

Design of a Suspension System for Cold-mass on Rotating Gantry for Medical Applications

LUCA PIACENTINI

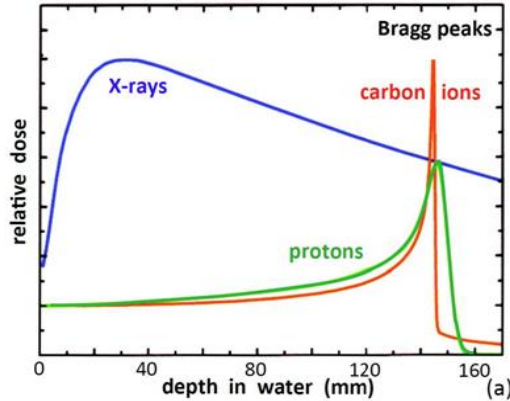


This project has received funding from the European Union's Horizon 2020 research and innovation programme under grant agreement No 101008548

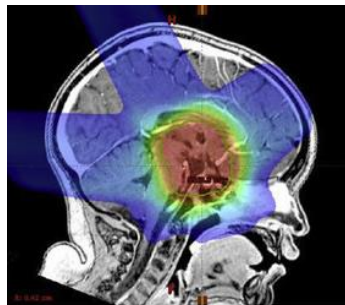
Why Hadron Therapy and Gantries

Introduction

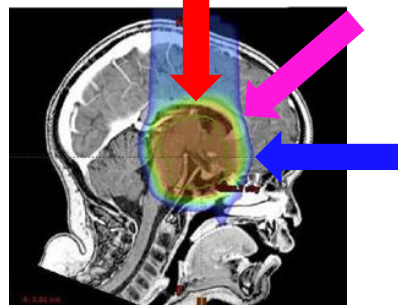
- Method
- Models
- Results



Bragg Peak: hadrons deposit energy @ specific depth, depending on the beam energy



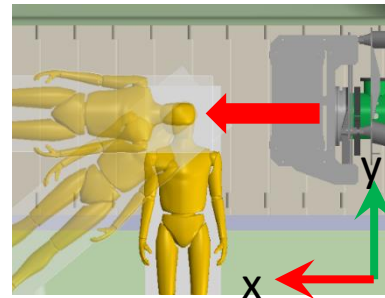
photons IMRT



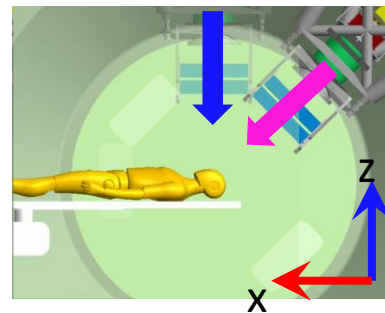
protons

<https://doi.org/10.1016/j.ijrobp.2016.06.2446>

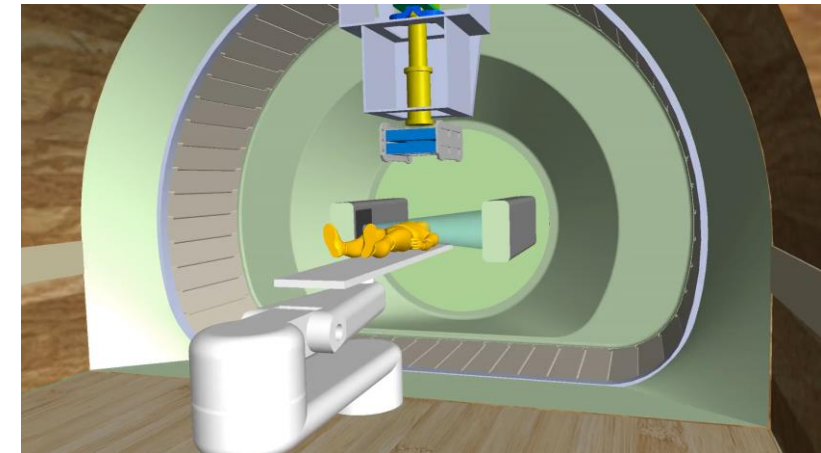
Fixed beam (XY)



Gantry (XZ)



Gantry

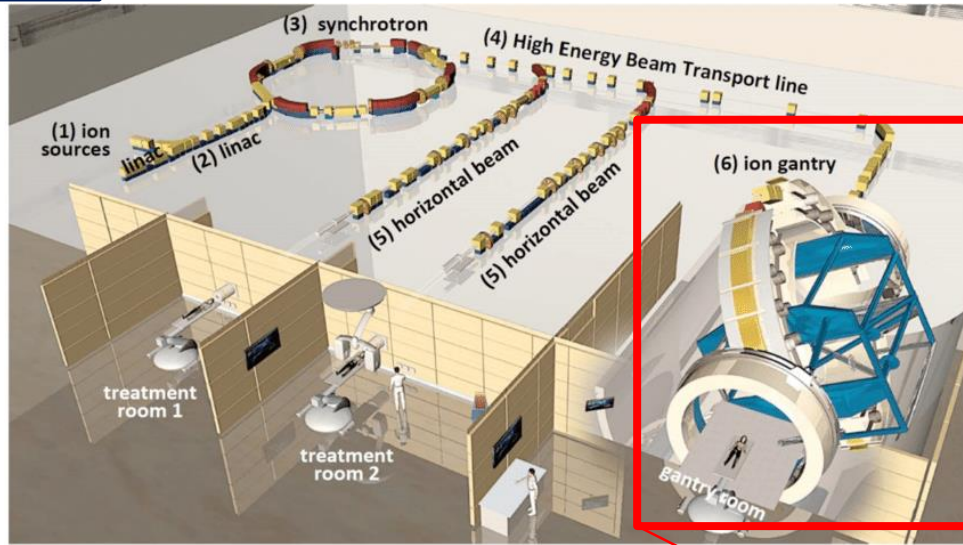


Medical community wants any degree of freedom possible to personalize the treatment plan -> higher efficacy, lower risks

What is a Gantry

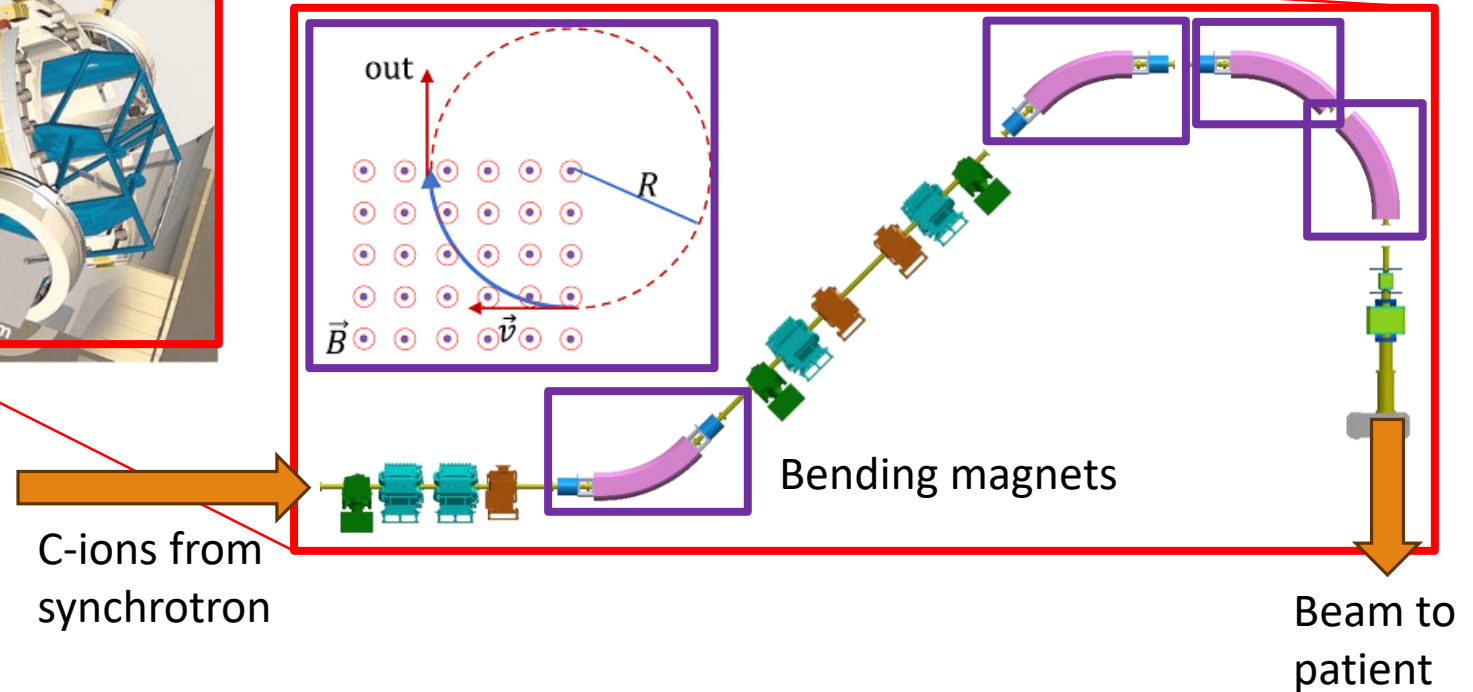
Introduction

- Method
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Source: HIT facility scheme

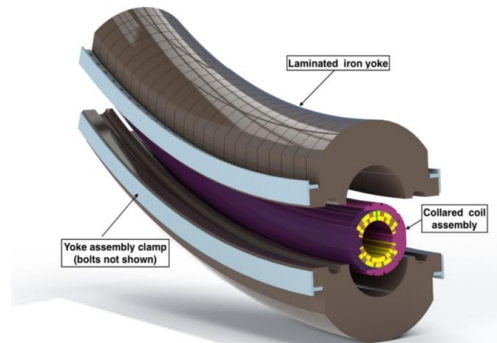
HIT gantry: 600 tons (normal conducting)
HITRIplus gantry: **200 tons (superconducting)**



Challenges for the suspension system

Introduction

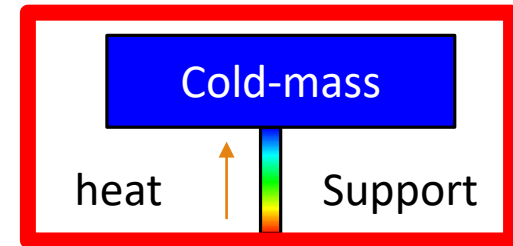
- Method
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[1] CT superconducting magnet
Courtesy of M. Karppinen

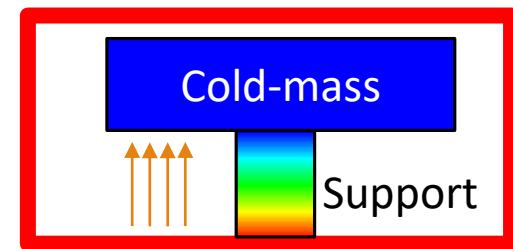
Features of Superconducting Dipole			
Material	Nb-Ti	Bending angle	45 °
Dipole Field	4 T	Bending radius	1.65 m
Aperture	80 mm	Mass	1500 kg
Length	1.3 m	Operating T°	4.5 K (-268 °C)

Thinner support – Less heat – Less cost
– **Less Treatment accuracy**



cryostat

Thicker support – More treatment accuracy –
more heat – more costs



Literature review of suspension systems

Introduction

Method

Models

Results



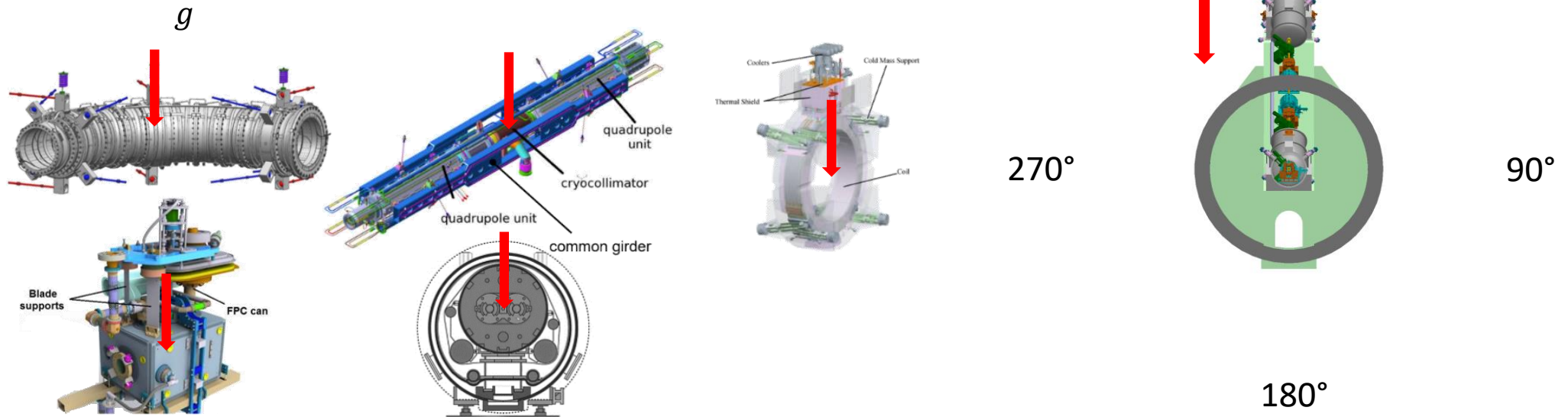
machines <https://doi.org/10.3390/machines11100929>



Review

Literature Review of Suspension Systems for Superconducting Elements

Luca Piacentini ^{1,*}, Luca Dassa ², Diego Perini ², Andris Ratkus ¹, Toms Torims ¹ and Stefano Uberti ³



Proposal of multiple solutions

Introduction

Method

Models

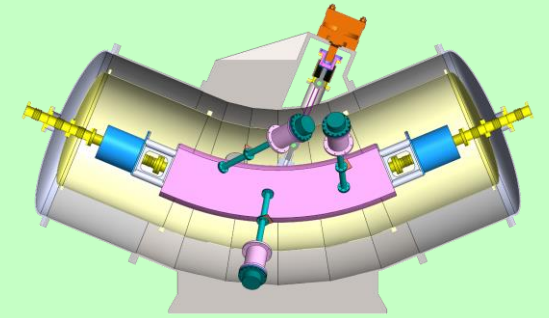
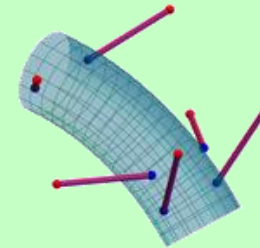
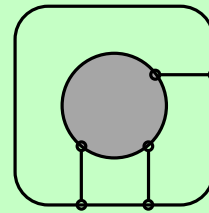
Results

Structure design

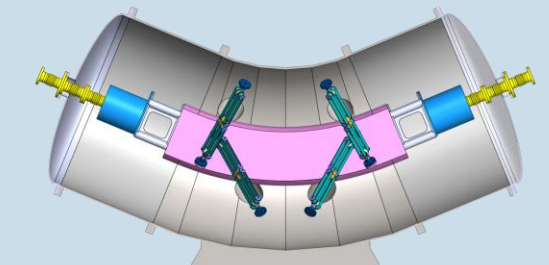
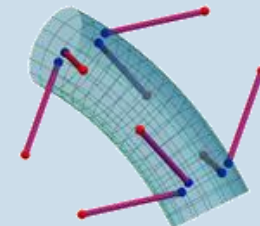
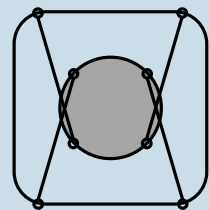
- **Exactly constrained**
- $N \text{ equations} = N \text{ variables}$

- **Over-constrained**
- $N \text{ equations} < N \text{ variables}$

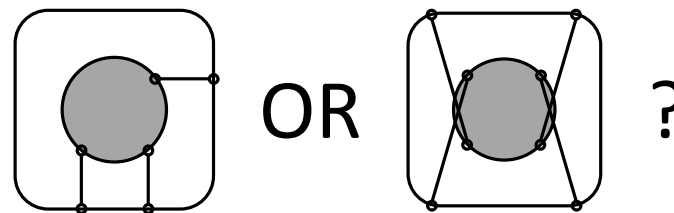
NEW!



Concepts from Literature



Fair comparison



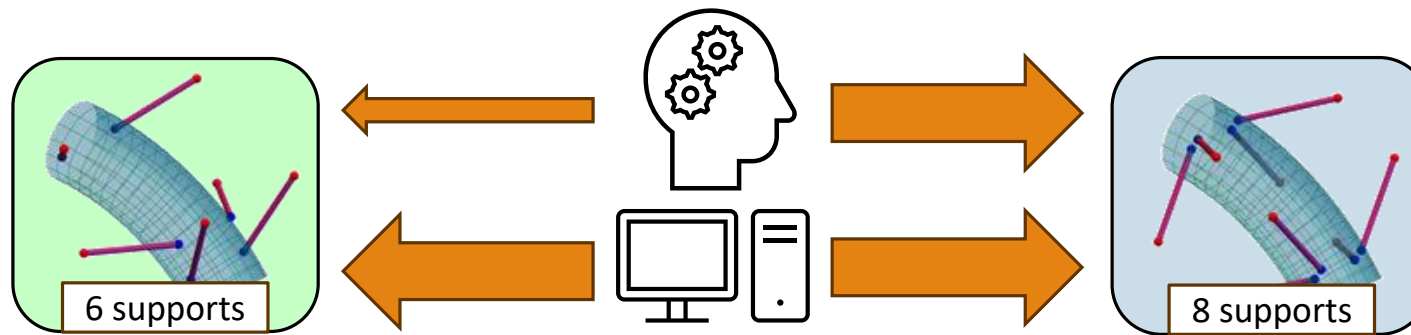
Introduction

Method

Models

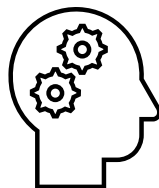
Results

How to place the supports so that the rigidity of the system is maximized during the rotation?



- **20** parameters

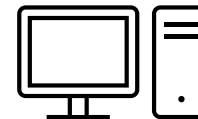
- **6** parameters (symmetry)
- Existing examples



Reduced solution space
imposing symmetries



Optimum



Larger solution space



Optimum

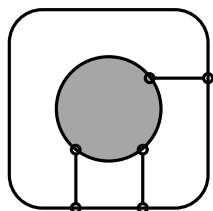
Fast comparison

Introduction

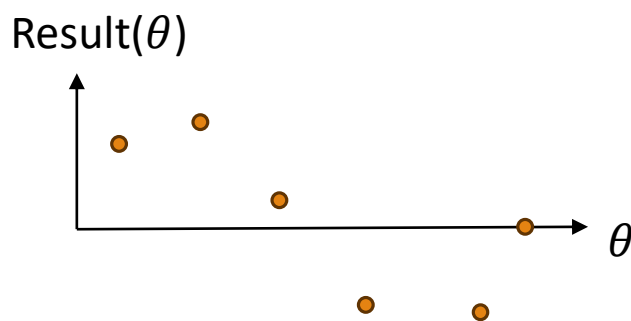
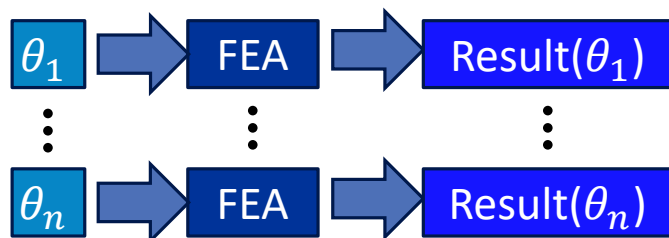
Method

Models

Results

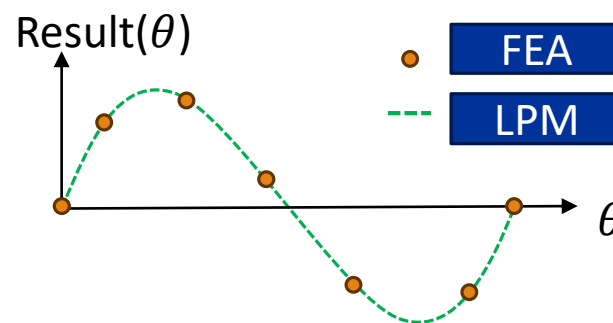
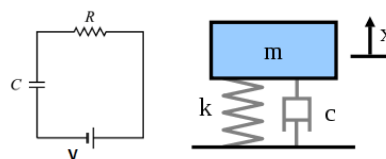


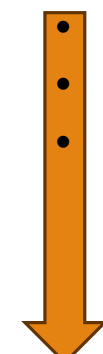

- Material A
- Dimension B



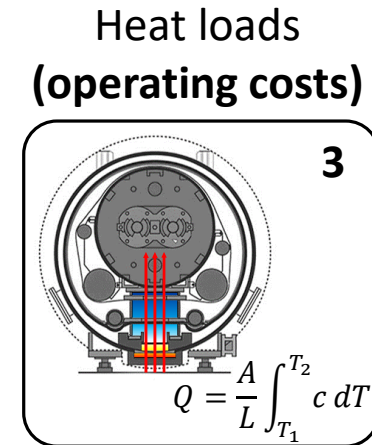
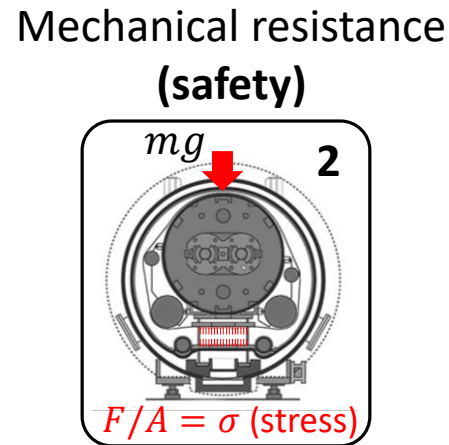
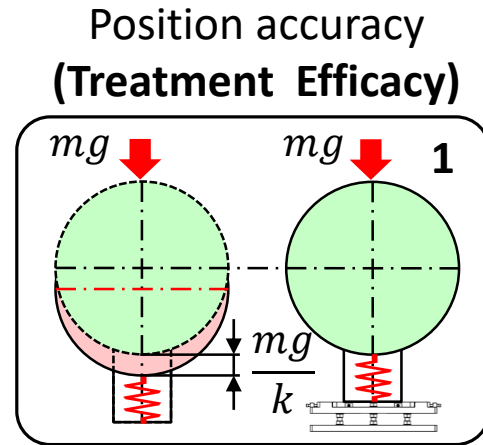
What if I change material, dimensions?

Lumped Parameter Model:



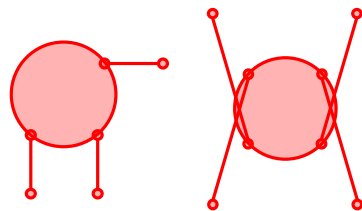
- Faster
 - Global view
 - Possibility to change most input parameters (material, length, ...)
- 
 **optimization**

Lumped Parameter Models Desirables

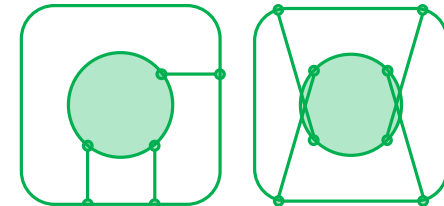


The LPM must be able to:

1. Estimate the **position of the cold mass** to ensure **treatment efficacy**
2. Estimate **loads on the supports** to ensure the mechanical reliability of the system (**safety**)
3. Model all components required to **compare** the two solutions



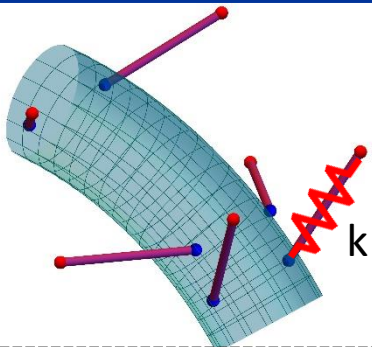
The **Vacuum Vessel** must be included as it is subject to **different loads – different deformations**



LPM 6S: linearization of the Vacuum vessel

- Introduction
- Method
- Models**
- Results

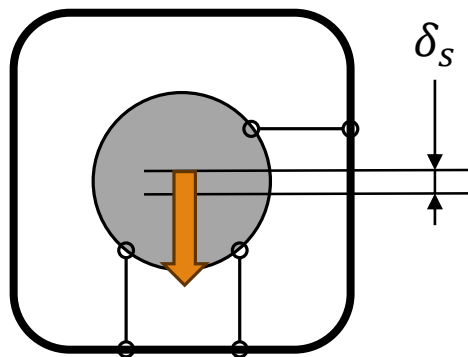
Deformation of supports



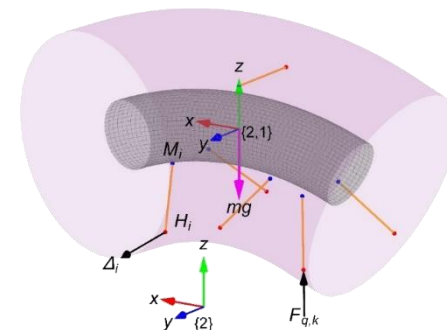
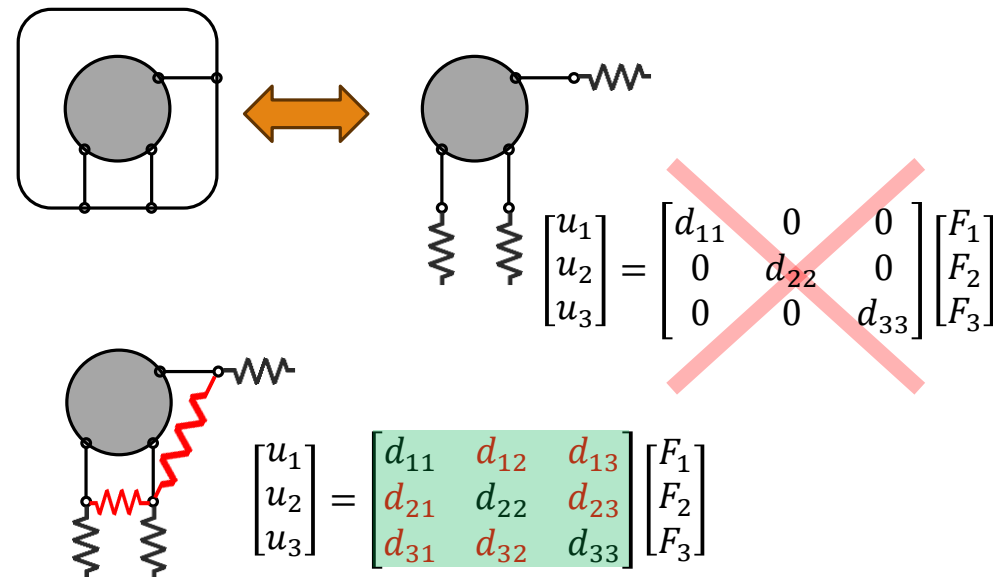
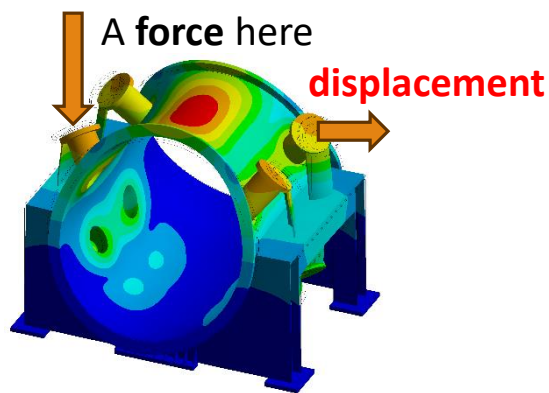
Literature

$$\mathbf{F}_q = \mathbf{J}^T \mathbf{F}_s$$

$$d\mathbf{S} \simeq \mathbf{J} D_q \mathbf{J}^T \mathbf{F}_s = \frac{1}{k} \mathbf{J} \mathbf{J}^T \mathbf{F}_s$$



Deformation of vacuum vessel



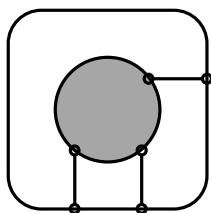
$$\Delta_{ij} = D_{v,ijk} F_q^k \text{ NEW!}$$

Valid for "n" number of supports

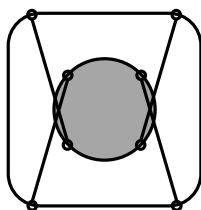
LPM 8S: Principle virtual work solution

- Introduction
- Method
- Models**
- Results

Deformation of supports

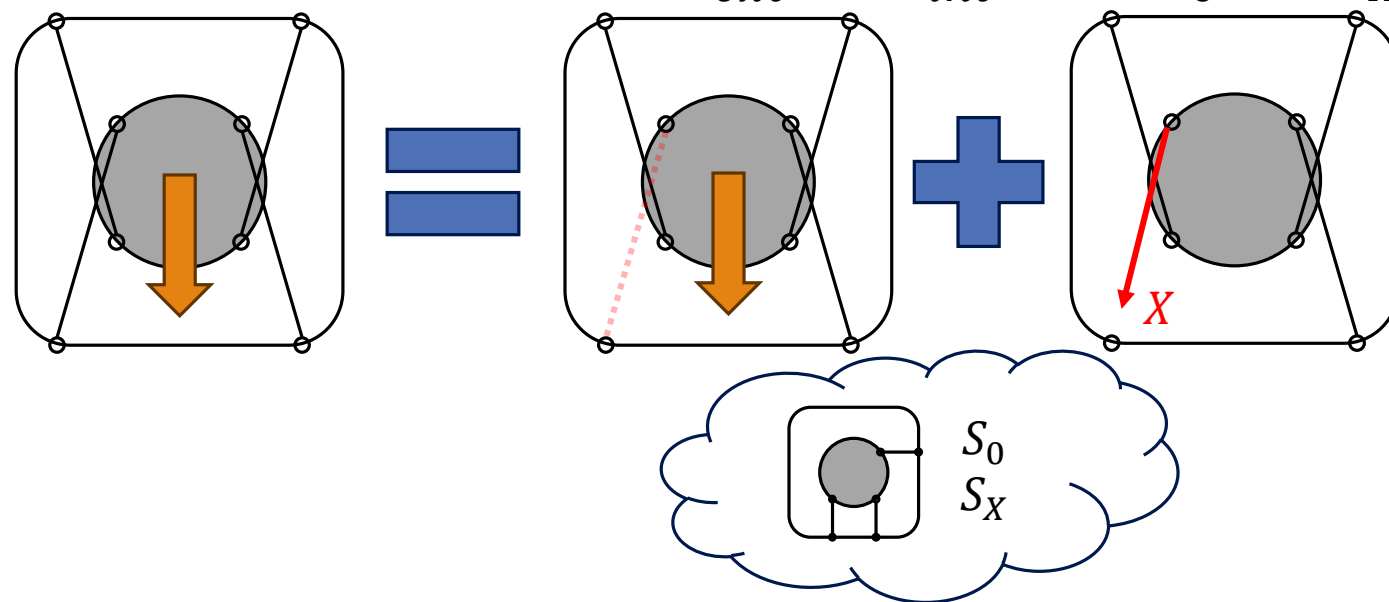


- 3 equilibrium equations
- 3 variables



- 3 equilibrium equations
- **4 variables**

Principle of Virtual work: $W_{ext} = W_{int}$ $S = S_0 + XS_X$



$$\sum_{i=1}^n \int_0^{L_i} N_{j,i} \left(\frac{N_{0,i} + \sum_{p=1}^{n-6} X_p N_{X_p,i}}{E_i A_i} + \frac{\delta_i}{L_i} - \frac{\delta_{P,i}}{L_i} \right) ds = W_{ext,j} \quad \text{NEW!}$$

Valid for “n” tie-rods (supports) of different shapes and dimensions

Validation of the models

Introduction

Method

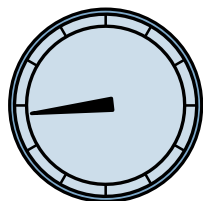
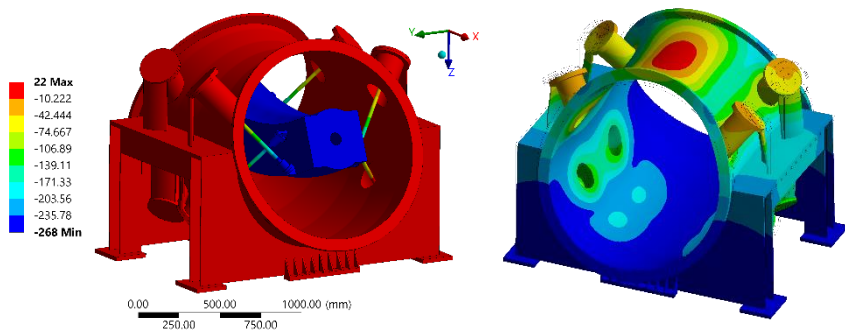
Models

Results

FEAs

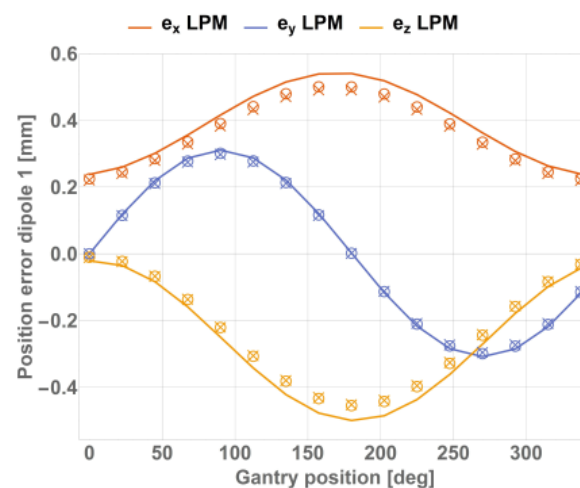
Thermo-mechanical analysis:

- Preload
- Cool-down
- Rotation
- Vacuum pressure

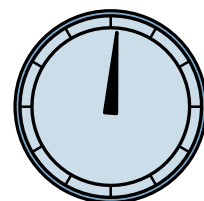


FEA evaluation > 40 min

Validation of 6S LPM

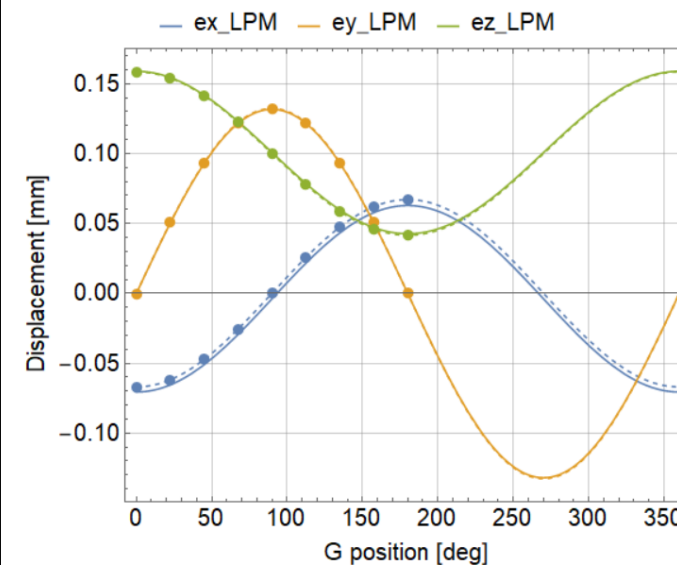


× e_x FEA (25 mm) × e_y FEA (25 mm) × e_z FEA (25 mm)
 ○ e_x FEA (15 mm) ○ e_y FEA (15 mm) ○ e_z FEA (15 mm)



LPM evaluation < 10 sec

Validation of 8S LPM

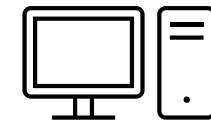
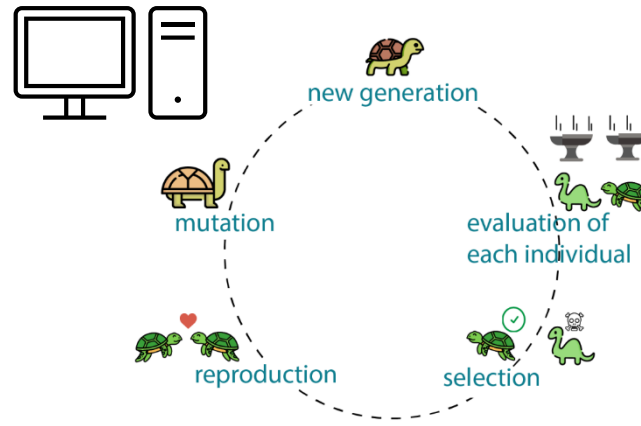
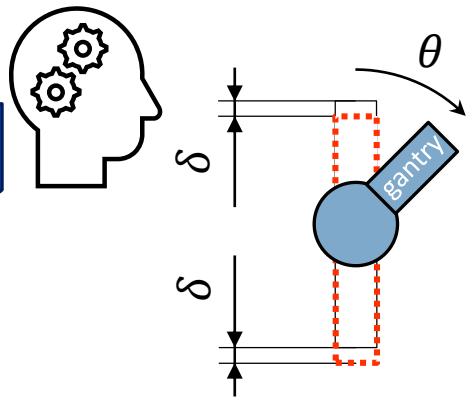


• e_x _FEA • e_y _FEA • e_z _FEA

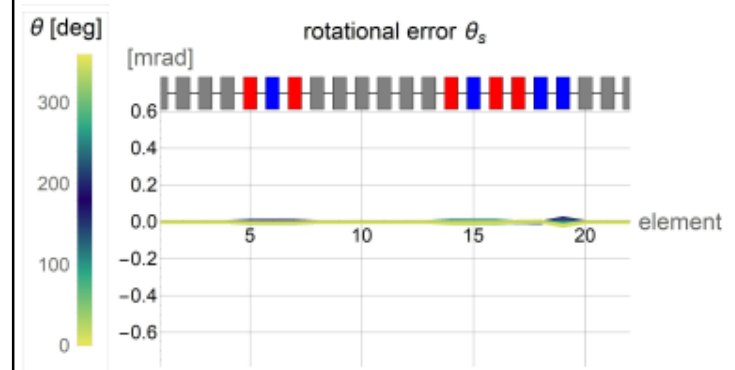
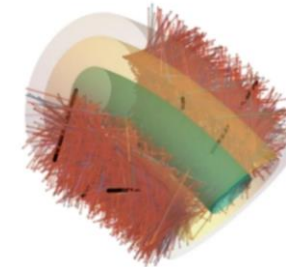
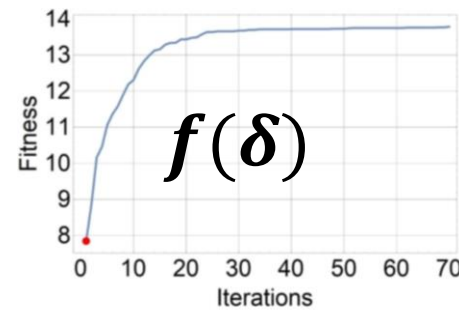
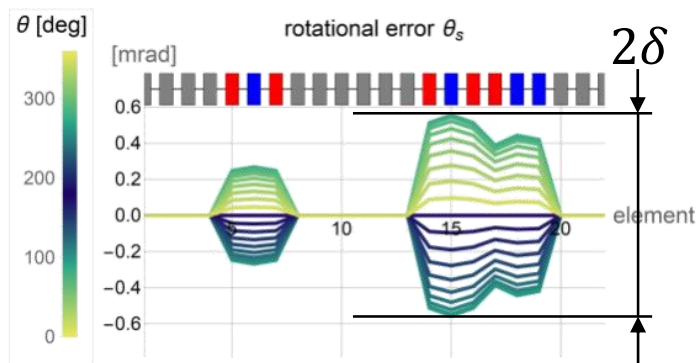
Optimization

- Introduction
- Method
- Models
- Results**

How to place the supports so that the rigidity of the system is maximized during the rotation?



45 to 80 % reduction of 2δ depending on the DOF considered



Design

Introduction

Method

Models

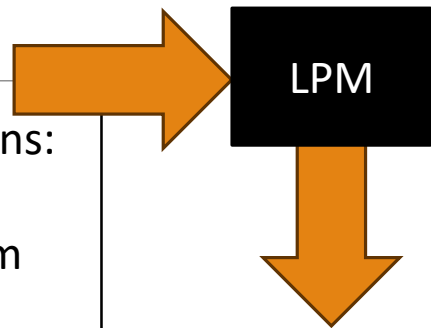
Results

Materials:

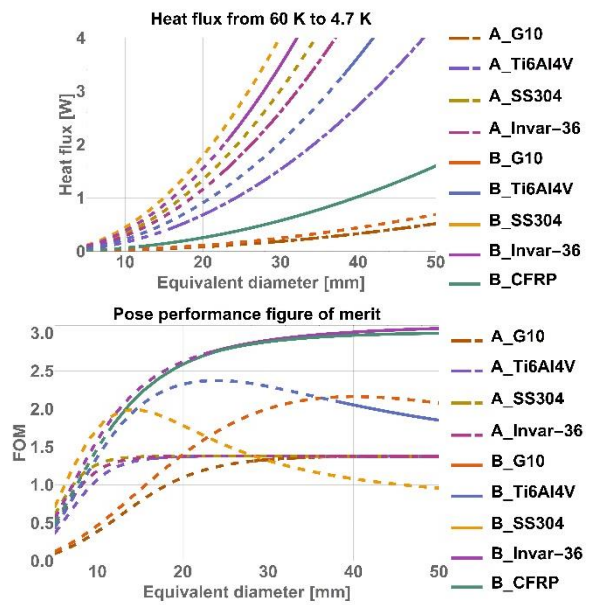
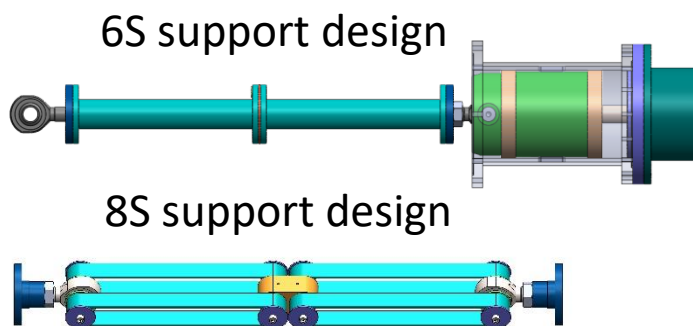
Titanium
Stainless steel
Invar
G10
CFRP

Dimensions:

5 – 50 mm



Comparison of 6S and 8S



	A	B	% improv.
Material	Ti6Al4V	CFRP	
Geometry	tube	double band	
Equivalent diameter	mm 18.0	mm 13.2	
e_x	mm 0.36	mm 0.37	-4%
e_y	mm 0.32	mm 0.24	23%
e_s	mm 0.45	mm 0.36	20%
θ_x	mrad 0.16	mrad 0.04	73%
θ_y	mrad 0.36	mrad 0.12	65%
θ_s	mrad 0.56	mrad 0.22	60%
FOM dipole 1	1.3	1.8	38%
FOM dipole 2	2.3	1.3	-43%
FOM dipole 3	2	1.4	-30%
FOM dipole 4	1.7	1.3	-24%
FOM average	1.79	1.44	-20%
Max load	kN 19.9	kN 22.3	
Safety factor	12.2	12.2	
Heat flux @ 4.5 K	W 0.55	W 0.11	80%
Compressor input power	kW 2.6	kW 0.5	80%

Conclusions

Introduction

Method

Models

Results

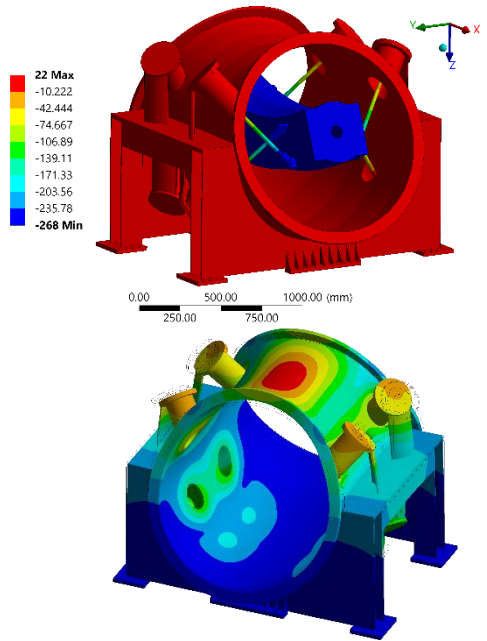
- A literature review highlighted the **lack of a solution** for the suspension system of **superconducting magnets on rotating machines**,
- Two conceptually **different architectures have been proposed**, one new to the field of superconducting elements,
- A machine-oriented optimization has been chosen to have a fair comparison between the two proposed structures,
- **Lumped Parameters Models (LPM)** for both architectures have been **formulated, enabling the machine-oriented optimization** at a much lower computational/time cost,
- **LPMs have been validated** by mean of a standard simulation software proving a **good accuracy** of the models,
- The **optimization** allowed for a **considerable improvement of the accuracy** of the systems,
- The two solutions have been compared: **Solution 6S** being **easier to align** and more **reliable in the alignment** while solution **8S** being **more accurate during operation** and **less expensive** to maintain in terms of **cryogenic cooling**.

Thank you for the attention!

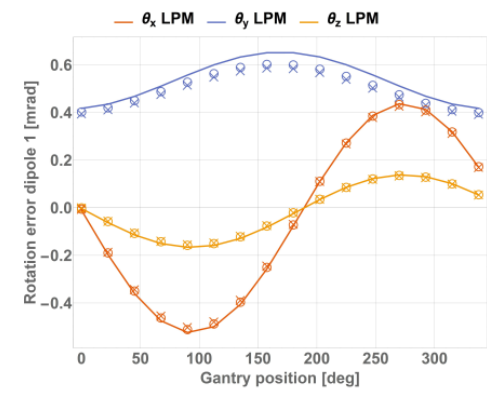
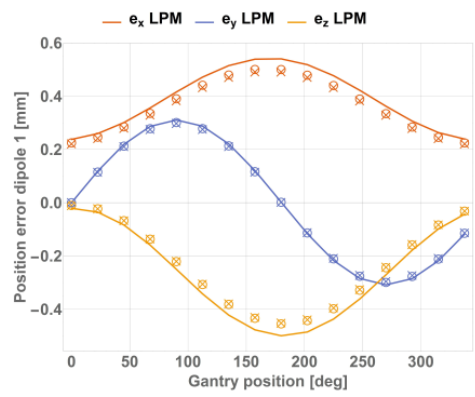


Validation of the models

- Introduction
- Method
- Models
- Results**



Validation of 6S LPM



Validation of 8S LPM

