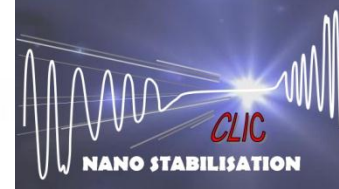




European Organization for Nuclear Research



FINAL FOCUS: COMBINATION OF PRE ISOLATOR AND ACTIVE STABILISATION

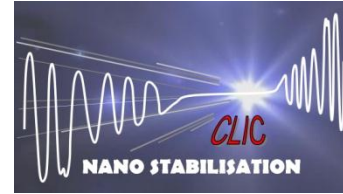
K. Artoos, C. Collette, R. Leuxe, C.Eymin, P. Fernandez, S. Janssens*



The research leading to these results has received funding from the European Commission under the FP7 Research Infrastructures project EuCARD



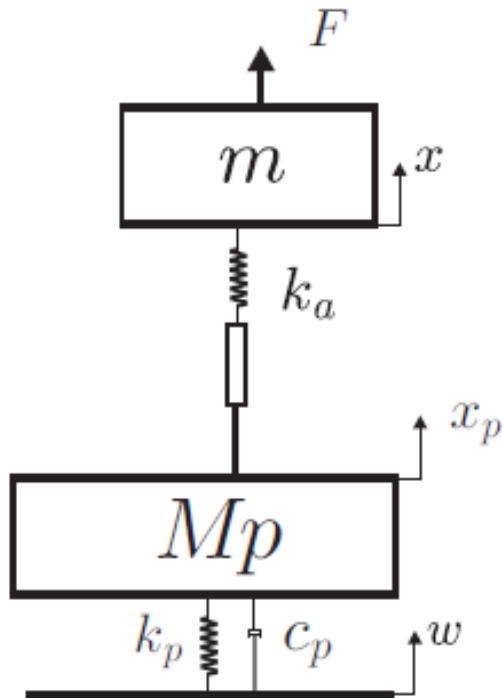
Outline



2

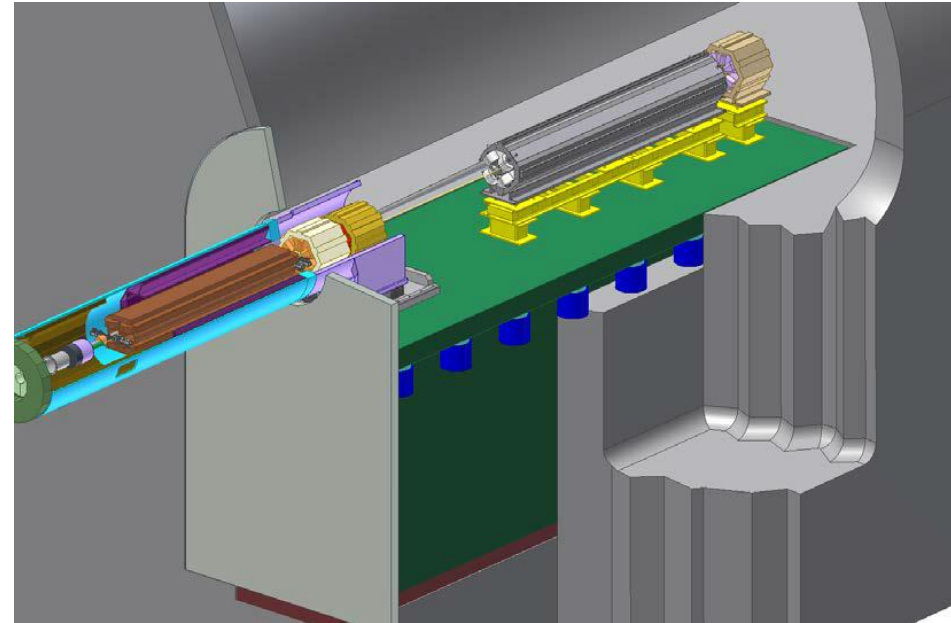
- How to perform active control on the preisolator to damp the 1 Hz resonance
- Issues with using the preisolator as it is
- Further research proposal
- Question about accelerometer feedback

$$M\ddot{X} + C\dot{X} + KX = F$$



$m=50$
 $k_a=480e6$
 $\xi=0.01$

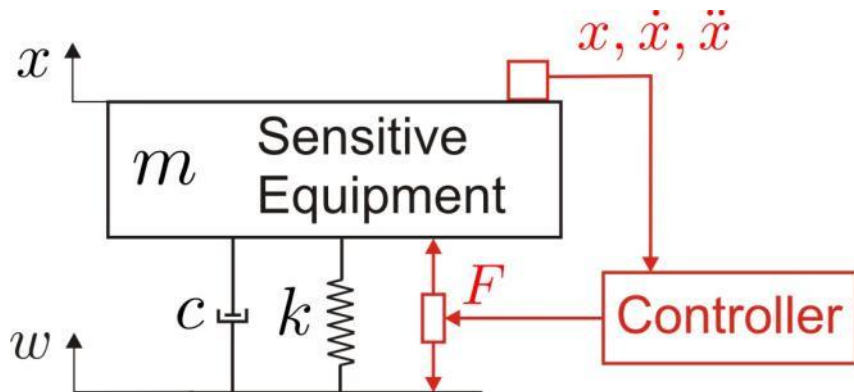
$M_p = 40000$
 $f_p = 1 \text{ Hz}$
 $\xi_p=0.01$



What is needed to damp/influence peak of pre-isolator mode?

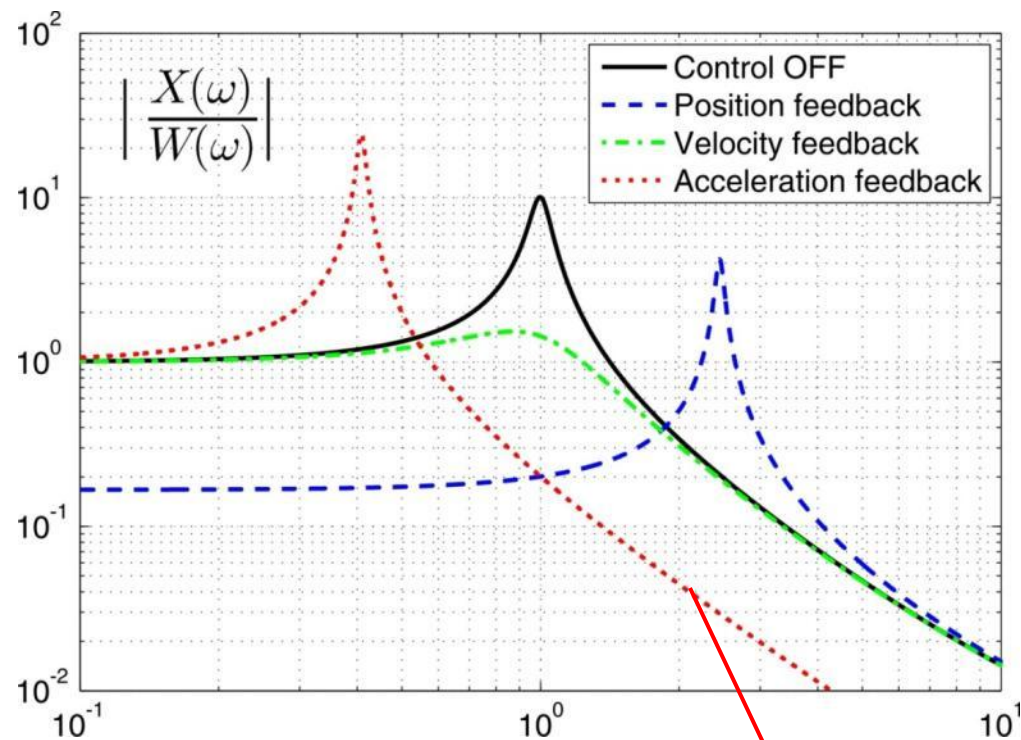
Perfect feedback (no filtering and sensors)

Feedback control principle



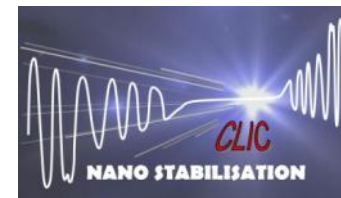
$$F(t) = k_d \dot{x} + k_v \ddot{x} + k_a \ddot{x}$$

$$\frac{X(s)}{W(s)} = \frac{cs+k}{(m+k_a)s^2+(c+k_v)s+(k+k_d)}$$



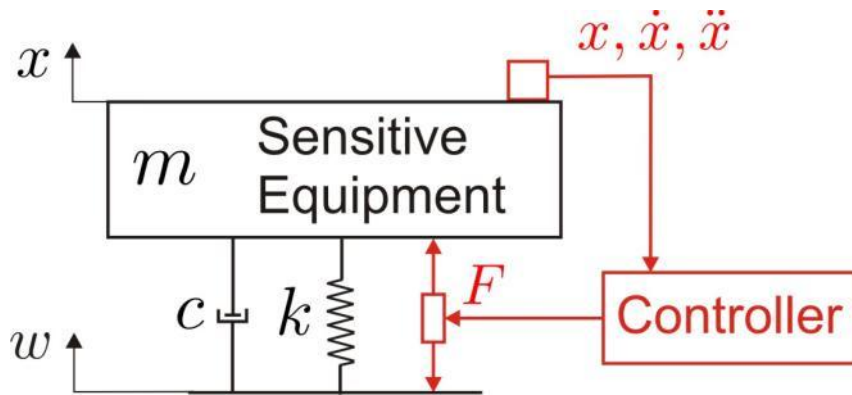
Add virtual mass

CLIC Active Isolation Strategies



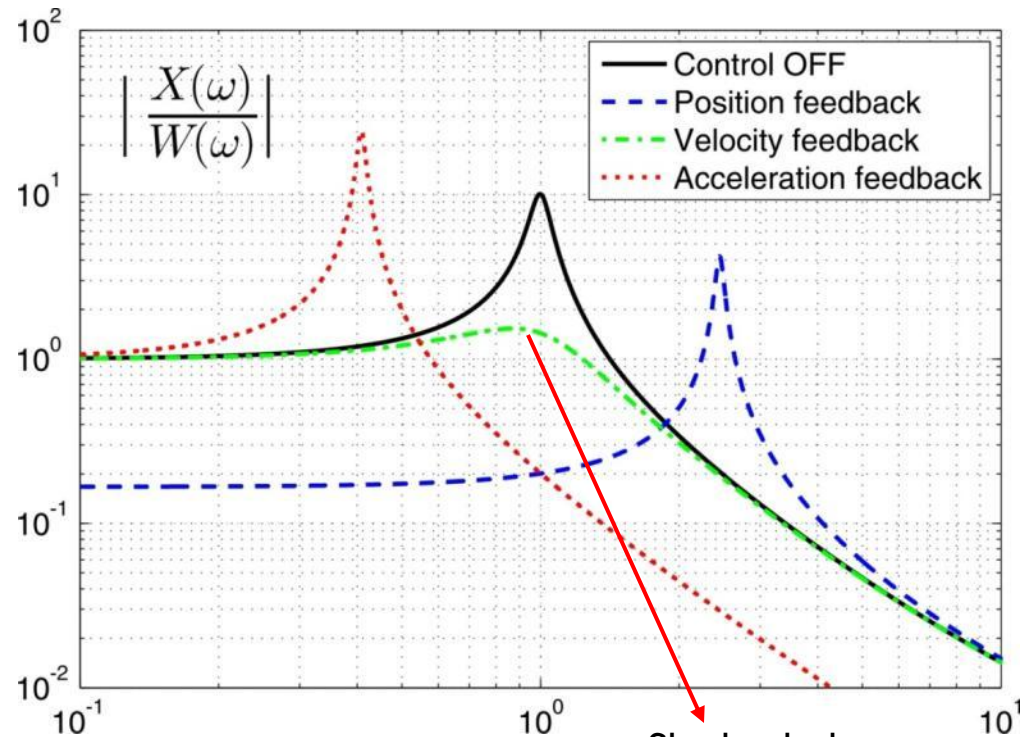
5

Feedback control principle

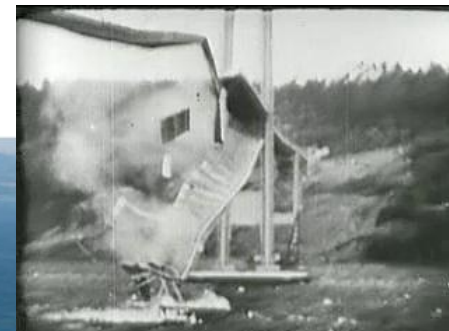


$$F(t) = k_d \dot{x} + k_v \ddot{x} + k_a \ddot{x}$$

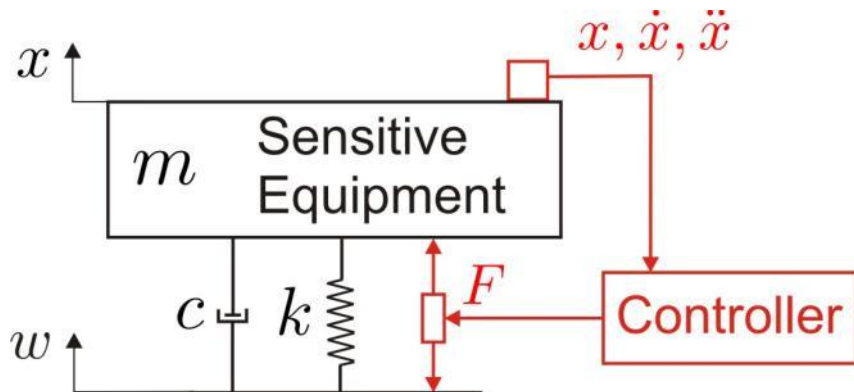
$$\frac{X(s)}{W(s)} = \frac{cs+k}{(m+k_a)s^2+(c+k_v)s+(k+k_d)}$$



Sky-hook damper
(D.C. Karnopp, 1969)

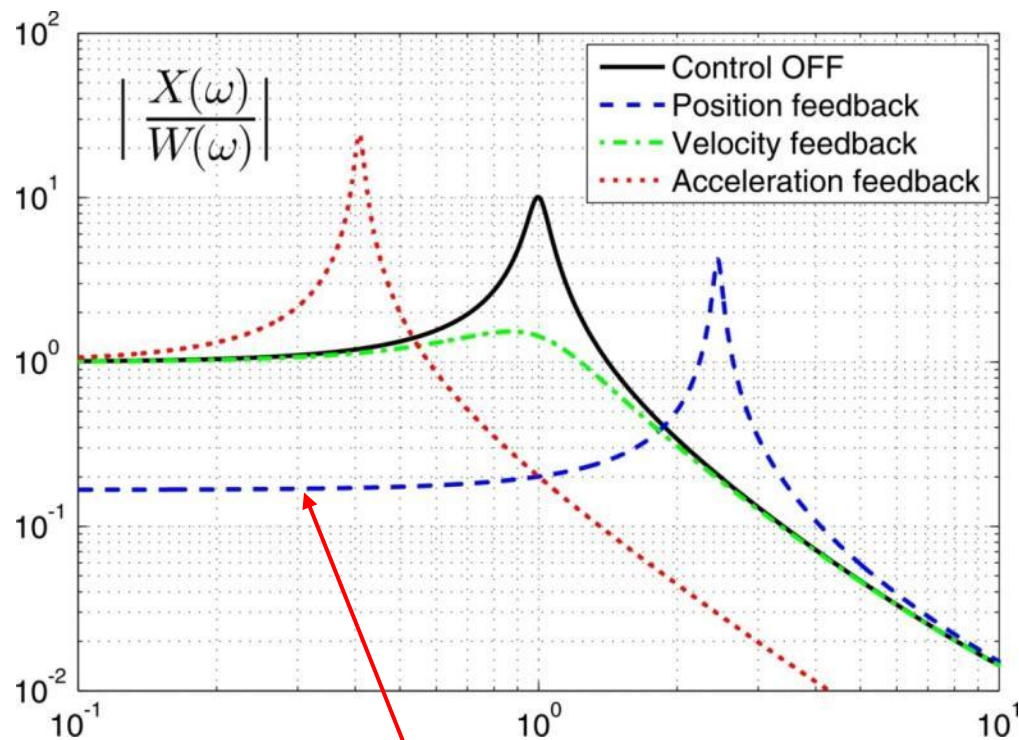


Feedback control principle



$$F(t) = k_d x + k_v \dot{x} + k_a \ddot{x}$$

$$\frac{X(s)}{W(s)} = \frac{cs+k}{(m+k_a)s^2+(c+k_v)s+(k+k_d)}$$

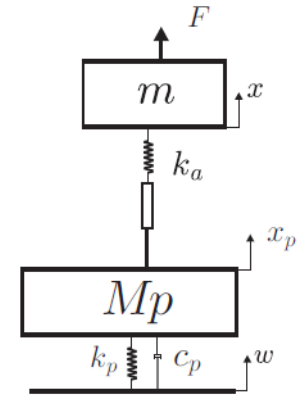


Virtual stiffness added

CLIC Active Isolation Strategies

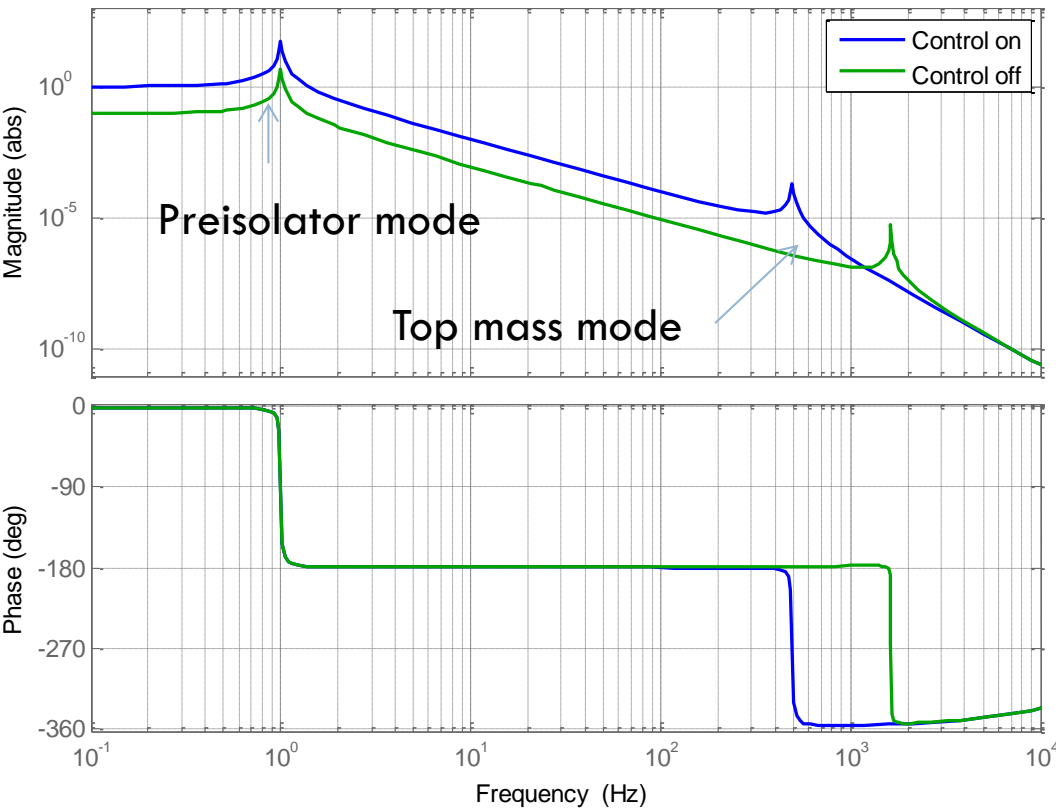
7

Qd0 stabilization system as it is
Position feedback x



Closed Twx

Bode Diagram



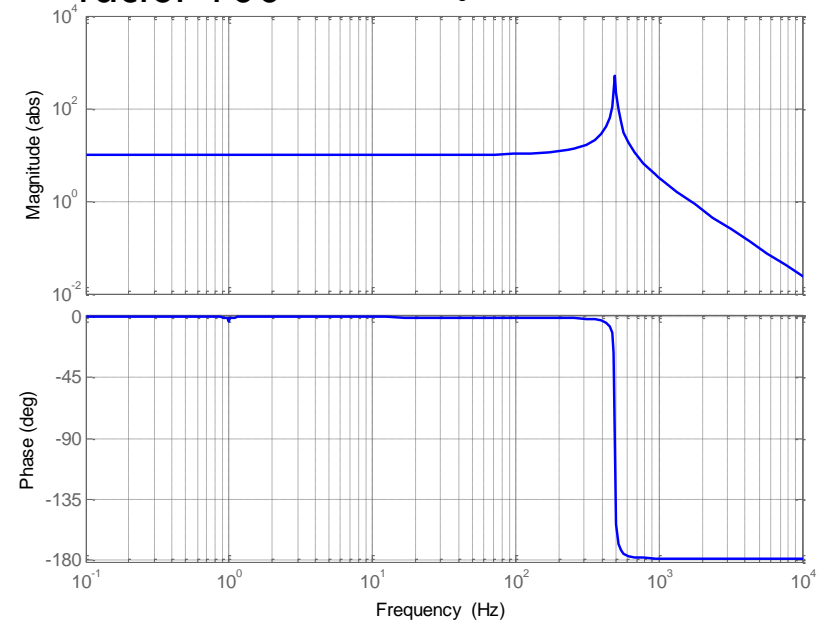
Reduces transmissibility

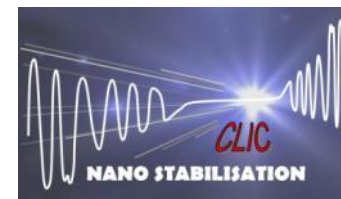
Doesn't damp first mode

Always stable

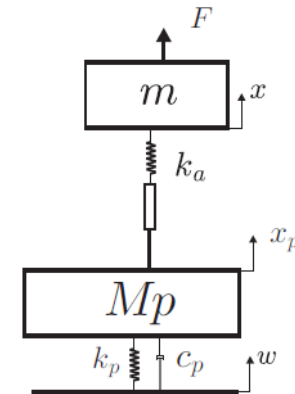
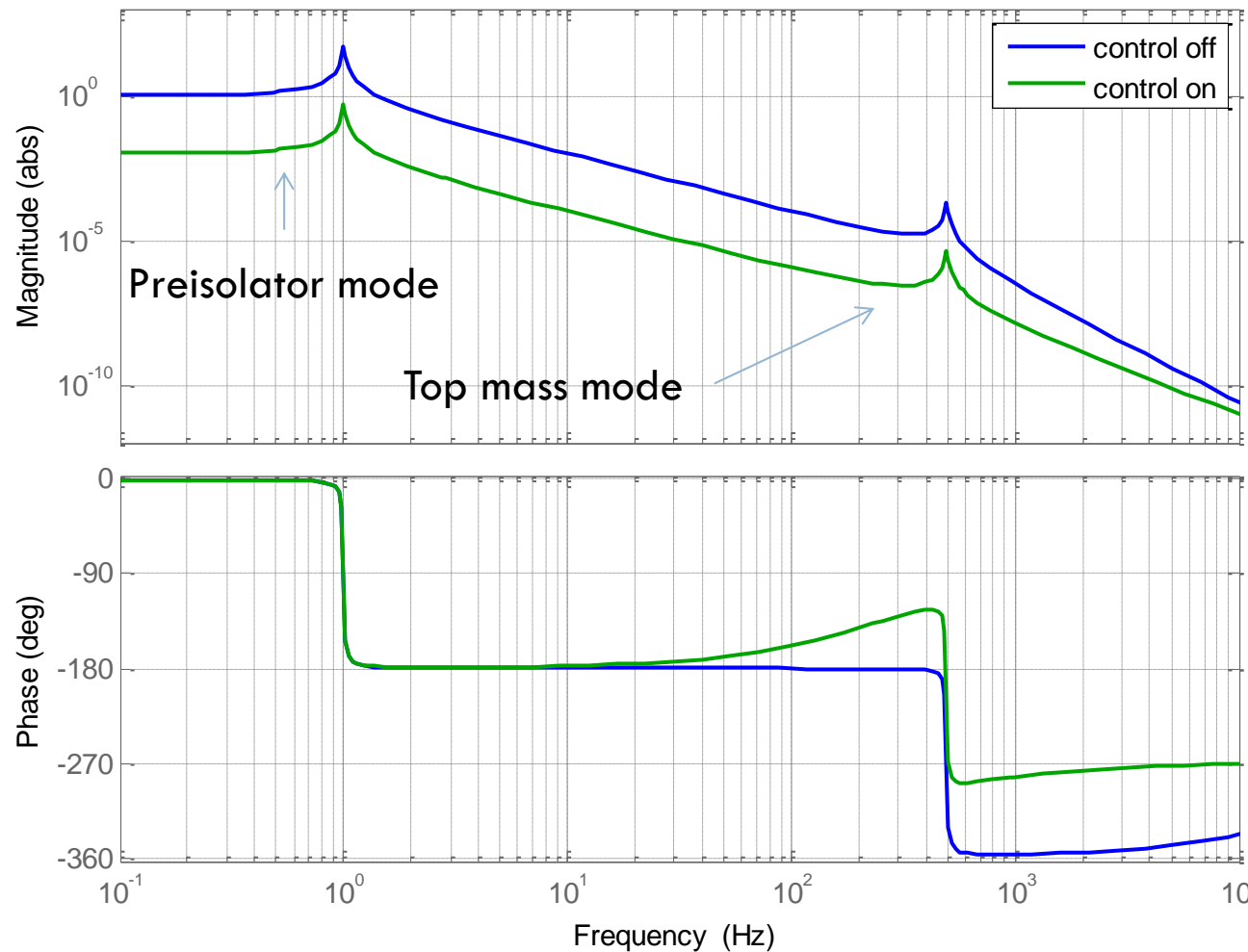
Authority at 1 Hz needed with
factor 100

Bode Diagram





Bode Diagram

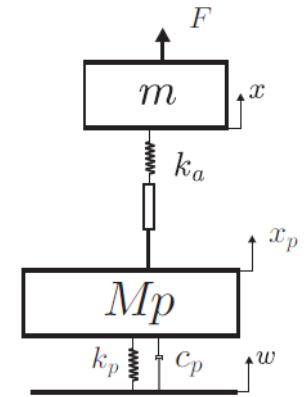


QDO stabilization as it is Feedforward of x_p

- Reduction of transmissibility
- No damping of the mode
- Top mass mode doesn't move (as no added stiffness due to ff)
- Authority at 1 Hz needed with factor 100

CLIC Active Isolation Strategies

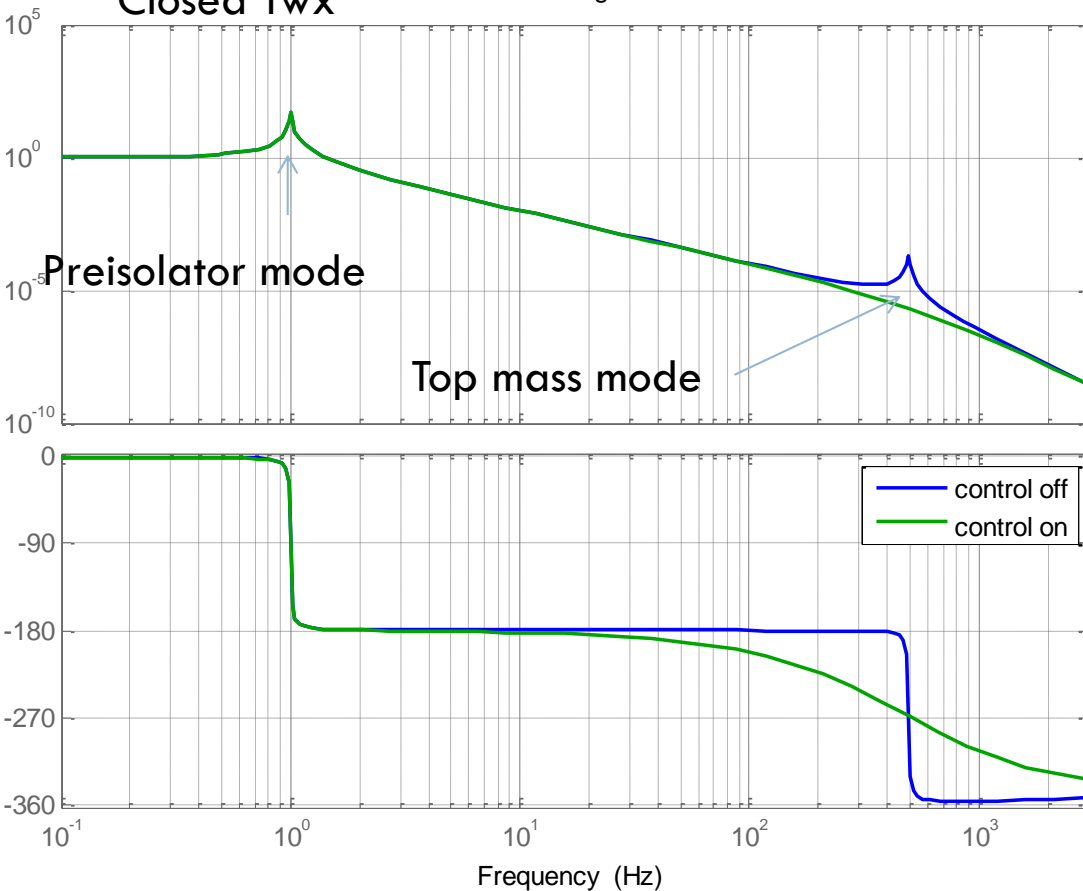
9



Damping with velocity feedback \dot{x}

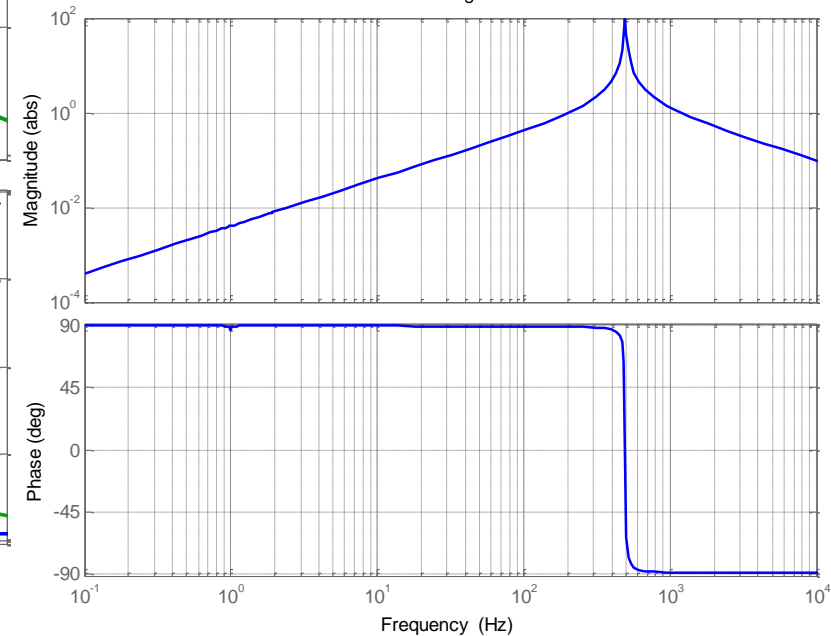
Closed T_{wx}

Bode Diagram



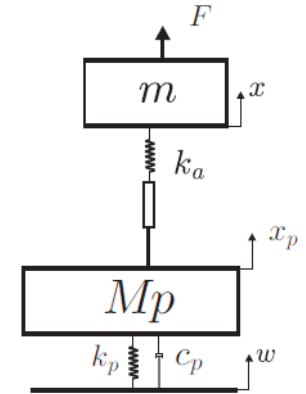
Only effect on second mode
Always stable
(similar results with IFF+noise issue)

Bode Diagram

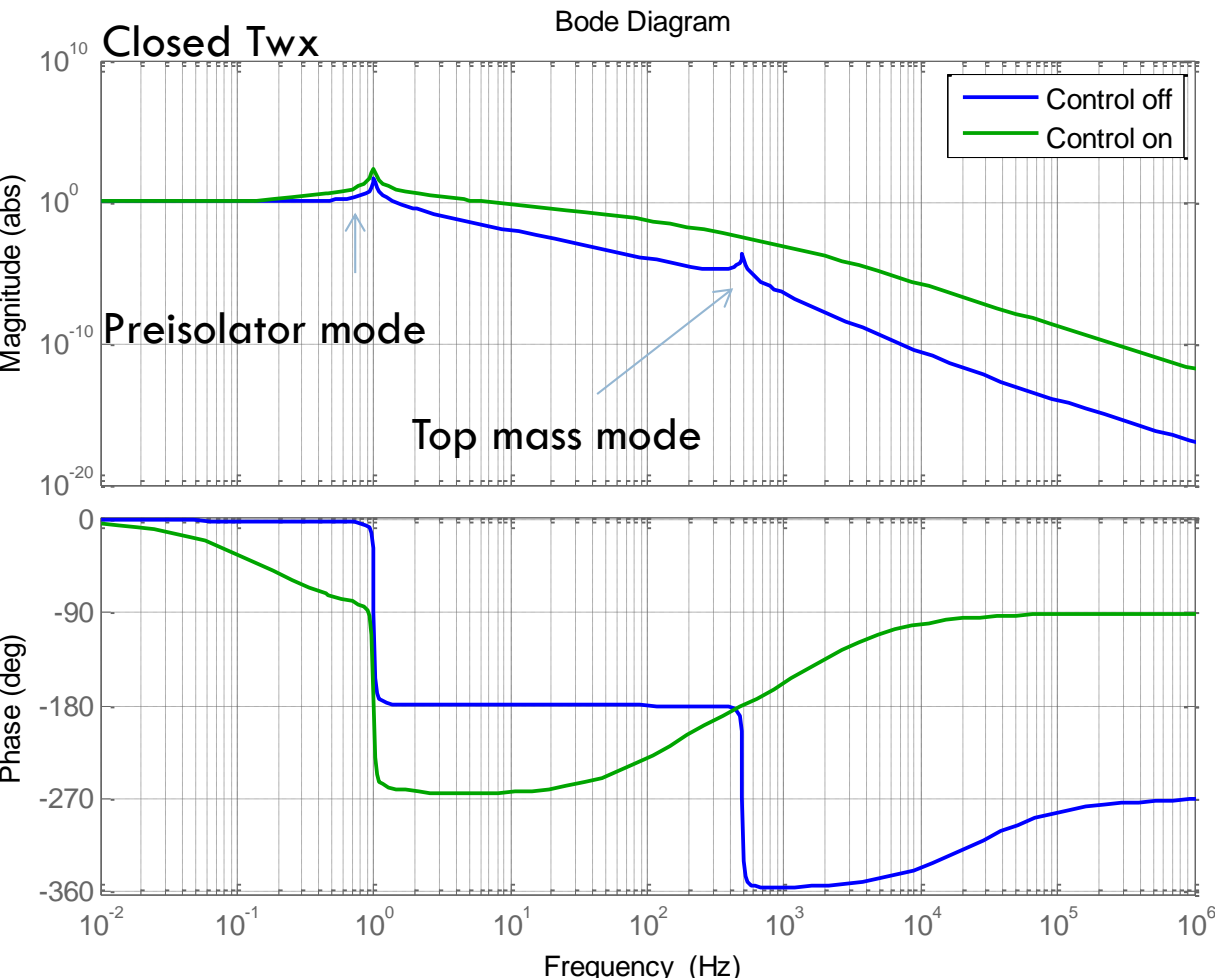


CLIC Active Isolation Strategies

10



Velocity feedforward \dot{x}_p



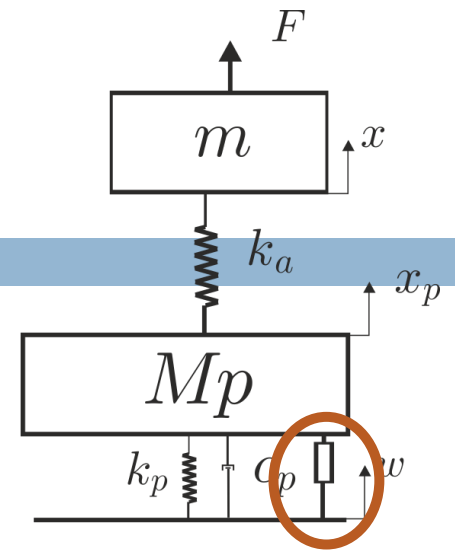
Doesn't work as it just injects noise due to phase difference



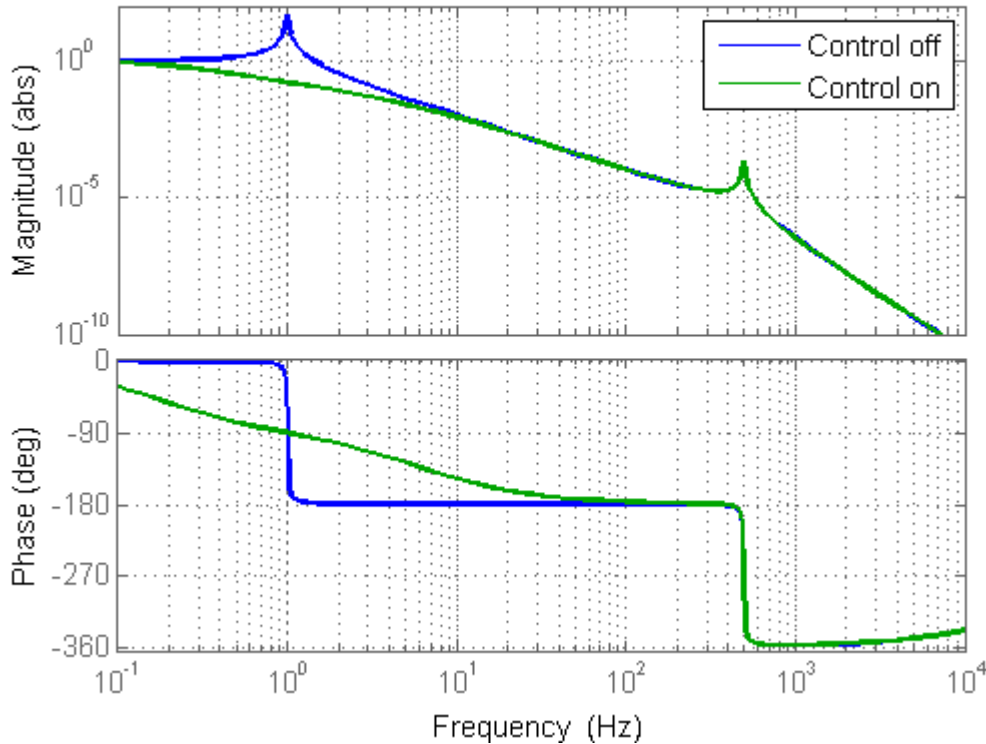
Active Isolation Strategies

Proposed velocity feedback of \dot{x}_p

Damps first mode critically

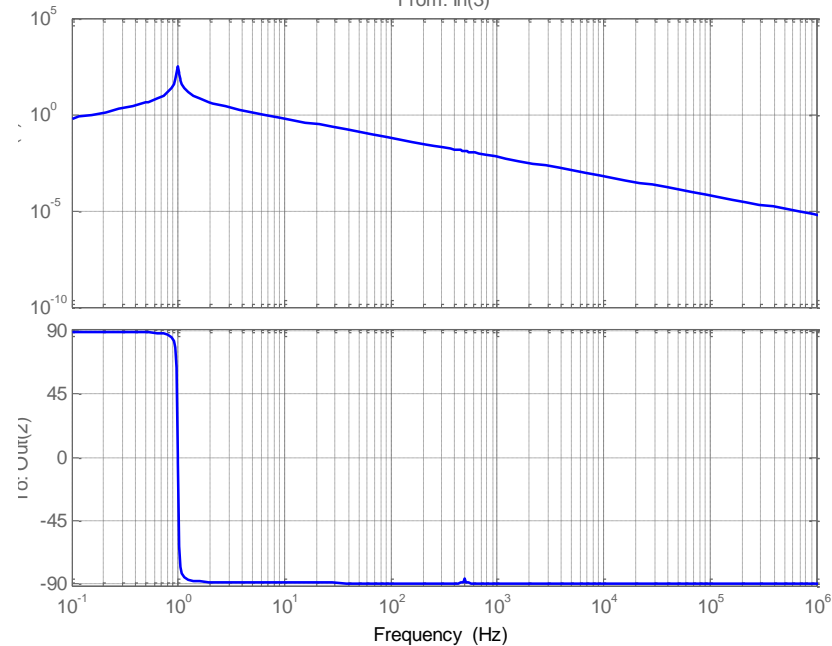


Bode Diagram

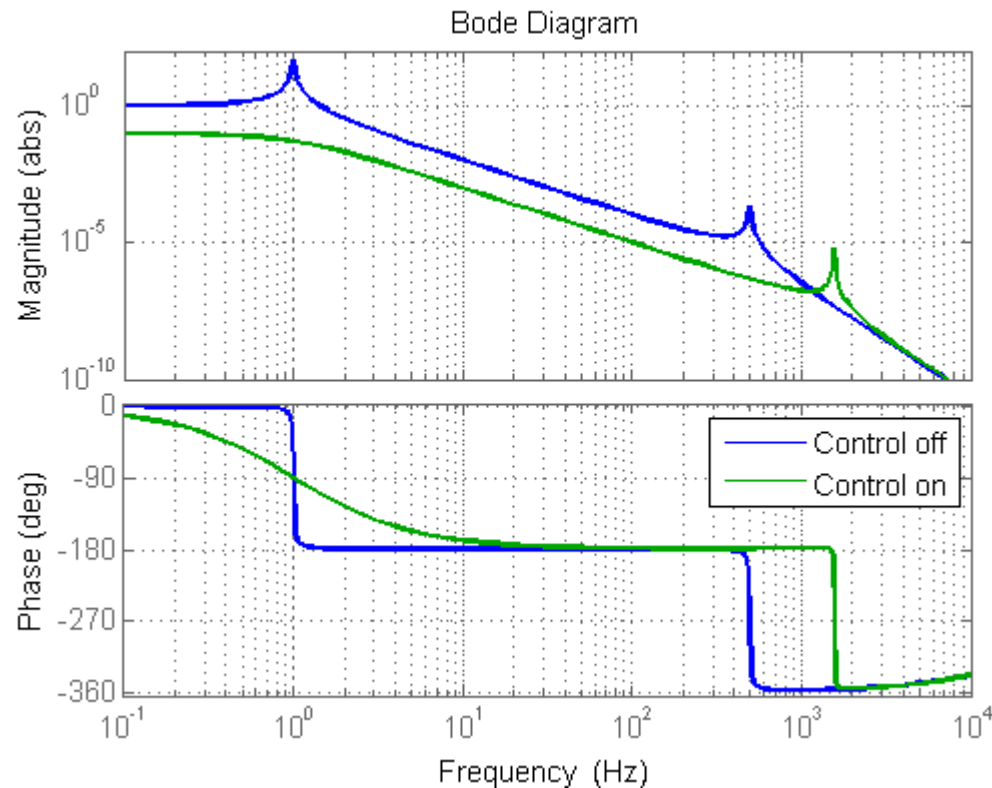
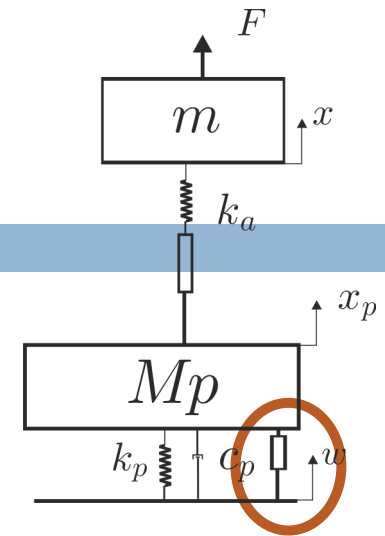


Bode Diagram

From: ln(3)

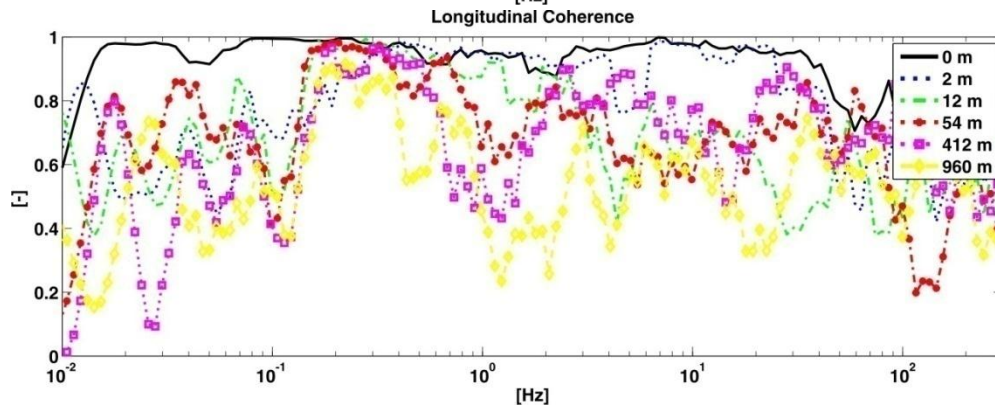
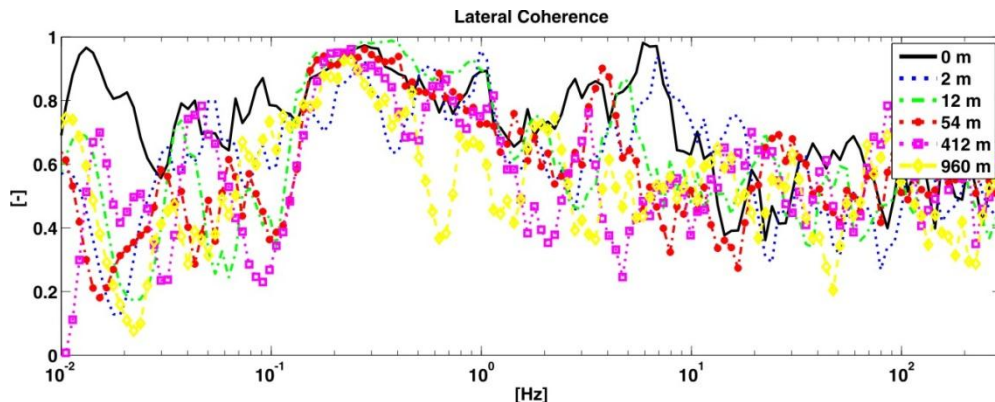
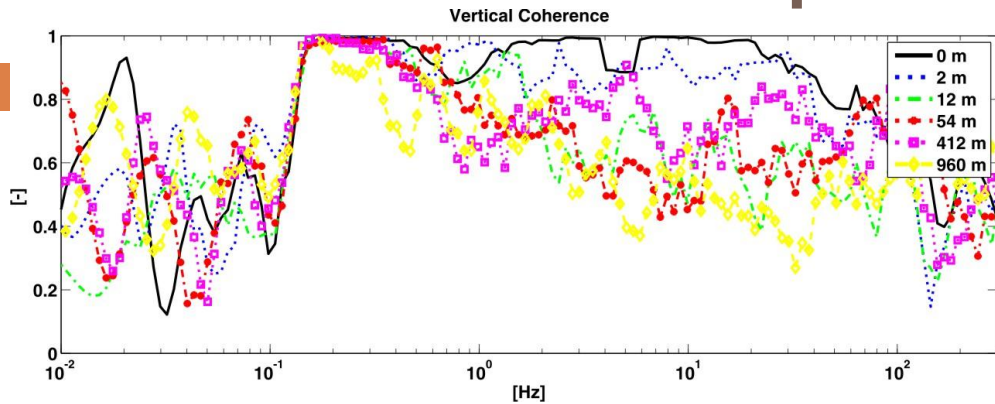


Combination of x and \dot{x}_p feedback



Damps the peak and reduces transmissibility

Issues for the preisolator



3D coherence of ground motion

Measurements performed in LHC tunnel
by K. Artoos and M. Guinchard.

Ground motion > 12 m @ 1 Hz is not
coherent

\Rightarrow Active damping and transmissibility
reduction necessary at 1 Hz

$$k_p = \omega_p^2 M_p = 6.31 \text{ N}/\mu\text{m}$$

$$F_{air} = \frac{1}{2} \rho v^2 A$$

$$\rho_{air} = 1.1644 \text{ kg}/\text{m}^3 @ 30^\circ\text{C}$$

$$A = 8\text{m} \times 2.5\text{m}$$

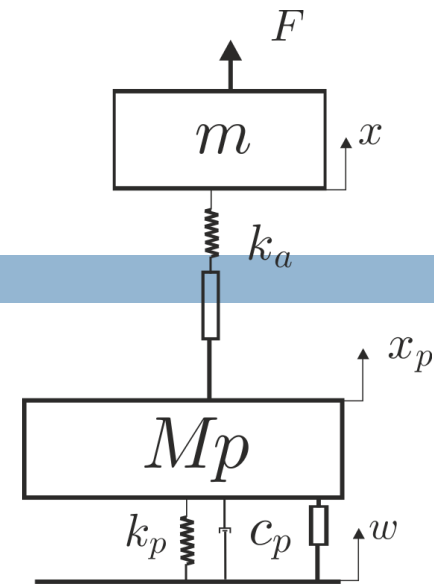
$$F_{air} = 11 \text{ N} @ 1 \text{ m}/\text{s}$$

$$\delta_{air} = 1.7 \mu\text{m}!$$

$$F_{air} = 0.11 \text{ N} @ 0.1 \text{ m}/\text{s}$$

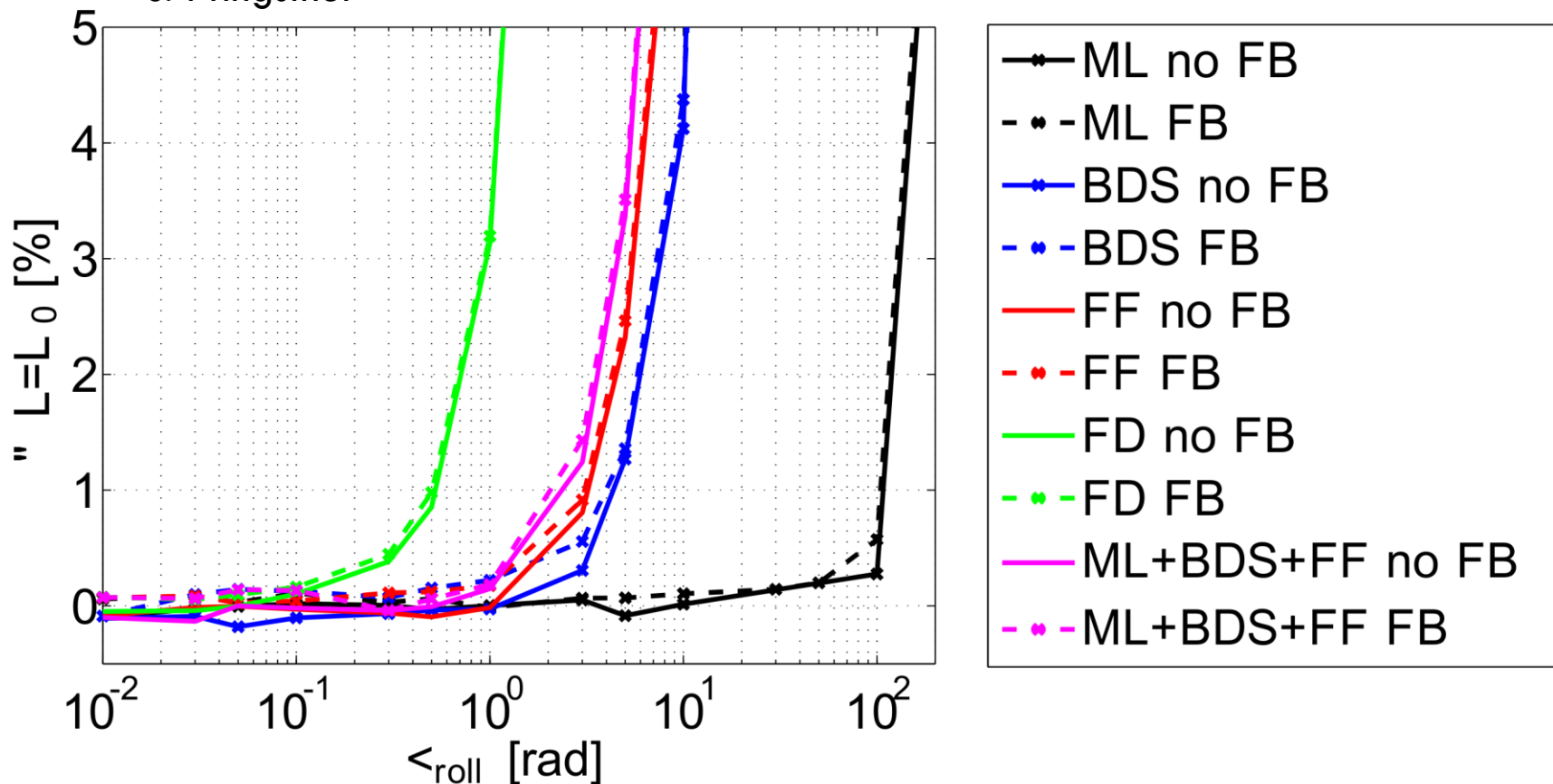
$$\delta_{air} = 17 \text{ nm}!$$

+pressure on vertical plane on surface of magnets which has a moment arm!! +other noise sources coming from the detector
=> Increasing with position feedback necessary!



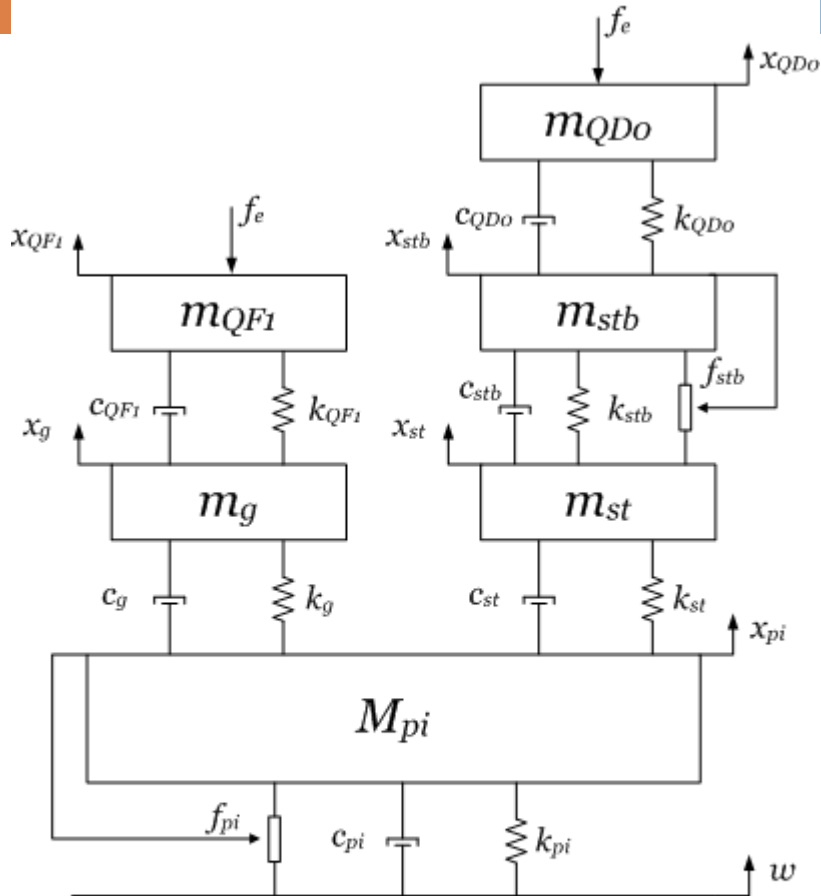
Preliminary max roll simulations vs luminosity loss
 => Max 1 μ rad!

J. Pfingstner





Further simulations with PID



More complex model has been made

With PID feedback for $x_p \Rightarrow$ improved position, damping and compliance

With proposed CERN MBQ stabilization (as example)

$$M_{pi}\ddot{x}_{pi} + k_{pi}(x_{pi} - w) + k_{st}(x_{pi} - x_{st}) + k_g(x_{pi} - x_g) = k_{pi}\delta_{pi}$$

$$M_g\ddot{x}_g + k_g(x_g - x_{pi}) + k_{QF1}(x_g - x_{QF1}) = 0$$

$$M_{QF1}\ddot{x}_{QF1} + k_{QF1}(x_{QF1} - x_g) = f_{eQF1}$$

$$M_{st}\ddot{x}_{st} + k_{st}(x_{st} - x_{pi}) + k_{stb}(x_{st} - x_{stb}) = k_{stb}\delta_{stb}$$

$$M_{stb}\ddot{x}_{stb} + k_{stb}(x_{stb} - x_{st}) + k_{QD0}(x_{stb} - x_{QD0}) = -k_{stb}\delta_{stb}$$

$$M_{QD0}\ddot{x}_{QD0} + k_{QD0}(x_{QD0} - x_{stb}) = f_{eQD0}$$

$$M_g = 50$$

$$M_{QD0} = 15000$$

$$M_{pi} = 40000$$

$$M_{QF1} = 15000$$

$$M_{st} = 40$$

$$M_{stb} = 150$$

$$w_g = 2\pi 200$$

$$w_{QD0} = 2\pi 50$$

$$w_{pi} = 2\pi 2$$

$$w_{QF1} = 2\pi 100$$

$$w_{st} = 2\pi 300$$

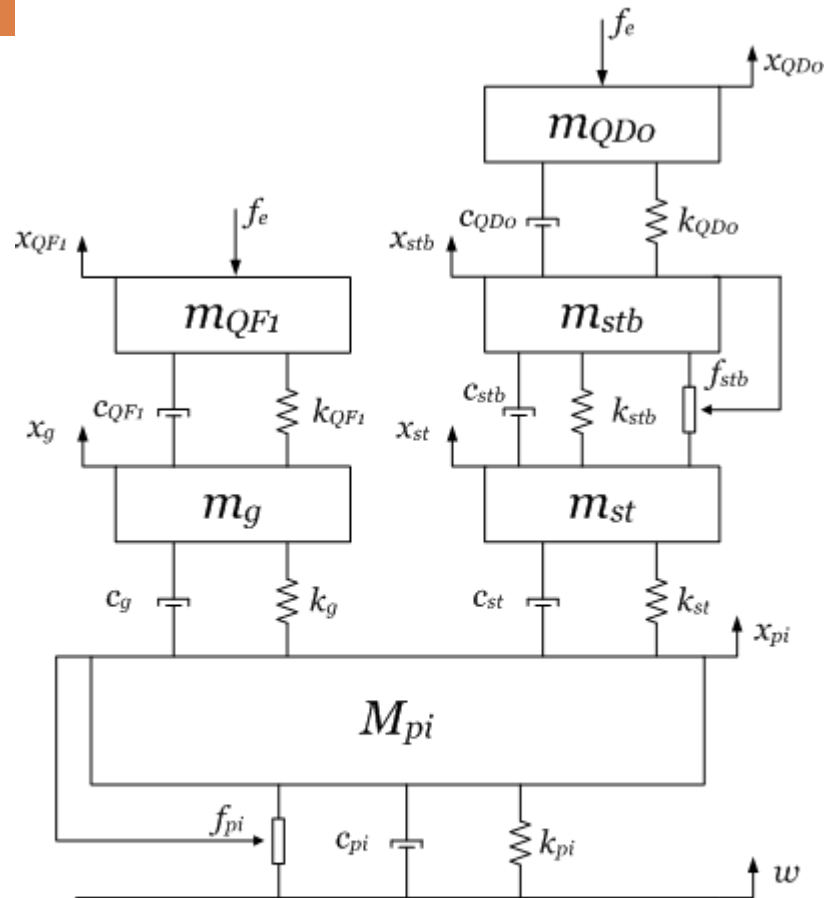
$$w_{stb} = 2\pi 100$$

$$\xi_i = 1\%$$

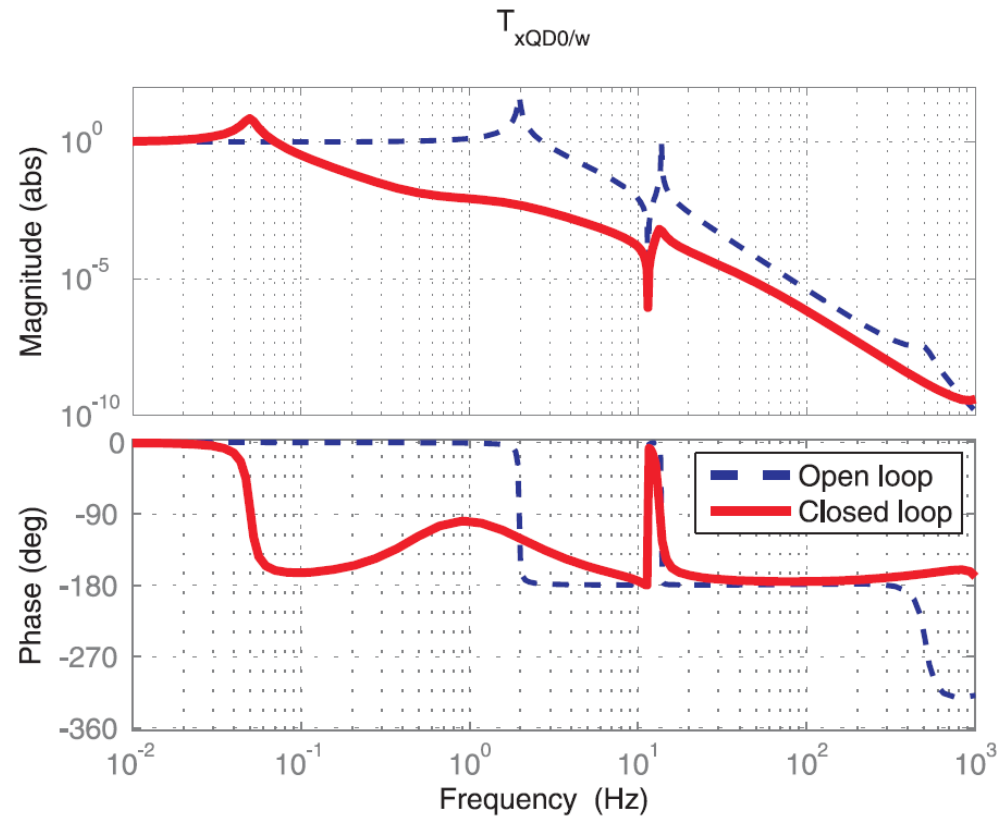
pi – pre-isolator
g – pre-alignment mechanics & support girder
st – pre-alignment mechanics & support tube
stb – LAPP stabilization system



Further simulations with PID



pi – pre-isolator
g – pre-alignment mechanics & support girder
st – pre-alignment mechanics & support tube
stb – LAPP stabilization system

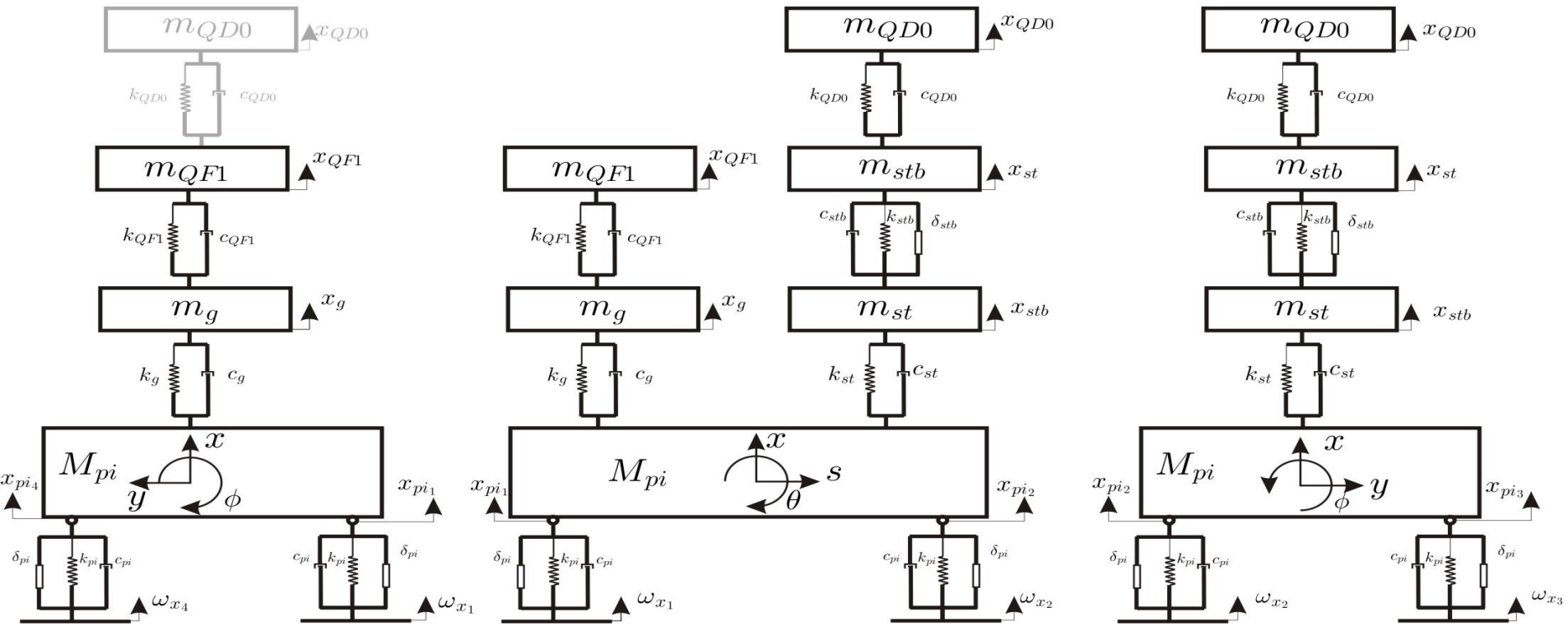
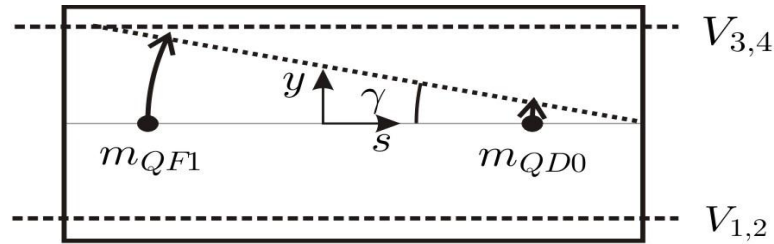


Possible transfer function between ground and QD0 + no drift at low freq
Possibility to change Preisolator position
!!!No noises or filters in it!!!!



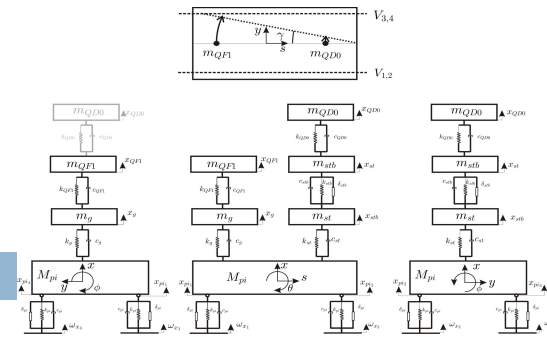
Further Research proposal (tender/inhouse)

More complex model
=> Investigate modes
with changing magnet
positions





Further Research proposal (tender/inhouse)



Research goals

1. Make a dynamic model of the system presented. For this model:

- give the transfer functions between all degrees of freedom and the ground
- make a table of all the modal frequencies and their decomposition in eigenvectors
- graphical representation of the evolution of the modal frequencies and their decomposition for a changing γ from 0 up to 15 degrees which changes the position of $m_g, m_{QF1}, m_{st}, m_{stb}, m_{QD0}$.
- What is the effect of increasing the 1st mode to 20 Hz?

~2 months

2. Propose the best active damping (velocity feedback, Integrated Force Feedback,...) system which:

- damps the 1st mode of M_{pi} critically
- Reduces the 1st mode from 20 Hz to 1 Hz through active control
- uses existing technologies compliant with the environmental parameters,
- does decrease the drop off above $2\omega_{pi}$ in the transfer function between w_{pi} and x_{pi} , for the ground vibrations specified, due to noise or any other limitations (actuator or sensors).
- Specify the number actuators/sensors (The 4 specified are a suggestion).
- Is it better to use a global controller or have each leg have its own SISO controller and decouple them with joints?
- Simulate the performance of the proposed isolation system in an environment with ground vibrations and applying actual sensor/actuator, sensitivity, noise and resolution.

~5 months



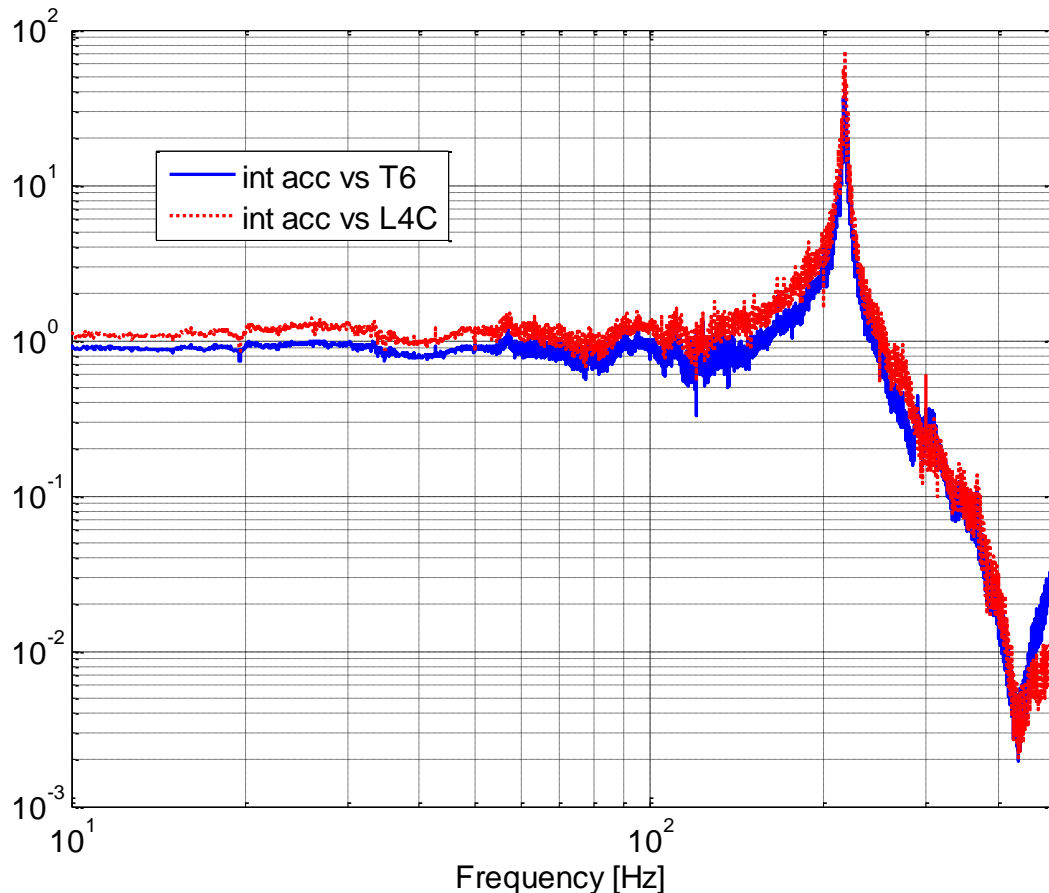
Further issues

- The alignment stage (would fall off now)
- Effect of the 2nd and 3rd mode on luminosity
- Effect of phase difference between two preisolator blocks
- Further issues?



Question accelerometer feedback

Measured sensitivity endeveco M86



Sensitivity curve of accelerometer has resonance at $\sim 200-300$ Hz

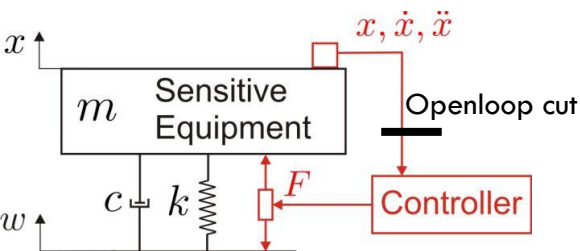
->issue with stability when resonance of system and accelerometer meet



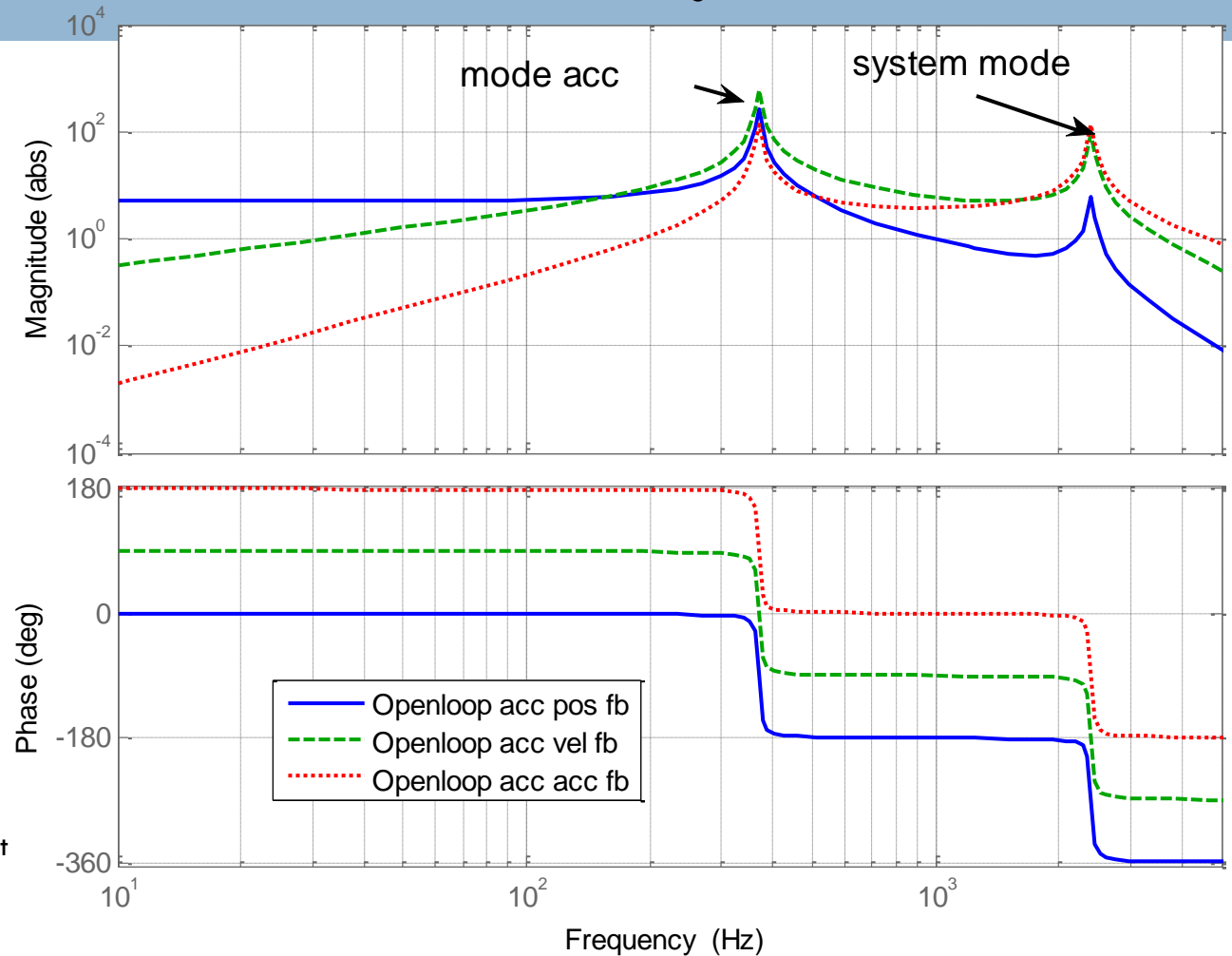
Question accelerometer feedback

Openloop:
Stable if for $\text{Mag} > 1$
Phase $-180 < \phi < 180$

System mode high:
Acc fb ok
Vel fb unstable
Pos fb unstable
Could be made stable
Pos fb unstable
Could be made stable



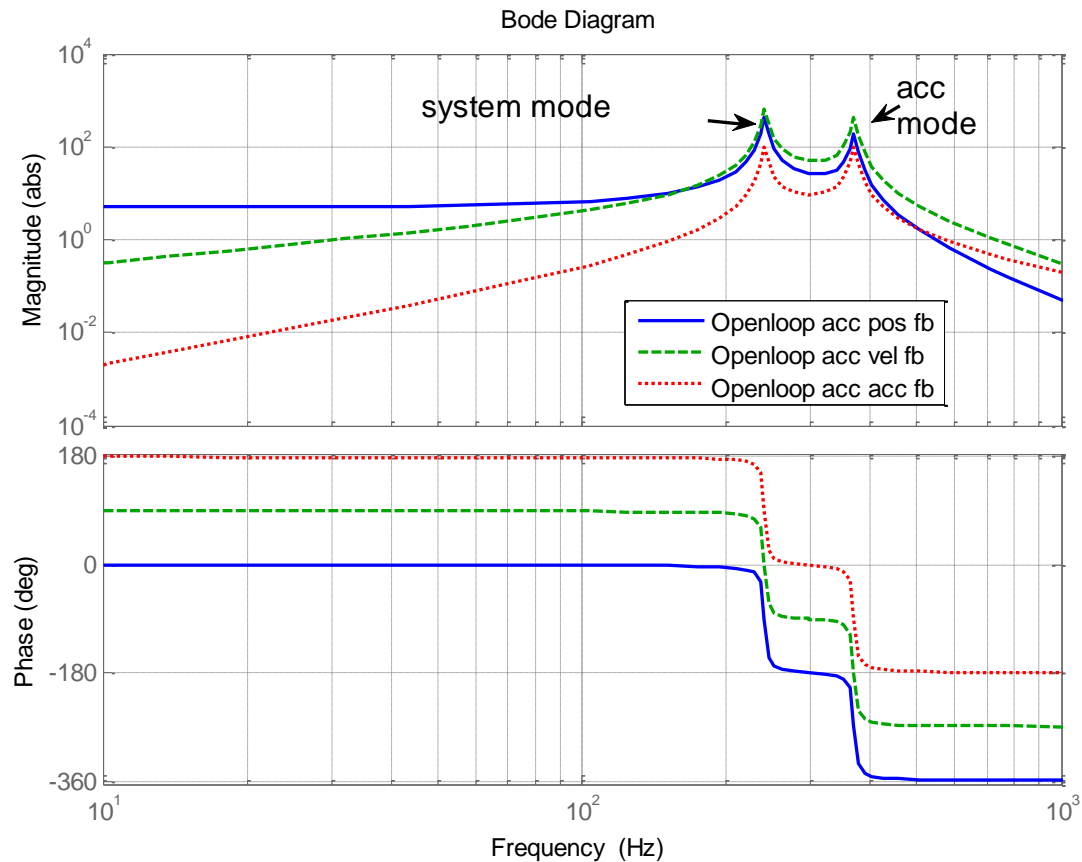
Bode Diagram





Question accelerometer feedback

System mode low:
Acc fb ok
Vel fb unstable
Pos fb unstable
Only pole cancellation



Pole cancellation -> very good knowledge of system necessary, difficult for complex system -> acc mode needs to be far away from system modes



Conclusion

Active control of the preisolator block is needed:

- to perform damping of the resonance
- to provide positioning/alignment capabilities/synchronize with other preisolator
- to improve compliance of the system

Proposed solution for a PID on the preisolator

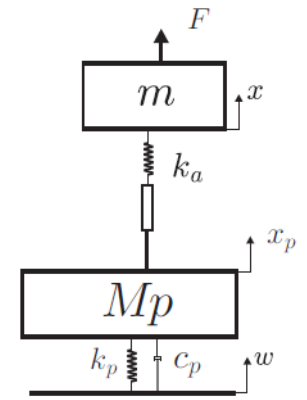
Further study required with full model=> Tender/inhouse study?

Stef's learning moment about accelerometer feedback

Spares

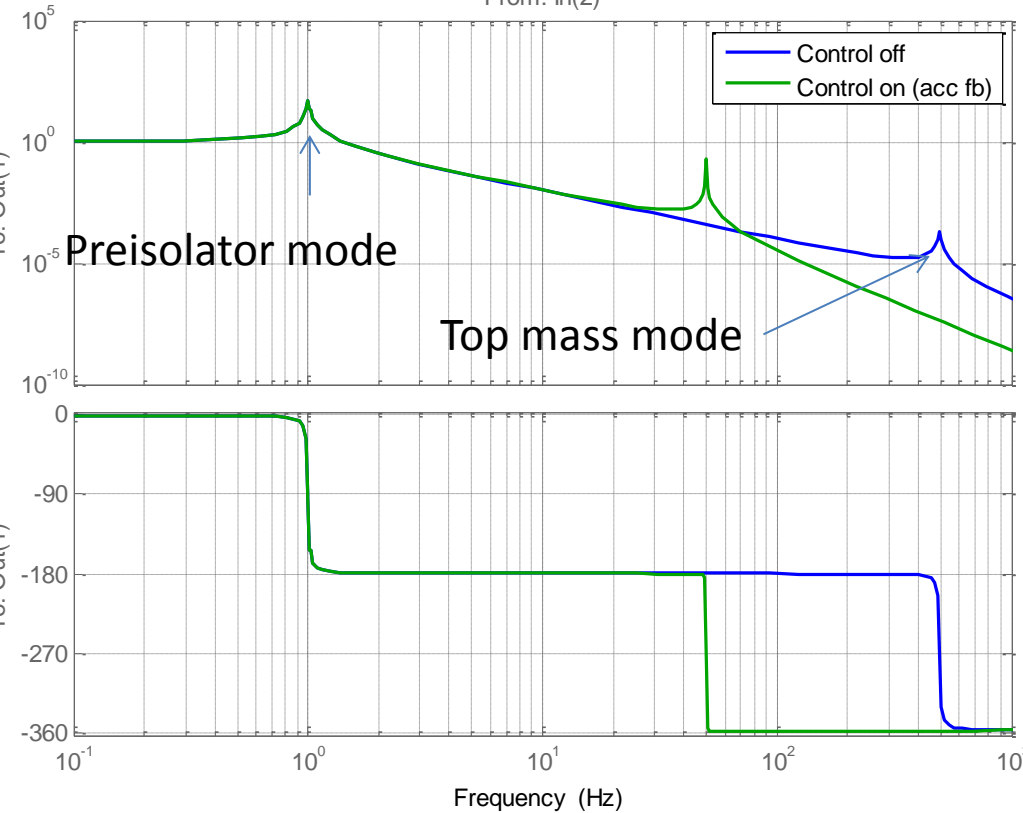
Pre-isolator feedback

\ddot{x} acc fb

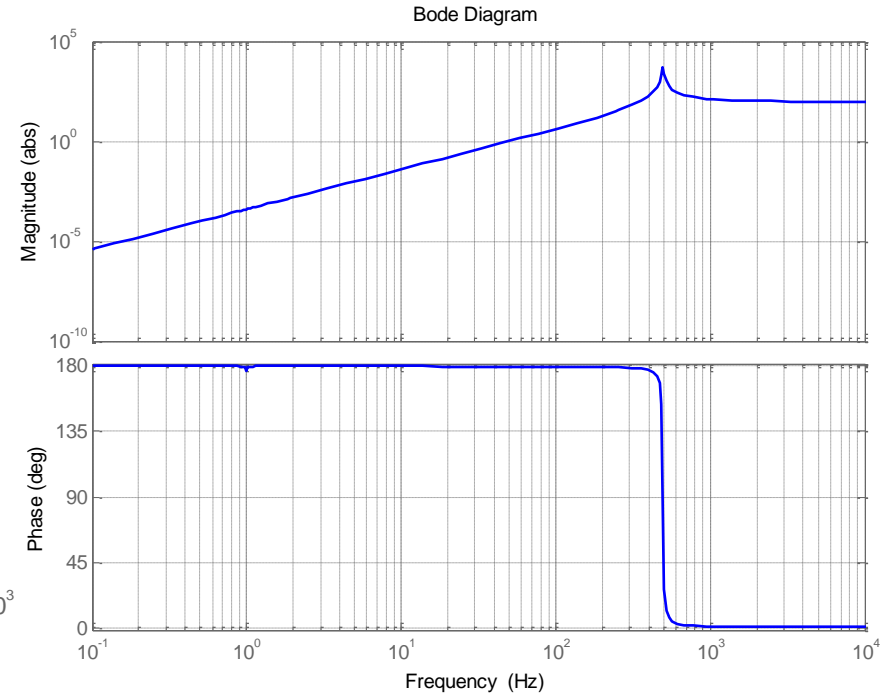


Closed Twx

Bode Diagram
From: ln(2)

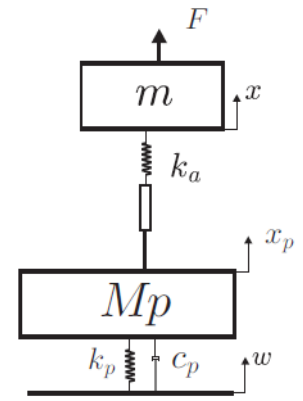


Only effect on second mode
Always stable



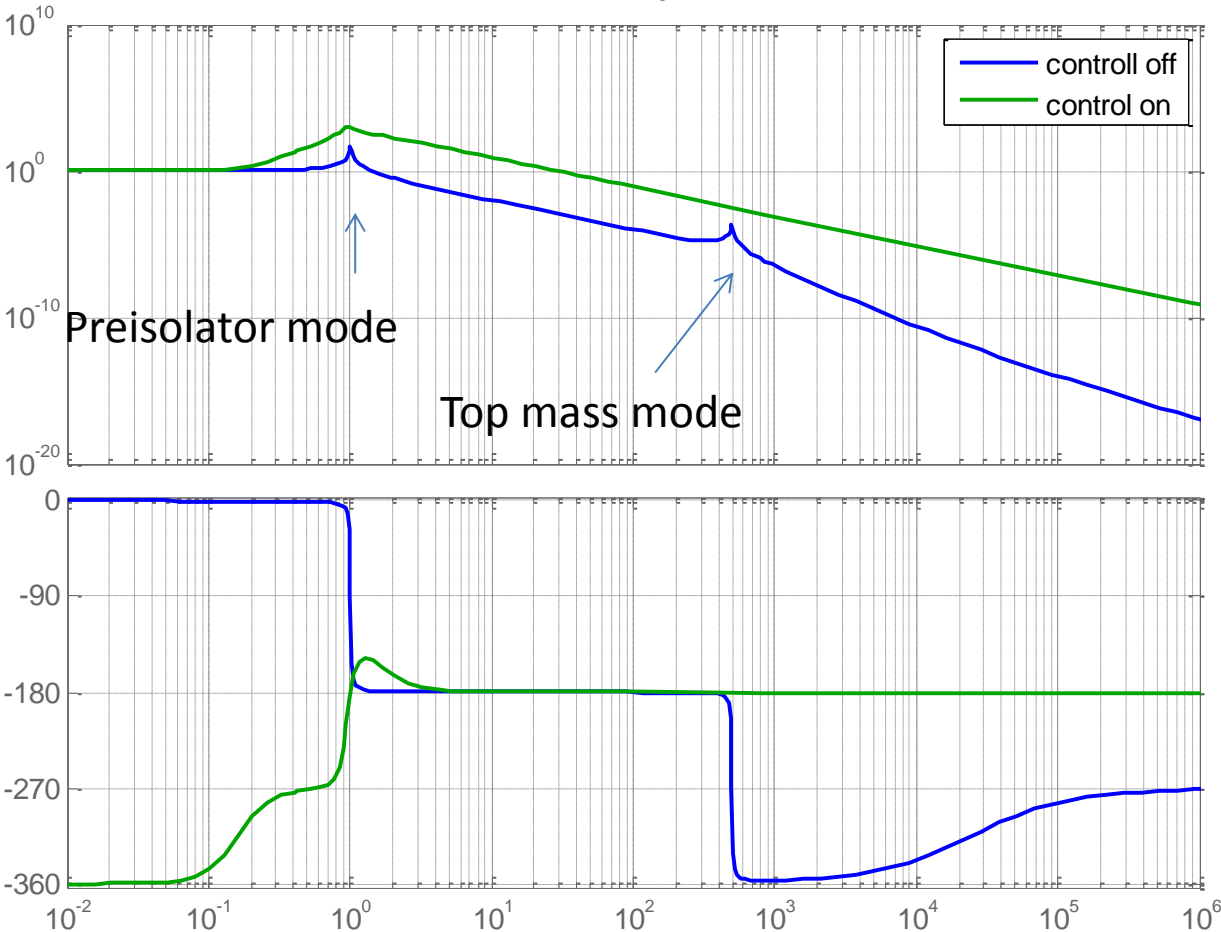
Pre-isolator feedback

\ddot{x}_p acc ff



Closed Twx

Bode Diagram



Doesn't work!