



29.05.2024

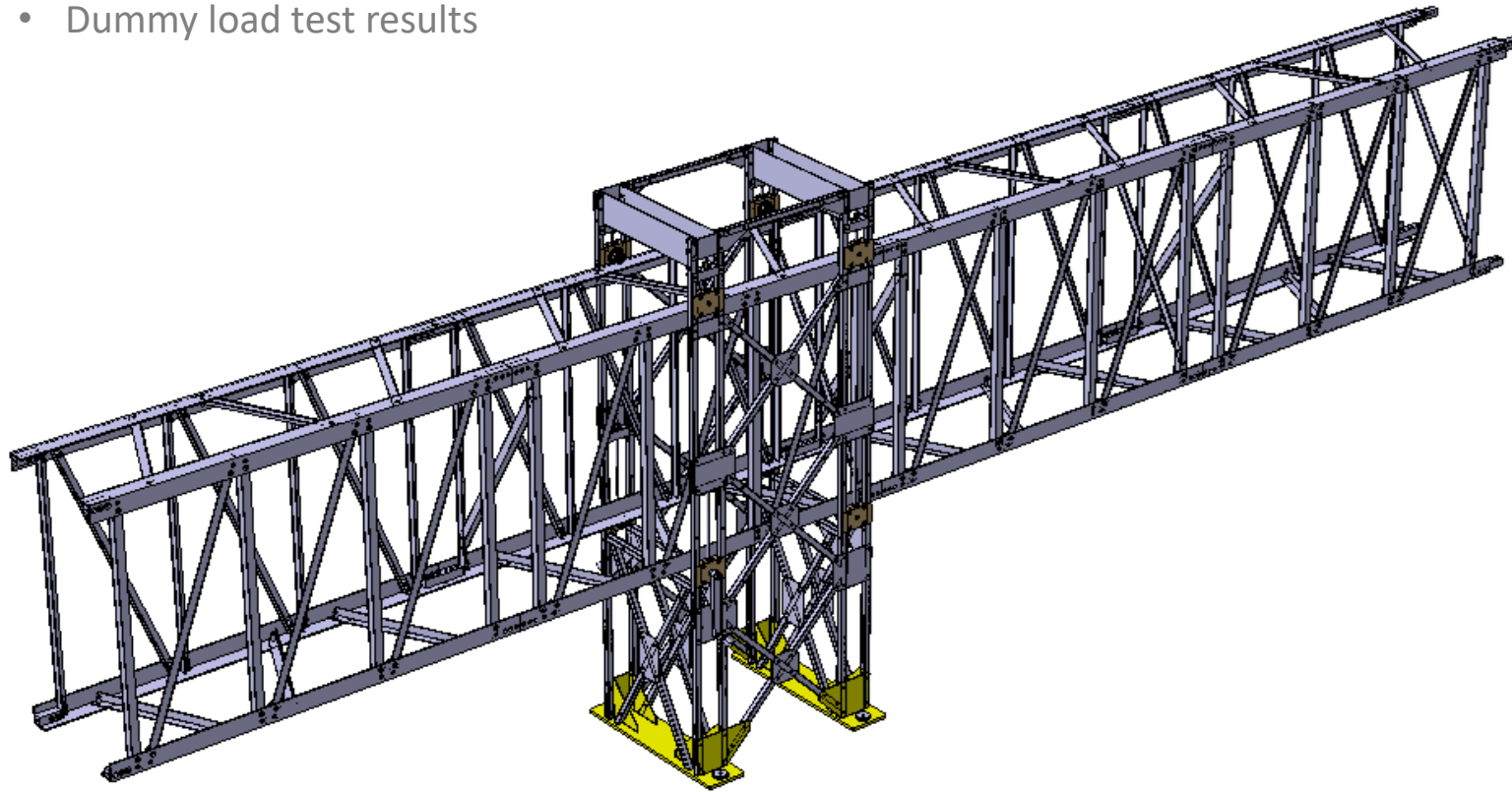
Design and testing of a dynamic support frame structure for the CMS tracker installation process

Mikko Barinoff, Nicolas Siegrist, Hans Postema

on behalf of the CMS Tracker -group

Contents

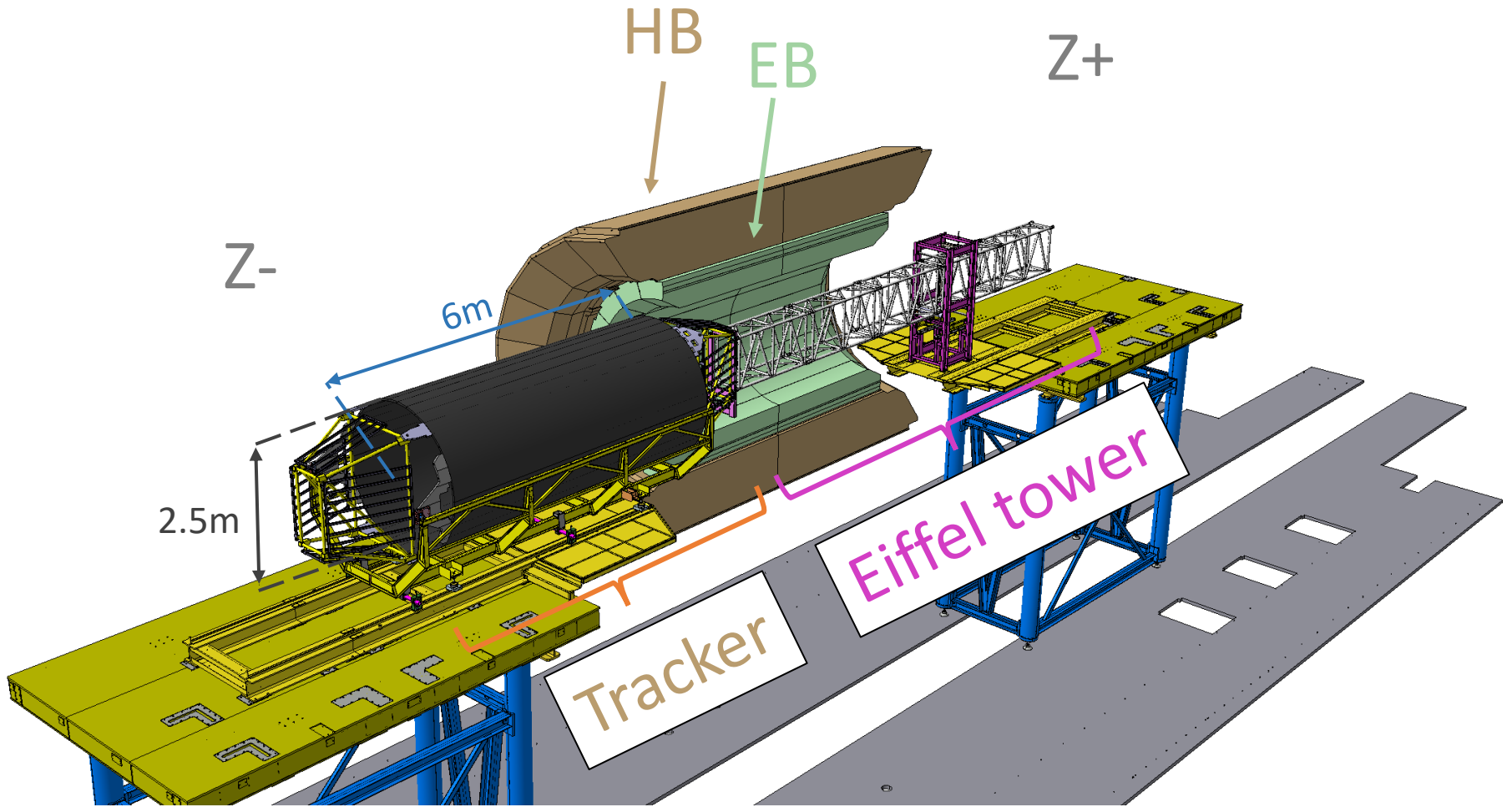
- Introduction to the eiffel tower concept
- Study of the old structure, motivation and overview of the new design
- Design validation
 - Dummy load test results





Eiffel tower concept

Eiffel tower concept



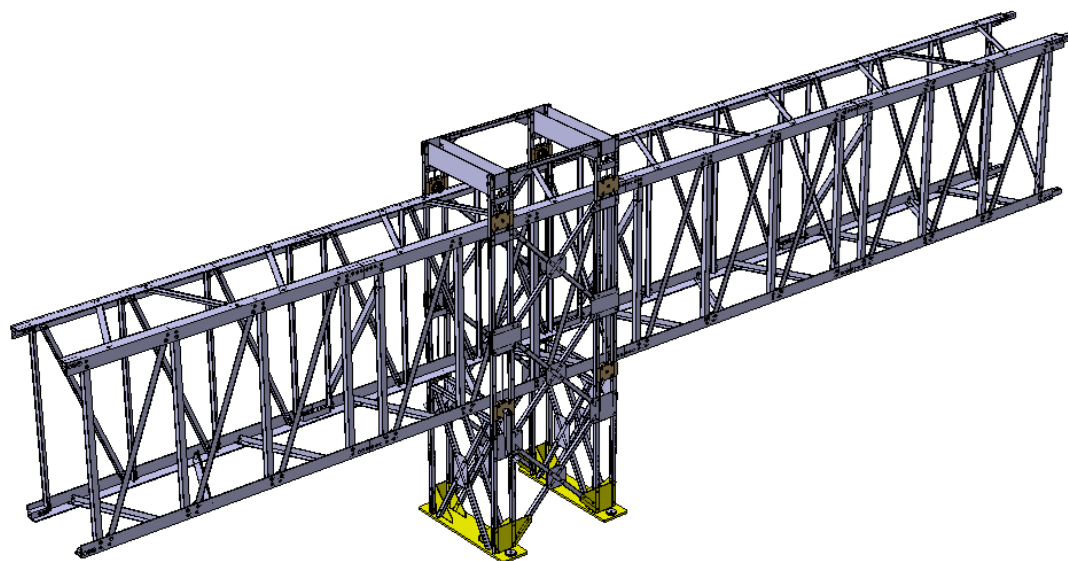
(video)



Study of the old structure

Starting point of the study

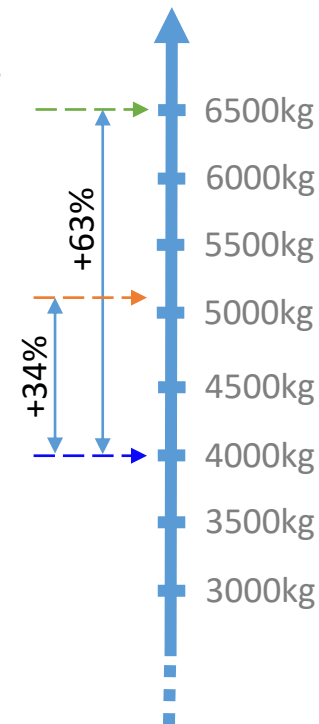
- Can we re-use the old eiffel tower?



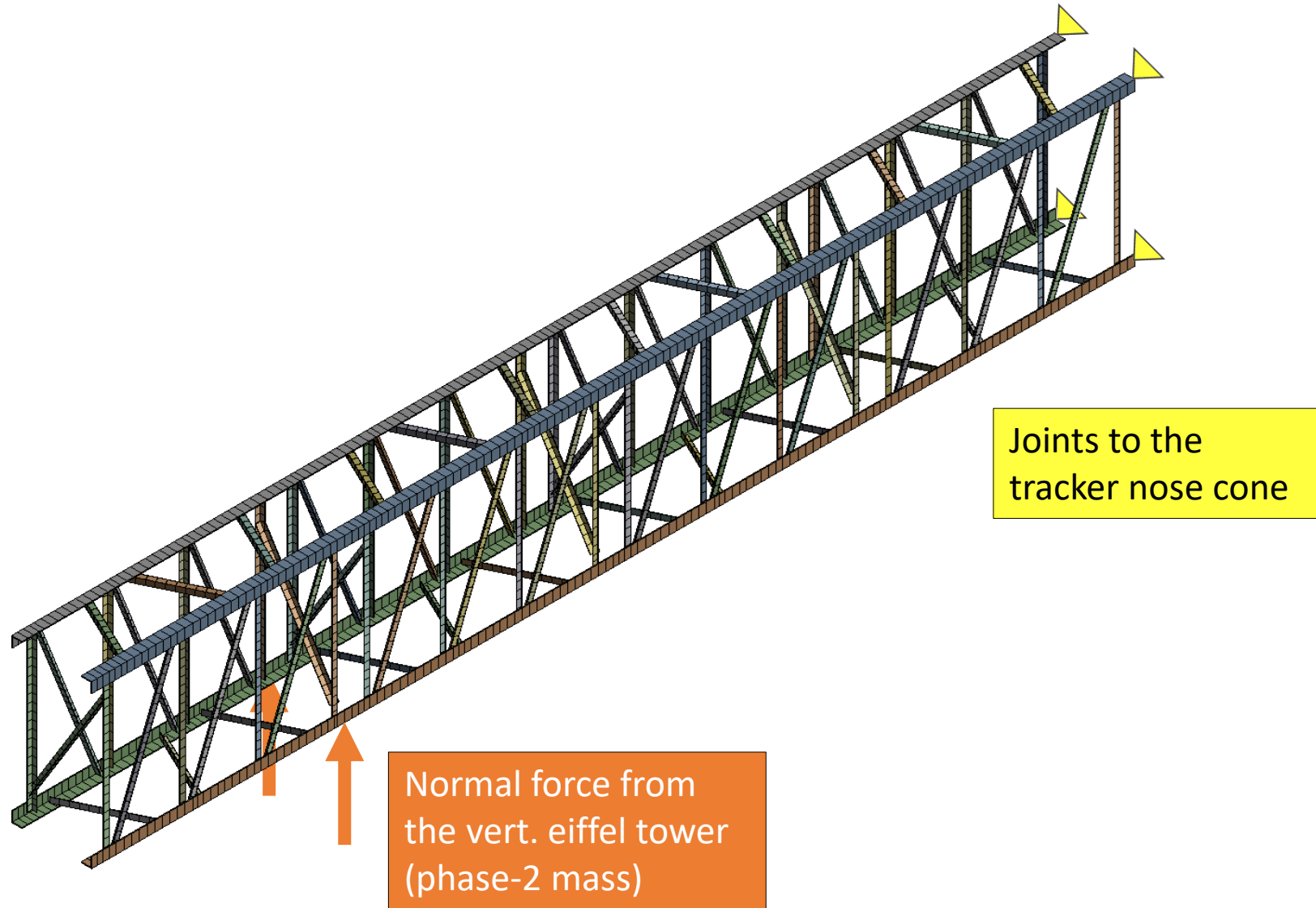
Phase-2 tracker
(estimate)

Current tracker
(actual)

Current tracker
(design value
for old ET)

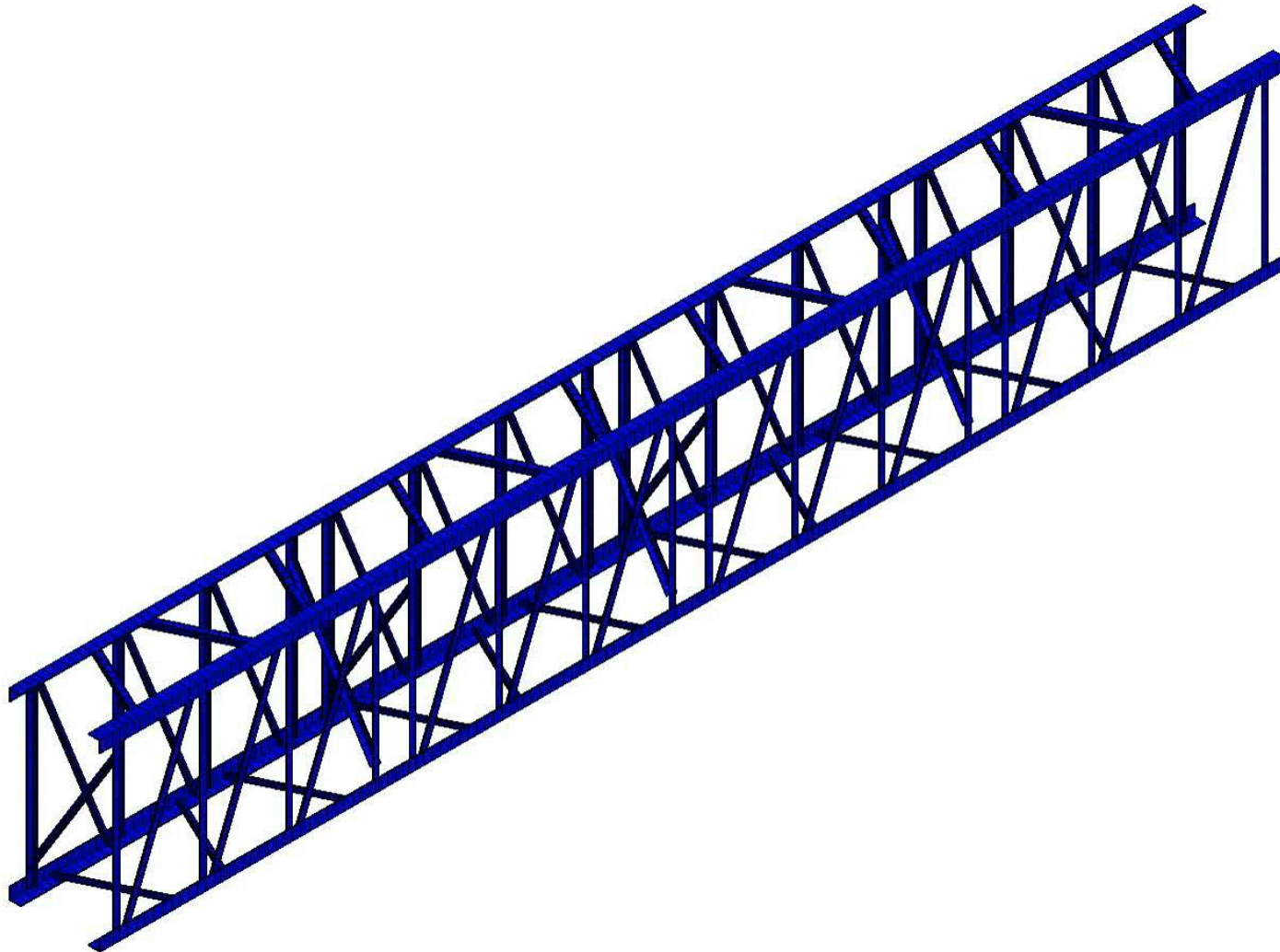


Structural study of the old Eiffel tower



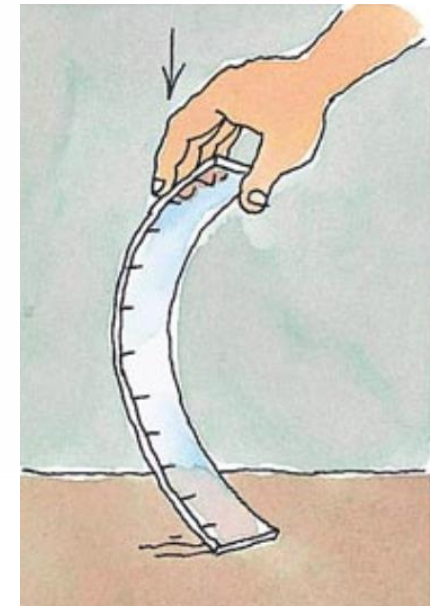
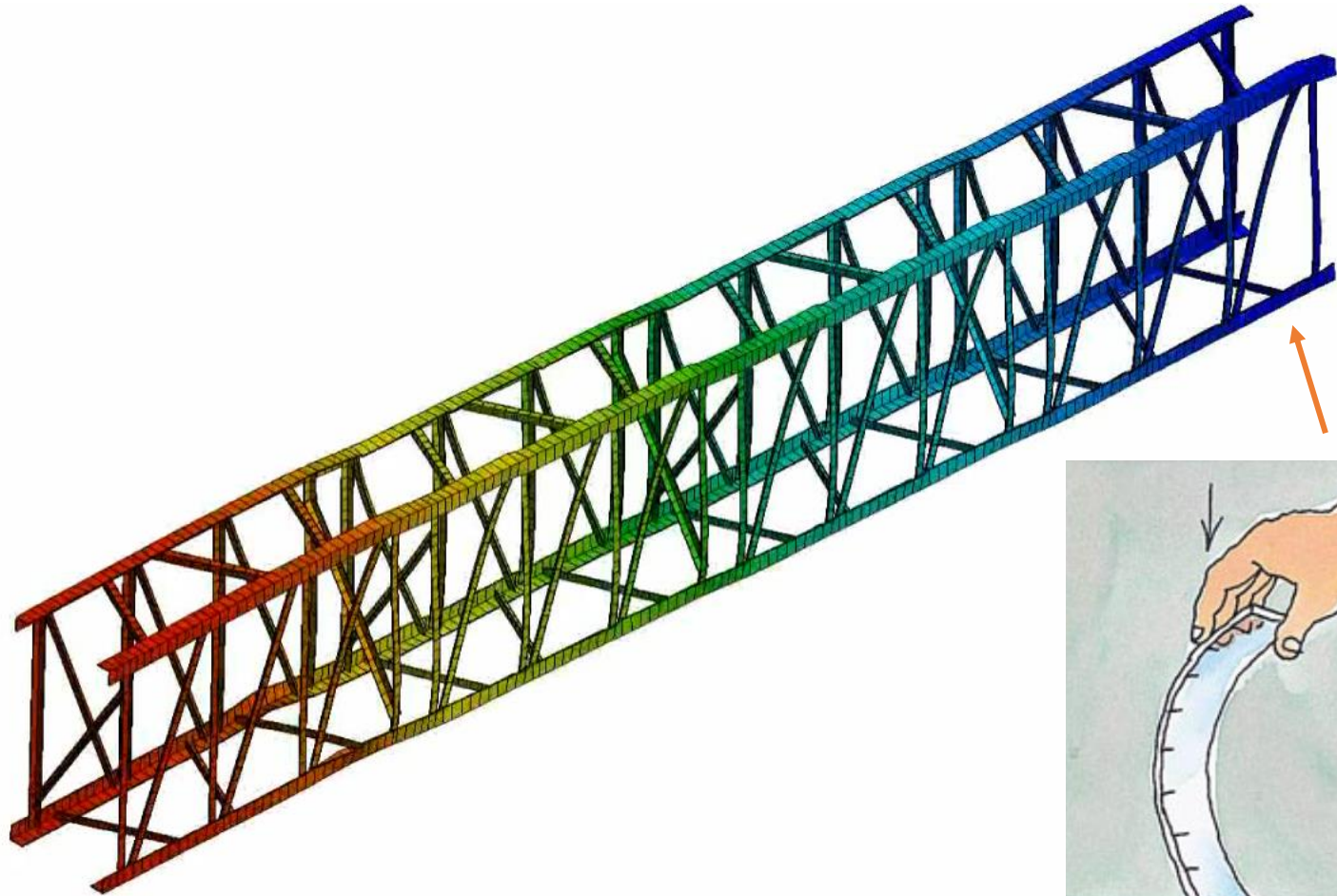


Structural study of the old Eiffel tower



(video)

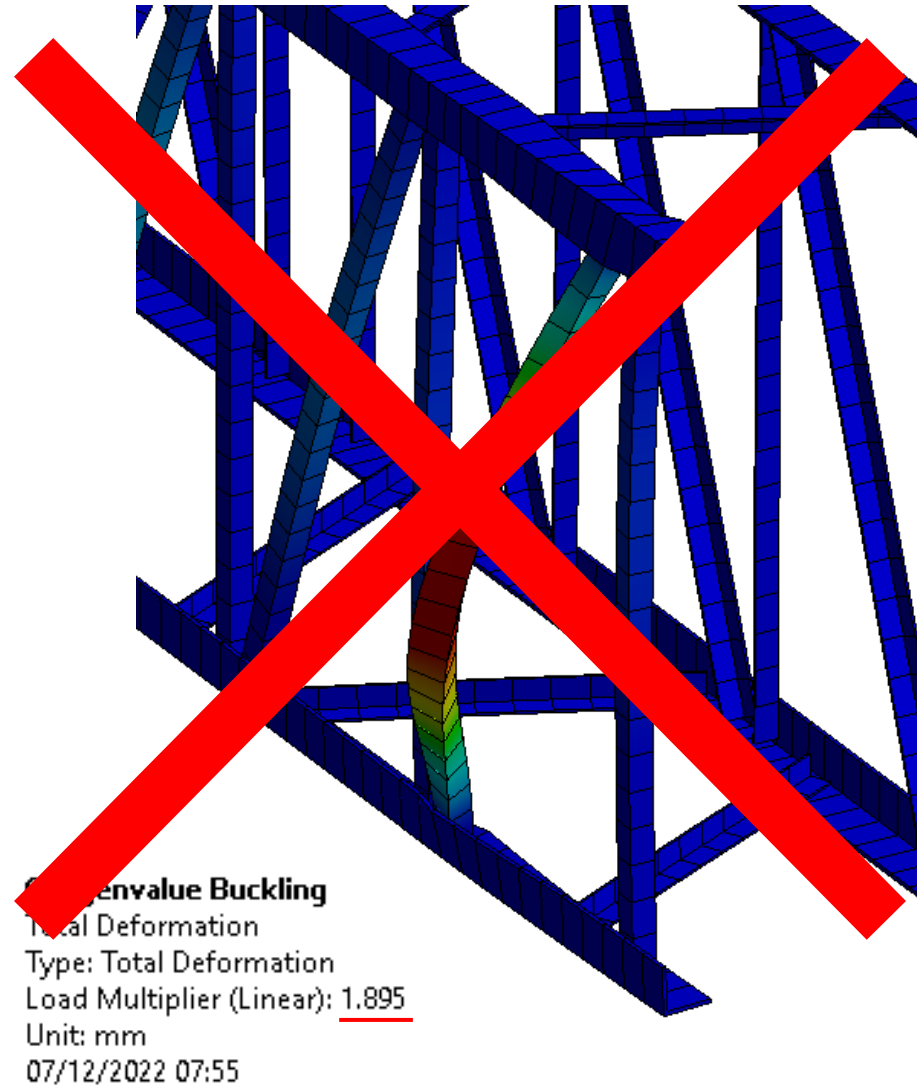
Structural study of the old Eiffel tower



Buckling



Structural study of the old Eiffel tower





New eiffel tower design



Make or buy?



New eiffel tower design



Installation tooling for particle trackers



Images

Videos

News

Books

Finance

About 0 results (0,37 seconds)

Your search - **Installation tooling for particle trackers** - did not match any documents.

Suggestions:

- Make sure that all words are spelled correctly.
- Try different keywords.
- Try more general keywords.
- Try fewer keywords.

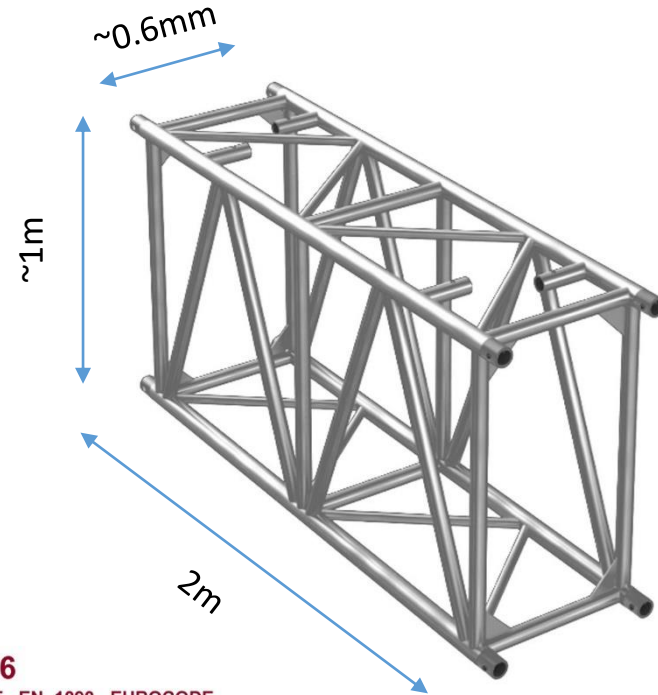
Inspiration for the new design





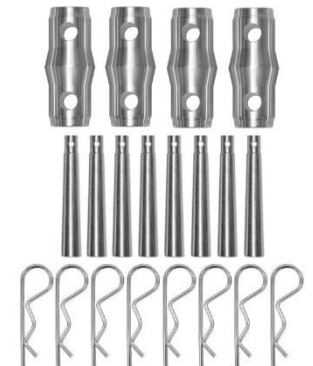
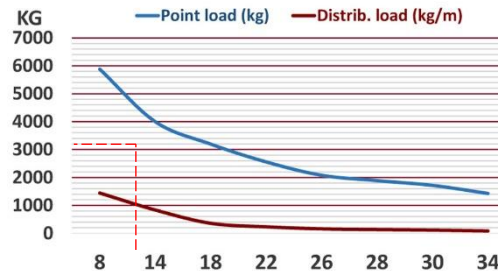
New horizontal eiffel tower

- Duratruss [DT 104/6-200 DT](#)
High Load Rigging Truss
 - Welded from 60x6 aluminum tube
- Would imply re-designing the vertical Eiffel tower and an adapter between the truss and the tracker nose-cone

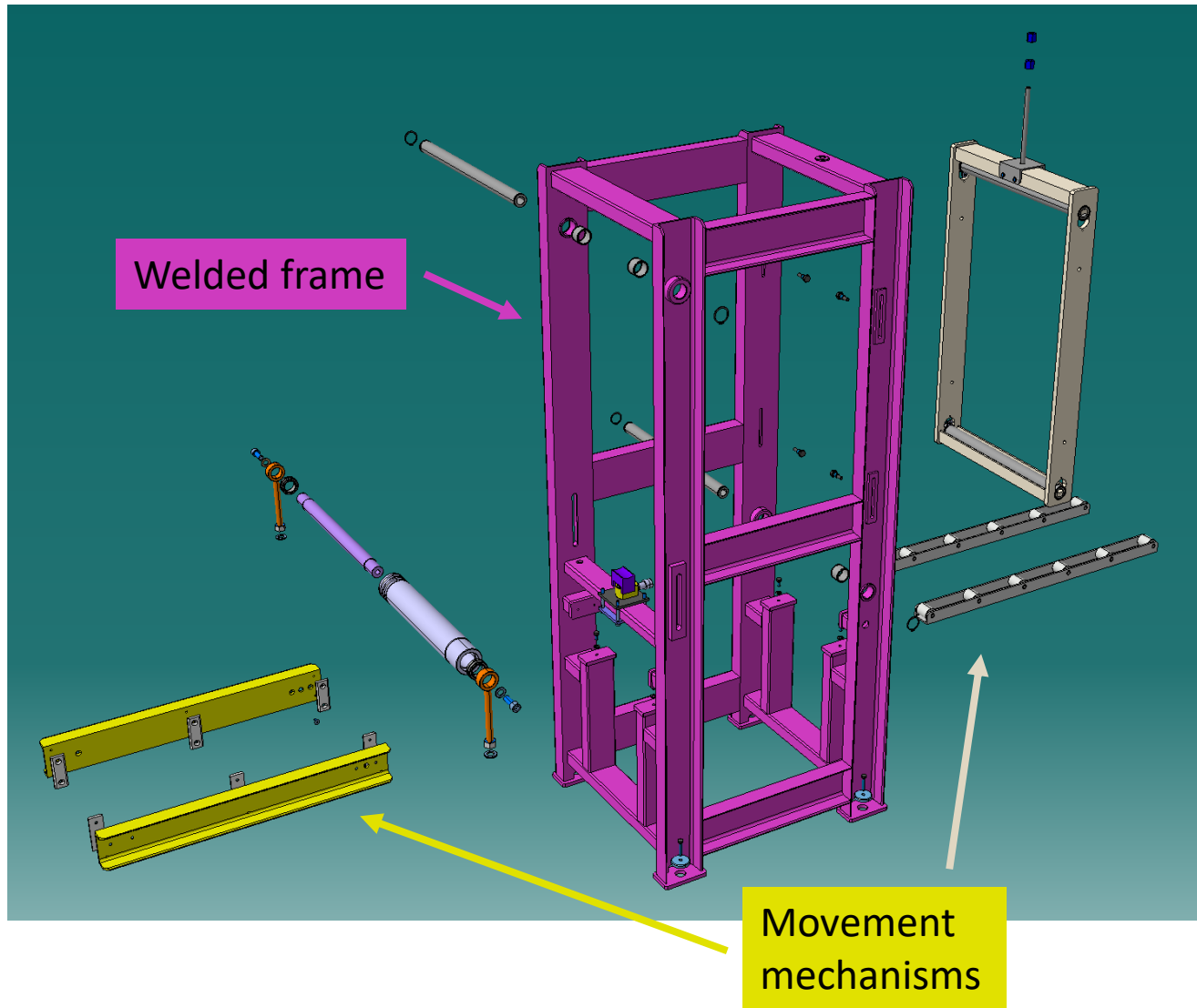


DT 104/6
LOAD CHART - EN -1090 - EUROCODE

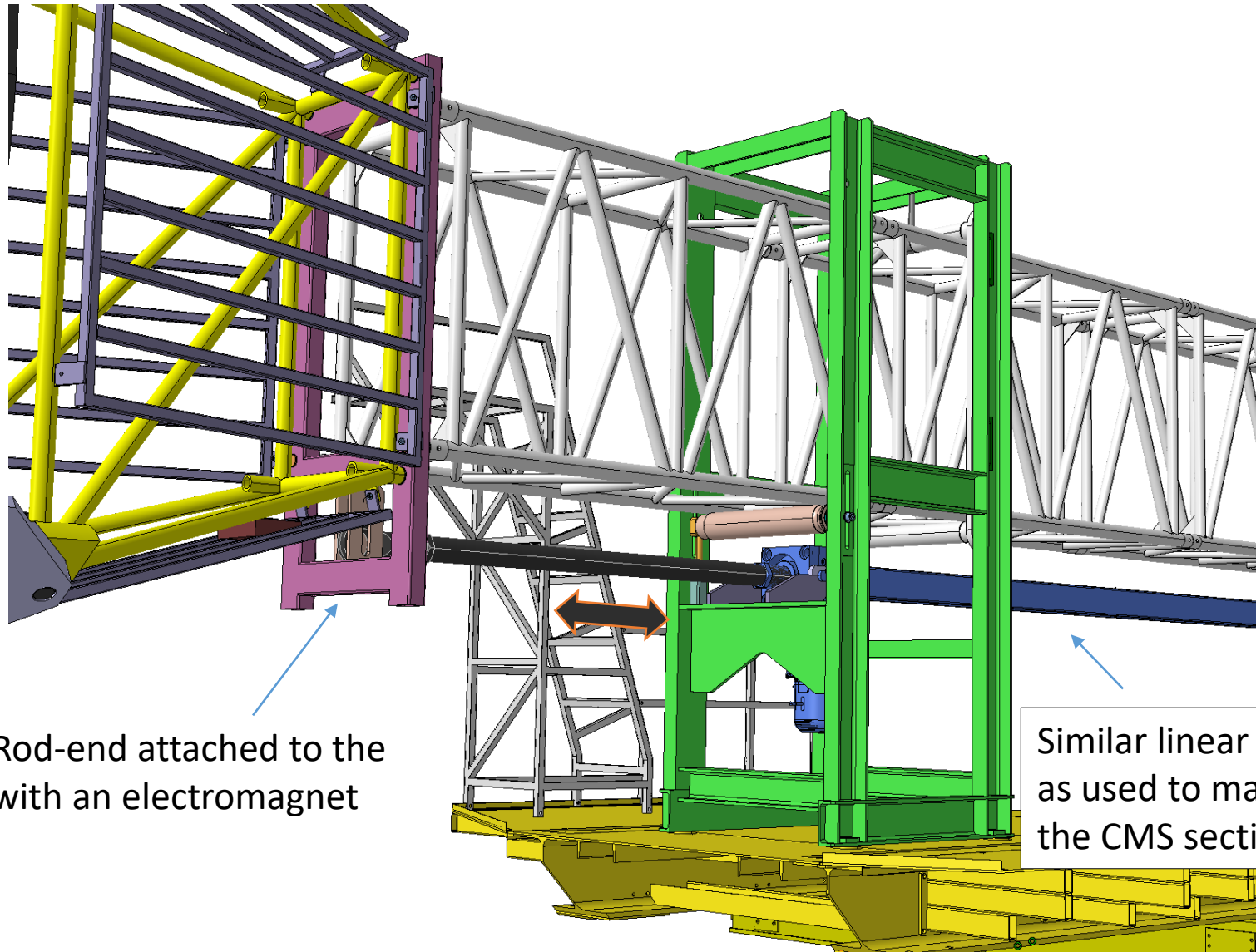
Length (m)	8	14	18	22	26	28	30	34
Point load (kg)	5880	4000	3196	2557,8	2079,6	1886,8	1716,7	1428,4
Deflection (mm)	9,8	36,8	64,4	98,3	139,1	162,5	188	245,7
Distrib. load (kg/m)	1441,6	835,7	358,7	232,5	160	134,8	114,4	84
Deflection (mm)	11,9	48,3	80	119,9	168,1	195,3	224,8	290,2



New vertical eiffel tower



New vertical eiffel tower



Rod-end attached to the with an electromagnet

Similar linear actuator as used to manipulate the CMS sections

New eiffel tower design



Horizontal tower assembled from the truss elements



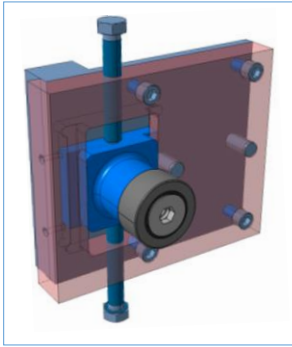
Custom design vertical eiffel tower welded from steel beams



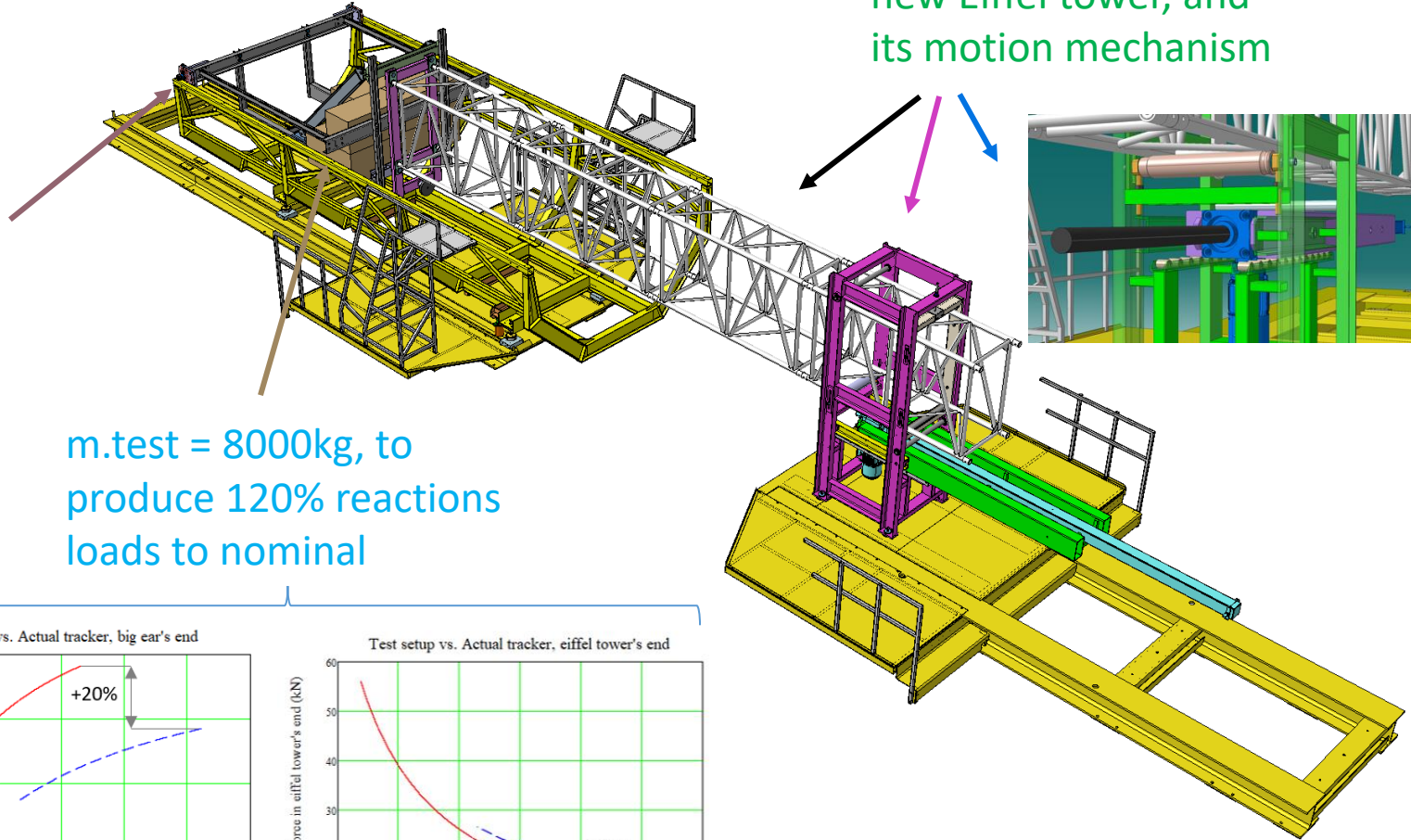
Testing

Test plan

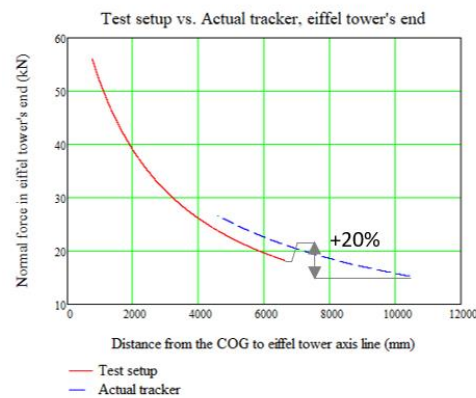
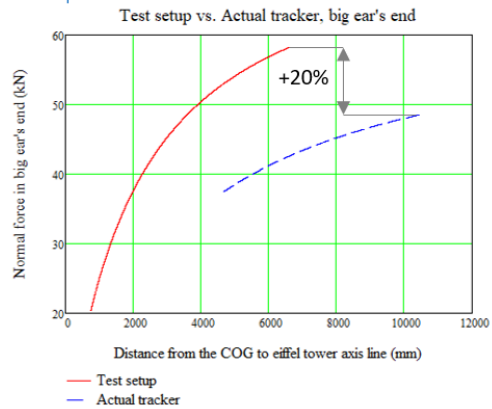
Test and validate the new Eiffel tower, and its motion mechanism



Test and validate the new big ear load transfer mechanisms



m.test = 8000kg, to produce 120% reactions loads to nominal



Setup at P5

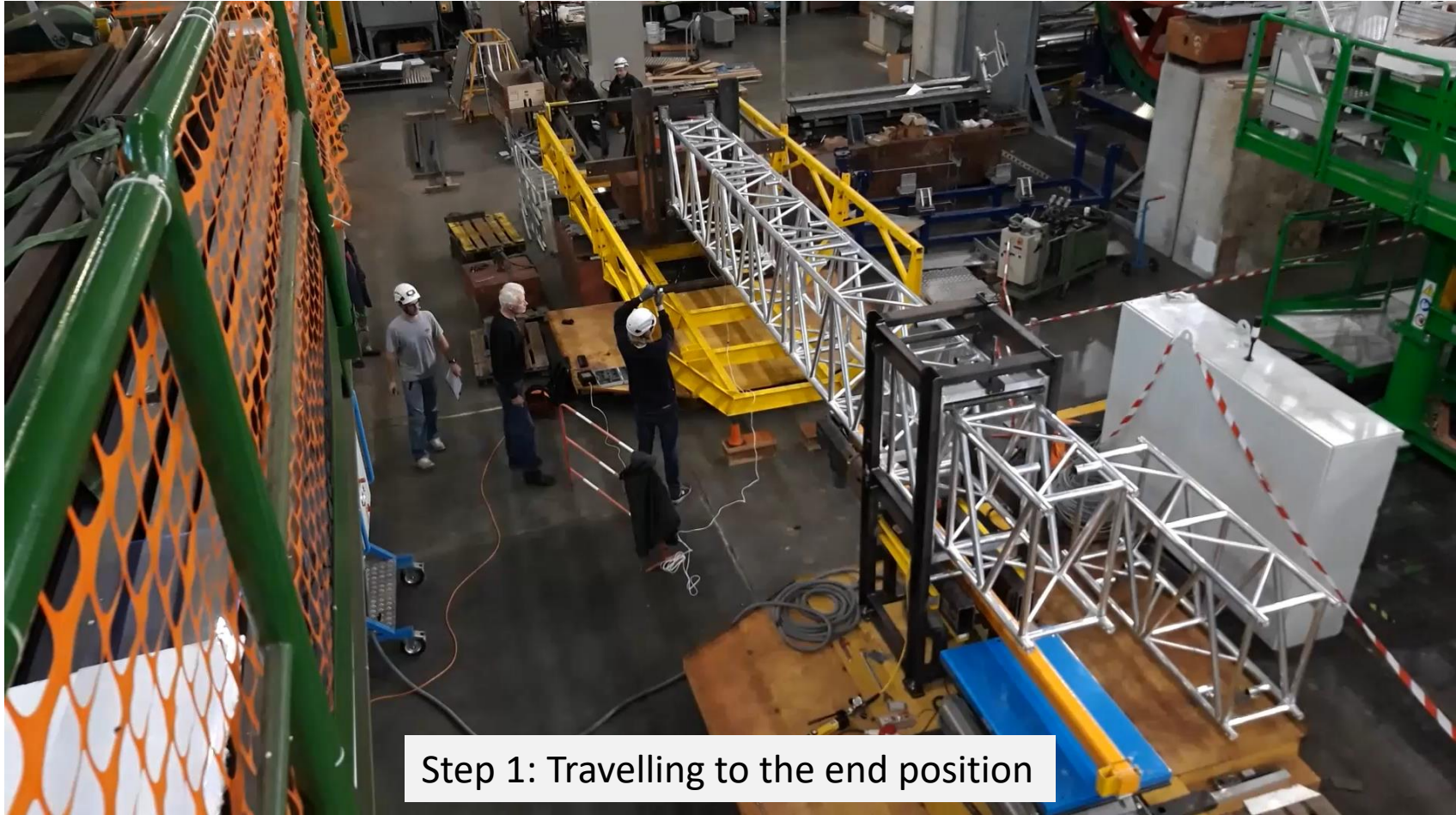


Test, scenario #1



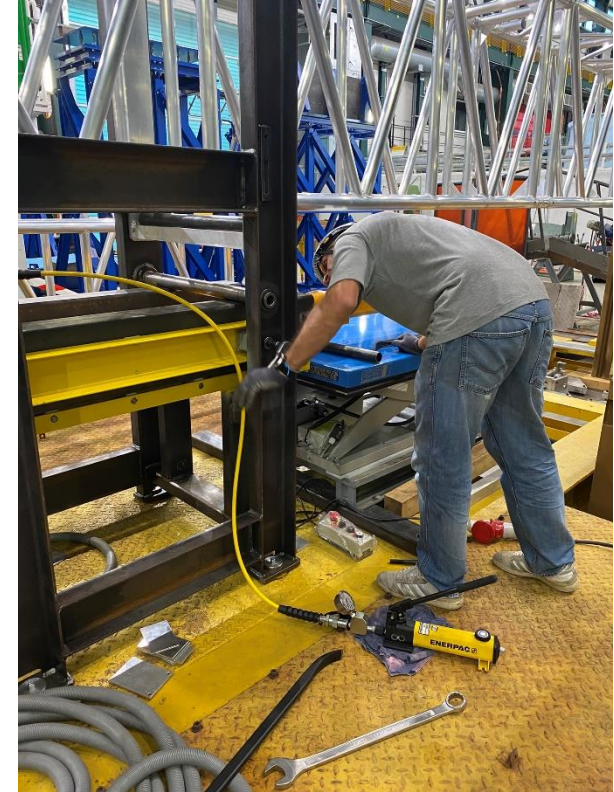
Step 0: 4000kg loaded in

Test, scenario #1



(video)

Test, scenario #1



Step 3: Heaviest loading case on the vertical eiffel tower, testing the height adjustment

Test, scenario #1



Step 4: Pushing back to the fully extended, lowering the load to the table

(video)

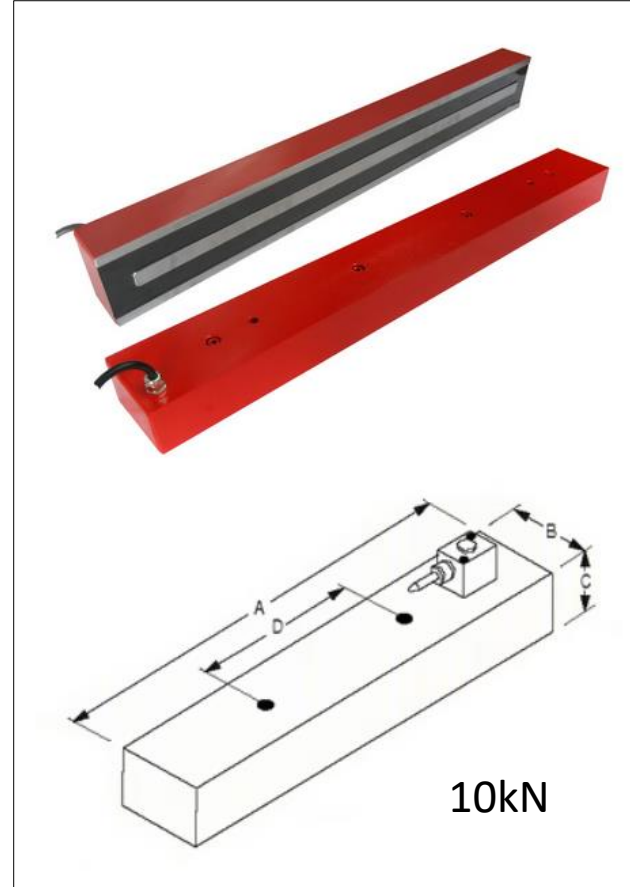
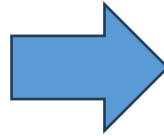
Test, scenario #1, challenges

Connection
break measured
around $\sim 4800\text{N}$



Experienced the magnet disconnecting especially when travelling over the truss element connection regions, where there's a step from the tube to the sleeve

New magnet



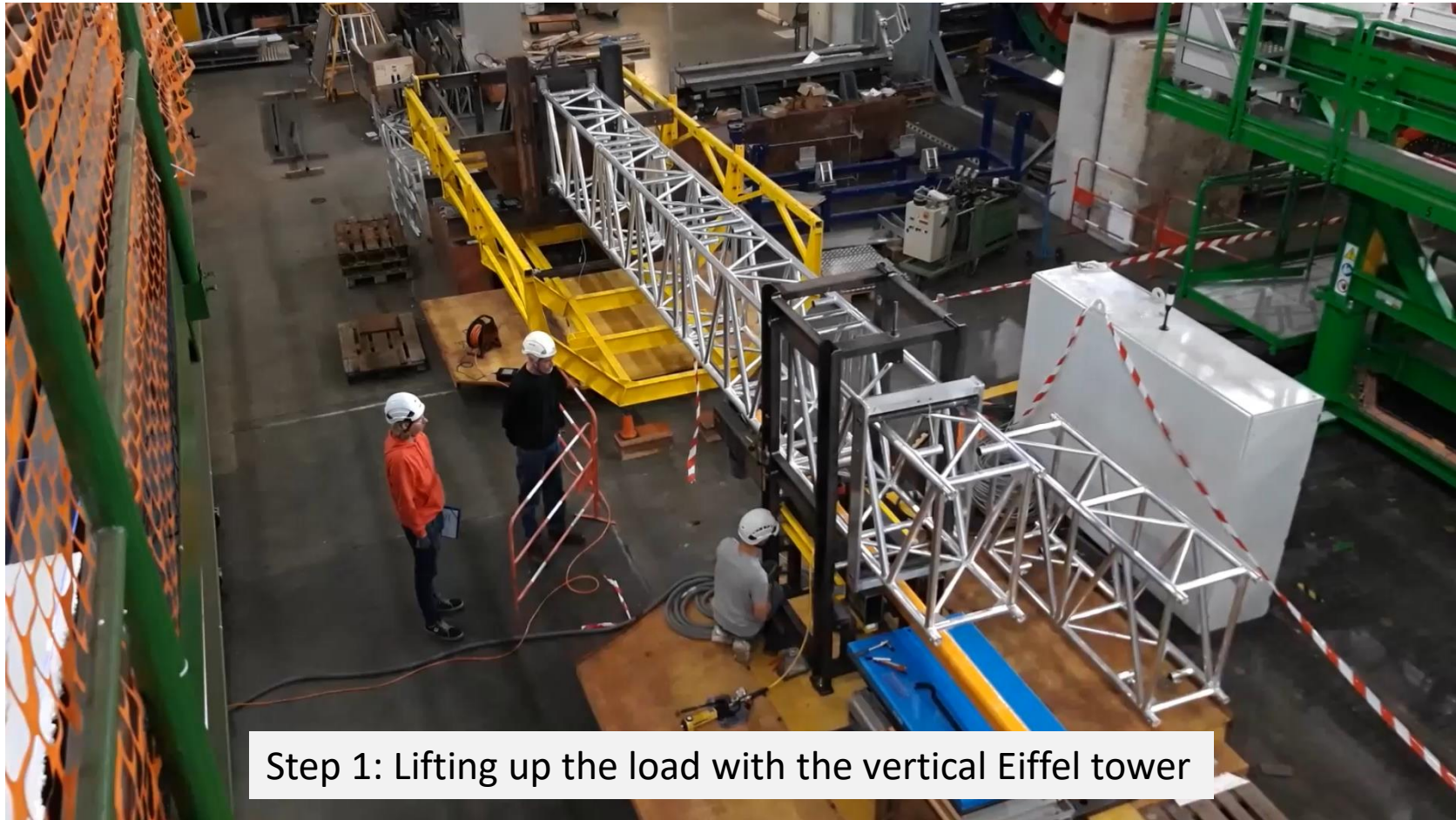
New magnet



Test, scenario #2

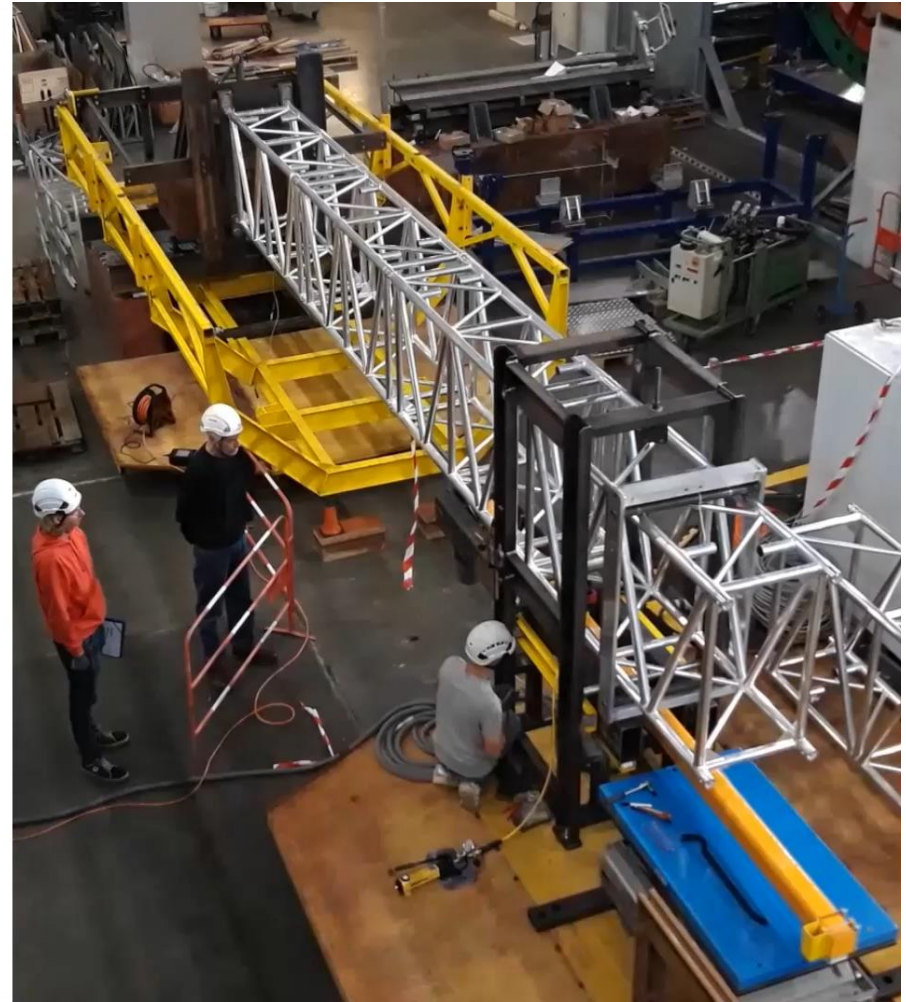
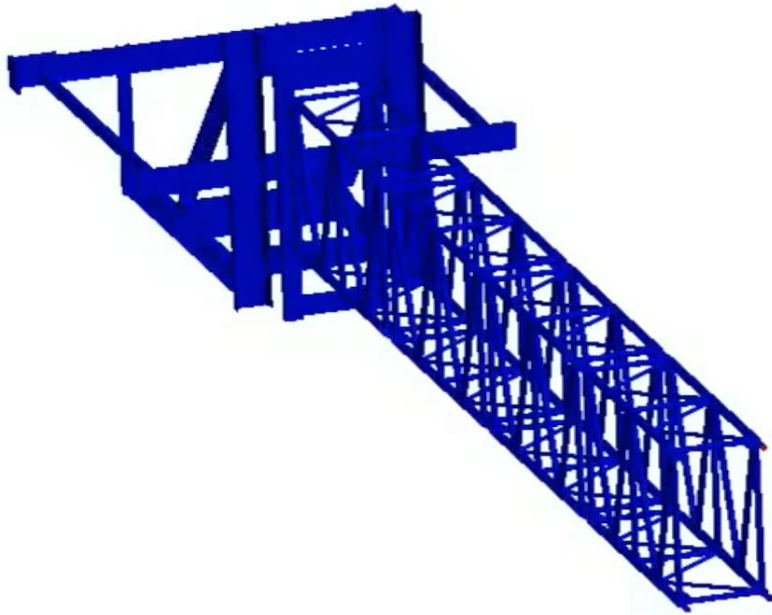


Test, scenario #2



(video)

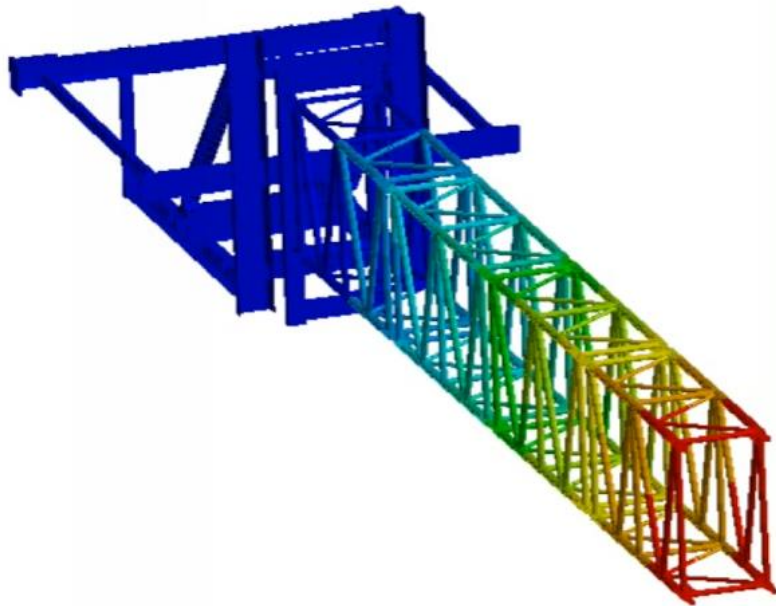
Test, scenario #2



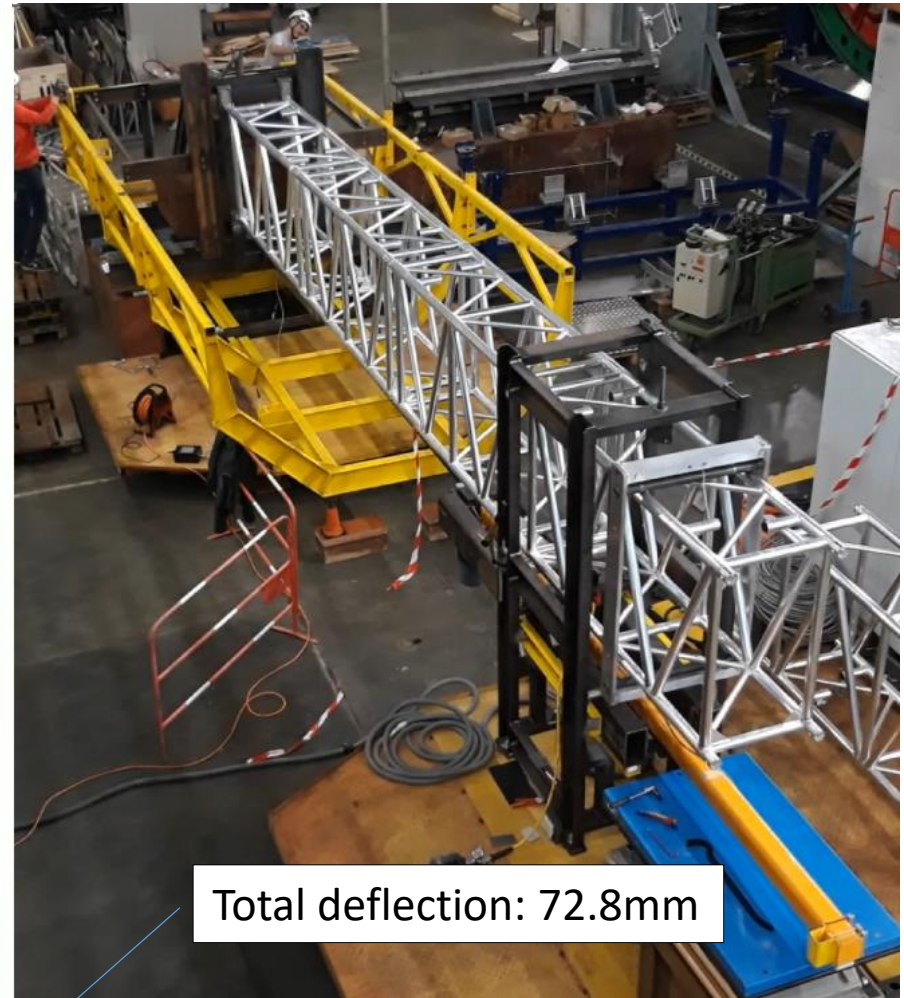
(video)



Test, scenario #2



Total deflection: 68.0mm



Total deflection: 72.8mm

-7%

Test, scenario #2



(video)

Test, scenario #2



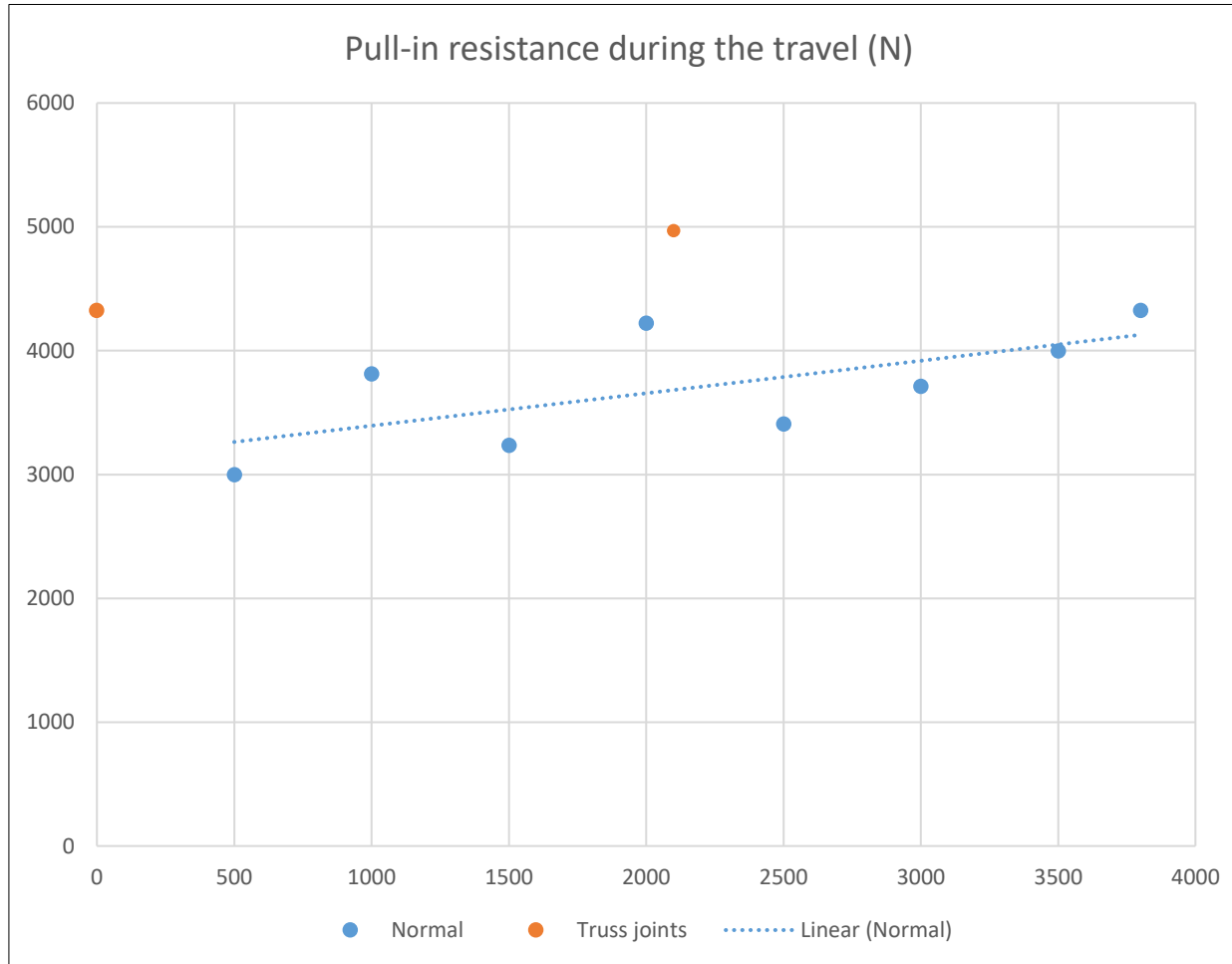
Test, scenario #2



Special
measurement
point when rolling
over the truss
element joints



Results



Test, scenario #2, challenges

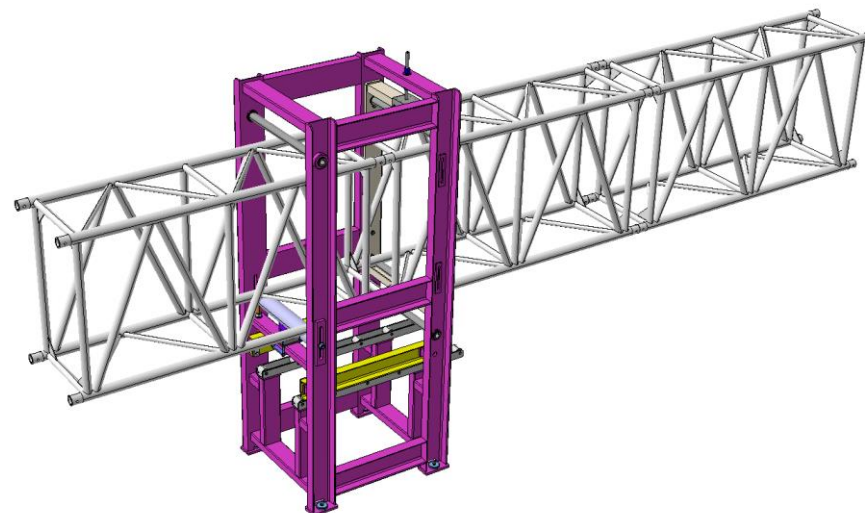


Some surface damage
(mainly paint peel-off) was
seen on the cradle beam

(video)

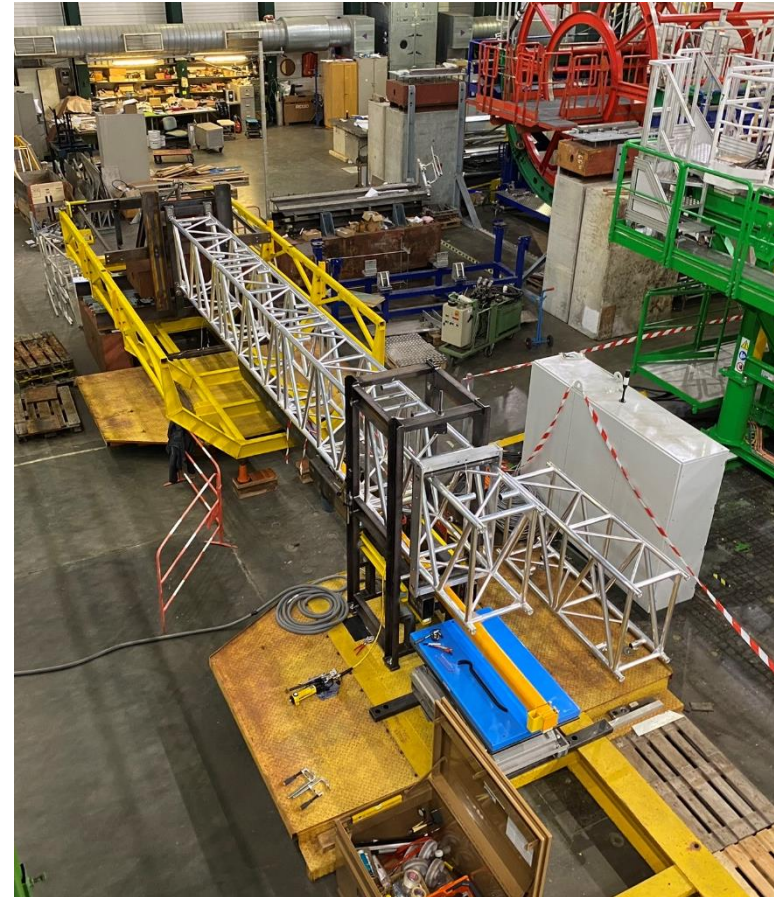
Conclusions (design)

- The old structure would've not been fit for the phase-2 loading conditions without notable adaptations
- It was easier to build a completely new one to implement all the “secondary” improvement in parallel as well
- The horizontal tower assembled from off-the-shelf welded truss elements, the vertical tower was custom built at P5



Conclusions (testing)

- Static structural configurations
 - Got all the worst case static load configurations done as planned
 - Structurally, no issues observed
 - Simulated value on the maximum deflection within 7% of the measured value
 - Good result, as limits in the measurement are a bit faint
- Z-direction motion mechanism
 - Got both of the motion configurations done as planned
 - The first magnet was too small, changed the to one which is 2x in capacity (10kN)
 - Highest measured rolling resistance values with that around 5kN
- The cradle deflections to be analyzed, will be used to benchmark the cradle FEM-models



A photograph of two men in a large industrial facility, likely a particle accelerator. They are wearing white hard hats and safety glasses. The man on the left is wearing a dark blue long-sleeved shirt with a logo and blue jeans. The man on the right is wearing a grey t-shirt and blue jeans. They are standing in front of a complex, multi-level metal structure made of silver-colored pipes and beams. The background shows more industrial equipment, including yellow overhead cranes and various pipes and cables. The lighting is bright, typical of an industrial environment.

Thanks!

Photos and videos:

- [Dry test](#)
- [Scenario 1](#)
- [Scenario 2](#)
- [Scenario 2 \(new magnet\)](#)

Documentation (in work)

- <https://edms.cern.ch/document/2872784>



Spares

Dry run



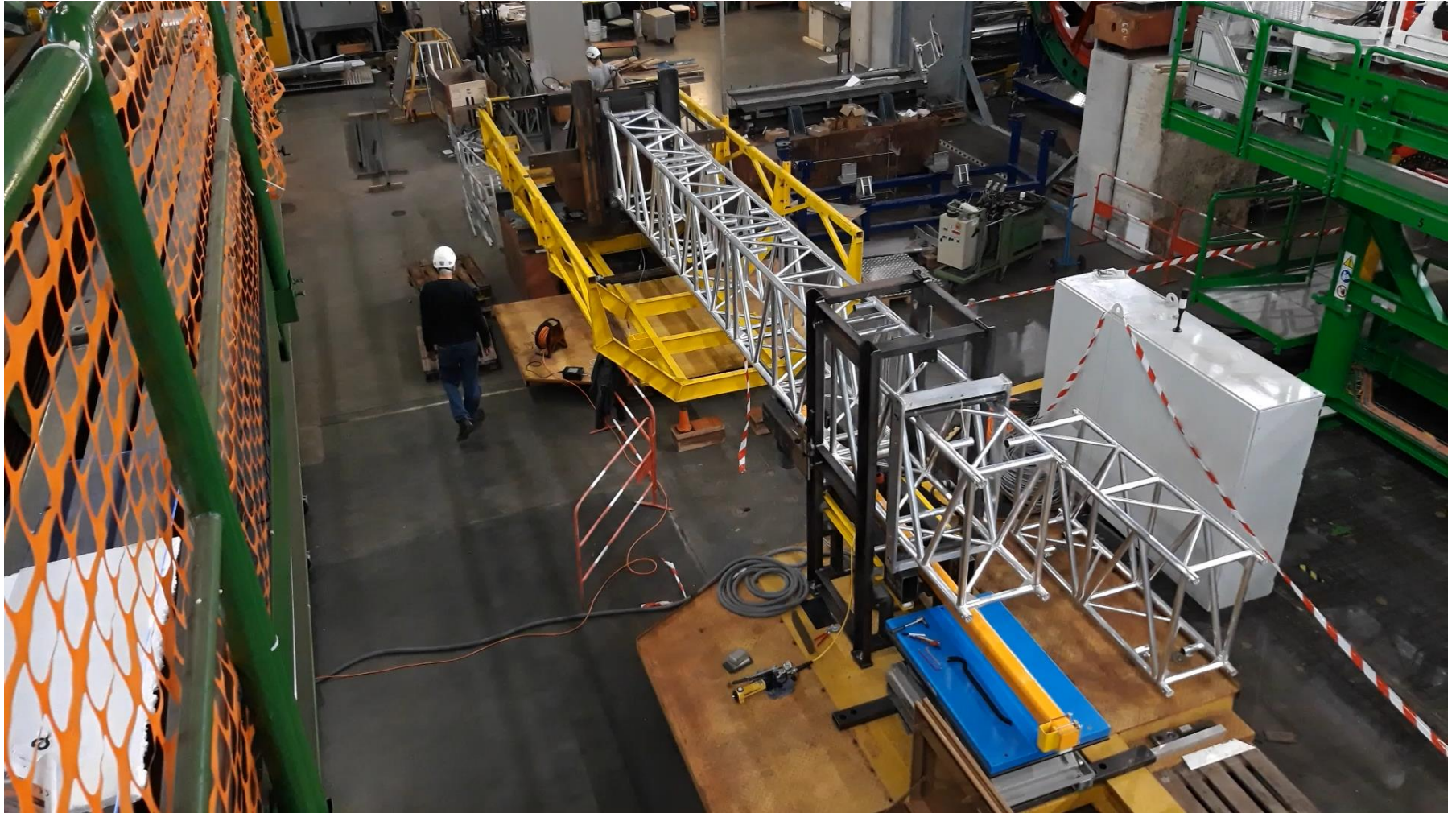
(video)

Scenario #1



(video)

Scenario #2



(video)

Scenario #2 (new magnet)

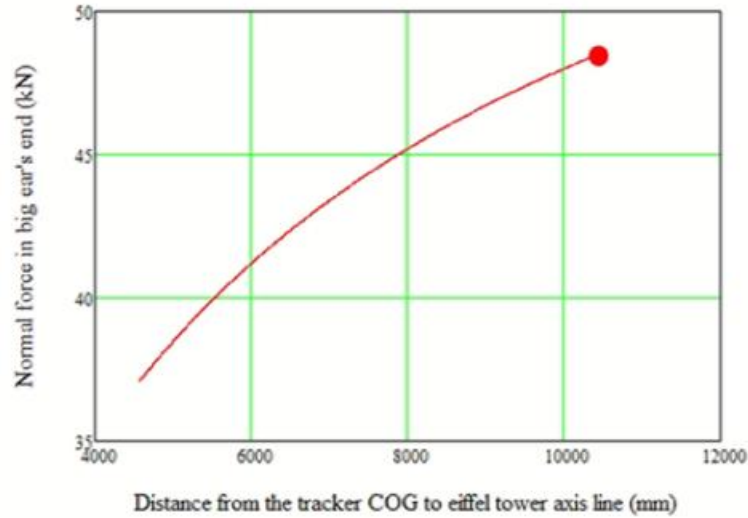


(video)

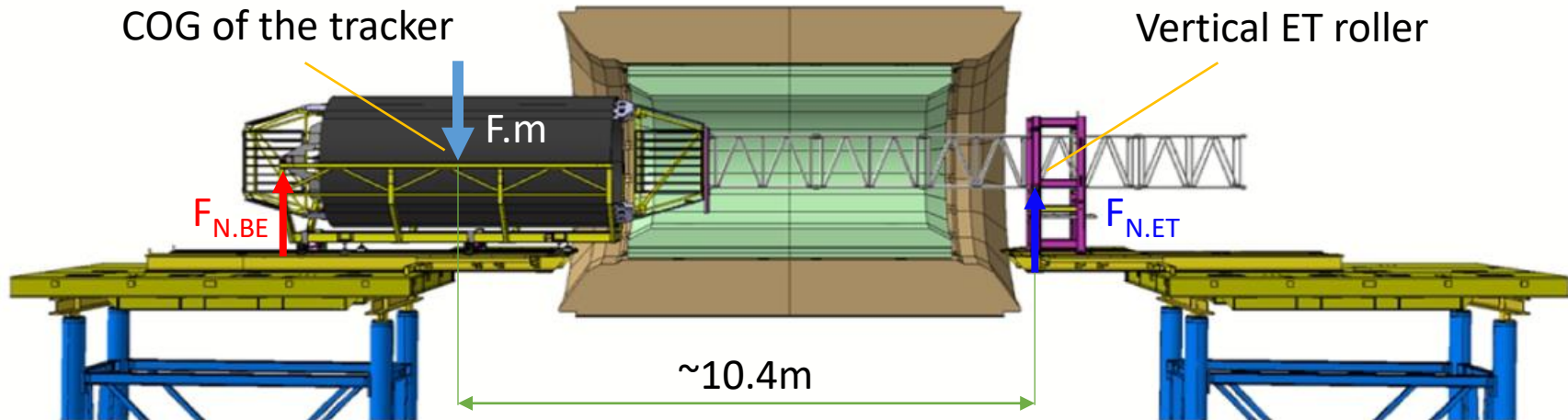
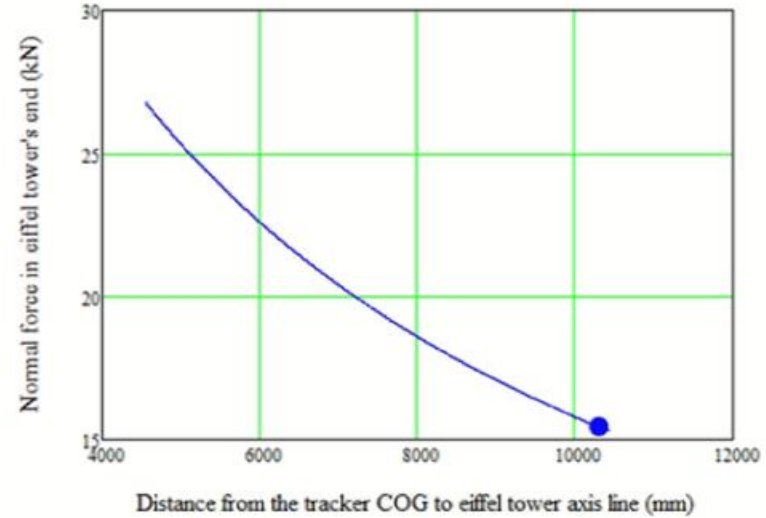


Phase-2 reaction loads

Reaction load on the BE-end

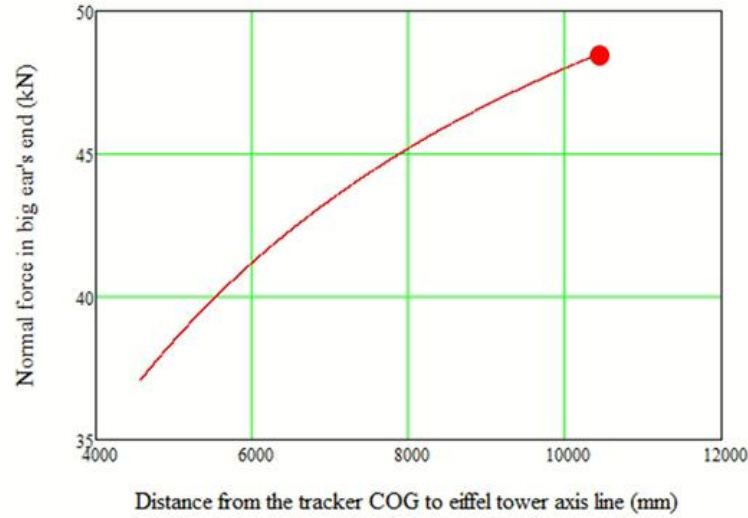


Reaction load on the ET-end

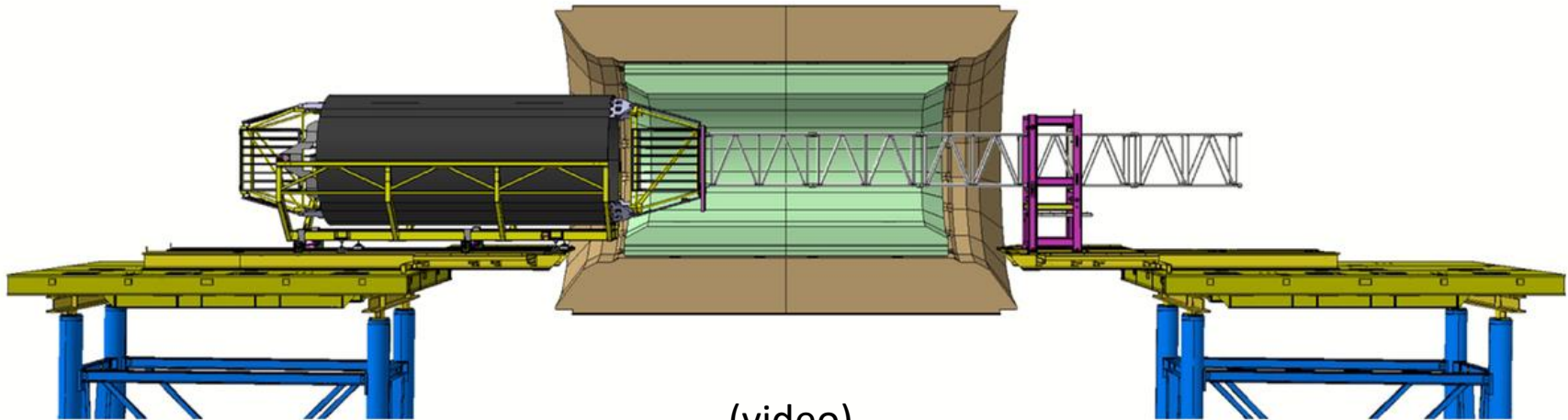
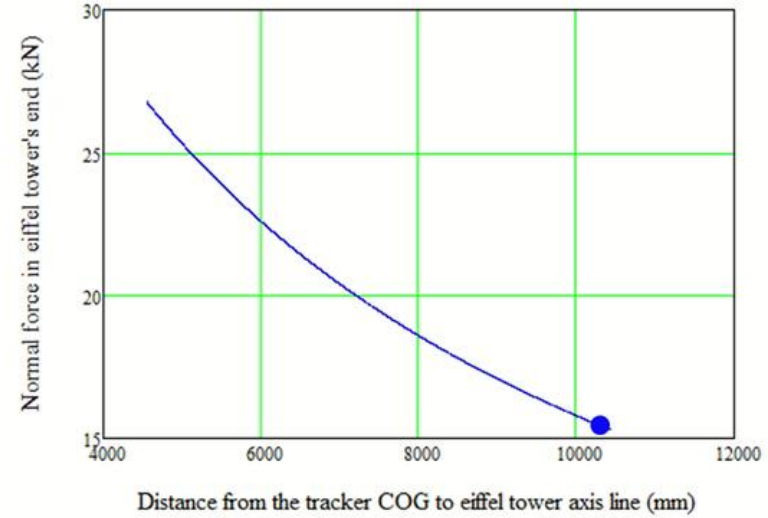


Phase-2 reaction loads

Reaction load on the BE-end

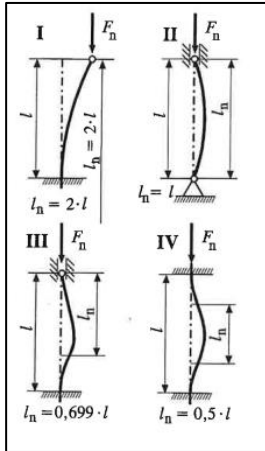


Reaction load on the ET-end

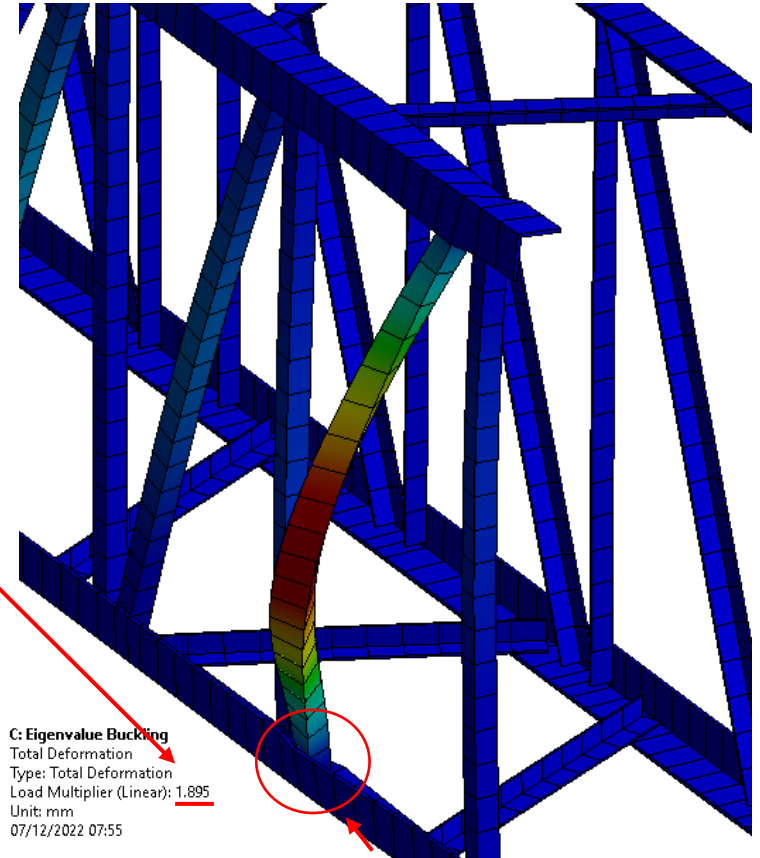


(video)

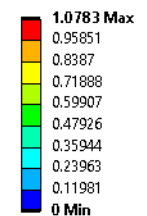
Structural study of the old Eiffel tower



Linear load multiplier is comparable to classical Euler's analysis, which is always non-conservative!



C: Eigenvalue Buckling
 Total Deformation
 Type: Total Deformation
 Load Multiplier (Linear): 1.895
 Unit: mm
 07/12/2022 07:55



Flange stiffness of the longitudinal profile supports to some extent

Critical buckling force

$$F_{n1} := \frac{\pi^2 \cdot E \cdot I_b}{4l^2} = 1.929 \text{ kN}$$

$$F_{n2} := \frac{\pi^2 \cdot E \cdot I_b}{l^2} = 7.717 \text{ kN}$$

$$F_{n3} := 2.046 \frac{\pi^2 \cdot E \cdot I_b}{l^2} = 15.789 \text{ kN}$$

$$F_{n4} := \frac{4 \cdot \pi^2 \cdot E \cdot I_b}{l^2} = 30.868 \text{ kN}$$

Safety factor

$$\frac{F_{n1}}{F_c} = 0.167$$

$$\frac{F_{n2}}{F_c} = 0.67$$

$$\frac{F_{n3}}{F_c} = 1.37$$

$$\frac{F_{n4}}{F_c} = 2.678$$

Buckling modes:

I

II

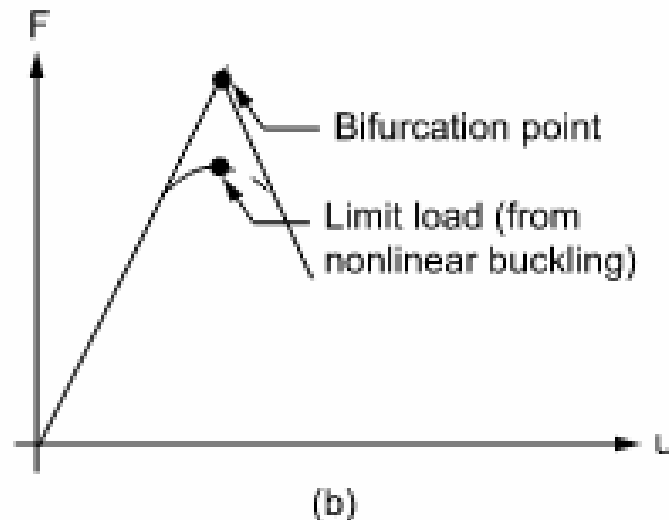
III

IIII



Horizontal Eiffel tower - stability

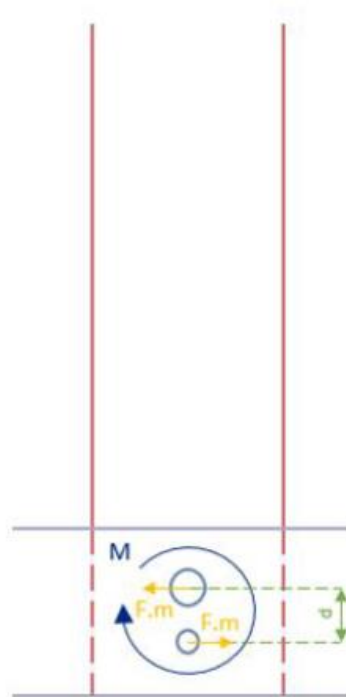
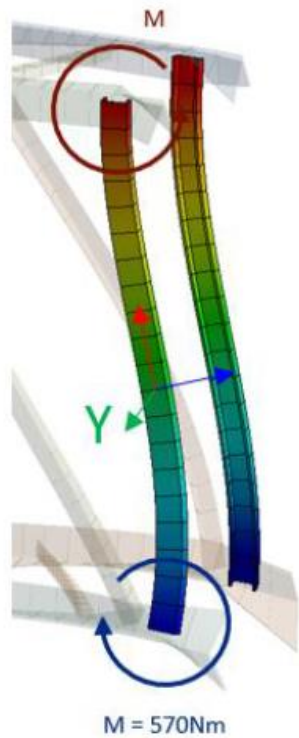
An **Eigenvalue Buckling** analysis predicts the theoretical buckling strength of an ideal elastic structure. This method corresponds to the textbook approach to an elastic buckling analysis: for instance, an **eigenvalue buckling** analysis of a column matches the classical Euler solution. However, imperfections and nonlinearities prevent most real-world structures from achieving their theoretical elastic buckling strength. Therefore, an **Eigenvalue Buckling** analysis often yields quick but non-conservative results.



(Ansys Help 2022 R1)



Structural study of the old Eiffel tower



$$d := 31.5\text{mm}$$

distance between bolts

$$M := 570\text{N}\cdot\text{m}$$

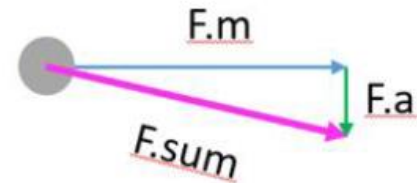
moment through the joint

$$F_M := \frac{M}{d} = 18.095\text{kN}$$

shear component by moment

$$F_a := 4450\text{N}$$

shear component by axial force



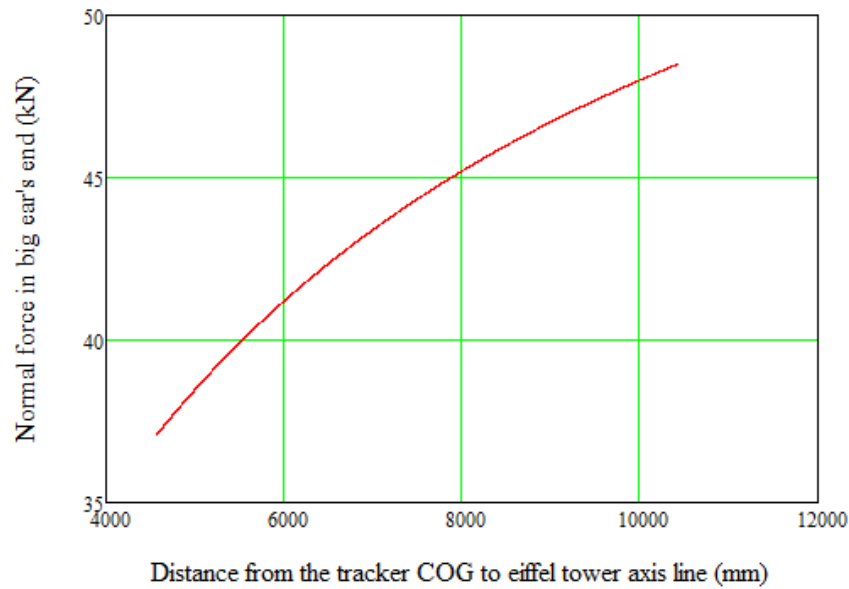
$$F_{\text{sum}} := \sqrt{F_M^2 + F_a^2} = \underline{18.634\text{kN}}$$

sum shear force on the bolt

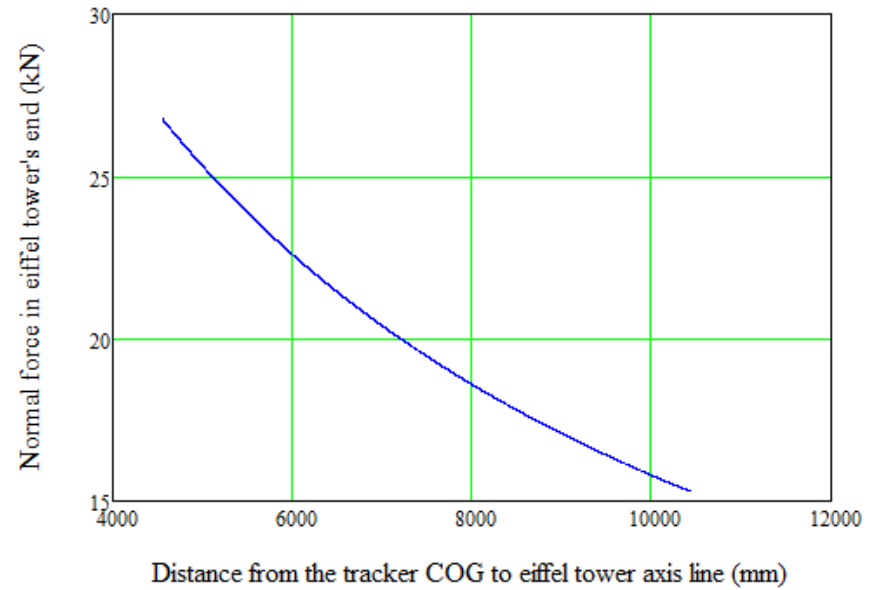


Reaction loads

Reaction load on the BE-end



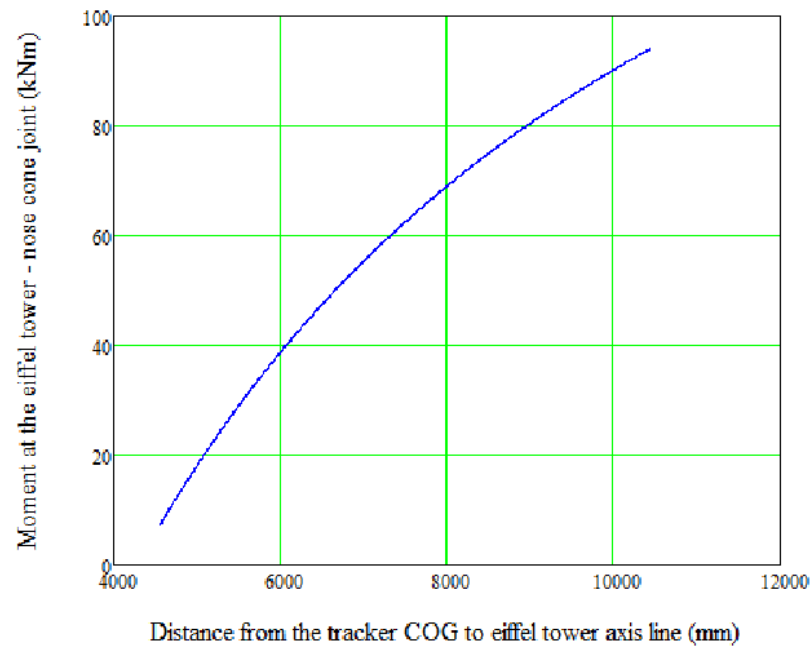
Reaction load on the ET-end





Appendix F: The moment in the inner end of the truss during the installation travel

$$M_{\text{light_end}}(b) := \frac{\overbrace{m_{\text{test}} \cdot g \cdot a}^{\text{F}}}{a + b} \cdot \underbrace{(b - 4283\text{mm})}_{\text{I}}$$



The moment on the inner end of the truss decreases along the installation travel.



Extraction of the current- vs. installation of the Phase-2 tracker

- For the extraction of the current tracker, some elements of the tooling are going to be already replaced with new designs
 - Mainly the roller assemblies on the ears, and Eiffel tower push/pull mechanism
- Ideally, most of the installation tooling could be re-used with the phase-2 -tracker with minor modifications due to changed interfaces
- The Phase-2 –tracker will be heavier than the current tracker
 - Current tracker: 5350kg (as measured, minus the mass of cradle)
 - Phase-2 tracker: ~6500kg ([A. Filenius & G. Reales 02/03/2021](#))
- To avoid duplicate work, all the dimensioning and validation of the new designs are done already with phase-2 boundary conditions

Summary

	Weight	Uncertainty	
TK in simulation	3963		
cooling fluid	-153	-53	77
pixel tube and beam pipe supports	45		
services from TST to PP1	670	-10	20
installation and packaging stuff	1648	-100	100
Total	6173	-163	197

So far compares well with the measurements of 6330 kg and 6350 kg (3% difference)

1.6 tons still to be verified by direct measurements

Final word in a few weeks

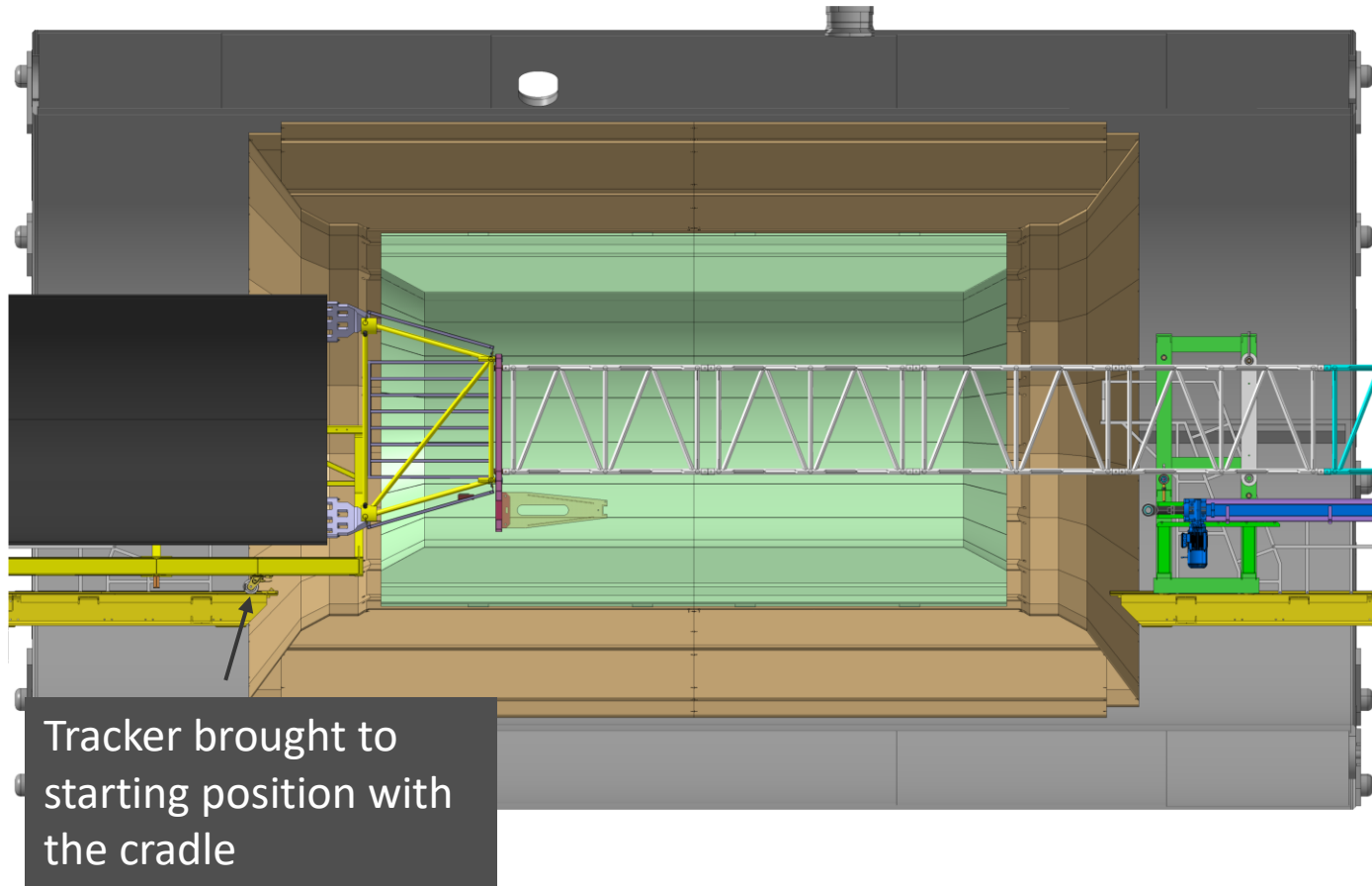
(2008, current tracker weighing)

Group	Item	Mass [kg]	Group mass (kg)
Detector assemblies	TBPS flat	40	3274
	TBPS tilted	120	
	TB2S	500	
	TEDD	800	
	BTL Tray	1814	
Detector mechanics	BTST	316	793
	TK rails	37	
	Banana	9	
	BTL I-Beams	70	
	Service support ribs	18	
	Electrical PPO Frames	19	
	TBPS/TB2S periphery seals	11	
	Bulkhead	220	
	ITST Small section	16	
	ITST Large section	57	
Service sup. Mech. On TB2S wheel	20		
Integration tooling	Steel nose cone	164	630
	Steel nose cone link	67	
	Aluminum nose cone	76	
	Aluminum nose cone link	34	
	Small ears	31	
	Big ears	73	
OT PPO-PP1 services	Hor. Eiffel Tower	185	1807
	BTL PPO-PP1 Power	430	
	BTL PPO-PP1 OF	12	
	TBPS-TB2S PPO-PP1 Power	710	
	TBPS-TB2S PPO-PP1 OF	51	
	TEDD PPO-PP1 Power	564	
TEDD PPO-PP1 OF	40		
TOTAL MASS		6505	
TOTAL MASS, WITH 1.2 SAFETY FACTOR		7805	

(2021, phase-2 tracker estimate)



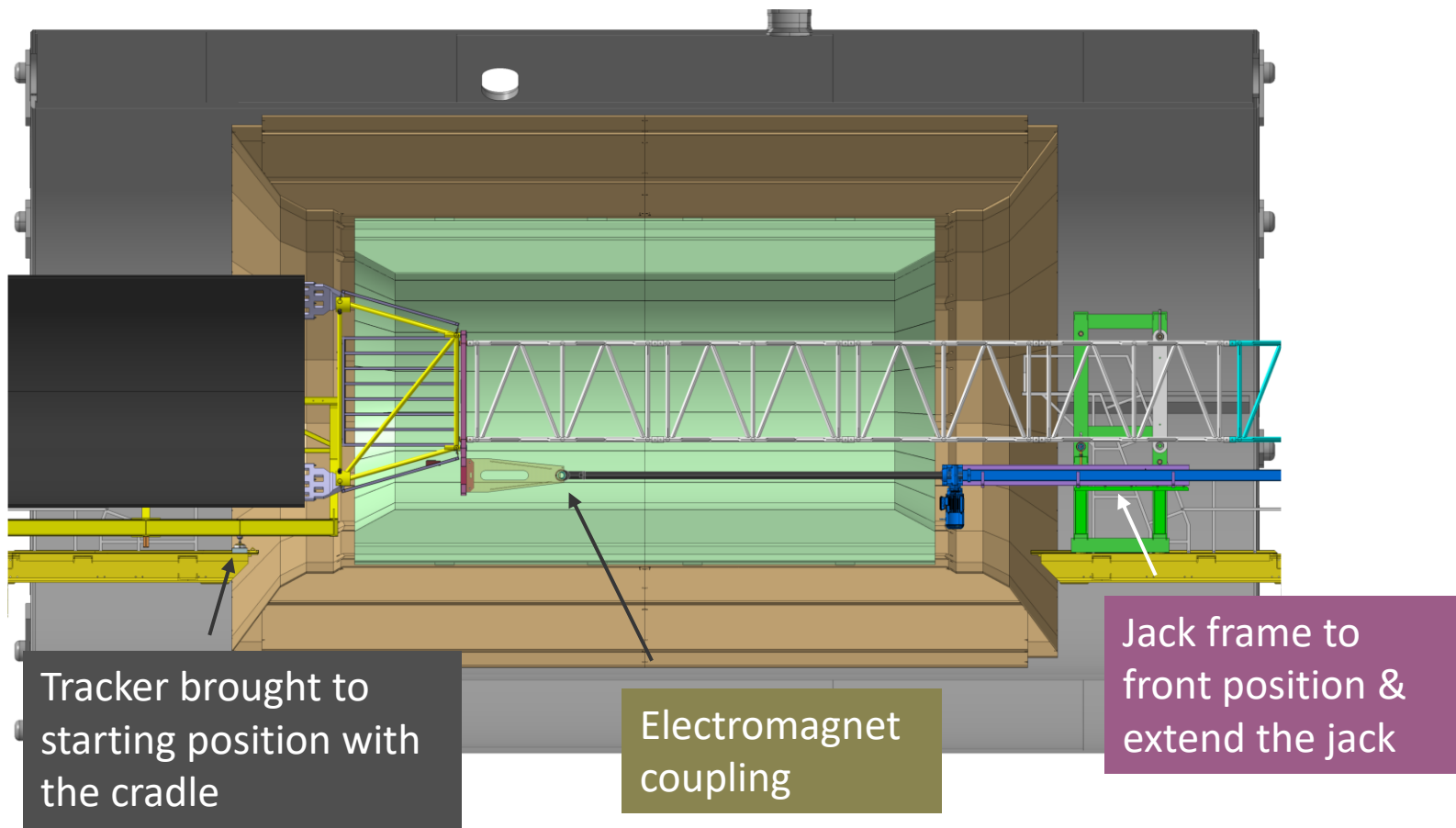
New vertical eiffel tower



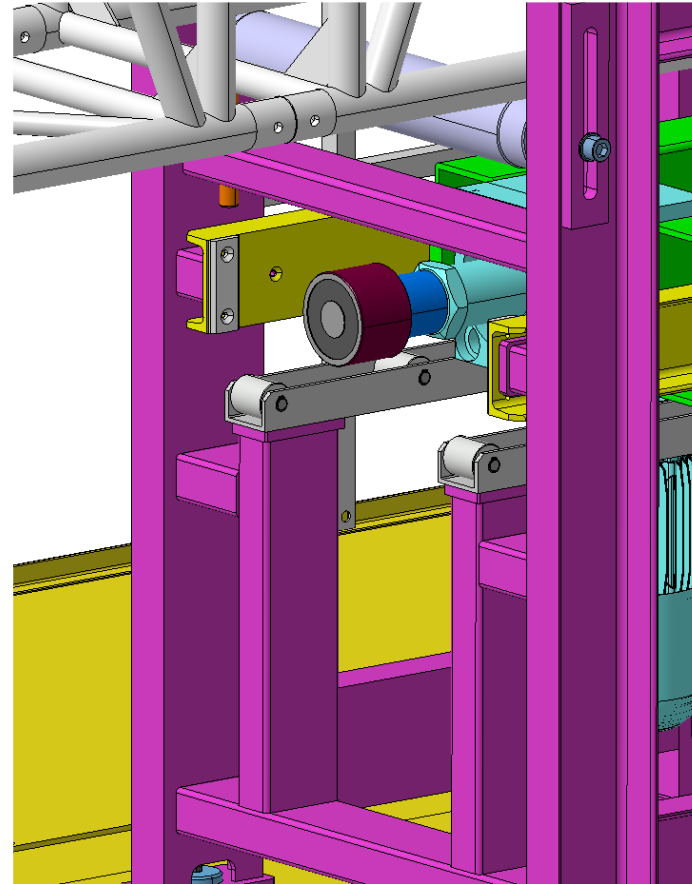
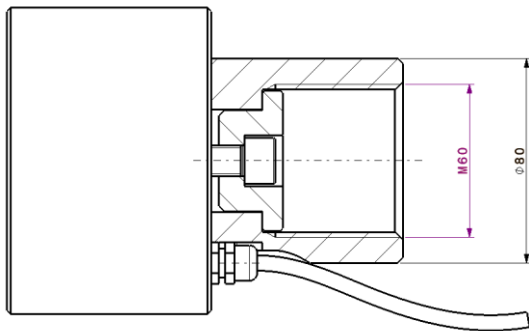
Tracker brought to starting position with the cradle



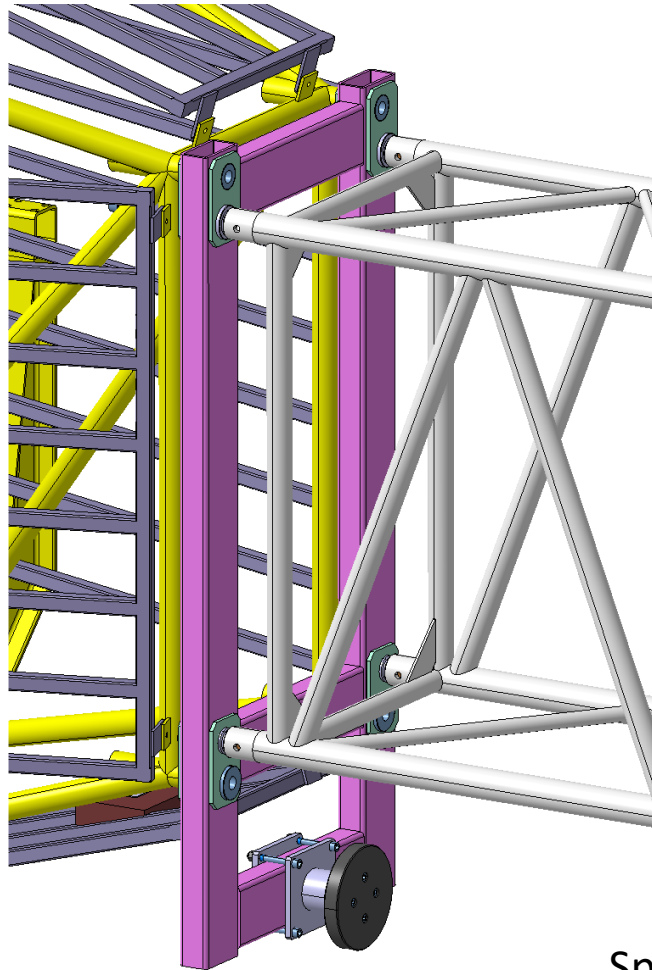
New vertical eiffel tower



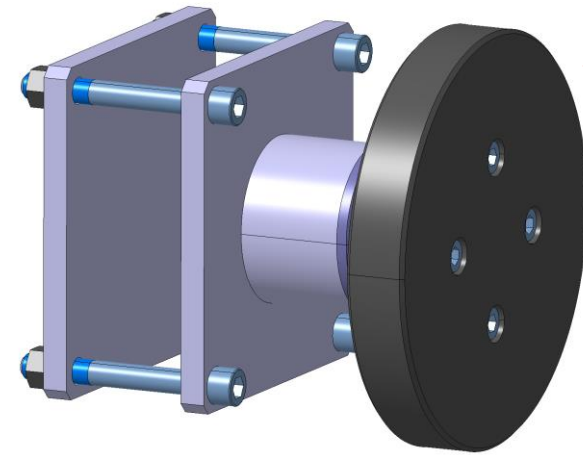
New vertical eiffel tower



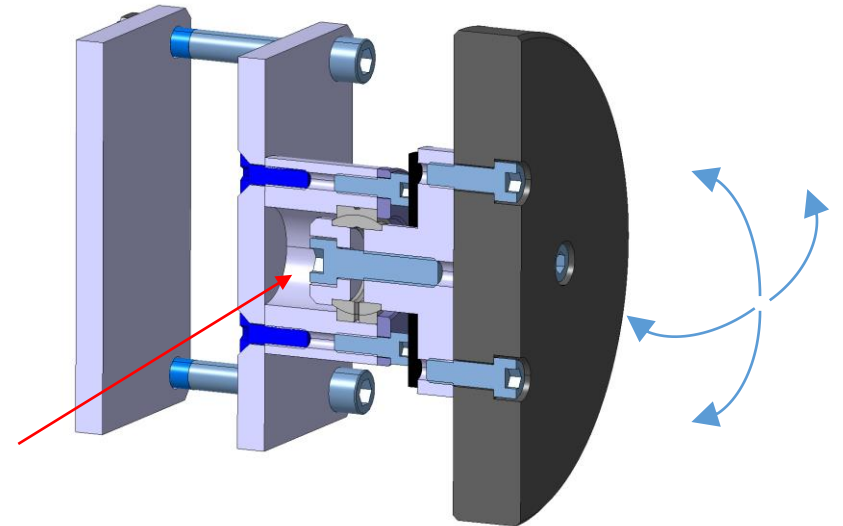
New vertical eiffel tower



Spherical joint

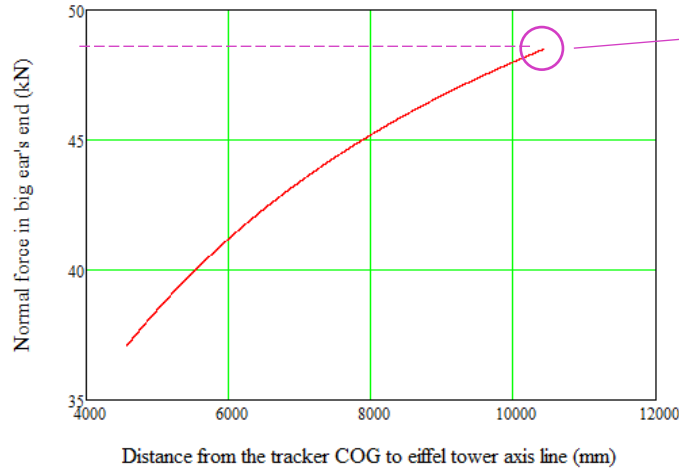


Steel plate for magnet



Beginning of installation

Reaction load on the BE-end



$N.BE.fully_out.nominal = 48.5kN$

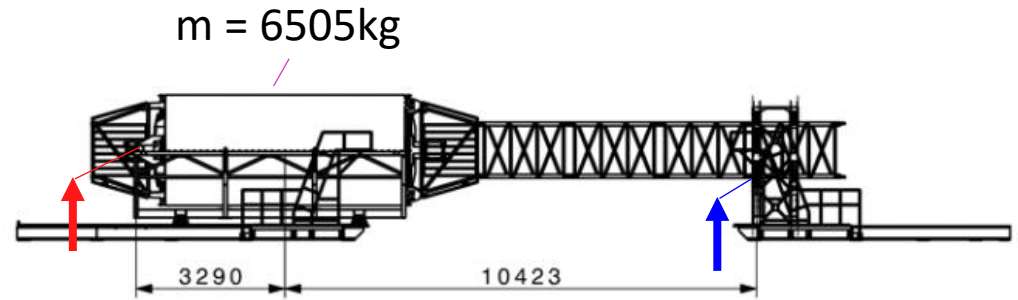
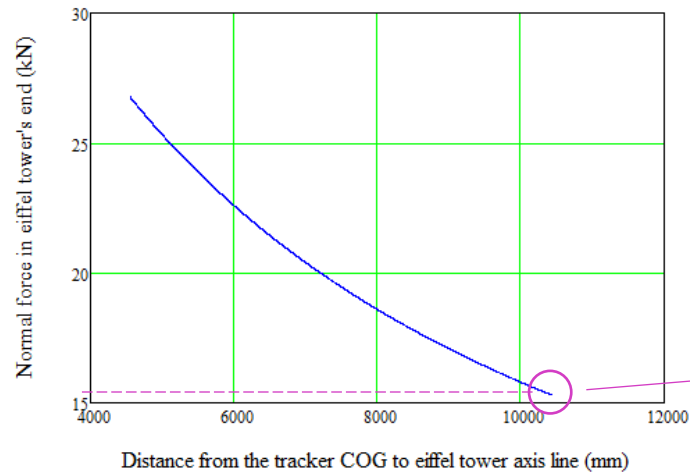


Figure 1 Beginning of installation, distances from tracker COG to rolling axles

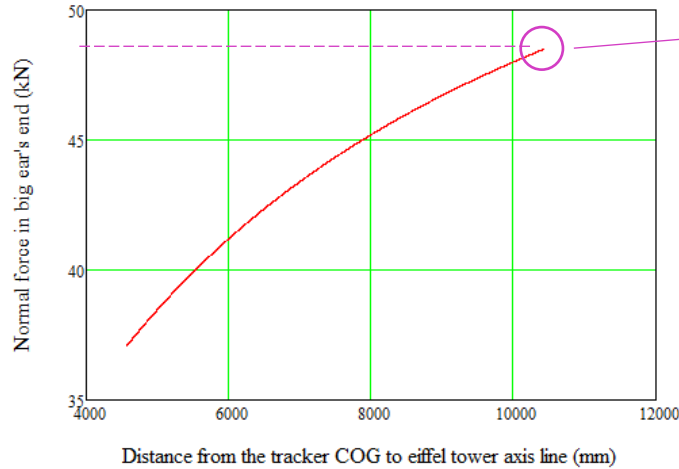
Reaction load on the ET-end



$N.ET.fully_out.nominal = 15.3kN$

Beginning of installation

Reaction load on the BE-end



$N.BE.fully_out.nominal = 48.5kN$

$m = 6505kg$

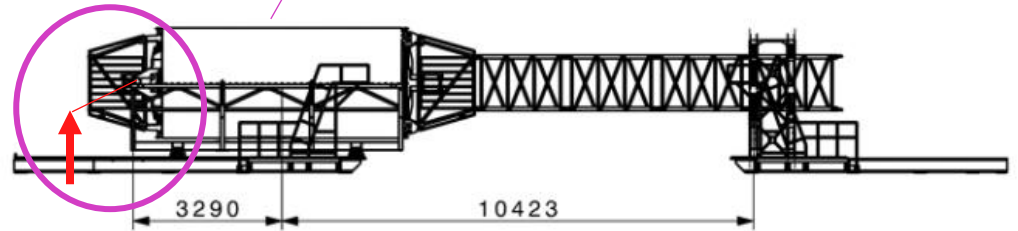
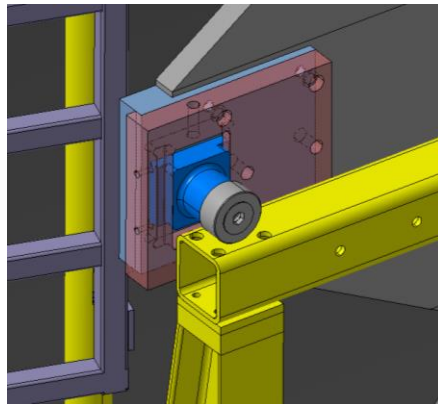


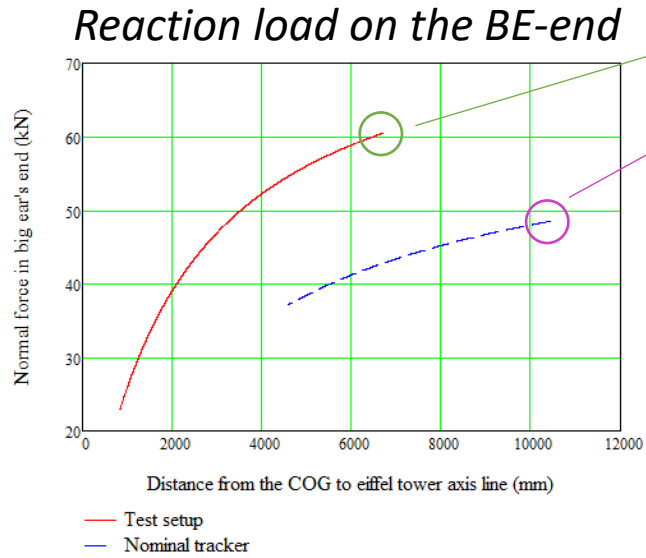
Figure 1 Beginning of installation, distances from tracker COG to rolling axes

Reaction load on the ET-end



- The beginning of installation is:
 - Worst case for the BE-mechanism
 - Heaviest loading for the cradle

Beginning of installation (scenario #2)



$N.BE.fully_out.test = 60.5kN$
 $N.BE.fully_out.nominal = 48.5kN$
+24.4%

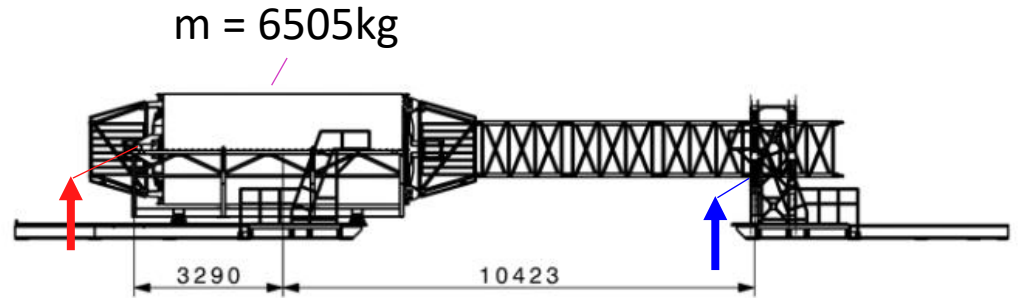
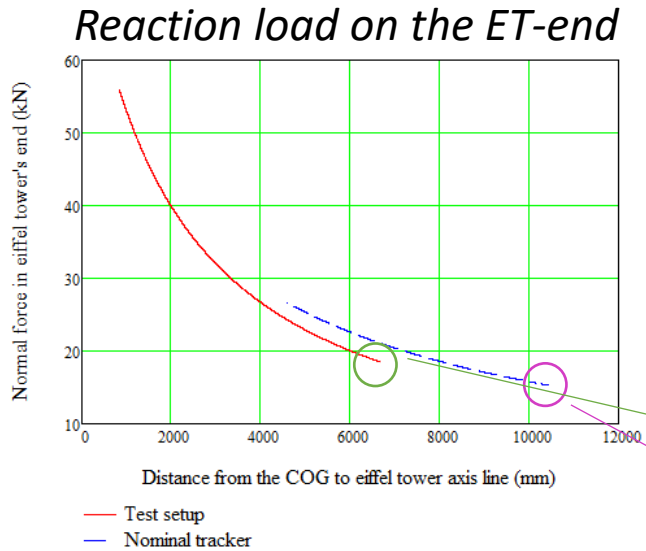


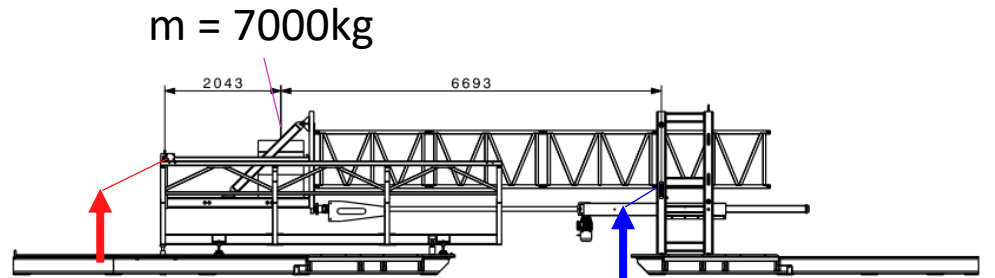
Figure 1 Beginning of installation, distances from tracker COG to rolling axes



$m = 7000kg$

2043 6693

$N.ET.fully_out.test = 18.5kN$
 $N.ET.fully_out.nominal = 15.3kN$
+21.6%



End of installation (scenario #1)

Reaction load on the ET-end

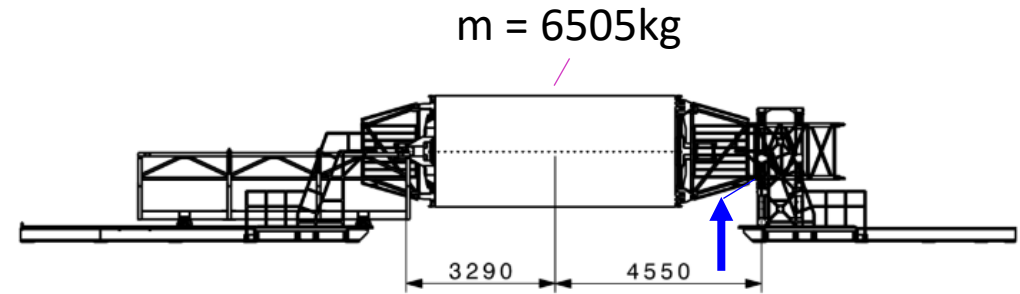
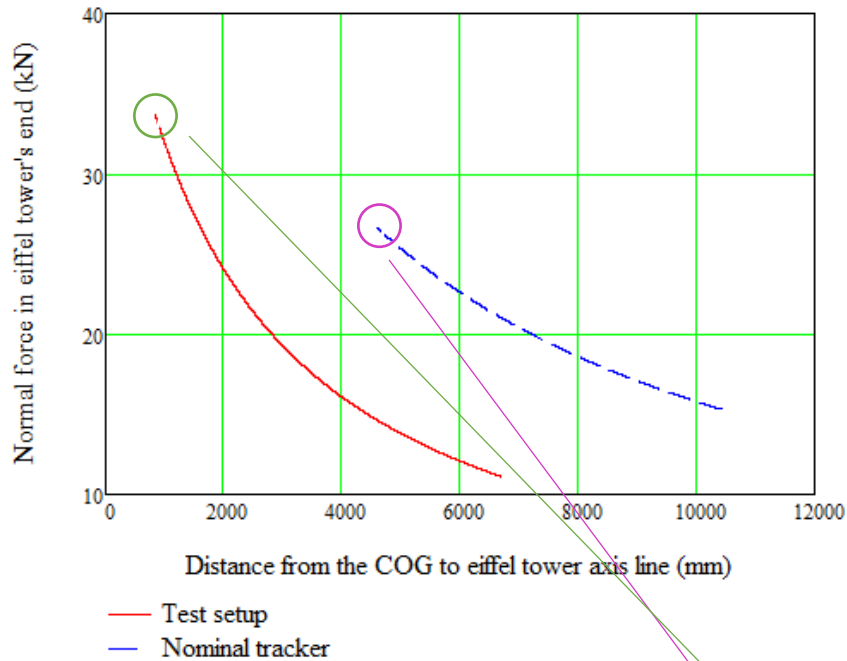
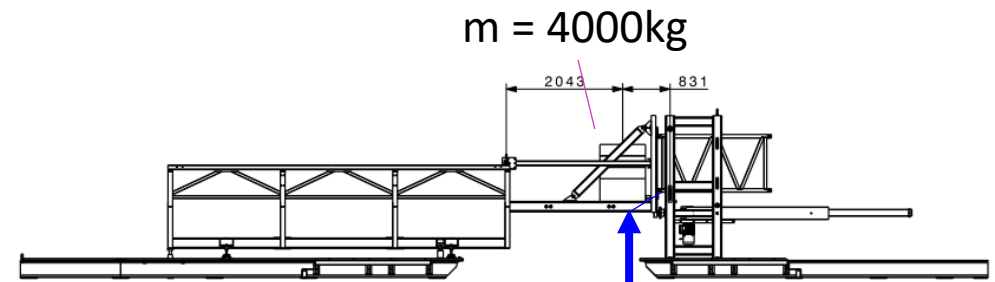


Figure 2 End of installation, distances from tracker COG to rolling axles

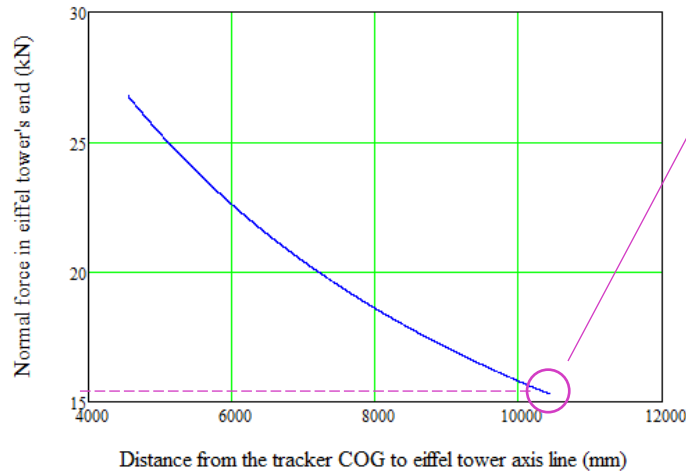


$N_{ET,fully_in_test} = 33.6\text{kN}$
 $N_{ET,fully_in_nominal} = 26.7\text{kN}$

+20.5%

Beginning of installation

Reaction load on the ET-end



$N_{ET,fully_out,nominal} = 15.3\text{kN}$

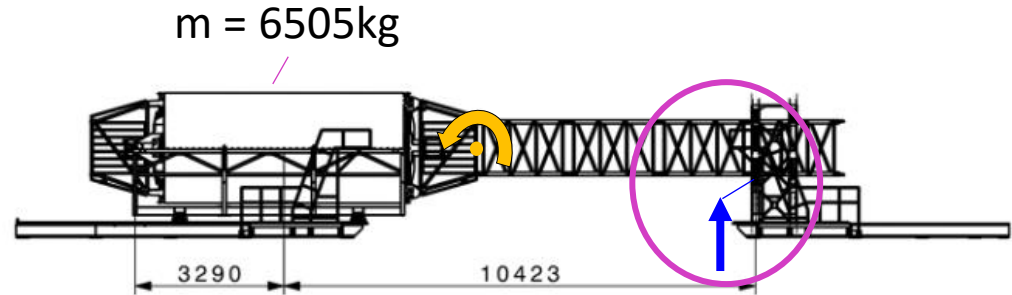
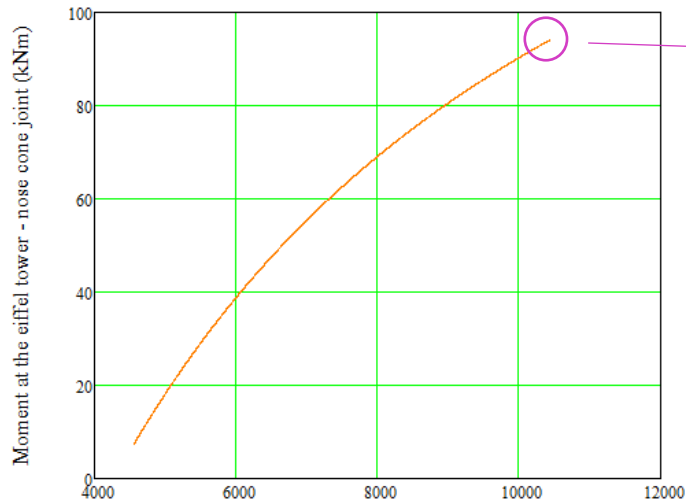


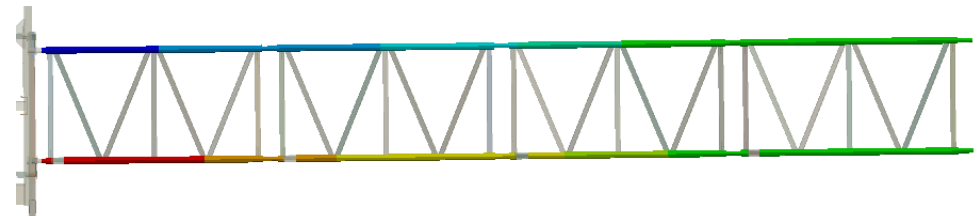
Figure 1 Beginning of installation, distances from tracker COG to rolling axles

Reaction moment on the ET-NS-joint



$M_{ET,fully_out,nominal} = 94.0\text{kNm}$


- The beginning of installation is:
 - Worst case for the horizontal Eiffel tower in stress and in deflection



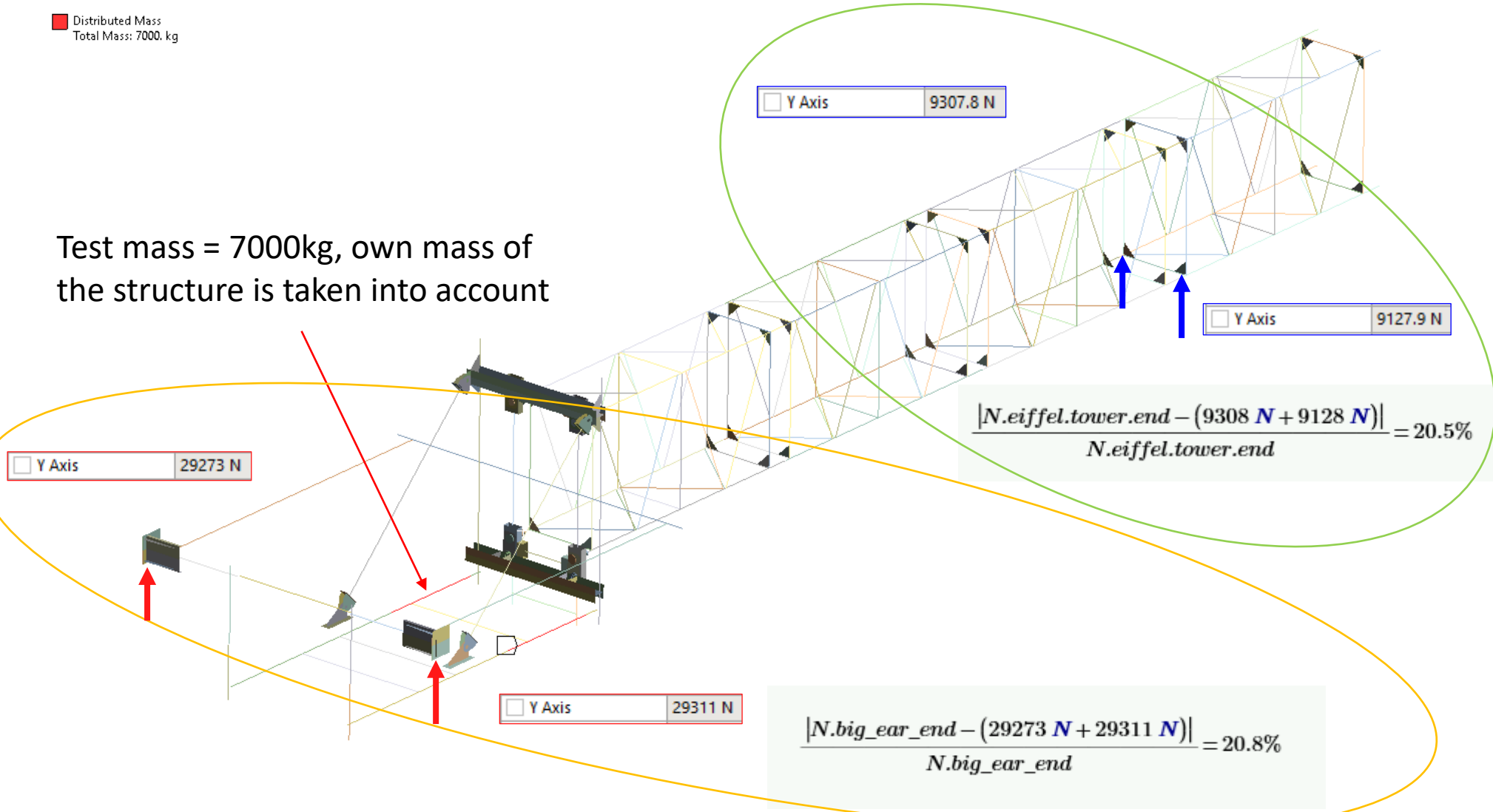


Test scenario #2, fully extended

Distributed Mass

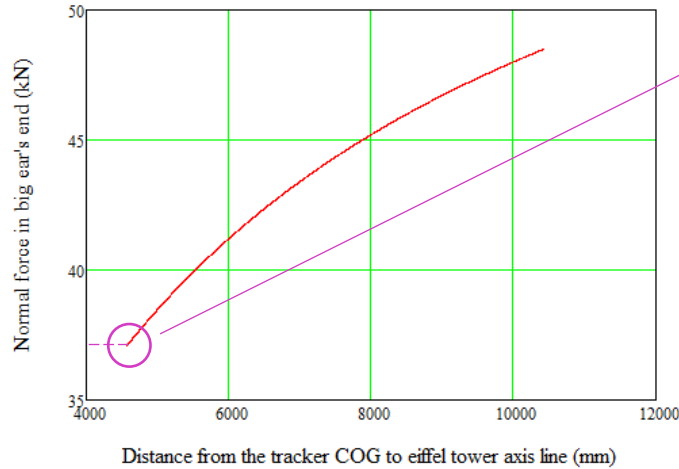
 Distributed Mass
Total Mass: 7000. kg

Test mass = 7000kg, own mass of the structure is taken into account



End of installation

Reaction load on the BE-end



$N.BE.fully_in.nominal = 37.0kN$

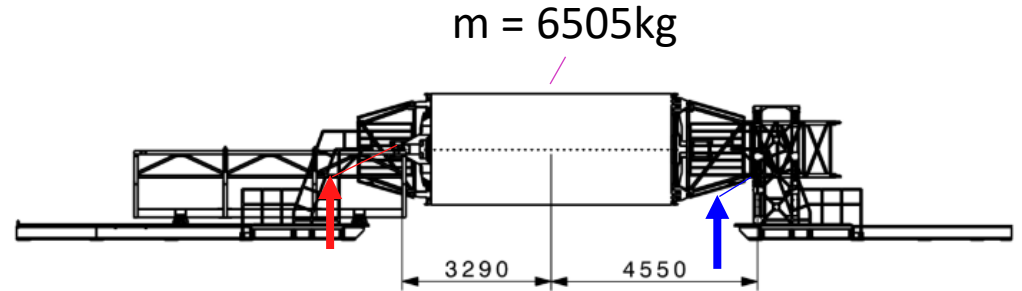
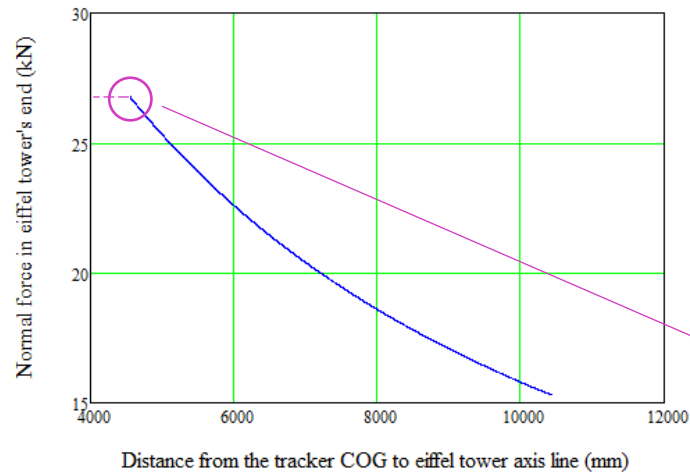


Figure 2 End of installation, distances from tracker COG to rolling axles

Reaction load on the ET-end



$N.ET.fully_in.nominal = 26.8kN$

End of installation

Reaction load on the ET-end

$N_{ET,fully_in,nominal} = 26.8\text{kN}$

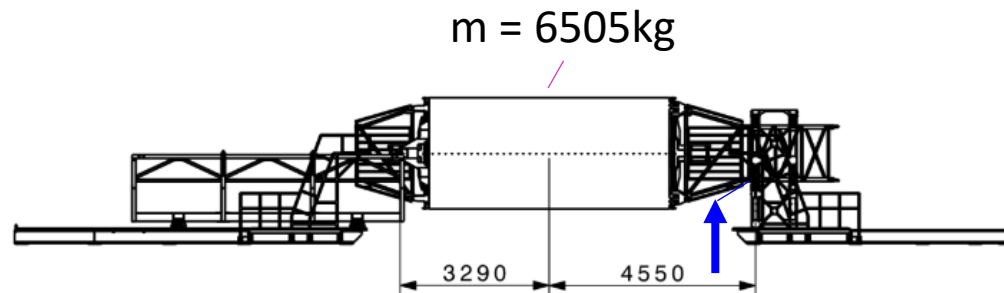
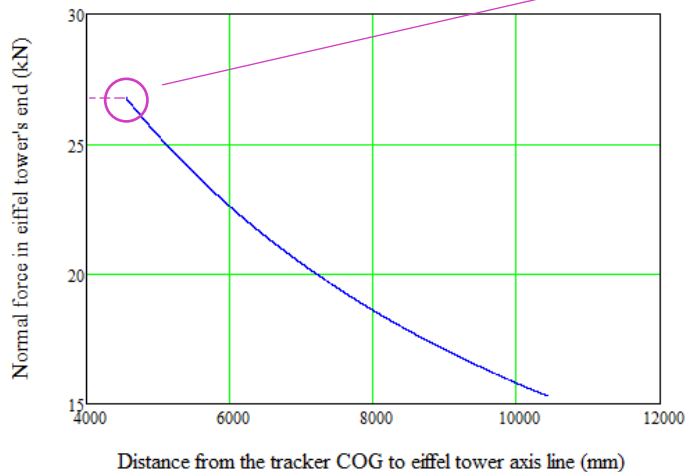
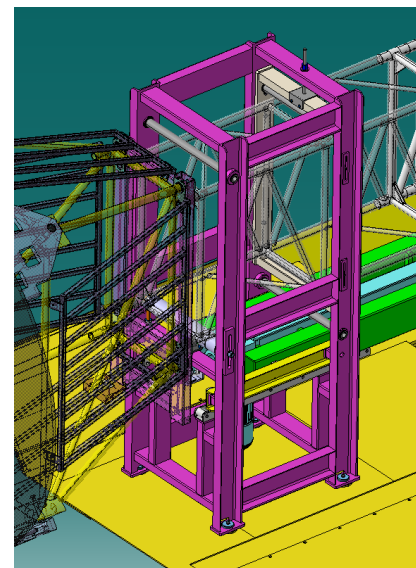


Figure 2 End of installation, distances from tracker COG to rolling axles


- The end of installation is:
 - Worst case for the vertical Eiffel tower structure, and its height adjustment mechanism



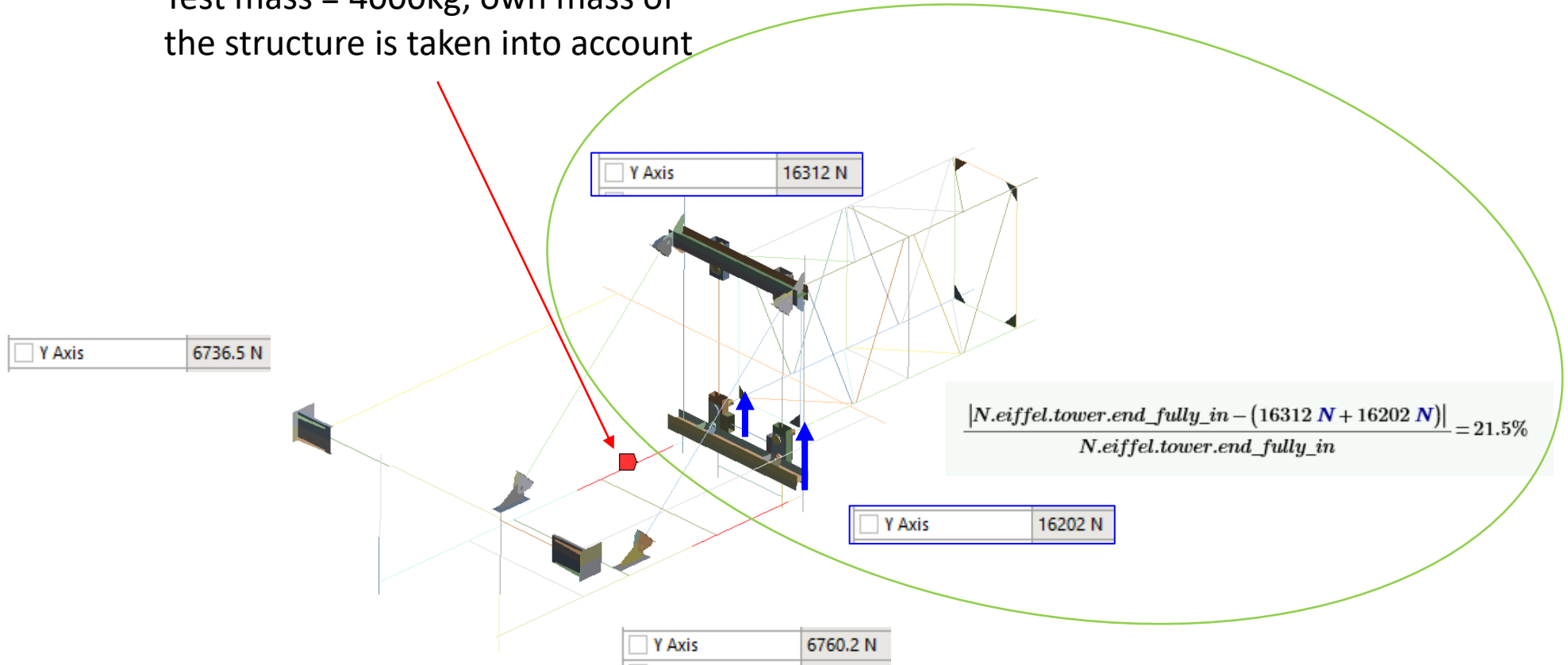


Test scenario #1, fully in

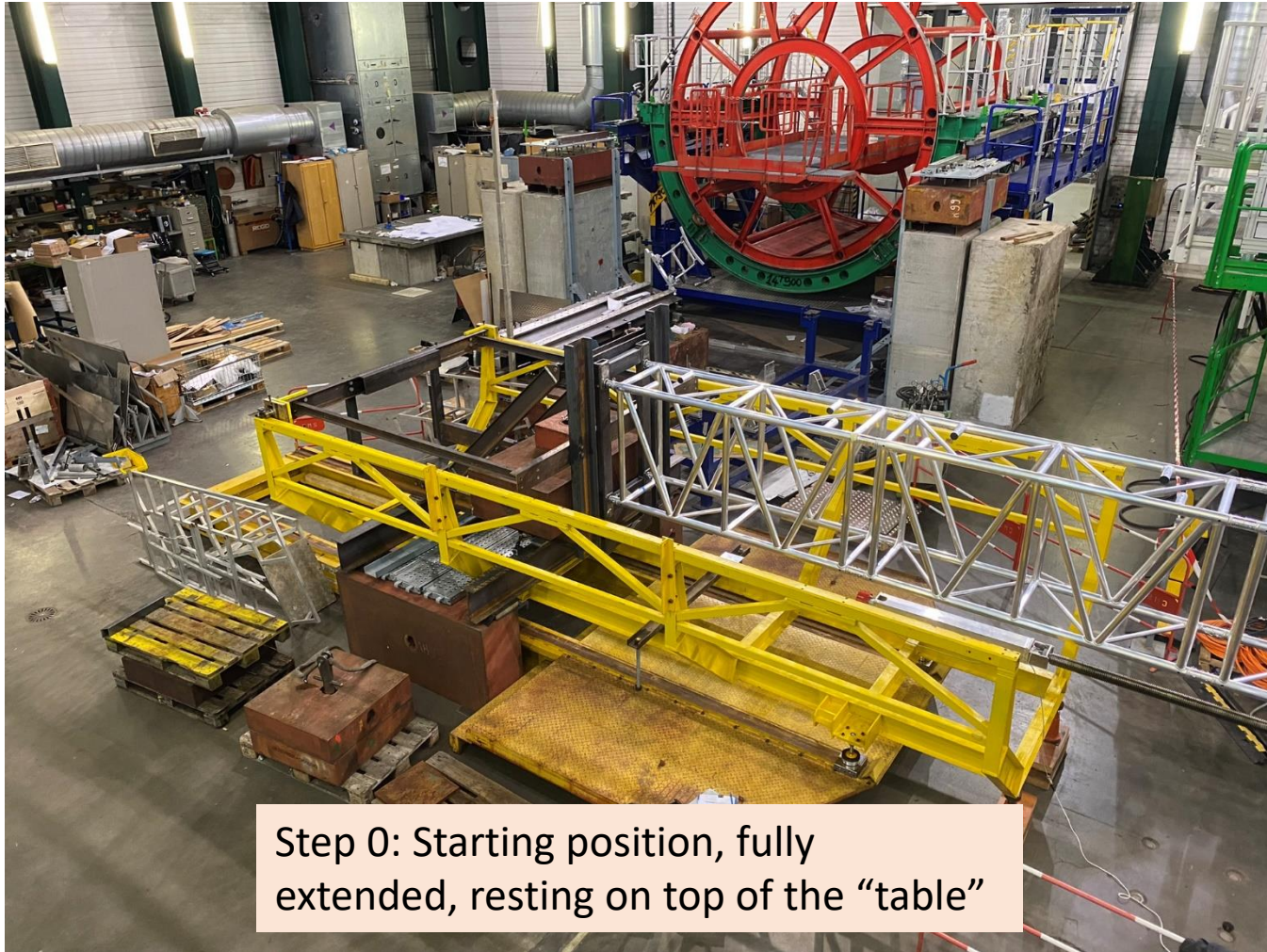
Distributed Mass

 Distributed Mass
Total Mass: 4000. kg

Test mass = 4000kg, own mass of the structure is taken into account

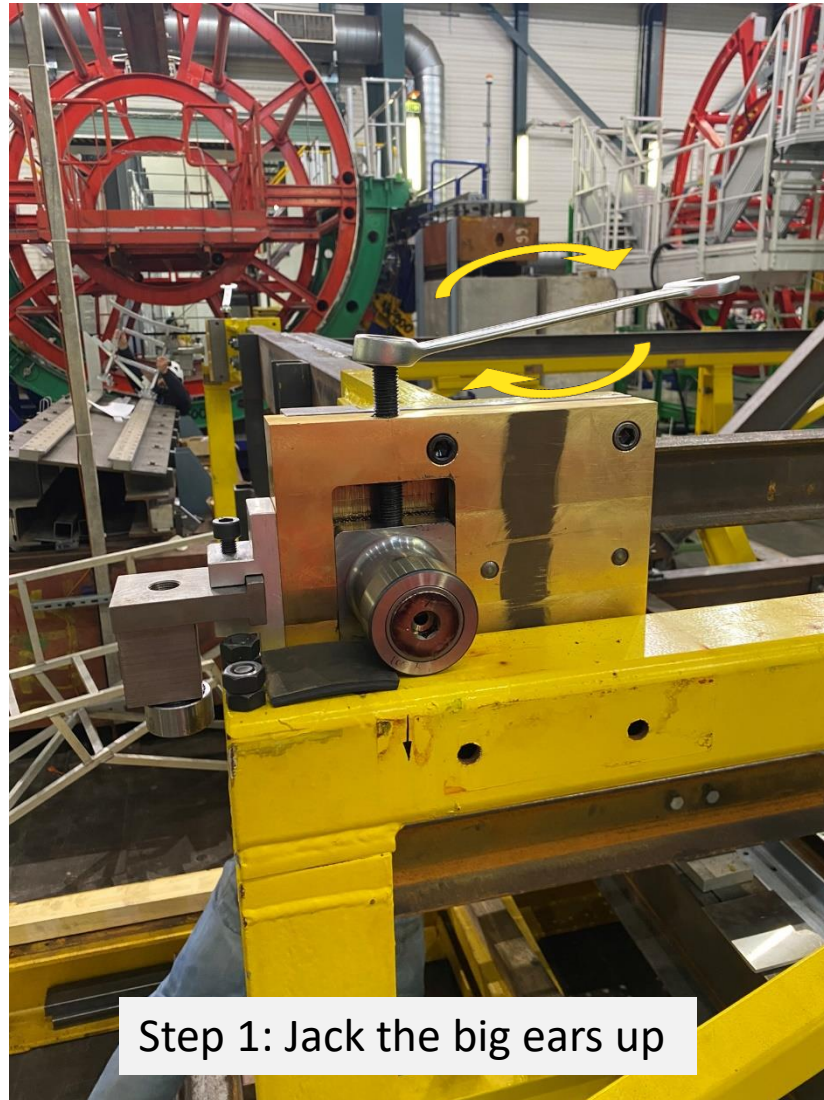


Test, scenario #1



Step 0: Starting position, fully extended, resting on top of the “table”

Test, scenario #1



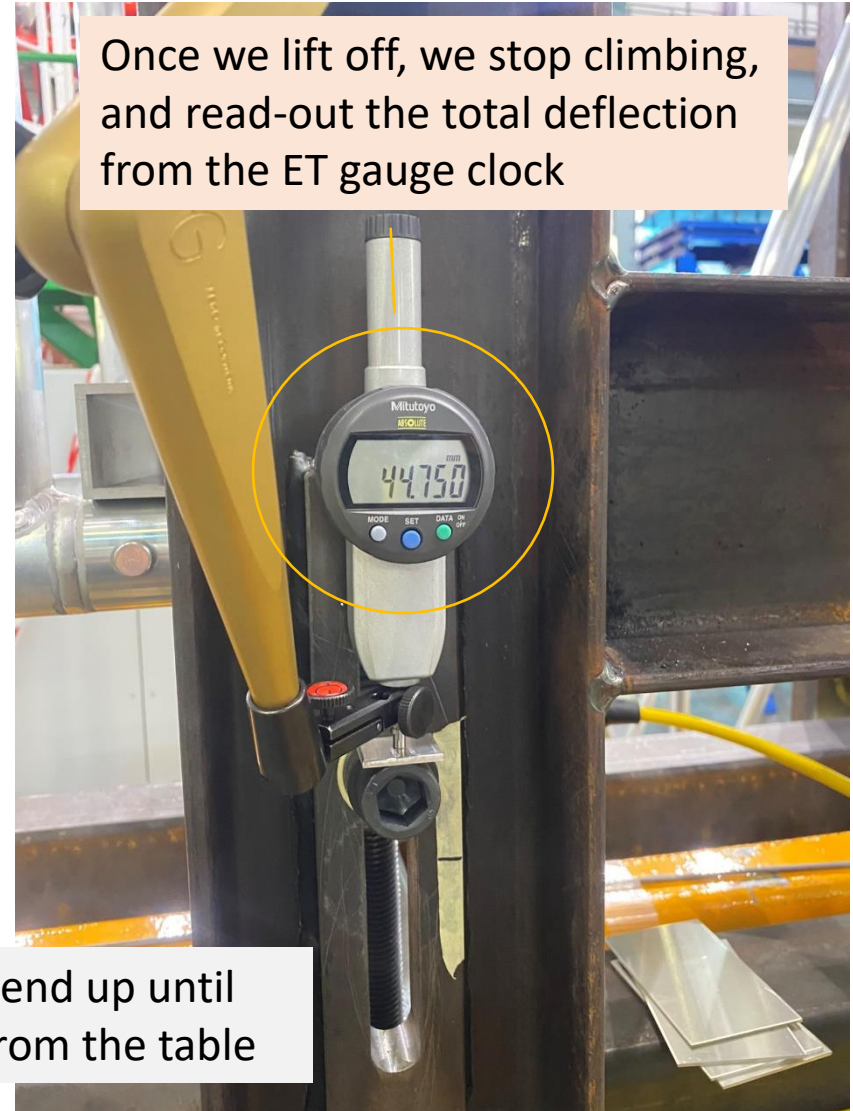
Step 1: Jack the big ears up

Test, scenario #1

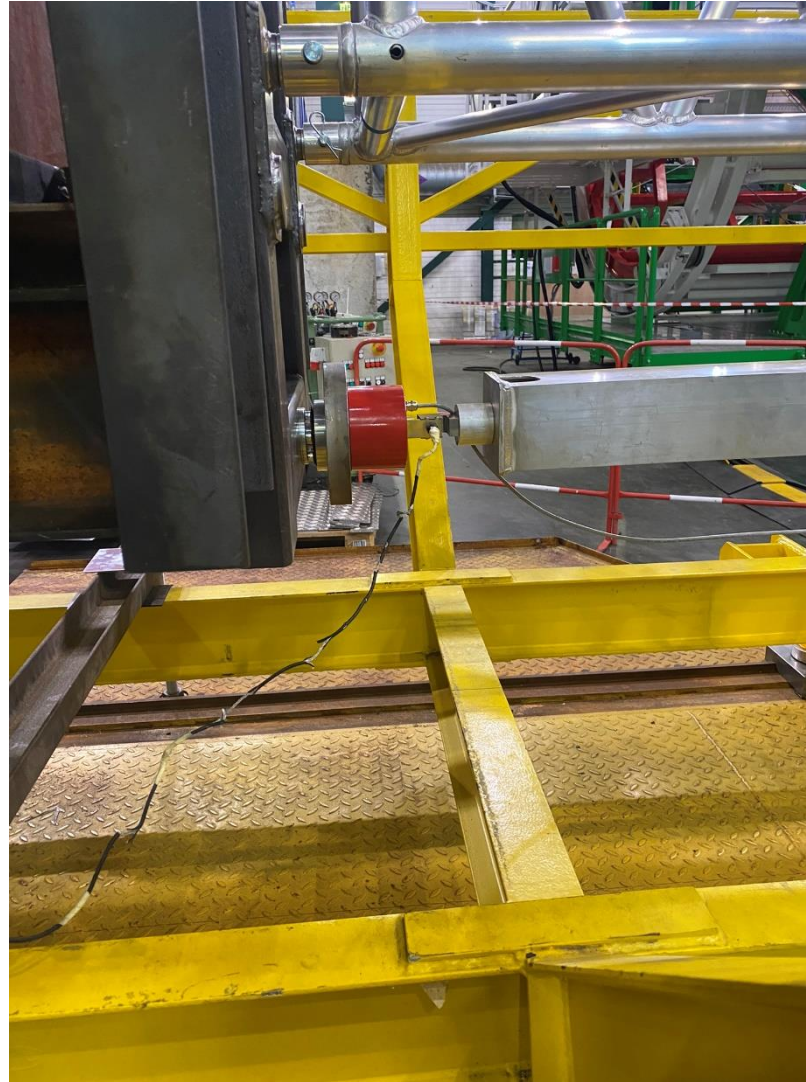


Step 2: Jack the eiffel tower's end up

Test, scenario #1



Test, scenario #1



Step 3: extending the actuator, connecting the magnet

Test, scenario #1



Step 3: extending the actuator, connecting the magnet

Load cell in between to measure push/pull force

Test, scenario #1

(video)



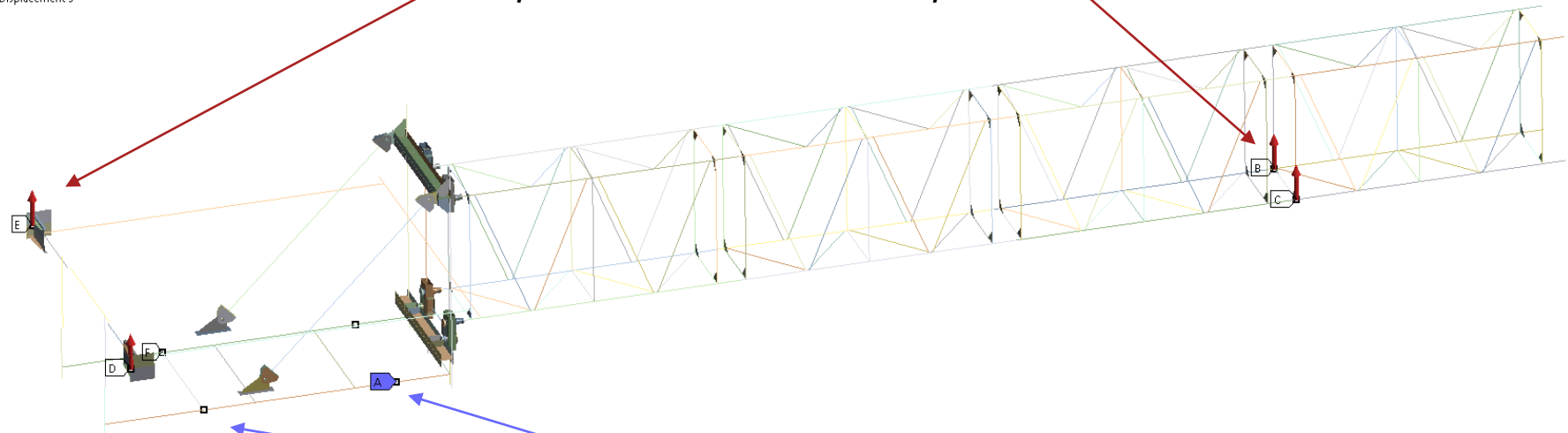
Test, scenario #2

“Inverse” boundary conditions:

BS: Copy of Horizontal eiffel tower (Duratruss), solid pins, Nose cone adapter, Flipped truss, corrected cross-section
Static Structural
Time: 1. s

- A** Fixed Support 3
- B** Force 3: 9307.8 N
- C** Force 4: 9127.9 N
- D** Force 5: 29311 N
- E** Force 6: 29273 N
- F** Displacement 5

Reaction loads from 7000kg load input to the roller contact points



Fixed displacements applied to the table contact points

Test, scenario #2



(video)



Test, scenario #2

Nominal phase-2, end of installation scenario

$$m_{nom} := 6505\text{kg}$$

$$a_{ww} := 3290\text{mm}$$

distance from COG to heavy end wheel axis

$$b_{ww} := 10423\text{mm}, 10422\text{mm}.. 4550\text{mm}$$

distance from COG to light end wheel axis

$$N_{light_end_defined} := \frac{m_{nom} \cdot g \cdot a}{a + 4550\text{mm}} = 26.77\text{ kN}$$

Normal force in eiffel tower's end, end of installation

$$N_{test_target} := N_{light_end_defined} \cdot 1.2 = \underline{32.124\text{ kN}}$$

Target normal force in test, +20%

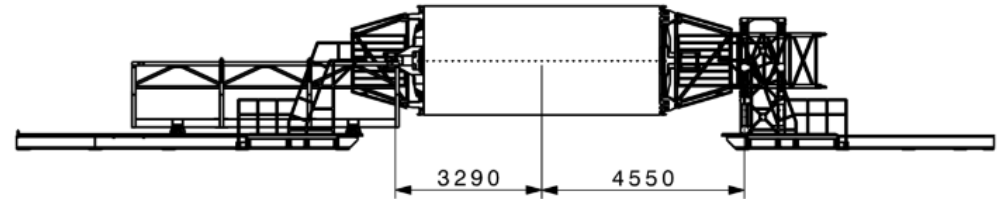


Figure 2 End of installation, distances from tracker COG to rolling axles

Scenario #2, searching for equal reaction

$$m_{test} := 8047\text{kg}$$

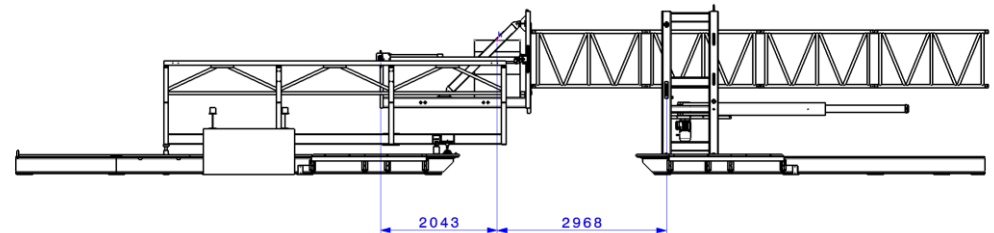
$$a_{test} := 2043\text{mm}$$

distance from COG to heavy end wheel axis

$$b_{test} := 2968\text{mm}$$

distance from COG to light end wheel axis

$$N_{light_end_test_defined} := \frac{m_{test} \cdot g \cdot a_{test}}{a_{test} + b_{test}} = \underline{32.174\text{ kN}}$$



Test cage, Scenario #2 furthest point in