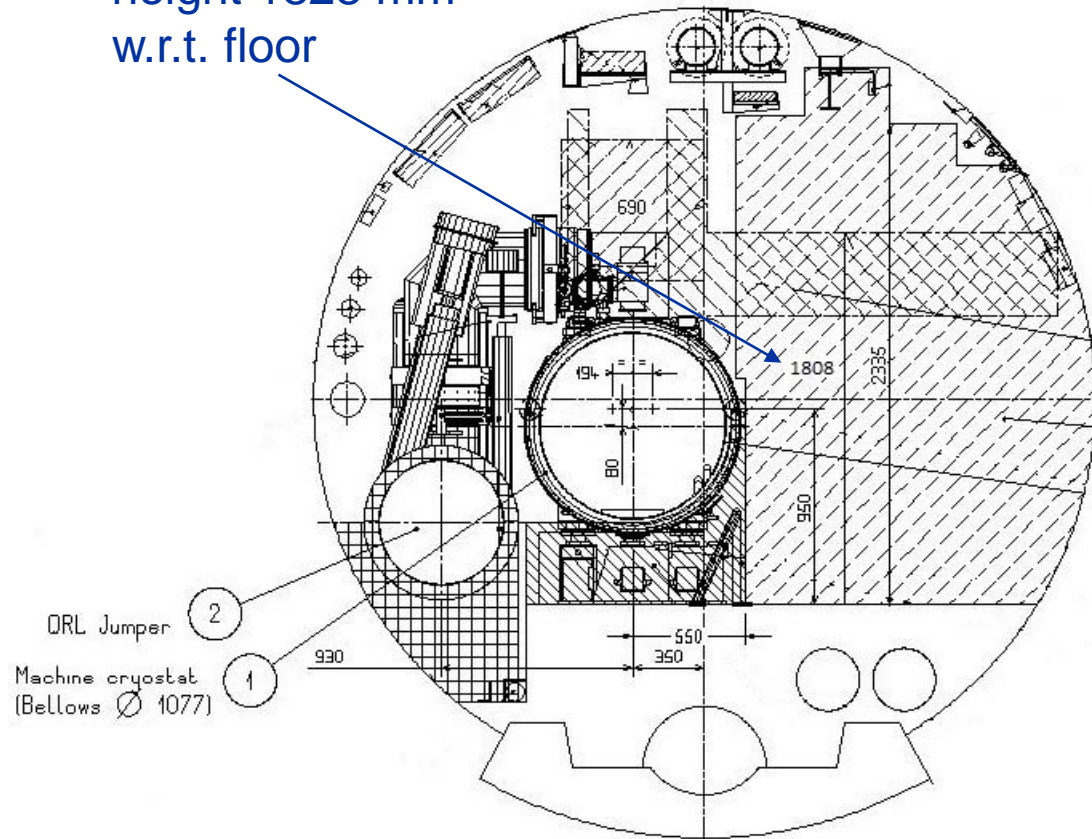


**Muon Collider specification requires a new design – concept work initiated with a jack supplier**

# TUNNEL SPACE

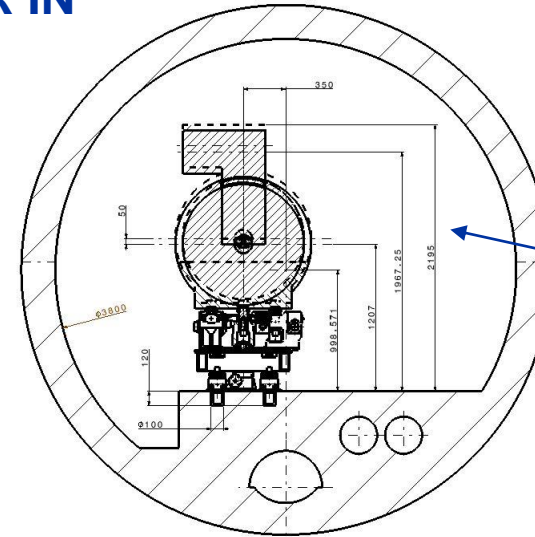
## LHC

- Jumper max height 1828 mm w.r.t. floor



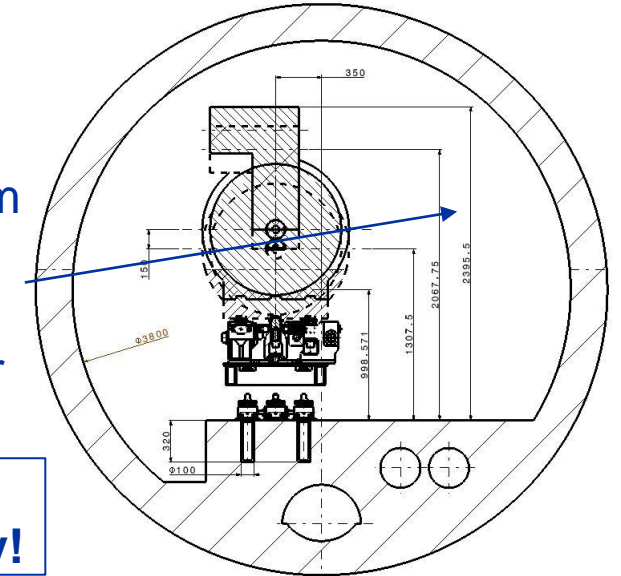
## MUON COLLIDER IN LHC TUNNEL

- 28 Space reserved for survey
- 26 Space reserved for transport



- Jacks with  $\pm 50$  mm movement
- Jumper max height 2215 mm w.r.t. floor

- Jacks with  $\pm 150$  mm movement
- Jumper max height 2415 mm w.r.t. floor



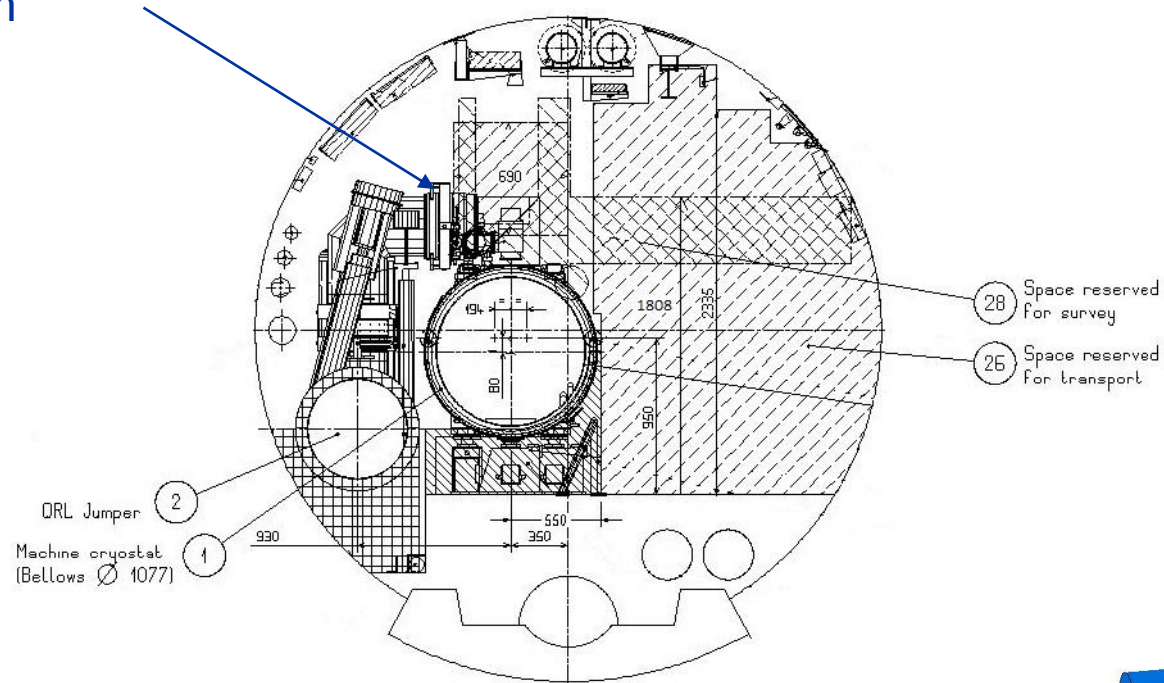
**NOTE: This assumes part of the jack frame sits in a cavity!**



# He SUPPLY

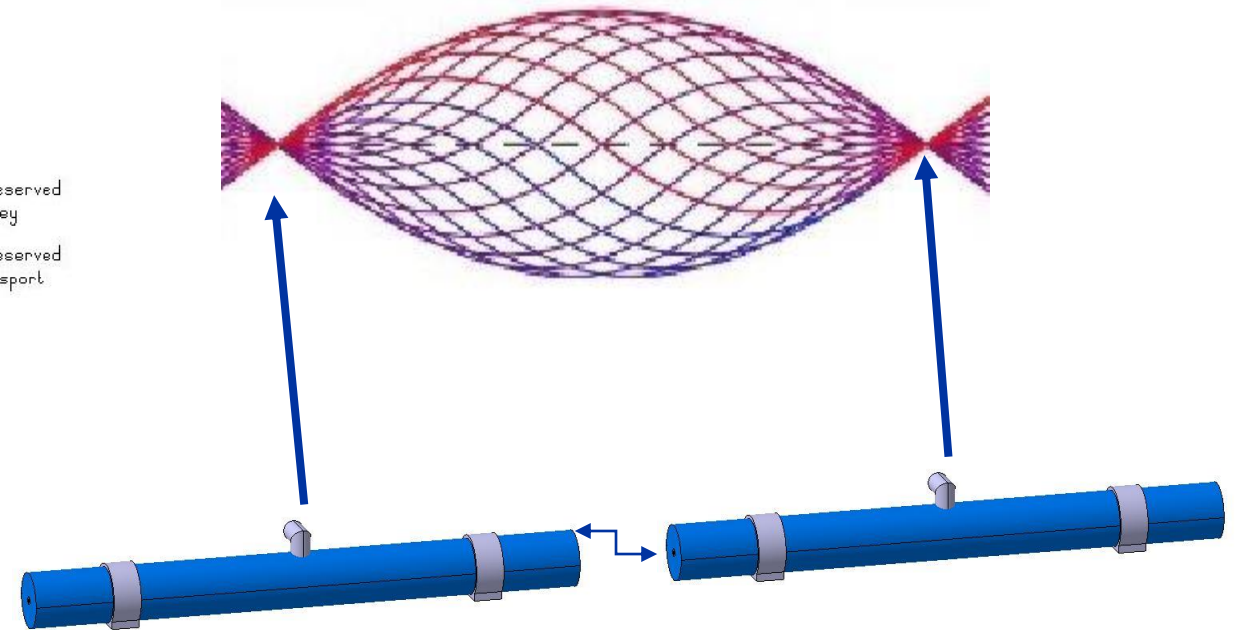
## LHC

- Jumper connection to He supply every 110 m



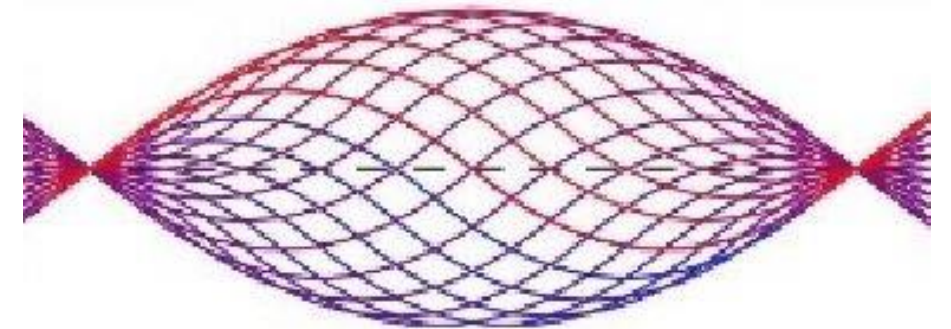
## He SUPPLY CONNECTIONS FOR MUON COLLIDER

- Must be placed around the period node points – current jumper designs do not allow the vertical movement of the collider dipoles w.r.t rigid cryogenic supply installation



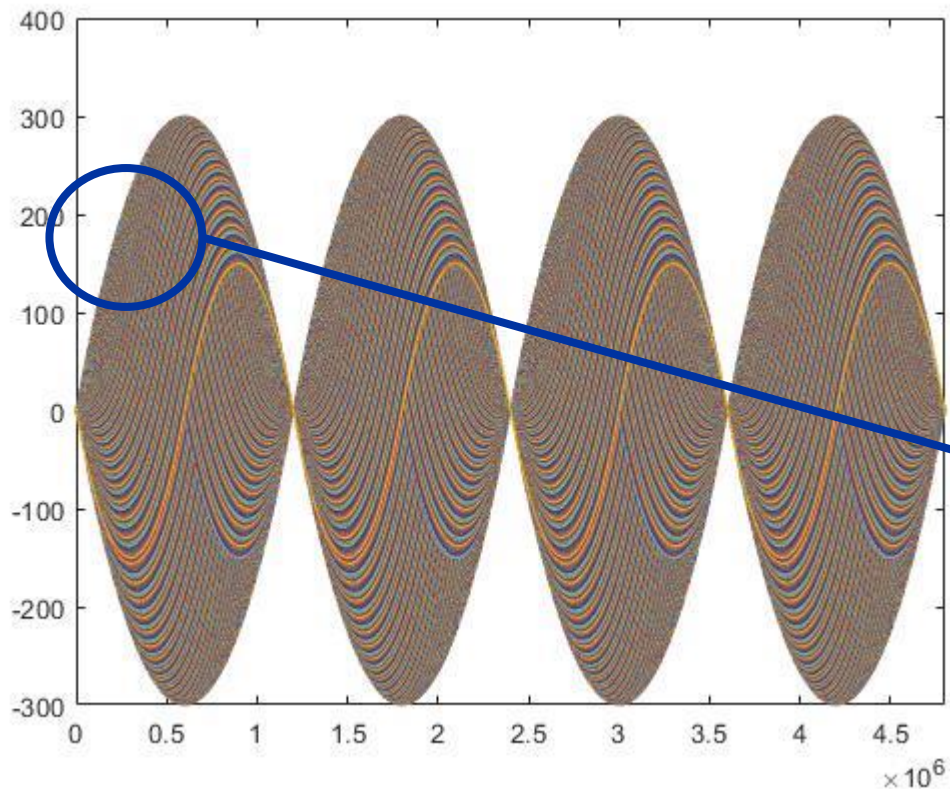
# POWER

- **200 days of physics annually(55%) leads to 400 trajectory changes**
  - LHC 2023 – 2038 planning is 43%
  - 20 years 8000 changes
  - How many physical lay-out changes w.r.t. Beam trajectory change? – more on the next slide
- **Powering lay-out changes**
  - $Power_{total} = N_{magnets} \times N_{jacks} \times Power_{motor}$  ,if 10 m dipoles and 0.8 m interconnection > 800 magnets
  - With 1 kW motor around 1 MW for parallel operation of jacks
  - Control system for movement and monitoring
    - Reliable push-button lay-out change!
  - Do we need to confirm measurements with external system?
- **Assume change propagated in smaller groups**
  - Max number of motors to be defined
  - Assume: group of 50 motors -> around 20 groups to move
  - 3 minutes per group leads to one hour

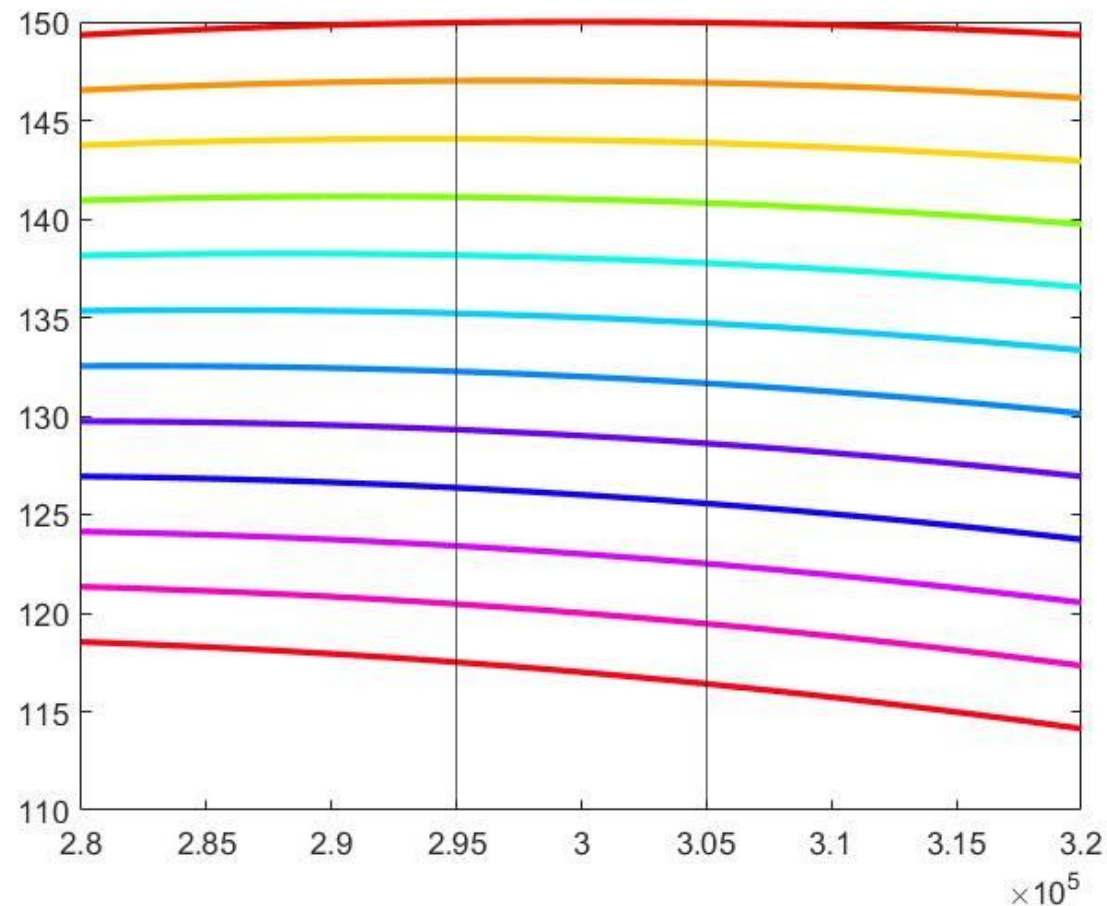




# BEAM TRAJECTORIES vs. MAGNET MOVEMENTS



To be studied: how many beam trajectories could be fit into one magnet lay-out

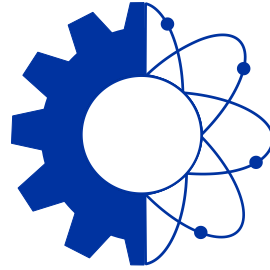
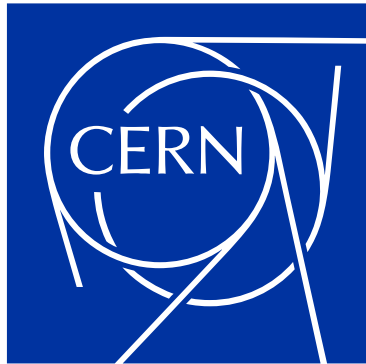


# CONCLUSIONS

- **Two solutions to move the magnets presented**
- **Information required**
  - Aperture confirmation
  - Period confirmation
- **Concept selection and design engineering**

THANK YOU FOR YOUR ATTENTION  
YOUR QUESTIONS PLEASE

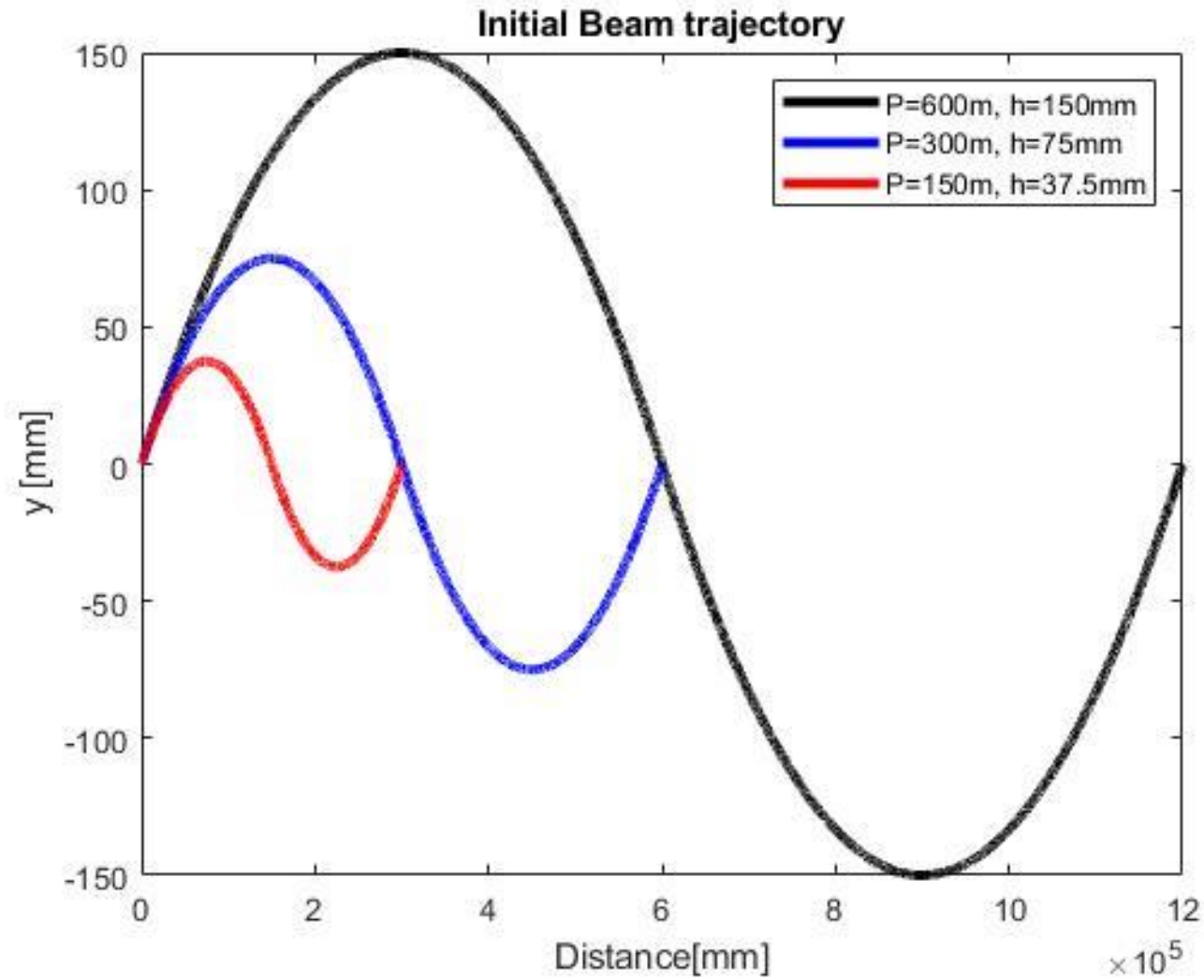




**ENGINEERING  
DEPARTMENT**

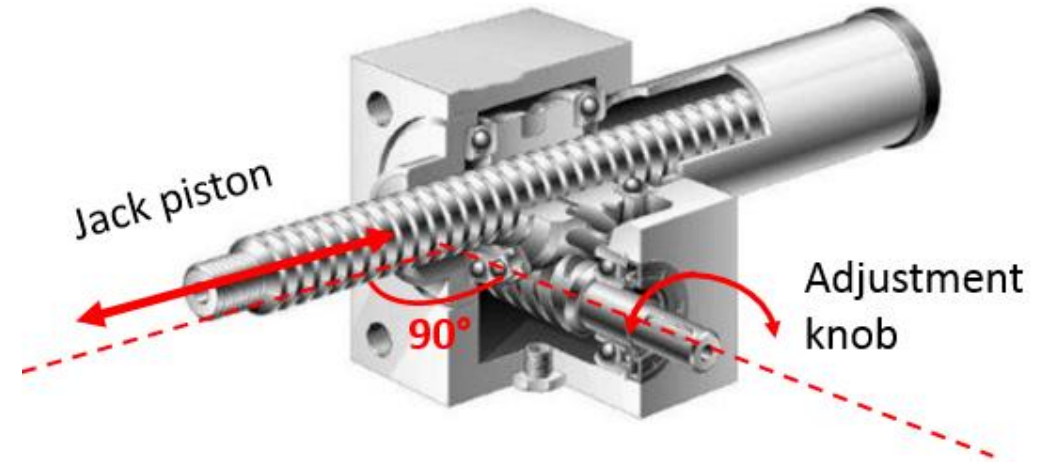
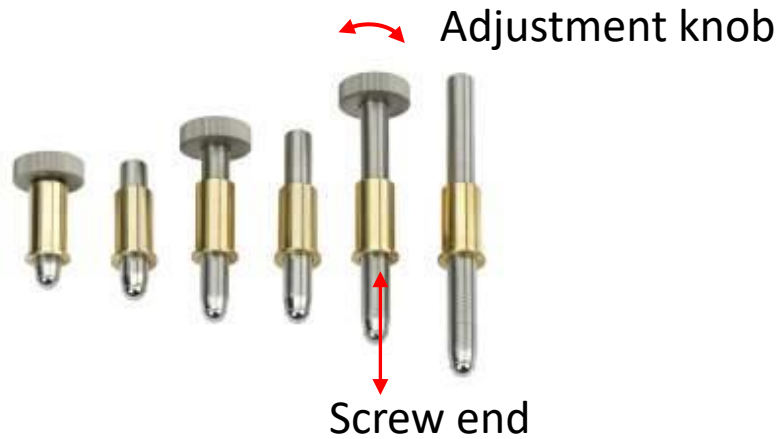
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# PERIODS





# Real impact of play in adjustment jigs (screw mechanisms, jacks) on adjustment performance



## • Simple screw adjustment

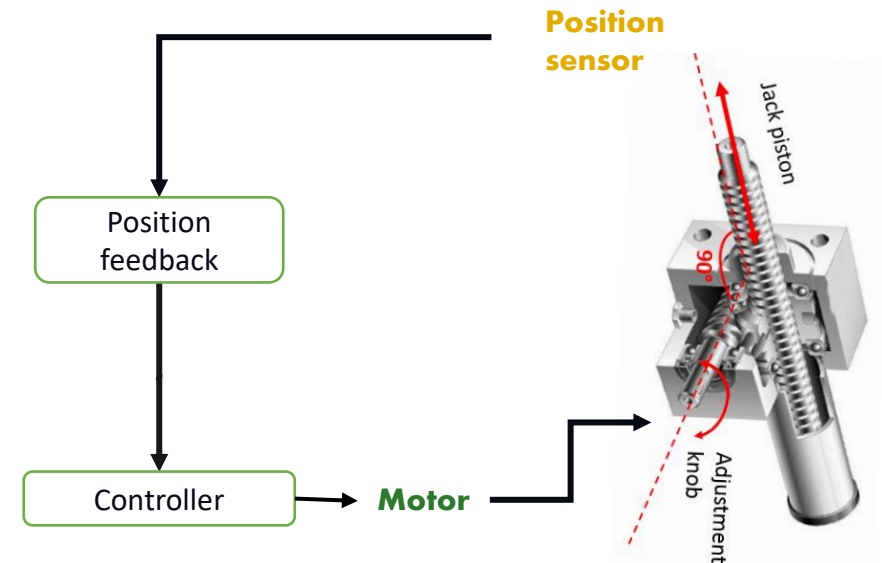
- Play (seen on *screw end*) depends on quality of the thread machining
  - For typical machined threads (M x, Tr x, etc ...) – play at *screw end* can vary from tenths um-s to even mm for bigger screws
  - For ball screws play depends on fitting quality – it is much lower than for ‘standard’ threads (typically 0 .. 20 um)
- Angular backlash ‘seen’ on adjustment knob is proportional to screw pitch

## • More complex mechanisms (e.g. power jack)

- Play (seen on jack piston end) depends on quality of the *Jack piston* thread machining, and is relevant as in simple screw mechanisms
- Backlash seen on *Adjustment knob* is much bigger and depends on *Jack piston* thread play and worm mechanism play

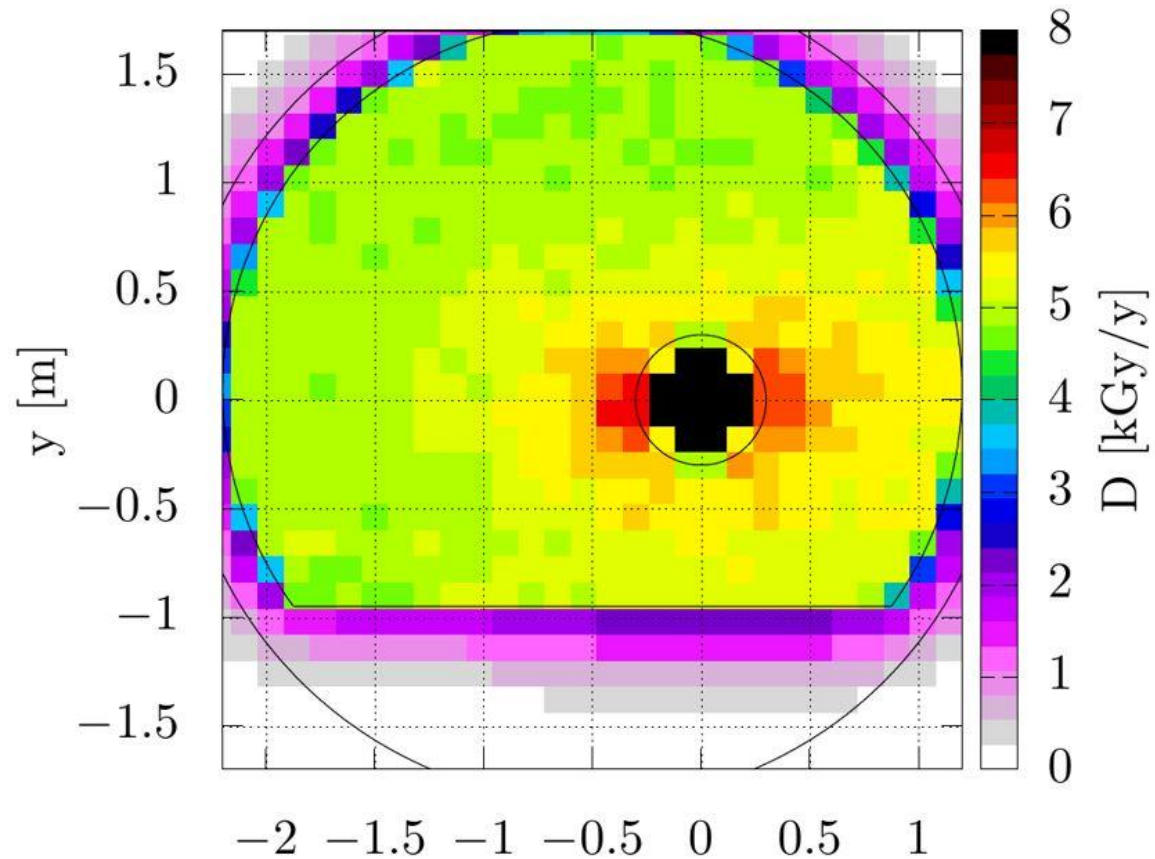
# Real impact of play in adjustment jigs (screw mechanisms, jacks) on adjustment performance

- For mechanisms using adjustment screws, the preloading of screws/thread pair plays big role to get best adjustment performance
  - For pre-loaded 'classical' screws, the resolution (minimum motion) and precision (repeatability of position) of adjustment is typically in range of  $\sim 5..50\mu\text{m}$  (this is e.g. classical configuration for vertical screws, supporting the load of adjusted components; for radial adjustment pre-loading springs are useful, to suppress screw play)
    - Pre loading of 'classical' screws can give resolution/precision parameters which are satisfactory and even comparable with ball screws in some cases
  - For non-pre-loaded screws, the adjustment resolution can be still  $\sim 5..50\mu\text{m}$  (in single motion direction), but precision of adjustment will be defined by play on screw/thread
- Use of closed-feedback-loop position control
  - Adding of position sensor, motorization and closed control loop to adjustment mechanisms allows to minimize the play effects in mechanisms – the controller follows the position measured by sensor
  - The control system components and mechanics shall be designed/chosen in a way to fulfil also the other system requirements (stiffness, 3D play in supporting system, safety, etc..)



# RADIATION

Yearly dose projection, 200 days of operation



*Courtesy of Daniele Calzolari*