

Vaporization of cryogenic liquid stored in damaged vacuum-insulated tank

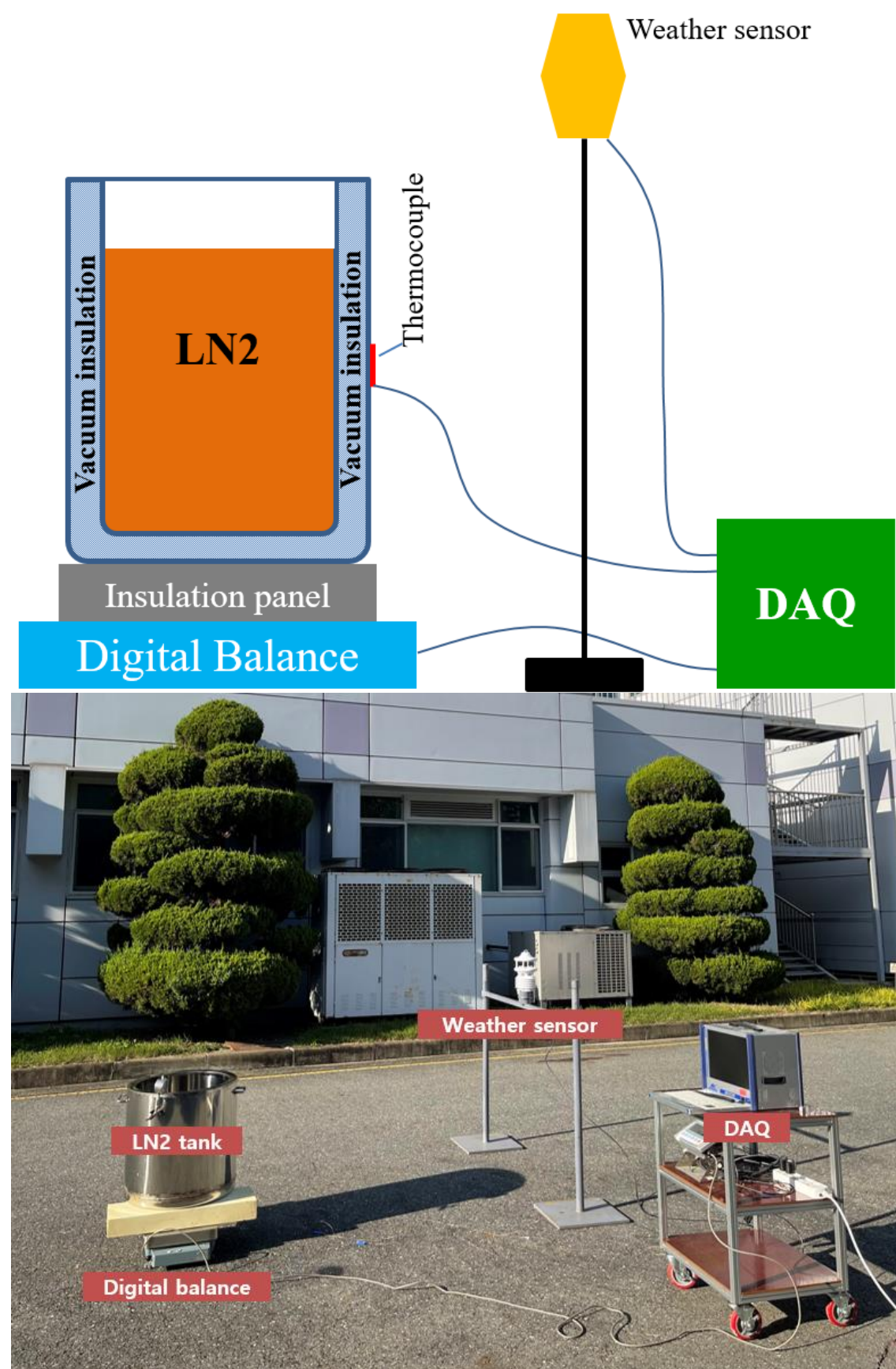
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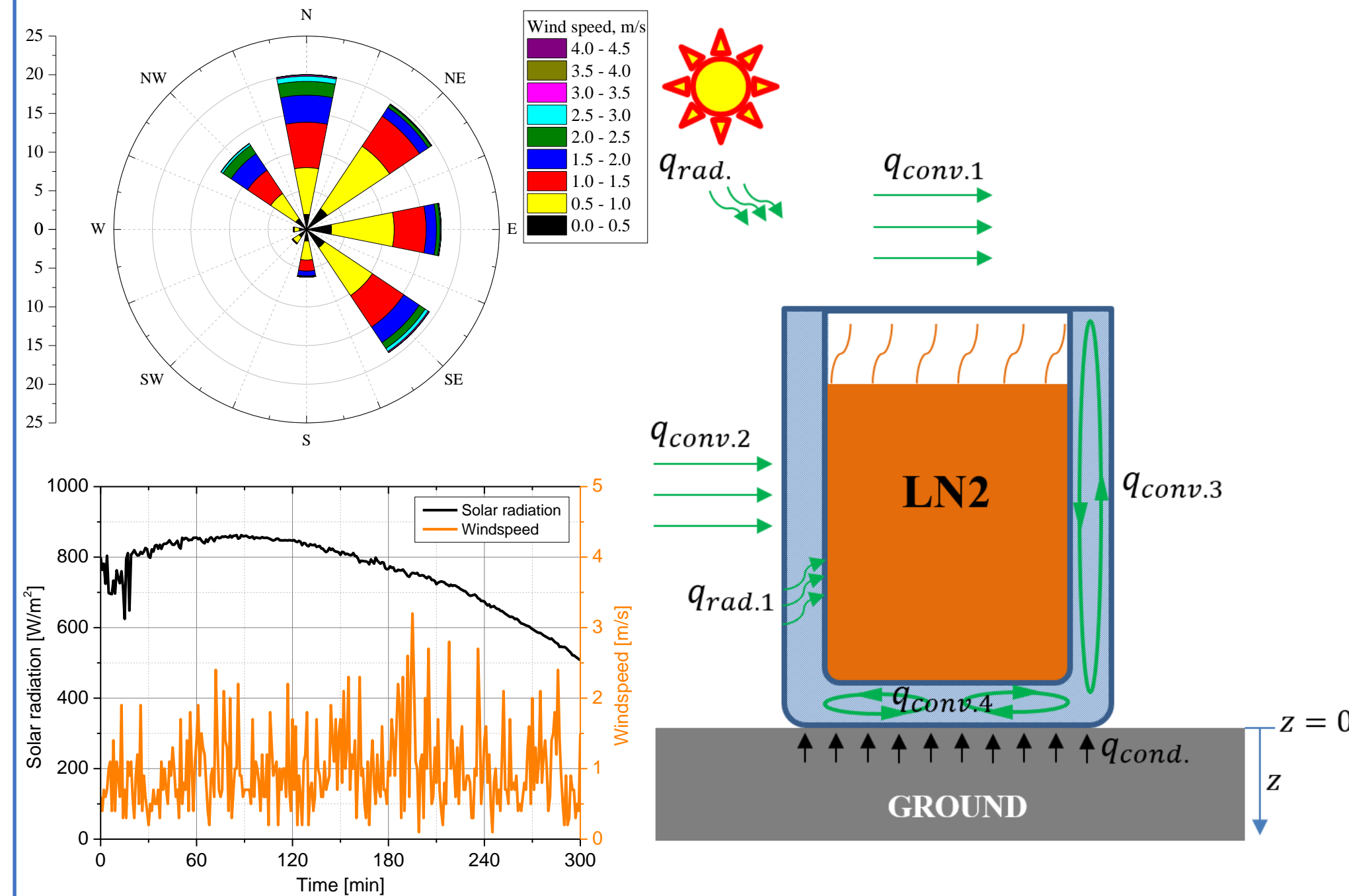
INTRODUCTION

- Vacuum insulation is commonly used in cryogenic liquid storage because of its excellent insulation performance. However, the accidental loss of insulating vacuum might result in significant boil-off of the stored liquid. The loss of the stored liquid through boil-off not only impacts energy efficiency but also poses a serious safety concern. The hazards associated with the storage of cryogenic liquids must be thoroughly considered to protect humans, assets, and the environment.
- Quantitative risk assessment (QRA) is an essential tool for evaluating the risks. As a critical step in QRA, the consequence estimation process requires the evaluation of the boil-off rate of the liquid in the case of vacuum failure to analyze incident outcomes.
- In this study, the boil-off of liquid nitrogen stored in a double-walled vacuum-insulated tank in the case of vacuum loss was numerically and experimentally investigated.
- The findings of this study are expected to enhance the understanding of the boil-off behavior of cryogenic liquids in the case of the failure of vacuum insulation and also provide an efficient numerical tool for analyzing the incident outcomes for consequence estimation as a part of QRA.

EXPERIMENTAL SETUP



MODELING



$$q_{conv.1} = \frac{\pi d^2}{4} \times \frac{\lambda Nu}{d} (T_a - T_b)$$

$$Nu = 0.664 Pr^{1/2} Re^{1/2} \text{ if } Re < 320,000 \text{ else } Nu = 0.037 Pr^{1/3} (Re^{0.8} - 15,200)$$

$$Re = (u_{10} \rho d) / \mu; Pr = (c_p \mu) / \lambda$$

$$q_{conv.2} = \frac{\pi DH}{2} \times \frac{\lambda Nu}{D} (T_a - T_w)$$

$$Nu = 0.3 + \frac{0.62 Re^{1/2} Pr^{1/3}}{[1 + (0.4 Pr)^{2/3}]^{1/4}} \left[1 + \left(\frac{Re}{282,000} \right)^{5/8} \right]^{4/5}$$

$$Re = (u \rho D) / \mu$$

$$q_{conv.3} = \pi d \times 0.364 \lambda (T_w - T_b) Ra_h^{1/4}$$

$$Ra_h = g \beta (T_w - T_b) h^3 / (\alpha \nu)$$

$$q_{conv.4} = \frac{\pi d^2}{4} \times \frac{\lambda Nu}{\delta} (T_{wb} - T_b)$$

$$Nu = 0.069 Ra^{1/3} Pr^{0.074}$$

$$Ra = g \beta (T_{wb} - T_b) \delta^3 / (\alpha \nu)$$

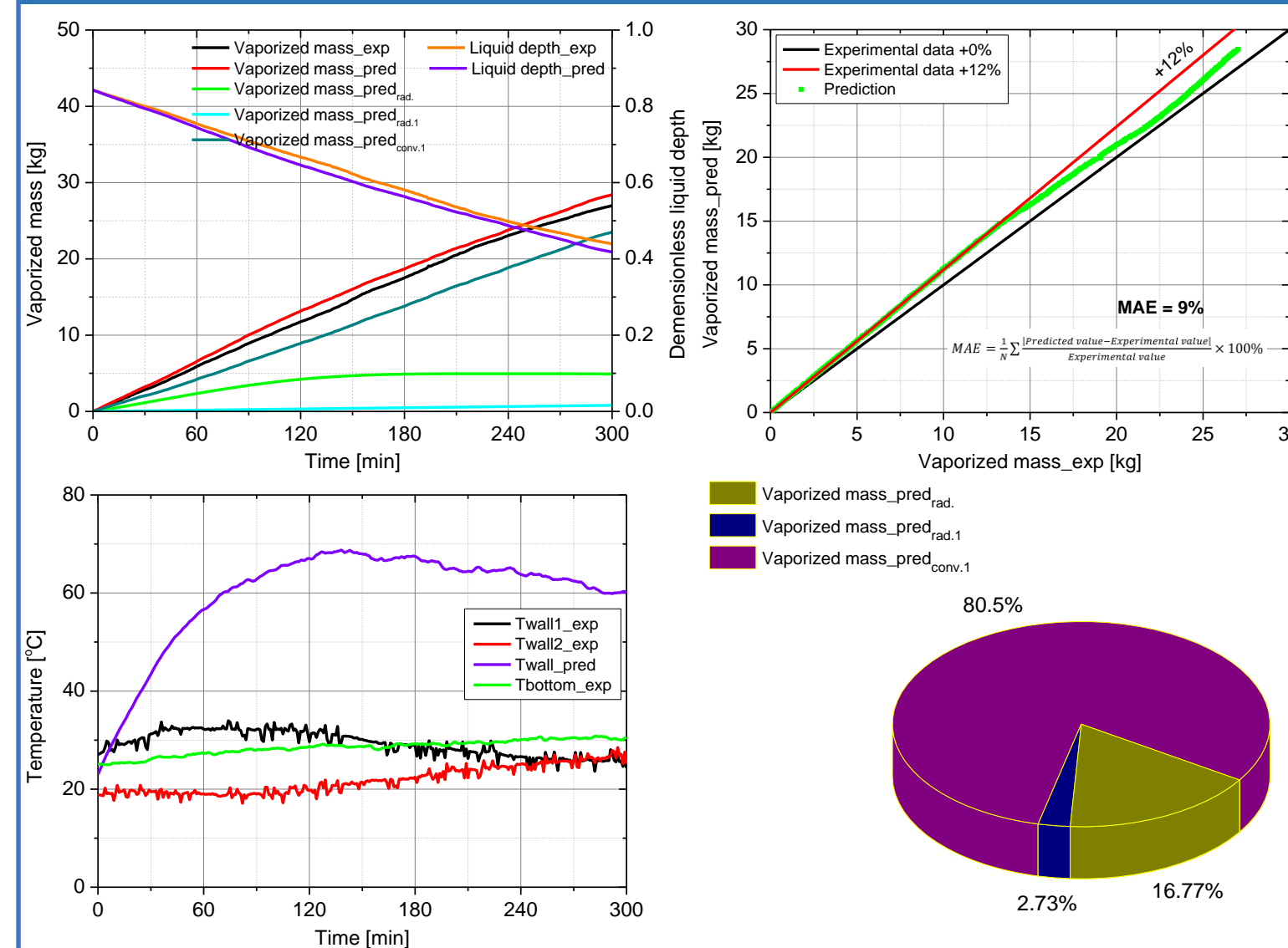
$$q_{cond} = \frac{\pi d^2}{4} \times \lambda \frac{\partial T}{\partial z} \Big|_{z=0^+}$$

$$q_{rad.1} = \pi d h \frac{\sigma (T_w^4 - T_b^4)}{1/\epsilon_w + 1/\epsilon_b - 1}$$

$$m_w c_w \frac{dT_w}{dt} = q_{rad} + q_{conv.2} - q_{rad.1} - q_{conv.3}$$

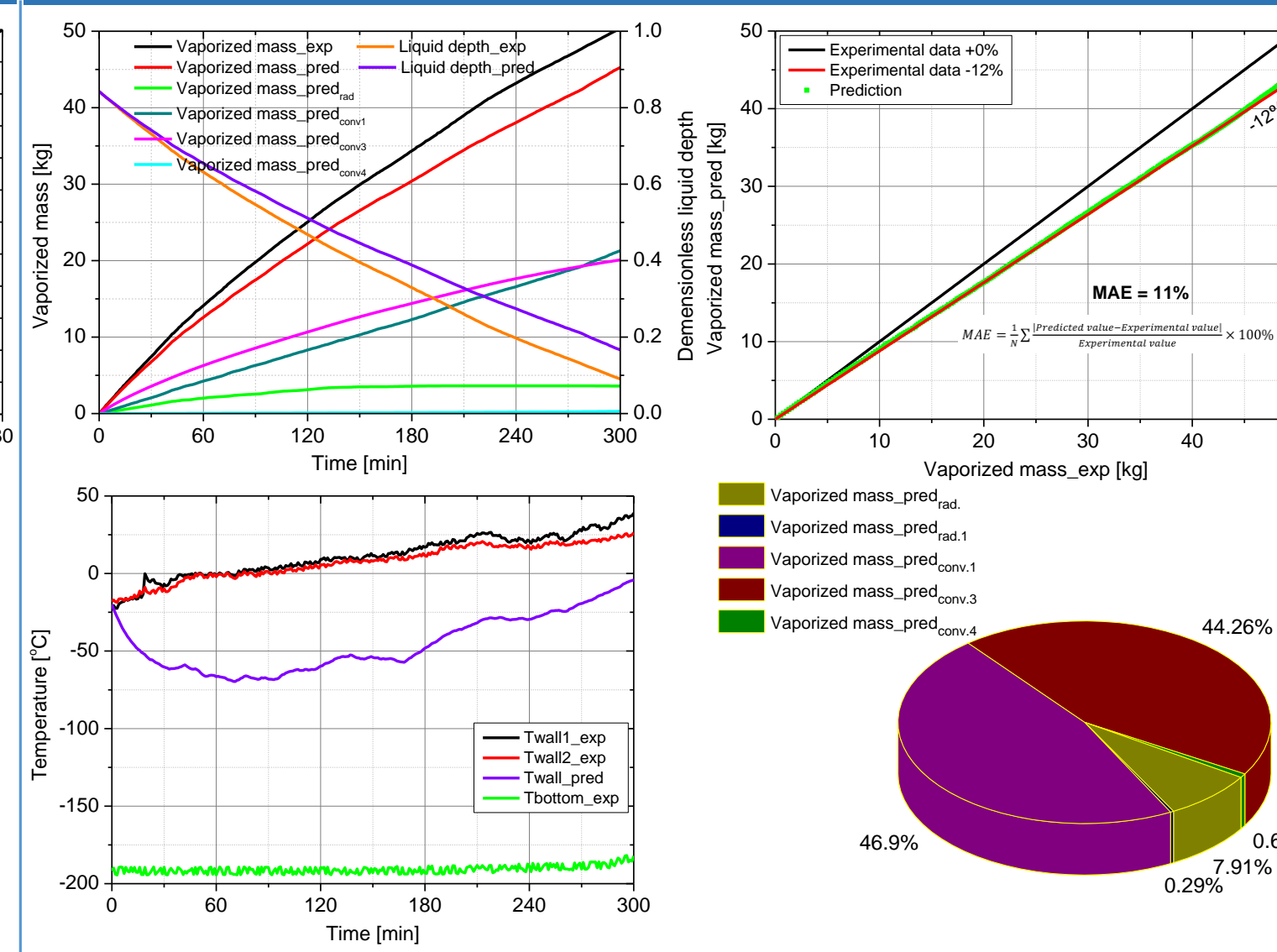
RESULTS

INTACT VACUUM-INSULATED TANK



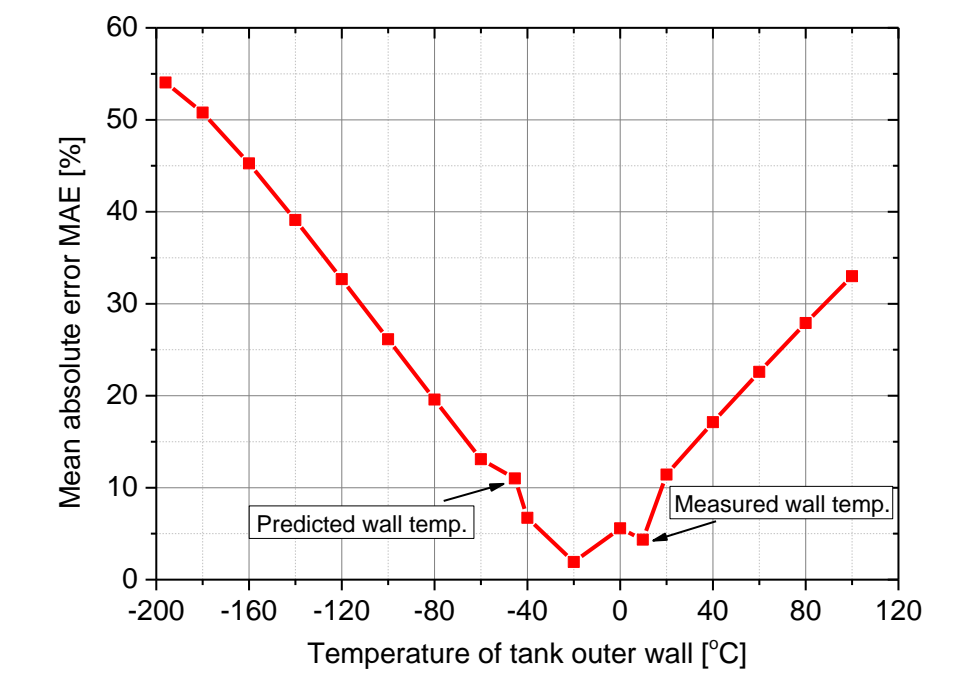
- The heat sources include $q_{conv.1}$, $q_{rad.1}$, and $q_{conv.4}$ with the contribution of the former to the vaporization being significantly greater than that of the latter.
- The prediction agreed well with experiment, with the MAE of 9%.

DAMAGED VACUUM-INSULATED TANK



- The contributions of $q_{conv.1}$ and $q_{conv.3}$ to the vaporization were quite similar.
- The model significantly underestimated the tank wall temperature.
- The predicted vaporized mass agreed well with experiment, with the MAE of 11%.

RESULTS (cont.)



- The effect of outer wall temperature on the predicted vaporized mass in the case of damaged insulation was investigated.
- The neglect of convection heat transfer between tank walls resulted in the MAE of 55%.

CONCLUSIONS

- The heat sources for liquid vaporization, including radiation heats q_{rad} and $q_{rad.1}$, convection heat from the ambient air to the liquid surface $q_{conv.1}$, convection heat from the ambient air to the outer tank wall $q_{conv.2}$, and convection heats from the outer to the inner tank wall $q_{conv.3}$ and $q_{conv.4}$ were modeled.
- The prediction for vaporized mass agreed well with experiments. However, the discrepancies between predicted temperature of tank wall with measurements were observed.
- The contributions of $q_{conv.1}$ and $q_{conv.3}$ to the vaporization were quite similar.
- The heat transferred by radiation between two tank walls $q_{rad.1}$ was negligible compared to convection heat $q_{conv.3}$.

REFERENCES

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ACKNOWLEDGEMENTS

This research was supported by a research program funded by the Ministry of Trade, Industry and Energy of the Republic of Korea and Korea Institute of Energy Technology Evaluation and Planning (Grant number: 20215810100020).

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