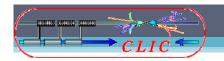
Main Beam Quadrupole Support

Friedrich Lackner (TS-SU)

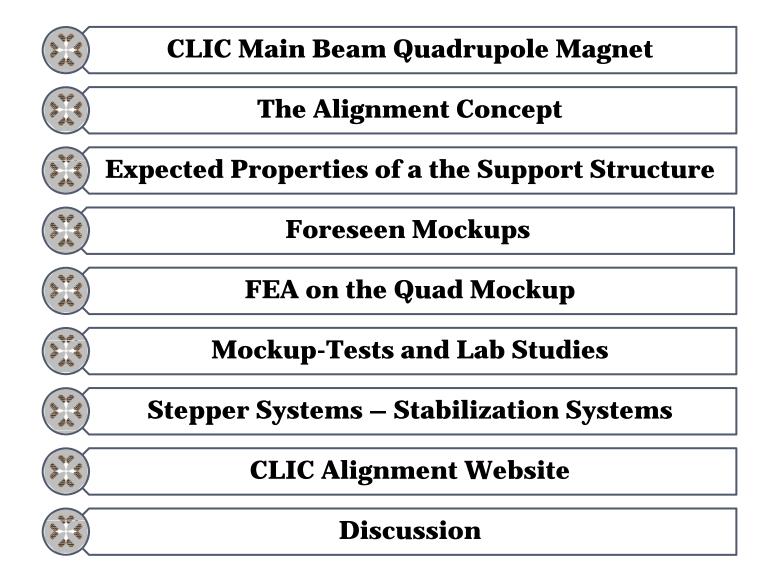


CLIC08 Workshop

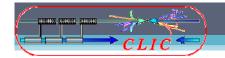




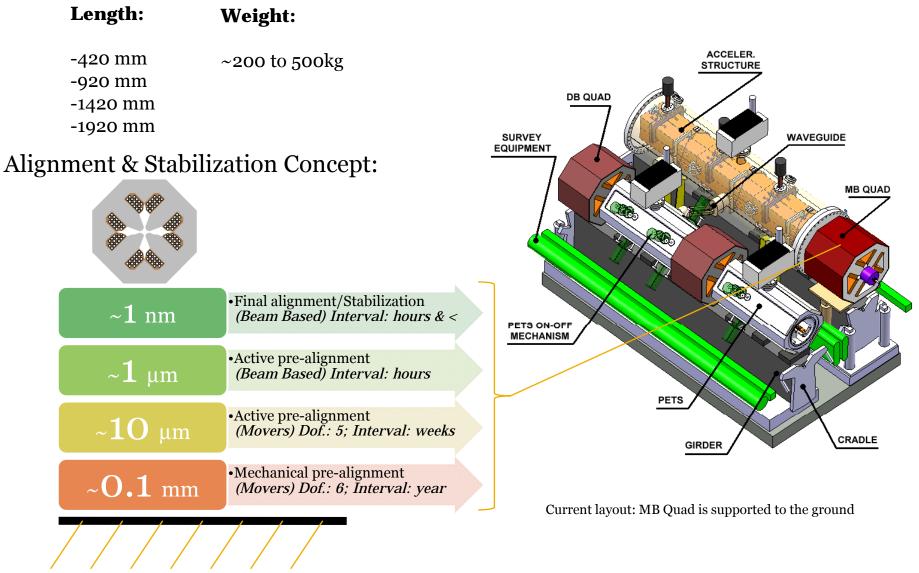
Outline:



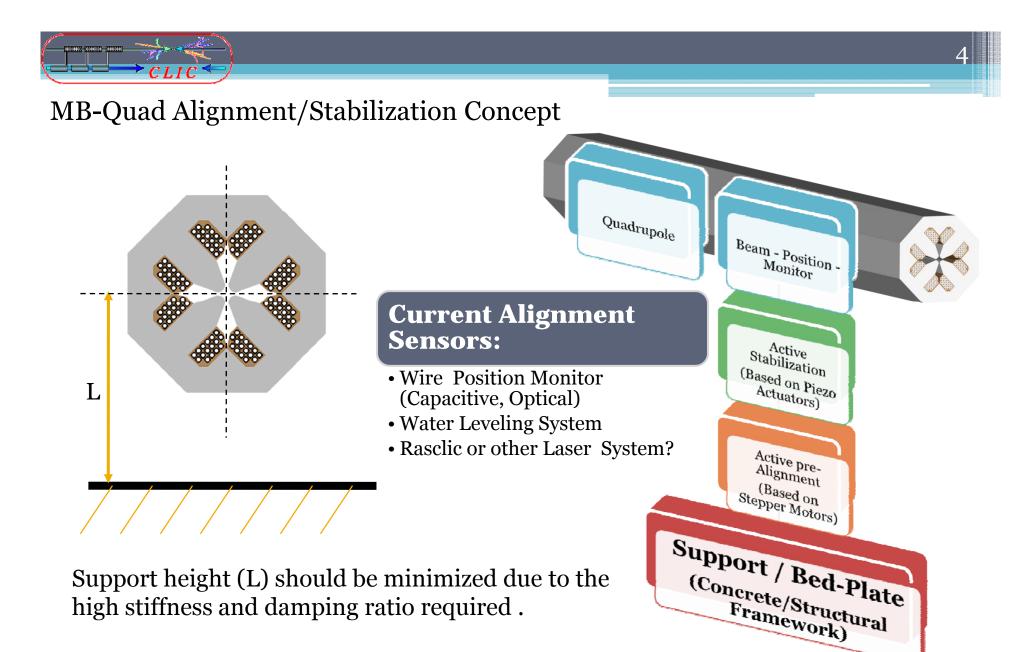
Friedrich Lackner, October 16th, 2008



Baseline: Four different types of Main Beam Quadrupoles:

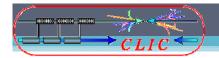


Friedrich Lackner, October 16th , 2008



$$k = \underbrace{AE}_{L} \qquad \zeta = \frac{c}{2\sqrt{km}}$$

Friedrich Lackner, October 16th , 2008



Expected Properties of a CLIC Mainbeam Quadrupole Support

InstallationWeight (optimized)Compact DesignEasy to TransportEasy to AlignEasy to re-AdjustInstallation Time (low)

Cost-Efficient

Operation

Damping Ratio (high)

Stiffness Characteristics (excellent)

Availability (high during entire life cycle)

Temperature *(low sensitivity)*

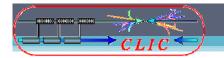
Humidity (low sensitivity)

Radiation (high resistance)

Magnetic field (high resistance)

Requires mockup studies to find best solution

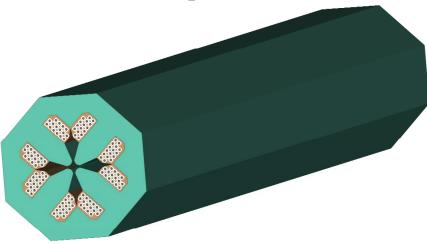
5



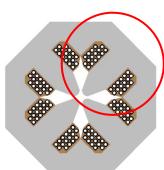
Most complicate version will be studied based on a mockup:

CLIC Main Linac Quadrupole (V4e)	
Magnet	
Nominal gradient	200.1 T/m
Nominal integrated gradient	370.0 Tm/m
Aperture radius	5.0 mm
Iron length	1844.0 mm
Effective length	1849.0 mm
Total magnet weight	393.3 kg
Total magnet length	1914.7 mm
Total magnet width	192.0 mm
Total magnet height	192.0 mm
Coil	
Conductor height	5.6 mm
Conductor width	5.6 mm
Cooling hole diameter	3.6 mm
Total number of turns	16
Cooling	
Number of cooling circuits per coil	1.0
Pressure drop	4 bar
Current density	6.59 A/mm2
Temperature rise	22.3 K
Coolant velocity	1.1 m/s
Total cooling flow	2.6 l/min
Electrical parameters	•
Nominal current	140 A
Magnet resistance (hot)	201.0 mOhm
Power consumption	4108.5 W
Total stored energy	420.7 kJ
Inductance	42.9 mH
Voltage drop (R*I)	29.3 V

Magnet Mockup Design: Thomas Zickler (CERN)

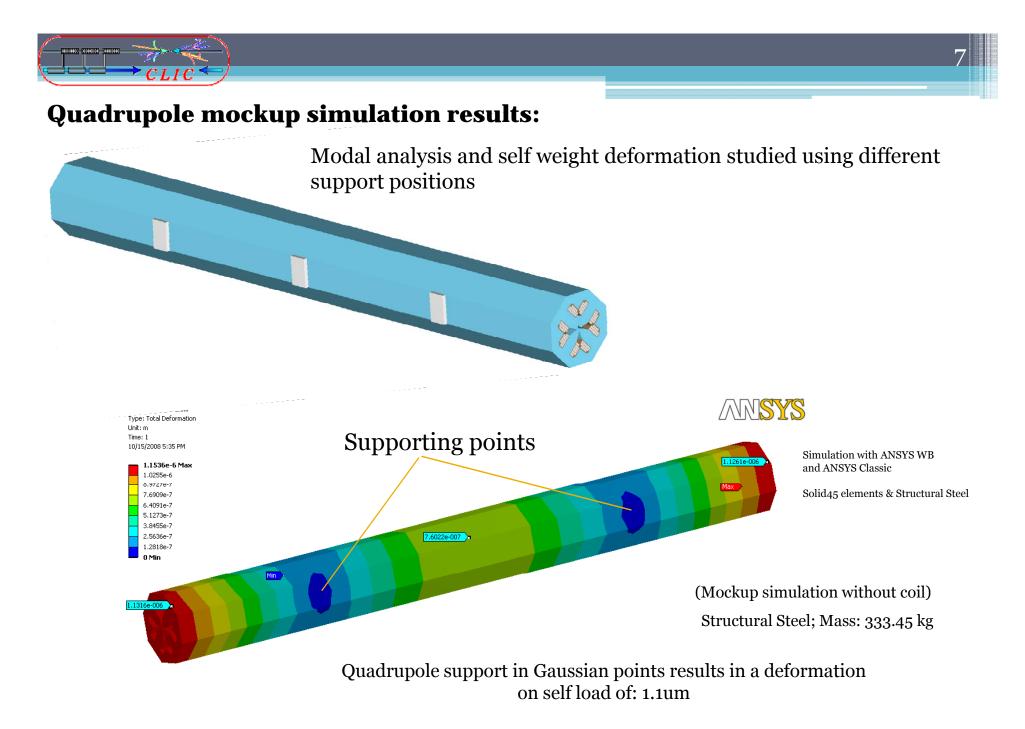


Quadrupole Mockup (Length = 1920mm)



Preliminary Mockup based on a T-shaped Profile will be studied based on a layered assembly and as a solid.

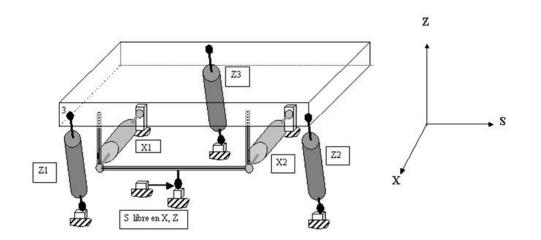
Layerd Profils will be based on Electromagnetic steel (e.g. Stabolit 70 – Thyssen Krupp)





Mockup-Tests and Lab Studies

Previous tests (S. Redaelli, CERN):





Main focus in the 90th was given in the vertical stabilization of the quadrupole magnet

5 Stepper motors were used to align the support to 5 DOF

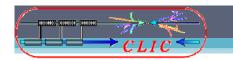
Supports were joints and bearings with low friction

Change of requirements in pre-alignment and especially higher loads requires complete redevelopment. Furthermore documentation regarding reachable Repeatability, Accuracy and Resolution in the 5 DOF of the former System undiscoverable.



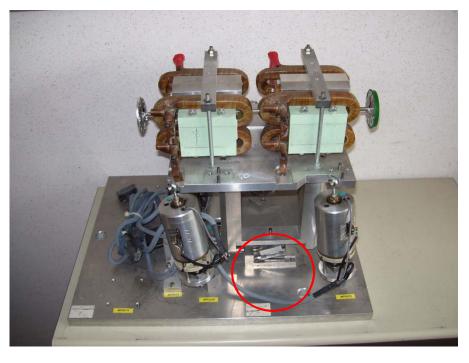
Behaviour of Stepper System can be studied by using the old setup as occasionally mockup

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CTF2 Stepper System reactivated:



Foreseen steps:

- Implementation of feedback loop
- Redesign of critical parts (guiding)

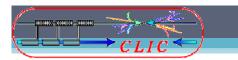
Frictionless Operation Possible?



New software version still communicates with the old setup...



First operations...



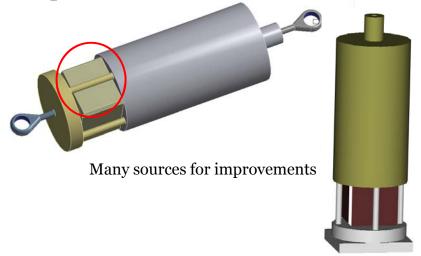
What to learn from an old Setup?



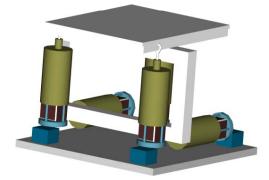
- Will provide important information for the stiffness of the entire support system.
 - Important information of friction and clearance behaviour
- Repeatability of important alignment parameters
 - Important input for ANSYS and for all further studies

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•Modal measurement for CTF2 stepper motor planned within the next weeks.



CTF2 Alignment Concept studied using the FEM



Will help to select adequate movers for CLIC Also answers the question -> Frictionless prealignment required?

Regarding Movers:

Main issue: Insufficient load capacity -> most μ movers and hexapods on the market can adjust loads up to max. 40kg

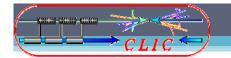
Market research on suppliers for stepper motor development and "frictionless" joints

- PI- Karlsruhe (D)
- ZTS -VVU – Kosice (Sk): Stepper system supplier for LHC low beta magnets
- Midi ingenierie – Labege (F): Stepper system supplier for CTF2



E.g.: PI MIKE M-238 unidirectional repeatability: 0.3 µm, max. load: 40kg

Further development of the M-238 for a load of 100kg -> 15000 CHF/Piece and complete new design for loads up to 200kg



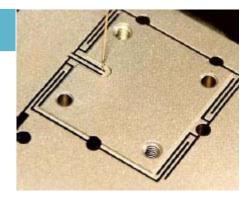
Progress on research for the nano stabilization of the MB-quad:

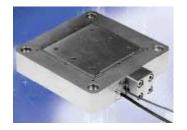
Nano membran -> studied by Kurt Artoos et al. (TS-MME)

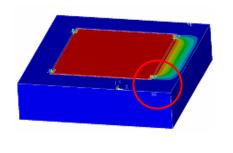
Further possibilities using piezo stacks are studied

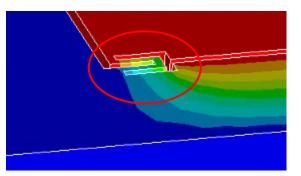
Guiding Flexures: Available solutions are not able to provide required load capacity

Foreseen: Production of a guiding flexure at CERN using wire -electro discharge machining is foreseen.



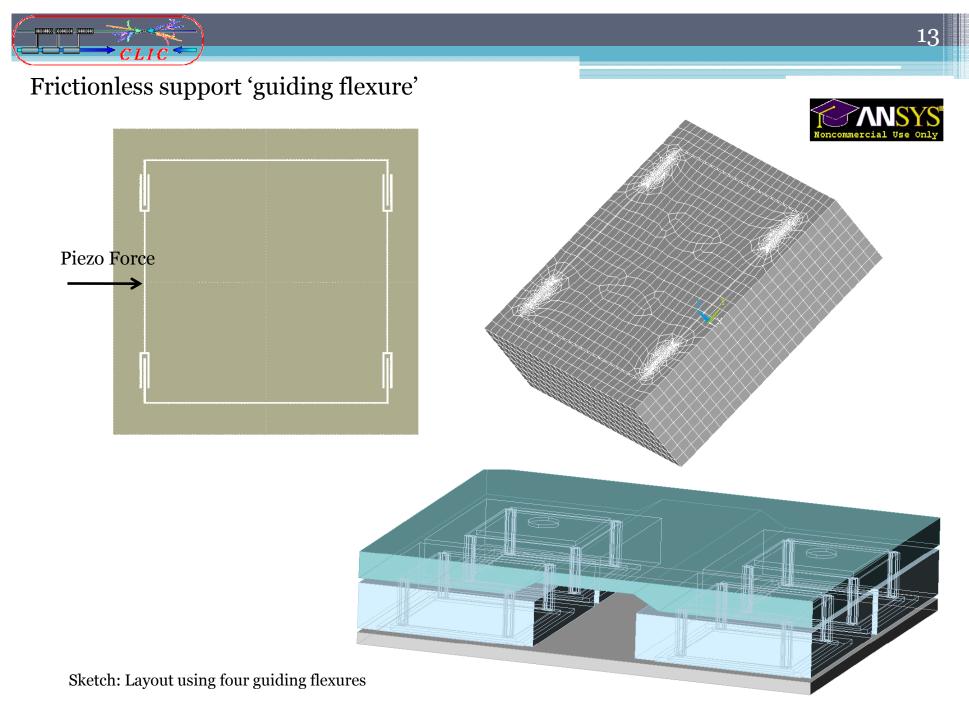




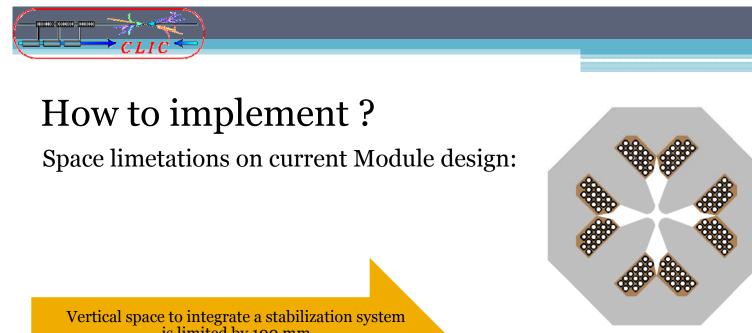


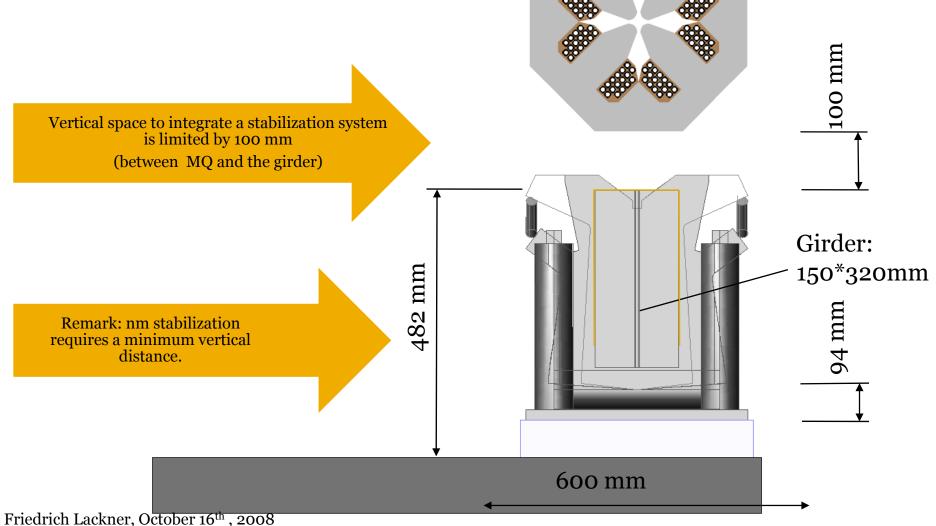
Critical: Implementation and fixation of the piezo actuator

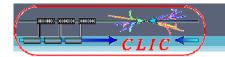
Friedrich Lackner, October $16^{\rm th}$, 2008



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Many solutions, proposals and open work -> still there are so many more open questions to be studied, e.g.:

Frictionless operation of the pre-alignment system by applying linear elastic deformation?

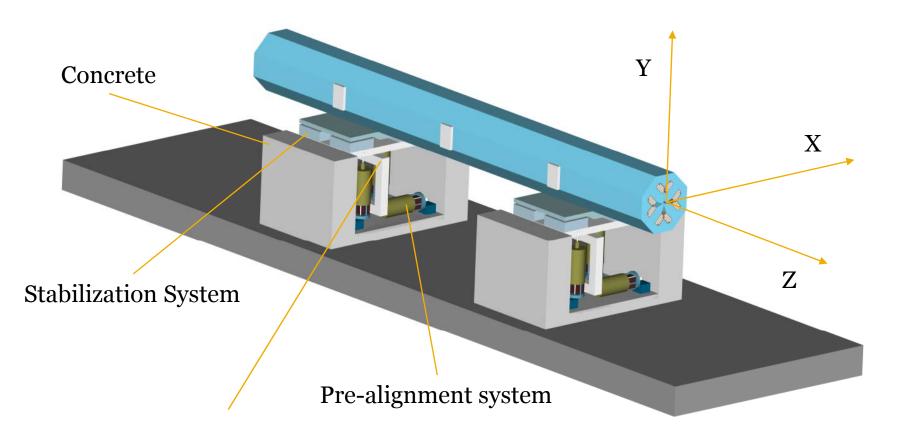
How to provide optimal stiffness characteristics for the stabilization system on beam level?

Decoupling between pre-alignment and stabilization system.

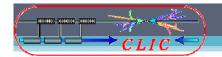
Friedrich Lackner, October 16th , 2008



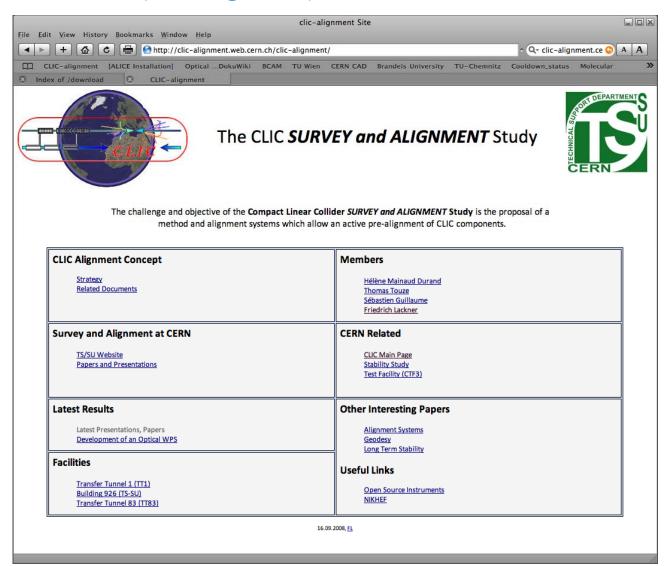
Proposal for Mockup studies:

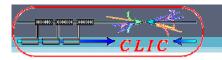


Idea: Increasing the stiffness for a stabilization system by clamping the prealignment system to concrete support (applying clamping force in non critical Z direction).



http://clic-alignment.web.cern.ch/clic-alignment/





Outline:

