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Cooling the FCC beam screens

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Content



- Beam screen cooling requirements and constraints
- Cooling and main sources of exergetic inefficiency
- Cooling of half-cell beam screen: first results
- Conclusions



Functional design map of beam screen



FUNCTION

PROCESS

DESIGN FEATURE

Reduce beam-induced cryogenic loads

Limit residual heat load to cold mass

Low-conduction supports

Intercept synchrotron radiation

High-conductivity copper plating

Increase development time of transverse resistive-wall instability

Limit resistive wall impedance

Cooling at low temperature

Resist eddy-current forces at magnet quench

Structural material with high resistivity

Austenitic stainless steel structure

Preserve field quality in magnet aperture

Low-permeability materials

Pumping slots

Maintain good beam vacuum

Provide pumping from shielded cold surface

Avoid temperatures favoring desorption of common gas species

Limit development of electron cloud

Limit reflectivity and SEY of beam screen surface

Sawtooth absorber

Beam scrubbing



The synchrotron radiation

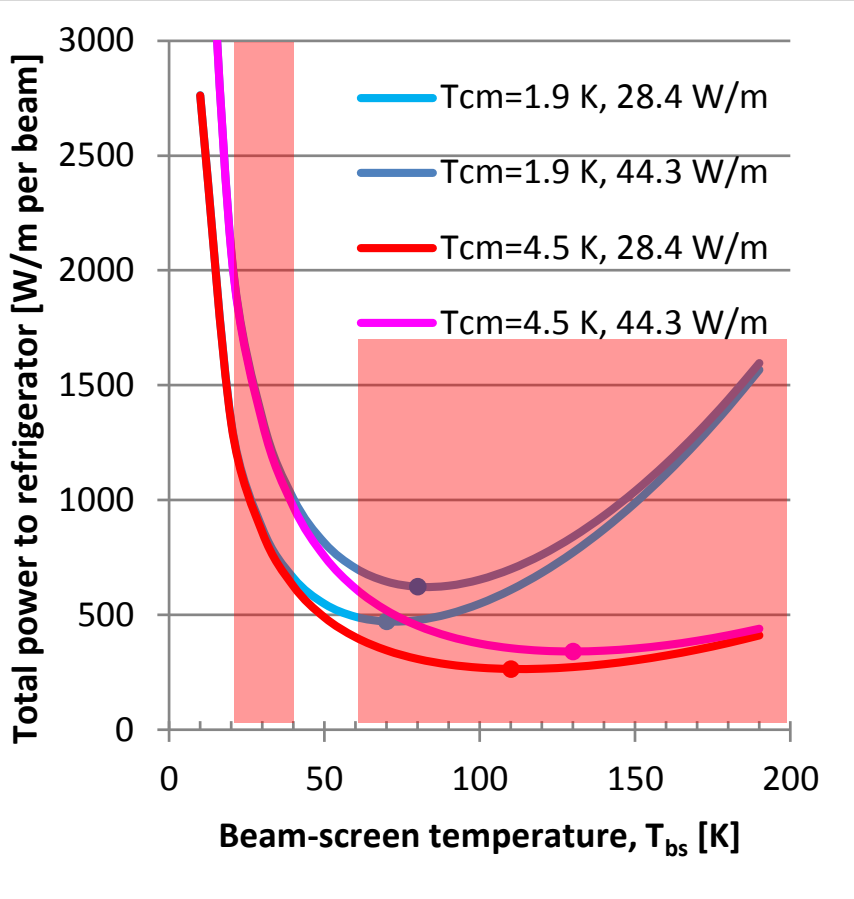


	LHC	FCC (100 km)	FCC (83 km)
P_{SR} in W/m	0.17	28.4	44.3
P_{SR} total in MW	0.008	4.8	5.8

- If this load is falling directly on the magnet cold masses working at 1.9 K, the corresponding total electrical power to refrigerators is 4.3 to 5.2 GW
- Beam screens are mandatory to stop the synchrotron radiation at a higher temperature reducing the electrical power to refrigerator.
 - → Is there a optimum operating temperature?



Main considerations



Forbidden by vacuum and/or by surface impedance

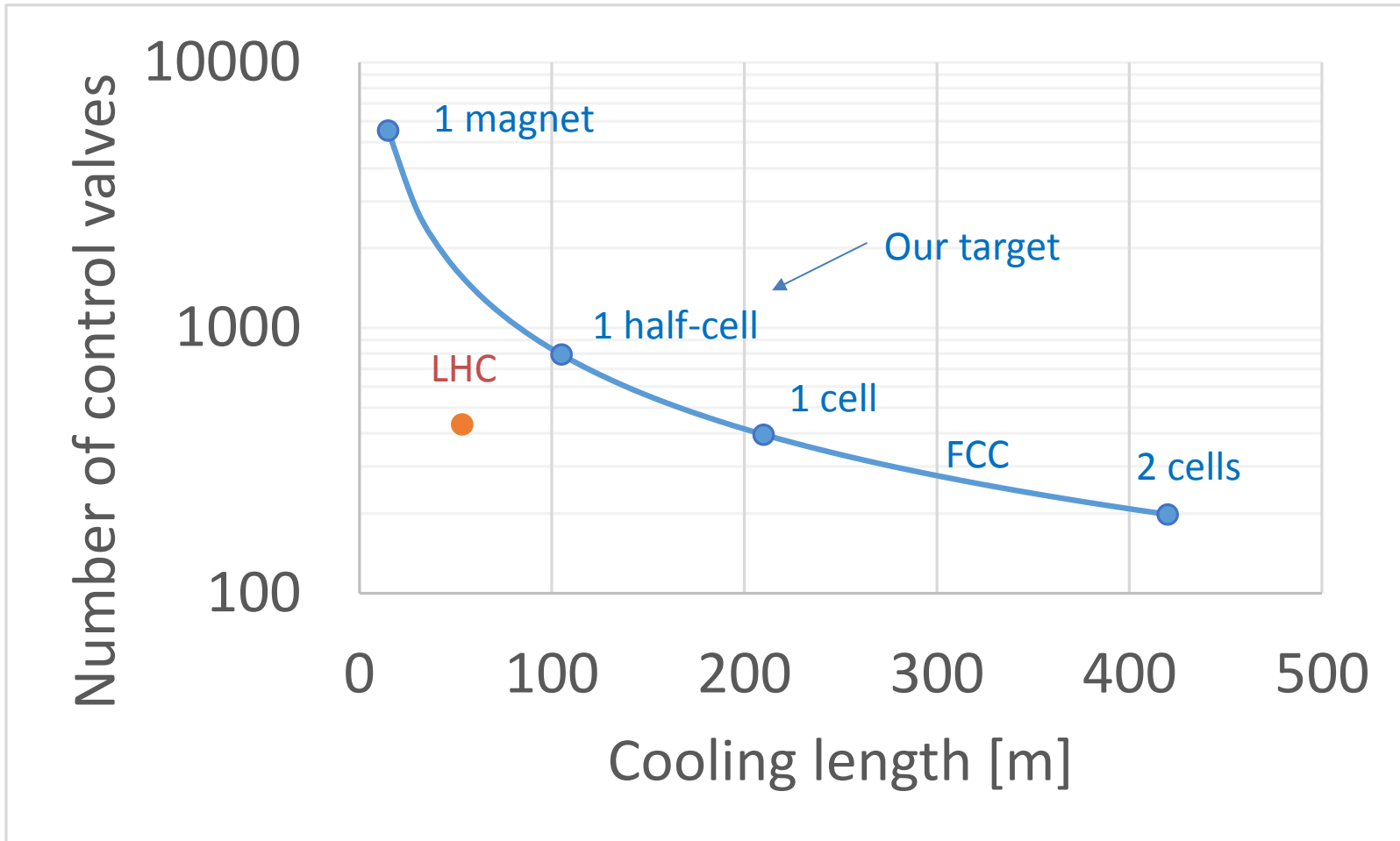
Entropic load ratio for FCC

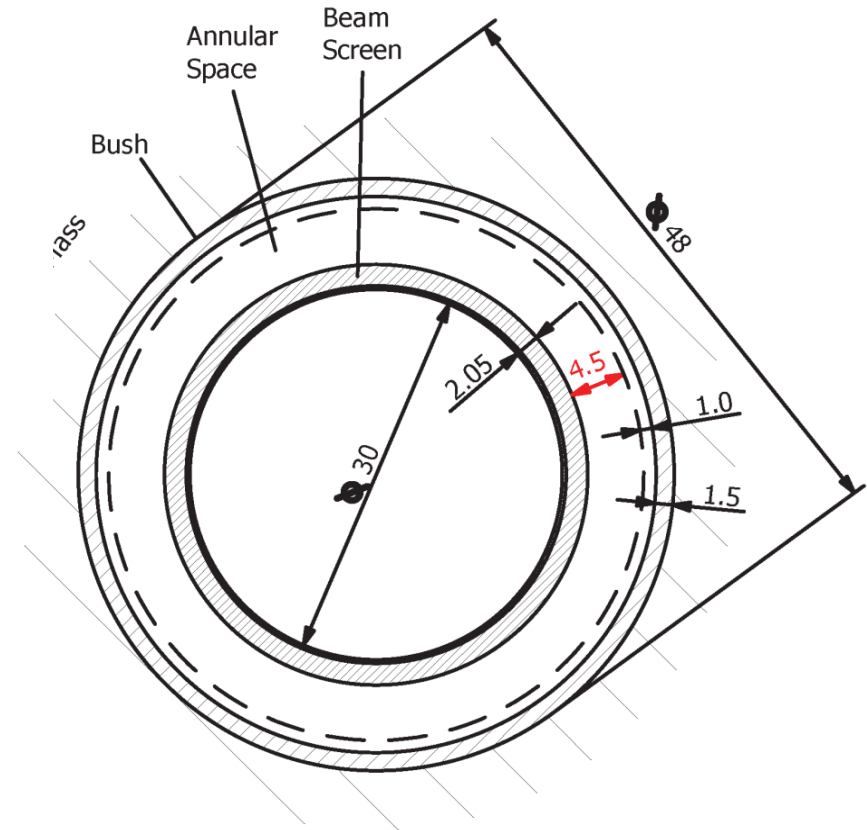
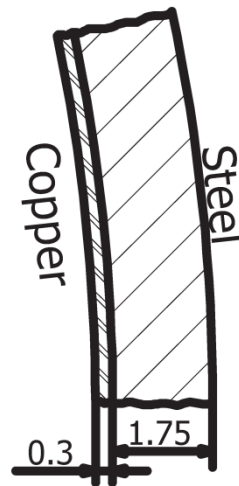
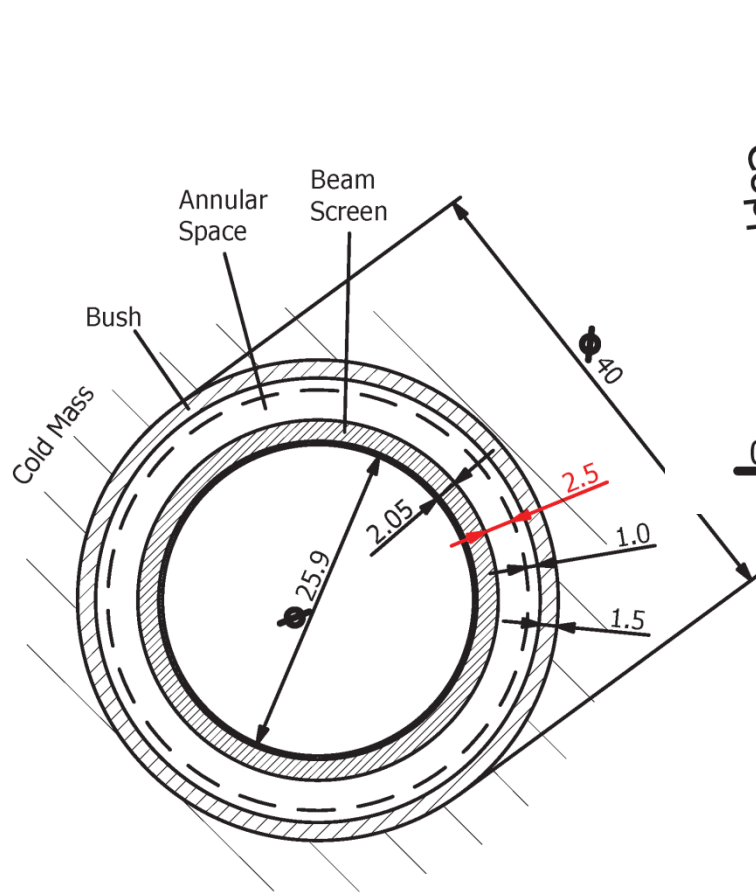
Cold-mass @ 1.9 K	46 %
Beam screen and thermal shield	51 %
Current lead	3 %

Beam screen and thermal shield cooling is the largest refrigeration load of FCC which will required more than 100 MW of electrical power to the refrigerator: Optimization of the corresponding cooling is mandatory.



Control valve number vs cooling length



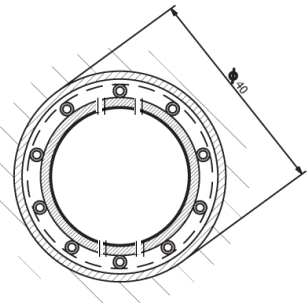


Inner coil diameter: 40 mm
 Beam aperture: 26 mm
 Remaining annular space: **2.5 mm**

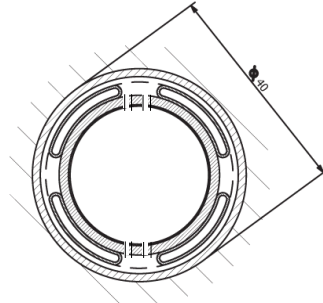
Inner coil diameter: 50 mm
 Beam aperture: 30 mm
 Remaining annular space: **4.5 mm**

- 2 different cold bore diameters
- Cylindrical or oval cooling channels
- Design constraints: free space for pumping slot and beam welding process (LHC type)

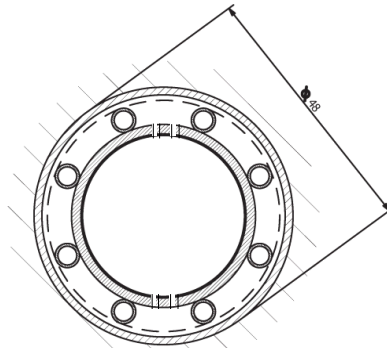
Small CB C10



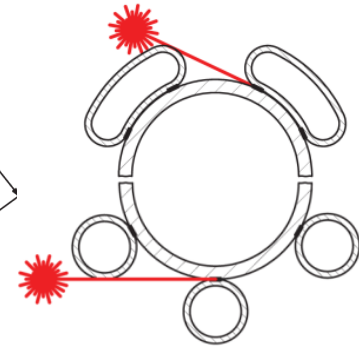
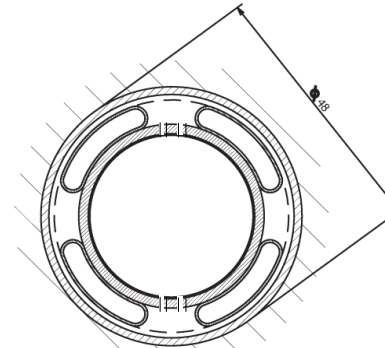
Small CB O4



Big CB C8



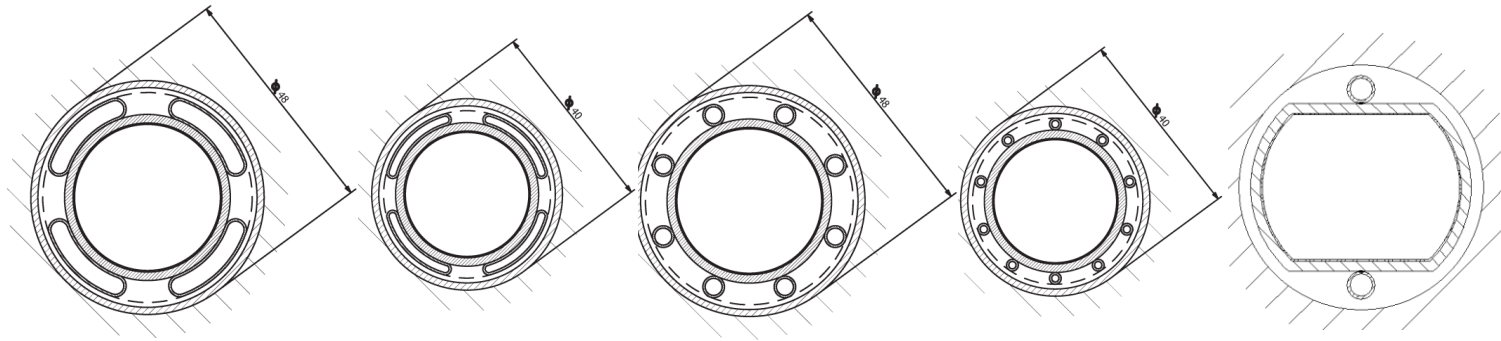
Big CB O4



- 2 different cryogenes (helium or neon)
- Different supply pressures (4 with helium, 2 with neon)
- 2 different heat loads (28.4W/m and 44.3 W/m)
- Cooling constraint:
 - Maximum pressure drop: 80 % of the absolute supply pressure (remaining 20 % for the valve controllability)
 - Maximum flow velocity: 10 % of the sound velocity (to avoid vibrations)

$\Delta P \sim 1/(A^2 \cdot Dh)$, A Section, Dh Hydraulic diameter

	Big CB O4	Small CB O4	Big CB C8	Small CB C10	LHC type
A [mm ²]	274	105	77	18	22
Dh [mm]	6.3	2.7	3.5	1.5	3.7
A ² .Dh [mm ⁵]	471856	30215	20735	468	1711
A ² .Dh ratio w/r to LHC	276	18	12	0.3	1

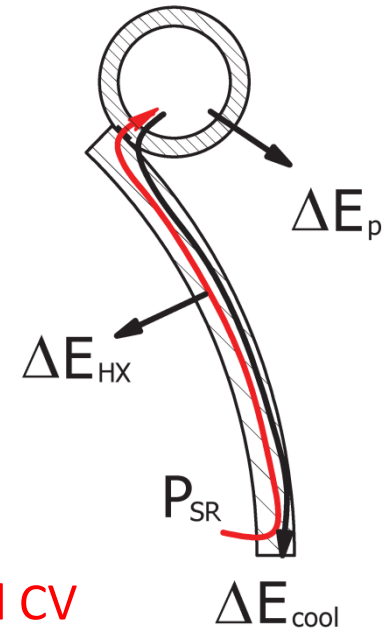


Exergy balance for cooling

Exergy balance only for the BS cooling

Exergy difference between inlet and outlet consists of

- Exergy flow to the Beam Screen
- Exergy loss due to pressure drop in channels and CV
- Exergy loss due to heat transition in Beam Screen (i.e. temperature difference between cryogen and BS)
- Exergy losses in the mixing chambers (unbalanced cooling)



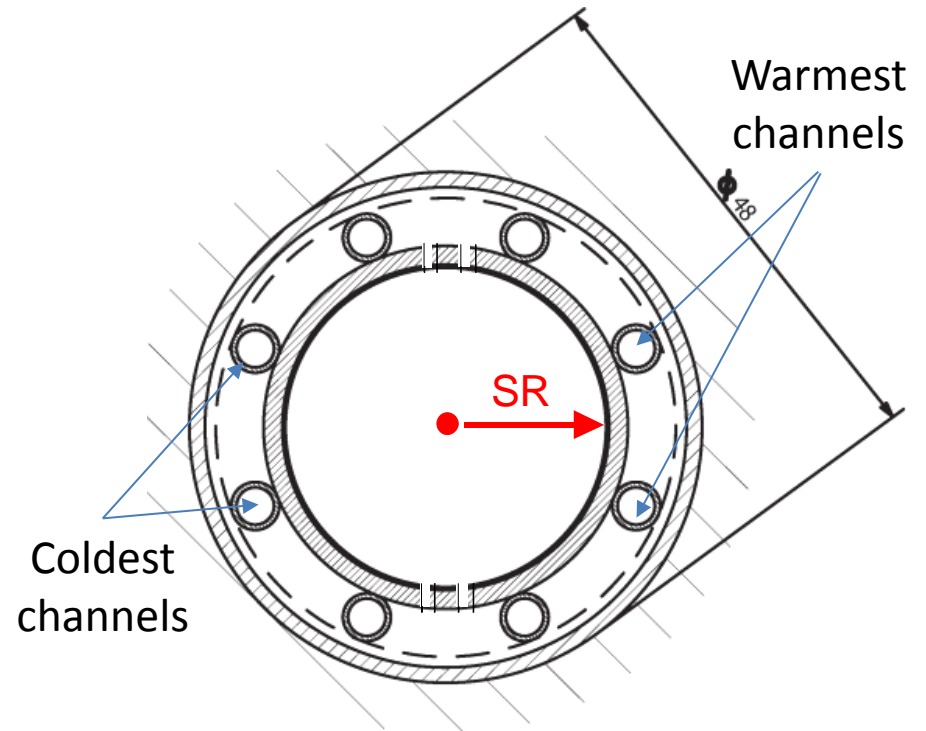
$$\text{Exergetic efficiency} = \frac{\text{Exergy of extracted heat at } T_{BS}}{\text{Exergy difference of cryogen}}$$

Main sources of unefficiency: Unbalance cooling

- Thanks to the copper layer (0.3 mm), the heat is well redistributed in the different parallel cooling channels.

Mass-flow ratio between warmest and coldest channel: $< 1 \%$

Temperature ratio between warmest and coldest channel: $< 2 \%$

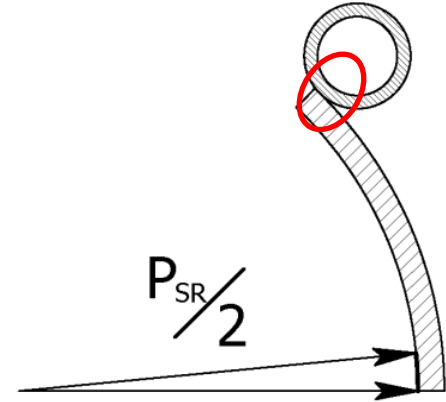




Main sources of inefficiency: BS thermal impedance



- The thermal impedance of the stainless steel part is preponderant in particular at the welding interface with the cooling channel:
 - Thermal impedance: 0.8 W/K.m per welding interface,
 - i.e. for beam screen with 8 welding interfaces, we have a temperature difference of:
 - ~4 K to extract a SR of 44 W/m
 - ~3 K to extract a SR of 28 W/m
 - - i.e. for beam screen with 2 welding interfaces (LHC type), we have a temperature difference of:
 - ~17 K to extract a SR of 44 W/m
 - ~11 K to extract a SR of 28 W/m
- To guaranty a T_{max} of 60 K, we have to lower the cooling temperature of the cryogen, i.e. increase the refrigeration cost.





Main sources of inefficiency: pressure drop



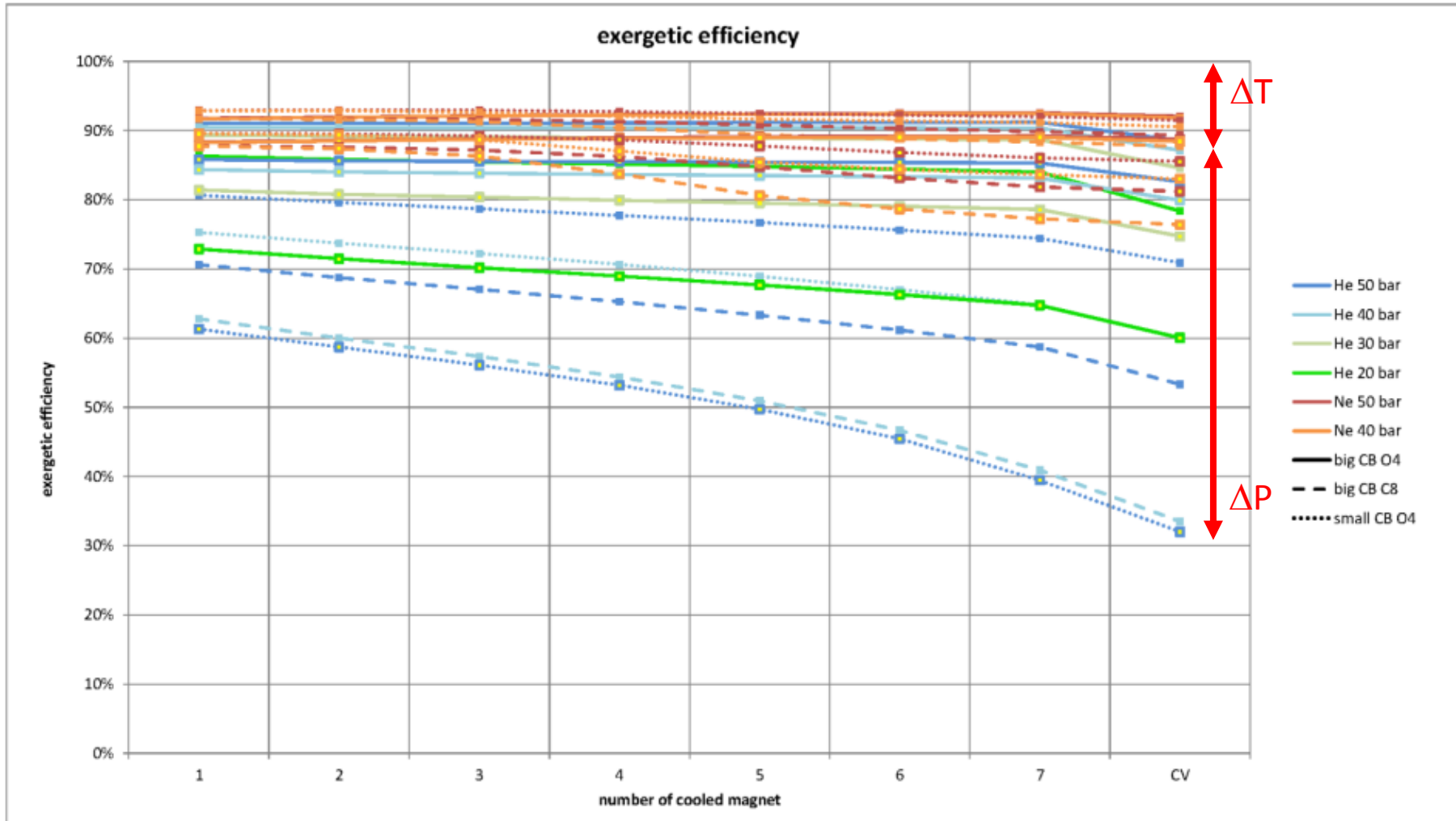
- Pressure drop is a source of irreversibility, which requires to be compensated by additional compression power dissipated:
 - at cold temperature in case a cold circulator/compressor
 - at cold and ambient temperature in case of a warm circulator/compressor.



Calculation results

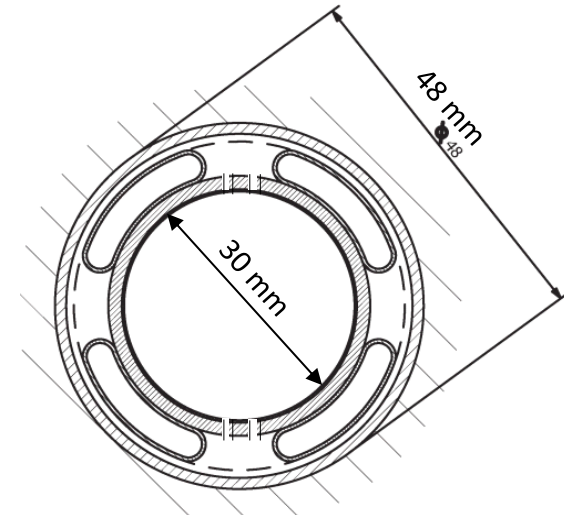


Out of the 48 possible combinations 21 were able to meet the half-cell cooling conditions

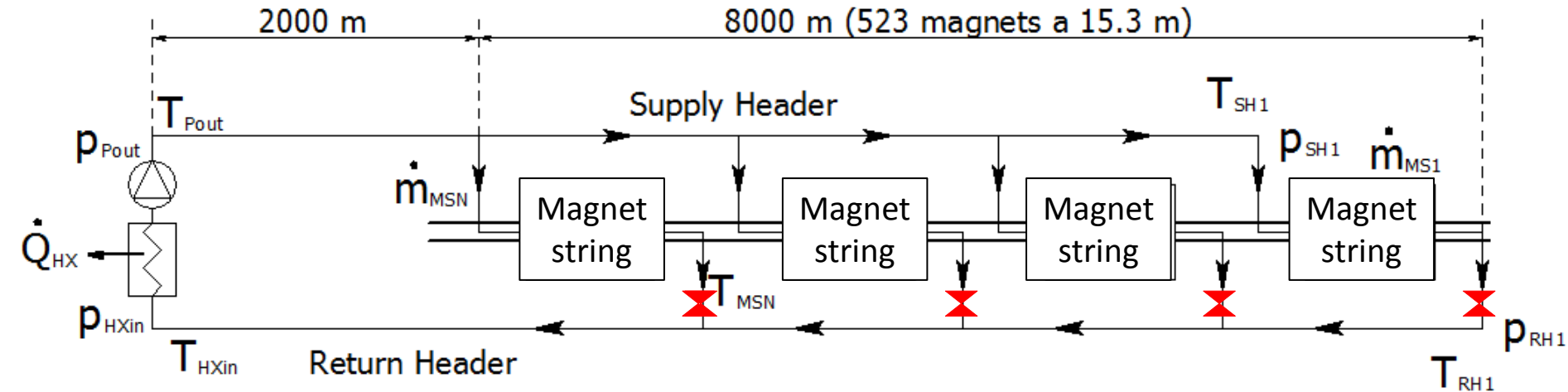


- Considering the cooling performance in nominal condition, the optimum configuration is large oval channels cooled with neon.
- But considering:
 - off design operation with neon becoming solid after a cooling stop (power cut).
 - cost and availability of neon,
 - possible activation of neon by the beam losses,

The best compromise is large oval channels cooled with helium. (loss of few points on the exergetic efficiency)



Supply pressure [bar]	Exergetic efficiency [%]	
	He	Ne
20	78	87
30	84	89
40	87	91
50	89	91



- Removing control valve could allow to decrease the cooling length (one loop per magnet ?) and consequently the required cooling channel diameter/number. In addition it will reduce the capital and maintenance cost.
- However, pressure drop in supply/return headers could create unbalances in the loop cooling which will impact the exergetic efficiency. In addition, off-design operation (no beam, ramp-up/down, operation at reduced beam energy...) must be carefully analysed.
- Study is in progress...

- Study of the beam screen cooling is in progress.
- Different beam screen geometries have been studied: large oval channels cooled with helium at 40-50 bar is recommended for efficient cooling.
- Iteration with other design features must continue in collaboration with vacuum colleagues.

