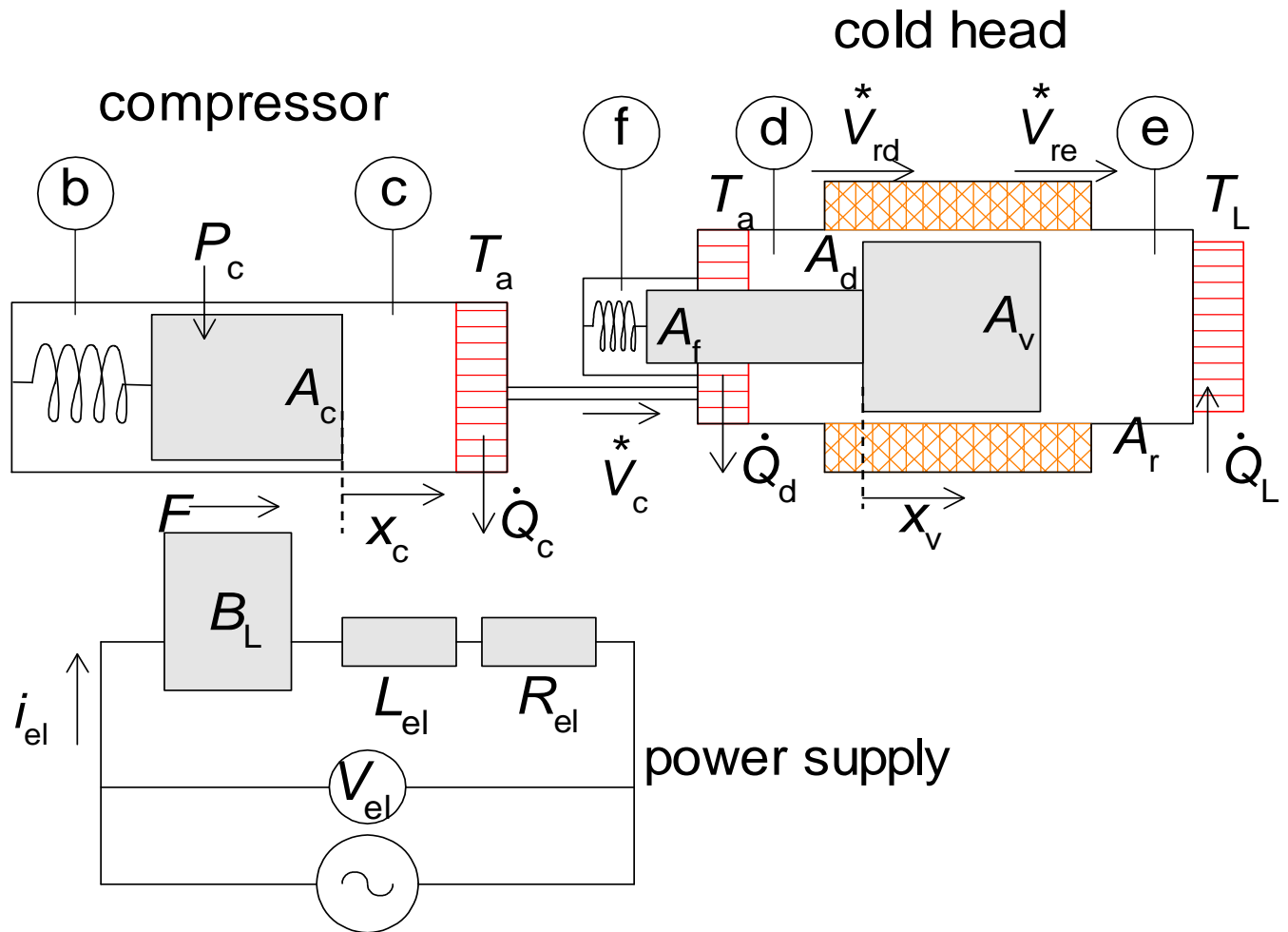


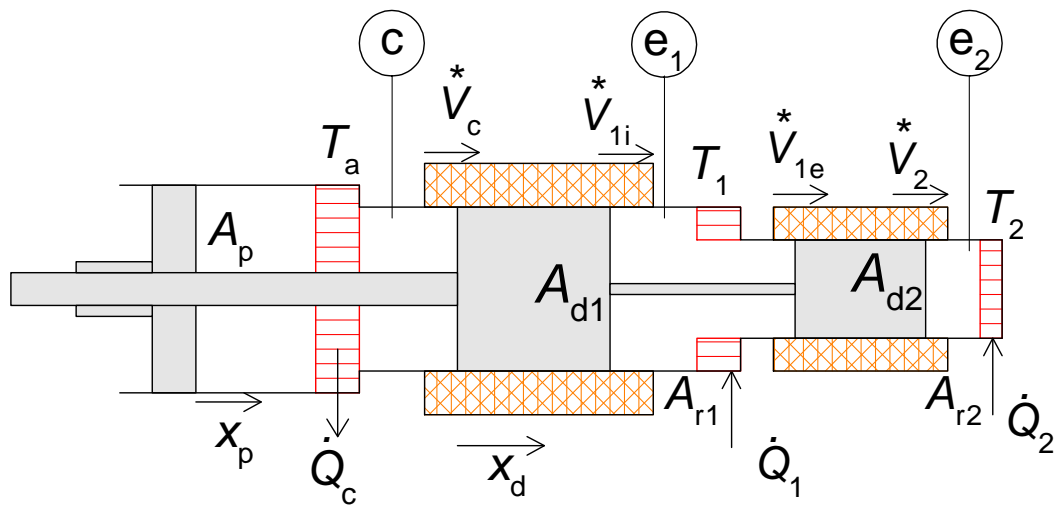
modeling/thermoacoustics

in the field of cryocoolers there are many cases of small periodic variations such as

# free-piston Stirling refrigerator

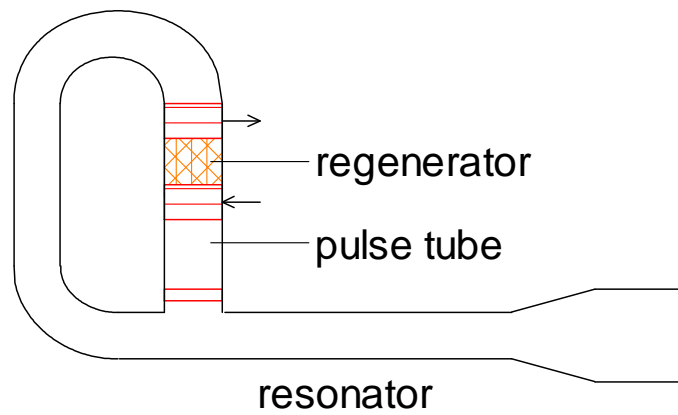
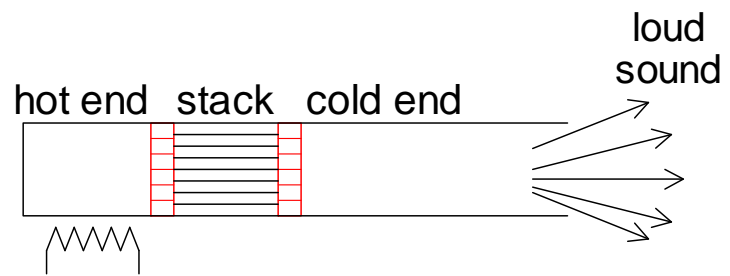
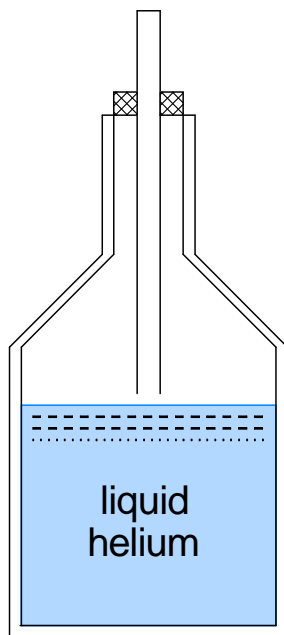


## two-stage crank-driven Stirling refrigerator



# thermoacoustic systems

Taconis oscillations



## harmonic approximation

$$p = p_0 + \delta p$$

small pressure variations ( $p_1 \ll p_0$ ) and steady state

$$\delta p = p_1 \cos(\omega t + \varphi) = \operatorname{Re}(\hat{p}e^{i\omega t})$$

with the complex amplitude

$$\hat{p} = p_r + ip_i$$

and

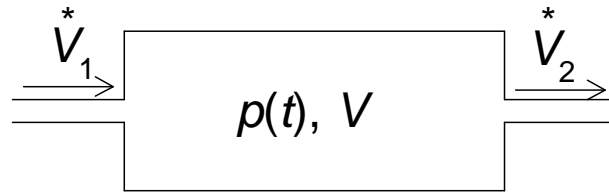
$$e^{i\omega t} = \cos \omega t + i \sin \omega t$$

replaces e.g.  $dp/dt$  by  $i\omega\hat{p}$ : a set of differential equations becomes a set of linear equations

a complicated system can be analyzed in a few microseconds!

for higher accuracy but (much) longer calculation times:  
Sage/Regen/DeltaE/CFD

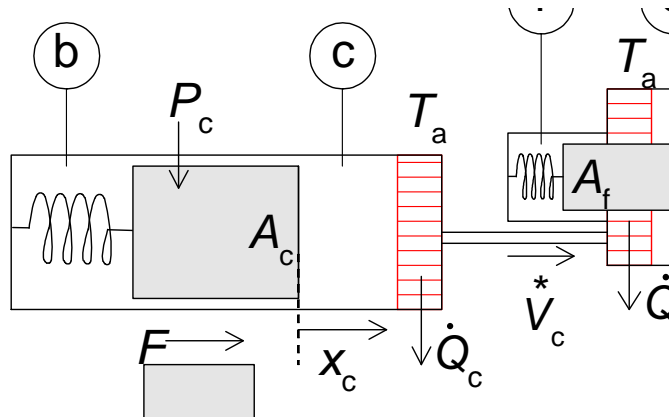
volume-flow equation



$$\dot{V}_1^* = \dot{V}_2^* + \frac{V}{\gamma p} \frac{dp}{dt}$$

with  $\hat{U}$  the complex amplitude of  $\dot{V}^*$  becomes

$$\hat{U}_1 = \hat{U}_2 + \frac{V}{\gamma p_0} i\omega \hat{p}$$



equation of motion of the piston

$$m_p \frac{d^2 x_c}{dt^2} = B_L i_{el} - A_c (p_b - p_c) - f_r \frac{dx_c}{dt} - k_c x_c$$

with  $d/dt \rightarrow i\omega$  this becomes

$$-\omega^2 m_p \hat{x}_c = B_L \hat{i}_{el} - A_c (\hat{p}_b - \hat{p}_c) - i\omega f_r \hat{x}_c - k_c \hat{x}_c$$

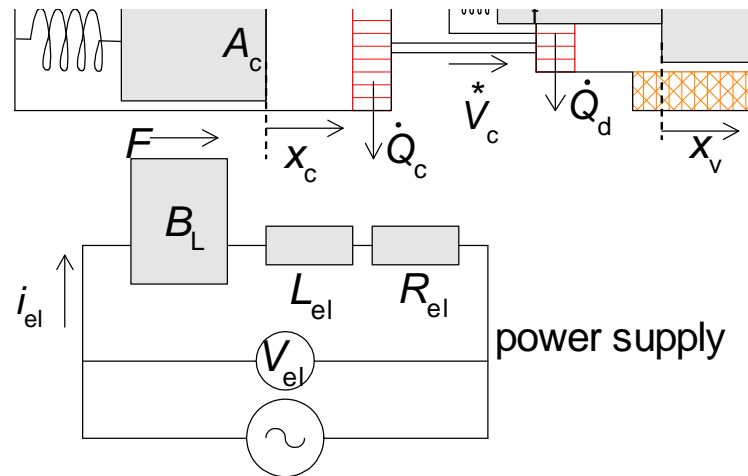
or

$$0 = B_L \hat{i}_{el} - A_c (\hat{p}_b - \hat{p}_c) + (\omega^2 m_p - i\omega f_r - k_c) \hat{x}_c$$

or

$$0 = B_L \hat{i}_{el} - A_c (\hat{p}_b - \hat{p}_c) - k_{ef} \hat{x}_c$$

power supply



$$V_{el} = B_L \frac{dx_c}{dt} + L_{el} \frac{di_{el}}{dt} + R_{el} i_{el}$$

becomes

$$\hat{V}_{el} = (R_{el} + i\omega L_{el}) \hat{i}_{el} + i\omega B_L \hat{x}_c$$

or

$$\hat{V}_{el} = Z_{el} \hat{i}_{el} + i\omega B_L \hat{x}_c$$



## average powers

applied power

$$P = \overline{Fv} = A\overline{v\delta p}$$

with

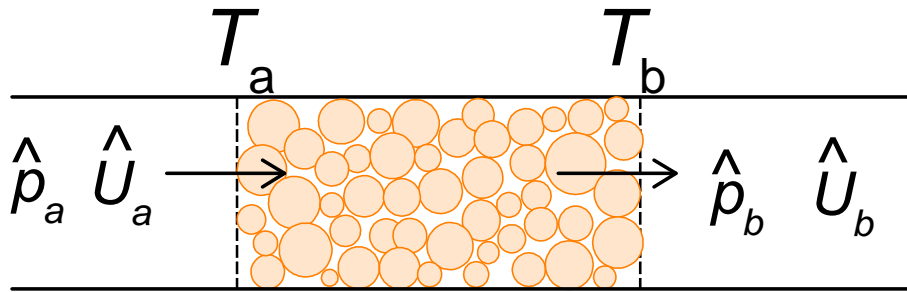
$$\overline{v\delta p} = \overline{\operatorname{Re}(\hat{v}e^{i\omega t}) \operatorname{Re}(\hat{p}e^{i\omega t})}$$

gives

$$\overline{v\delta p} = \frac{1}{2} (v_r p_r + v_i p_i)$$

choose your phase here?

## transfer matrix/functions



connects  $\hat{p}_a$  and  $\hat{U}_a$  to  $\hat{p}_b$  and  $\hat{U}_b$  via

$$\begin{pmatrix} \hat{p} \\ \hat{U} \end{pmatrix}_a = \overrightarrow{T} \begin{pmatrix} \hat{p} \\ \hat{U} \end{pmatrix}_b = \begin{pmatrix} T_{11} & T_{12} \\ T_{21} & T_{22} \end{pmatrix} \begin{pmatrix} \hat{p} \\ \hat{U} \end{pmatrix}_b$$

means

$$\begin{aligned} \hat{p}_a &= T_{11}\hat{p}_b + T_{12}\hat{U}_b \\ \hat{U}_a &= T_{21}\hat{p}_b + T_{22}\hat{U}_b \end{aligned}$$

components in series give products of transfer matrices

orifice with flow resistance  $R_O$

$$p_a - p_b = R_O \dot{V}^*$$

so

$$\begin{aligned}\hat{p}_a &= 1 \times \hat{p}_b + R_O \hat{U}_b \\ \hat{U}_a &= 0 + 1 \times \hat{U}_b\end{aligned}$$

so

$$\vec{T} = \begin{pmatrix} 1 & R_O \\ 0 & 1 \end{pmatrix}$$

volume-flow equation

$$\dot{V}_a^* = \dot{V}_b^* + \frac{V}{\gamma p} \frac{dp}{dt}$$

so

$$\begin{aligned}\hat{p}_a &= 1 \times \hat{p}_b + 0 \\ \hat{U}_a &= i\omega \frac{V}{\gamma p_0} \hat{p}_b + 1 \times \hat{U}_b\end{aligned}$$

so

$$\vec{T} = \begin{pmatrix} 1 & 0 \\ i\omega \frac{V}{\gamma p_0} & 1 \end{pmatrix}$$

ideal regenerator

$$\begin{aligned}\hat{p}_a &= \hat{p}_b + 0 \\ \hat{U}_a &= 0 + \frac{T_a}{T_b} \hat{U}_b\end{aligned}$$

so

$$\vec{T} = \begin{pmatrix} 1 & 0 \\ 0 & \frac{T_a}{T_b} \end{pmatrix}$$

regenerator with flow resistance and a linear temperature  
profile  $\vec{T}$  contains Bessel functions of complex parameters  
makes everybody nervous!

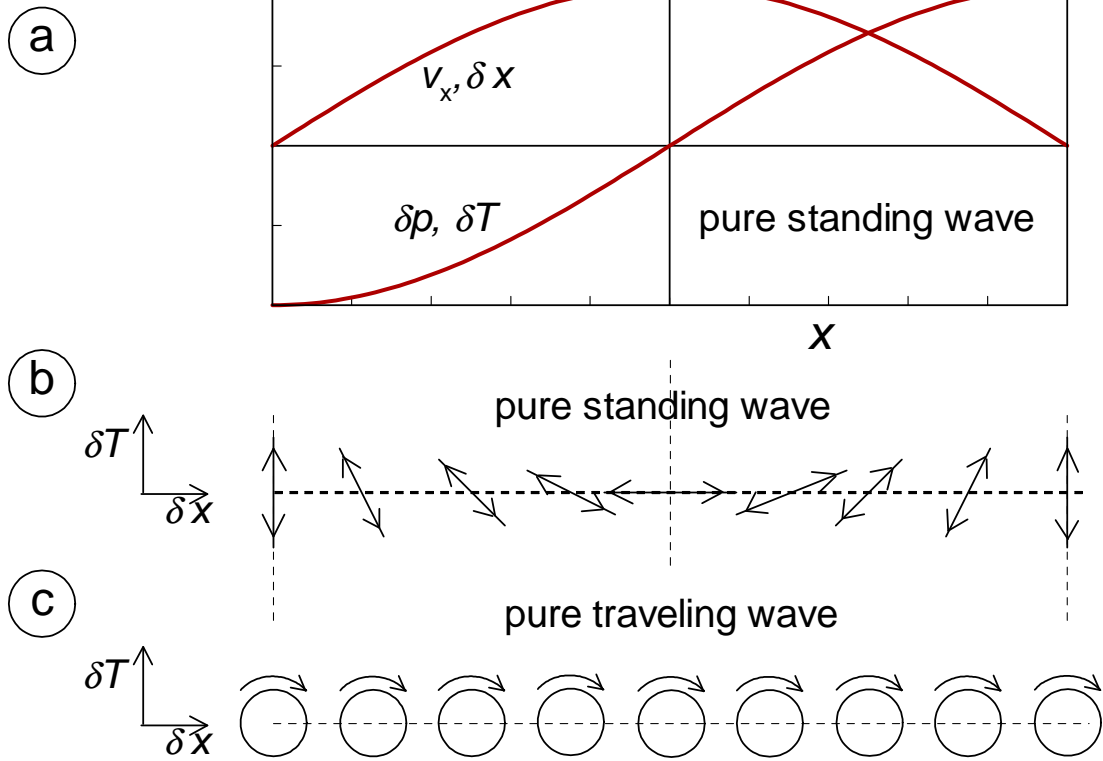
Let commercial software solve your problem!

sound duct of length  $L$  (no proof here!)

$$\begin{pmatrix} T_{11} & T_{12} \\ T_{21} & T_{22} \end{pmatrix} = \begin{pmatrix} \cos \frac{\omega L}{c} & i \frac{c\rho_0}{A} \sin \frac{\omega L}{c} \\ i \frac{A}{c\rho_0} \sin \frac{\omega L}{c} & \cos \frac{\omega L}{c} \end{pmatrix}$$

plate stack: Rott functions

# SOUND



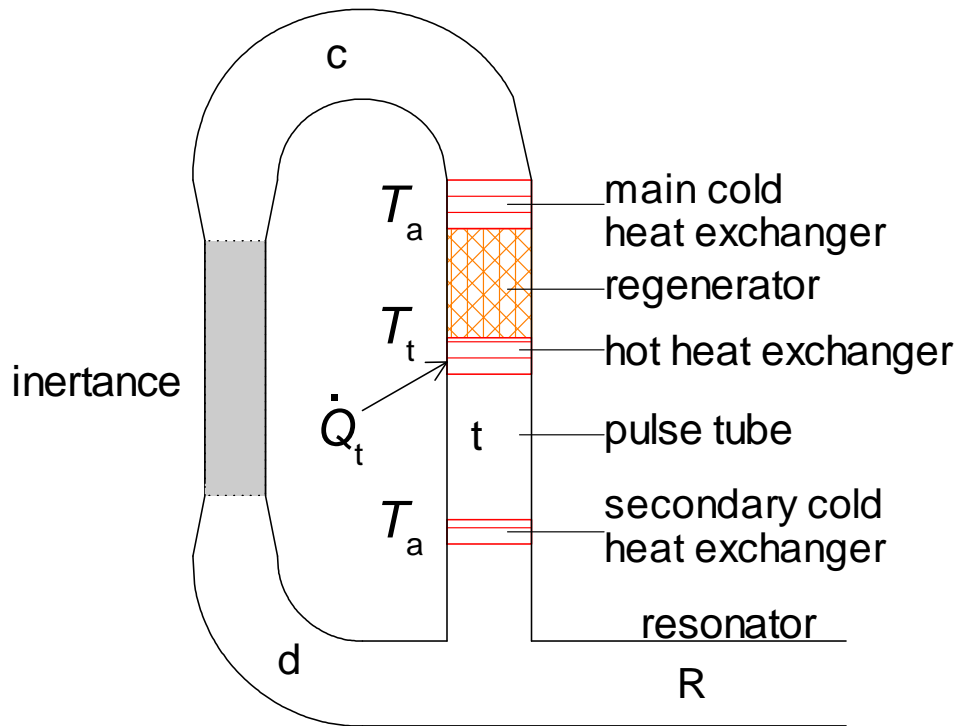
average enthalpy flow (acoustic power) is

$$\overline{H^*} = \overline{n^* H_m} = \frac{p_0 V^*}{RT_0} C_p \delta T = \overline{V^* \delta p}$$

with

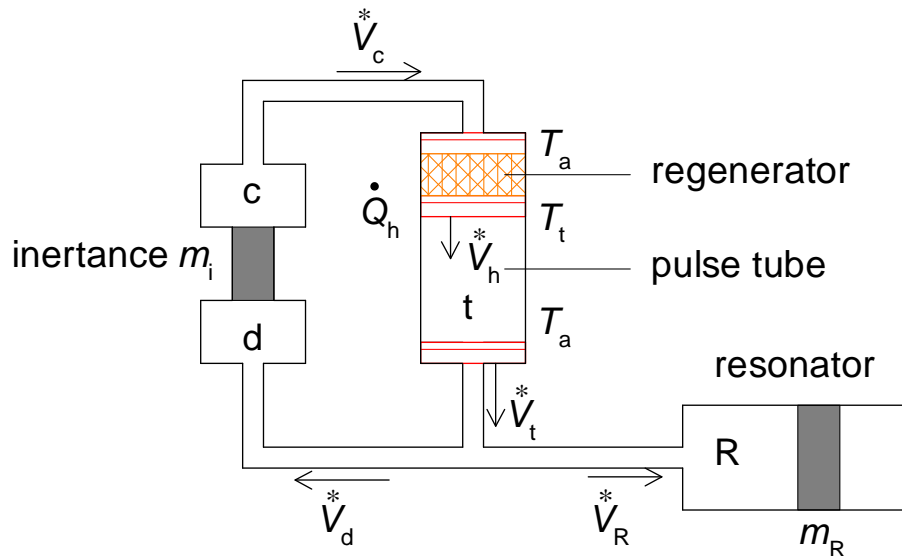
$$\delta p = \frac{C_p p_0}{RT_0} \delta T$$

# calculation of onset of oscillations





## model



$$\frac{d^4 \delta p_t}{dt^4} + a_3 \frac{d^3 \delta p_t}{dt^3} + a_2 \frac{d^2 \delta p_t}{dt^2} + a_1 \frac{d \delta p_t}{dt} + a_0 \delta p_t = 0$$

try

$$\delta p_t = \sum_{n=1}^4 c_n e^{z_n t} = \sum_{n=1}^4 c_n e^{-\alpha_n t} e^{i \omega_n t}$$

$\alpha_n > 0$  dying oscillations

$\alpha_n < 0$  growing amplitudes

$\alpha_n = 0$  steady oscillations at  $\omega_n$