

# Cooling topology options for HTS rotating electric machinery



Leslie Bromberg<sup>1</sup>, Edward Chen<sup>2</sup>, Philip C. Michael<sup>1</sup>, Seungyong Hahn<sup>3</sup>, John P. Voccio<sup>1</sup>, and Haran Karmaker<sup>2</sup>

<sup>1</sup>MIT Plasma Science and Fusion Center, Cambridge, MA, USA

<sup>2</sup>TECO-Westinghouse Motor Company, Round Rock, TX, USA

<sup>3</sup>Applied Superconductivity Center, NHMFL, Tallahassee, FL, USA

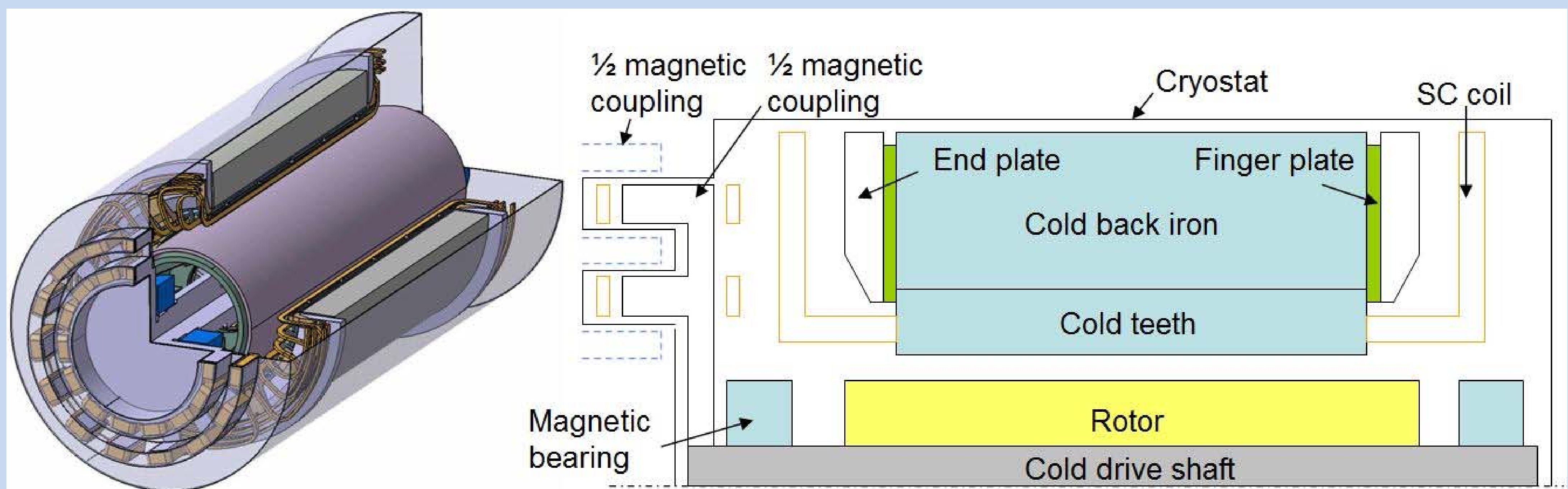
# Motivation

- High-temperature-superconductors are being developed for use in rotating electric machinery
  - Rotors for low speed, medium power (5~10 MW) wind turbines
  - Fully-superconducting high-speed machines for airpower & shipboard generation and propulsion
- Challenging cryosystem design for high-speed, fully-superconducting machines
  - High AC losses in stator
  - Rotor cooling typically requires high-speed cryogen transfer coupling

# Objective

- Examine unconventional cryostat configurations to reduce cryosystem demands
- Emphasize cooling options for high ac loss stator windings

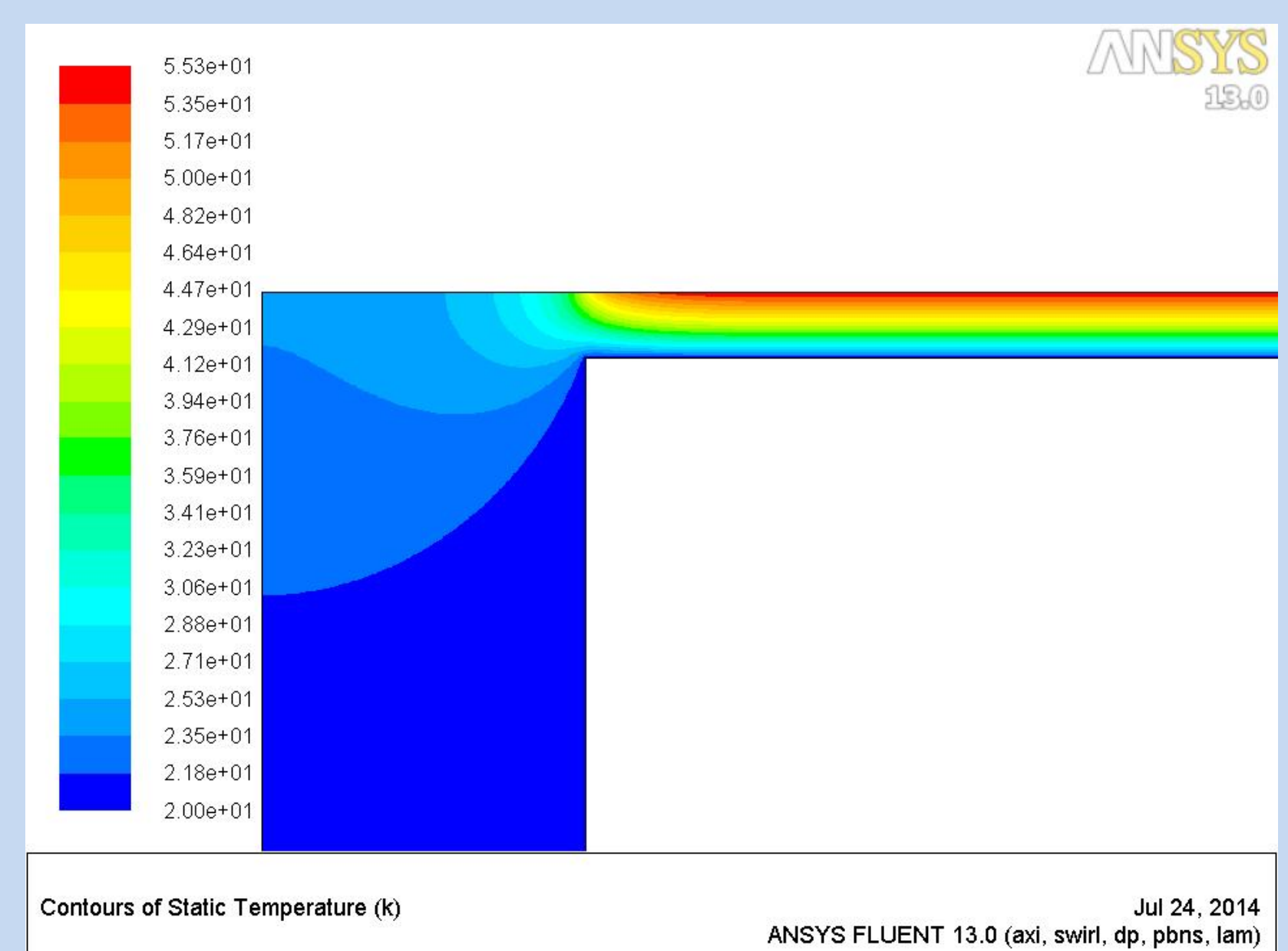
# Common cryostat configuration



Feature	Pro	Con
Active 20 K cooling of $MgB_2$ stator, convection cooling of 50~60 K ReBCO rotor	✓ Elimination of high-speed rotor cryocoupling	✗ Need to control cryostat residual gas pressure – balancing windage vs. heat transfer ✗ High heat load from cold back iron
Torque transfer through magnetic coupling and non-conducting cryostat boundary	✓ No hermetic shaft seals	✗ Large size, complex configuration, large cantilever for rated torque
Magnetic bearings inside cryostat	✓ Large reduction in rotating friction heat load	✗ The load (motor weight and torque) gets transmitted through the cryostat. ✗ Low radial stiffness may affect dynamic mechanical stability

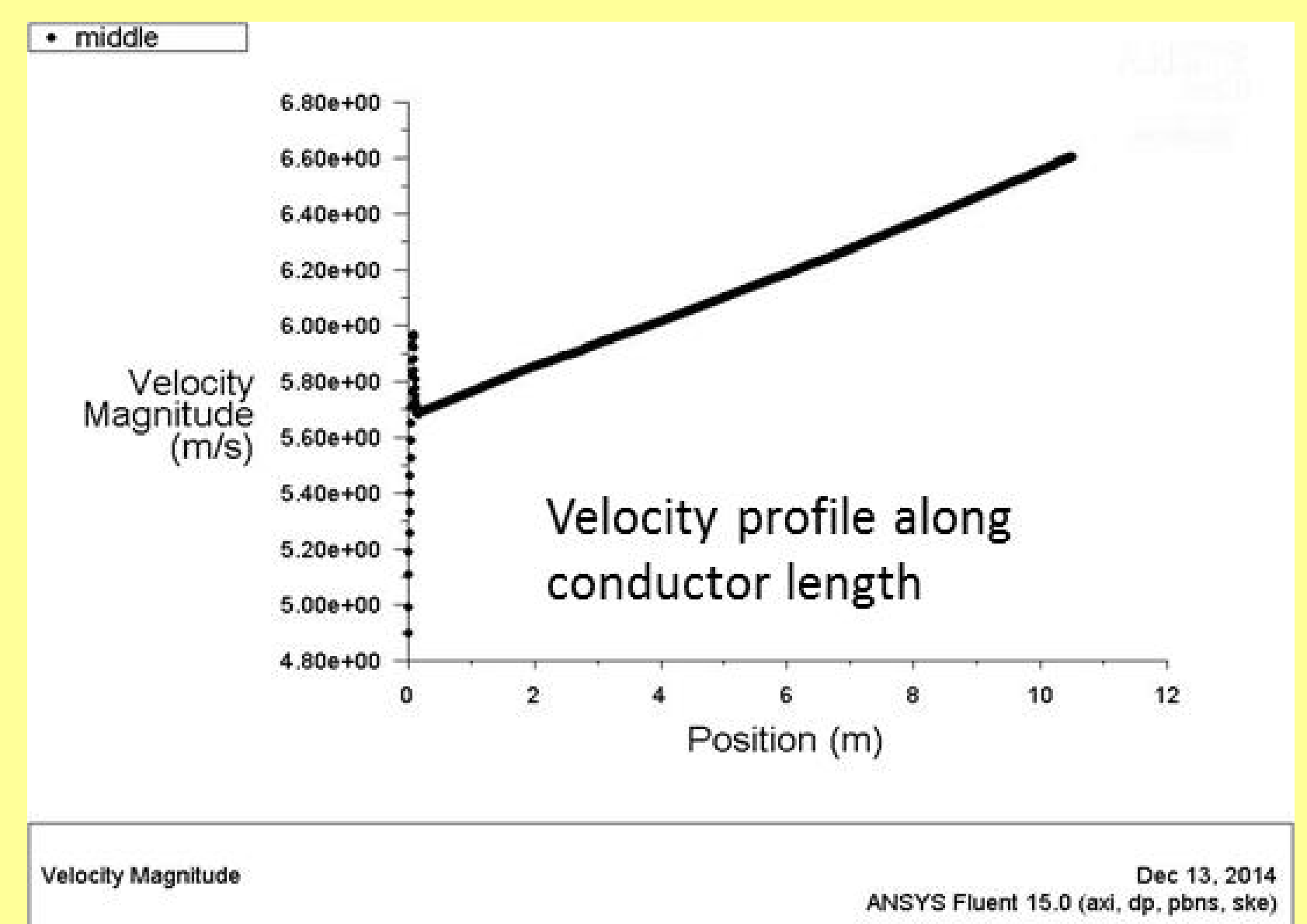
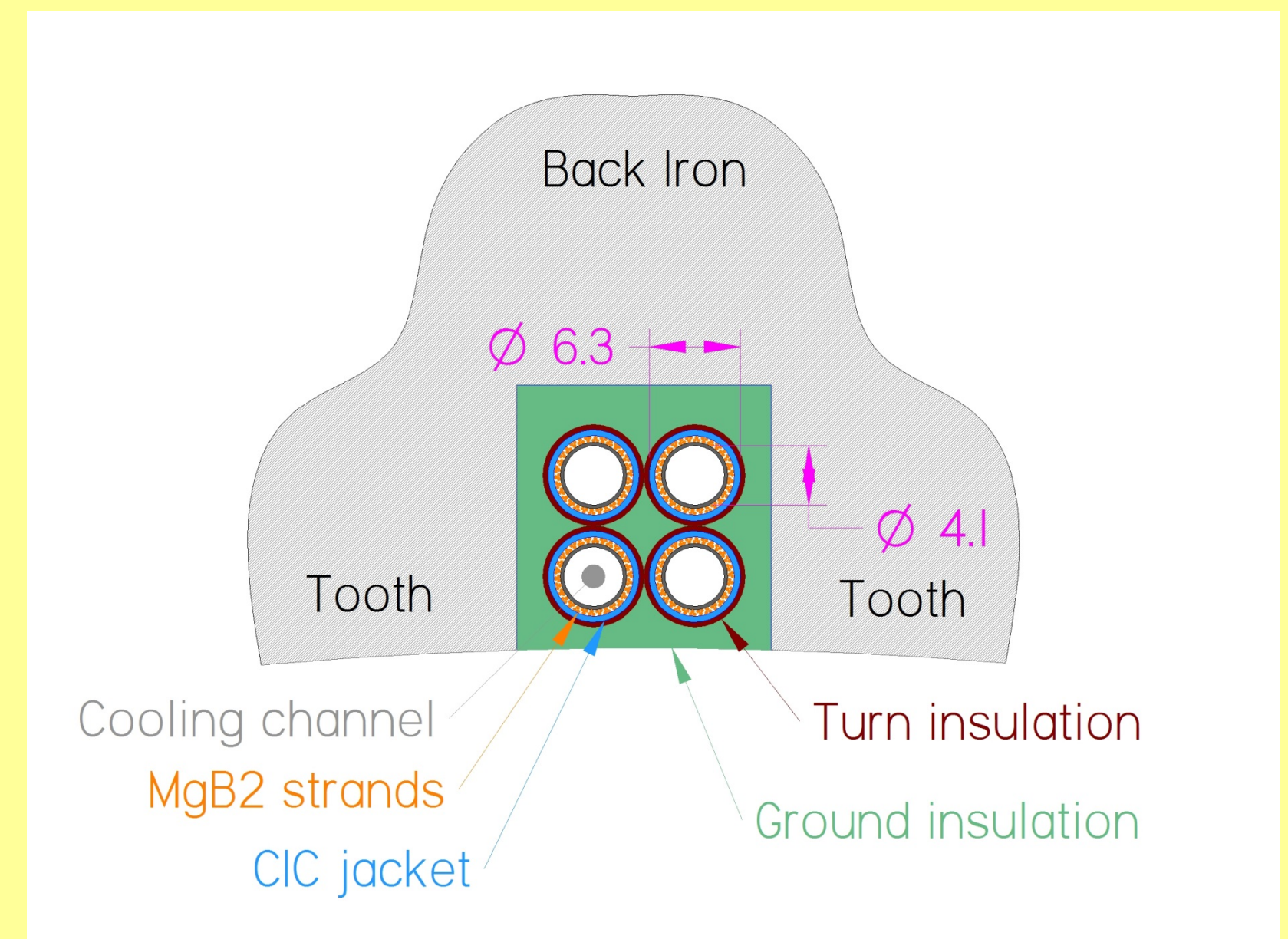
# Residual gas cooling for ReBCO-based rotor

- Assume 1 m long, 0.36 m diameter rotor
- Allow a rotor heat load of 130 W ( $\sim 100 \text{ W/m}^2$ )
- Use  $\sim 1$  mTorr  $H_2$  or He gas pressure in cryostat for cooling where, conductivity,  $k$ , viscosity,  $\nu$ , are pressure independent
- Actively cool stator to  $\sim 20$  K
- Simulate operation at 7000 rpm, with 1 cm gap
- Roughly 35 K temperature difference stator to rotor
- 0.01 N-m torque and 10 W loss to windage

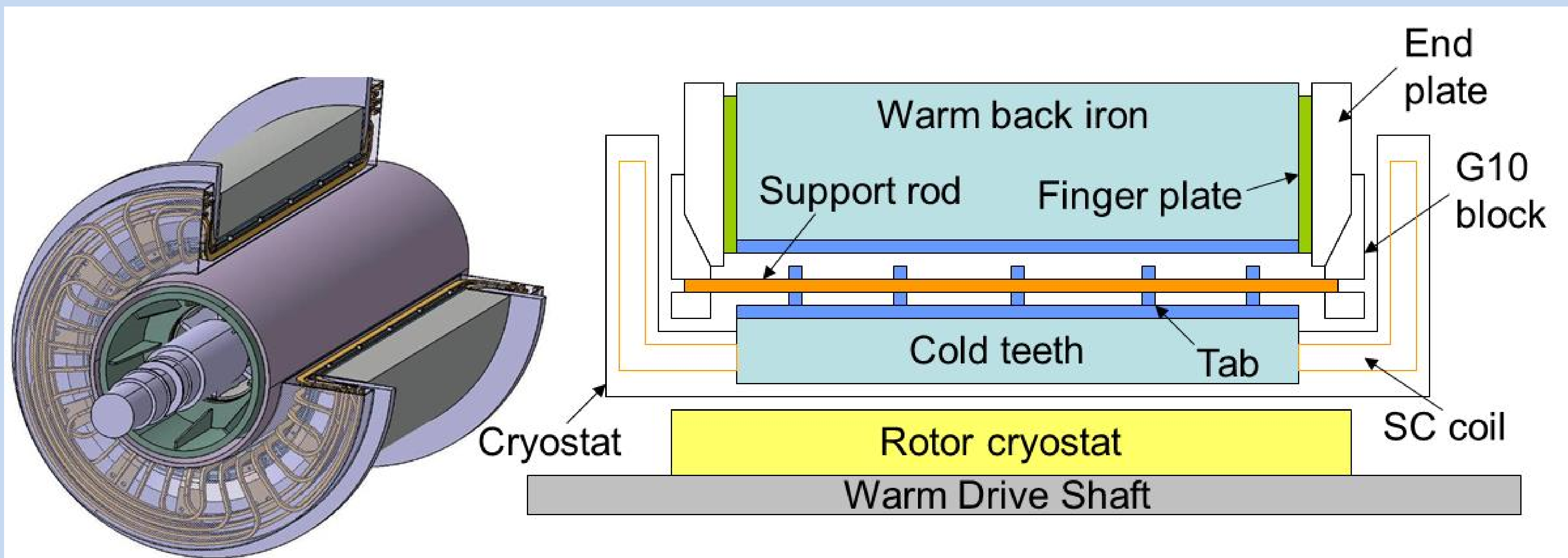


# Stator conductor and cooling design

- Conductor based on fine-filament MgB<sub>2</sub> strand
  - ~10 μm filament diameter
  - ~10 mm twist pitch
  - ~ 10<sup>-6</sup> Ohm-m matrix resistivity
- Two-channel cable-in-conduit configuration
  - Inner cooling channel diameter accommodates conductor ac losses (hysteresis, coupling, transport current)
  - 15~20 m conductor length per stator winding
  - Strand diameter and number to fit annular cable space
- Helium gas cooling
  - 20 K, 20 atm gas supplied at conductor inlet
  - ½ atm pressure drop, 3 K temperature rise at outlet
- Estimated stator losses for 14 MW, 7 krpm, 6.6 kV design
  - 1.7 ~ 3 W/m conductor ac loss
  - 1.0 ~ 1.5 W in iron teeth
  - 25~30 kW in back iron

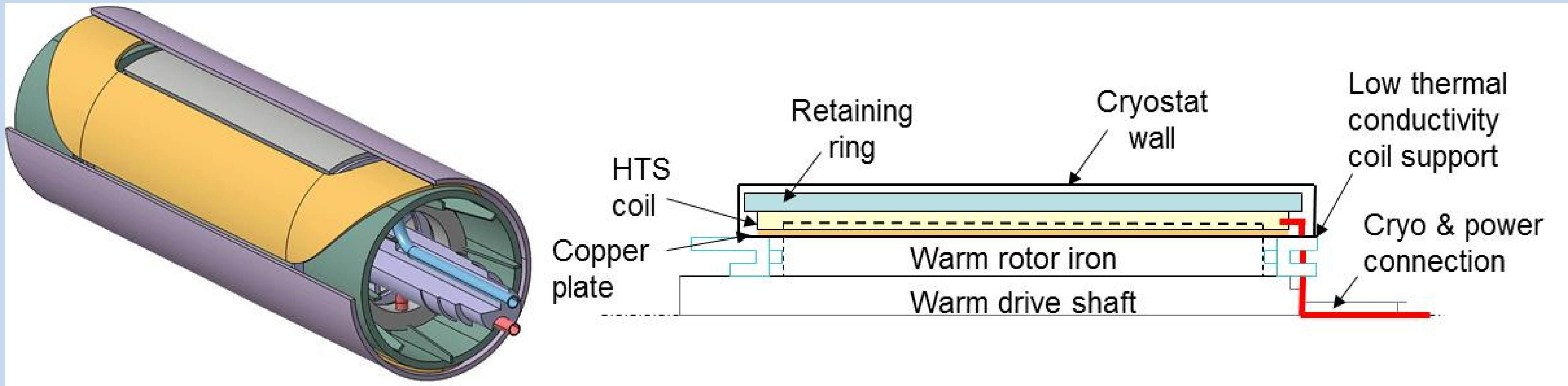


## Separate cryostat - for rotor, and entire stator



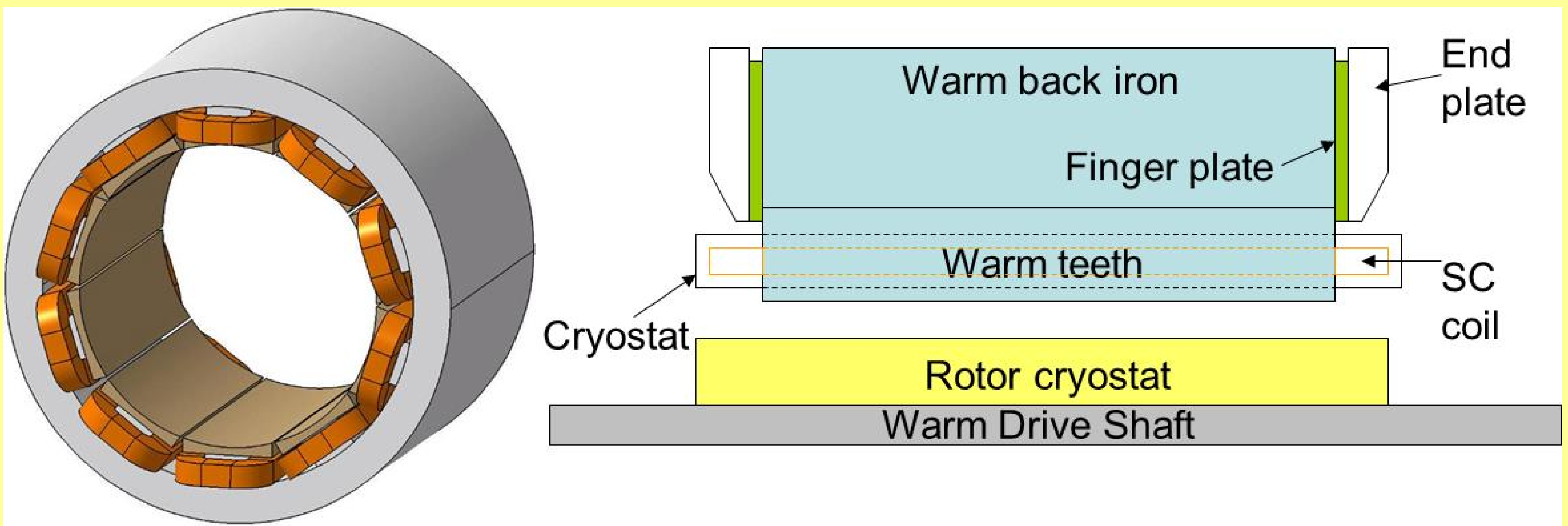
Feature	Pro	Con
Separation of teeth from back iron	✓ Removes large back iron loss from cryogenic environment	✗ Complicated cryostat design and stator assembly
Use of non-magnetic teeth between stator windings	✓ Removes iron tooth loss from cryogenic environment ✓ Provides robust mechanical support for stator windings	✗ Increases magnetic field amplitude (and hysteresis ac loss) on stator conductor
Use of conventional room temperature rotor bearings	✓ Proven technology	✗ Challenging design to minimize heat leak to rotor while maintaining high shaft stiffness for dynamic stability

# Rotor cryostat design considerations



Feature	Pro	Con
Warm rotor iron	✓ Faster cool down time	✗ Space constraint ✗ Complicated cryostat design
Cryogenic transfer coupling	✓ Increased cooling reliability compared to residual gas transfer	✗ Challenging design at this rotational speed

# Stator with individual coil cryostats



Feature	Pro	Con
Elimination of cold stator iron and cold stator teeth	✓ Significant reduction in cryogenic heat load	✗ Reduction in mechanical support for stator windings; Large number of load cycles (7 Billion/yr) may limit lifetime of internal cryostat structural/thermal elements
Introduction of cold bus between stator windings	✓ Permits replacement of individual faulty coil	✗ Complicated assembly and QA procedures
Introduction of adjacent phase winding in common slots	✓ Simplifies assembly ✓ Maximizes use of rotor field	✗ Leads to significant magnetic field peaking ✗ Cryostat design significantly limited by the space constraint between the teeth

## Summary

- Significant challenges to the development of high-speed, HTS rotating electric machinery exist
  - Relatively high conductor ac loss, compared to equivalent LTS conductors
  - Carnot efficiency gain from higher temperature (20~50 K) gas cooling, offset by low coolant density
  - Use of high-speed cryo-transfer coupling for rotor cooling remains challenging
- Additional cryostat topologies will be examined in effort to improve feasibility
  - No significant break throughs yet
  - Working to reduce magnetic field peaking for individual stator winding cryostat configurations