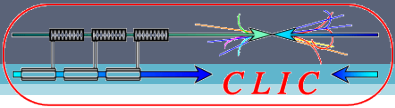


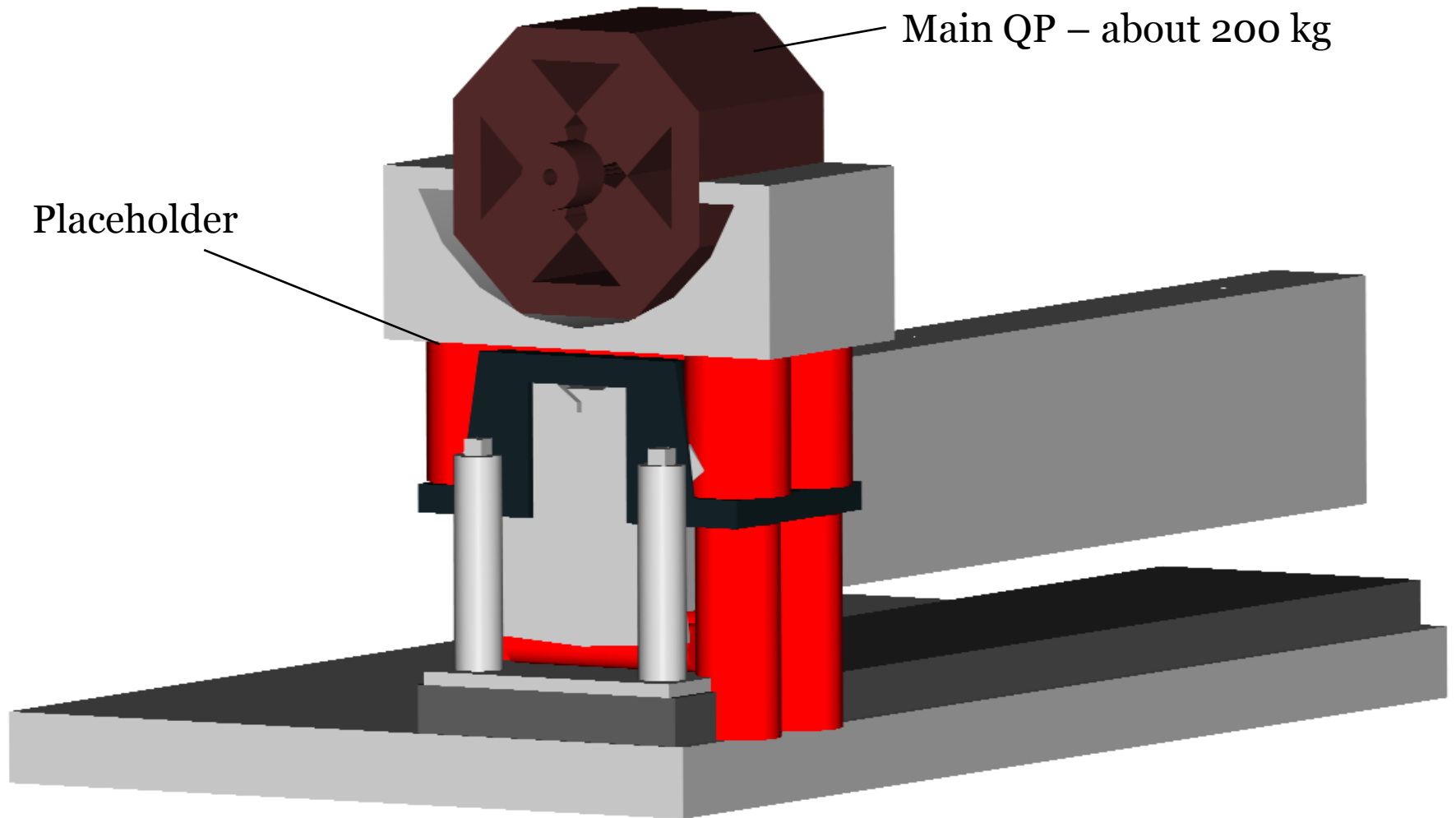
# Concept for pre and final alignment

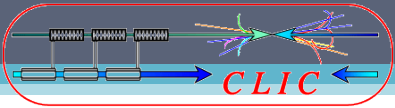
Module and Beam Dynamics working group meeting  
14.02.2008

F. Lackner



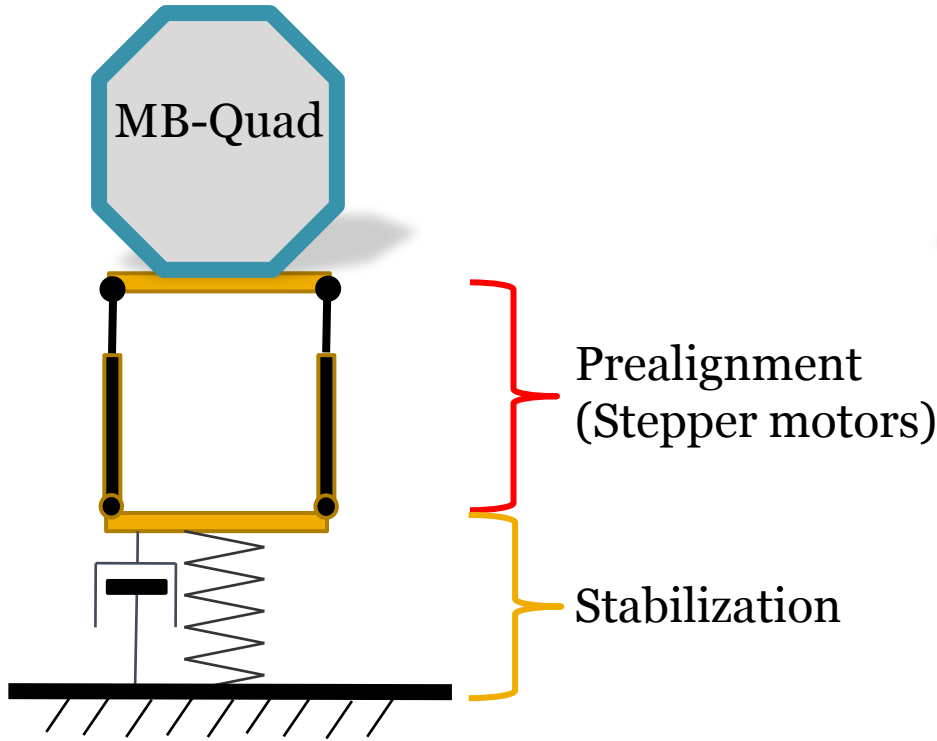
Current situation:



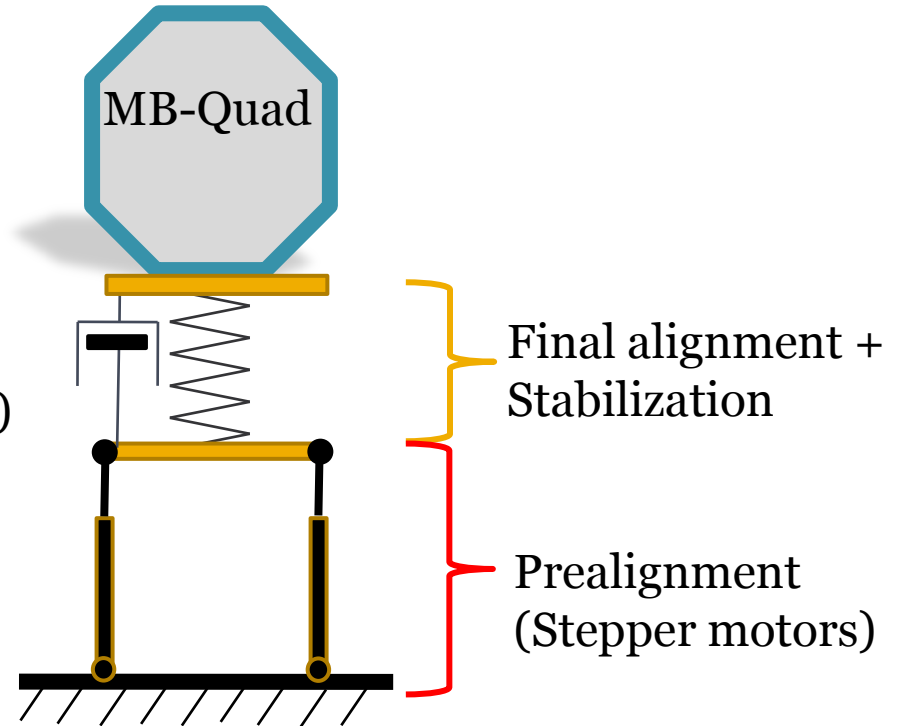


# Alignment Concepts:

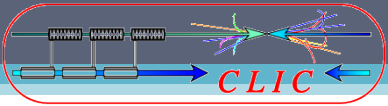
Previous:



Proposal:



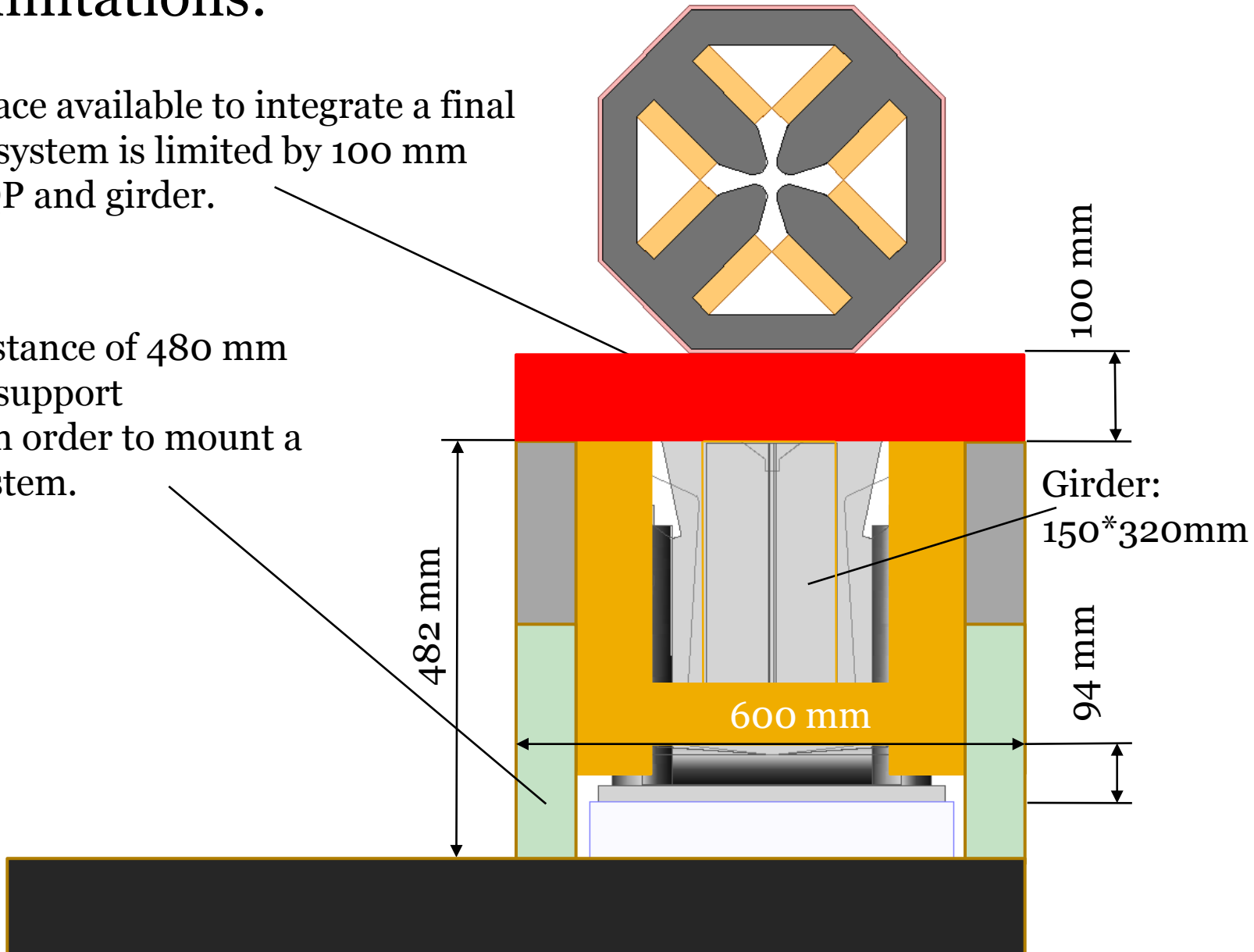
First estimation of the quadrupole weight: 200 kg (acc.to T. Zickler)

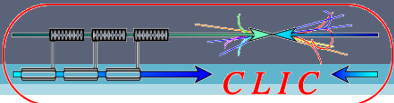


## Space limitations:

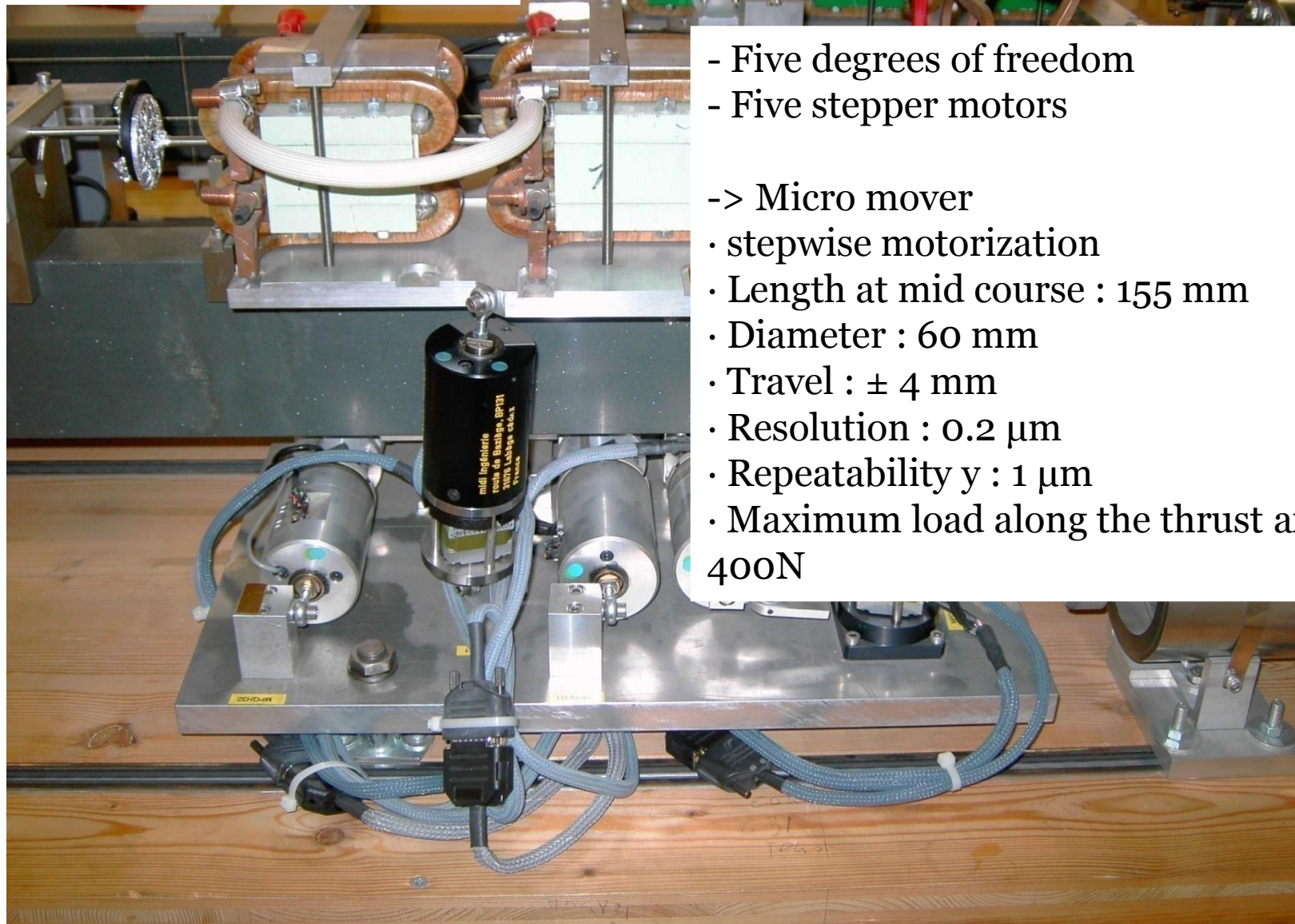
Vertical space available to integrate a final alignment system is limited by 100 mm between QP and girder.

Vertical distance of 480 mm requires a support structure in order to mount a stepper system.

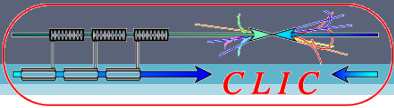




## Prealignment System CTF2:



- Five degrees of freedom
- Five stepper motors
- > Micro mover
  - stepwise motorization
  - Length at mid course : 155 mm
  - Diameter : 60 mm
  - Travel :  $\pm 4$  mm
  - Resolution : 0.2  $\mu\text{m}$
  - Repeatability y : 1  $\mu\text{m}$
  - Maximum load along the thrust axis:  
400N

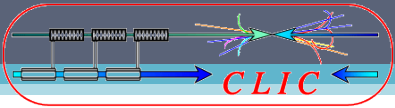


Pre - alignment System (accuracy:  $< 10 \mu\text{m}$ ):

- Keeping the concept from the CTF2 experience (CLIC Note 553 et.al.)
  - Update the design and stepper motor system to the actual state of technology
  - Taking into account the quadrupole weight of about 220 kg  
(incl. fine alignment system)
- > Market research in order to find adequate stepper motors:

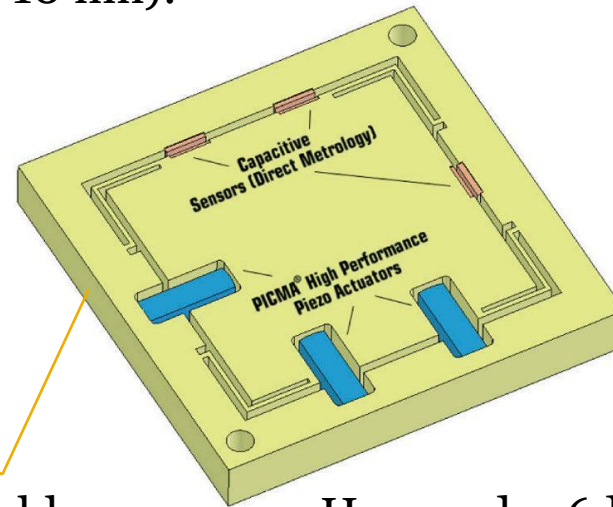
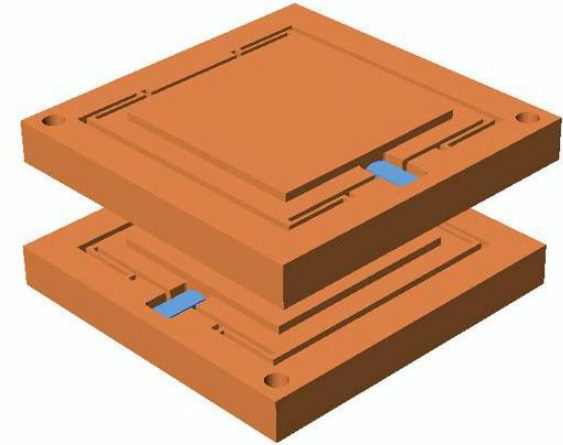
Various acceptable products providing the requirements in order to build a stepper based pre - alignment system with an accuracy to align to better than  $10\mu\text{m}$ .

-> Further steps...



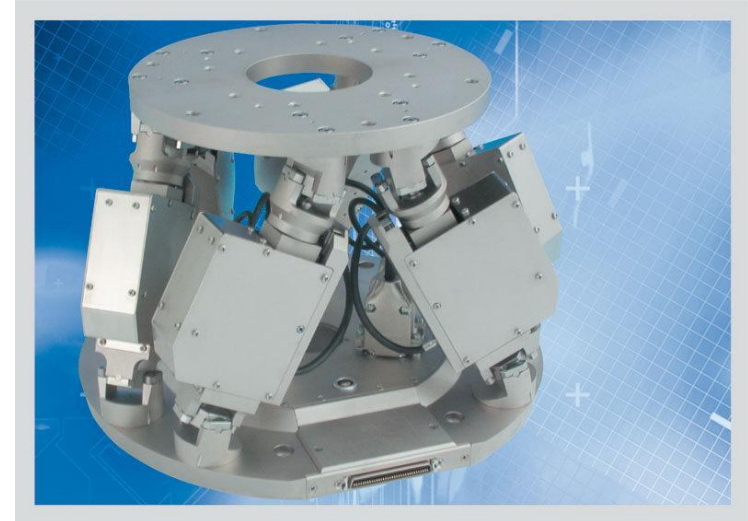
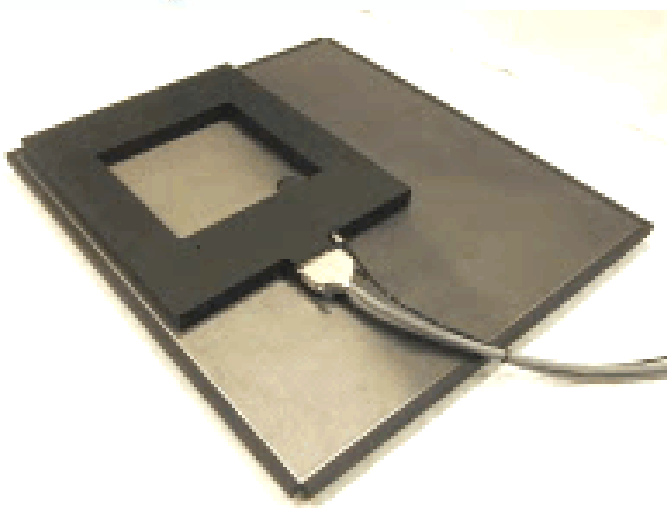
Final - alignment System (accuracy:  $< 10$  nm):

Nano alignment 'available':



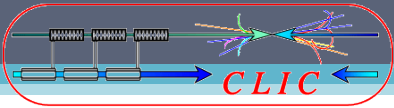
X-Y tables:

Hexapod – 6d alignment:



Main issue:

Load capacitance:  $\leq 10$  kg or slightly above (not satisfactory for a CLIC alignment system)

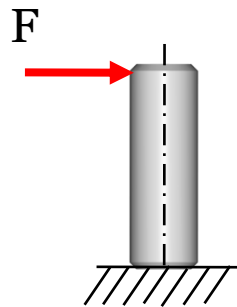
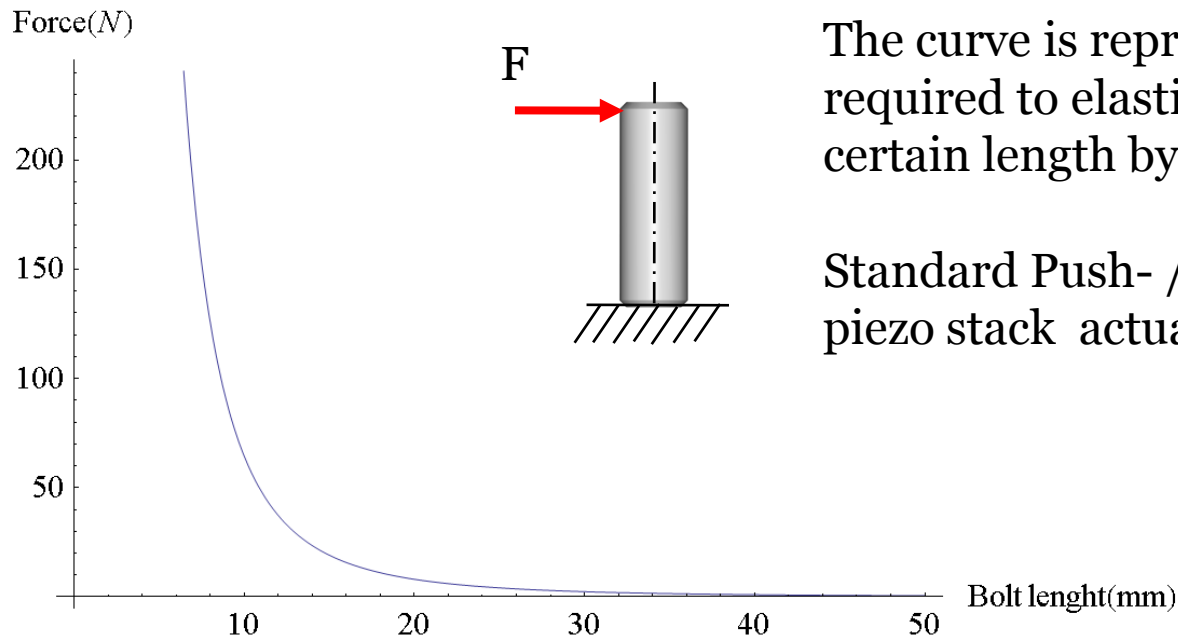


Concept: Combining nano alignment and stabilization for both, vertical and horizontal motion. Developing a fine alignment and stabilization system according to CLIC specifications .

Possible approach:

Controlled deformation of a defined object using piezo ceramics.

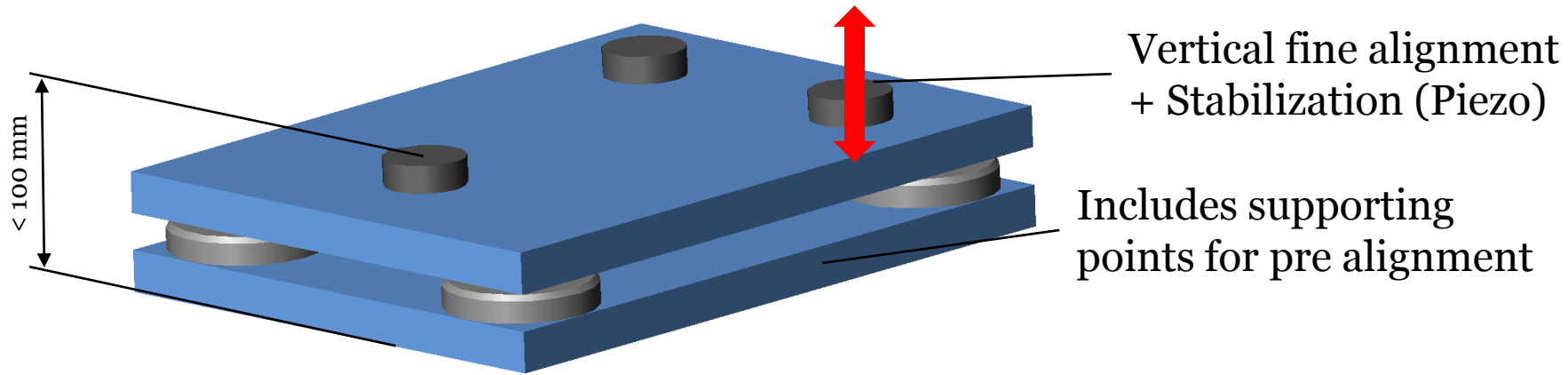
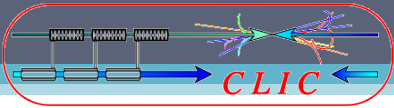
E.g. Al – bolt (standard alloy, Youngs Modulus  $70000 \text{ N/mm}^2$ ), diameter 5 mm



The curve is representing the Force  $F$  required to elastically deform a bolt of certain length by a magnitude of  $10 \mu\text{m}$ .

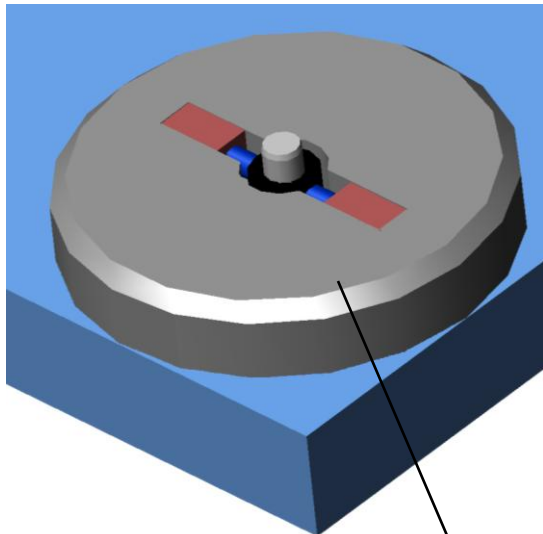
Standard Push- /pull force capacity of small piezo stack actuators is about 1000 N





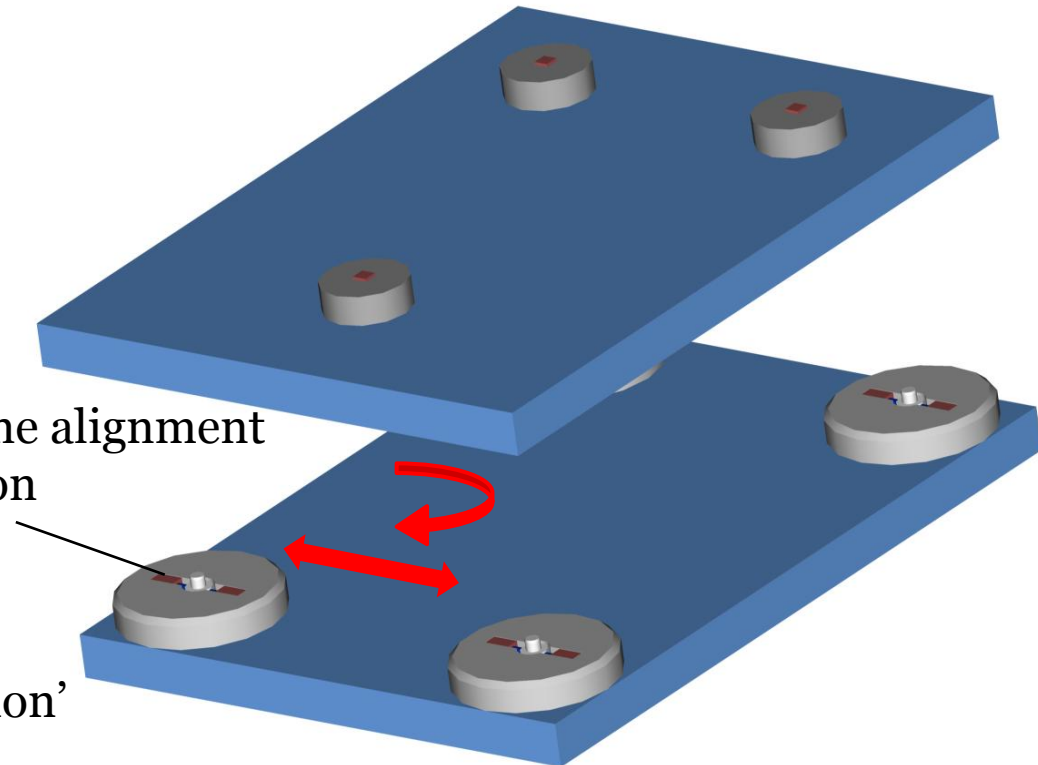
Vertical fine alignment  
+ Stabilization (Piezo)

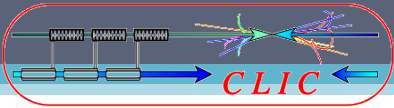
Includes supporting  
points for pre alignment



horizontal fine alignment  
+ Stabilization

Two piezo stacks, precise supports  
applying Force (elastic deformation  
on bolt) 'with or without synchronization'





Possible Piezo: PI (Physikinstrumente, D-Karlsruhe)

PICMA Monolithic Multilayer Piezo Actuator (LVPZT)



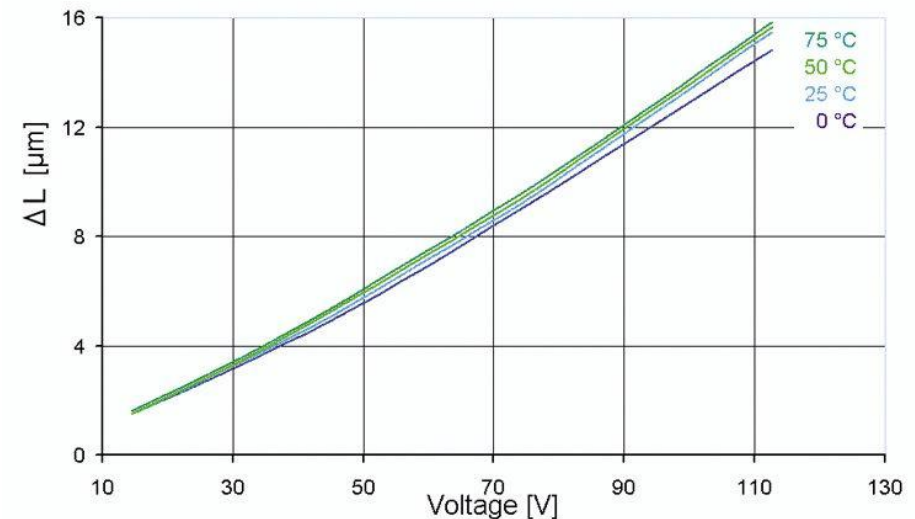
P-885.50 (5 x 5 x 18mm)  
Max. displacement: 18  $\mu\text{m}$   $\pm 10\%$

Force: 900N

Costs: 350 Chf

#### Application Examples

- Precision mechanics and mechanical engineering
- Nanopositioning / high-speed switching
- Active and adaptive optics
- Vibration cancellation
- Pneumatic & hydraulic valves
- Metrology / interferometry
- Life sciences, medicine and biology



# Design and Feasibility study:

**CAD:** Design and support optimization

**FEA et al.:**

- Optimizing the structural stability and interconnection between the two alignment plates.
- Optimize actuator support and bolt dimensions
- Modal analysis,
- Control Technology
- Estimate thermal influence

**Prototyping:**

- First mockup (CERN workshop), production limitations
- Component selection (In collaboration with?)
- Prototype (In collaboration)

**LAB test:** - First Lab tests using a mockup

*~Additionally ~*

**Important question:** - Environment for nanometer precision measurements?

## Operating schedule:

