



TWEPP-2008

Topical Workshop on Electronics for Particle Physics

CO₂ Cooling for HEP experiments

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Naxos, 18 September 2008

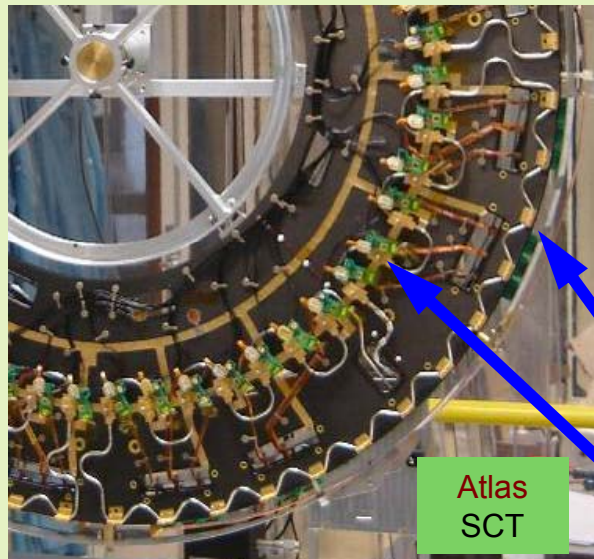


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- Introduction to evaporative CO_2 -Cooling
- Introduction to the LHCb Velo Thermal Control System (VTCS)
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- Conclusions and outlook.

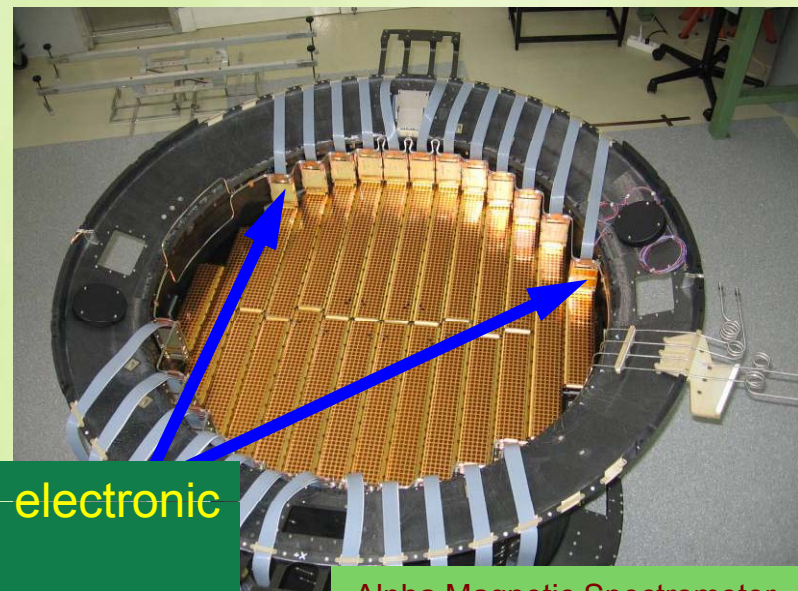
(Silicon) Particle Detectors and Cooling

- (Silicon) Particle detectors have specific needs for thermal control:
 - Many distributed heat sources over large volumes.
 - Low temperature gradients between these sources.
 - Permanent cooling
 - *To avoid thermal runaway of the irradiated silicon*
 - Cooling pipes should have low mass inside detectors
 - Cooling pipes should have low structural impact
 - Radiation resistant cooling fluid



Atlas
SCT

Multiple electronic
stations
(All need cooling)



Alpha Magnetic Spectrometer
Silicon Tracker

Why Evaporative CO_2 Cooling?

The lightest way of cooling is:

Evaporate at high pressure!

Why?

Vapor expansion is limited under high pressure

Volume stays low

Pipe diameter stays low

Mass stays low

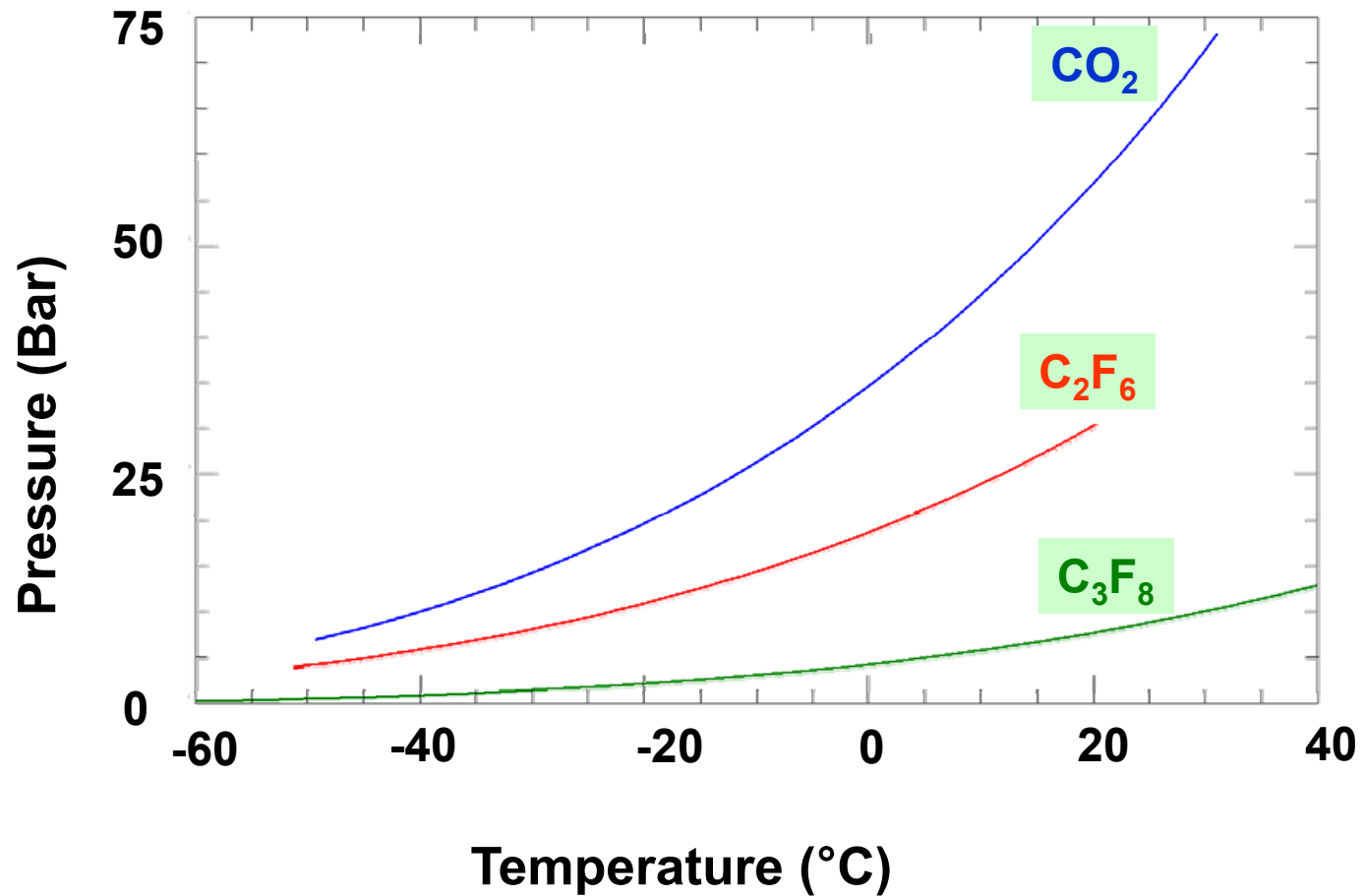
Low mass flow

High latent heat

Carbon Dioxide

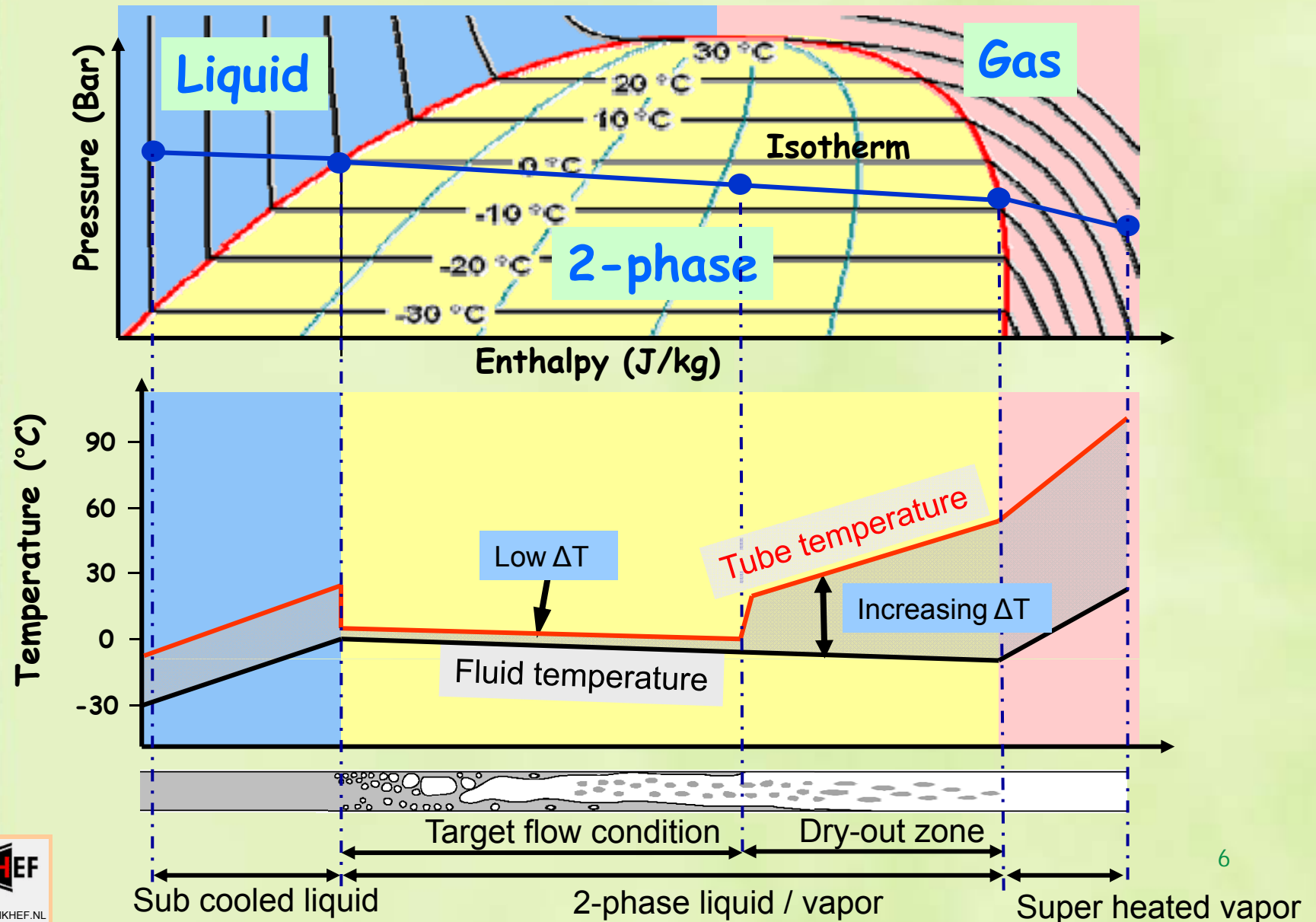
Later on in this presentation this will be illustrated by calculations

Saturation curves in the PT Diagram for CO_2 , C_2F_6 & C_3F_8 (3 used or considered refrigerants at CERN)

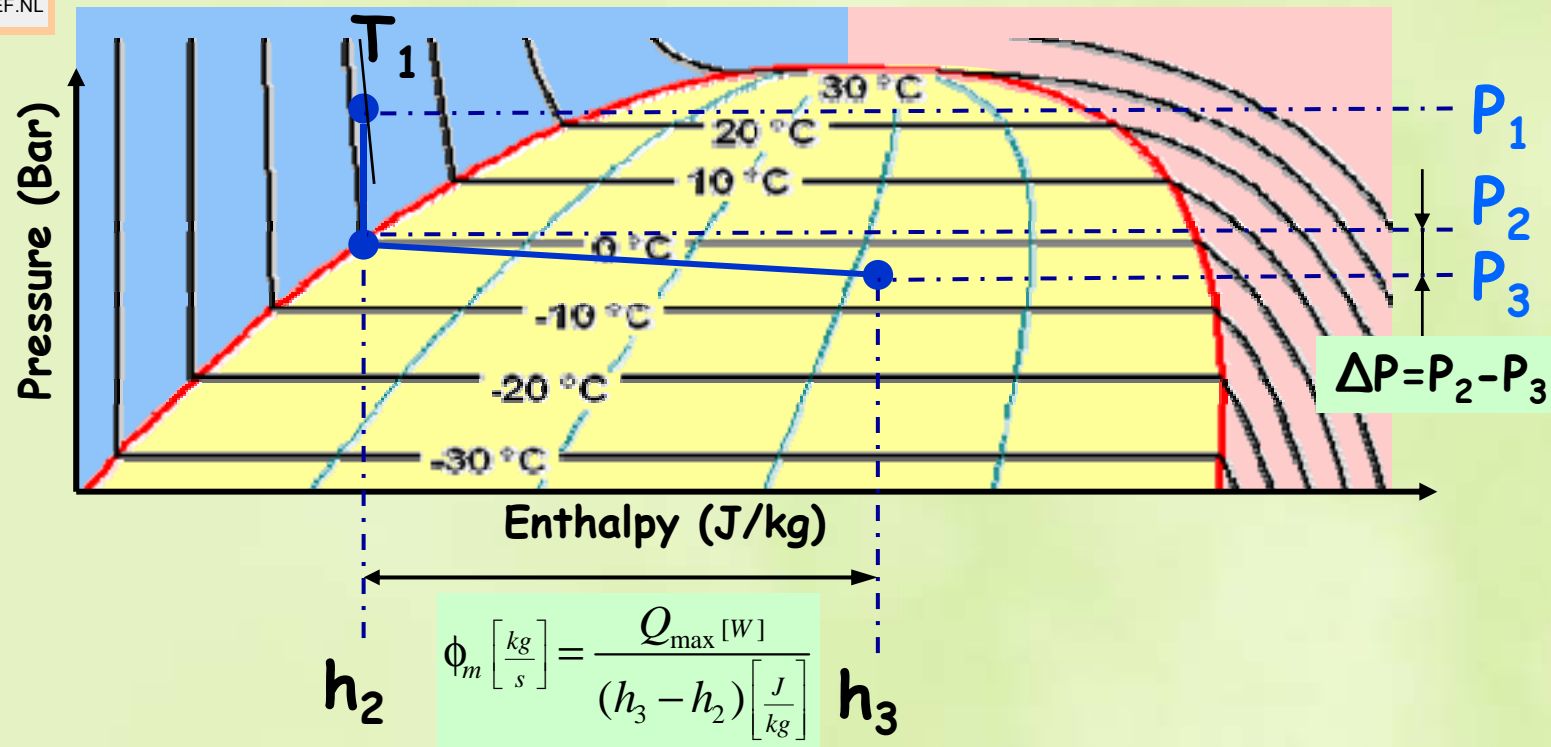


What happens inside a cooling tube?

Heating a flow from liquid to gas



How to design a cooling tube?



For evaporative cooling 5 items are important:

- Controlling inlet enthalpy (h_2) by expanding from liquid (P_1, T_1)
- Setting mass flow (ϕ_m) as function of maximum heat load (Q_{max})
- Controlling outlet pressure (P_3)
- Minimize pressure drop (ΔP)
- Optimize heat transfer (α) with contact area.

2-Phase flow prediction

2-phase flow theory =

Well understood single phase flow theory

×

Magic empirical correlations in a black box

Pressure drop:

$$\Delta P_{2-Phase} = \Delta P_{Liquid} *$$

$$\frac{E + 3.24 * F * H}{Fr^{0.045} * We^{0.035}}$$

For example: Friedel Correlation

Heat transfer:

$$\alpha_{2-Phase} = \alpha_{Liquid} *$$

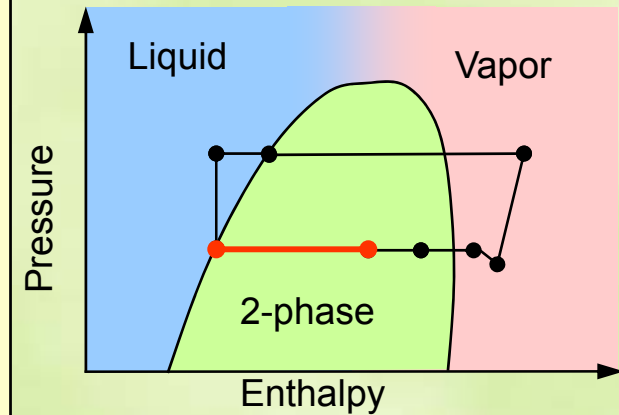
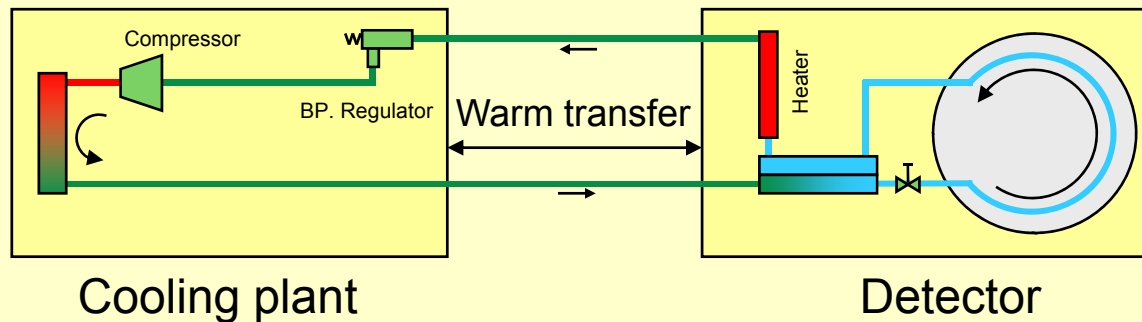
$$\frac{(C_1 * Co^{C_2} * (25 * Fr_{Lo})^{C_5} + C_3 * Bo^{C_4} * F_{fl})}{...}$$

For example: Kandlikar Correlation

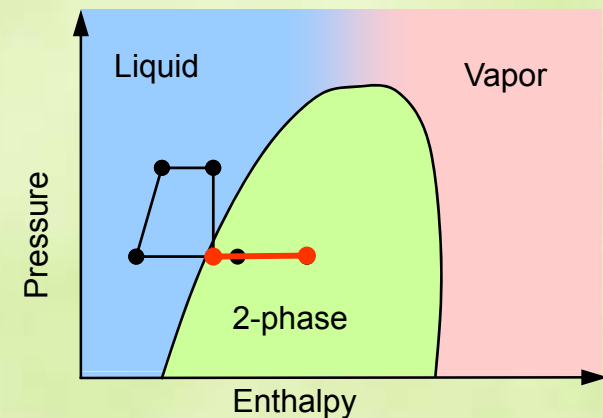
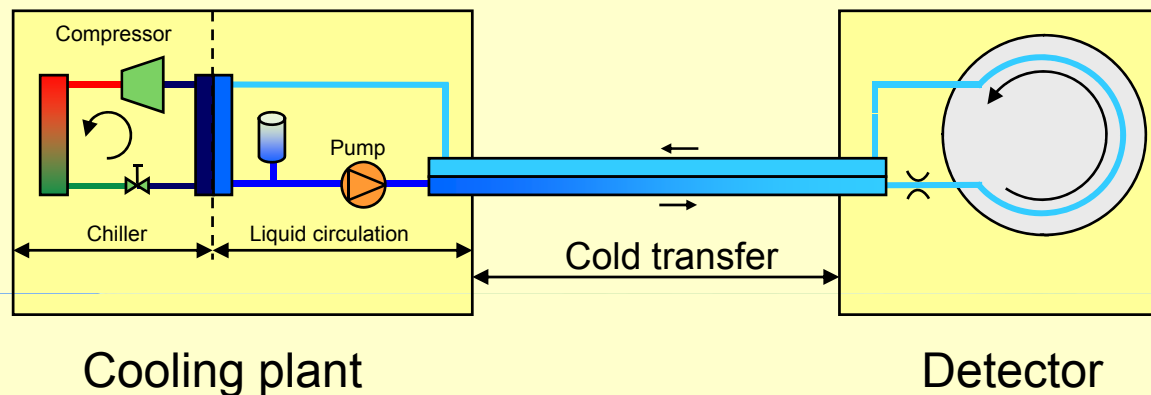
2-Phase coefficient order 1~10

How to get the ideal 2-phase flow in the detector?

Traditional method: **Vapor compression system** (Atlas)



2PACL method: **Pumped liquid system** (LHCb)



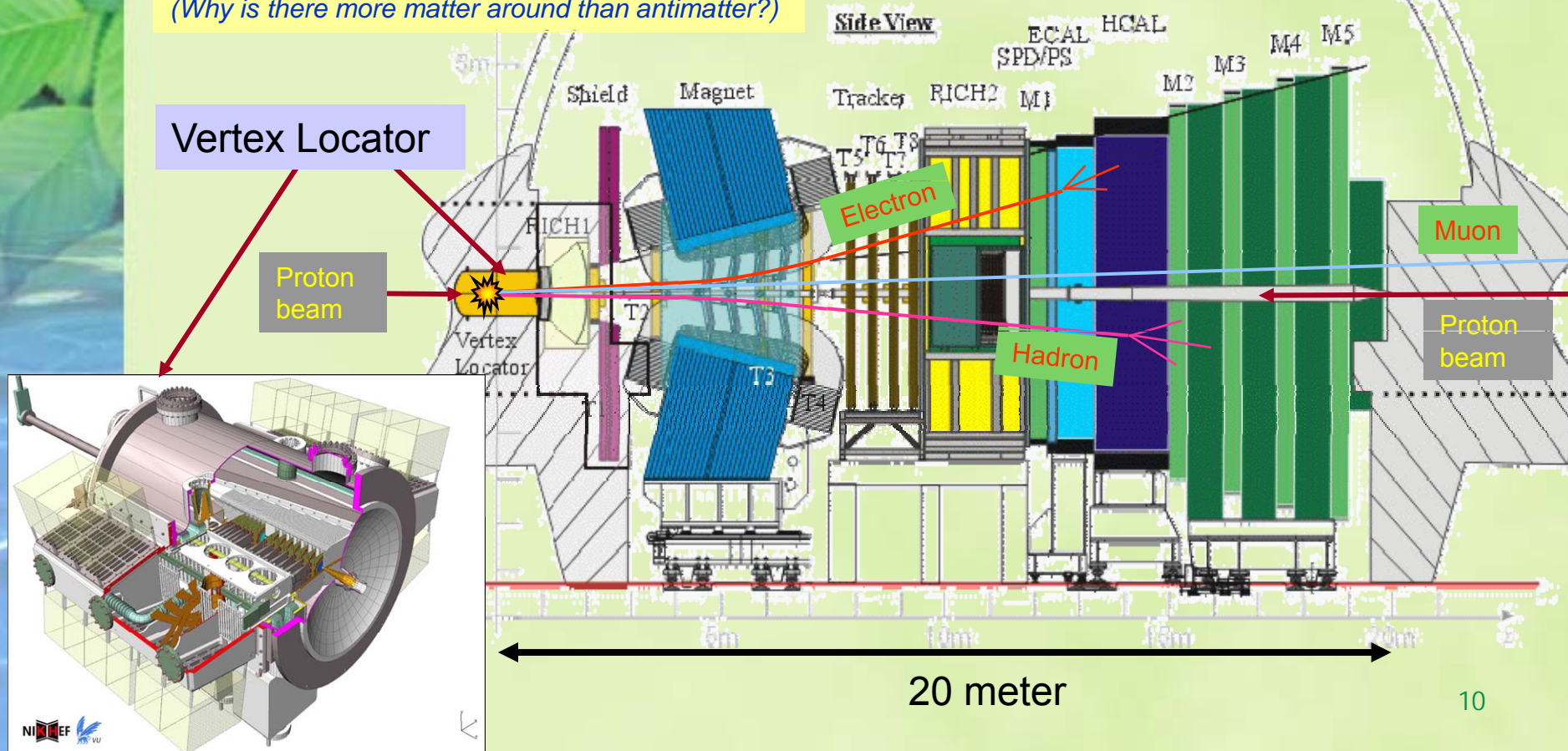
LHCb Detector Overview

Goals of LHCb:

Studying the decay of B-mesons to find evidence of CP-violation

(Why is there more matter around than antimatter?)

LHCb Cross section



The LHCb-VELO Detector (**VE**rtex **LO**cator)

Detectors and
electronics

- Temperature detectors: -7°C
- Heat generation: max 1600 W

23 parallel evaporator stations

capillaries
and return hose

VELO Thermal Control System
 CO_2 evaporator section



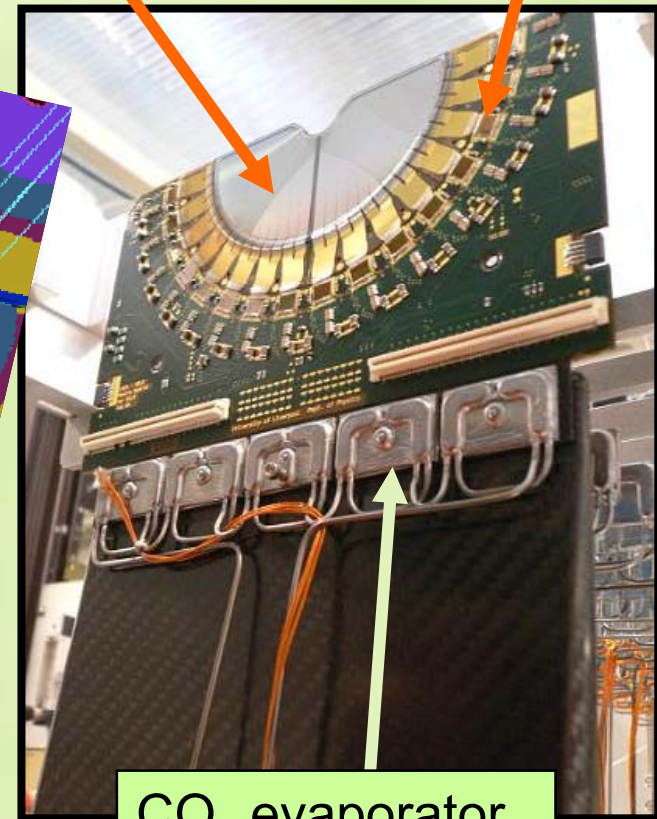
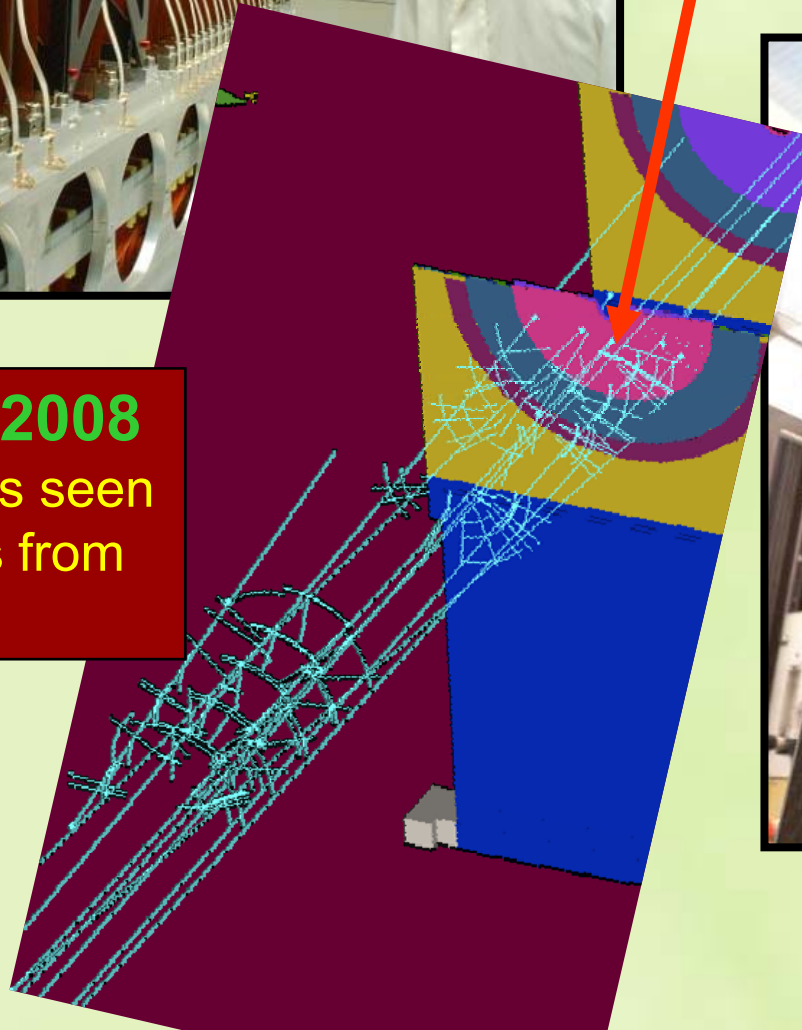
The Velo Detector



Detection Silicon

Heat producing electronics

22 August 2008
The VELO has seen particle tracks from an LHC test!



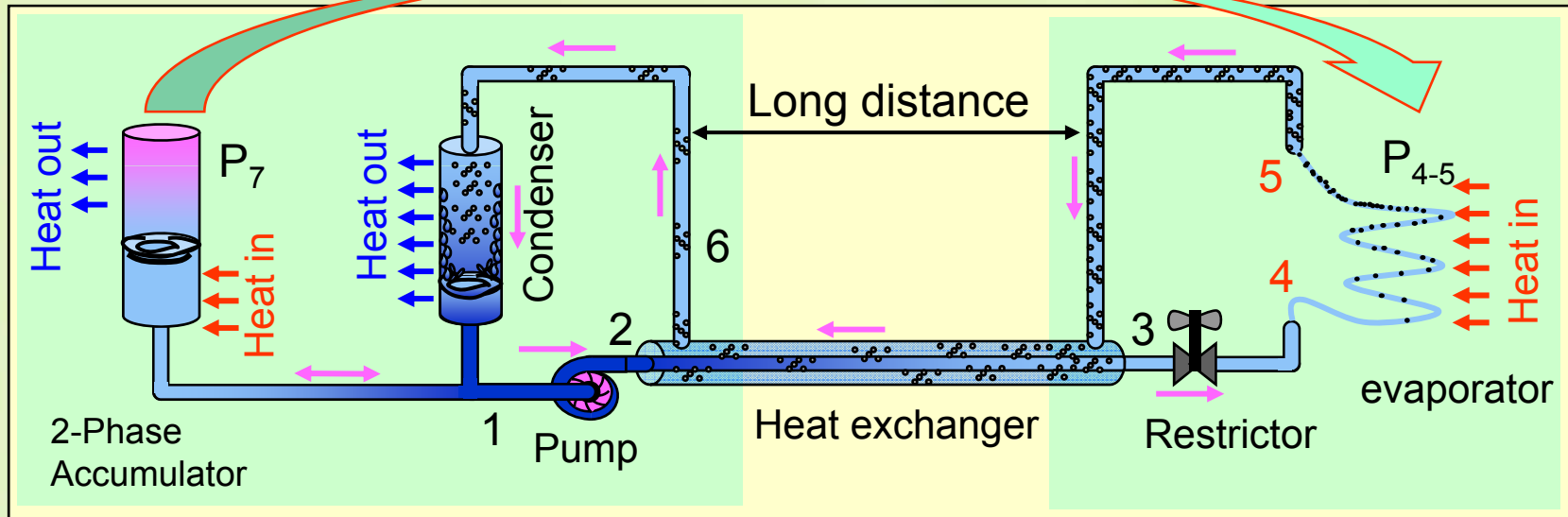
CO₂ evaporator
(Stainless steel tube
casted in aluminum)

VELO Cooling Challenges

- VELO electronics must be cooled in vacuum.
 - Good conductive connection
 - Absolute leak free
- Maximum power of the electronics: 1.6 kW
- Silicon sensors must stay below -7°C at all times (on or off).
 - To avoid thermal runaway of the irradiated silicon
- Adjustable temperature for commissioning.
- Maintenance free in inaccessible detector area

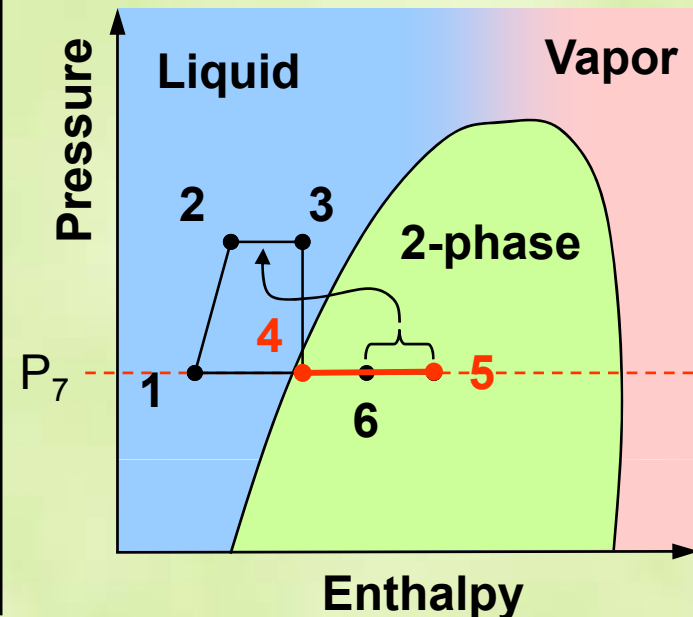


The 2-Phase Accumulator Controlled Loop (2PACL)

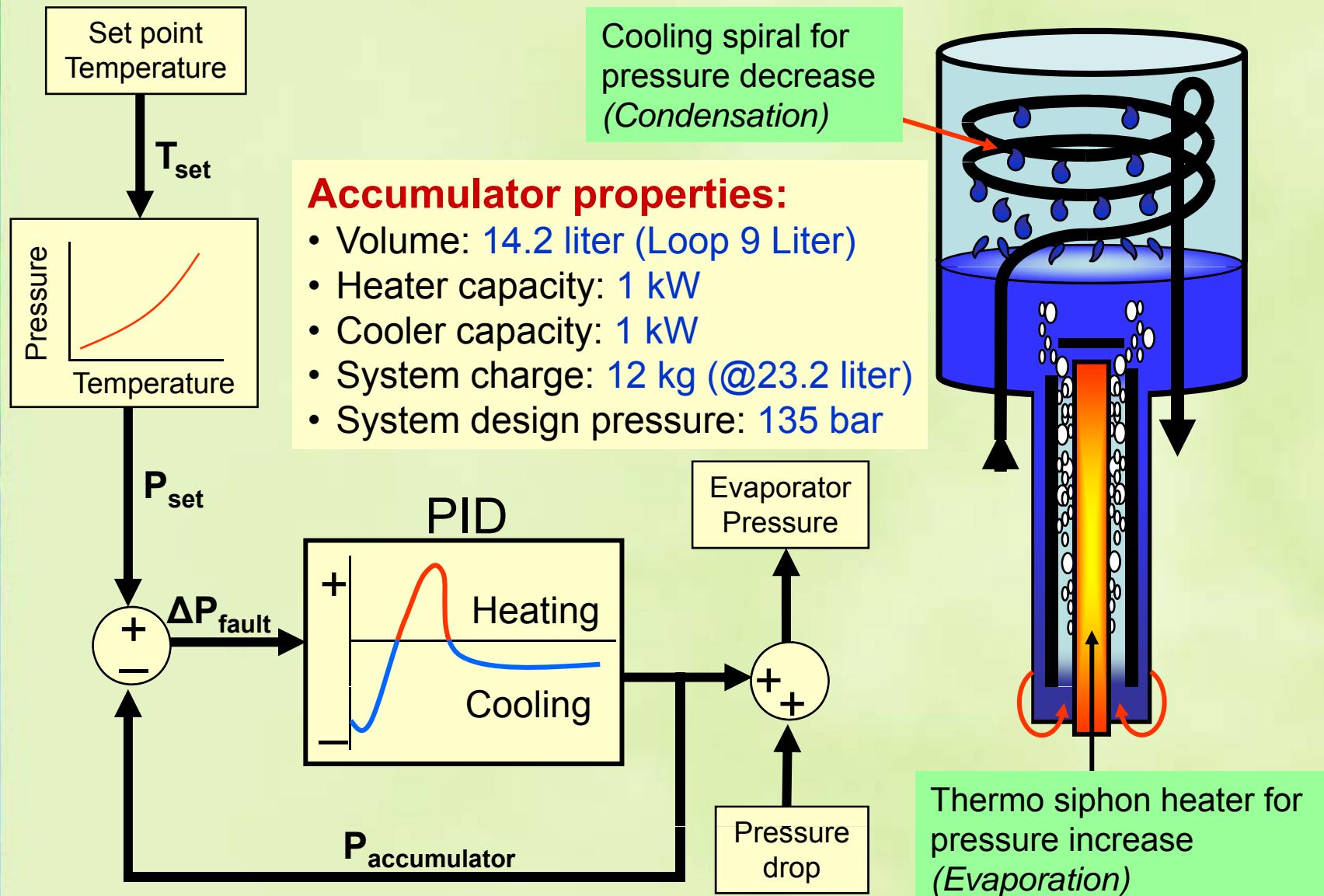


2PACL principle ideal for detector cooling:

- Liquid overflow => no mass flow control
- Low vapor quality => good heat transfer
- No local evaporator control, evaporator is passive in detector.
- Very stable evaporator temperature control at a distance ($P_{4-5} = P_7$)

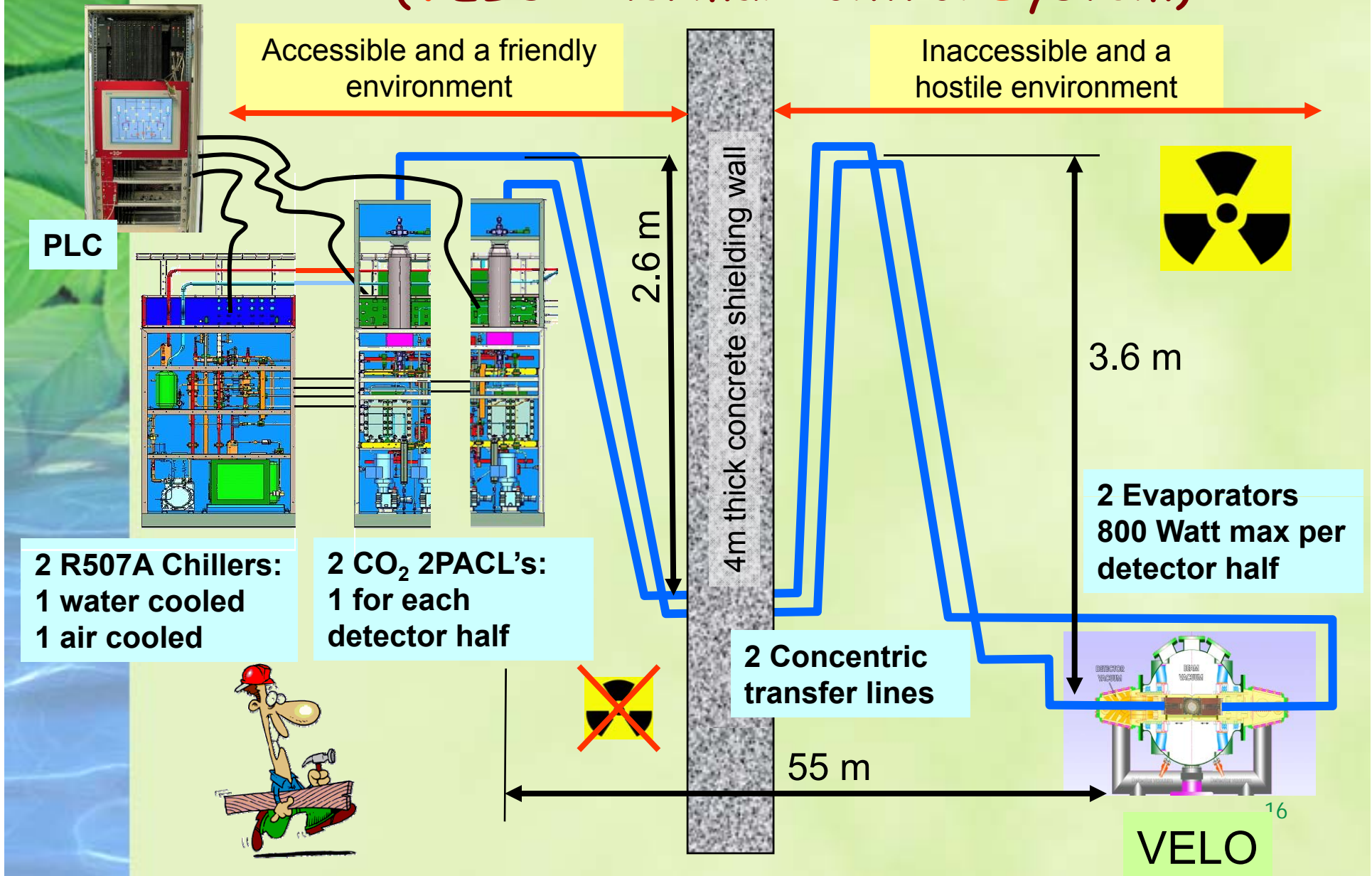


VTCS Accumulator Control



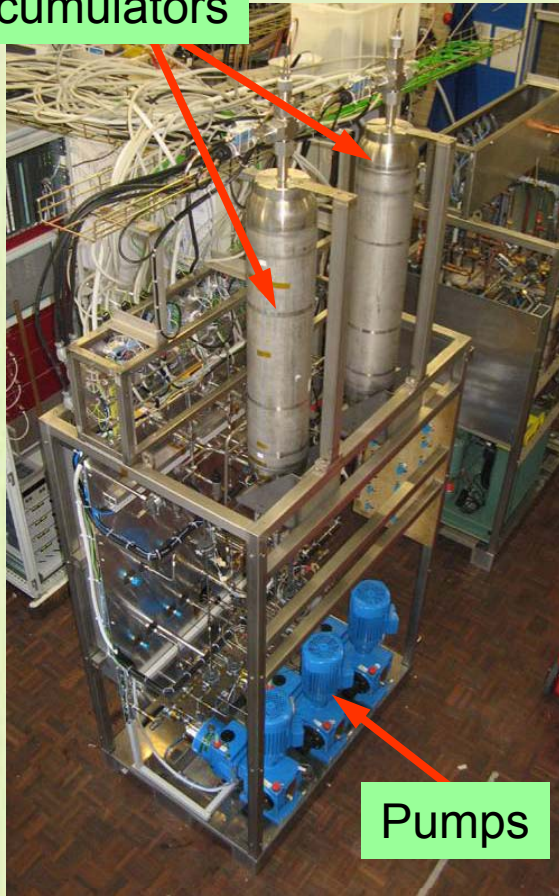
LHCb-VTCS Overview

(VELO Thermal Control System)



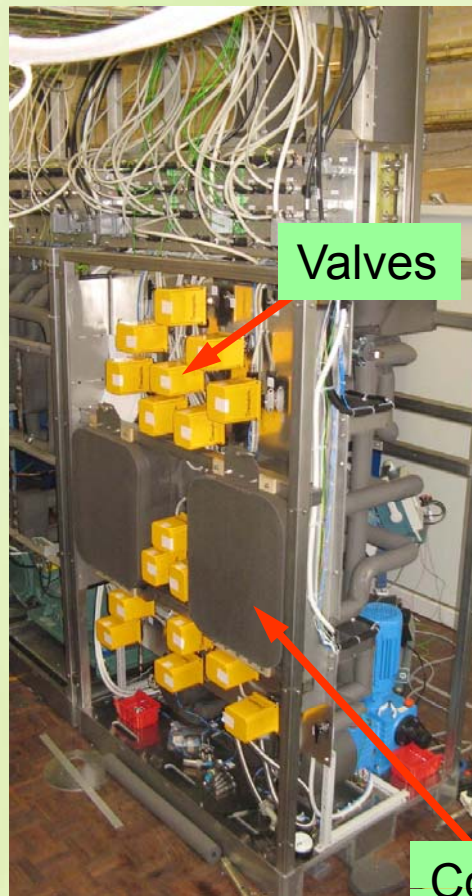
LHCb-VTCS Cooling Components

Accumulators

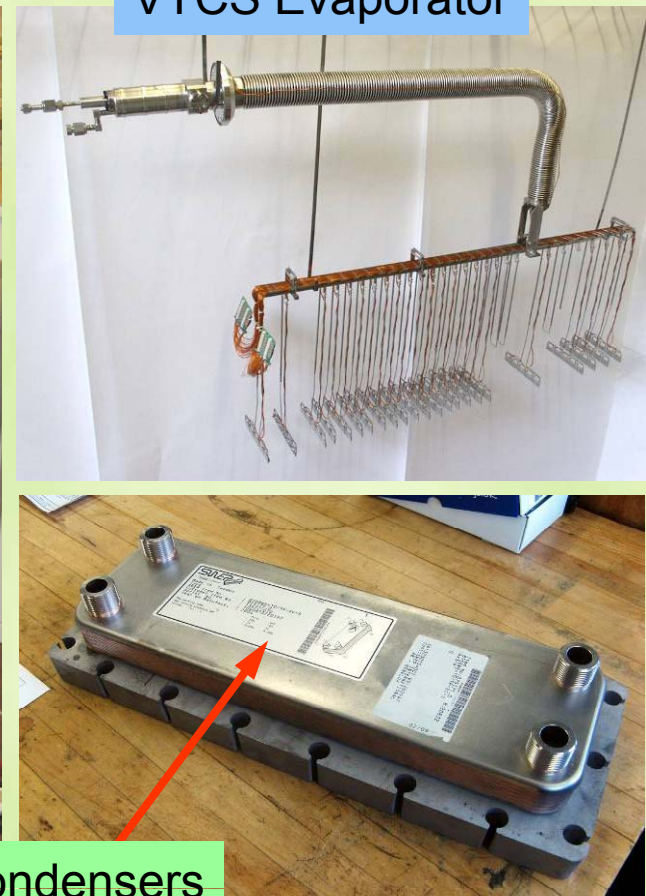


Pumps

Valves



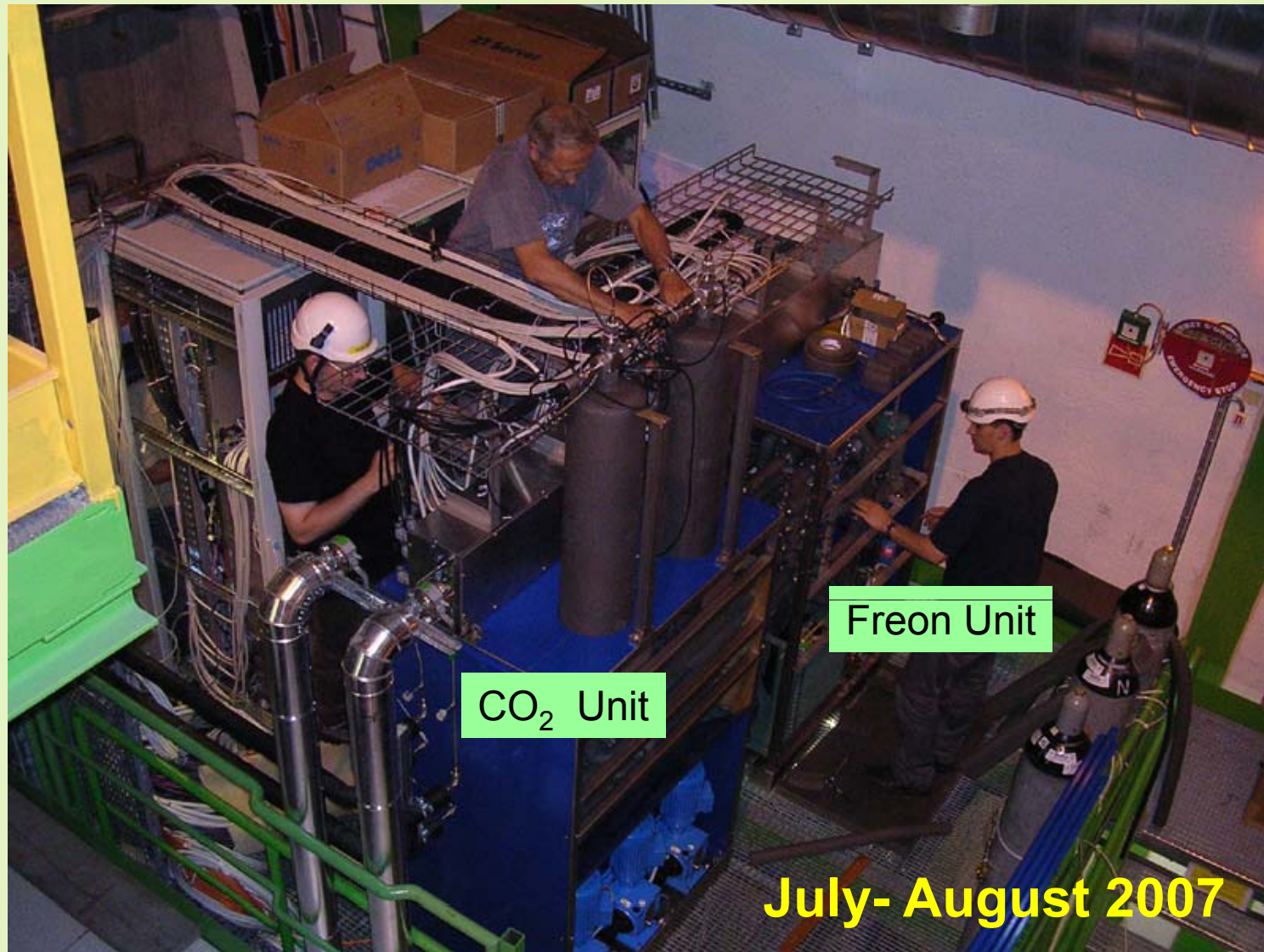
Condensers



VTCS Evaporator



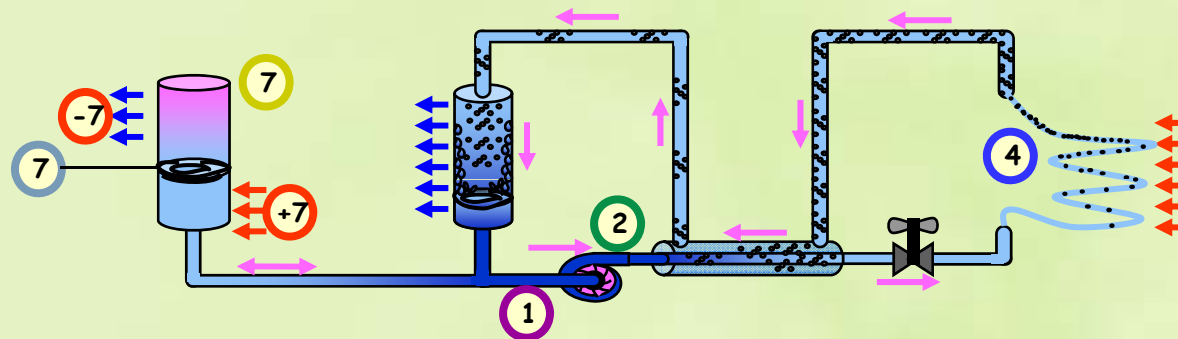
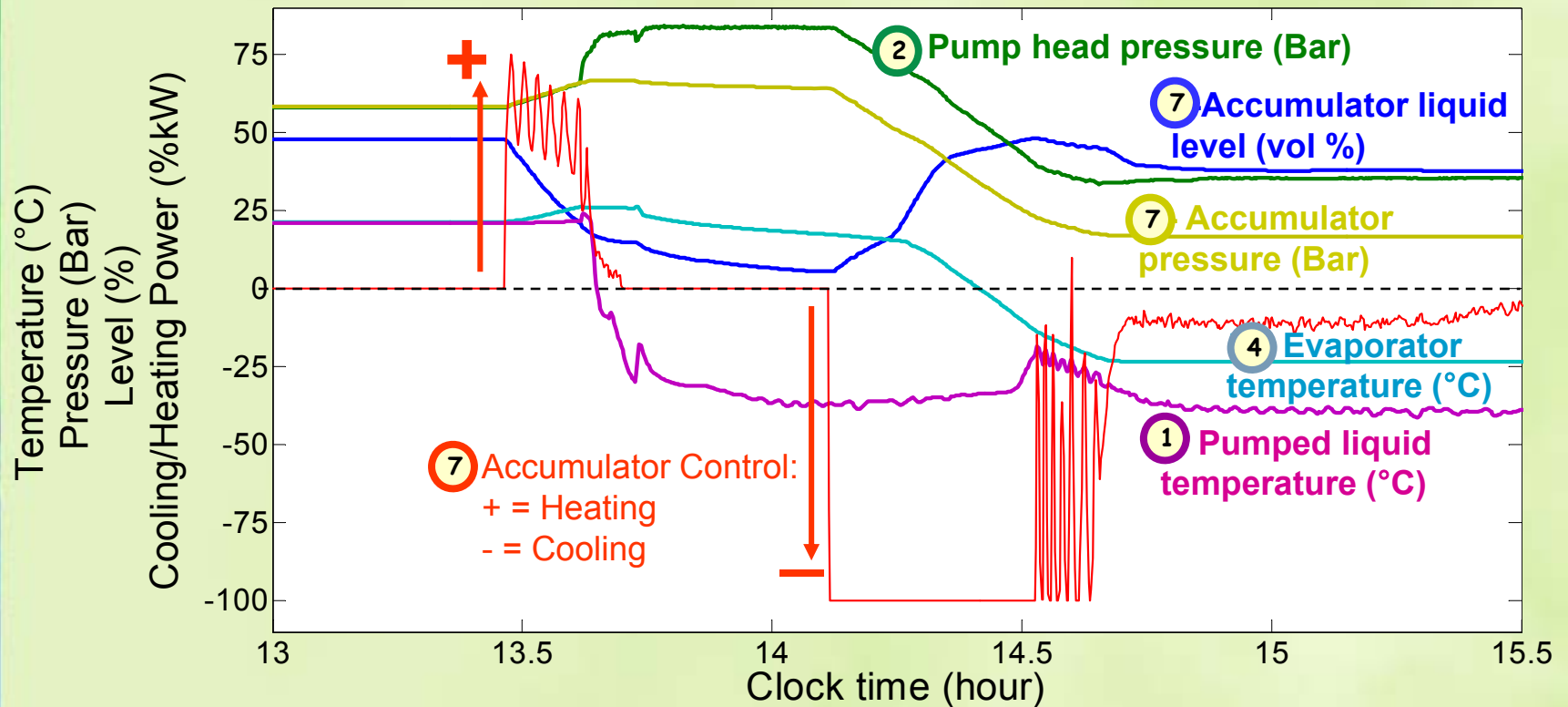
VTCS Units Installed @ CERN



July- August 2007

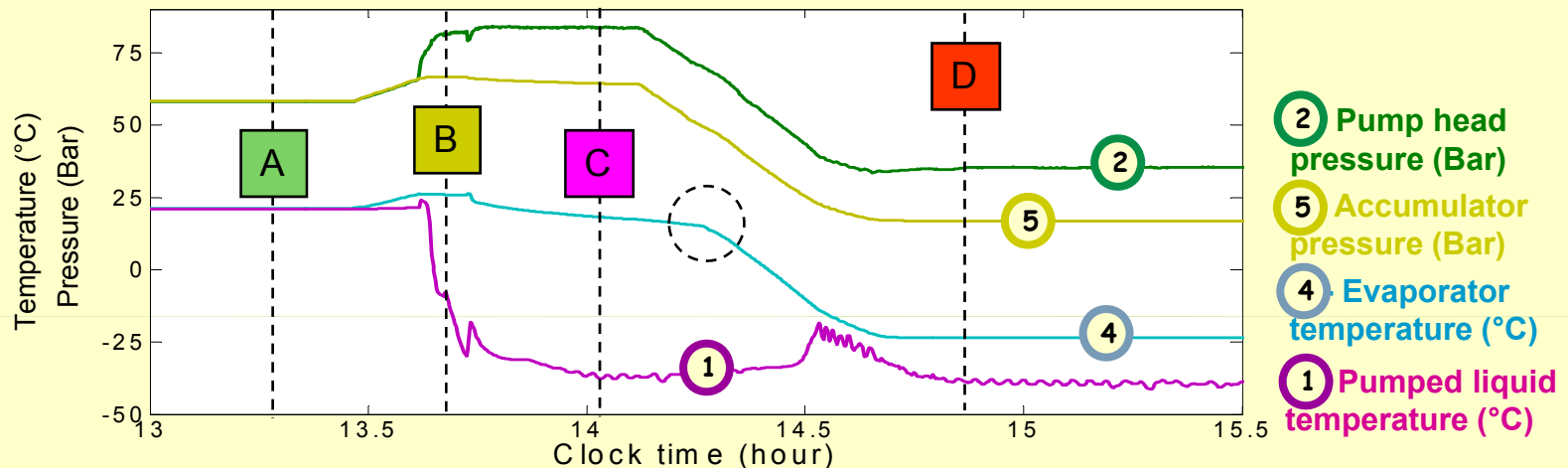
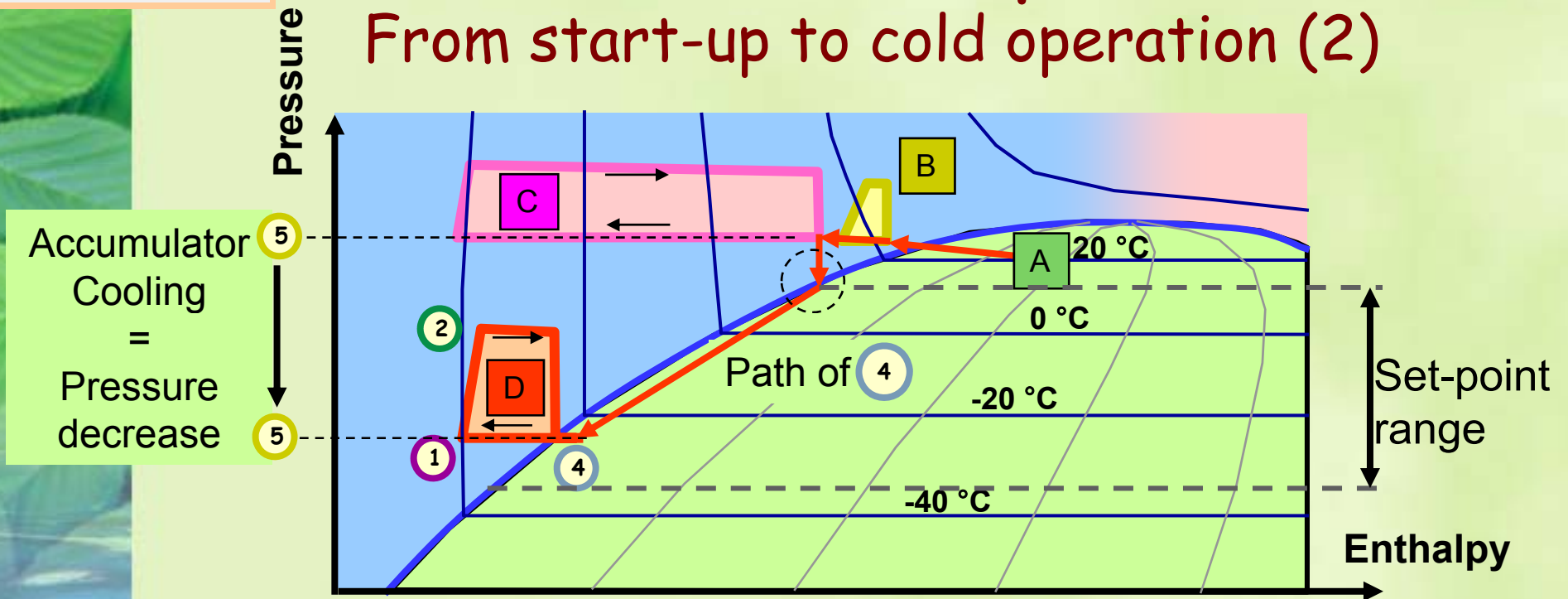
VTCS 2PACL Operation

From start-up to cold operation (1)

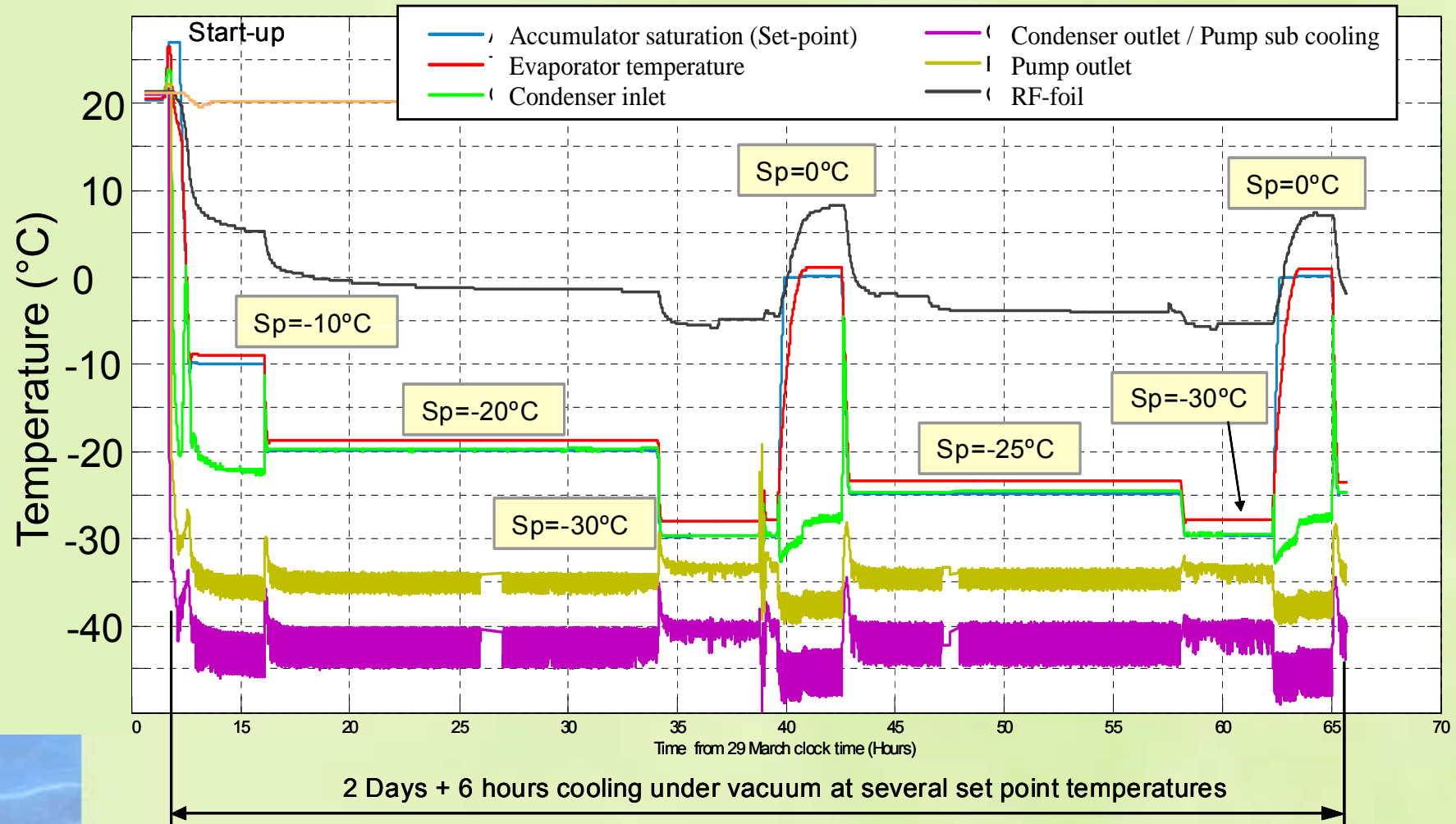


VTCS 2PACL Operation

From start-up to cold operation (2)



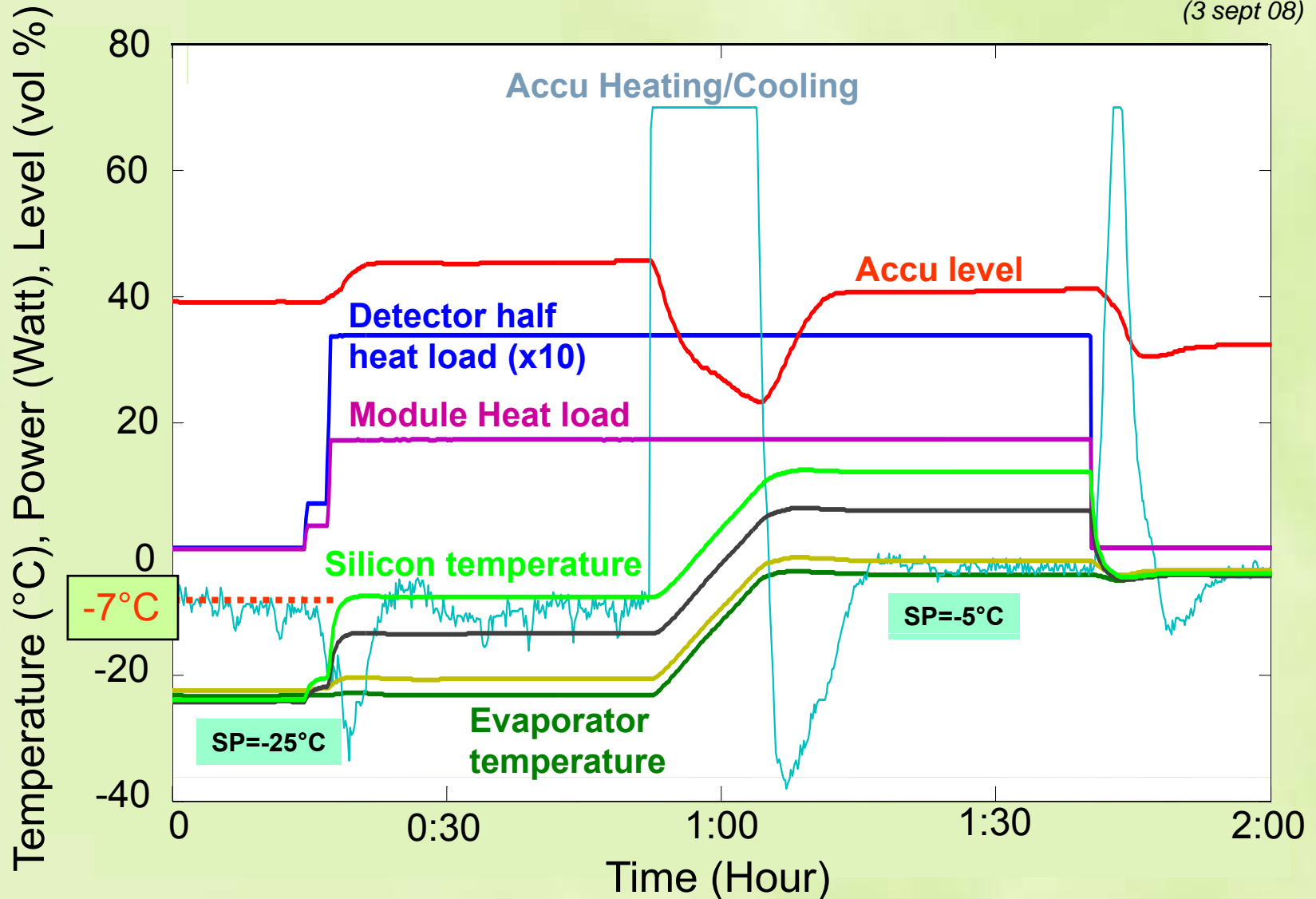
March '08: Commissioning of the VTCS *Detector under vacuum and unpowered*



24 June '08: After a succesful commisioning of the detector at -25°C , the setpoint is increased to -5°C .

And has been running since then smoothly!

(3 sept 08)



VTCS performance overview for a set point of -5°C (Detector switched on, fully powered)

$dP=0.6 \text{ bar} = 6.2 \text{ m static height}$

Detector offset from accu control: 0.7°C

CO_2 heat transfer
 $dT=1.4^{\circ}\text{C}$

Evaporator Pressure
 31.15 bar = -4.18°C

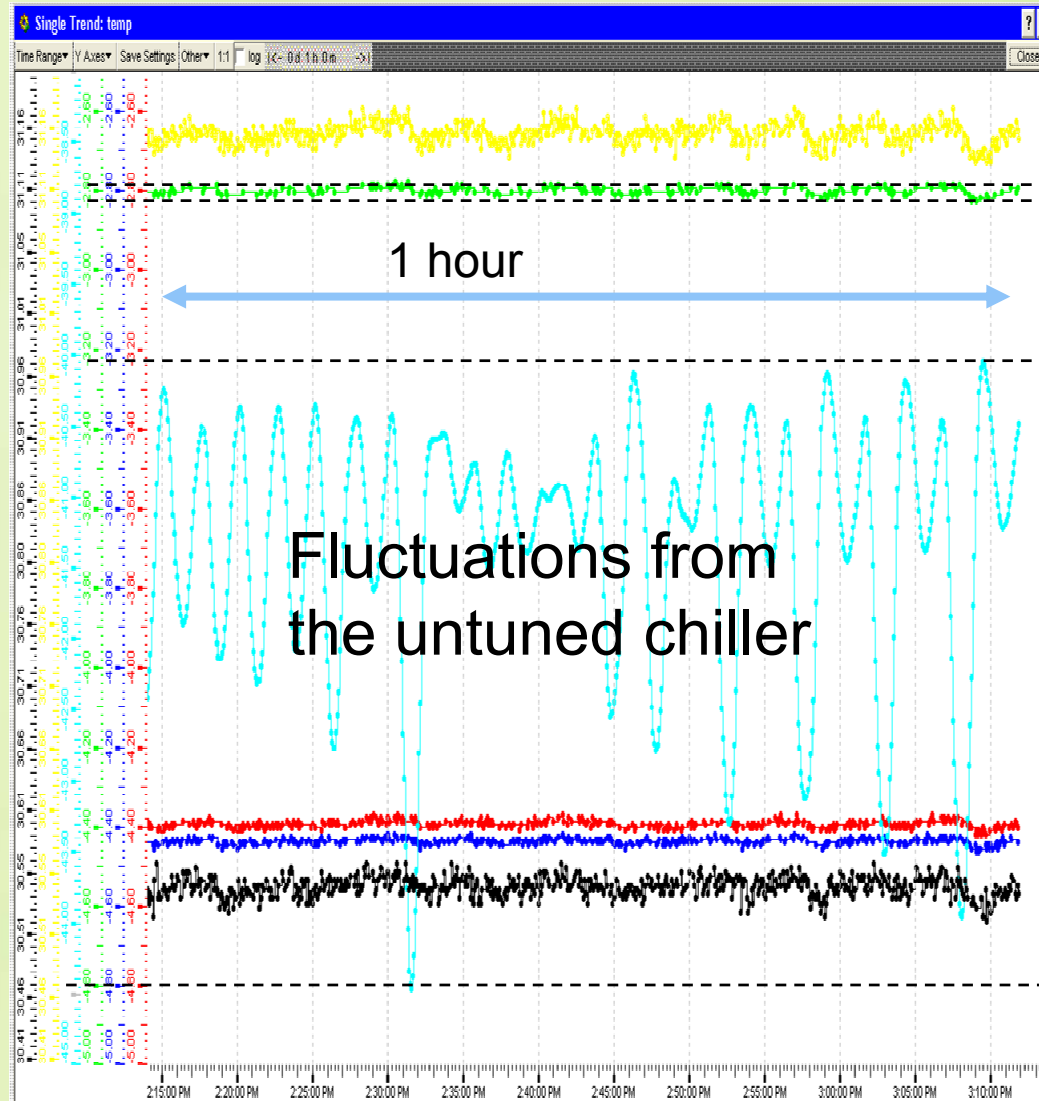
Cooling block
 temperature = -2.8°C

CO_2 liquid temp = -42°C

Evaporator liquid
 inlet temp = -4.40°C

Evaporator vapor
 outlet temp = -4.44°C

Accumulator Pressure
 30.54 bar = -4.90°C



Cooling block
 $dT=0.04^{\circ}\text{C}$

CO_2 liquid
 $dT=4.5^{\circ}\text{C}$

VTCS Summary

- The installation at CERN started in July 2007.
- The VTCS has successfully passed the 1st commissioning phase between march and June '08 and is ready to be used in the experiment
- Operational temperature range is between 0°C and -30°C set point (+10°C with the back-up chiller)
- It has run for 2½ months continuously without any problem (only 3 interruption due to power or cooling water failures)
- It behaves very stable (<0.1°C fluctuation), with the chiller still to be tuned.
- The silicon temperature is below the required -7°C @ -25°C set point temperature. (This is consistent with the prediction)

Some Lessons Learned

- The accumulator sometimes gives the pump a 2-phase mixture => cavitation. Problem is solved by connecting the accumulator to the inlet of the condenser instead of the outlet where it is now.
- Operational temperature range of the evaporator is larger than expected. This is due to the "Duck Foot Cooling"¹ principle of the transfer line.
- The main pressure drop of 2-phase flow in the return line was not caused by friction but by the heights of the upward columns combined. Minimization of the upward columns is more important in the design. Were CO₂ gives only about 1°C/bar ($\approx 0.1^\circ\text{C}/\text{m}$), the use of other fluids will make upward 2-phase flow impossible without a big temperature penalty.

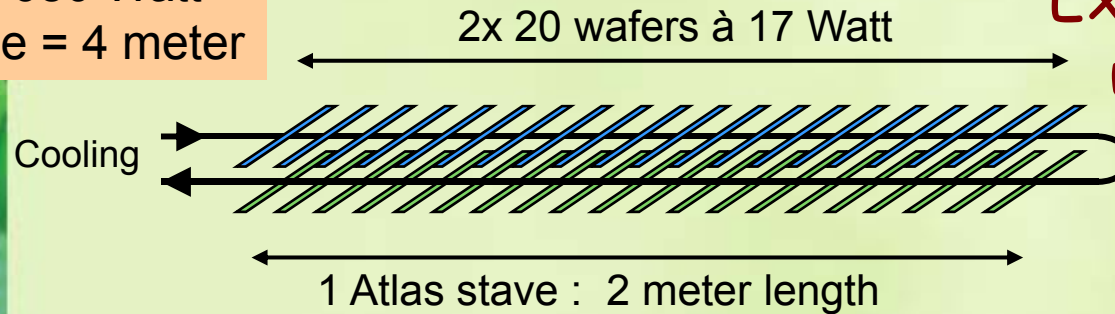
¹ The way a duck can have cold feet without losing body heat, by exchanging heat between the in- and outlet bloodstream.

Outlook

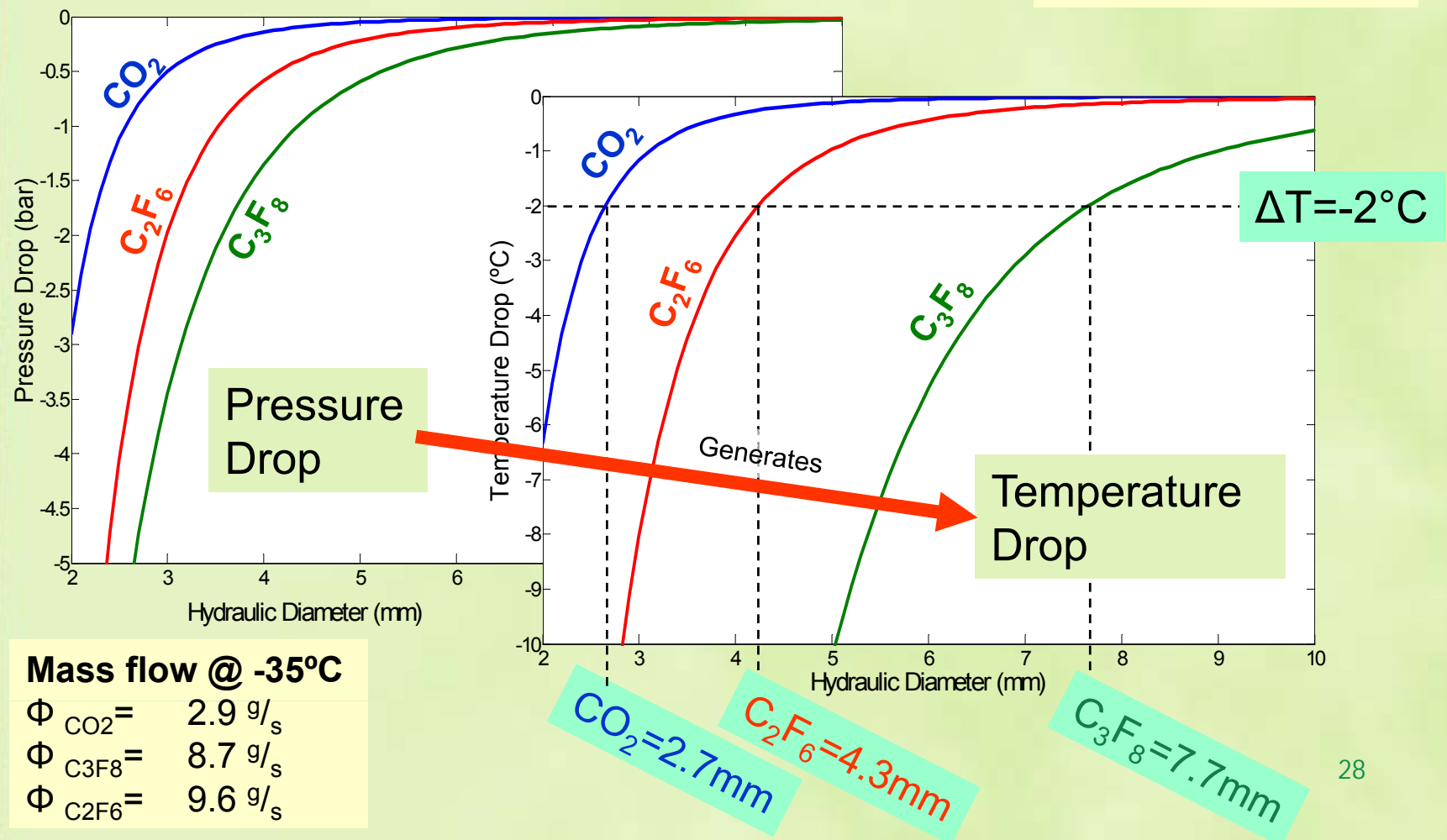
- The VTCS is not yet finished, some things have to be done:
 - Implementing automatic back-up procedure.
 - Changing the accumulator connection.
 - Tuning the chiller.
 - Analyze data for publication.
- Construction of a mini desktop 2PACL CO₂ circulator for general purpose laboratory use.
- Other CERN detectors (Atlas/CMS) have shown interest in the VTCS for their inner tracker upgrades.
 - Challenge: Scaling of the 1.6kW VTCS to a 100kW system.
 - An example of fluid trade-off is given in an Atlas upgrade stave calculation example.

Q = 680 Watt
Tube = 4 meter

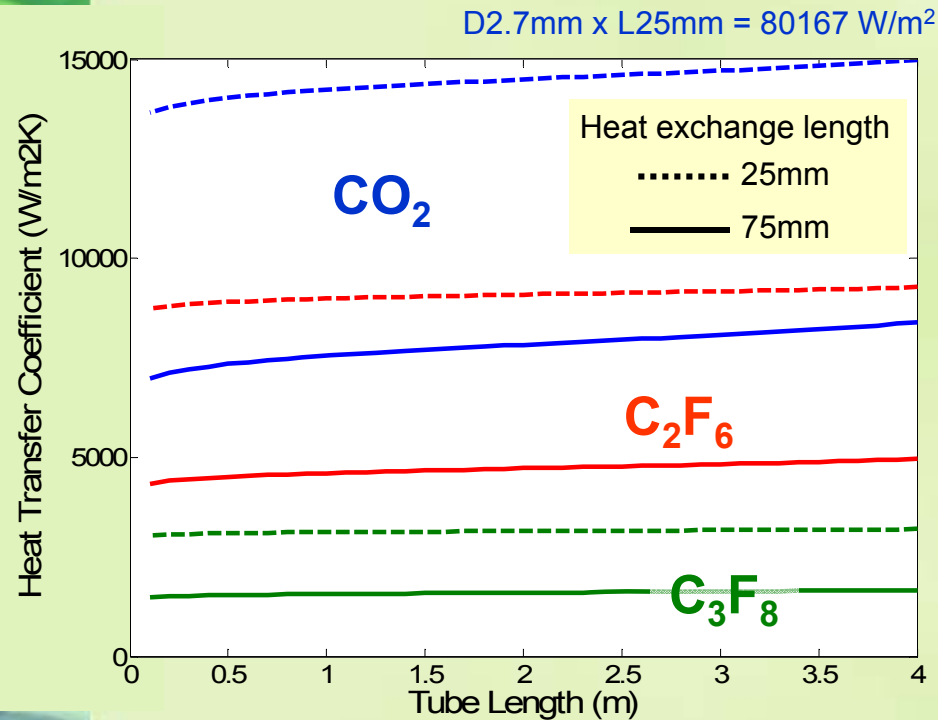
Example of and Atlas upgrade stave (1)



Calculations based on -35°C and 75% vapor quality at exit



Example of and Atlas upgrade stave (2)



Mass flux @ -35°C

$$\Phi'_{\text{CO}_2} = 506 \text{ kg/m}^2\text{s}$$

$$\Phi'_{\text{C}_3\text{F}_8} = 661 \text{ kg/m}^2\text{s}$$

$$\Phi'_{\text{C}_2\text{F}_6} = 186 \text{ kg/m}^2\text{s}$$

Critical Heat Flux (Bowering/Ahmad):
 $\text{CHF}(\text{CO}_2) = 313 \text{ kW/m}^2, x=1.1$

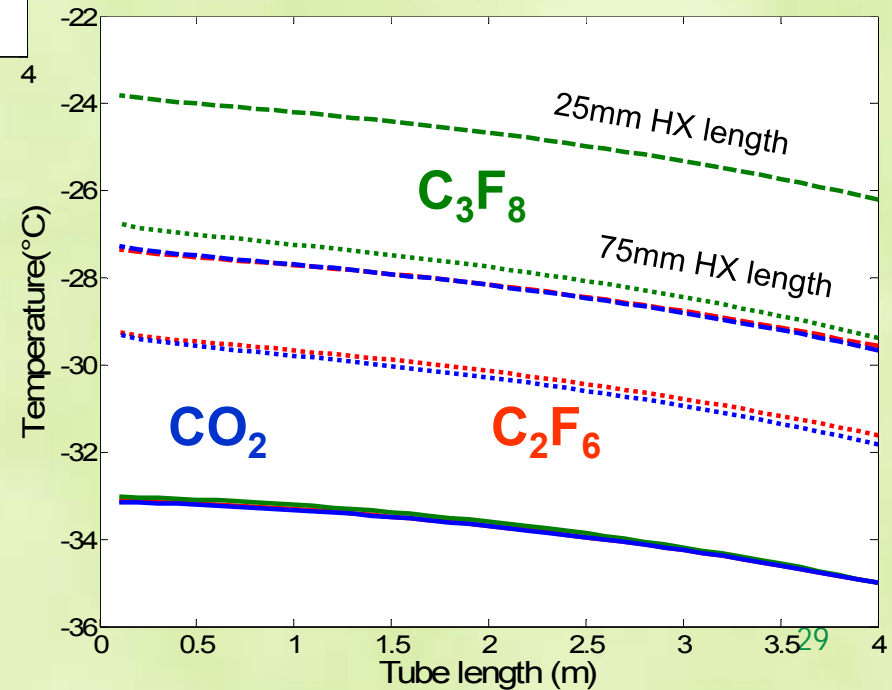
$$\text{D4.3mm x L25mm} = 50337 \text{ W/m}^2$$


$$\text{D2.7mm x L75mm} = 26722 \text{ W/m}^2$$

$$\text{D4.3mm x L75mm} = 16779 \text{ W/m}^2$$

$$\text{D7.7mm x L25mm} = 28110 \text{ W/m}^2$$

$$\text{D7.7mm x L75mm} = 9370 \text{ W/m}^2$$





Thank you for your attention:

Questions?