

# MATERIALS CHARACTERIZATION AT CRYOGENIC TEMPERATURES

Arman Nyilas

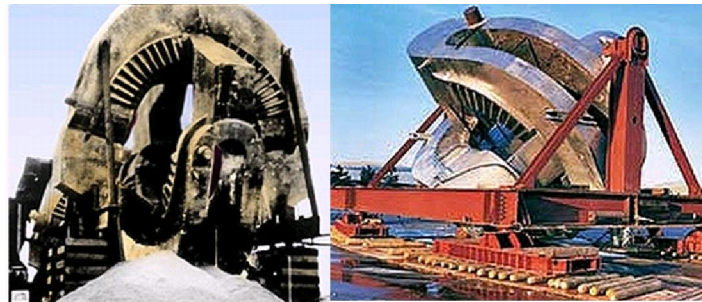
Cryogenic Engineering & Materials Expertise , (CEME), 76297 Stutensee, Germany

- Overview about materials performance for structural design
- Tensile measurements
- Fracture toughness measurements
- History of FCGR and its concept
- FCGR evaluation between 80s and 90s, and today
- Concept of uncertainty and its basics (true value ?)
- Application of uncertainty in case of FCGR
- Paris law region as model equation
- Evaluation of the best estimate and its uncertainty term

## Structural steels: A common feature is the heavy wall thickness



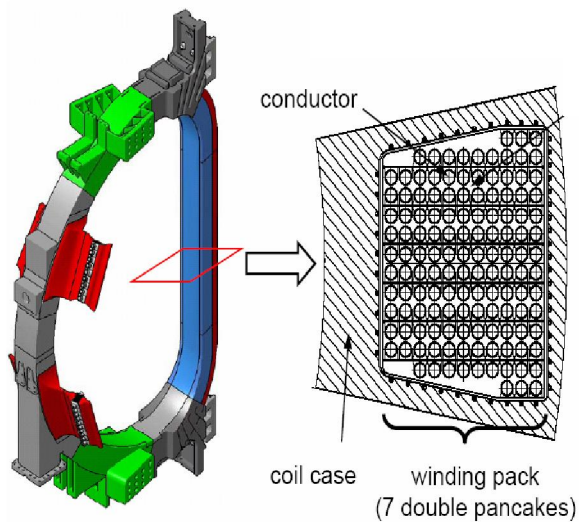
BEBC, Cern (1975-1985)  
3.7 m Ø x 4 m (316LN)



MFTF-B, Livermore (304N): Built in 80s, 1986 shut down  
without any service



Tore Supra, Cadarache  
(1988 – 2003; 6 min Plasma)  
1.8 K service (316LN)



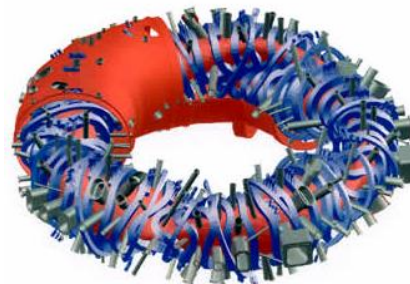
ITER, Cadarache TF coil design  
2008 (+~ 10 y)



LHD, NIFS, Japan SUS 316L  
1998 March 31: 1<sup>st</sup> Plasma  
Total cold mass 850 t



LCT- EU, Karlsruhe & TFMC  
1.8 K service (316LN) (1984-2005)



W7 – X, Greifswald: 50 non-planar/20 planar coils ~ 2013/2014

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## General addresses for the service of structural materials at 4 K

- A common feature of all these magnets is the similarity to pressure vessels likened to gas pressure, magnetic field lines try to expand the structure with equivalent pressures up to 1000 bar at temperatures near 4 K
- Beside the tension also bending stresses are inevitable due to manufacturing and fault conditions
- No code or standard guidance exists for the design of magnet structures for service at 4 K
- Designers usually follow parts of various codes like Secs. III or VIII of ASME Boiler and Pressure Vessel Code
- However, blind obedience to existing codes may result either excessive heavy or dangerous fracture-prone structures

## An example from past times about the design rules: MFTF B

### Historical: Recommended design factors for Mirror magnets

#### Base metal

Design stress < 2/3 yield (primary tension and combined)

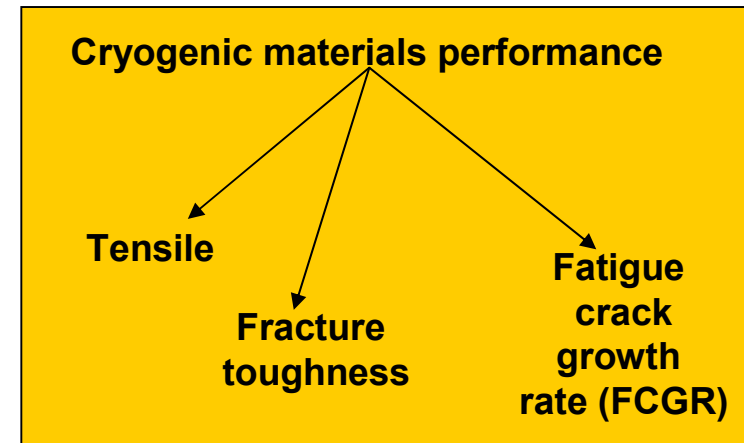
Design stress < 90% yield (primary bending)

Design stress < 1/3 UTS

Design stress <  $\frac{1}{2} \cdot \frac{K_{IC}}{\sqrt{\pi \cdot a}}$

Design stress cycles < 4 lifetimes

The lowest stress values should be used



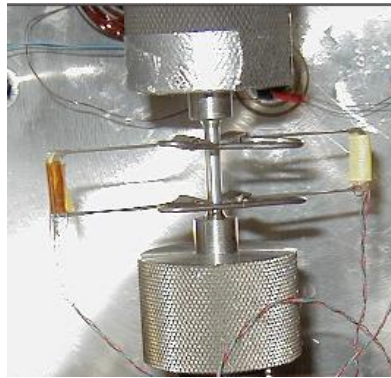
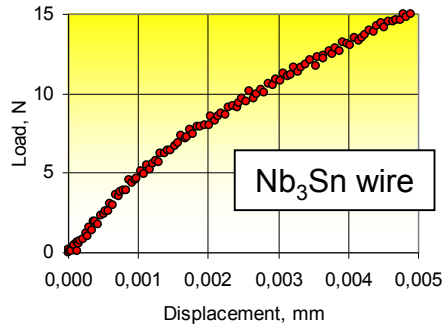
At this stage according to these past tests the toughness was the major problem with the consumable welds

Welds were manufactured by SMA (practically manual)

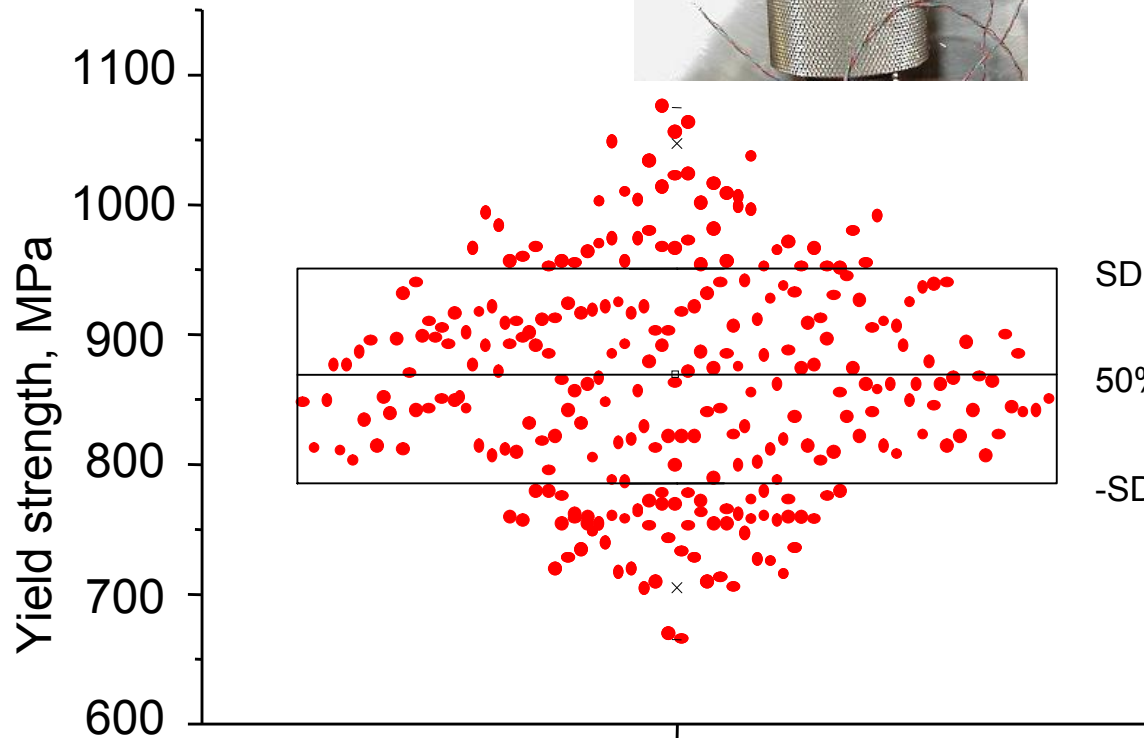
Yield strengths of 304N base metal values were 840 MPa and  $K_{IC}$  values were 60 – 120 MPa $\sqrt{m}$

$\delta$  - Ferrite versus hot cracking was one of the main culprit

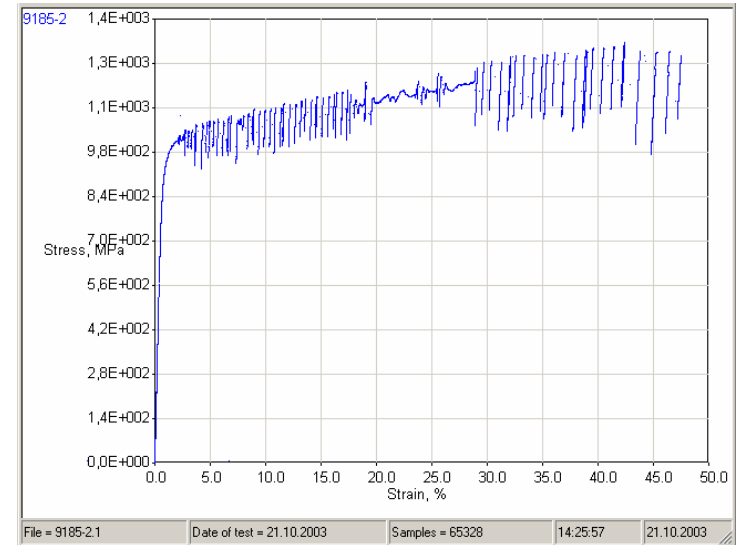
# Yield strength properties of cast nitrogen strengthened materials: W7 - X



Yield strength of cast material measurements of 259 specimens with respect to different heats

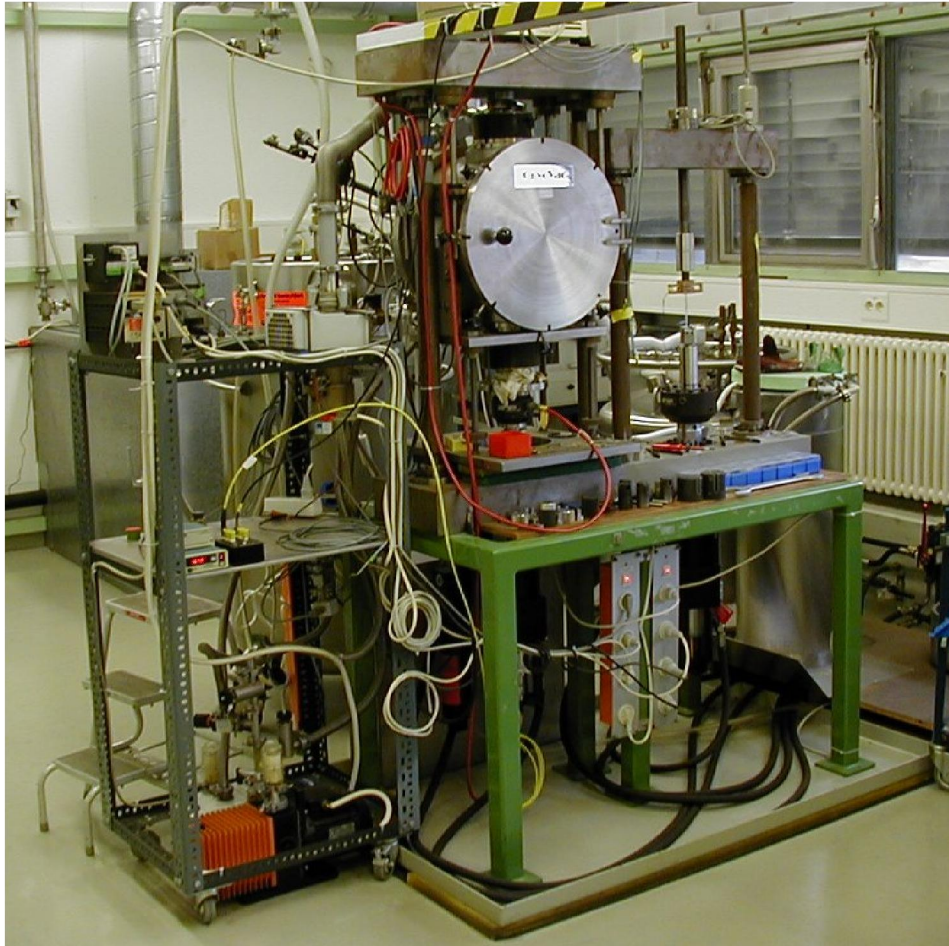


Collected data of 295 specimens

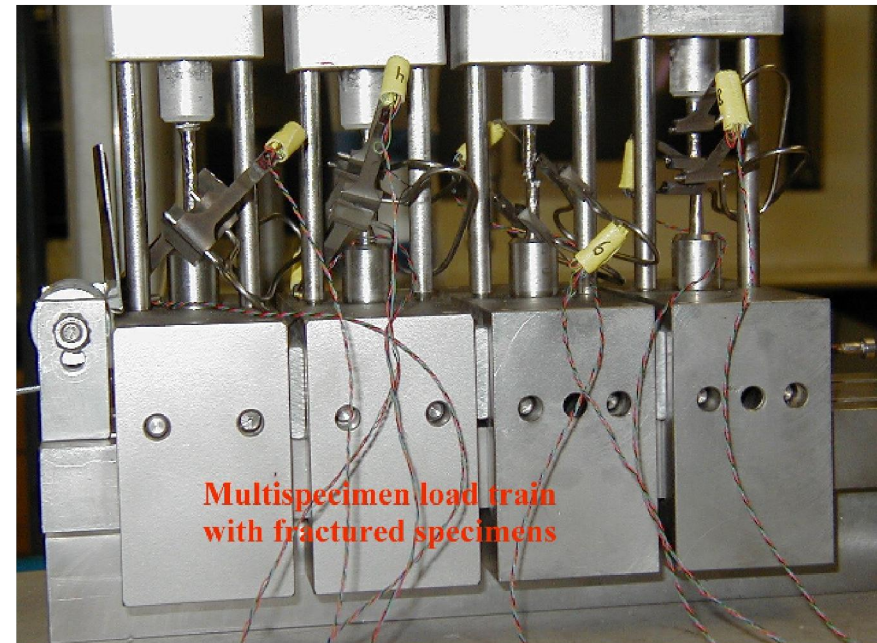


## Used cryogenic test facilities: +/- 25 kN MTS Type & 630 kN Schenck Type

The facilities are located at Karlsruhe Institute of Technology (KIT), former Forschungszentrum Karlsruhe. In addition all following measurements are done at Forschungszentrum Karlsruhe



Variable temperature (295 K -7 K)

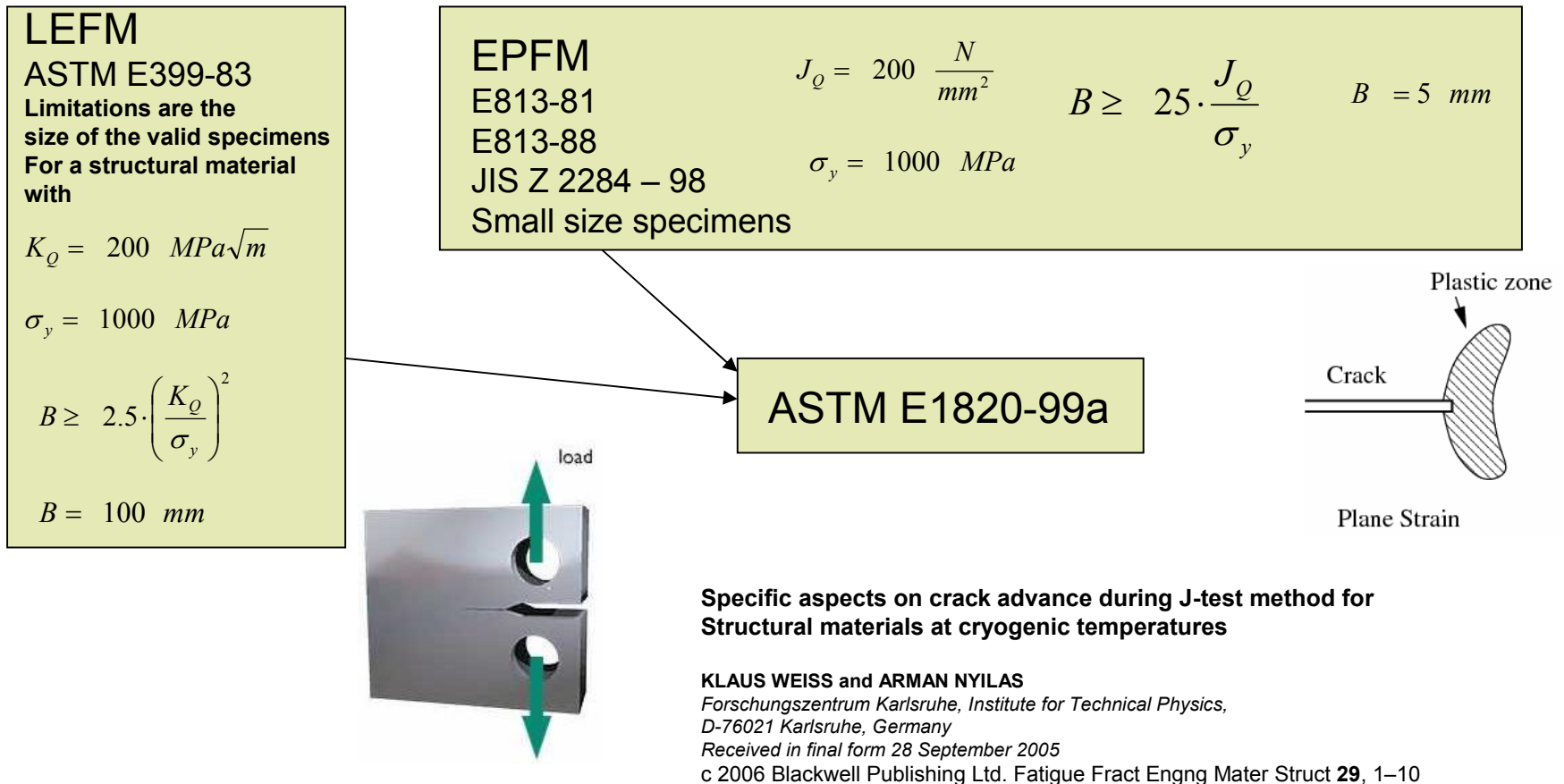


LHe, LN2, or RT

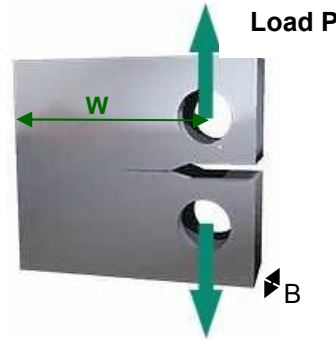
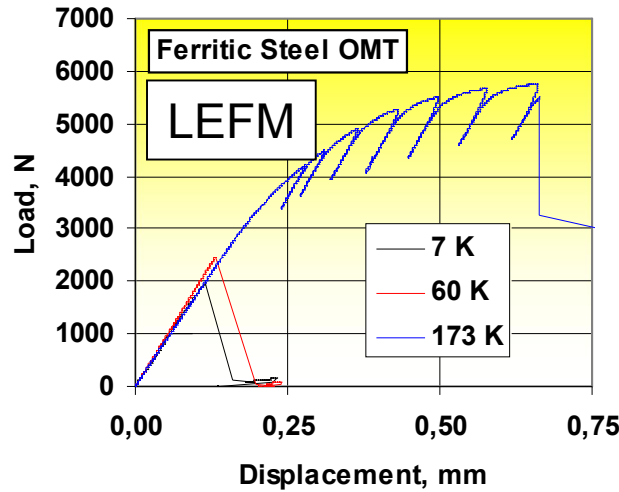
# Fracture mechanics, its history and situation of today

## The Struggle for Recognition of Engineering Fracture Mechanics

The post WW II work led to the development of modern fracture mechanics with a modification of the Griffith idea from WWI in which the resistance to crack extension was assumed to be plastic deformation at the leading edge of the crack rather than solid state surface energy (Irwin 1948, Orowan 1949). Linear elasticity theory predicts that the stresses become infinite as a crack tip is approached. In real materials, however, the stresses are limited by the flow stress, and a plastic zone develops at the end of the crack.



# J-Integral: Small scale specimens, its energy absorption, & fracture toughness



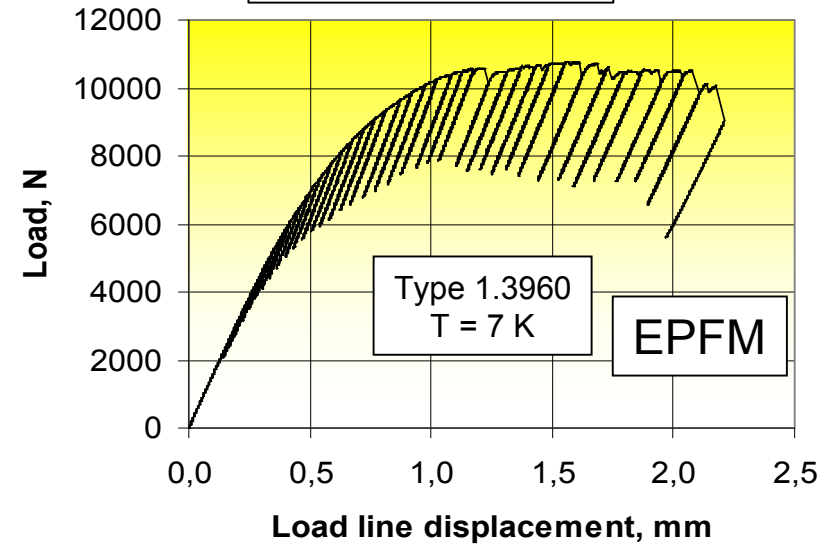
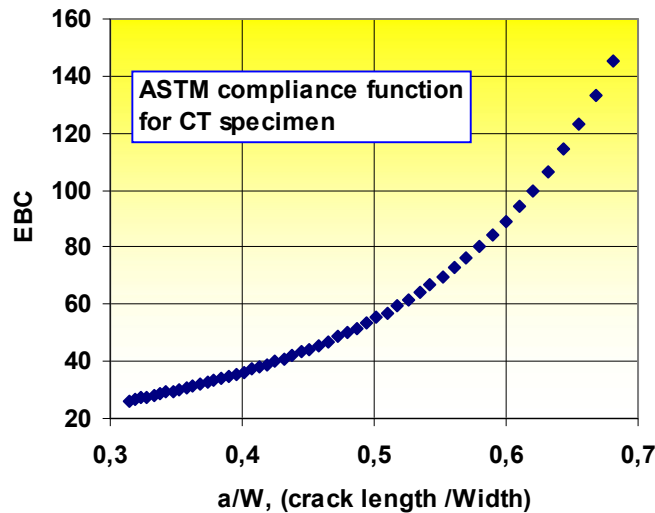
Crack advance (unloading compliance method)

Decrease of stiffness (or increase of compliance)

$$J_{El} = \frac{K^2 \cdot (1 - \nu^2)}{E}$$

$$J_{Pl} = \frac{2 \cdot Area}{B \cdot b}$$

$$J = J_{El} + J_{Pl}$$

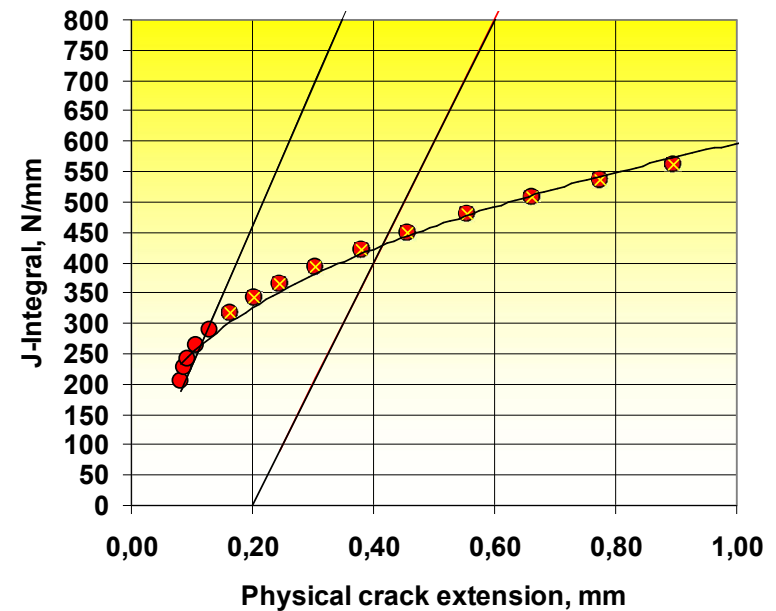
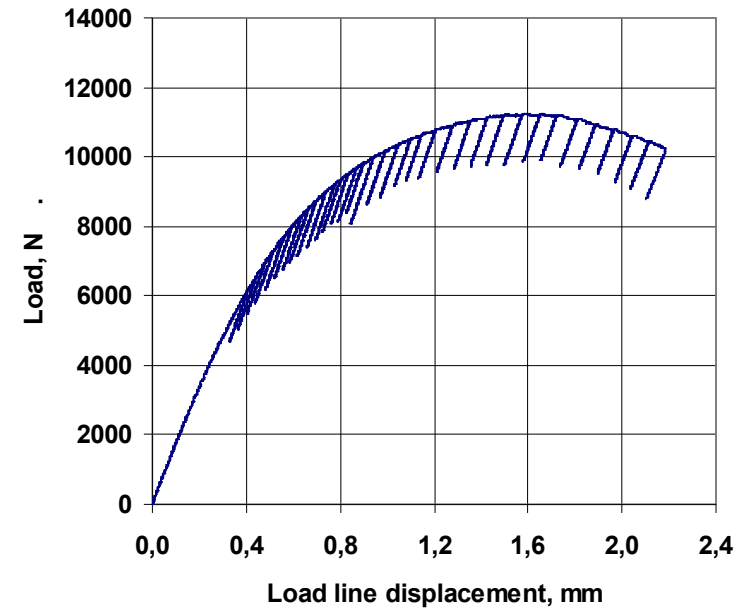
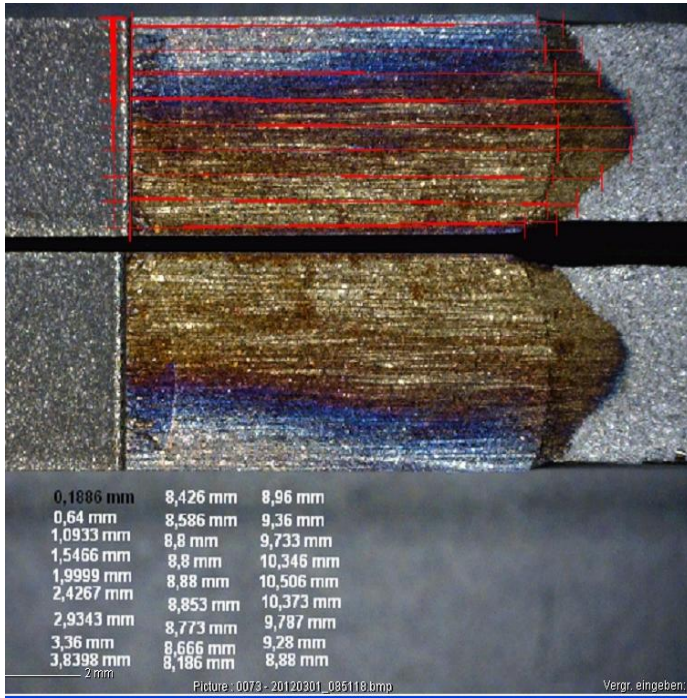


Stress intensity of a compact tension specimen

$$\Delta K = \frac{\Delta P}{B \cdot \sqrt{W}} \cdot \frac{\left(2 + \frac{a}{W}\right)}{\left(1 - \frac{a}{W}\right)^{1.5}} \cdot \left(0,886 + 4,64 \cdot \frac{a}{W} - 13,32 \cdot \left(\frac{a}{W}\right)^2 + 14,72 \cdot \left(\frac{a}{W}\right)^3 - 5,6 \cdot \left(\frac{a}{W}\right)^4\right)$$



# J-test of Type 316LN rolled plate material at 7 K



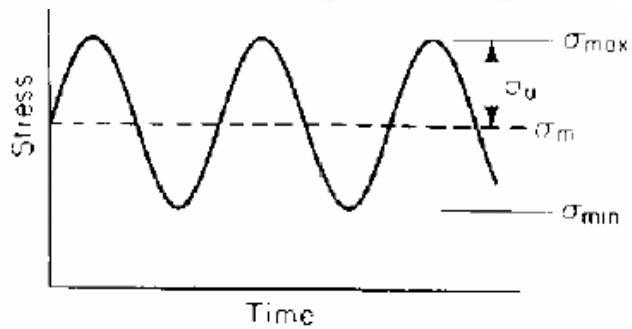
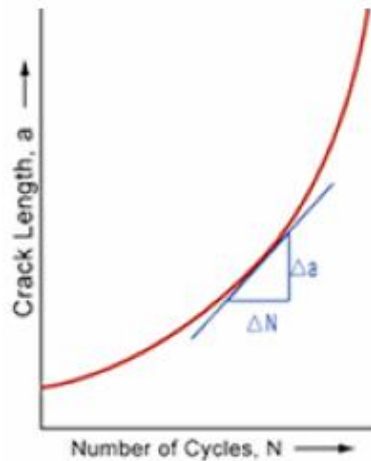
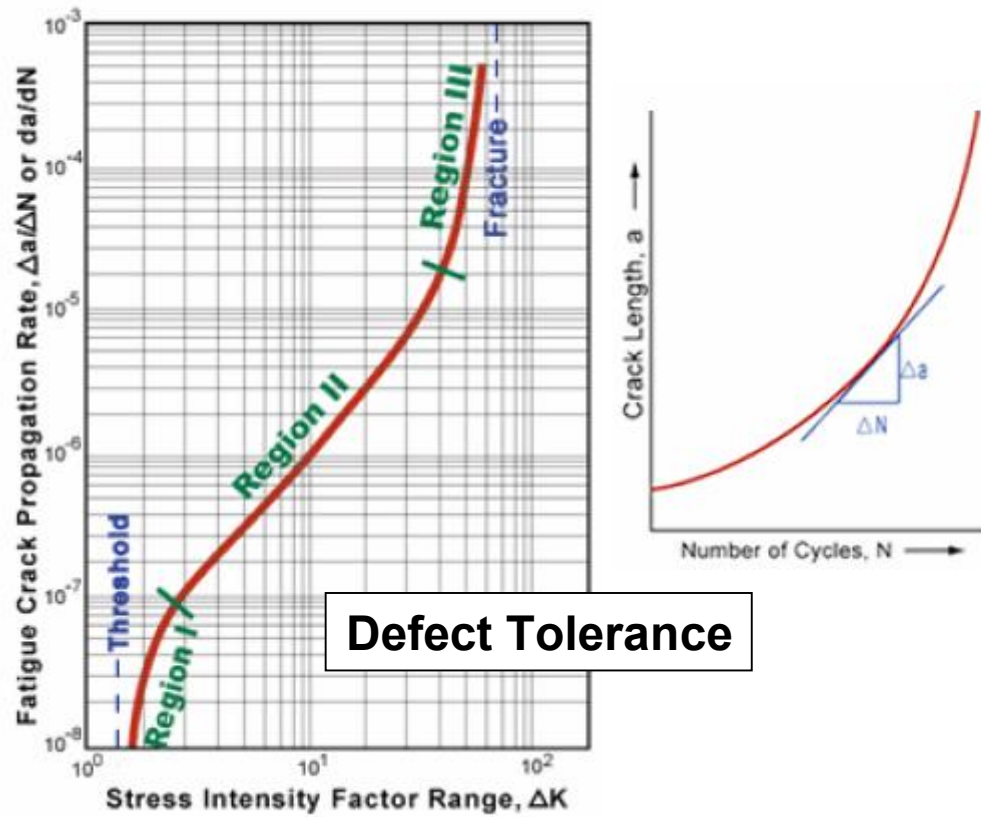
J-test with rolled plate material Specimen: Rol- L-1

Date: February 29, 2012; T = 7 K ;No side grooves

Specimen details:

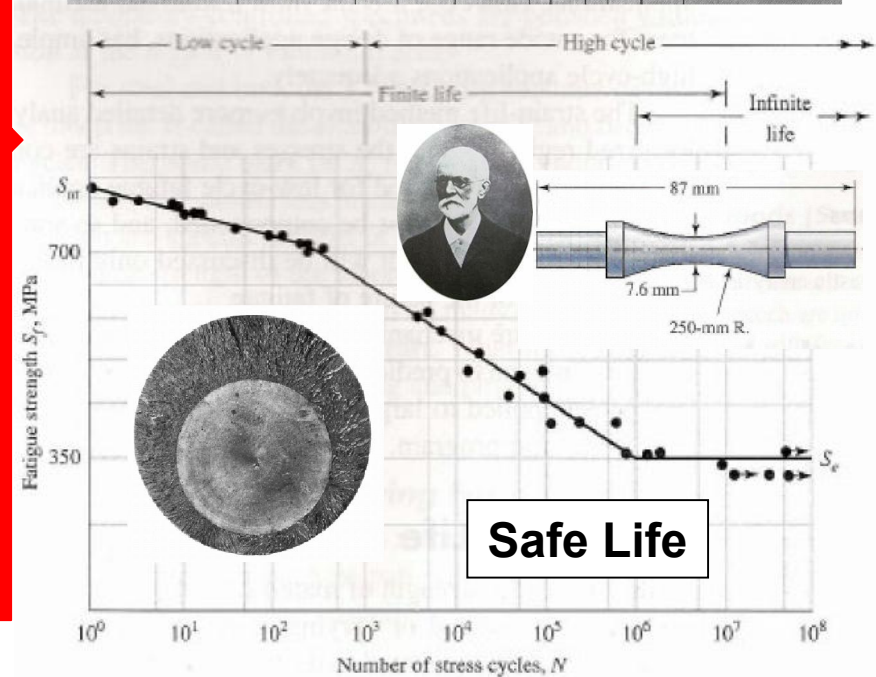
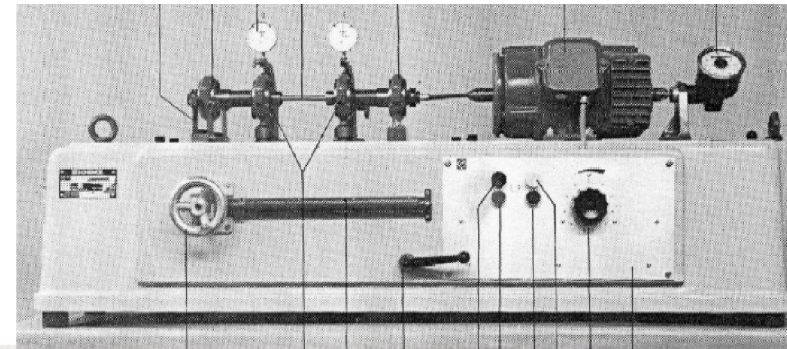
B = 4, W = 36 mm ,  $a_{\text{notch}} = 12$  mm,  $a_f = 20,708$  mm,  
 $a_p = 21,788$  mm;  $J_q = 425$  N/mm; Estimated  $K_{IC} = 295$  MPa $\sqrt{\text{m}}$

# Cyclic loading of components and load bearing members (FATIGUE)



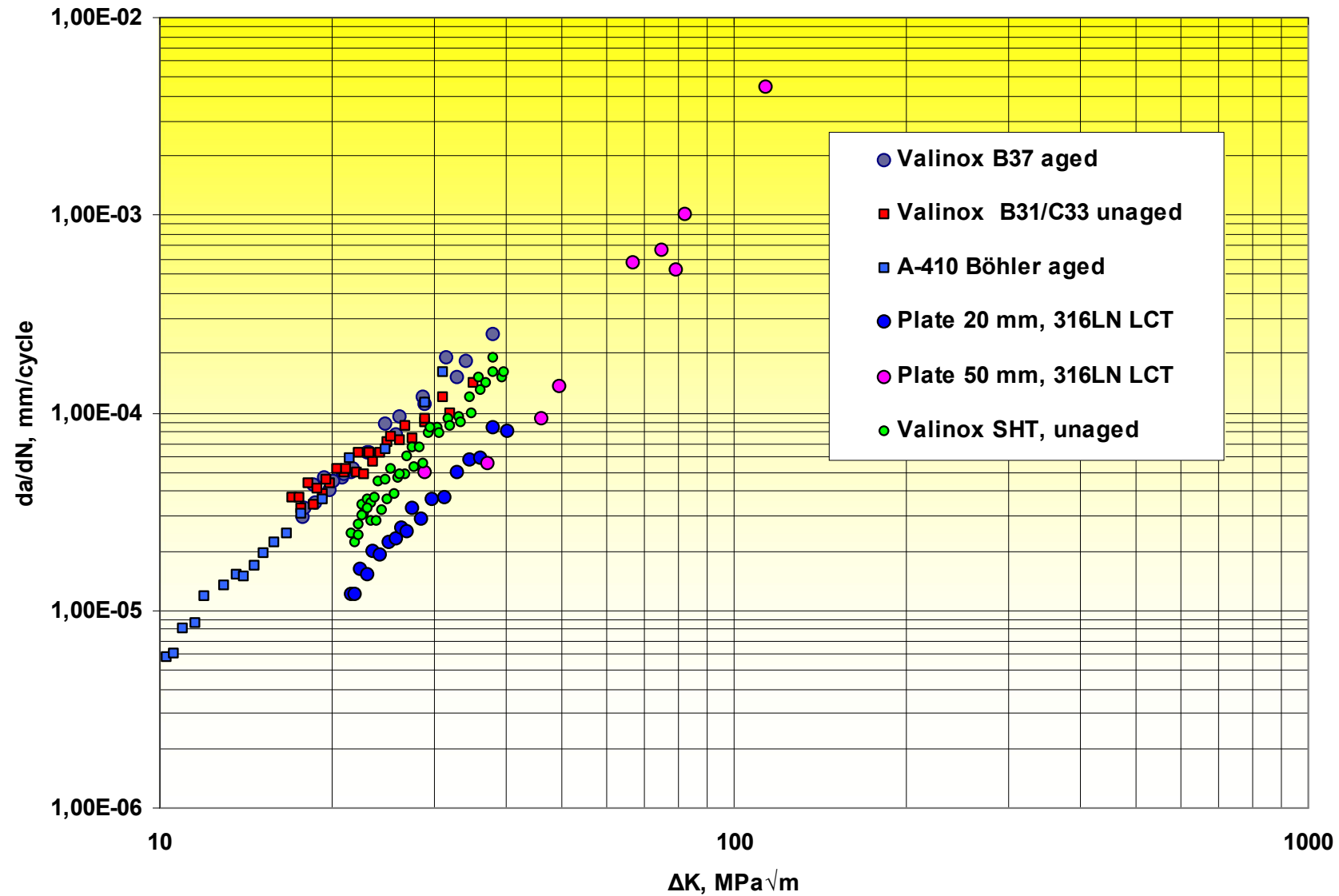
Load Range

$$R = \frac{\sigma_{\min}}{\sigma_{\max}} \approx \frac{K_{\min}}{K_{\max}}$$



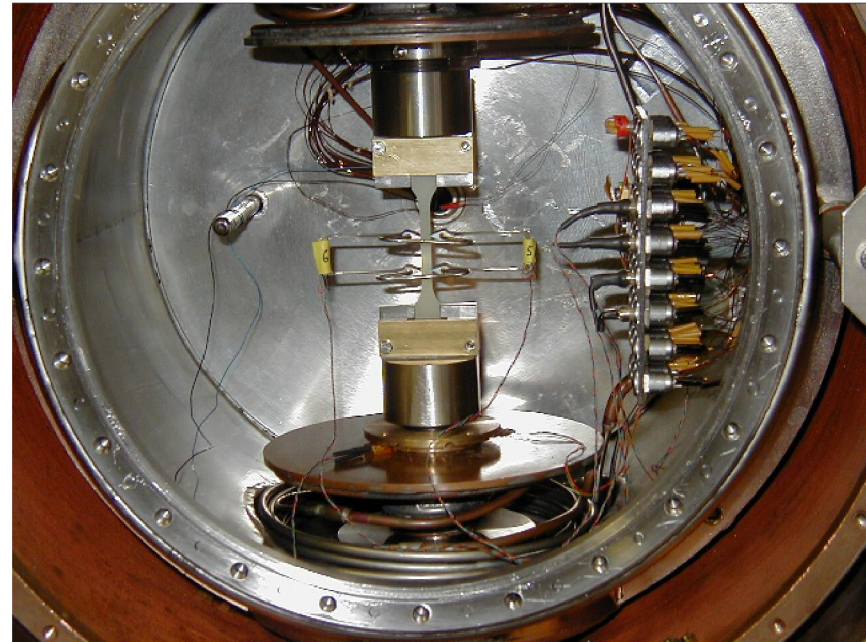
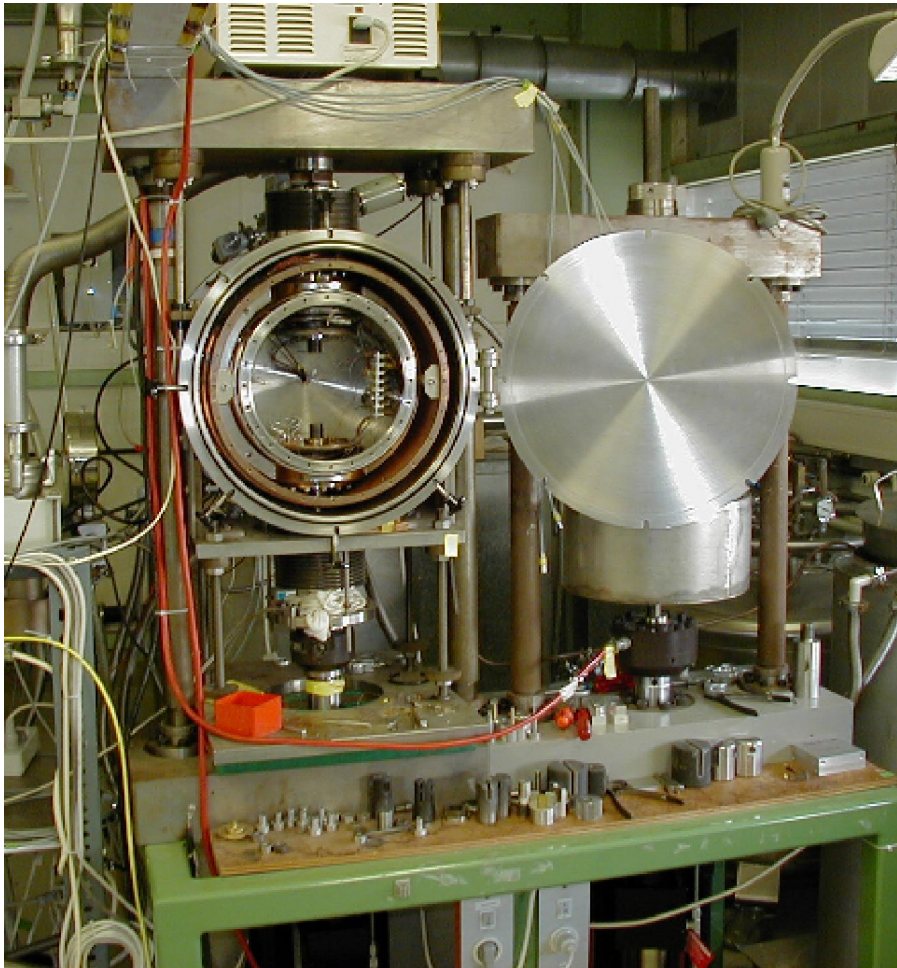


## Collection of FCGR data for 316LN at early 90s

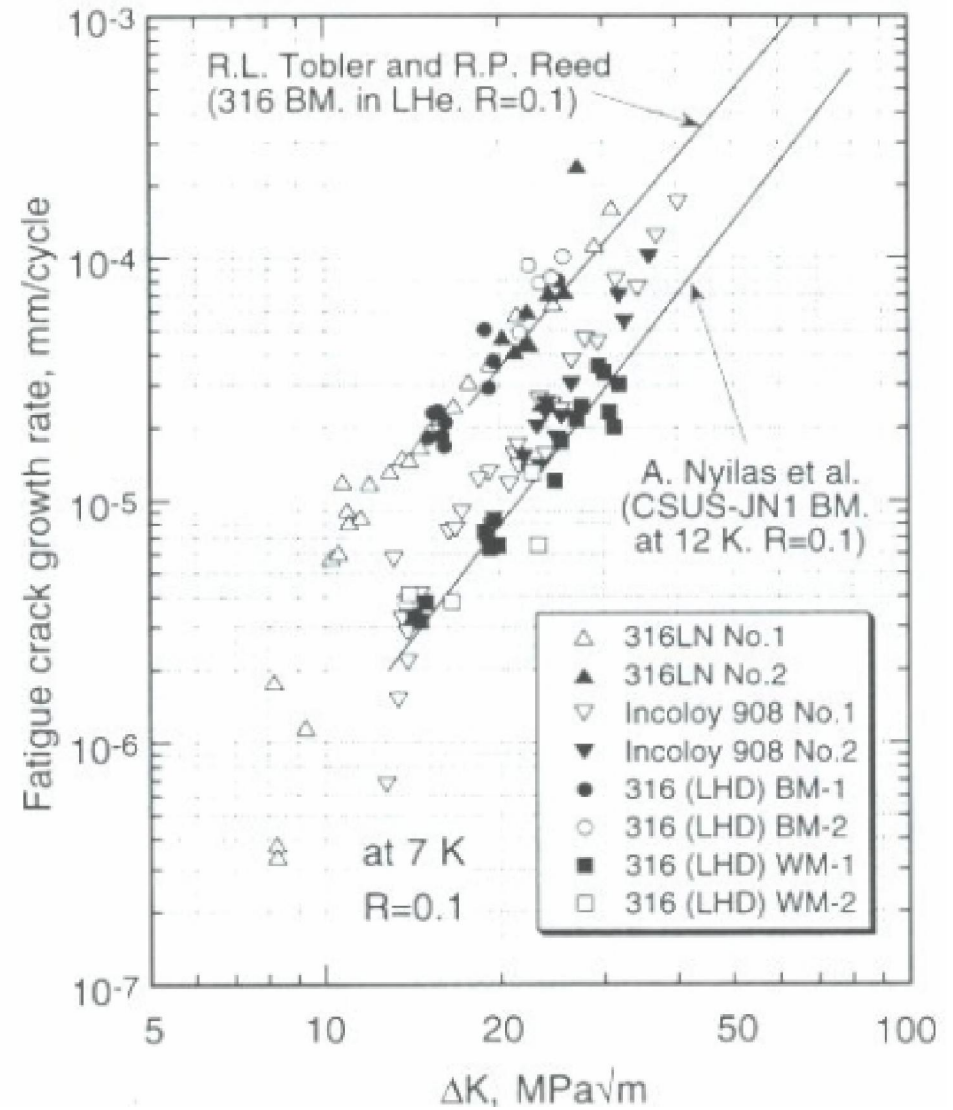
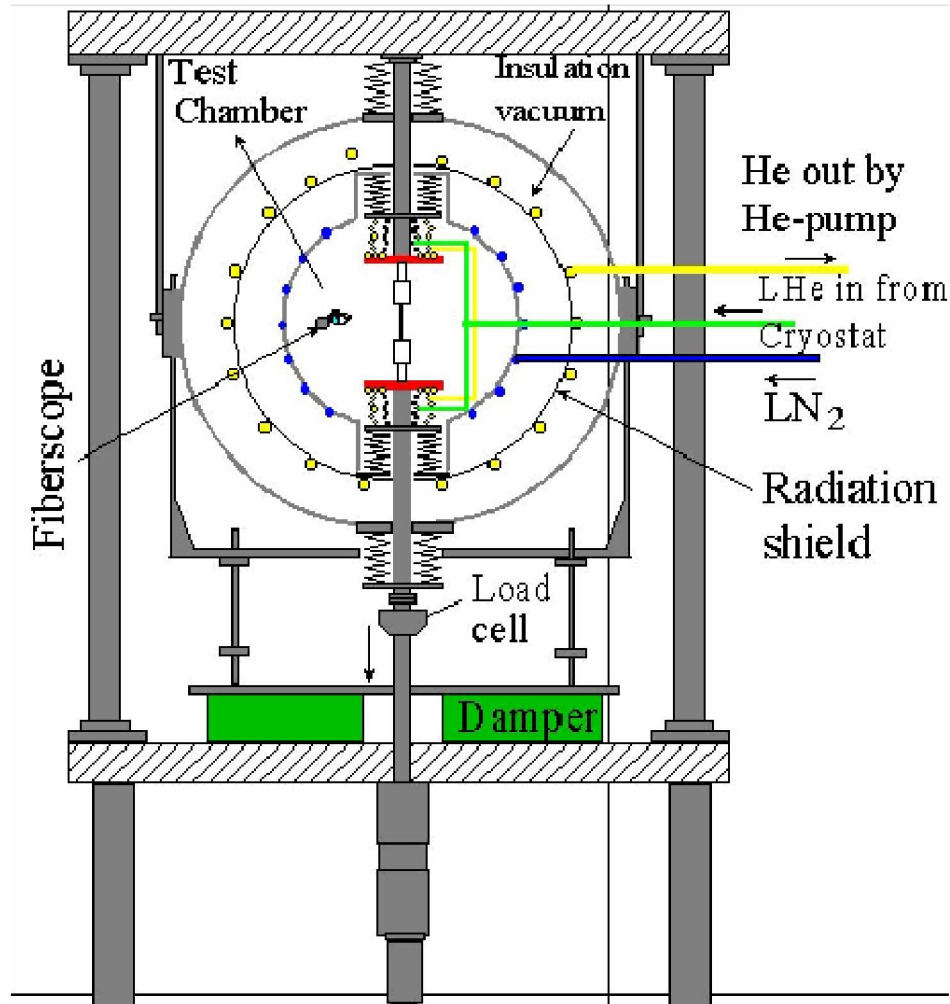


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**Existing variable temperature cryogenic test facility working since 1987**



## Historical measurements : Microscopic crack detection at cryogenics



## Historical measurements : Traveling microscope

### FATIGUE CRACK GROWTH RATE OF SUS 316 AND WELD JOINT WITH NATURAL CRACK AT 7 K

A. Nishimura,<sup>1</sup> J. Yamamoto,<sup>1</sup> and A. Nyilas<sup>2</sup>

<sup>1</sup>National Institute for Fusion Science  
Toki, Gifu 509-52 Japan

<sup>2</sup>Forschungszentrum Karlsruhe, ITP  
D-76021 Karlsruhe, Germany

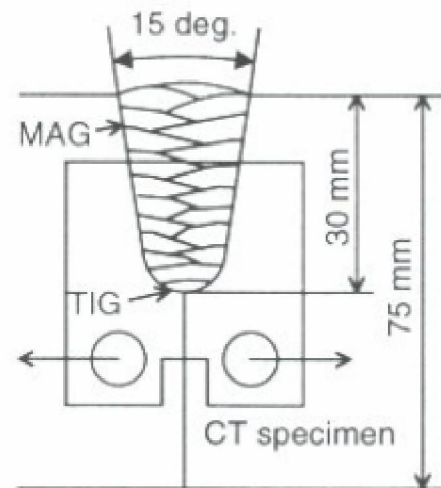


Figure 1. Configuration of welded joint.

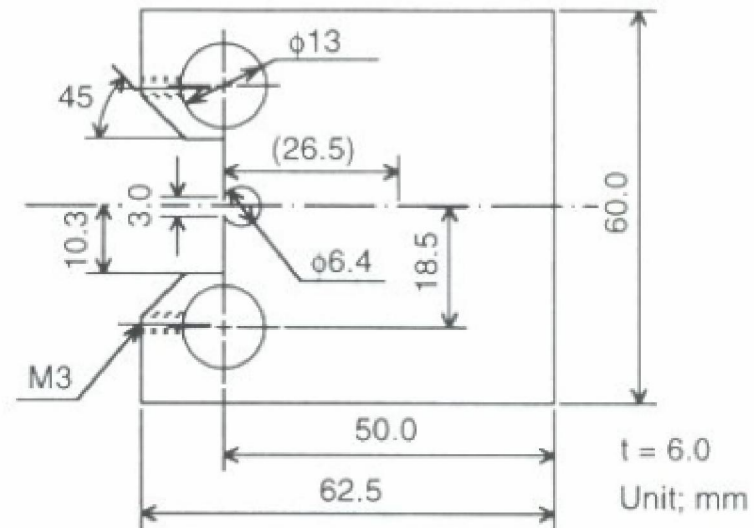
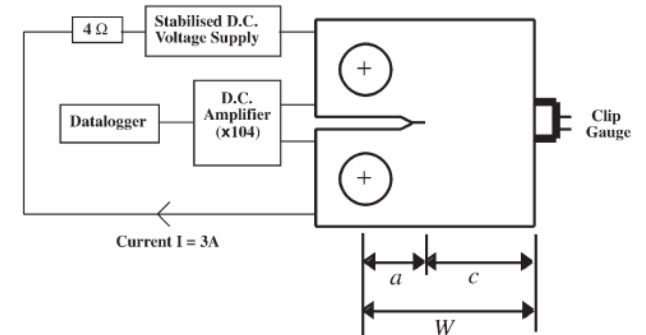


Figure 2. Compact tension specimen.

*Journal of Fatigue* 23 (2001) 375–382



## Life analysis using the Paris law

$$C_0 := 2.84 \cdot 10^{-13} \quad m := 3.58 \quad \Delta K := 114 \quad K_{Ic} := 200 \quad \pi = 3.142$$

$$Y := 1 \quad S := 2 \quad \Delta\sigma := 400 \quad R := 0.1 \quad a_0 := 0.002$$

$$a_c := \frac{1}{\pi} \cdot \left( \frac{K_{Ic}}{S \cdot Y \cdot \Delta\sigma} \right)^2 \quad a_c = 0.02 \quad K_c := 195$$

$$\frac{da}{dN} = C_0 \cdot \Delta K^m$$

**Paris law**

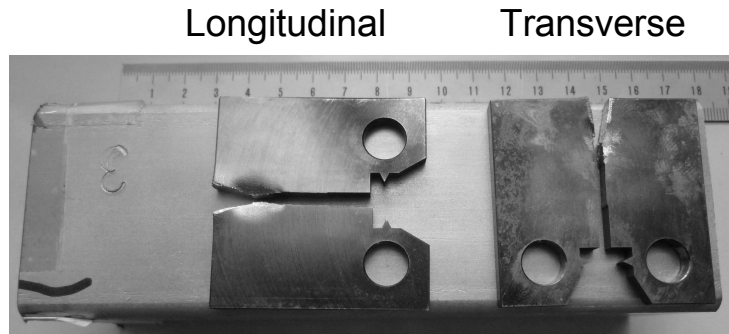
$$N_f := \left( \frac{1}{C_0 \cdot Y^m \cdot \Delta\sigma^m} \right) \cdot \int_{a_0}^{a_c} \frac{1}{(\sqrt{\pi \cdot a})^m} da \quad N_f = 31533$$

**Forman considers the load range R. Forman could determine that da/dN goes to infinity if  $K_{max}$  the critical K-value nears**

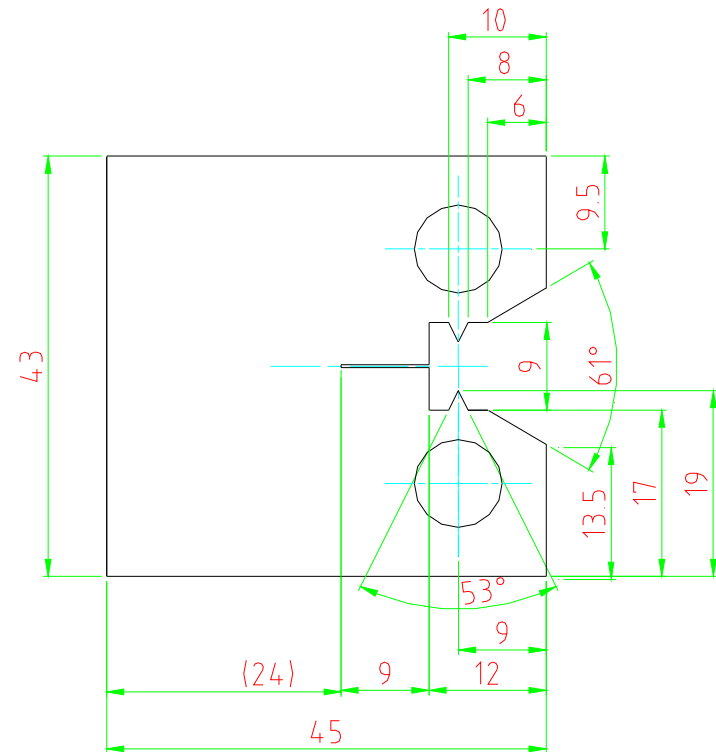
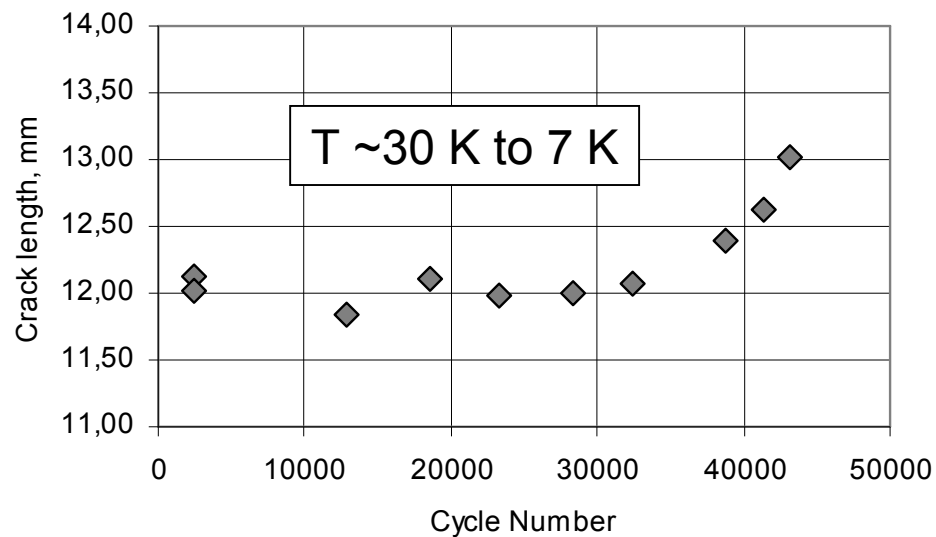
$$N_f := \int_{a_0}^{a_c} \frac{1 \cdot (1 - R) \cdot K_c - (\Delta\sigma \cdot Y \cdot S \cdot \sqrt{\pi \cdot a})}{\left[ C_0 \cdot (\Delta\sigma \cdot Y \cdot S \cdot \sqrt{\pi \cdot a})^m \right]} da \quad N_f = 198130$$



## Sample machining & specimen & test details today



Crack length versus cycle number at 7 K for  
46\_(3)\_CO+ST\_2.5%+AG\_FCGR+J1C\_LD\_1 specimen



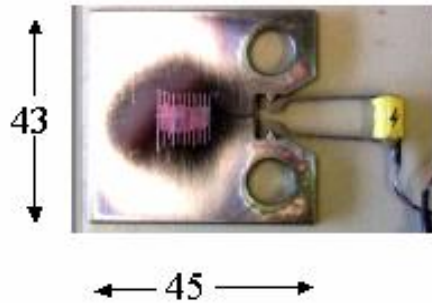
Pre-Cracking Start  
at the notch position  $a/W = 0.33$   
with  $\sim \Delta K 30 \text{ MPa}\sqrt{\text{m}}$

# Fatigue Crack Growth Rate (FCGR) Measurement Procedure

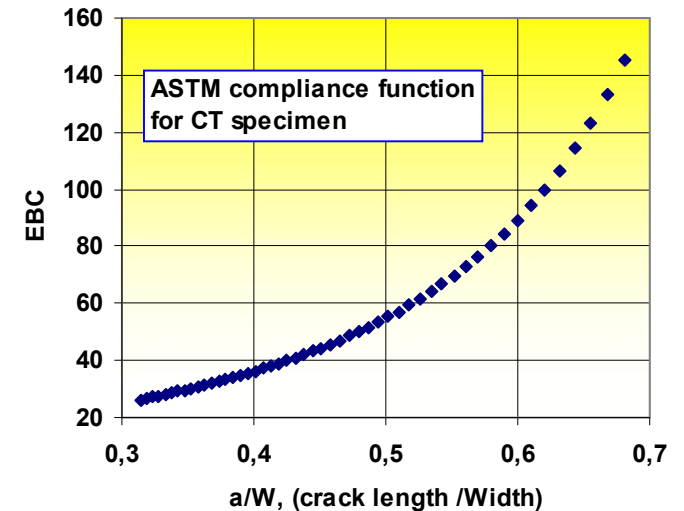
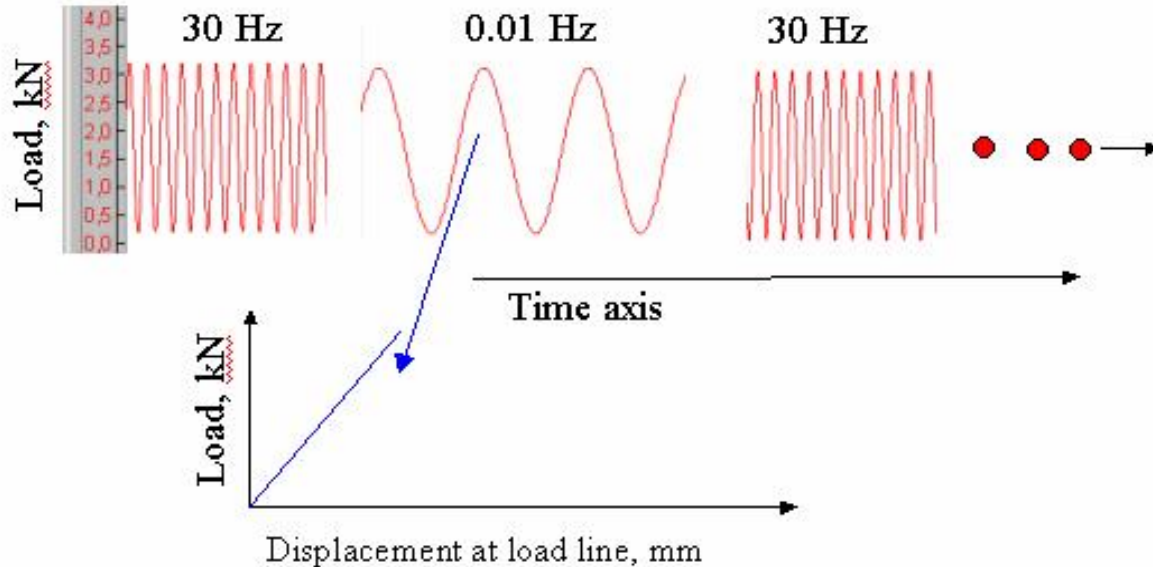
$$a/W = f( EBC )$$

$$EBC = Young's \text{ modulus} \cdot Thickness \cdot \frac{1}{Stiffness}$$

$$a/W = 0.35251 + 0.57213 \cdot \tanh(0.85068 \cdot (\log(EBC) - 1.26064))$$

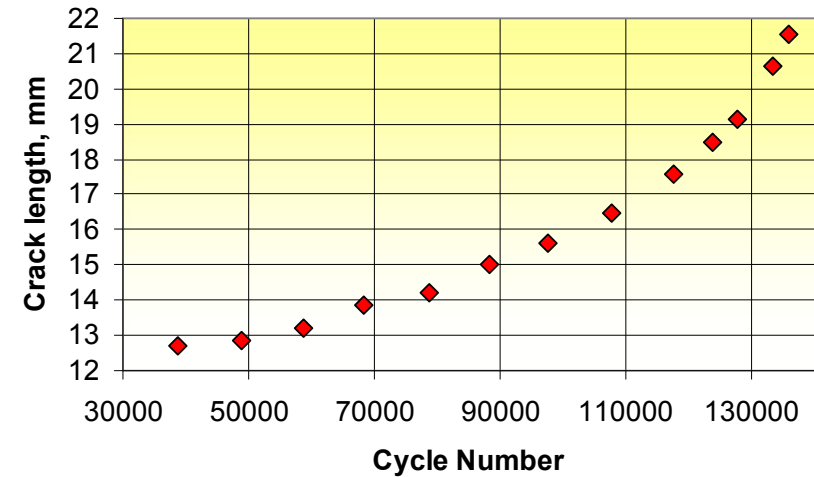


Pre-Cracking Start  
at the notch  $a/W$   
with  $\sim \Delta K 30 \text{ MPa}\sqrt{\text{m}}$

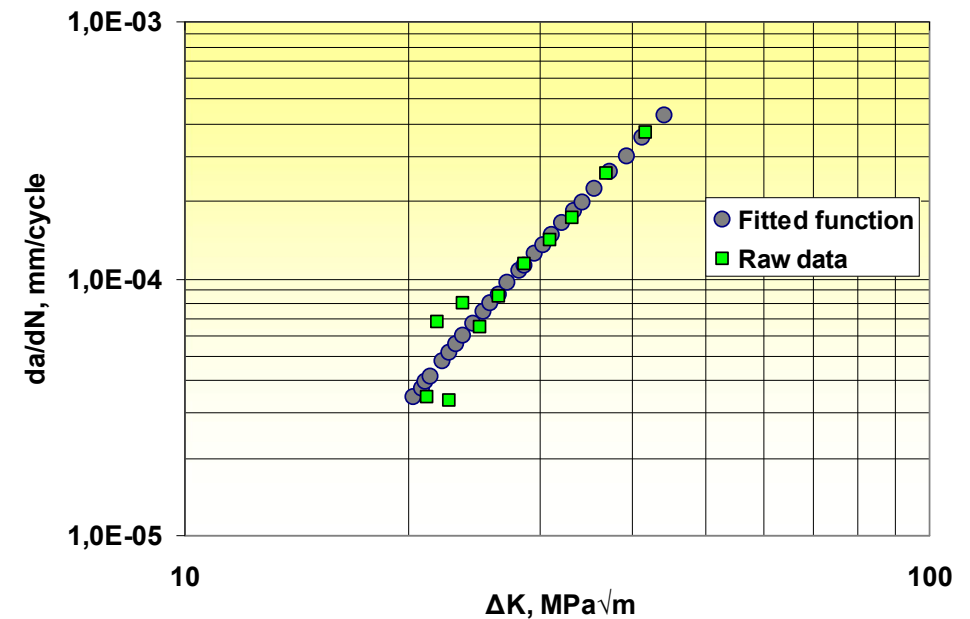


## Obtained FCGR raw data & its analysis by curve fitting

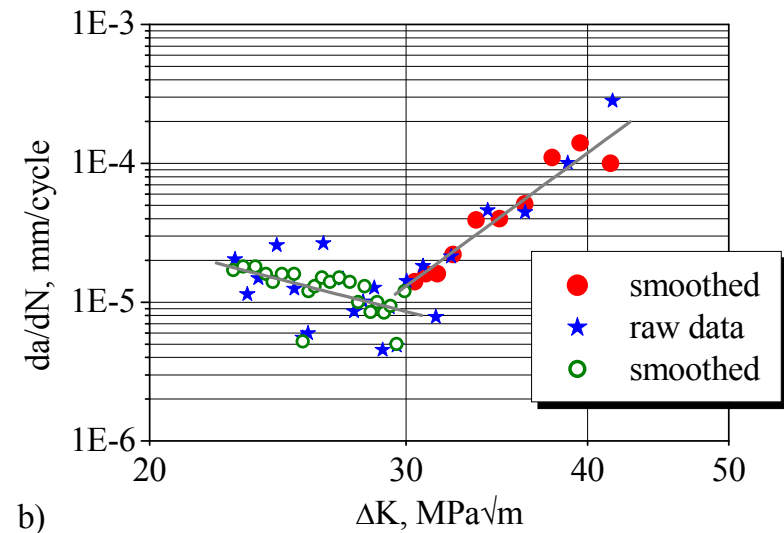
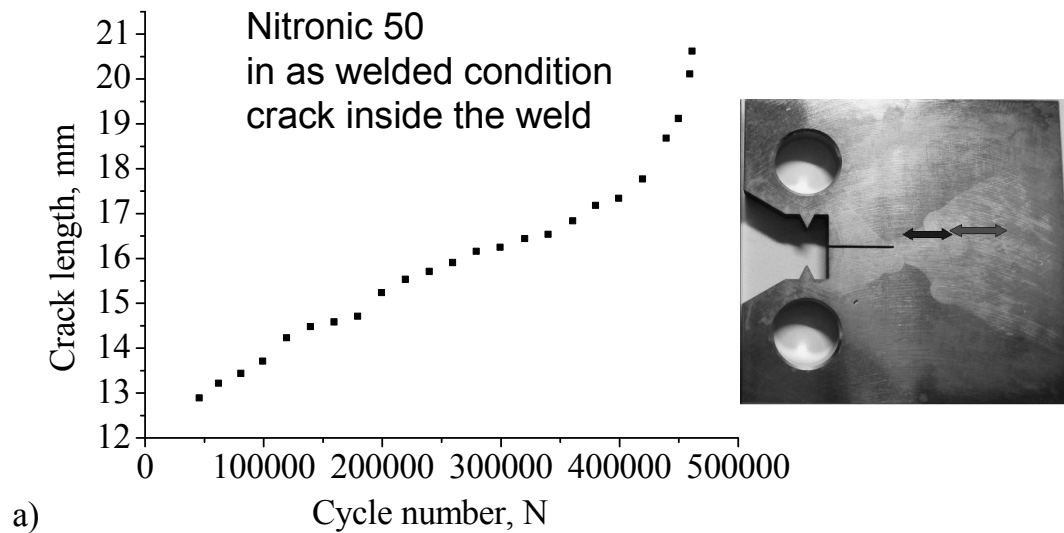
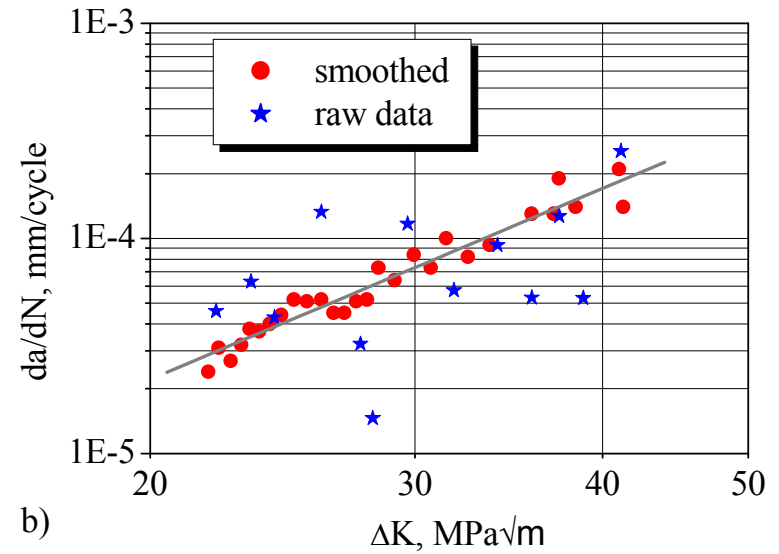
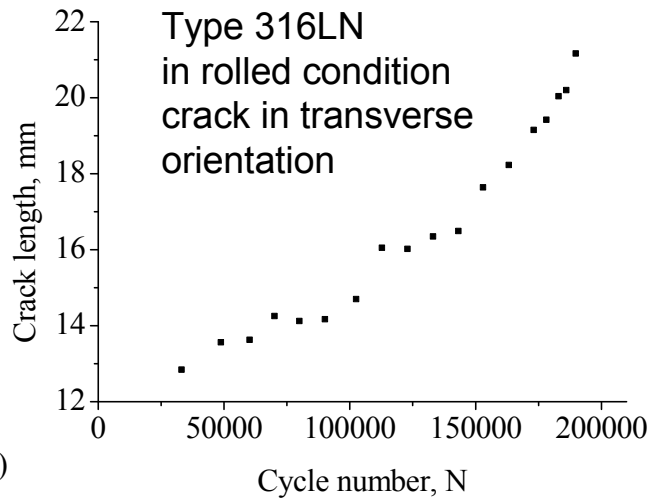
a/W	crack length, mm	Slope, kN/mm	Regression Coefficient %	Load min, kN	Load max, kN	Cycle number	$\Delta K$ , MPa $\sqrt{m}$
0,35	12,7	46759,6	99,93	0,27	2,70	38700	20,6
0,36	12,9	45799,0	99,93	0,27	2,70	48736	20,9
0,37	13,2	43828,5	99,94	0,27	2,70	58632	21,4
0,39	13,9	40253,1	99,90	0,27	2,70	68397	22,4
0,39	14,2	38515,7	99,93	0,27	2,70	78592	23,0
0,42	15,0	34783,2	99,95	0,27	2,70	88315	24,4
0,43	15,6	32156,0	99,93	0,27	2,70	97575	25,5
0,46	16,5	28690,1	99,96	0,27	2,70	107621	27,2
0,49	17,6	24631,1	99,97	0,27	2,70	117468	29,9
0,51	18,5	21816,6	99,97	0,27	2,70	123611	32,2
0,53	19,1	19819,6	99,98	0,27	2,70	127532	34,2
0,57	20,6	15868,4	99,93	0,27	2,70	133347	39,4
0,60	21,6	13643,9	99,92	0,27	2,70	135916	43,6



Specimen XX\_FCGRTD\_1  
 Raw data and smoothing of  
 raw data by using the DataFit Software



## FCGR investigations at 7 K of two different materials



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## Uncertainty as “new” tool since around 1990

### Measurement and its reliability

- For reporting the result of a measurement of a physical quantity, it should be obligatory to indicate the quality of the result quantitatively **so** that those who use it, can assess its reliability.
- Without such an indication, measurements cannot be compared. It **is** therefore necessary that there be a readily implemented, easily understood, and generally accepted procedure for characterizing the quality of a result of a measurement, that is, for evaluating and expressing its ***uncertainty***.

GUM Method : “**The Guide to Expression of Uncertainty in Measurement**” was originally published jointly by seven international organizations.

These organizations are BIPM, IEC, IFCC, ISO, IUPAC, IUPAP and OIML.

<http://www.gum.dk/e-gumfaq-publishers/GUMpublishers.html>

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## Uncertainty concept a brief description

The result of any physical measurement comprises two parts: an estimate of the true value of the measurand and the uncertainty of this “best” estimate.

One can attempt to measure the true value by measuring “**the best estimate**” using uncertainty evaluations such as Type A (repeated measurements in the laboratory in general expressed in form of Gaussian distribution) and Type B (previous experiments, literature data, manufacturer’s information, etc. in general in form of rectangular distribution) uncertainties.

Type A uncertainty

$$u_A = \frac{s}{\sqrt{N}}$$

Type B uncertainty

$$u_B = \sqrt{\left(\frac{1}{3}\right) \cdot M_1^2 + \left(\frac{1}{3}\right) \cdot M_2^2 + \dots}$$

Combined uncertainty

$$u_c = \sqrt{u_A^2 + u_B^2}$$

The GUM, within this context, is a guide for a transparent, standardized documentation of the measurement procedure.

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Model equation and the procedure of uncertainty determination

$$\frac{\partial a}{\partial N} = C_0 \cdot \Delta K^m \quad \text{Paris law (linear regime, "stage II") in double log graph}$$

$$\Delta K = \frac{\Delta P}{B \cdot \sqrt{W}} \cdot \frac{\left(2 + \frac{a}{W}\right)}{\left(1 - \frac{a}{W}\right)^{1.5}} \cdot \left(0,886 + 4,64 \cdot \frac{a}{W} - 13,32 \cdot \left(\frac{a}{W}\right)^2 + 14,72 \cdot \left(\frac{a}{W}\right)^3 - 5,6 \cdot \left(\frac{a}{W}\right)^4\right)$$

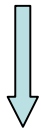
$$\frac{\partial a}{\partial N} = R = f(C_0, m, \Delta P, W, B, a)$$

## Combined standard uncertainty and its treatment

Model equation  $\longrightarrow \frac{\partial a}{\partial N} = C_0 \cdot \Delta K^m \longrightarrow \frac{\partial a}{\partial N} = R = f(C_0, m, \Delta P, W, B, a)$

Combined standard uncertainty using the Gaussian distribution law

$$u_c = \sqrt{\left(\frac{\partial R}{\partial C_0}\right)^2 u_1^2 + \left(\frac{\partial R}{\partial m}\right)^2 u_2^2 + \left(\frac{\partial R}{\partial \Delta P}\right)^2 u_3^2 + \left(\frac{\partial R}{\partial W}\right)^2 u_4^2 + \left(\frac{\partial R}{\partial B}\right)^2 u_5^2 + \left(\frac{\partial R}{\partial a}\right)^2 u_6^2}$$



$u_1 \cdots u_6$  are uncertainty coefficients,  
whilst the derivatives are the sensitivity coefficients

Partial differentiation & determination of sensitivity coefficients

$$\frac{\delta R}{\delta C_0} = \Delta K^m \quad \& \quad \frac{\delta R}{\delta m} = C_0 \cdot \Delta K^m \cdot \ln \Delta K$$

$$\frac{\partial R}{\partial \Delta P} = C_0 \cdot \Delta K^m \cdot \frac{m}{\Delta P} \quad \& \quad \frac{\partial R}{\partial B} = -C_0 \cdot \Delta K^m \cdot \frac{m}{B}$$



Partial differentials for width “W” and crack length “a”

$$\frac{\partial R}{\partial W} = C_0 \cdot \Delta K^m \cdot m \cdot \frac{[F_A + F_B + F_C + F_D]}{\Delta P} \cdot B \cdot F_E$$

$$\frac{\partial R}{\partial a} = C_0 \cdot \Delta K^m \cdot m \cdot \frac{\left[ -\frac{2 \cdot F_A}{\left(2 + \frac{a}{W}\right)} + F_F - \frac{F_D \cdot \sqrt[5]{W}}{\sqrt[3]{W}} \right]}{\Delta P} \cdot B \cdot F_E$$

## Functions $F_A$ ..... $F_F$ of the partial derivative equation

$$F_A = -\Delta P \cdot \left(2 + \frac{a}{W}\right) \cdot \frac{\left(8.886 + 4.64 \cdot \left(\frac{a}{W}\right) - 13.32 \cdot \left(\frac{a}{W}\right)^2 + 14.72 \cdot \left(\frac{a}{W}\right)^3 - 5.6 \cdot \left(\frac{a}{W}\right)^4\right)}{\left(1 - \frac{a}{W}\right)^{1.5} \cdot 2 \cdot B \cdot \sqrt[3]{W}}$$

$$F_B = -\Delta P \cdot a \cdot \frac{\left(8.886 + 4.64 \cdot \left(\frac{a}{W}\right) - 13.32 \cdot \left(\frac{a}{W}\right)^2 + 14.72 \cdot \left(\frac{a}{W}\right)^3 - 5.6 \cdot \left(\frac{a}{W}\right)^4\right)}{\left(1 - \frac{a}{W}\right)^{1.5} \cdot B \cdot \sqrt[3]{W}}$$

$$F_C = \Delta P \cdot \frac{\left(2 + \frac{a}{W}\right) \cdot \left(-4.64 \cdot \frac{a}{W^2} + 26.64 \cdot \frac{a^2}{W^3} - 44.16 \cdot \frac{a^3}{W^4} + 22.4 \cdot \frac{a^4}{W^5}\right)}{\left(1 - \frac{a}{W}\right)^{1.5} \cdot B \cdot \sqrt{W}}$$

$$F_D = -1.5 \cdot \Delta P \cdot \frac{\left(2 + \frac{a}{W}\right) \cdot \left(8.886 + 4.64 \cdot \left(\frac{a}{W}\right) - 13.32 \cdot \left(\frac{a}{W}\right)^2 + 14.72 \cdot \left(\frac{a}{W}\right)^3 - 5.6 \cdot \left(\frac{a}{W}\right)^4\right)}{\left(1 - \frac{a}{W}\right)^{2.5} \cdot B \cdot \sqrt[3]{W}}$$

$$F_E = \frac{\sqrt{W}}{\left(8.886 + 4.64 \cdot \frac{a}{W} - 13.32 \cdot \left(\frac{a}{W}\right)^2 + 14.72 \cdot \left(\frac{a}{W}\right)^3 - 5.6 \cdot \left(\frac{a}{W}\right)^4\right)} \cdot \frac{\left(1 - \frac{a}{W}\right)^{1.5}}{\left(2 + \frac{a}{W}\right)}$$

$$F_F = \Delta P \cdot \frac{\left(2 + \frac{a}{W}\right) \cdot \left(4.64 \cdot \frac{1}{W} - 26.64 \cdot \frac{a}{W^2} + 44.16 \cdot \frac{a^2}{W^3} - 22.4 \cdot \frac{a^3}{W^4}\right)}{\left(1 - \frac{a}{W}\right)^{1.5} \cdot B \cdot \sqrt{W}}$$

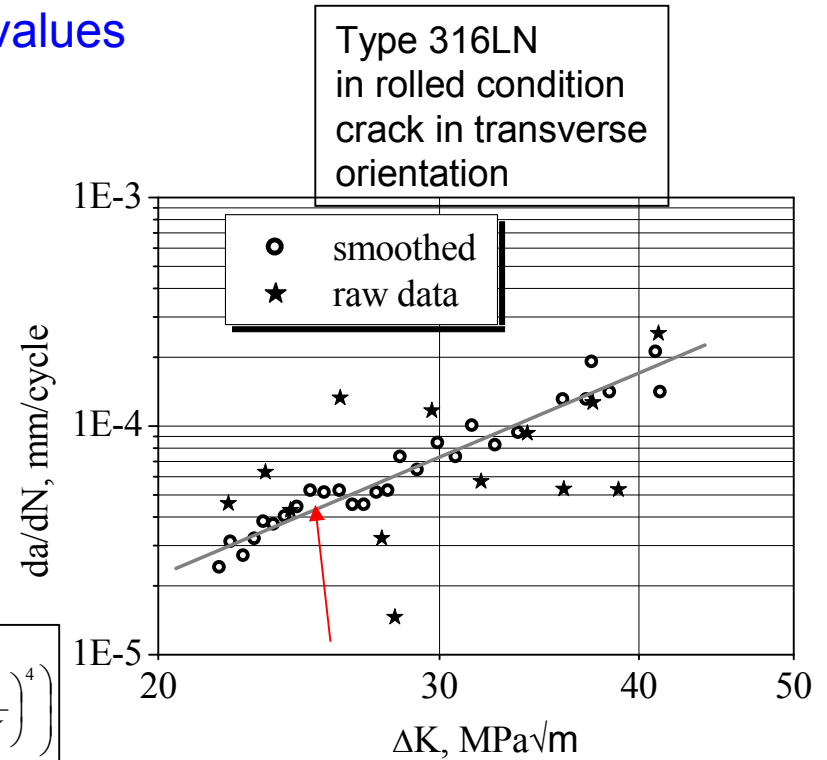
## Computation of sensitivity values

Computed Paris law constants  
 $C_0 = 3.226 \cdot 10^{-9}$  &  $m = 2.949$

with  
 $a = 1.5 \text{ cm}$ ,  $W = 3.6 \text{ cm}$ ,  $B = 0.4 \text{ cm}$ ,  $\Delta P = 2.5 \text{ kN}$

$$\Delta K = \frac{\Delta P}{B \cdot \sqrt{W}} \cdot \frac{\left(2 + \frac{a}{W}\right)}{\left(1 - \frac{a}{W}\right)^{1.5}} \cdot \left(0,886 + 4,64 \cdot \frac{a}{W} - 13,32 \cdot \left(\frac{a}{W}\right)^2 + 14,72 \cdot \left(\frac{a}{W}\right)^3 - 5,6 \cdot \left(\frac{a}{W}\right)^4\right)$$

$\Delta K$  is  $\sim 25 \text{ MPa}\sqrt{\text{m}}$



Results of the computed partial differentials (sensitivity coefficients)

$\frac{da}{dN}$	$\Delta K$	$\frac{\partial R}{\partial C_0}$	$\frac{\partial R}{\partial m}$	$\frac{\partial R}{\partial \Delta P}$	$\frac{\partial R}{\partial W}$	$\frac{\partial R}{\partial B}$	$\frac{\partial R}{\partial a}$
$4.311 \cdot 10^{-5}$	25.066	$1.336 \cdot 10^4$	$1.918 \cdot 10^4$	$5.085 \cdot 10^{-5}$	$-5.724 \cdot 10^{-5}$	$-3.178 \cdot 10^{-4}$	$9.50 \cdot 10^{-5}$

## Uncertainty terms for m and C<sub>0</sub> and its treatment

Assumption: experimental errors are identical for all four tests as the uncertainties resulting from the geometrical constraints has been considered elsewhere. Therefore, m and C<sub>0</sub> are determined using the experimental results.

Specimen, condition, & code	C <sub>0</sub> , specimen # 1	C <sub>0</sub> , specimen # 2	absolute difference	m, specimen # 1	m, specimen # 2	absolute difference
316LN rolled plate, longitudinal	1.79·10 <sup>-9</sup>	2.65·10 <sup>-9</sup>	8.64·10 <sup>-10</sup>	3.092	3.012	0.080
316LN rolled plate, transversal	3.23·10 <sup>-9</sup>	1.37·10 <sup>-9</sup>	1.86·10 <sup>-9</sup>	2.949	3.193	0.244

For m, a, C<sub>0</sub>, and, ΔP treatment according to Type B. W and B according to Type A

$$u_c = \sqrt{\left(1.336 \cdot 10^4\right)^2 \cdot \left(3.93 \cdot 10^{-10}\right)^2 + \left(1.918 \cdot 10^{-4}\right)^2 \cdot (0.047)^2 + \left(5.085 \cdot 10^{-5}\right)^2 \cdot (0.0008)^2 + \left(-5.724 \cdot 10^{-5}\right)^2 \cdot (0.002)^2 + \left(-3.178 \cdot 10^{-4}\right)^2 \cdot (0.002)^2 + \left(9.5 \cdot 10^{-5}\right)^2 \cdot (0.115)^2} \quad \frac{mm}{cycle}$$

$$u_c = 1.51 \cdot 10^{-5} \quad \frac{mm}{cycle}$$

$$\frac{da}{dN} = R = 4.311 \cdot 10^{-5} \quad \pm \quad 1.51 \cdot 10^{-5} \quad \frac{mm}{Cycles} \quad for \quad \Delta K = 25.07 \quad MPa\sqrt{m}$$

## Best estimate and the uncertainty of the measurement

