

---

# FRESCA2 magnet design latest version

P. Ferracin, J. E. Munoz Garcia

EuCARD ESAC review for the FRESCA2 dipole  
CEA, Saclay  
27-29 February, 2013

# Outline

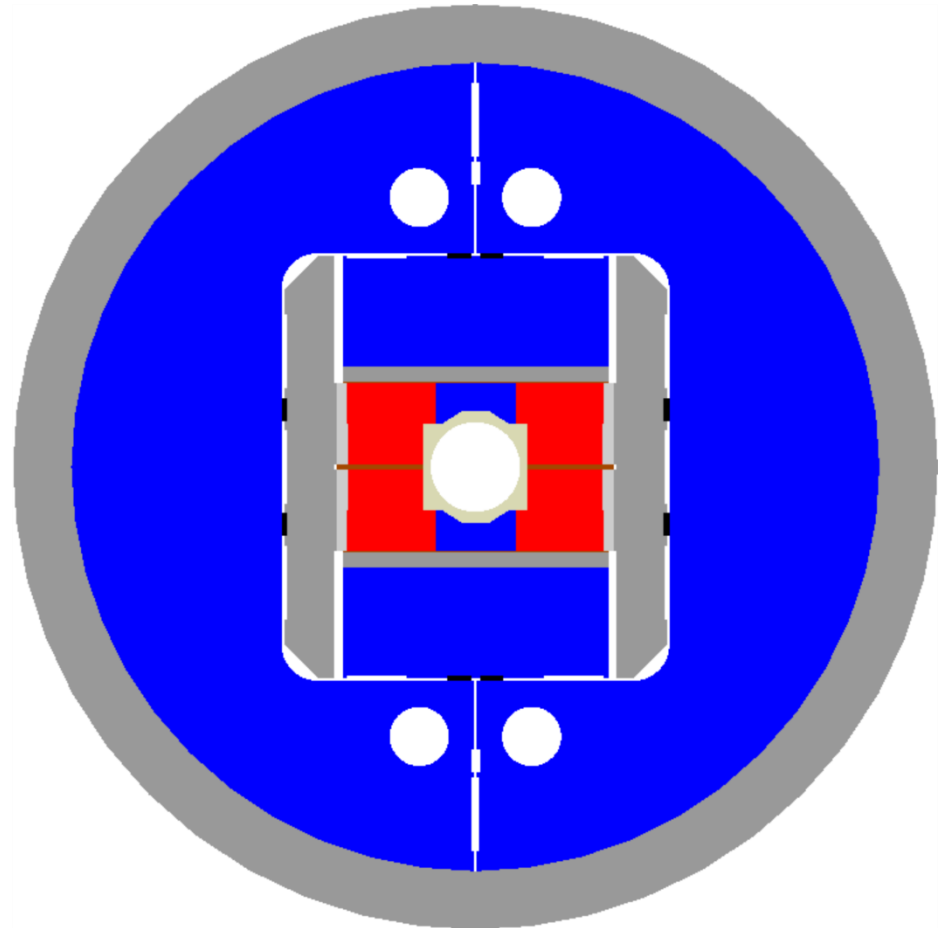
---

- Introduction: magnet overview
- Conductor and cable parameters
- Coil design and magnet parameters
- Support structure and stresses
- 3D FE analysis
  - Horizontal and axial pre-load conditions
- Conclusions

# Introduction

## Overview of magnet design

- Target: 13 T in 100 mm clear bore
- OD: 1.030 m; length: 2.255 m
- Al shell, 65 mm thick, 1.6 m long
- Bladder and key pre-load
- Iron yoke
  - Holes for axial rods (60 mm  $\varnothing$ )
- Horizontal stainless steel pad
  - 3 bladders per side, 75 mm wide
  - 2 load keys
- Vertical iron pad
  - 2 bladders per side, 60 mm wide
  - 2 load keys
- Four double-layer coils
- Iron and Ti alloy central posts



# Timeline

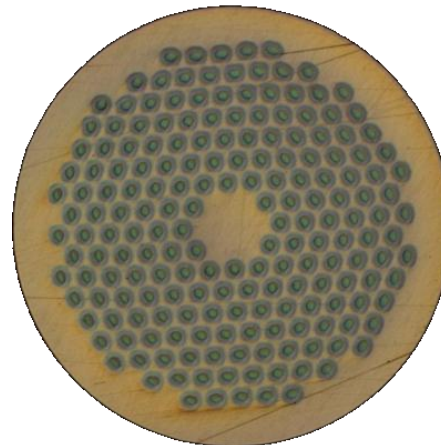
- Version tracker

Date	#	Status
2012 June	4.4	Structure has been realized and assembled successfully Winding tests will be done on this version
2011 October	3.4	Last corrections before realization of the structure
2011 July	2.3	Call-for-tenders of Coil pack mockup + Structure
2011 March	2.2	"Optimized" coil/structure geometry
2011 February	1.1	"Baseline" version

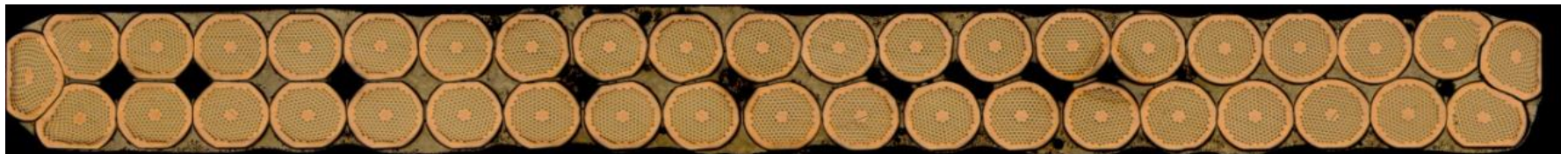
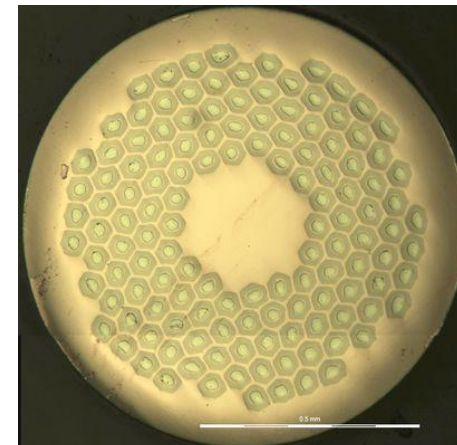
# Conductor and cable parameters

- PIT (192) and RRP (132/169)
- Strand diameter: 1 mm
- Cu/Sc: 1.3 → 56% Cu
- Strand #: 40
- Bare width after cabling: **20.90** mm
- Bare thickness after cabling: **1.82** mm
- Braided insulation: 0.2 mm
- Assumed growth during HT
  - 4% in thickness and 2% in width
  - Based on LARP and SMC experience
- Bare width after HT: **21.32** mm
- Bare thickness after HT: **1.89** mm

PIT strand



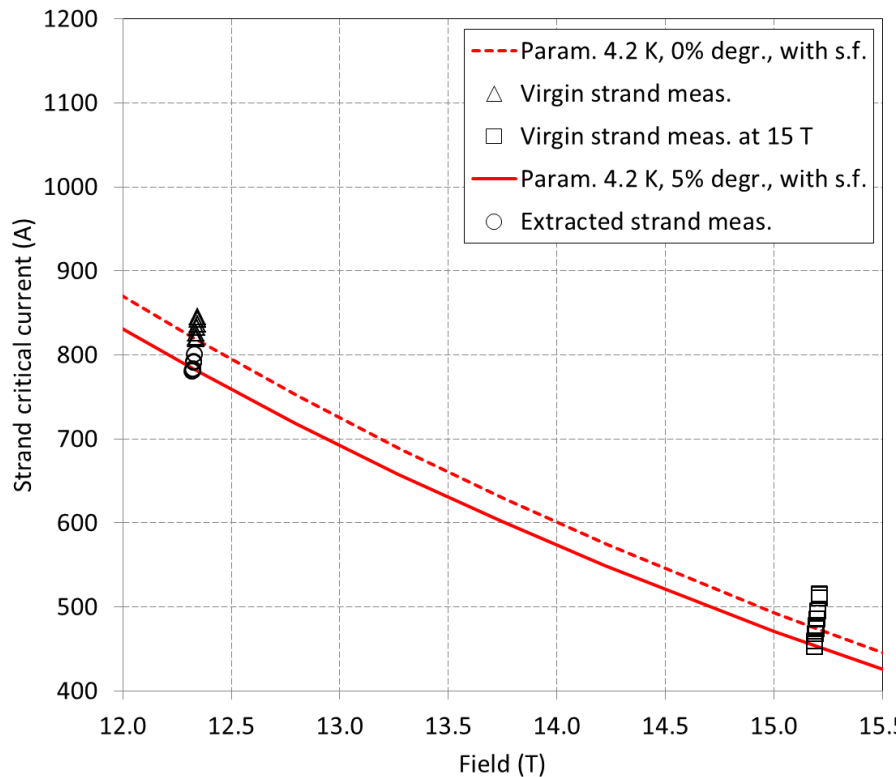
RRP strand



# Conductor properties: $I_c$ (with self-field correction)

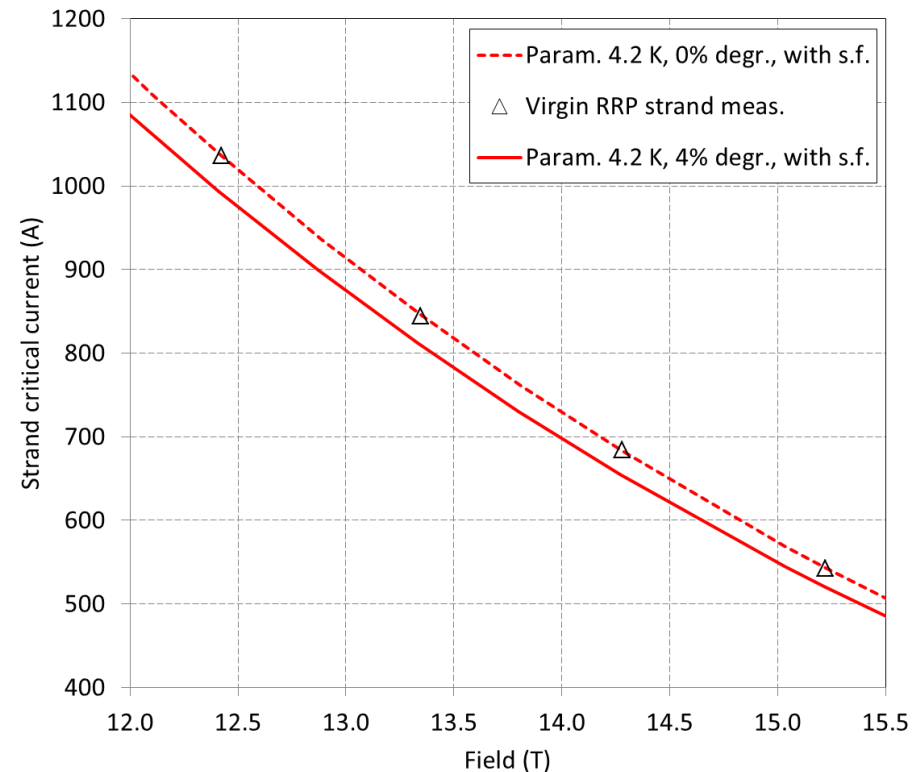
- PIT

– 5% cabling degradation



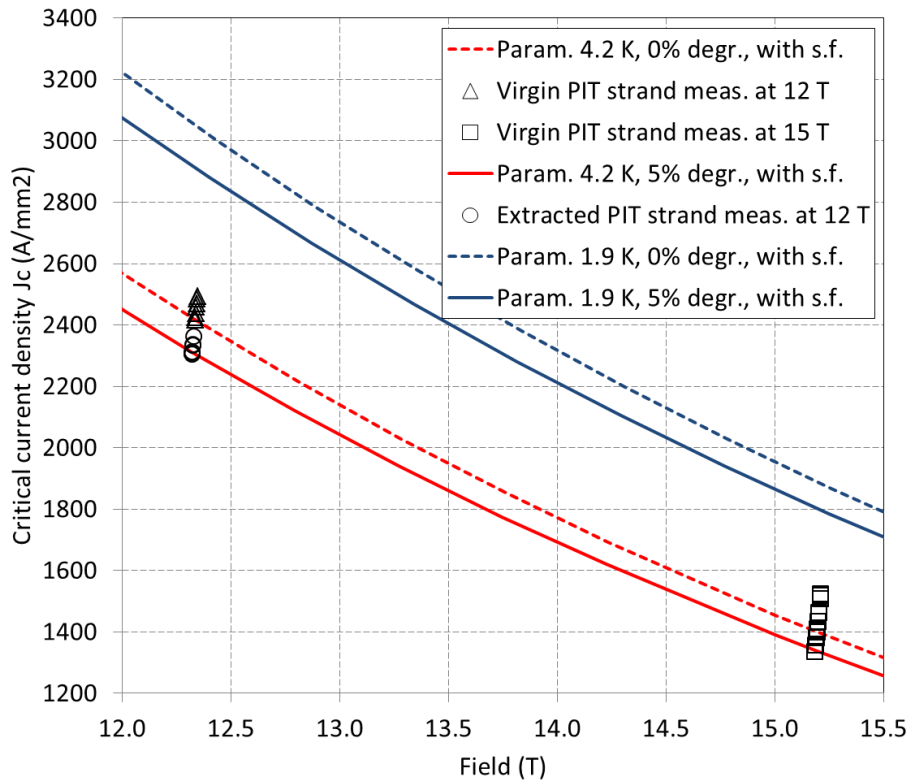
- RRP

– 4% cabling degradation

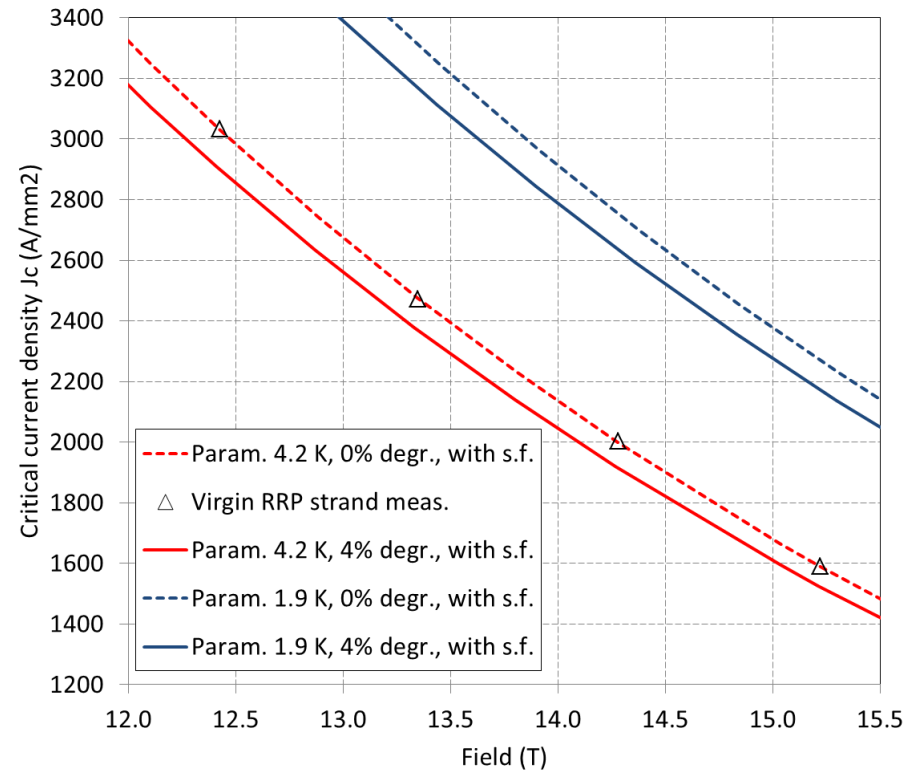


# Conductor properties: $J_c$ (with self-field correction)

- PIT
- $J_c$  (A/mm<sup>2</sup>): **2450** at 12 T,  
**1400** at 15 T

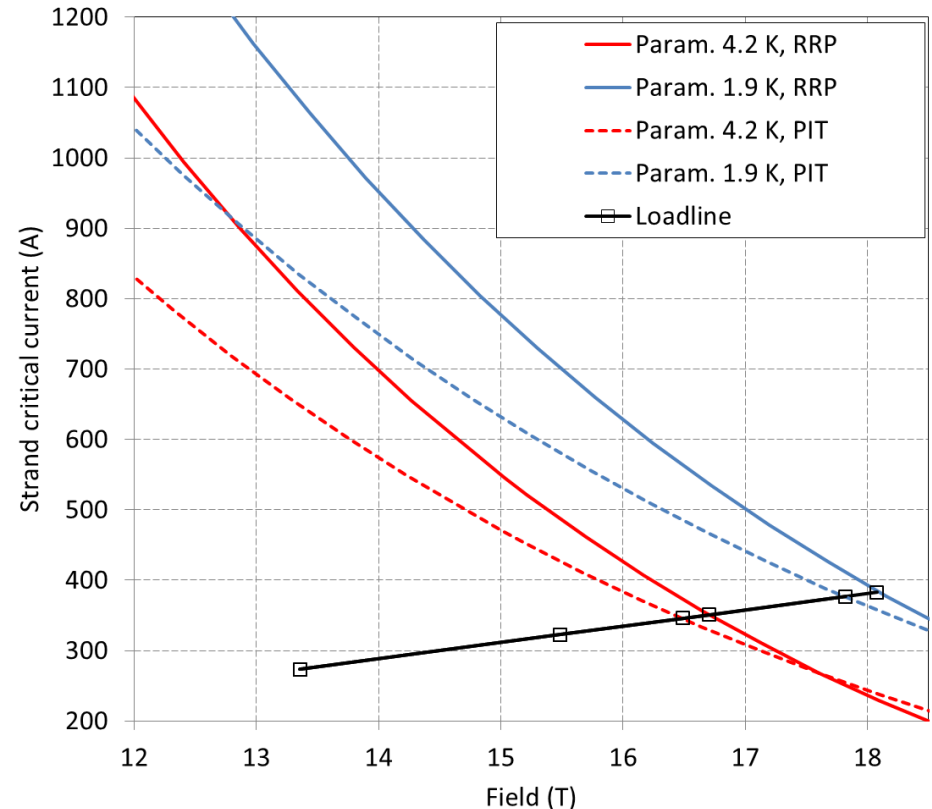


- RRP
- $J_c$  (A/mm<sup>2</sup>): **3150** at 12 T,  
**1600** at 15 T



# Magnet parameters

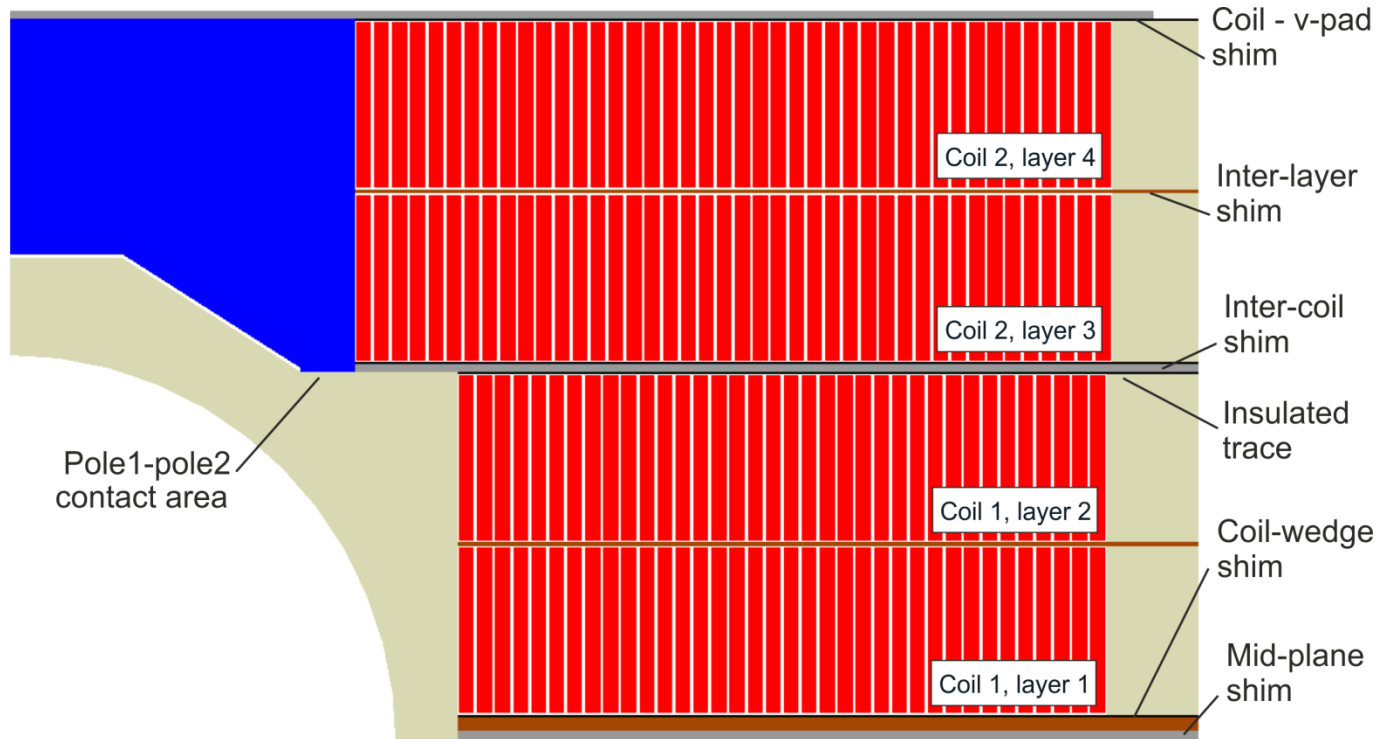
- Operational condition (**13 T**)
  - $I_{op}$ : 10.9 kA
  - $B_{peak\_op}$ : 13.4 T
  - **~79%** of  $I_{SS}$  at 4.2 K
    - $B_{bore\_ss}$ : 16.0 T
  - **~72%** of  $I_{SS}$  at 1.9 K
    - $B_{bore\_ss}$ : 17.2 T;
- **15 T** bore field (“ultimate”)
  - **86%** of 1.9 K  $I_{SS}$
- 1% difference (load-line) between PIT and RRP





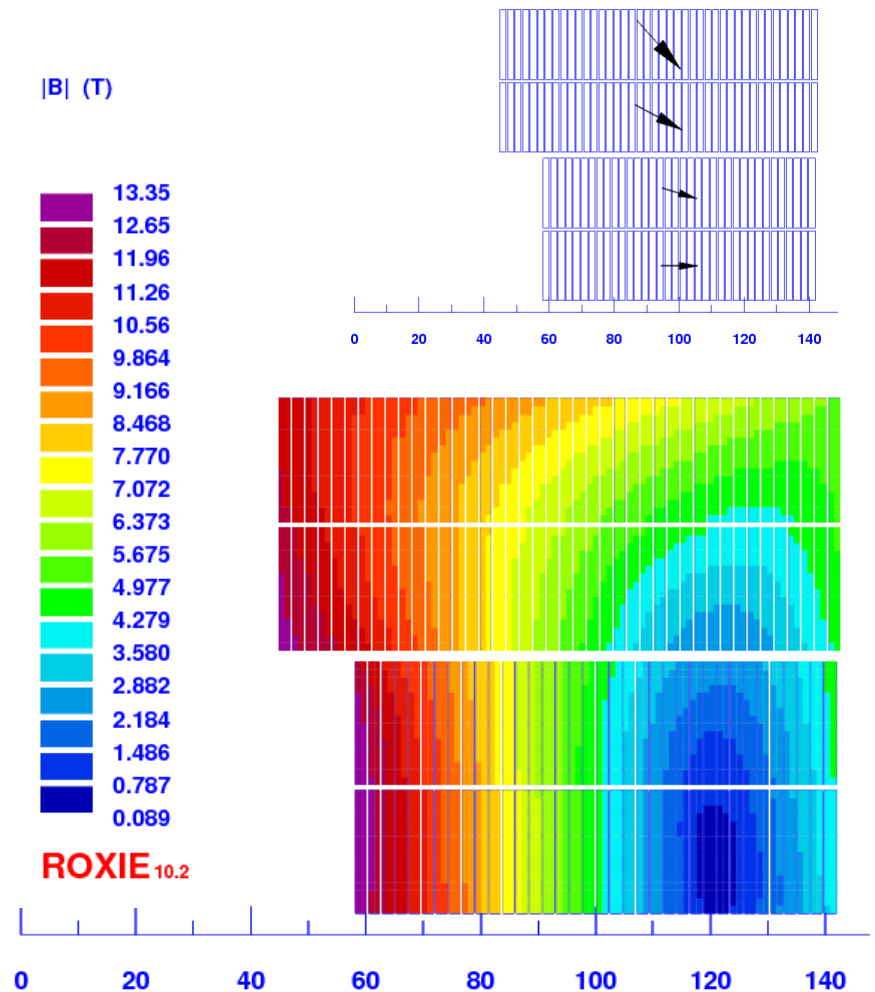
# Coil cross-section

- Two double-layers with 36 and 42 turns
- Bore aperture 100 mm
- Iron and Ti poles
- Inter-coil gap and mid-plane “tailored” shim



# 2D magnetic analysis

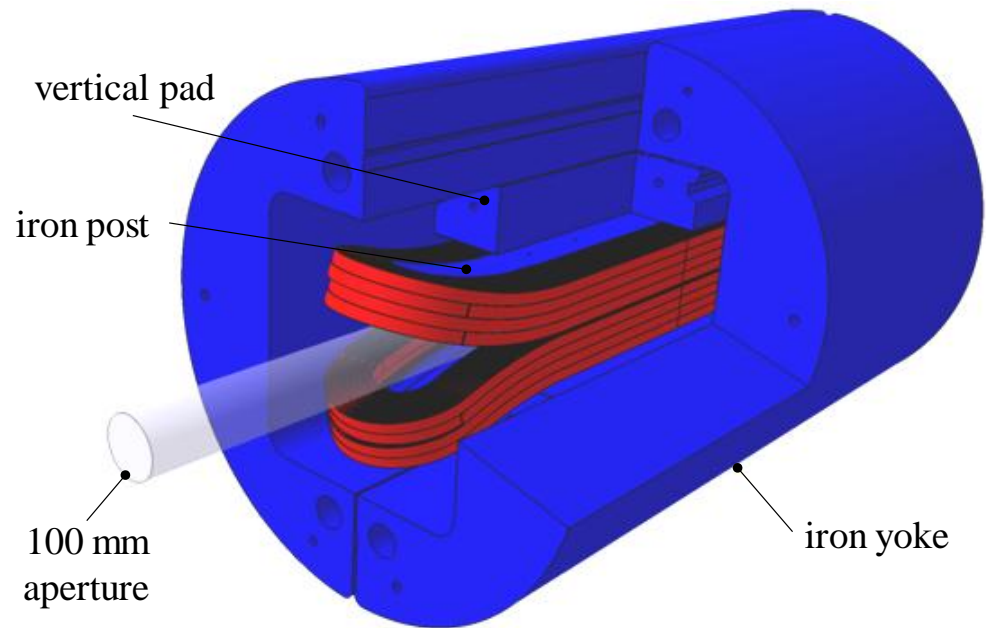
- Peak field in layer 1
  - Layer 2,3,4 with field 1%,3%,9% lower
- Field quality ( $2/3$  of  $R_{\text{bore}}$ )
  - <1% homogeneity
    - $\sim 70$  units of  $b_3$ ,  $\sim 30$  of  $b_5$
- E.m. forces
  - Lorentz stress
    - 75 MPa in L12
    - 95 MPa in L34
    - 130 MPa with 15 T bore field



# 3D magnetic analysis

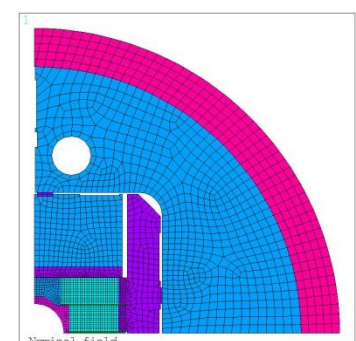
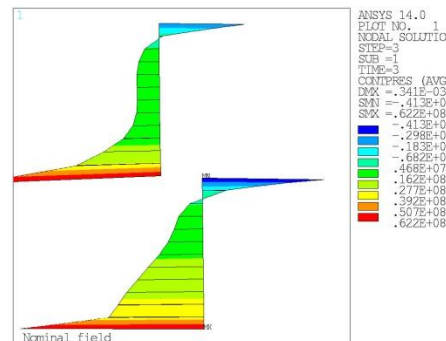
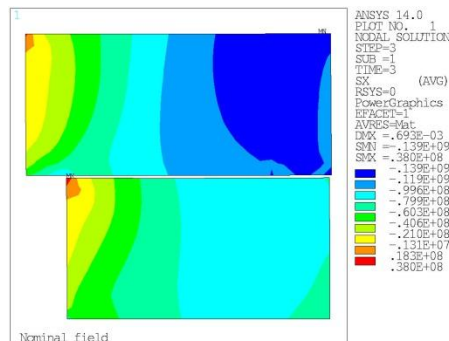
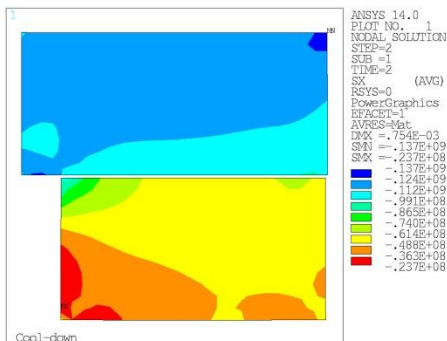
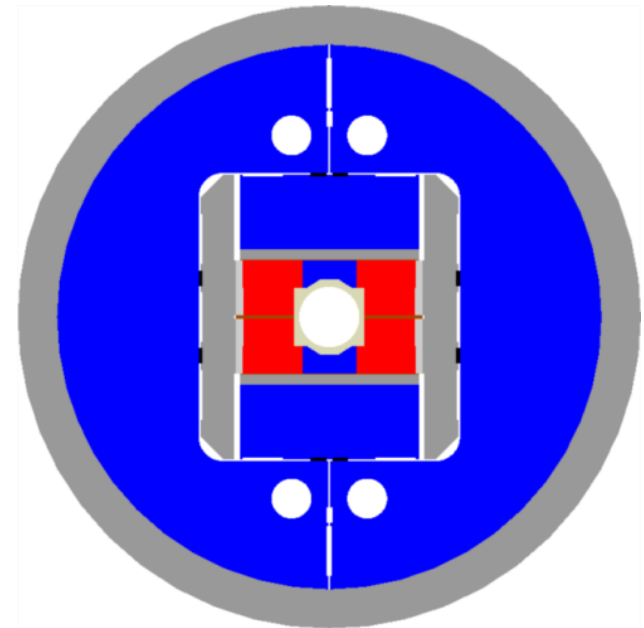
## Coil design

- 730 mm of total straight section
- Hard-way bend with minimum radius of 700 mm
- Inclined straight section (17 degr.) of about 30 mm
- Coil length of 1.5 m
- 10% field margin in the ends
- 1% uniformity of over 540 mm along z
- Stored energy (13 T):  
**3.8 MJ**
  - Stored energy density comparable to other  $\text{Nb}_3\text{Sn}$  magnets



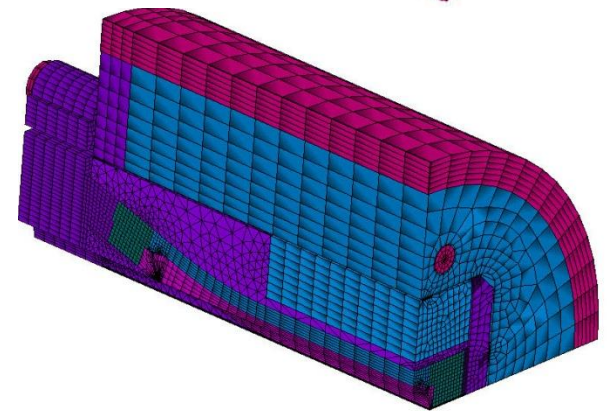
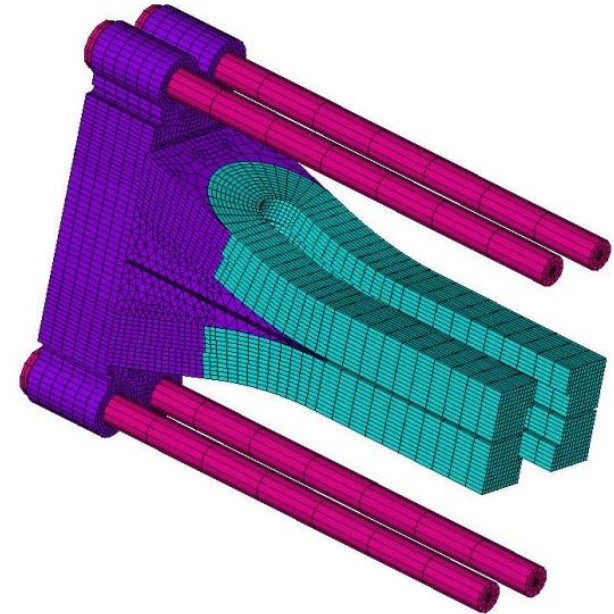
# Support structure

- Horizontal bladders pressurized to 30 MPa
  - Insertion of a shim in horizontal load keys of 0.6 mm
- Shell  $\sigma$  from 65 (293 K) to 185 (4.2 K) MPa
- Structure capable to withstand ultimate field (15 T)
- 2D model (13 T)
  - no coil-pole separation and  $\sigma_{\text{coil}} \sim 140$  MPa



# End forces and support

- Axial force: 2.8 MN
- Axial piston used to pre-load the rods
  - 60 mm diameter
- Room temperature pre-load
  - Rod stress: 150 MPa
  - 1.7 MN (170 t) provided by 200 t piston
- 4.2 K pre-load
  - Rod stress: 260 MPa
  - 2.8 MN (280 t) to end-shoes and wedge (glued)
- Alternative option
  - Axial force from rod to yoke (bullets on coil/wedge)
    - Infinitely rigid condition



# Conclusions

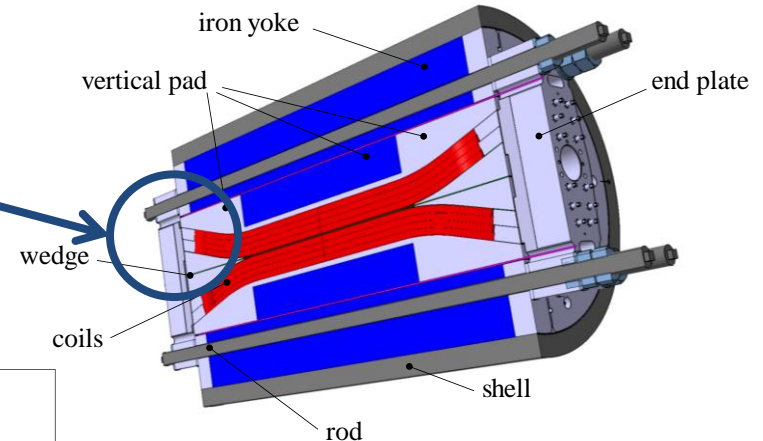
---

- FRESCA2 development from design to fabrication/assembly phase
  - Goal: 13 T in 100 mm bore
- According to measured strand properties, the magnet operates with more than 20% of current margin
- Coil peak stress below 150 MPa during all magnet operations
- Support structure capable of providing full pre-load (up to 15 T) in straight section, ramp, and end region

# Study in progress: an overview

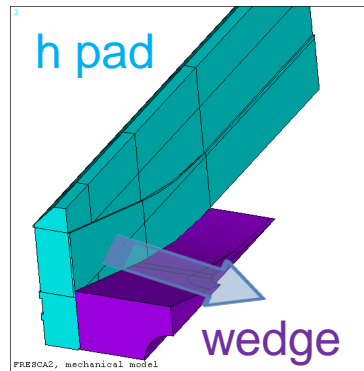
## Wedge:

- Impregnated with coil 1-2?
- Or not impregnated?



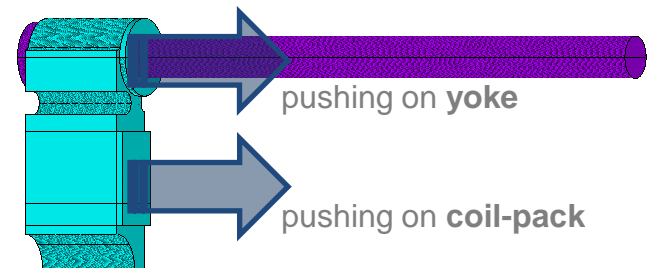
## Lateral support:

- Contact in **h-pad** / **wedge**?
- Or no contact?



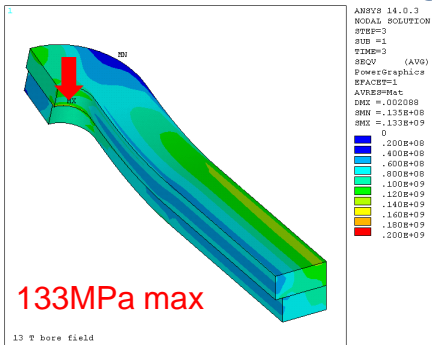
## Axial loading:

- Pushing on the **coil-pack** only
- Pushing on the **coil-pack** and **yoke**

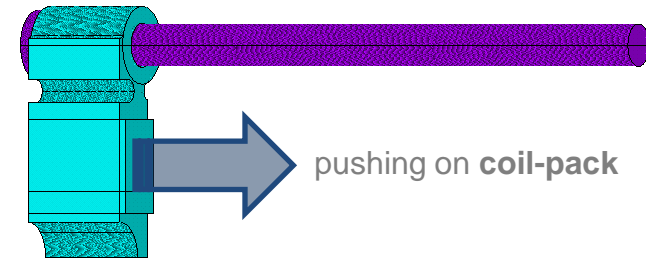
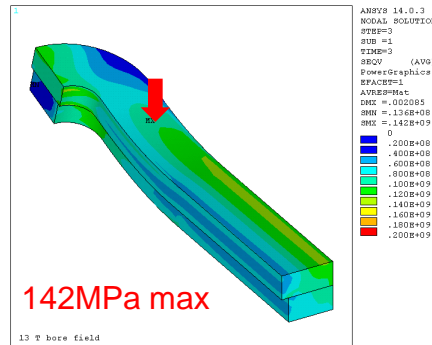


# Cases of study: impregnation of the wedge

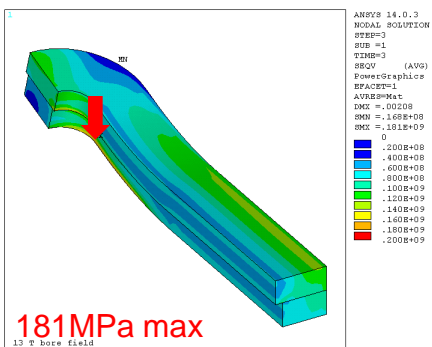
Case 1: Wedge **non** impregnated  
contact on the wedge



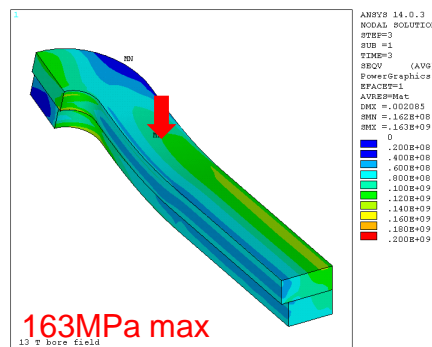
Case 2: Wedge **non** impregnated  
**no** contact on the wedge



Case 3: Wedge **impregnated**  
contact on the wedge



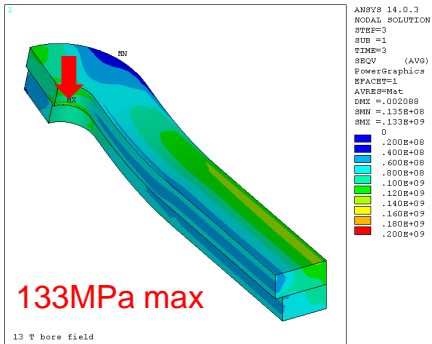
Case 4: Wedge **impregnated**  
**no** contact on the wedge



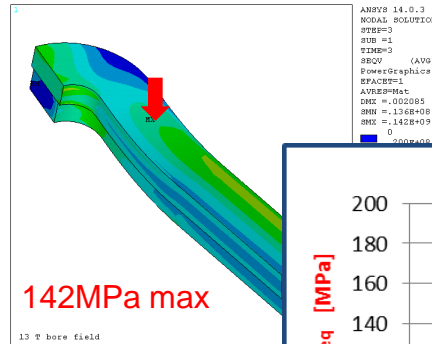


# Cases of study: impregnation of the wedge

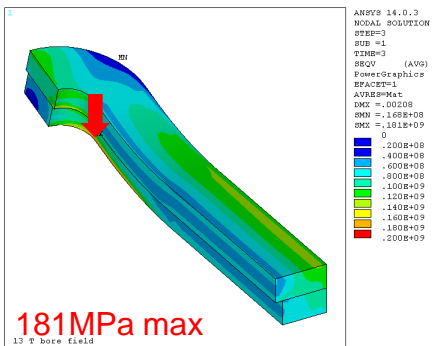
Case 1: Wedge **non** impregnated  
contact on the wedge



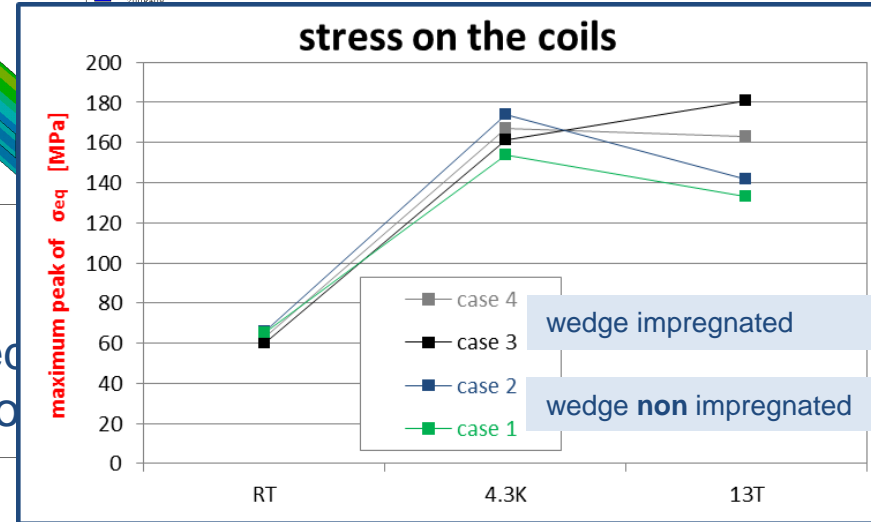
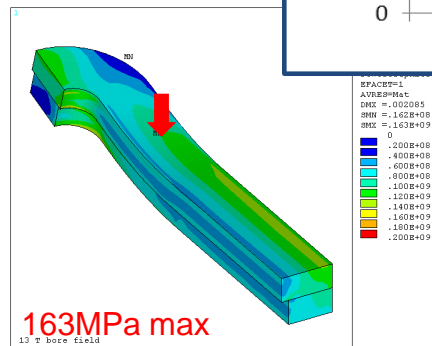
Case 2: Wedge **non** impregnated  
**no** contact on the wedge



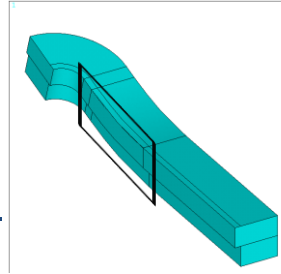
Case 3: Wedge **impregnated**  
contact on the wedge



Case 4: Wedge **impregnated**  
**no** contact on the wedge

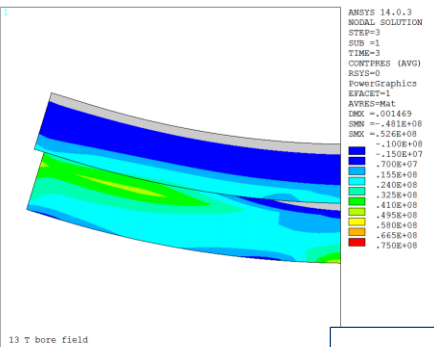


# Cases of study: lateral support

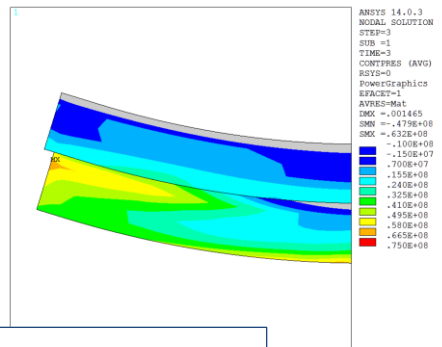


in-side  
hard-way-bend  
@13T

Case 1: Wedge **non** impregnated  
contact on the wedge

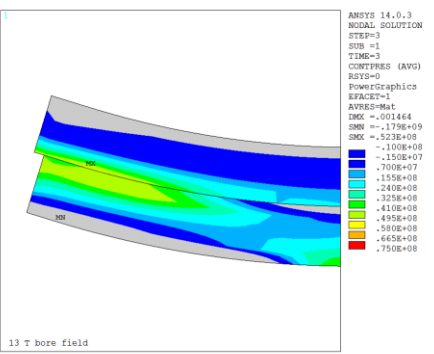


Case 2: Wedge **non** impregnated  
**no** contact on the wedge

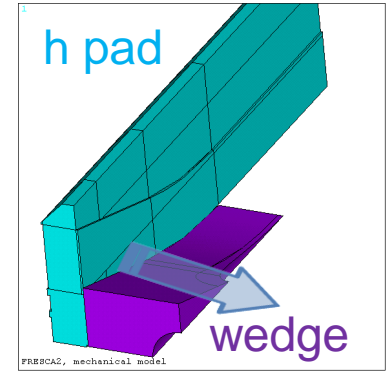
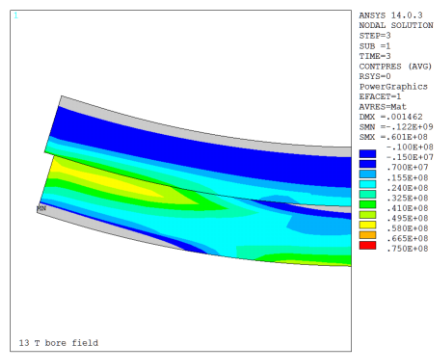


colored area: compression  
gray area: <10MPa in tension

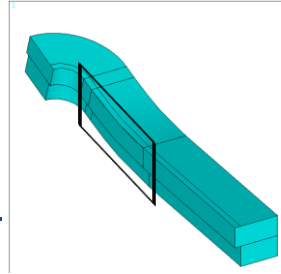
Case 3: Wedge **impregnated**  
contact on the wedge



Case 4: Wedge **impregnated**  
**no** contact on the wedge

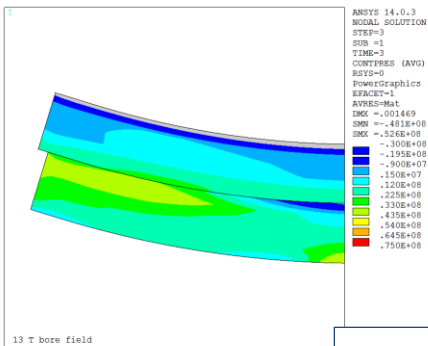


# Cases of study: lateral support

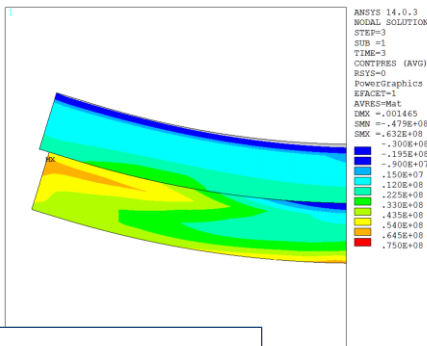


in-side  
hard-way-bend  
@13T

Case 1: Wedge **non** impregnated  
contact on the wedge

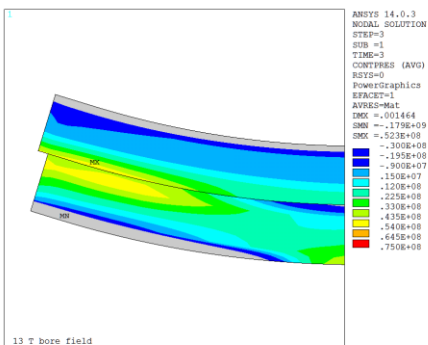


Case 2: Wedge **non** impregnated  
**no** contact on the wedge

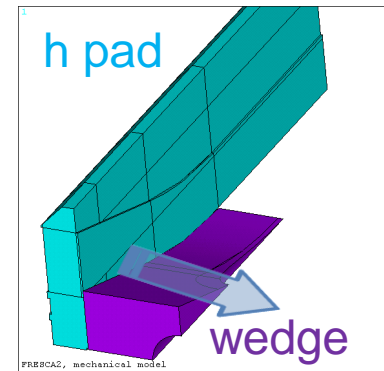
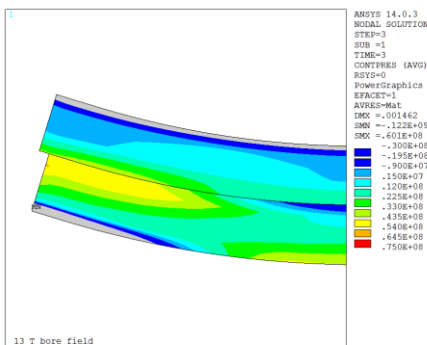


colored area: compression  
gray area: <30MPa in tension

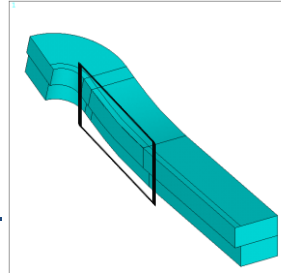
Case 3: Wedge **impregnated**  
contact on the wedge



Case 4: Wedge **impregnated**  
**no** contact on the wedge

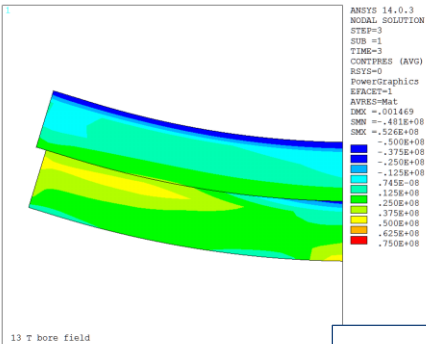


# Cases of study: lateral support

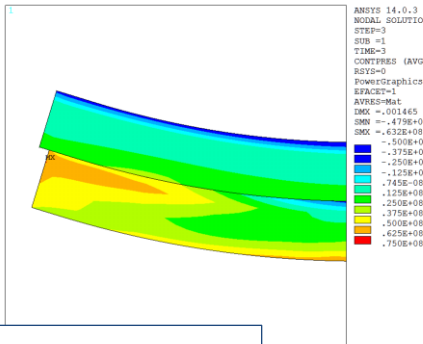


in-side  
hard-way-bend  
@13T

Case 1: Wedge **non** impregnated  
contact on the wedge

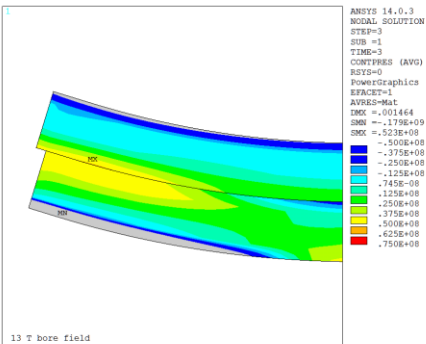


Case 2: Wedge **non** impregnated  
**no** contact on the wedge

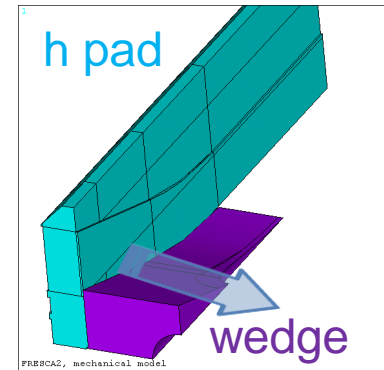
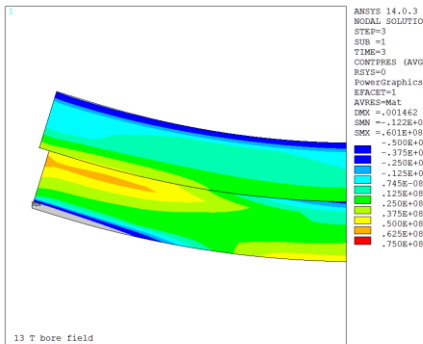


colored area: compression  
gray area: <50MPa in tension

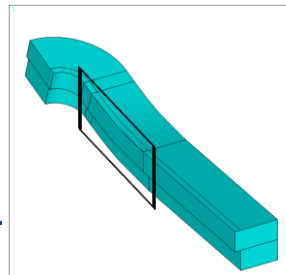
Case 3: Wedge **impregnated**  
contact on the wedge



Case 4: Wedge **impregnated**  
**no** contact on the wedge

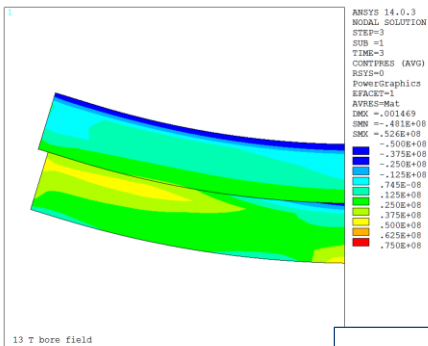


# Cases of study: lateral support

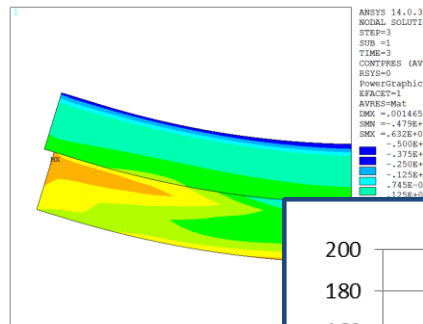


in-side  
hard-way-bend  
@13T

Case 1: Wedge **non** impregnated  
contact on the wedge

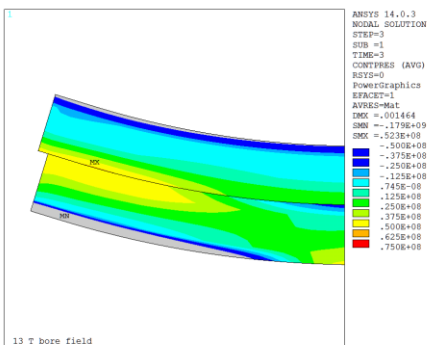


Case 2: Wedge **non** impregnated  
**no** contact on the wedge

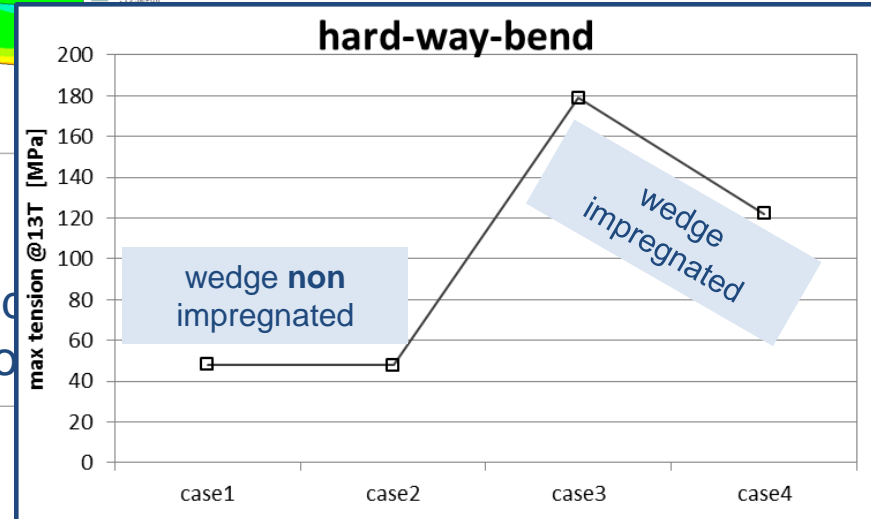
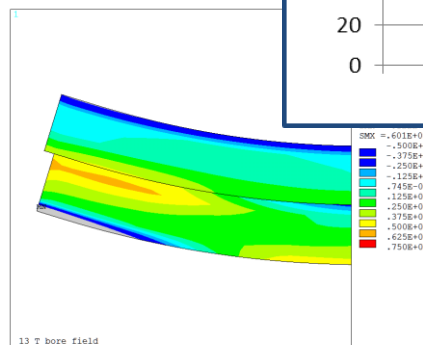


colored area: compression  
gray area: <math><50\text{MPa}</math> in tension

Case 3: Wedge **impregnated**  
contact on the wedge

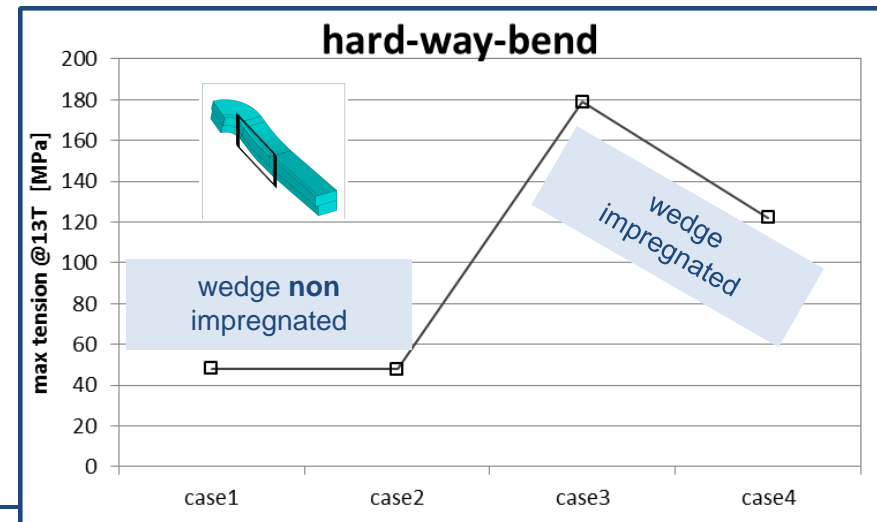
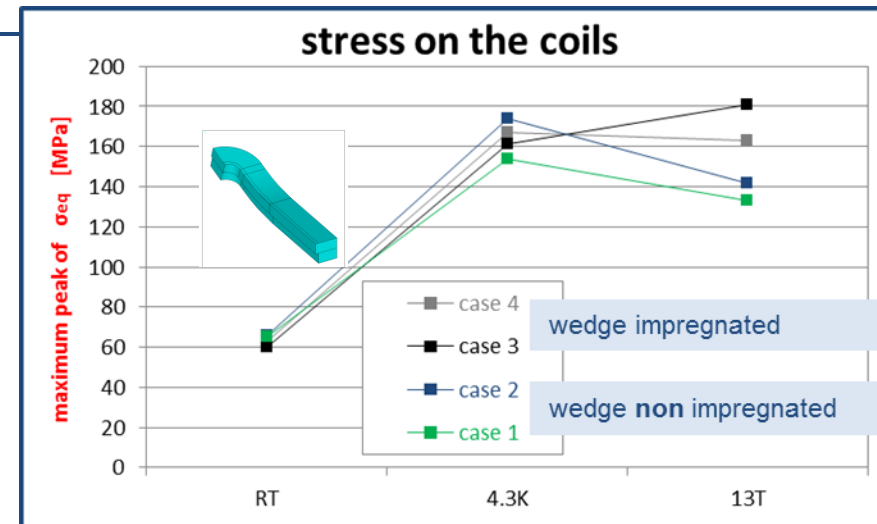


Case 4: Wedge **impregnated**  
**no** contact on the wedge



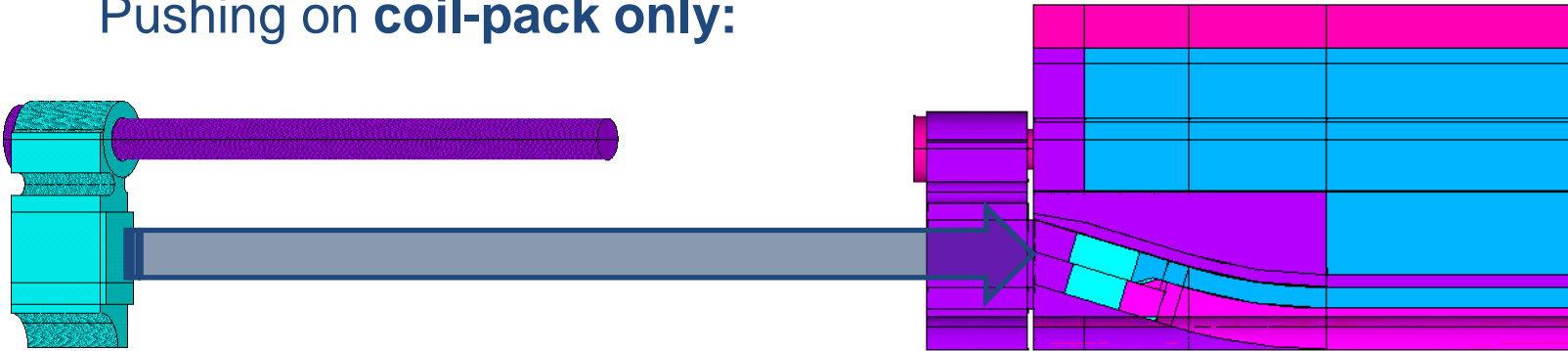
# Cases of study on wedge: preliminary conclusions

- ❑ Lower max stress peak in the coil with the **wedge not impregnated**
- ❑ Lower contact tension in the *hard-way-bend* side of the coil with the **wedge not impregnated**
- ❑ With the wedge not impregnated
  - ❑ Contact tension is slightly lower when there is no contact on the wedge (no lateral support )
  - ❑ Coil peak stress slightly lower with lateral contact with the wedge

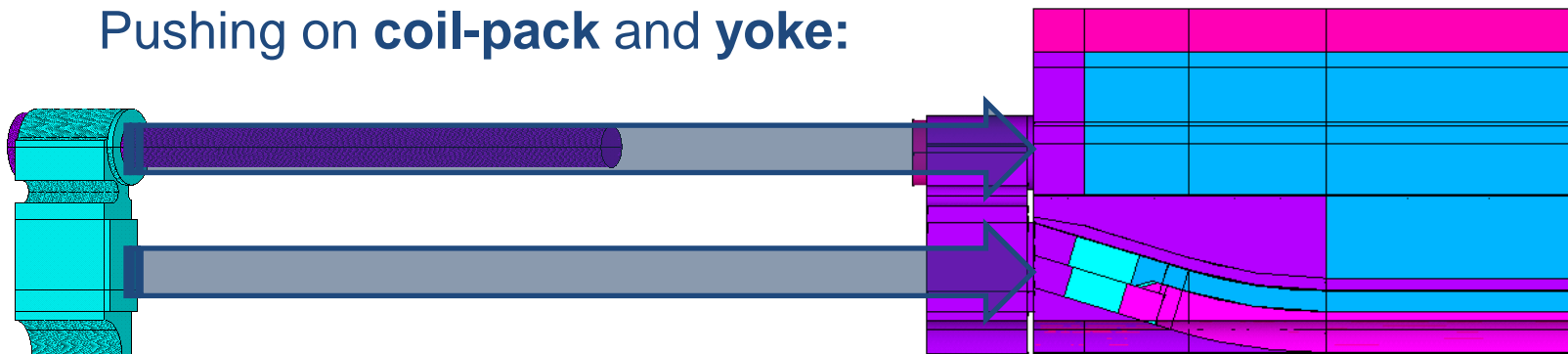


# Cases of study: axial loading

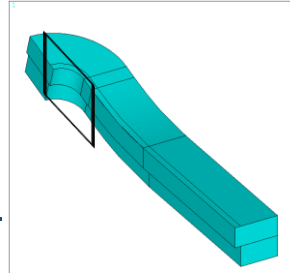
Pushing on **coil-pack only**:



Pushing on **coil-pack and yoke**:

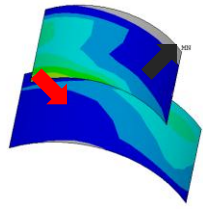


# Cases of study: axial loading



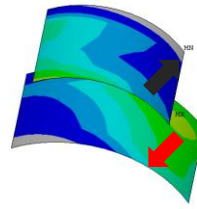
Case 1: Wedge **non** impregnated  
contact on the wedge

Case 2: Wedge **non** impregnated  
**no** contact on the wedge



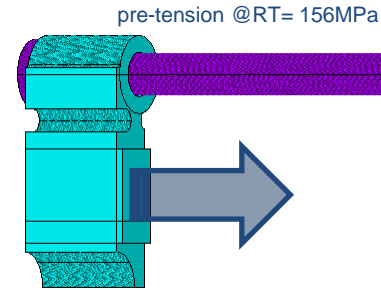
77.9MPa / 37.0MPa

	forces [kN]	
	293K	4.3K
endshoe1	-106	-176
endshoe2	-326	-529
wedge	-5	-41
total	-437	-746
rod	437	740

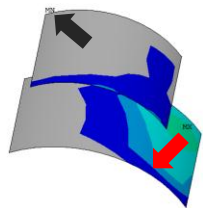


72.2MPa / 29.0MPa

	forces [kN]	
	293K	4.3K
endshoe1	-109	-196
endshoe2	-320	-516
wedge	-7	-27
total	-436	-739
rod	437	740



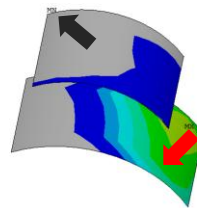
pushing on **coil-pack**



52MPa / 97MPa

No contact end-plate - coil

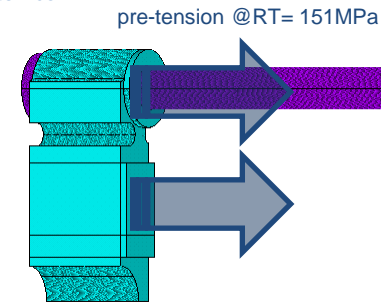
	forces [kN]	
	293K	4.3K
endshoe1	-31	0
endshoe2	-52	0
wedge	-16	0
yoke	-328	-882
total	-427	-882
rod	427	880



77MPa / 96MPa

No contact end-plate - coil

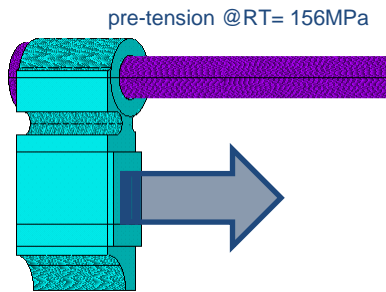
	forces [kN]	
	293K	4.3K
endshoe1	-40	0
endshoe2	-52	0
wedge	-10	0
yoke	-323	-878
total	-425	-878
rod	426	878



pushing on **coil-pack**  
and **yoke**



# Cases of study on axial load: preliminary conclusions

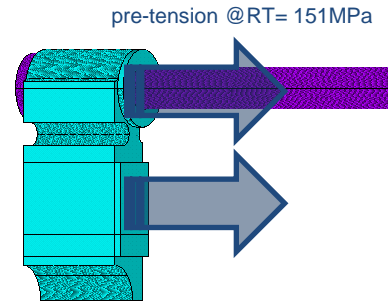


pushing on **coil-pack**

- At 293 K most of the force on endshoe2
  - Still contact with endshoe1 and wedge
- At 4.2 K, similar force distribution
- At 13 T, lower contact tension coil-pole

- **Next step**

- Try to simulate situation with contact with end-plate and yoke+coil after cool-down



pushing on **coil-pack**  
and **yoke**

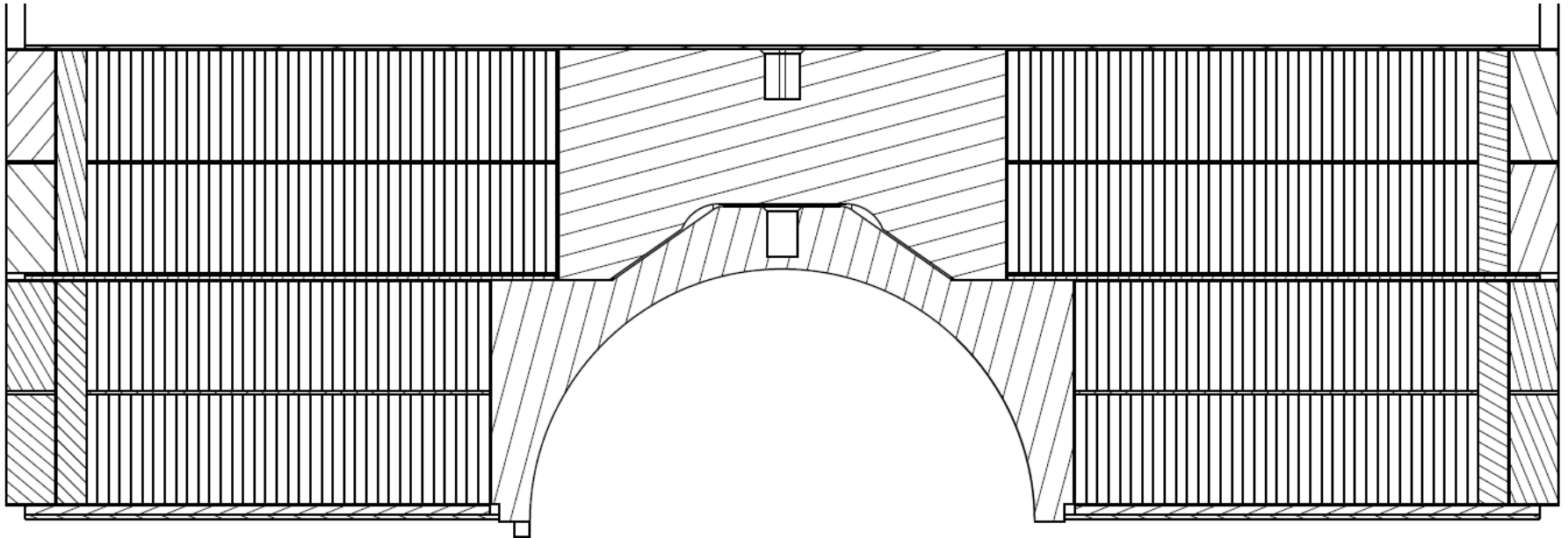
- At 293 K most of the force on the yoke
  - Still contact with coil and wedge
- At 4.2 K, loss of contact endplate-coil
- At 13 T, higher contact tension coil-pole

# Appendix

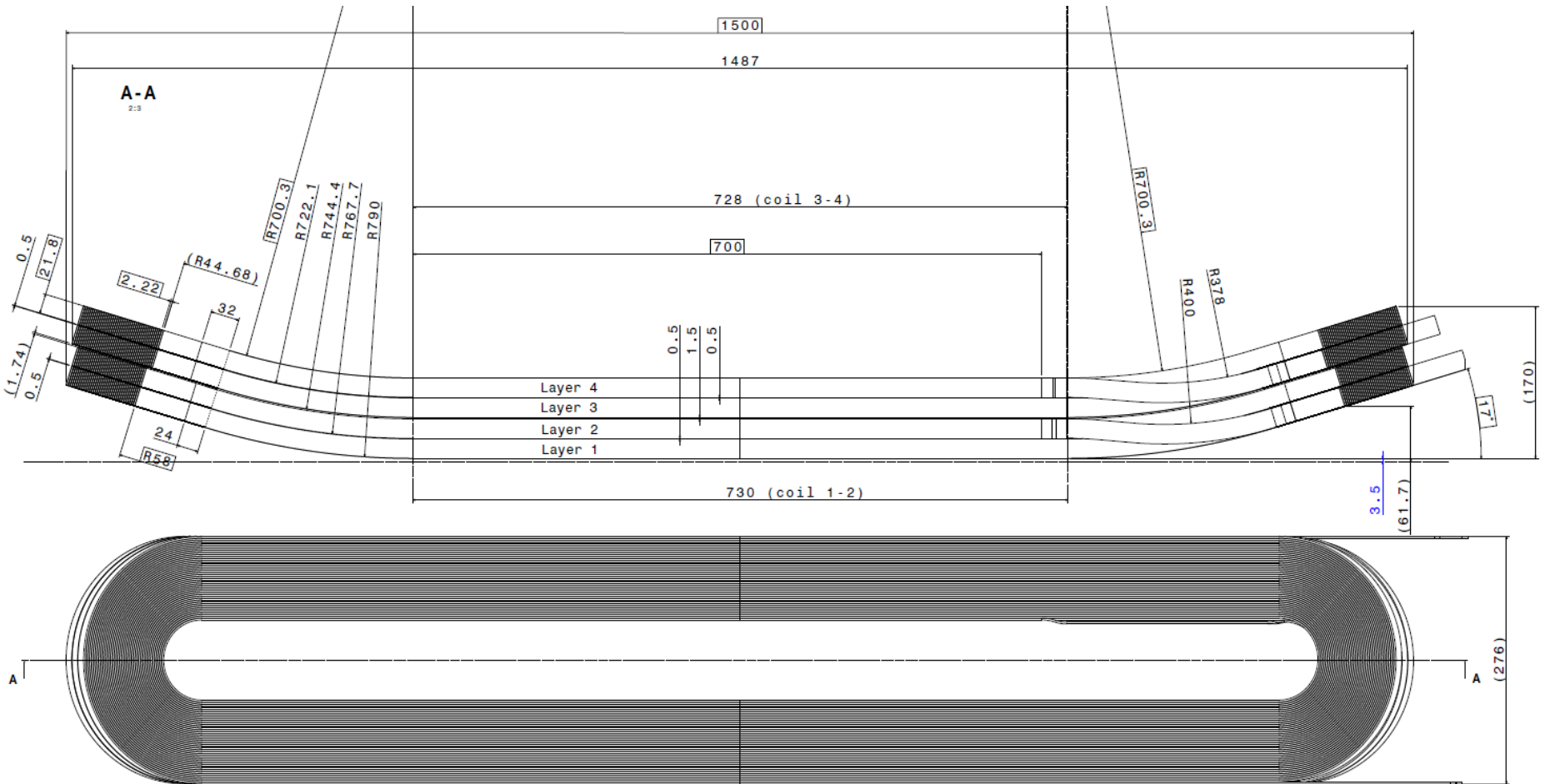
---

# Coil cross-section

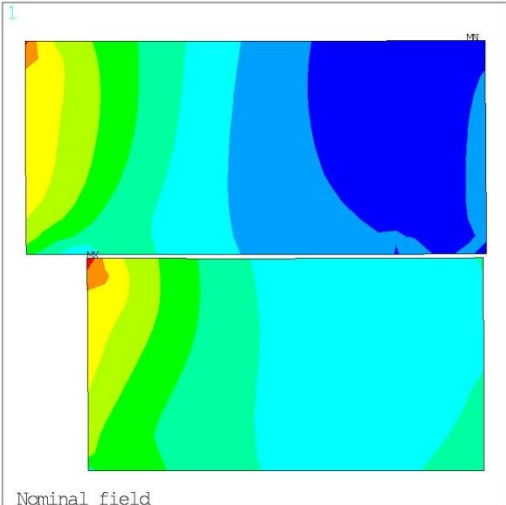
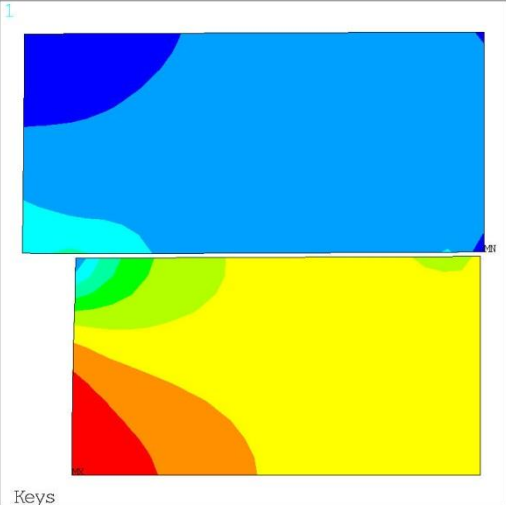
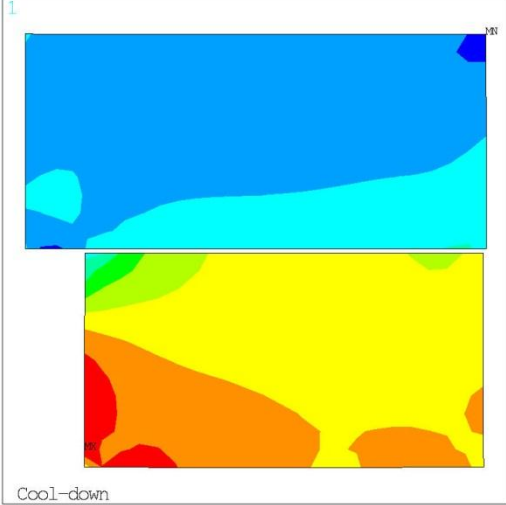
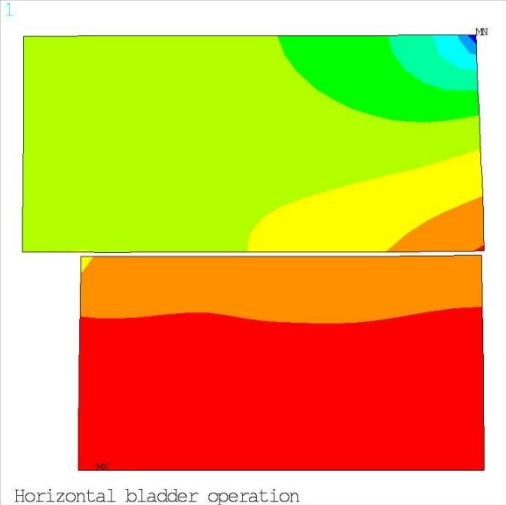
---



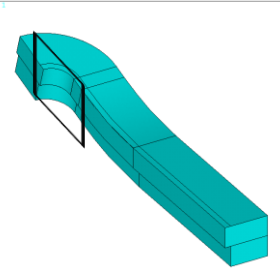
# Coil 3D design



# Coil stresses (MPa)

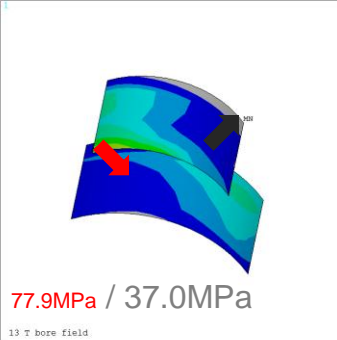


# Cases of study: pushing on the coil-pack only



**in-side  
easy-way-bend  
@13T**

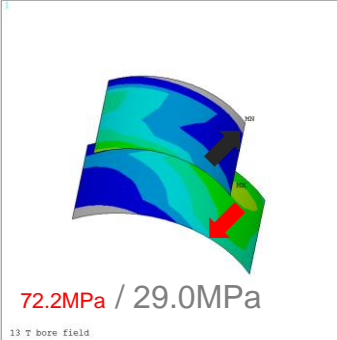
**Case 1: Wedge **non** impregnated  
contact on the wedge**



```
ANSYS 14.0.3
MODEL SOLUTION
STEP=3
SUB =1
TIME=3
CONTRES (AVG)
RESU=0
PowerGraphics
EFACET=1
AVRES=Max
DMX =.001558
SMN =-.370E+08
SMX =-.778E+08
0
-.110E+08
-.220E+08
-.330E+08
-.440E+08
-.550E+08
-.660E+08
-.770E+08
-.880E+08
-.990E+08
.110E+08
```

77.9MPa / 37.0MPa

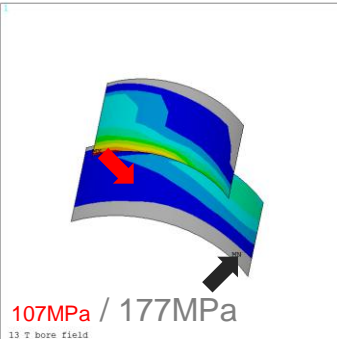
**Case 2: Wedge **non** impregnated  
no contact on the wedge**



```
ANSYS 14.0.3
MODEL SOLUTION
STEP=3
SUB =1
TIME=3
CONTRES (AVG)
RESU=0
PowerGraphics
EFACET=1
AVRES=Max
DMX =.001554
SMN =-.290E+08
SMX =-.722E+08
0
-.110E+08
-.220E+08
-.330E+08
-.440E+08
-.550E+08
-.660E+08
-.770E+08
-.880E+08
-.990E+08
.110E+08
```

72.2MPa / 29.0MPa

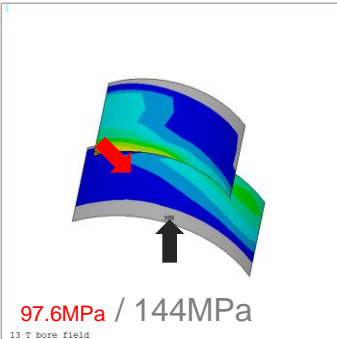
**Case 3: Wedge **impregnated**  
contact on the wedge**



```
ANSYS 14.0.3
MODEL SOLUTION
STEP=3
SUB =1
TIME=3
CONTRES (AVG)
RESU=0
PowerGraphics
EFACET=1
AVRES=Max
DMX =.001552
SMN =-.177E+09
SMX =-.107E+09
0
-.110E+08
-.220E+08
-.330E+08
-.440E+08
-.550E+08
-.660E+08
-.770E+08
-.880E+08
-.990E+08
.110E+08
```

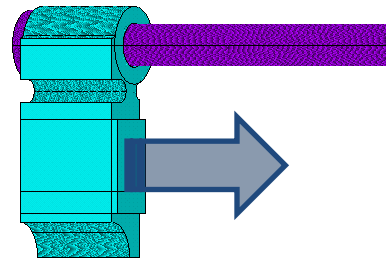
107MPa / 177MPa

**Case 4: Wedge **impregnated**  
no contact on the wedge**



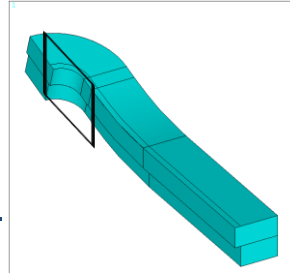
```
ANSYS 14.0.3
MODEL SOLUTION
STEP=3
SUB =1
TIME=3
CONTRES (AVG)
RESU=0
PowerGraphics
EFACET=1
AVRES=Max
DMX =.001551
SMN =-.144E+09
SMX =-.576E+08
0
-.110E+08
-.220E+08
-.330E+08
-.440E+08
-.550E+08
-.660E+08
-.770E+08
-.880E+08
-.990E+08
.110E+08
```

97.6MPa / 144MPa



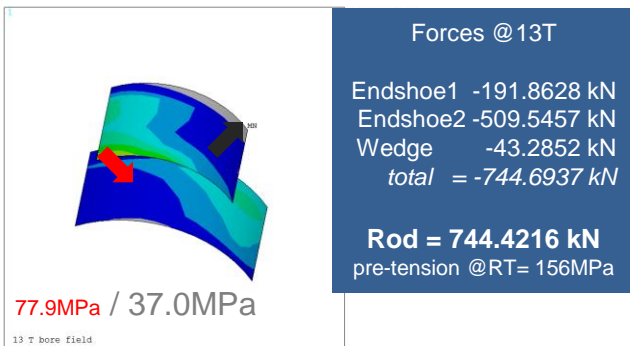
pushing on coil-pack

# Cases of study: axial loading



Case 1: Wedge **non** impregnated  
contact on the wedge

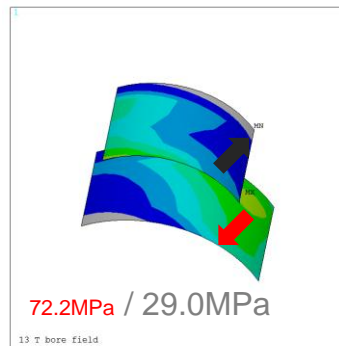
Case 2: Wedge **non** impregnated  
**no** contact on the wedge



**Forces @13T**

Endshoe1	-191.8628 kN
Endshoe2	-509.5457 kN
Wedge	-43.2852 kN
<b>total</b>	<b>-744.6937 kN</b>

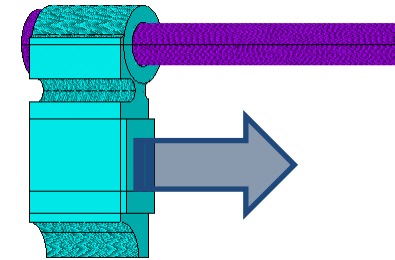
**Rod = 744.4216 kN**  
pre-tension @RT= 156MPa



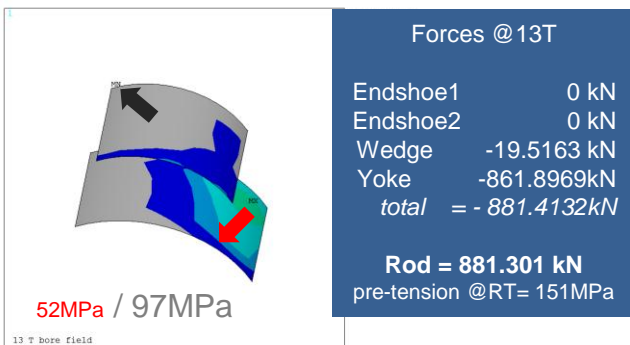
**Forces @13T**

Endshoe1	-210.6606 kN
Endshoe2	-489.4995 kN
Wedge	-44.444 kN
<b>total</b>	<b>-744.6041 kN</b>

**Rod = 744.4396 kN**  
pre-tension @RT= 156MPa



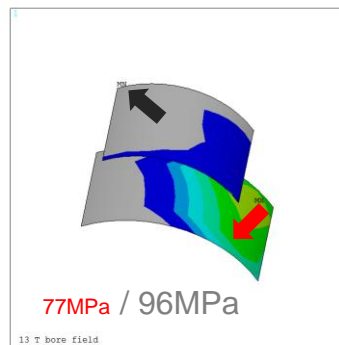
pushing on **coil-pack**



**Forces @13T**

Endshoe1	0 kN
Endshoe2	0 kN
Wedge	-19.5163 kN
Yoke	-861.8969 kN
<b>total</b>	<b>-881.4132 kN</b>

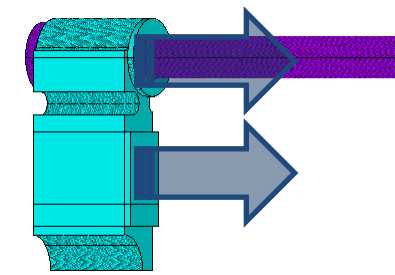
**Rod = 881.301 kN**  
pre-tension @RT= 151MPa



**Forces @13T**

Endshoe1	-3.75244 kN
Endshoe2	0 kN
Wedge	-40.1981 kN
Yoke	-835.9605 kN
<b>total</b>	<b>-879.91112 kN</b>

**Rod = 879.7984 kN**  
pre-tension @RT= 151MPa



pushing on **coil-pack**  
and **yoke**