



# How heat pumps work: criteria for heat sources evaluation

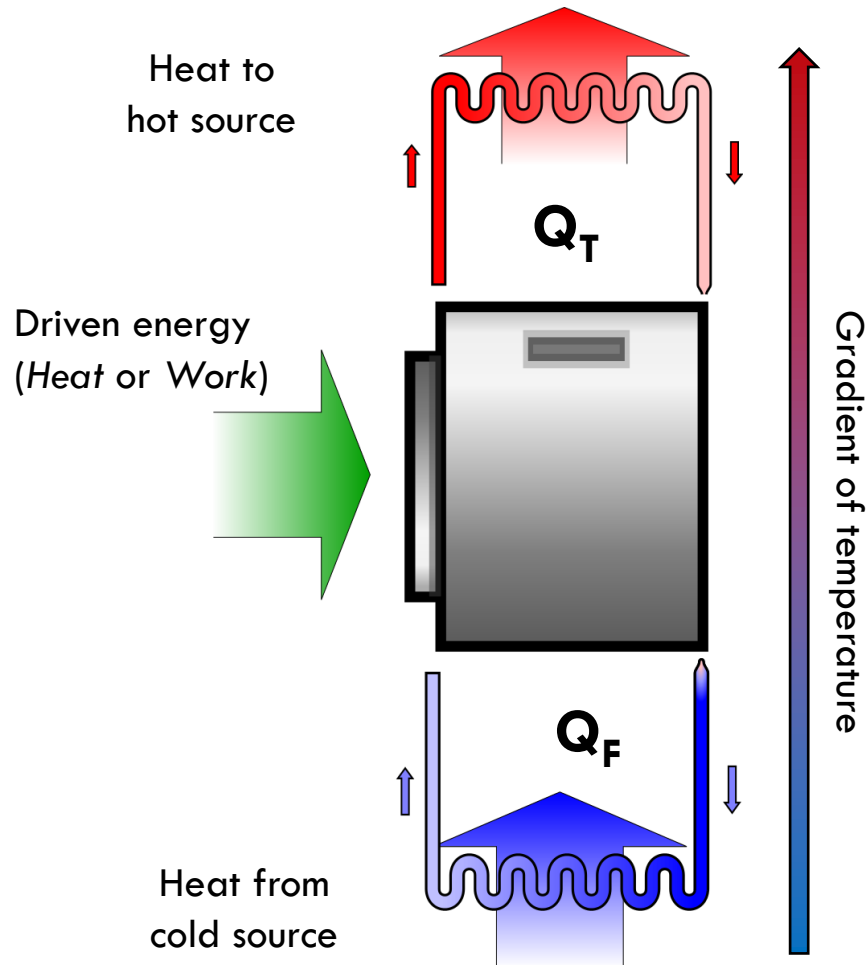
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# Heat pumps: basic theory

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## ✓ What is an heat pump?

Heat pumps is a device able to transfer heat from a cold source to an hot source, against the natural direction of flow. To do that, driven energy is required (heat or work)

## ✓ Coefficient of performance

- Heating mode

$$COP = \frac{Q_T}{Q_T - Q_F}$$

- Cooling mode

$$EER = \frac{Q_F}{Q_T - Q_F}$$

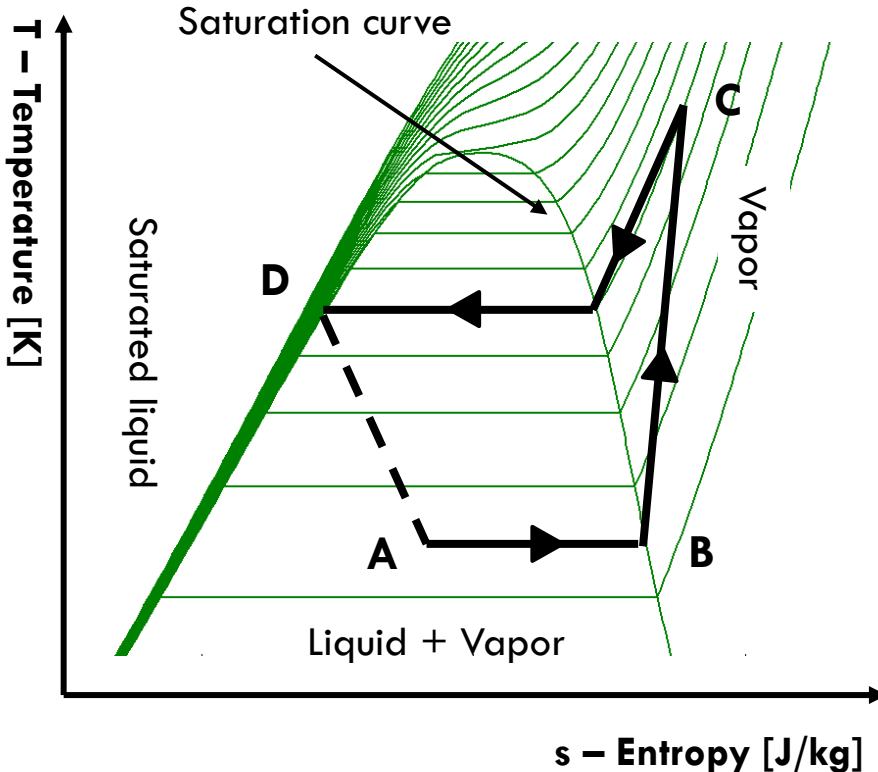
# Heat pumps: basic theory

**Table 1.** Energetic indexes for electrically-driven heat pumps.

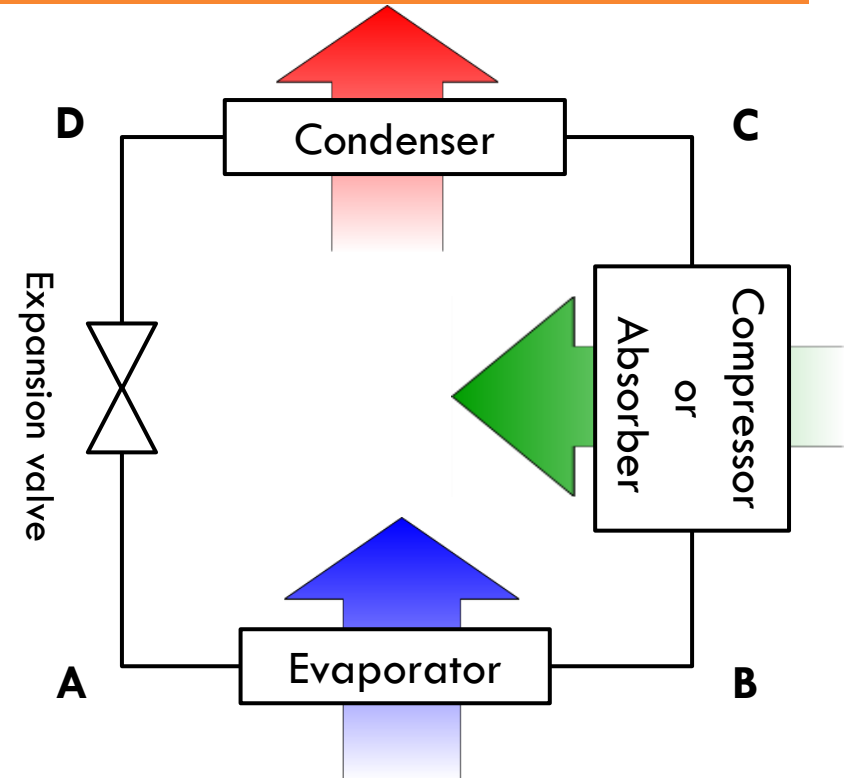
NAME	DEFINITION	TYPICAL VALUES
<b>COP/EER</b>	Useful <b>thermal power</b> divided by <b>power input</b>	$3.5 \div 5 / 2.5 \div 3.5$
<b>&lt;COP&gt; / &lt;EER&gt;</b> Average COP/EER	Useful <b>thermal energy</b> divided by <b>total energy</b> input. The coefficient refers to a <b>specified time interval</b> .	$3 \div 4 / 2.5 \div 3$
<b>SCOP / SEER</b> Seasonal COP/EER	Useful <b>thermal energy</b> divided by <b>energy input</b> . The coefficient refers to the <b>entire heating/cooling season</b> .	$3 \div 4 / 2 \div 3$
<b>PER</b> Primary Energy Ratio	Useful <b>thermal energy</b> during a season divided by <b>primary energy input</b>	$1.2 \div 1.6 / 0.8 \div 1.2$

# Heat pumps: basic theory

## Thermodynamic reference cycle



## Components diagram

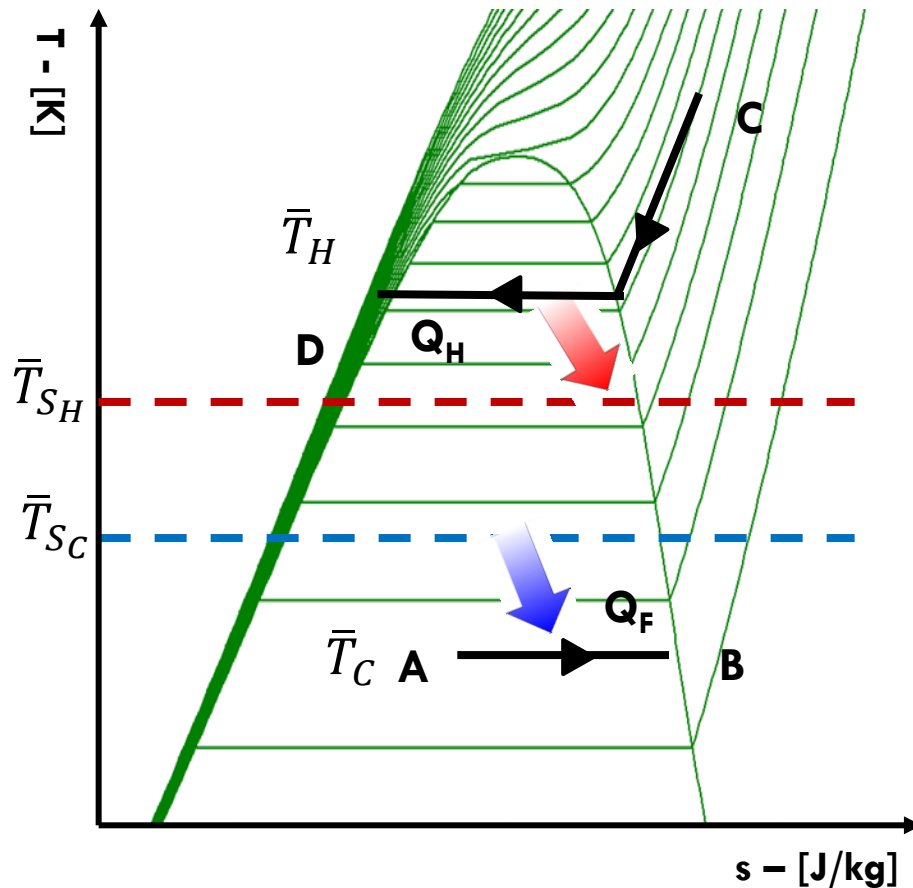


# Reference cycle vs. real systems



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## Thermal sources VS. operating fluid



- ✓ HP efficiency depends on condensing/evaporation temperatures (not sources)

$$\bar{T}_{SH} < \bar{T}_{fH}$$

$$\bar{T}_{Sc} < \bar{T}_{fC}$$

$\bar{T}_S$  -> thermal source

$\bar{T}_f$  -> operating fluid

$$COP(T_S) > COP(T_f)$$

- ✓ HP efficiency is strongly affected by components performance

# Reference cycle vs. real systems



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## Effects of the variation of evaporation/condensing temperatures

Nominal data refer to standard rating condition of thermal sources (e.g. UNI EN 14511-2:2013)

### Nominal performances

Heating capacity – kW	15.1
Total power input – kW	3.6
COP (only compressor)	4.2

Secondary fluid (Evaporator): Inlet 10°C / Outlet 7°C

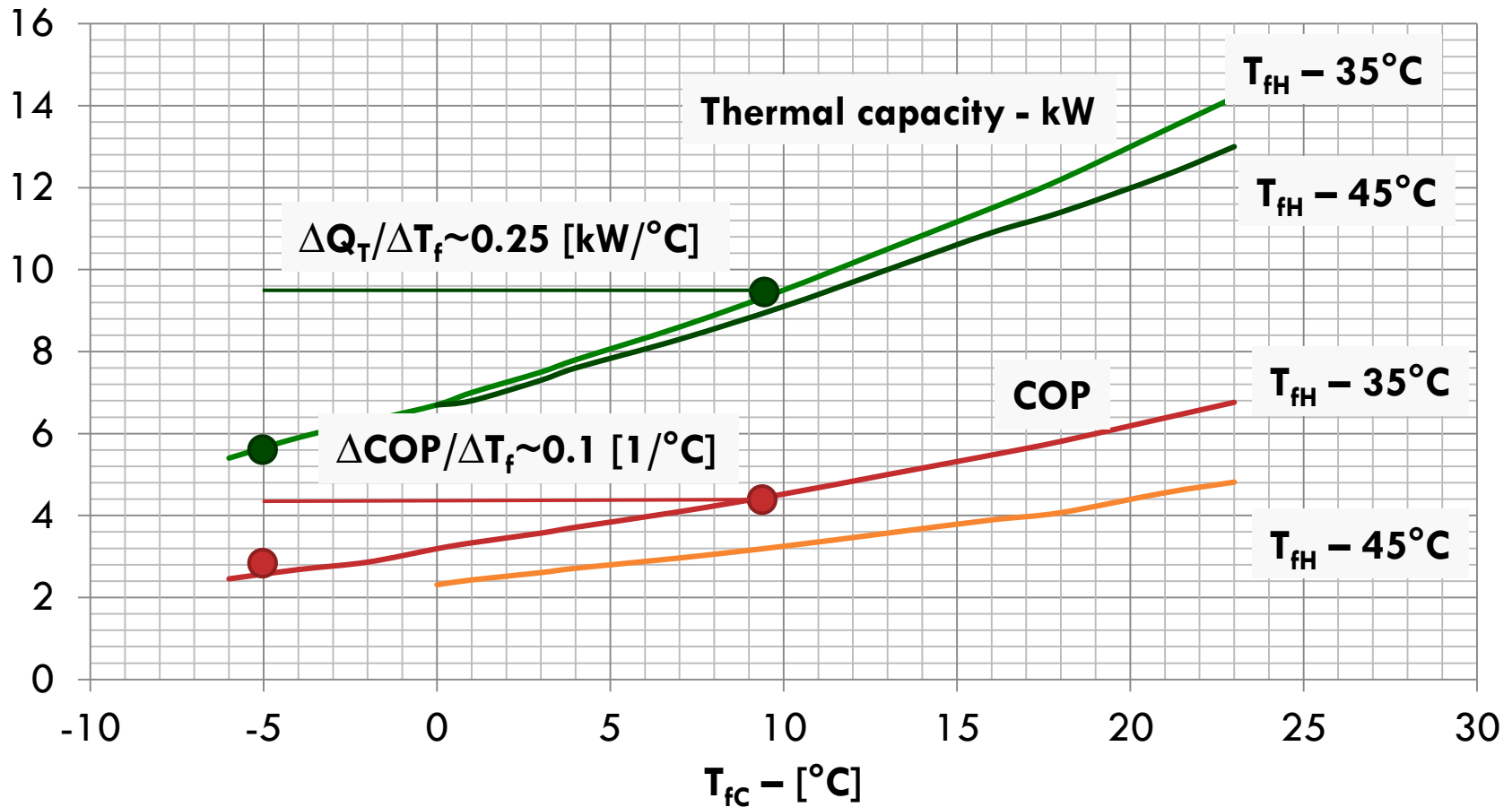
Secondary fluid (Condenser): Inlet 30°C / Outlet 35°C



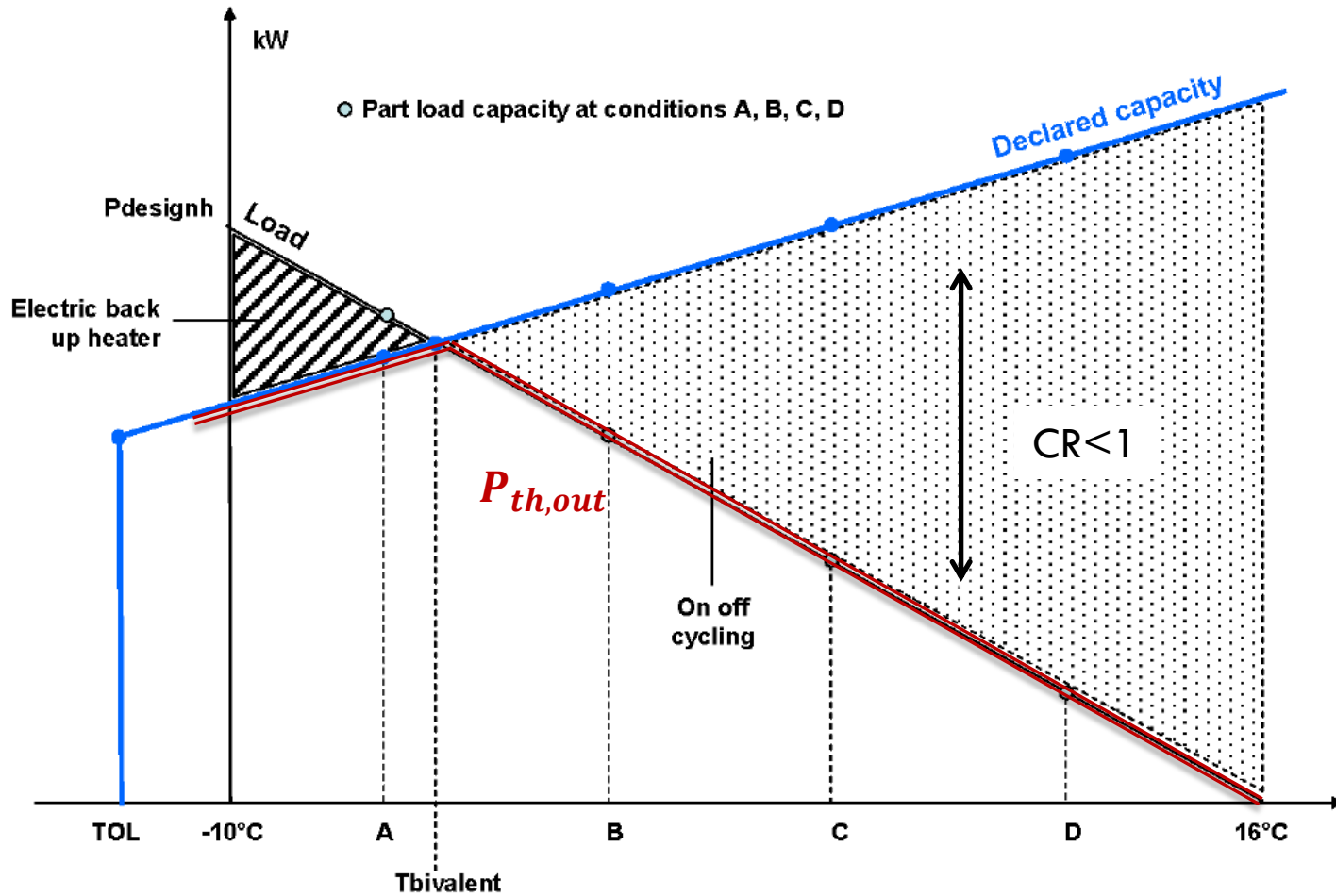
# Reference cycle vs. real systems



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# Reference cycle vs. real systems





# Reference cycle vs. real systems



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## Effects of the capacity ratio

$$CR = \frac{\dot{Q}_u}{\dot{Q}_{DC}}$$

where:

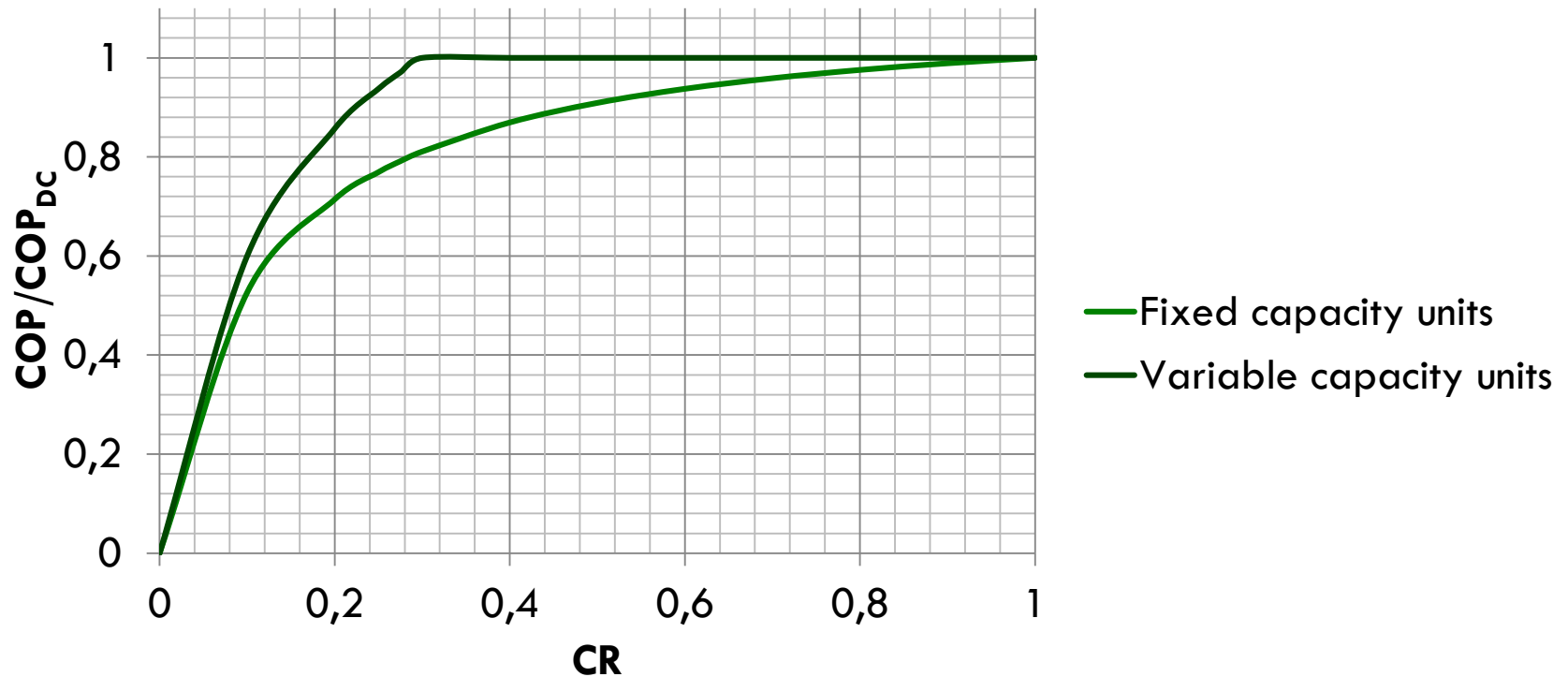
- $\dot{Q}_u$  is the useful thermal power delivered by the HP;
- $\dot{Q}_{DC}$  is the maximum capacity of the HP unit, when operating at the temperatures of the thermal sources.

# Reference cycle vs. real systems



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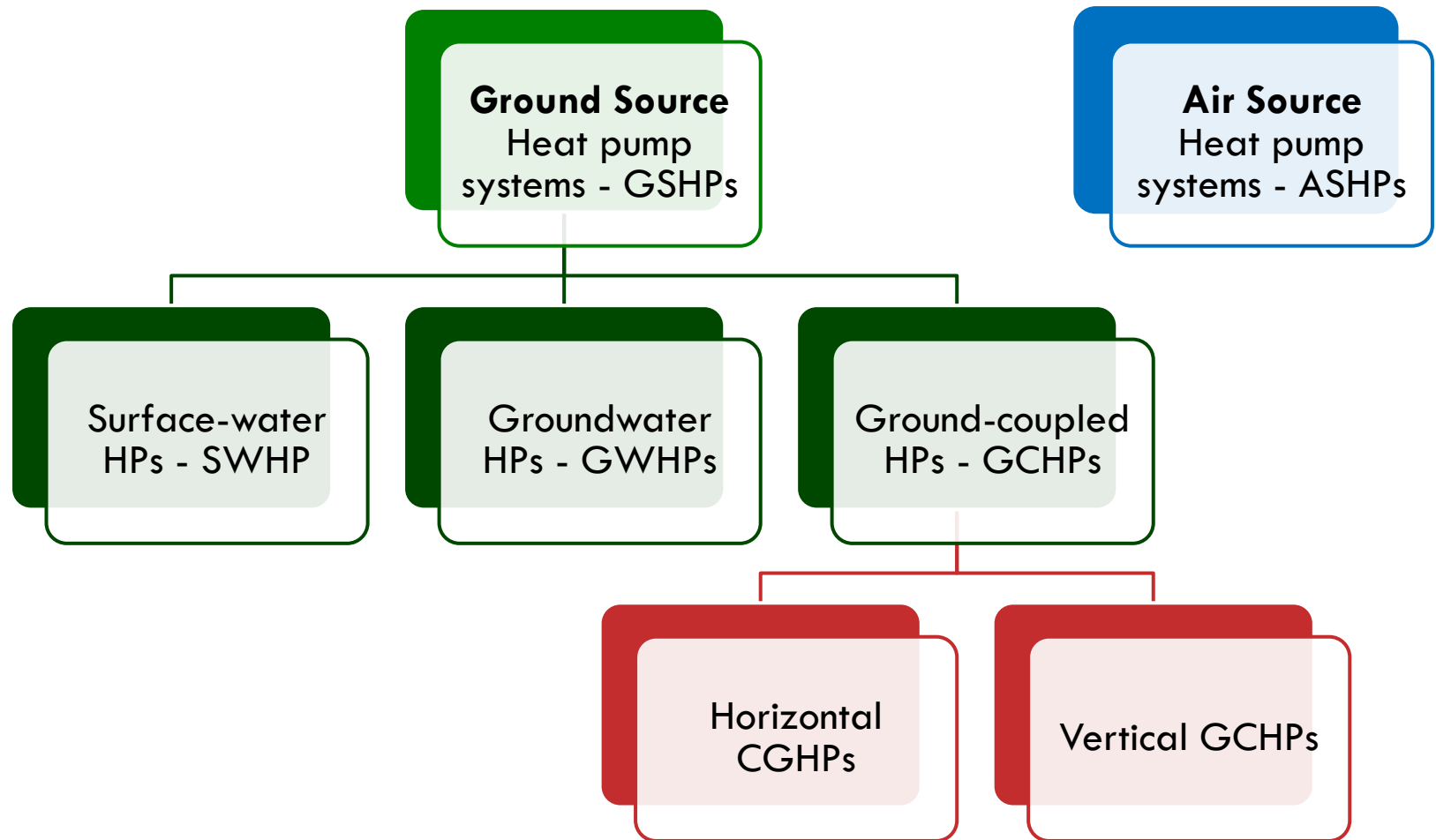
**COP penalization factor depending on control type  
(UNI EN 14825:2012)**



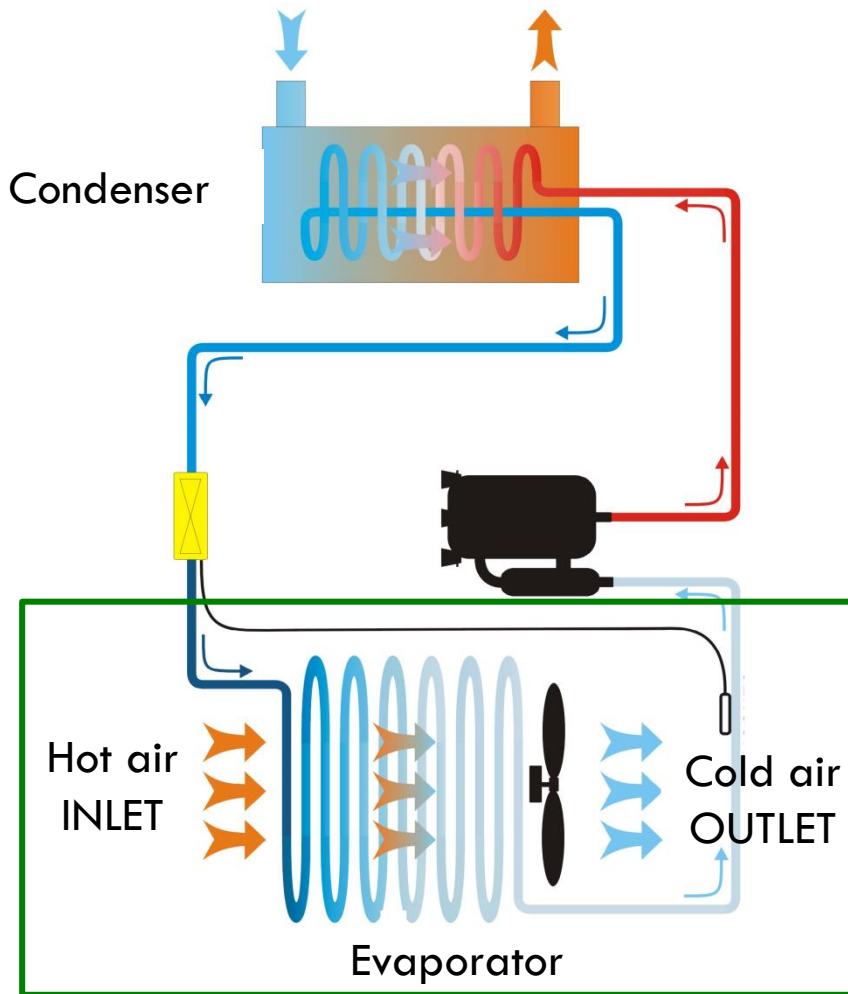
# Classification and Terminology



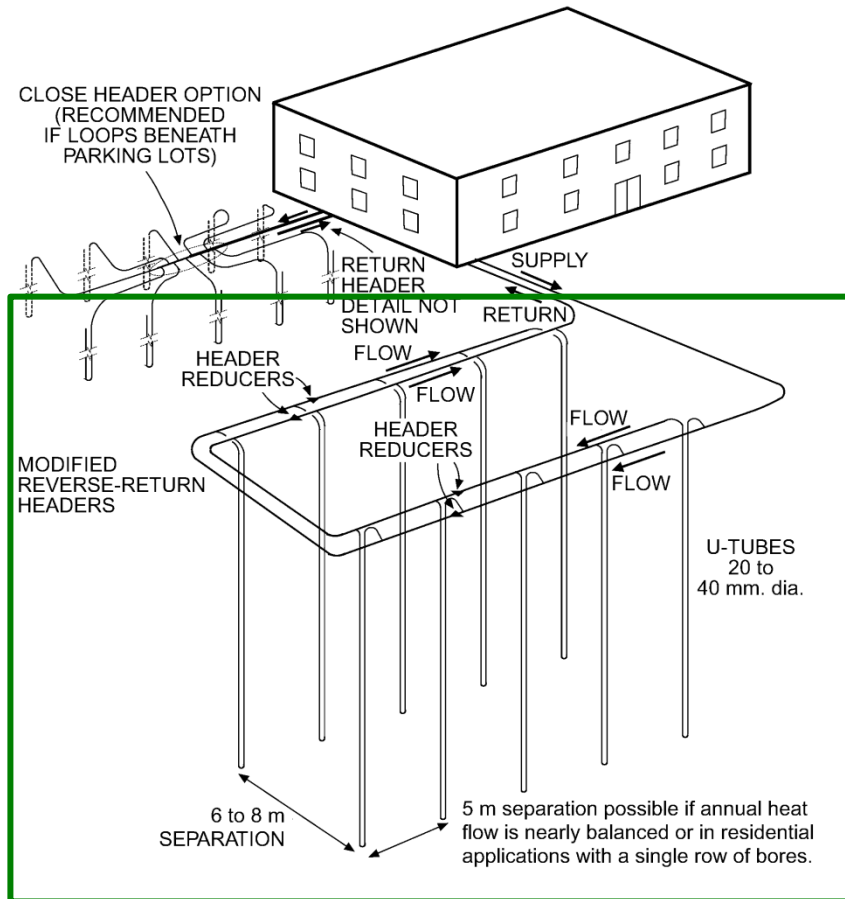
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# Air Source HPs

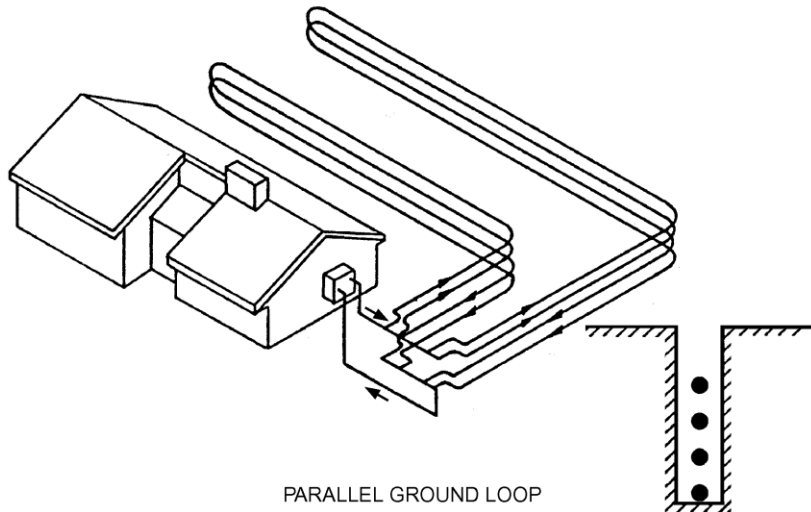
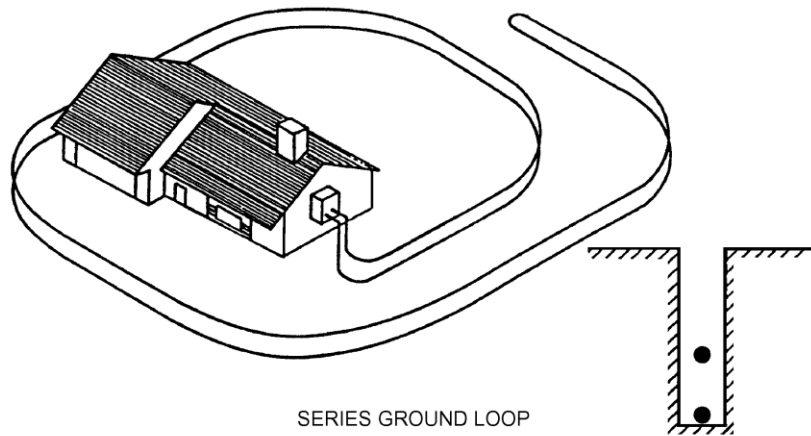


# Vertical GCHPs

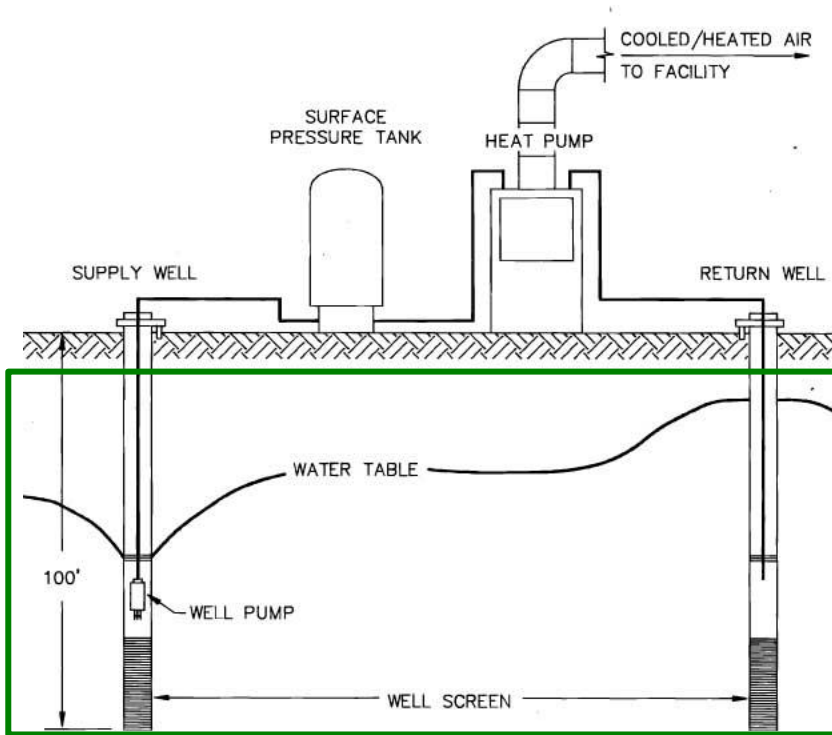


# Horizontal GCHPs

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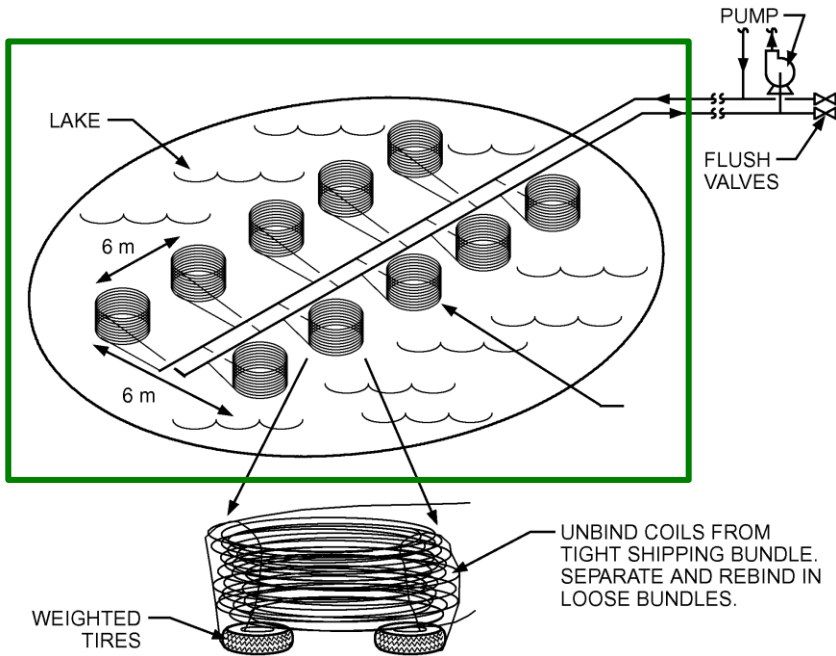
# Groundwater HPs



DUAL WELL GROUND COUPLED OPEN LOOP SYSTEM



# Surface-Water HPs

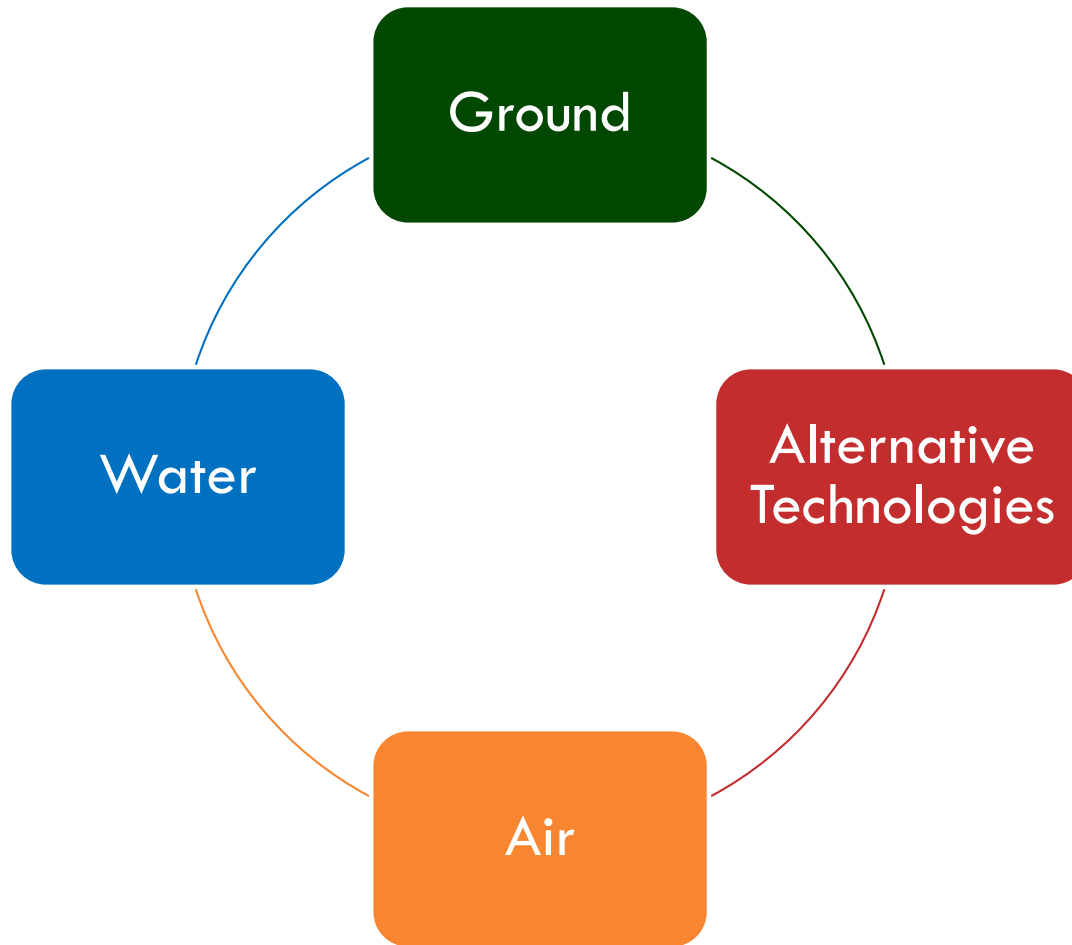




# Which source should I use?



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# Qualitative comparison

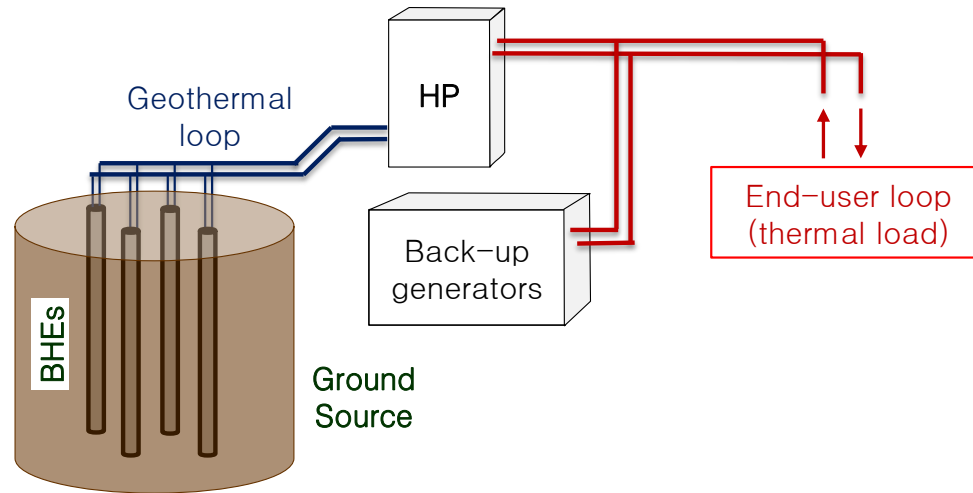


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	<b>Suitability</b>	<b>Availability</b>	<b>Installation Cost</b>	<b>O&amp;M Cost</b>	<b>Temperature</b>
<b>ASHPs</b>	GOOD	EXCELLENT	LOW	MODERATE	VARIABLE
<b>Vertical GCHPs</b>	MODERATE	GOOD / EXCELLENT	HIGH	MODERATE	GOOD
<b>Horizontal GCHPs</b>	MODERATE	MODERATE / GOOD	MODERATE	MODERATE	GOOD / EXCELLENT
<b>GWHPs</b>	GOOD	GOOD	MODERATE	MODERATE / HIGH	GOOD / EXCELLENT
<b>SWHPs</b>	GOOD	MODERATE	MODERATE	MODERATE / HIGH	GOOD

# Real systems layout

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Ground	Ground Heat Exchangers (GHEx)	Geothermal loop	HP equipment	Back-up generators	End user loop
Thermophysical properties	Number, depth, diameter, arrangement	Flow rate and operating temperature	Capacity, Type, Control	Capacity, efficiency	Puffer and thermal loads (power and energy)

**GSHP Systems**

**HVAC Systems**

# What determines a proper work by GSHPs?



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## Primary ENERGY savings

- ✓ Share of loads delivered with a SCOP/SEER higher than alternative technologies
  - **Operating Temperature** at the evaporator/condenser
    - Thermal sources
    - Heat transfer apparatus
  - **Capacity ratio**
    - HP sizing and control
    - Thermal load pattern

## Economy

- ✓ Capacity ratio

$$CR \propto \frac{\text{Energy savings}}{\text{Initial Costs}}$$

- Initial costs
  - HP capacity + GHEXs
- Total energy savings
  - (HP + Back-up)
- Prices/Fares of electricity and natural gas

# What should be the goal?

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- ✓ **What is the goal?**
  - ▣ Maximize **benefit-cost ratio** (energy/economy) of overall system (GSHP & Back-up generators)
  - ▣ Evaluate the **optimal fraction of building load** (*energy!*) delivered by GSHPs.
  - ▣ Nominal capacity of GSHP system is a variable and not the final result
- ✓ **What are the main design variables?**
  - ▣ **HP capacity & GHEXs** (depth, number, arrangement...)
  - ▣ Management strategy during operation time.
- ✓ **What should be avoided?**
  - ▣ **Rough & hasty approach** to design process
  - ▣ No consideration on the **management of system** during operation time.

# The design process: a cost-benefit optimization



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- ✓ Energetic systems design (especially GSHPs) consists in a comprehensive procedure aimed at evaluating the performance of the system during its operational life for cost-benefit considerations
- ✓ Sizing process, feasibility study, performance analysis should to be considered as the very same activity
- ✓ Further details on this design approach can be found in:
  - “Proposal of a Novel Design Procedure for GSHP Systems”  
W. Grassi, P. Conti, D. Testi – EGC2013 proceedings.
  - “A Design Method for Ground Source Heat Pump Systems Based on Optimal Year-Round Performance – Part 1: Model Definition and Discussion”, *submitted to Applied Energy*.

# Thermal sources assessment

## operative conditions vs. initial conditions



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- ✓ Collect all possible information about undisturbed/initial state of the external thermal sources
  - **Air (ASHPs):** annual climate;
  - **Water (GWHPs and SWHPs):** Aquifer depth, temperature, pumping test...
  - **Ground (GCHPs):** temperature, thermophysical properties, TRT...
  
- ✓ Evaluate thermal load profile (second thermal sources)
  - Heat delivery temperature
  - Power and Energy needs
  
- ✓ **Does GSHP operation alter the thermal sources?**

# Air/water vs. ground

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- ✓ Ground has a **huge thermal capacity**, governing its temperature evolution
- ✓ Current performance of the GSHP depends on the **full history of heat exchanges** (sizing and control strategy)
- ✓ Air and water sources are **weakly affected** by past operation

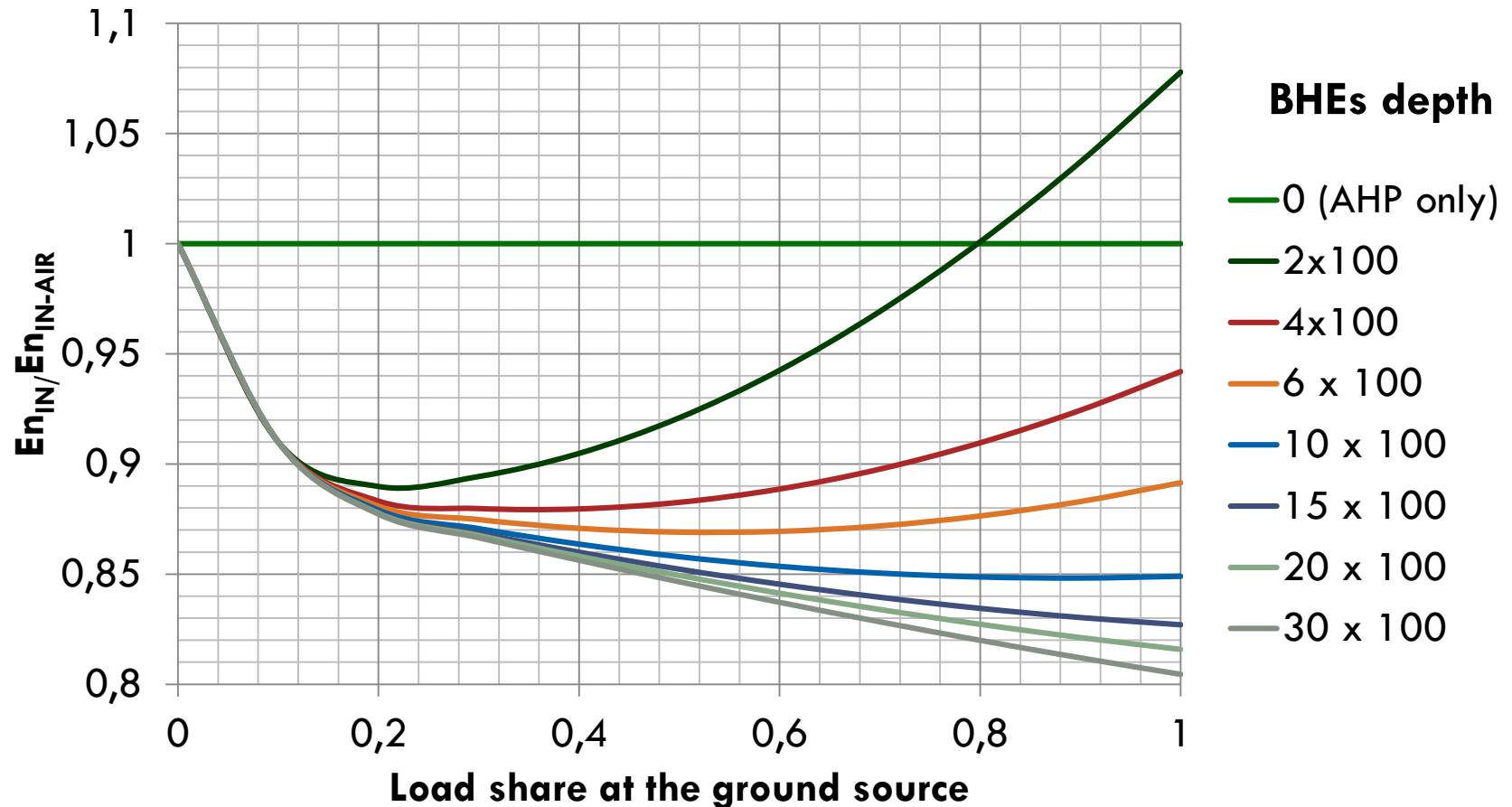


# Feasibility of GSHP projects: a case study



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We consider an heating system made of a **GCHP** and an **ASHP** (as back-up).



# Feasibility of GSHP projects: a case study



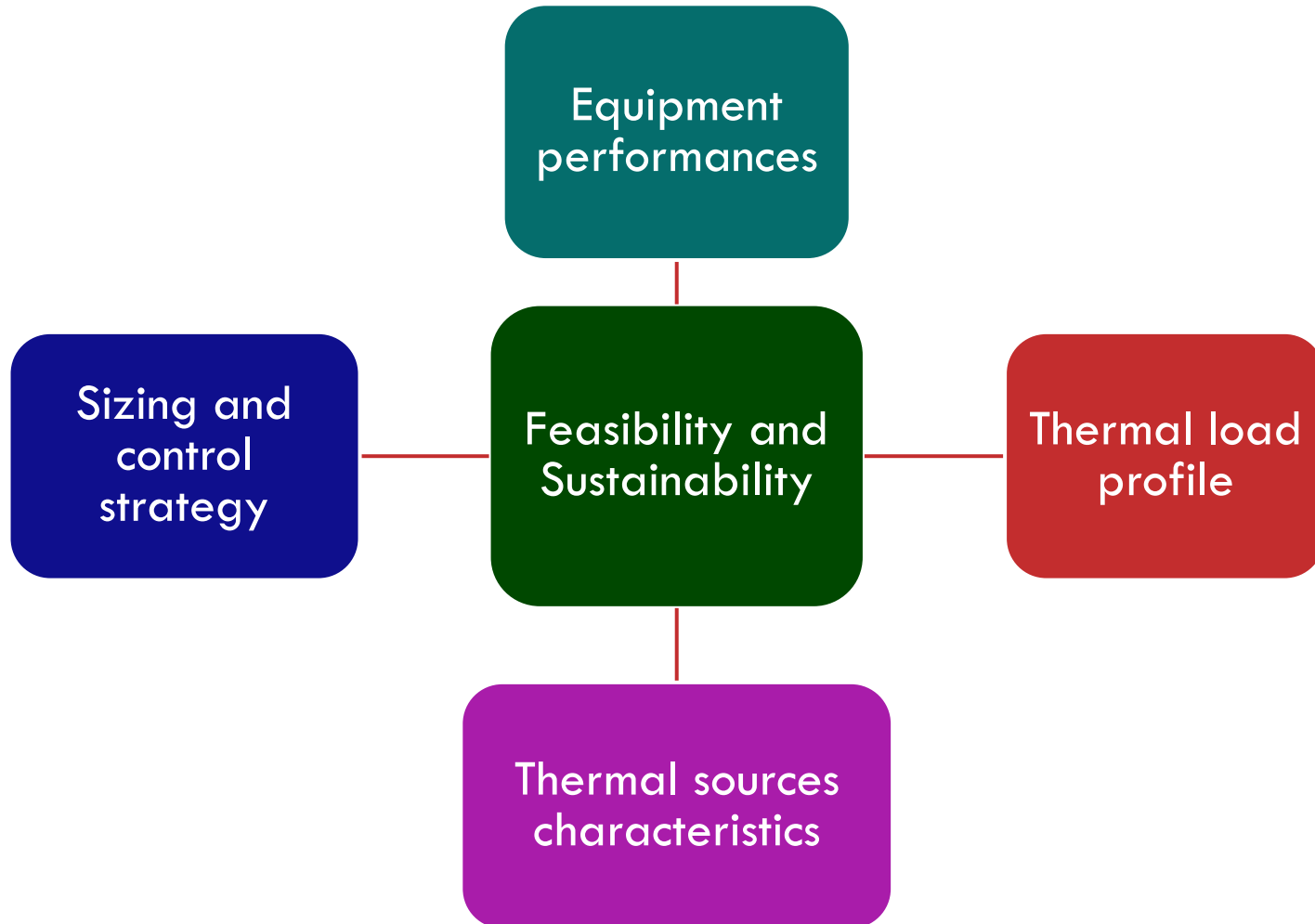
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- ✓ **Minimum energy consumption** corresponds to the **optimal synergy** between air and ground sources.
- ✓ Energy savings show a saturation trend at elevate BHEs number, hinting an **unfavorable benefit-cost ratio for oversized systems**.
- ✓ BHEs number and depth have to be chosen seeking **the optimal tradeoff** among **installation costs** and **corresponding achievable savings**.

# Feasibility of GSHP projects

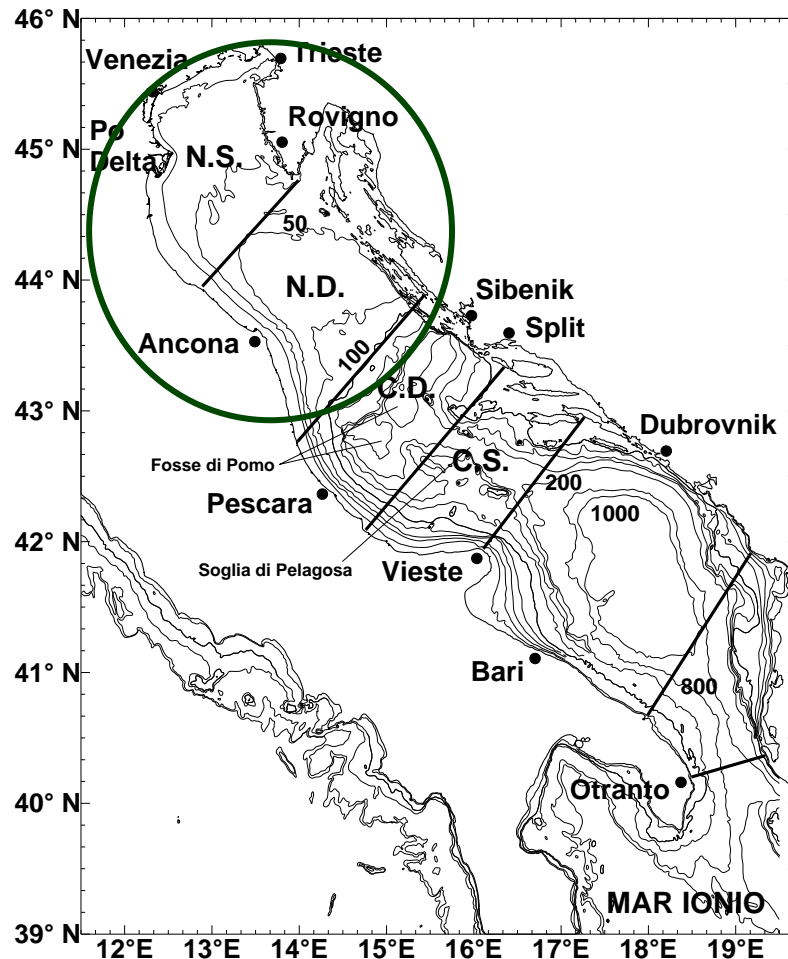


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# Seawater as thermal source

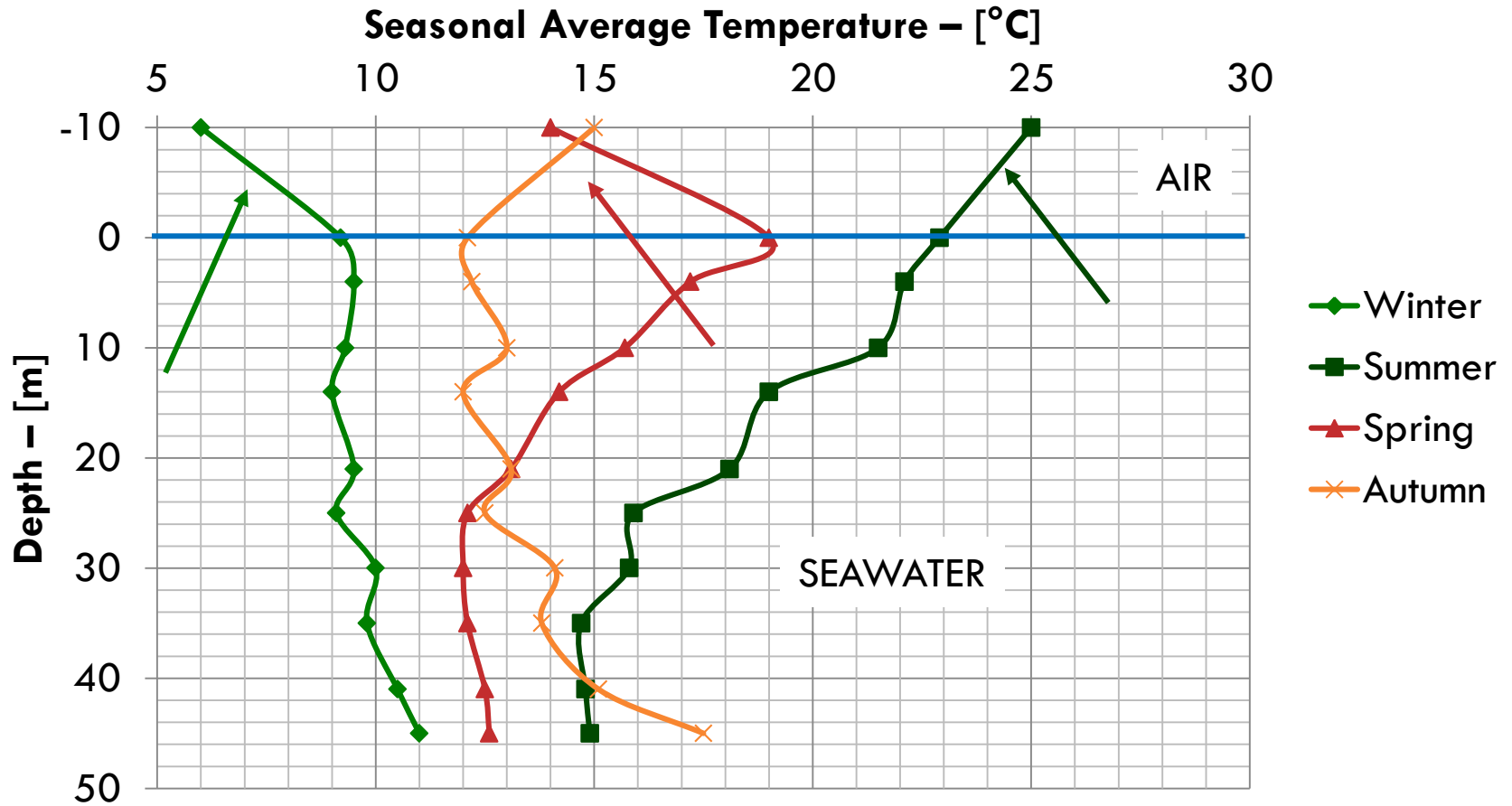
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## Preliminary considerations

- ✓ Seawater temperature is **less variable** than air one
- ✓ Seawater temperature is **weakly affected** by HPs operation
- ✓ **Maintenance** required
- ✓ Feasibility study required (always!)

# Seawater as thermal source



Data from: UNI 10349:1994 and ISMAR (CNR)

# Thanks for your attention!



ECSAC

