Thermomag-07

a CARE-HHH-AMT Workshop on Heat Generation & Transfer in Superconducting Magnets

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Heat transfer from the coils to the helium heat sink

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Outline

- •Issues and first order cold mass design values for magnets operating in superfluid helium
- Generalized numerical approach

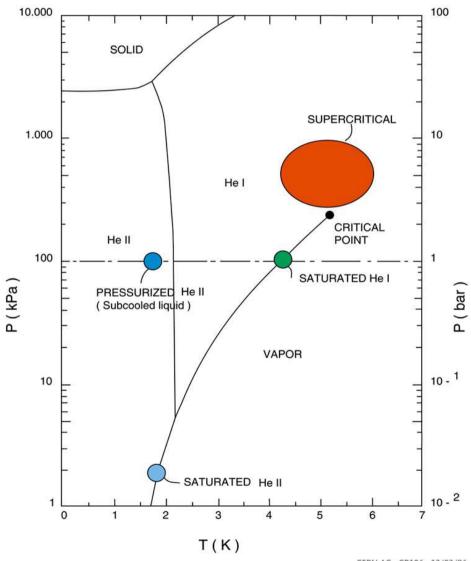
Issues

The cooling concepts and associated cold mass structure has to be determined such as to satisfy the following requirements:

- Static heat loads
- Dynamic heat loads
- Superconductive cable properties
- Coolant (Helium) properties
- Mechanical (space) constraints

Coolant (Helium) properties

Phase diagram of helium

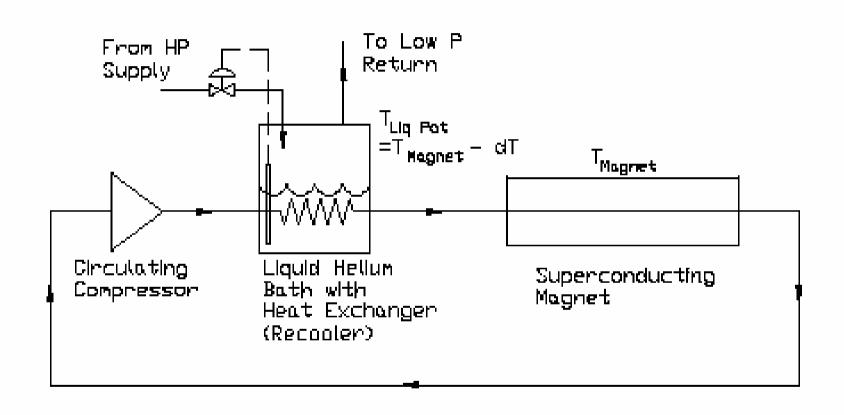


Basic cooling schemes and their practical limits

Cooling by means of:

- Forced convection of supercritical helium
- Pool boiling of normal helium (not covered)
- Forced convection of superfluid helium (not covered)
- Conductive cooling in pressurized superfluid helium

Forced convection of supercritical helium



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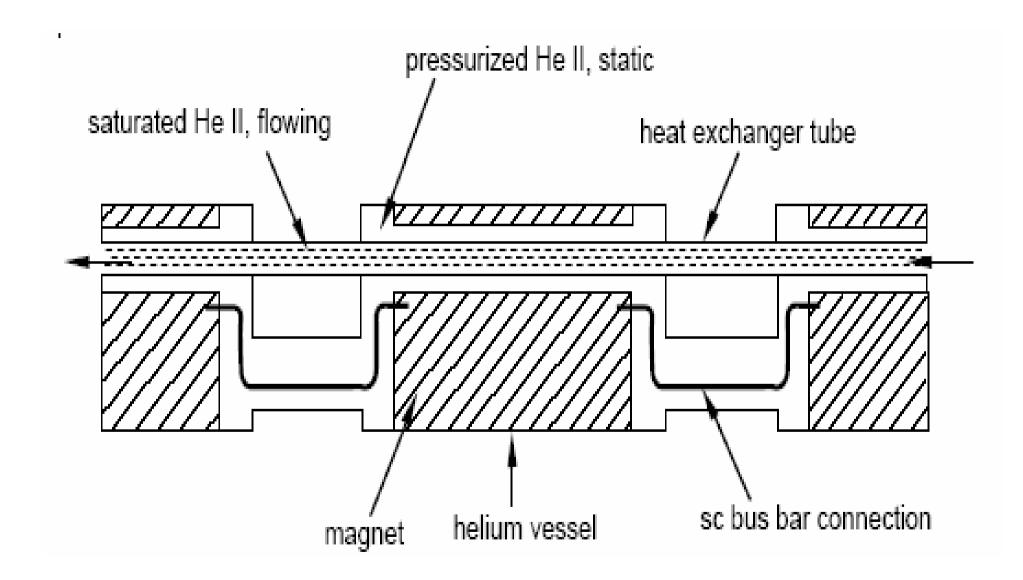
Forced convection of supercritical helium

- Forced convection cooling by supercritical helium is characterized by available pressure drop over a given length, and by allowed T difference between inlet and outlet.
- Over a given length covering a string of magnets, usually it is not the available pressure drop, but the allowed T difference that is limiting. As a consequence intermediate re-coolers need to be installed.
- Typical pressures are P≈3-8 bar, ∆P≈1-2 mbar per magnet
- Typical T≈4.4 K, ∆T≈50-150 mK per magnet
- Typical heat loads are ≈2 W per magnet (through flow, RHIC magnets) or ≈6 W per magnet (cross flow, i.e. helium forced close to the coil, as envisaged for SSC magnets).
- Horizontal flows. Typical gap sizes between cold bore and coil ≈2 mm
- Eric Willen: "Calculations and measurements of the heating in 17 m SSC magnets3 indicate that with crossflow cooling, a heat input to the 40 mm diameter SSC magnet coil of 2.4 W/m would result in a coil temperature rise of only 0.14 K. It is estimated that at least 30 W/m of heat could be tolerated in this coil structure."
- If we propose to use extra T margin (6 K 4.5 K), and at a ΔP =5 bar (8 in , 3 out) over the entire magnet we could in principle extract 1 kW using about 53 g/s of mass flow. Possible problems: high ΔP , orifice sizes, flow paths may lead to badly cooled "dead" spots.

Conductive cooling of pressurized superfluid helium

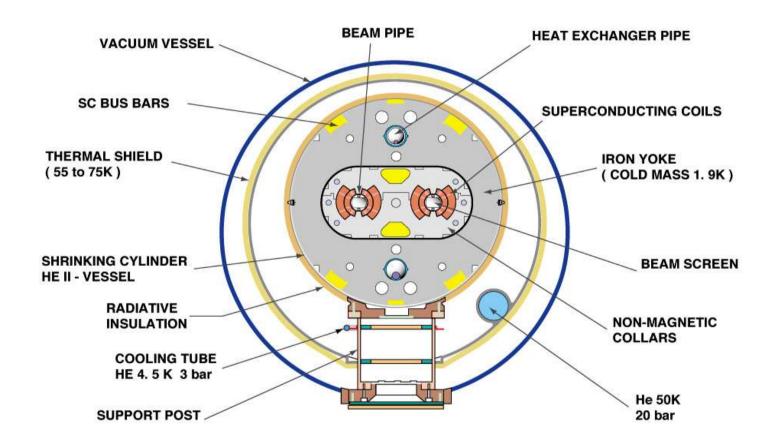
- Efficient, but if cold source is far away requires high conducting cross section (example 1 W/m over 50 m requires about 90 cm² between 1.8 and 1.9 K).
- If cold source can be distributed over the length (thus only radial distances to cover) very efficient. Cold source provided by two-phase flow of saturated superfluid helium.
- Applied in LHC (main magnets ≈1 W/m), suitable for high heat loads (inner triplets ≈15 W/m).
- Requires attention to conduction paths, but is certainly extendable to heat loads of about 50-100 W/m.

Two-phase flow of saturated Hell



Two-phase flow of saturated Hell

CROSS SECTION OF LHC DIPOLE



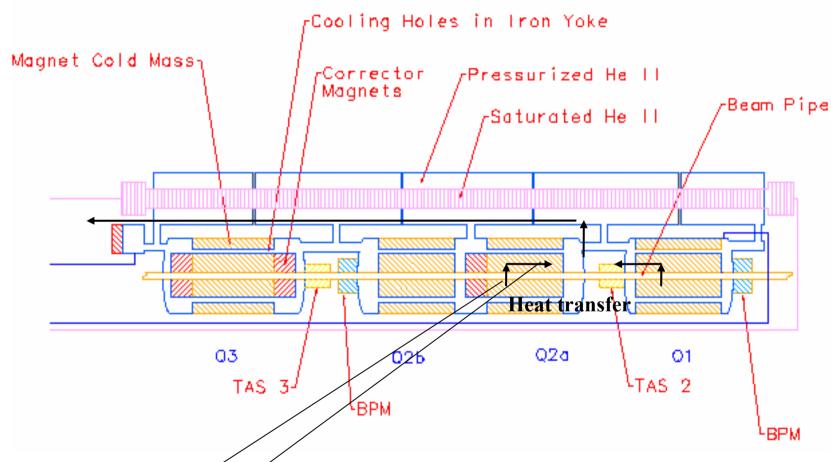
CERN AC _HE107A_ V02/02/98

The total available ΔT for heat extraction is determined by T_{λ} =2.17 K at the coil to the suction pressure temperature of 1.776 K (15 mbar) provided by the cold compressors:

$$\Delta T = 394 \text{ mK}$$

We subdivide the total available ΔT into the parts relevant to the system design.

- ATcoil: from the coil to the helium
- ΔTcoil-freeA (radial): the channels which collect the heat from the vicinity of the coil until it reaches the free conduction zone
- ΔTfreeA-bHX (longitudinal): the free helium conduction zone until the cold source (bayonet heat exchanger, bHX)
- △TbHXA: the bHX heat exchange area
- <u>ATbHXdp</u>: the bHX pressure drop induced temperature loss
- <u>ATcfHX</u>: the counterflow HX (cfHX) pressure drop induced temperature loss
- △Tpiping: the piping to cold compressors, pressure drop induced temperature loss



ΔTcoil: typically 80-90 mK available down from 2.17 K max ΔTcoil-freeA (radial): typically 60-70 mK available around 2.050 K ΔTfreeA-bHX (longitudinal): typically 80-90 mK available around 1.98 K about 160 mK remains for heat transfer to cold source and up to cold

In general heat transfer through a HeII channel of length *L* and cross section *A*, subject to a linear heat load *q* can be characterized by:

$$L^{n+1}(q/A)^n=C(Tmax,Tmin),$$

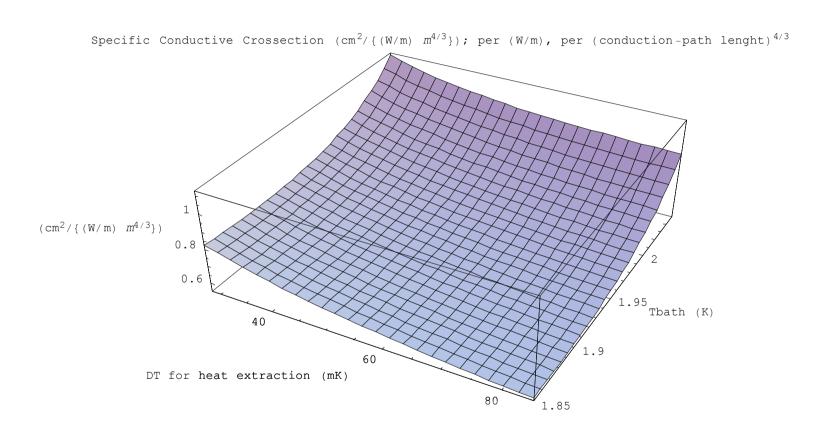
were n=3 (or 3.4), and Tmax, Tmin are temperatures at the channel extremities (ref.: LHC project report 144).

For initial conduction paths scaling estimates we define the specific area:

$$Aspec=A/(qL^{4/3})=1/C(Tmax,Tmin)^{1/3}$$

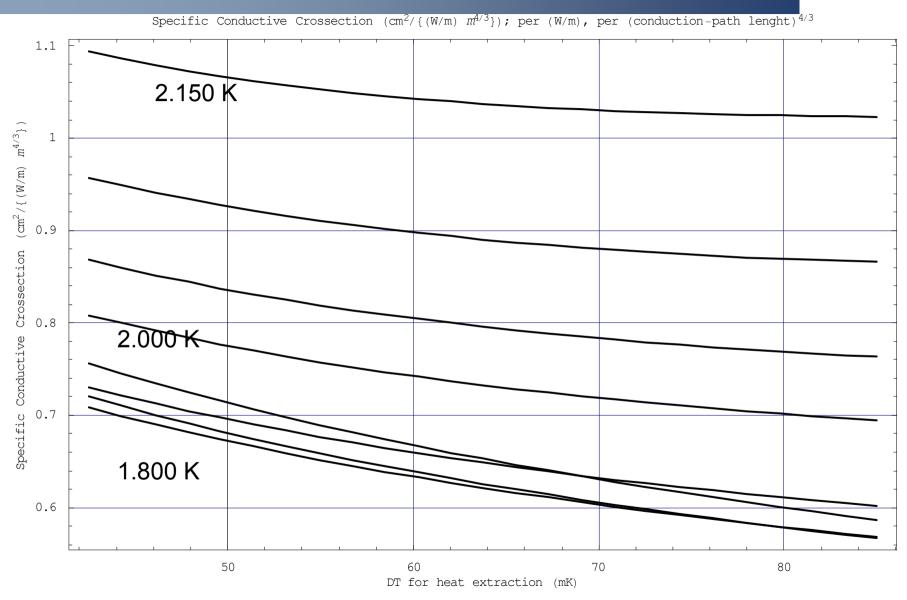
The above formula can be applied, using the typical temperature ranges, to provide an estimate for the necessary conduction areas inside the magnet.

Specific Conductive Cross section (cm²/[W/m m³/4]])

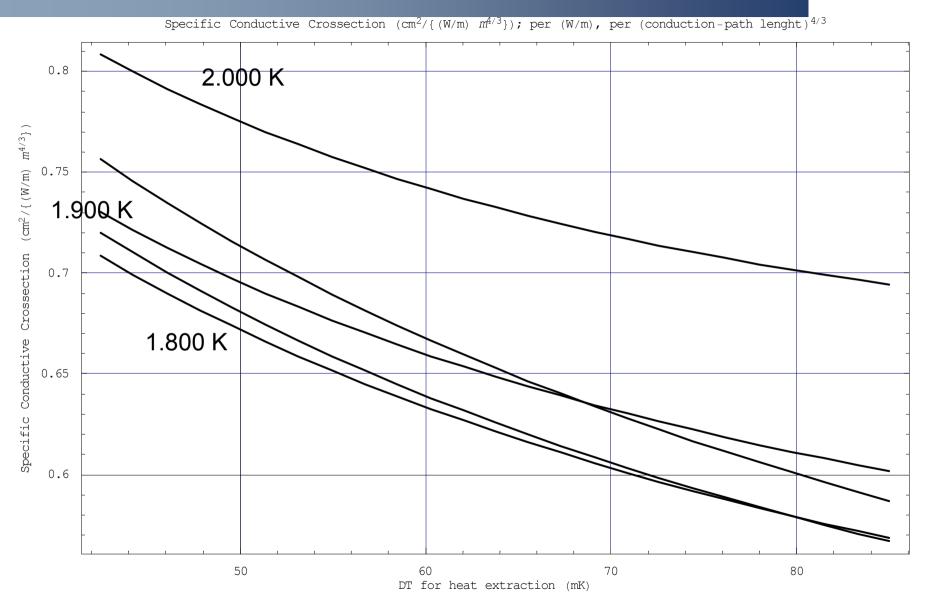


Aspec is in the range of 0.3 to 1.1 cm²/[W/m m^{3/4}]

Aspec for Tmn (=Tbath) 1.800 K to 2.150 K (cm²/[W/m m³/4])



Aspec for Tmn (=Tbath) 1.800 K to 2.000 K (cm²/[W/m m³/4])



Example:

NED dipole/Q1 LHC inner triplet upgrade, 100 W/m, up to 5 m longitudinal heat extraction length, Tbath~1.935 K, ∆T~85 mK, Aspec ~0.55:

--> A~470 cm² to be made in the yoke

Assuming 15% of the cold mass volume taken up by the coil, which is what needs to be condcuted our radially over ~ 0.05 m at Tbath ~ 2.020 K, $\Delta T \sim 65$ mK, Aspec ~ 0.75 :

--> A~0.21 cm² to be provided in the collar & yoke laminations every 10 cm

Conclusion: values are still in the "constructable" range

Beyond analytical estimates: framework of the CERN/WUT collaboration

Within the framework of K944/AT/LHC Cooperation on operational safety for the LHC cryogenic system. Addendum No 2. Article 3: Part III Heat transfer flow from the magnet structure to the helium after resistive transition, between CERN and Wroclaw University of Technology:

- Analysis of heat extraction from the magnet is made. This
 effort has evolved into a direction which is more generally
 applicable than to resistive transitions only.
- A cryostat for performing measurements of the heat transfer to and heat propagation in superfluid pressurized helium is designed and fabricated.

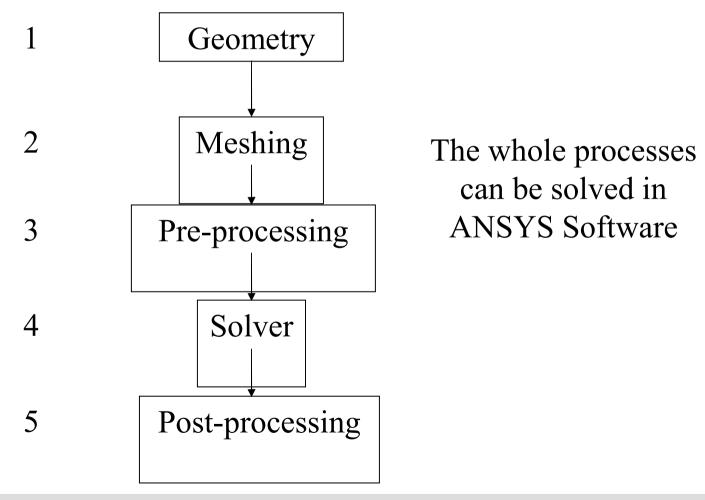
Numerical Approach: goals

The goals of the present work are to provide a numerical code which:

- •Models the heat transfer throughout the whole magnet cold mass structure and the helium it contains with the exception of the heat transfer processes specific to the superconducting cable itself.
- •Treats fluid hydro & thermodynamics in the same time as thermal conduction through solids.
- •Can model magnets cooled in pool boiling-, supercritical- and pressurized superfluid helium.
- Can solve steady state and transients.
- •Can use heat deposition data as function of geometry and time (i.e. Fluka results).
- •Can use arbitrary heat transfer correlations specific to the accelerator magnet case.
- •Is generic. I.e. in order to facilitate possibilities for long term use and development, the code should not be linked to a specific institute's environment, and use as much as possible widely available software.

Numerical Approach: steps

The 5 steps of numerical solution



Numerical Approach: the CFD software

The software available at Wrocław University of Technology

Meshing:

- ANSYS ICEM v10.0;

Pre – processing, solver, post – processing:

- ANSYS CFX v10.0

CFD tool

CERN

The properties of Helium - Hepak

The possibility of creation the *.rgp (real gas properties) files which can be used in ANSYS CFX software

Properties of Helium

Numerical Approach: status

- Helium properties have been integrated
- •Superfluid helium conduction module under development
- code comparison with analytical & literature data has started
- •Trial examples, like the MQY magnet, are being explored

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