



# Cryogenic refrigeration with neon-helium mixtures for FCC beam screens

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- Collaboration between CERN and TUD
- Refrigeration below 80 K / The Helium concept
  - Compressors
  - Heat Exchangers
  - Coolant circulation
  - Cycle variants
- Summary and next steps

## Scenario:

For the FCC-hh, 12 cryogenic plants are planned, each should provide refrigeration for:

- beam screens
  - 500 kW @ 40..60 K
- thermal shields
  - 70 kW @ 40..60 K
- current leads
  - 2000 kA per plant (1000 kA in and out)
  - additional stream @ 40 K

In comparison to LHC, refrigeration in the temperature range 40..60 K became a new challenge → more than 50 % of the equivalent refrigeration capacity are needed in this range

## Restrictions:

- hydraulic impedance beam screen: 230 parallel pipes, length 30 m, inner diameter 5.55 mm
- hydraulic impedance thermal shields: straight pipe, 16 km length, inner diameter 100 mm
- pressure is limited to 50 bar

Scope of the study:

Design of a highly efficient refrigeration cycle to provide the refrigeration capacity

Focus on:

- cooling of beam screens, thermal shields and current leads
- refrigeration cycle, efficiency
- requirement and characteristics of specific hardware
- proposals of a reference cycle
- recommendations and inventory of specific developments

For refrigeration below 80 K the standard is an oil flooded *screw compressor* with an isothermal efficiency of 50..60 %

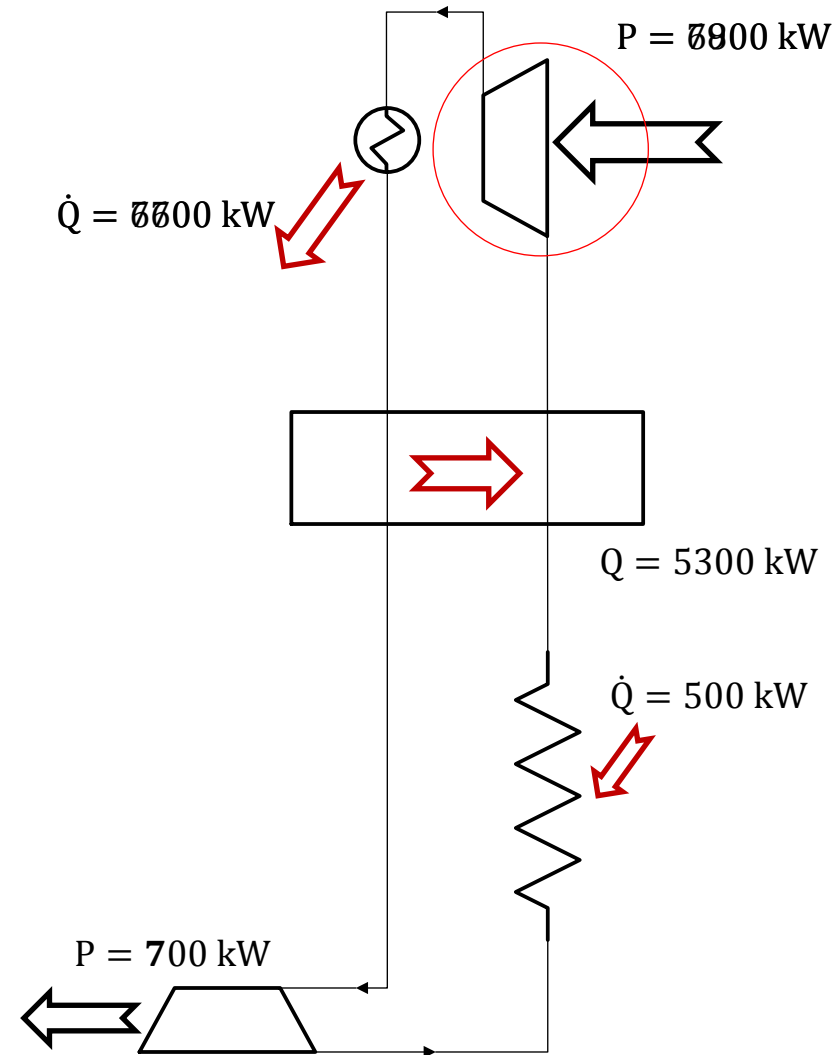
→ Is there a more efficient alternative?

A *centrifugal compressor* with isentropic efficiency of 83 % with an aftercooler can reach a isothermal efficiency of 70 %.

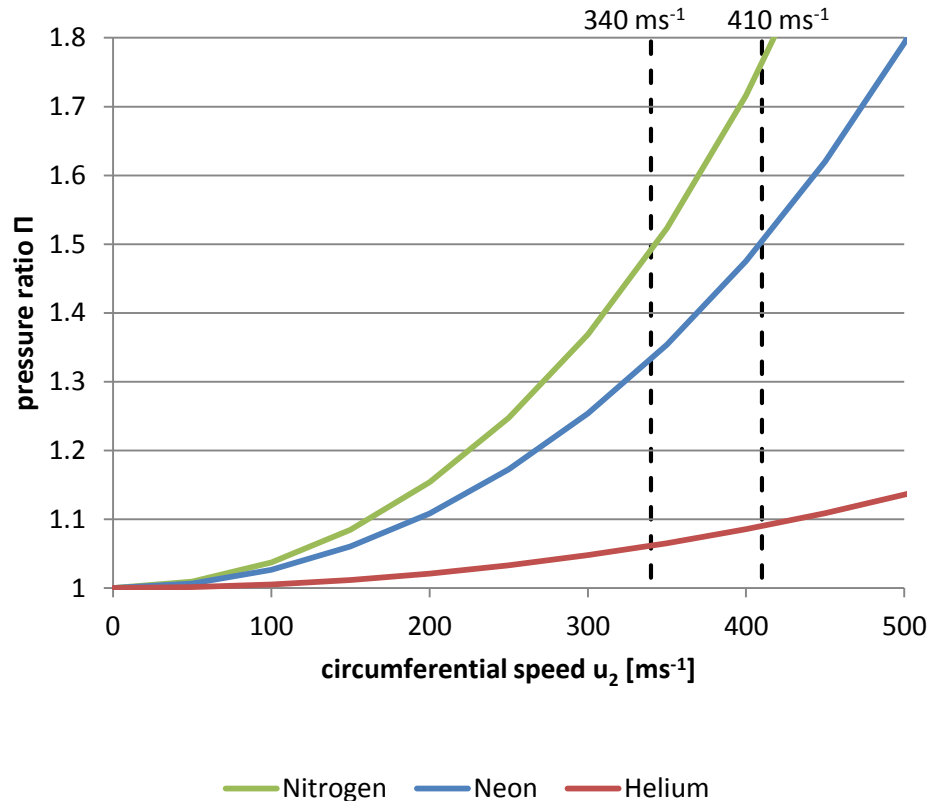
→ Why not use a centrifugal compressor?  
Optimal compressor,  $u_2 = 340 \text{ m/s}$ ,  $\psi = 0.65$

	Pressure ratio
Helium	1.06
Nitrogen	1.49

→ Very low pressure ratio per stage with helium, high number of stages



- Is it possible to increase the pressure ratio per stage?
- A higher circumferential speed allows a higher pressure ratio
  - Limitations due to structural robustness, rotor dynamics etc.
  - The other possibility is to use a gas with a higher molecular weight
  - Only alternative useable gas in the respective temperature region is neon

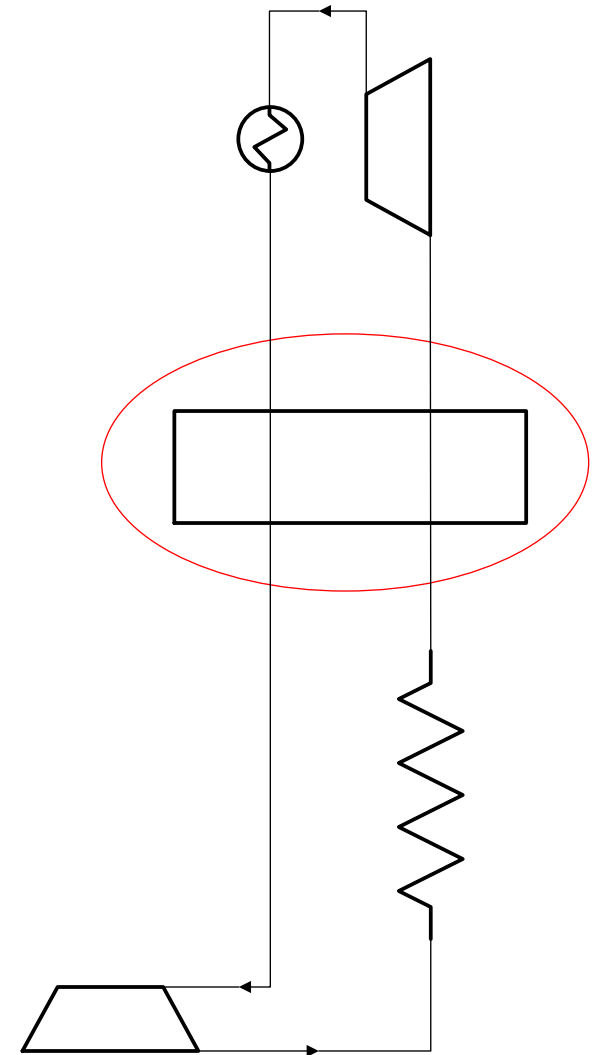


→ Why not use pure neon?

Neon has inferior heat exchange properties compared to helium:

	Neon	Helium
$\lambda$ [W/mK]	48	156
$c_p$ [kJ/kg]	1.0	5.2
$\rho$ [kg/m <sup>3</sup> ]	0.81	0.16
$\eta$ [ $\mu$ Pa s]	31	20

Heat exchangers (HX) with neon will be bigger, less efficient and/or have higher pressure drop



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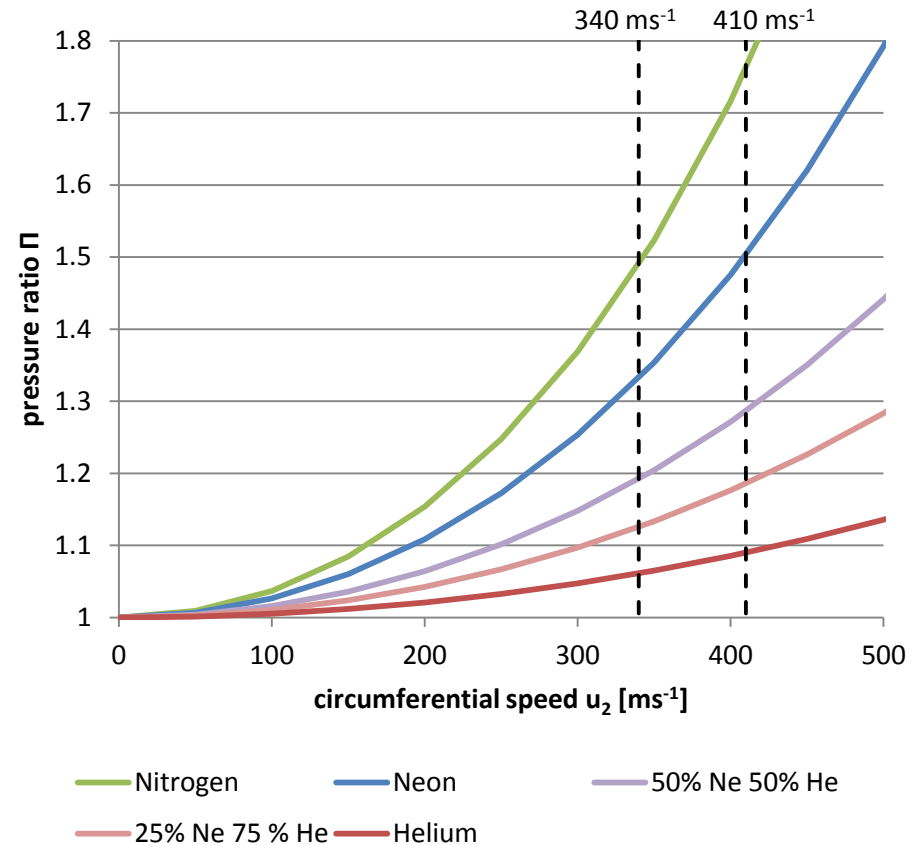
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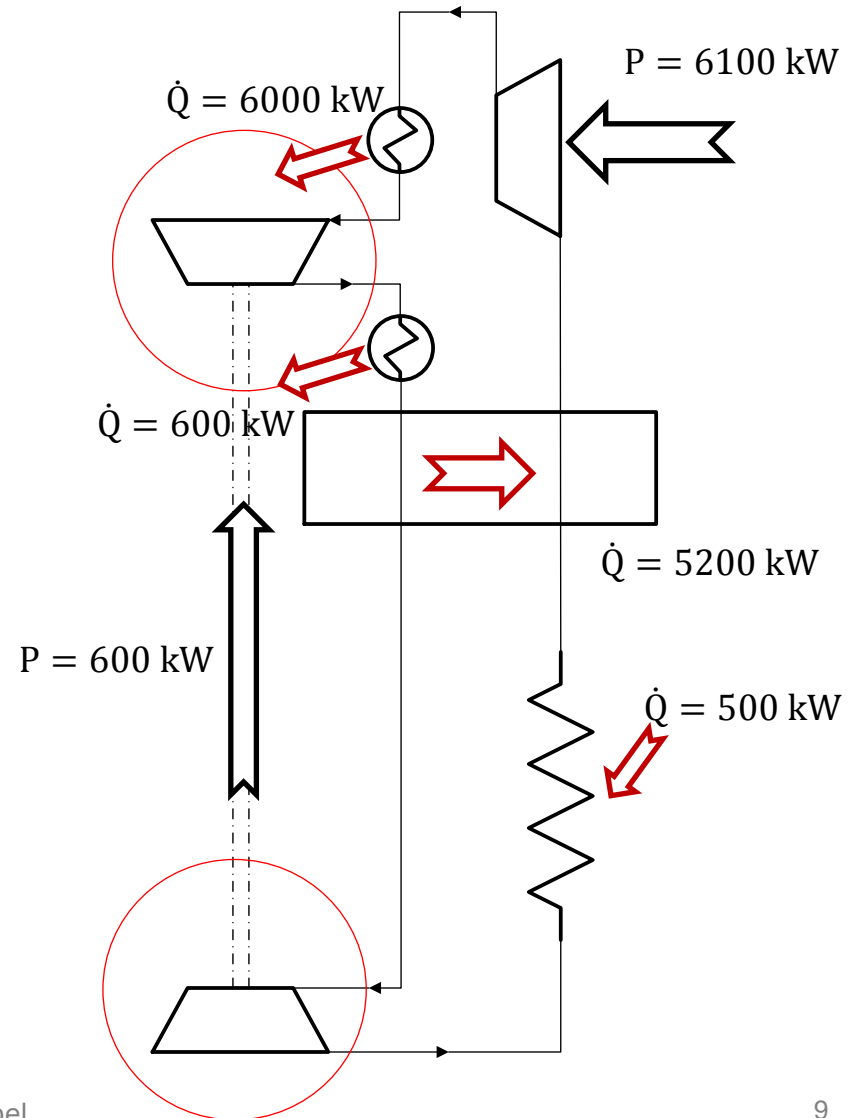
→ A mixture of neon and helium (*Nelium*) can be compressed in a centrifugal compressor and has moderate pressure losses in the HX





With pure helium, the work of the expansion turbines is dissipated:

Using a mixture of helium and neon allows to recover the mechanical energy of the turbines



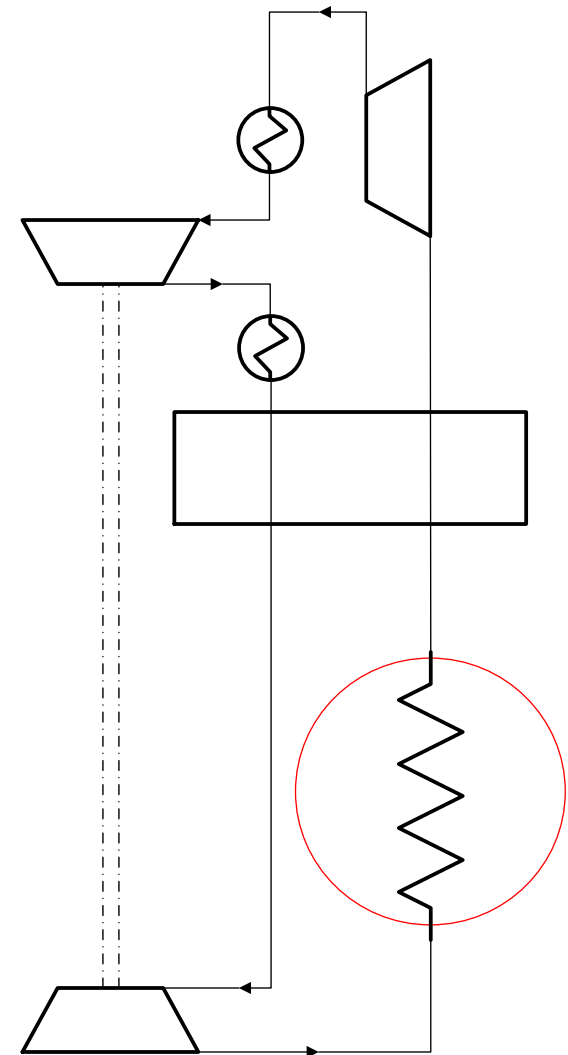
- Is it possible to use the refrigerant directly in the beam screen?

Capillaries in the beam screen are long and narrow: necessary inlet pressure for exhaust pressure of 1 bar:

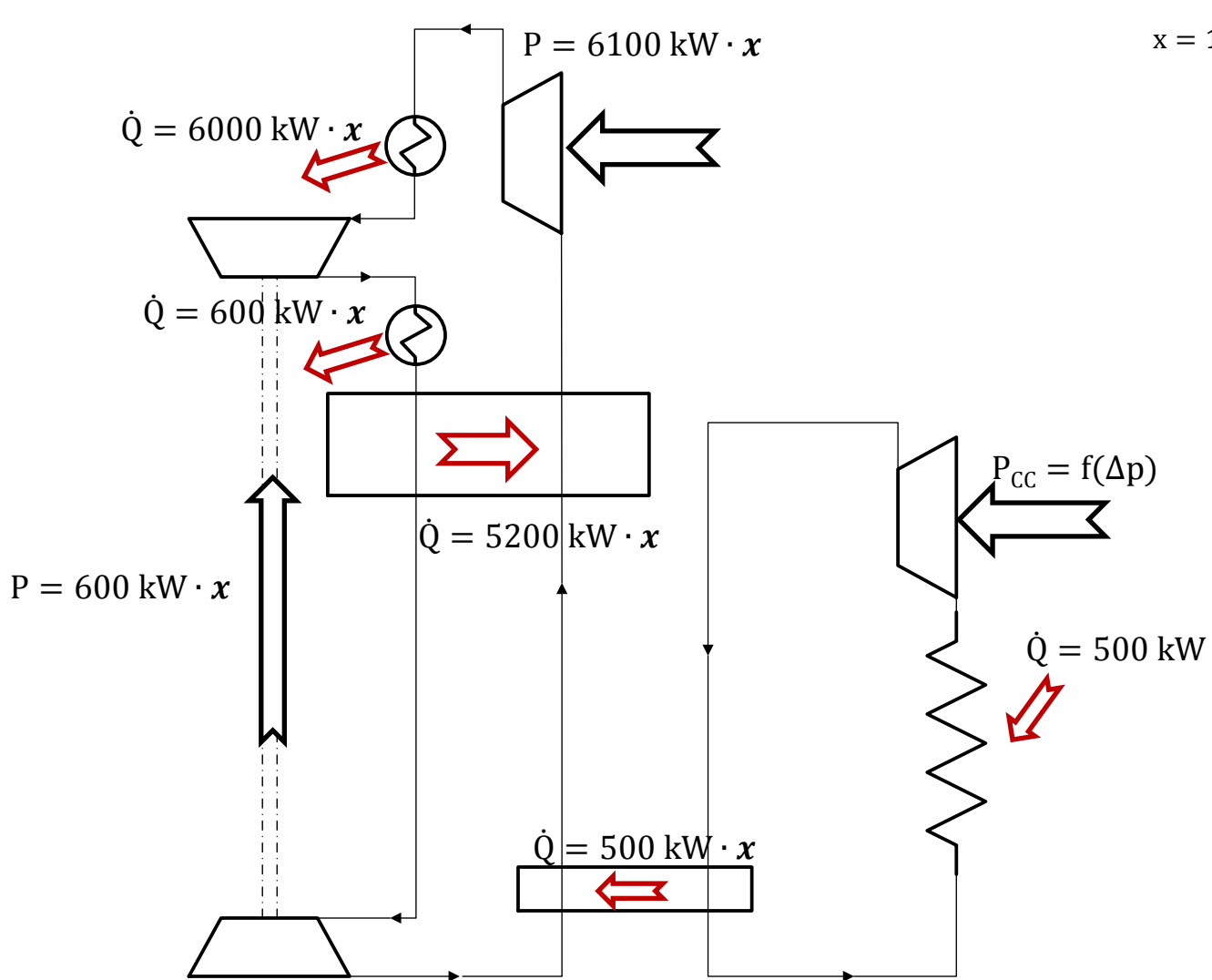
	<b>p [bar]</b>
Helium	26
Neonium 25 (25 mol-% Ne)	34

Pressure ratio of the expansion turbine is about 5

- The pressure at the outlet of the main and booster compressors would be very high
- Instead of cooling the beam screen directly use a second loop

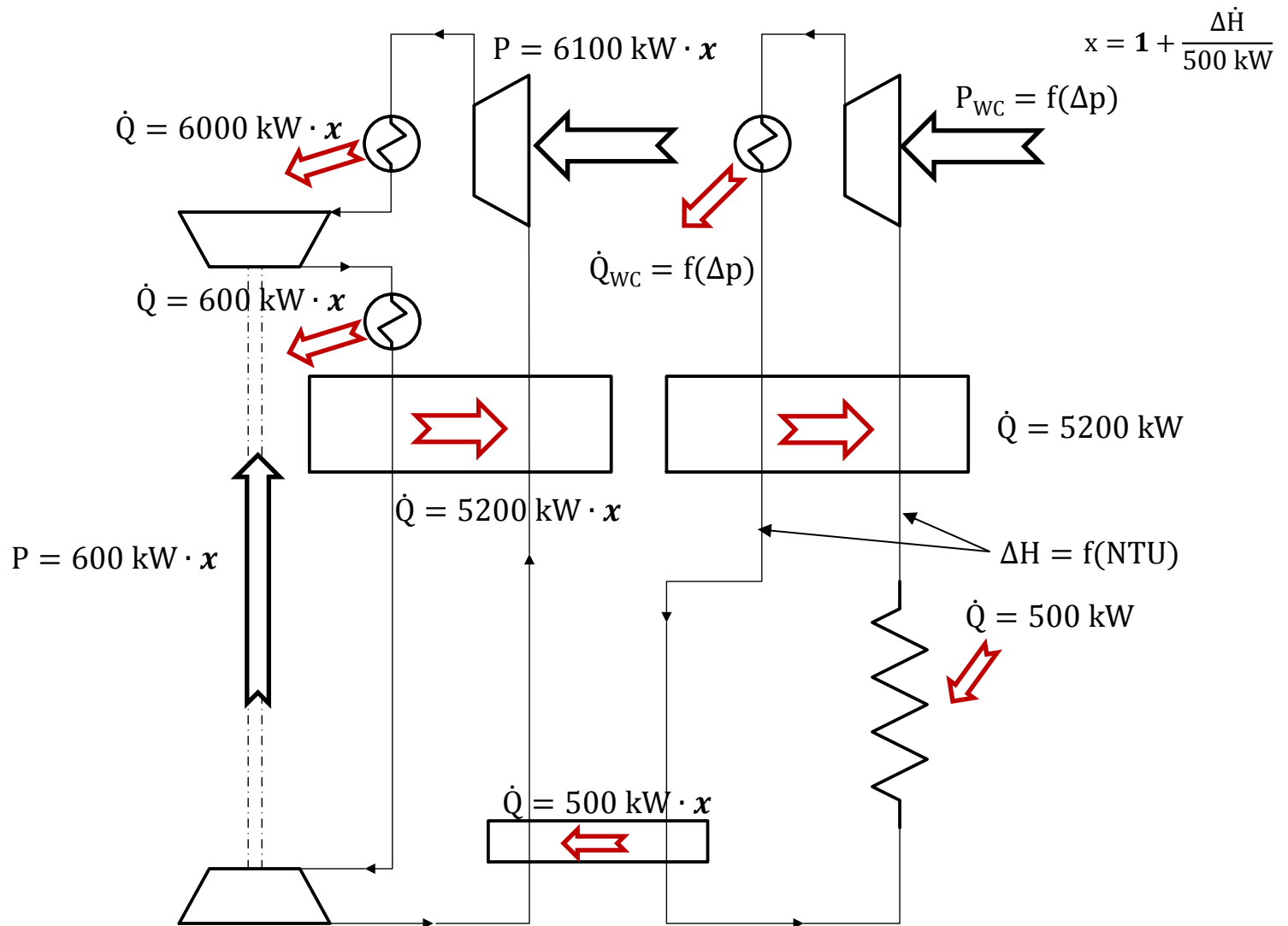


# Secondary Loop: Cold Compressor

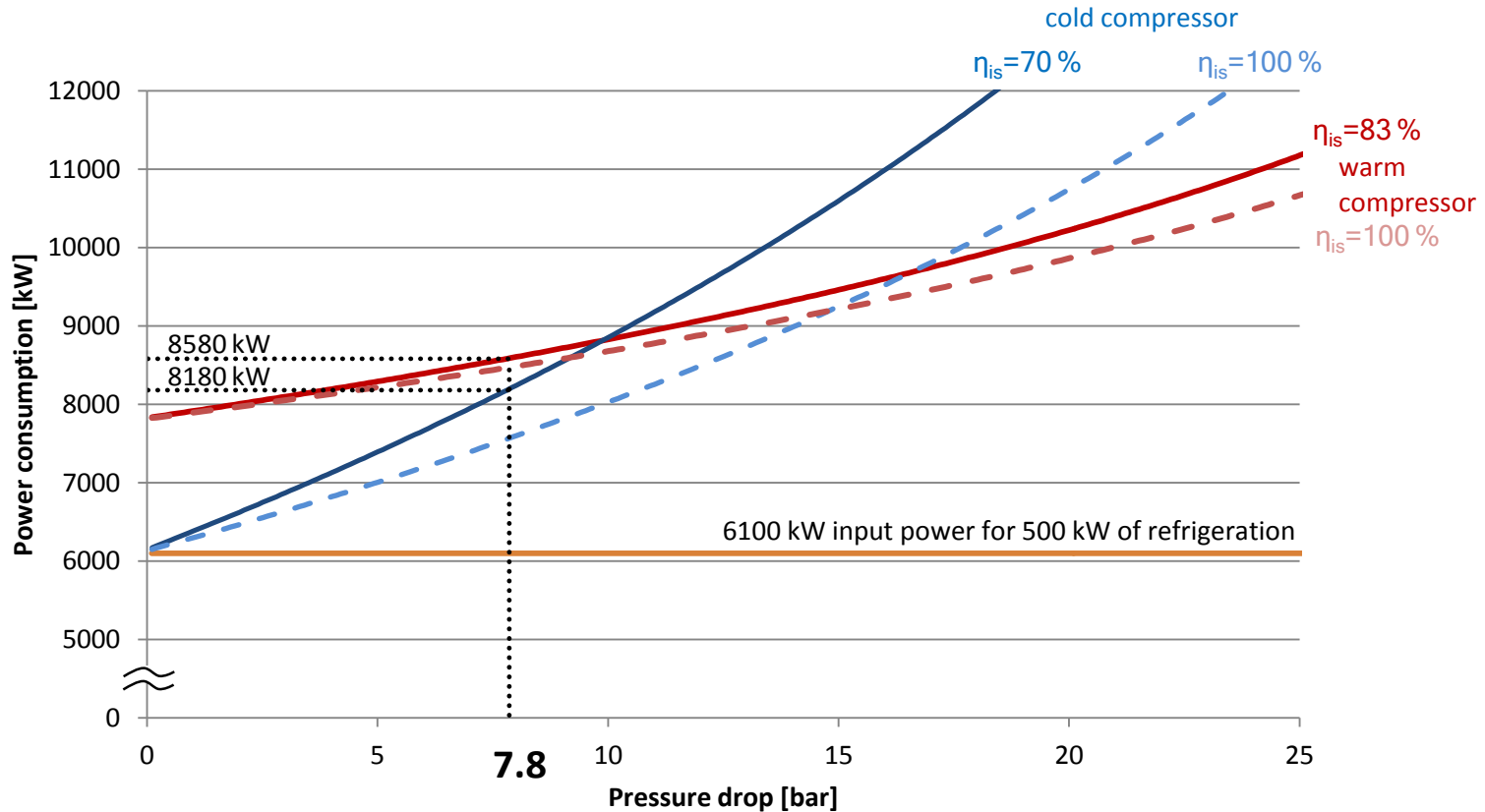


$$x = 1 + \frac{P_{CC}}{500 \text{ kW}}$$

# Secondary Loop: Warm Compressor

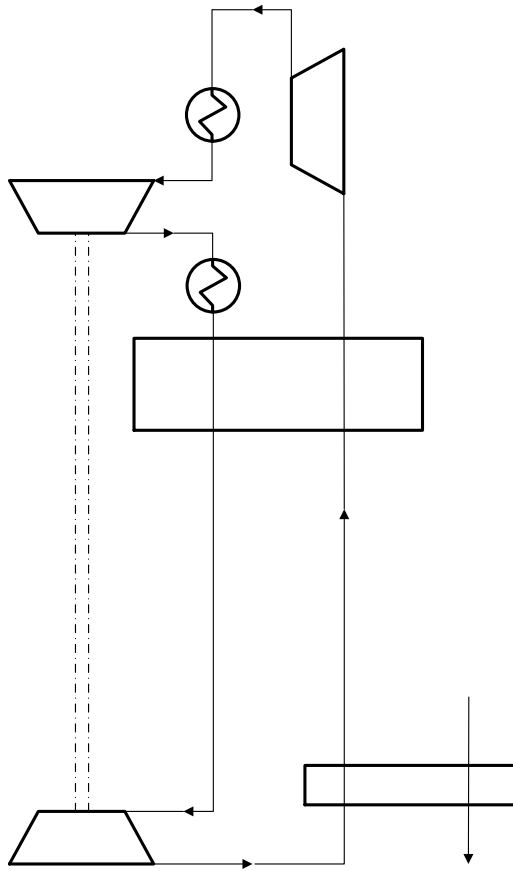


Assuming a circulation compressor outlet pressure of 50 bar and a NTU of the HX of 40:

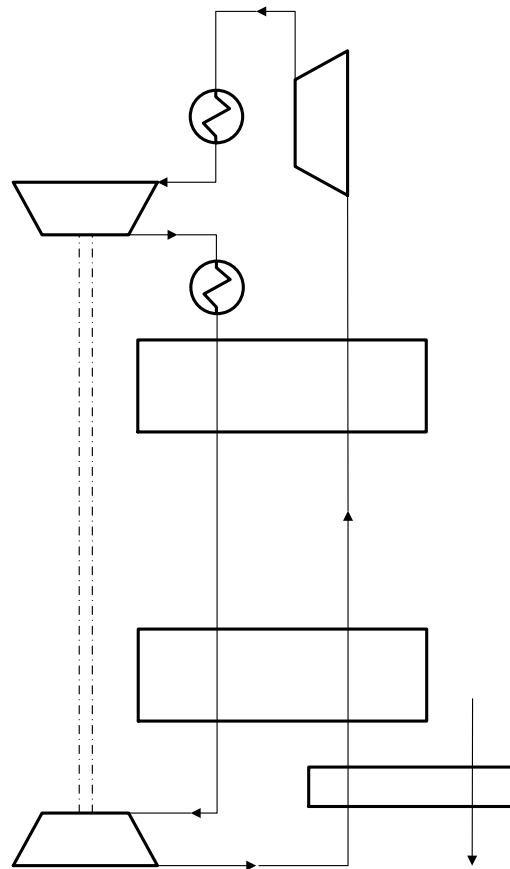


- A cold compressor is the better choice for this configuration
- One quarter of the overall power consumption is required for the circulation

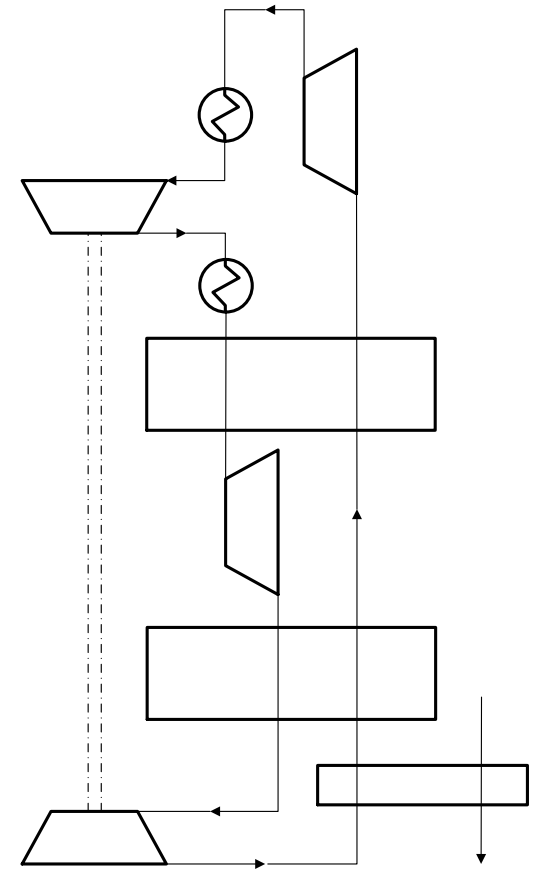
Variant A: as shown before



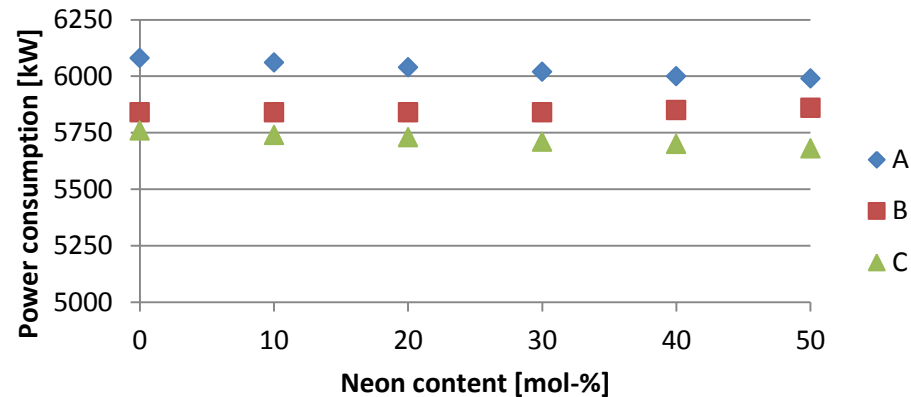
Variant B: with a double inner HX



Variant C: with a precooling turbine between the HX



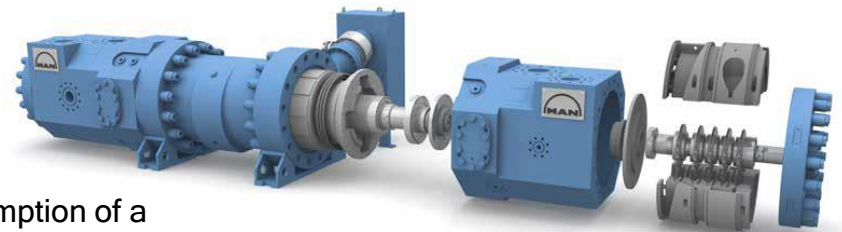
Results of the first order cycle calculation (constant NTUs, efficiencies etc.)



- Neon content has only a small impact on the cycle efficiency (neglecting influence on HX etc.)
- Small improvements are possible through better HX and/or precooling turbine
- The largest influence on the power consumption has the main compressor



Power consumption of a system with a screw compressor: **7400 kW**



Power consumption of a system with a centrifugal compressor: **5760 kW**

A highly efficient beam screen cooling can be realized with a Brayton cycle using a mixture of helium and neon instead of pure helium

A concept was worked out, which allows to compress this mixture in a centrifugal compressor with a reasonable number of stages

For comparison, three different cycles are investigated

A current lead cooling concept will be added, including a new design of the current leads

Next step:

Preliminary configuration of the main compressor:

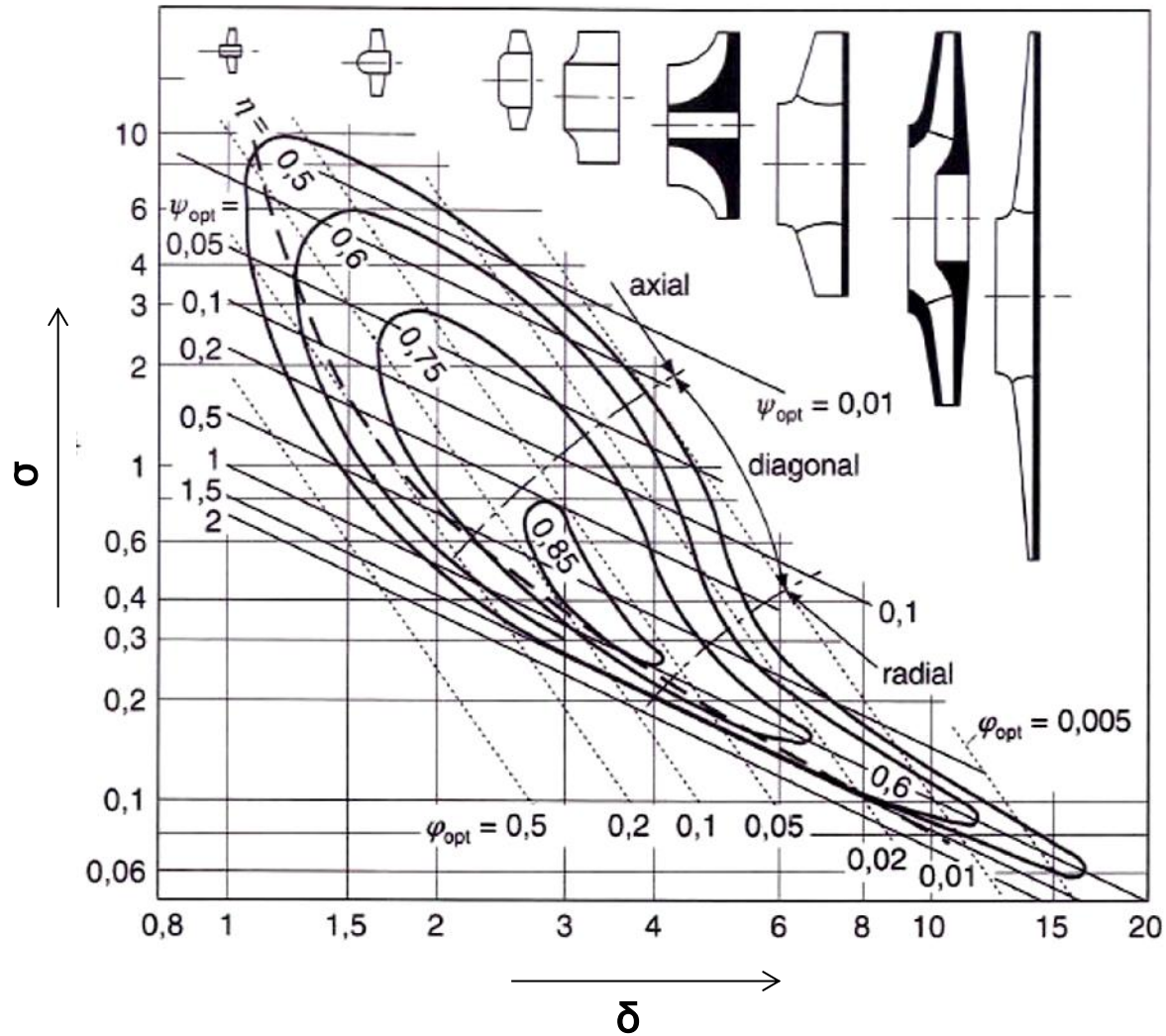
- Number of stages, stage groups, machines

**Thank you for your attention!**

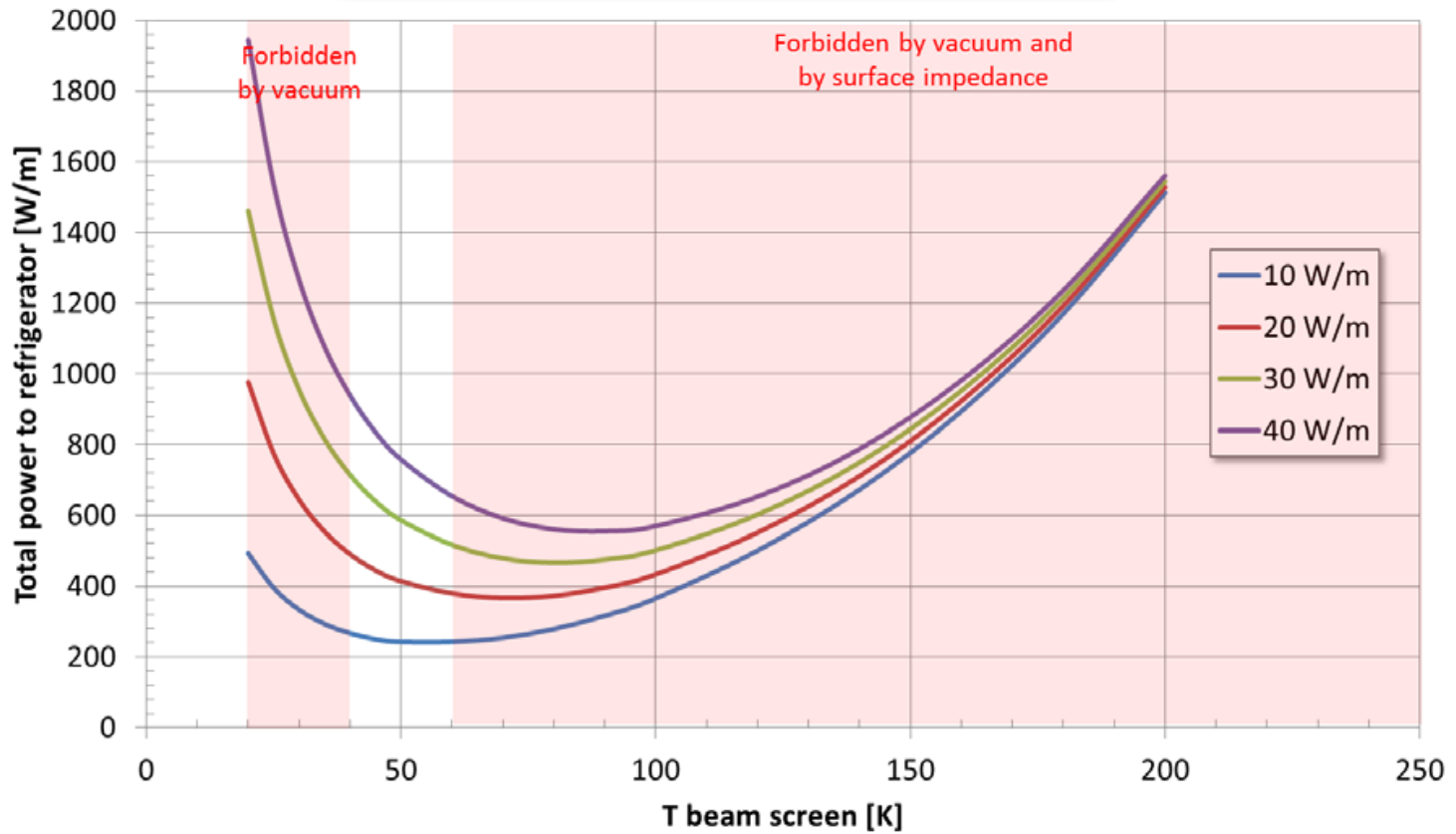




# Cordier Diagram



**Power to refrigerator vs beam screen temperature**  
**Ta = 290 K; LHC type beam screen**



# Stages needed

