

# A New 2 K Superconducting Half-Wave Cavity Cryomodule for PIP-II

Zachary Conway

On Behalf of the ANL Physics Division Linac Development Group

June 29, 2015



## **Acknowledgements**

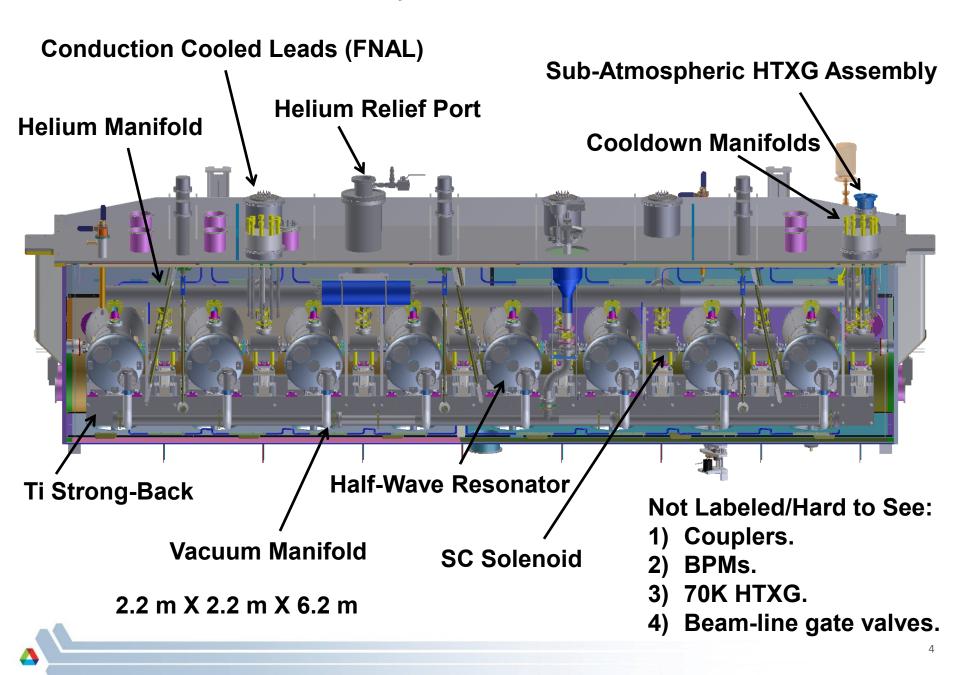
- People Working at ANL:
  - PHY: P. Ostroumov, M. Kelly, S. Gerbick, M. Kedzie, G. Zinkann, S. MacDonald, S.H. Kim and C. Hopper.
  - NE: R. Fischer, A. Barcikowski and G. Cherry.
  - HEP: T. Reid and B. Guilfolye.
  - APS: J. Fuerst and W. Jansma.
  - TechSource: K. Shepard.
  - Towson U.: N. Prins.
  - Elmhurst College: D. McWilliams.
- FNAL Cryogenics Group & Tech Division.
  - T. Nicol and M. White.
- Many Vendors:
  - Meyer Tool and Manufacturing, IL.
  - Advanced Energy Systems, NY.
  - Adron EDM, WI.
  - Ti Fab, PA.
  - Numerical Precision, IL.
  - M-1 Tool Works, IL.



#### Introduction

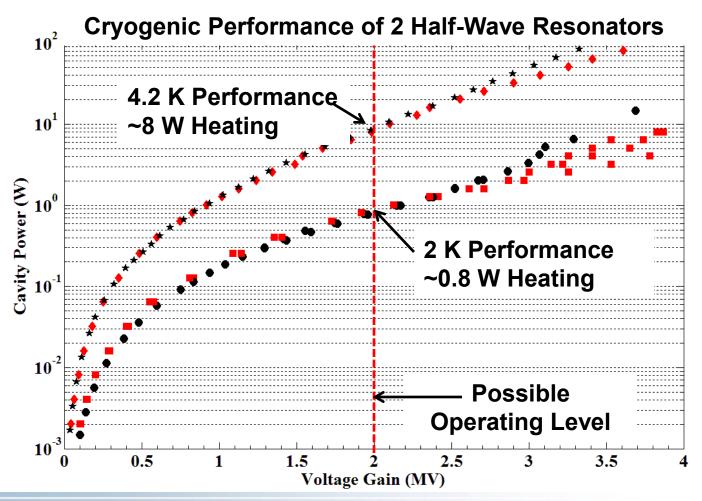
- We are building a cryogenic accelerator system which will contain accelerator cavities and magnets operating at 2 K.
- Will be the first operational 2 K cryomodule for superconducting accelerator cavities with low-beta (beta = v/c < 0.5) structures.
  - Using many techniques developed by velocity-of-light (or close to) accelerators; e.g., elliptical cell cavities.
  - Others are in development too; e.g., IFMIF, MSU-FRIB.
- Design goals for the:
  - Operate at 2 K instead of 4 K.
  - Further reduce static cryogenic loads relative to previous low-velocity cavity cryomodules.
  - Comply with DOE, ANL and FNAL safety guidelines for cryogenic, vacuum and pressure vessels.
  - Enable faster more-accurate alignment.

## Half-Wave Resonator Cryomodule

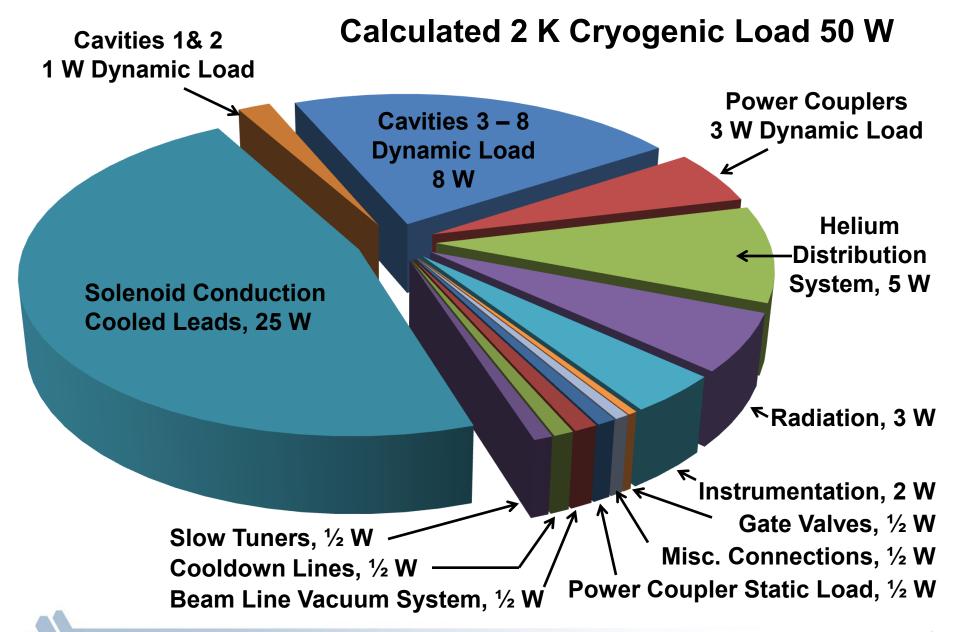


## 2 K Low-Beta Cavity Cryomodules

- Low-beta = low-frequency and losses scale as  $f^2$ . Low-beta cavities have traditionally operated at 4.2 K to save on refrigeration.
- Why operate at 2 K now?
  - The rest of the system is 2 K = Simplified Cryogenic Distribution.
  - The performance improvement justifies the extra cryogenic cost.



## Cryomodule 2 K Design Thermal Loads

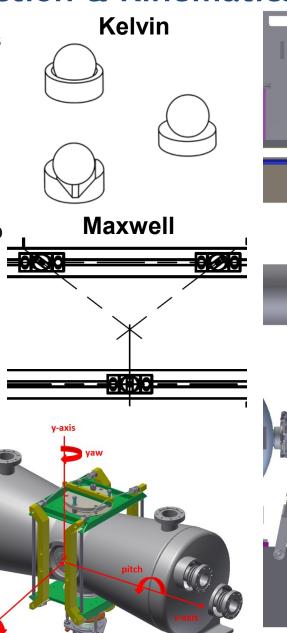


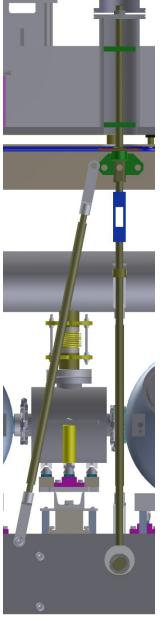
## Alignment - 1: Thermal Contraction & Kinematics

- Need to align solenoids to ±250 μm<sub>rms</sub> and ±0.1° in pitch, yaw and roll relative to the beam axis.
- Transverse shift ~ negligible.
  - We have changed from a Kelvin to a Maxwell planar kinematic coupling.
    - Maxwell geometry can be designed to be thermally invariant.
    - Kelvin geometry shifts toward fixed point.
- Vertical Shift = 650 μm up.
  - Hanger Contraction =  $+ 1,640 \mu m up$ .
  - Alignment System contraction = -990
     μm up.
  - Possible to zero.

"Design of three-grove kinematic couplings," A.H. Slocum, Precision Engineering **14**, Pg. 67 (1992).

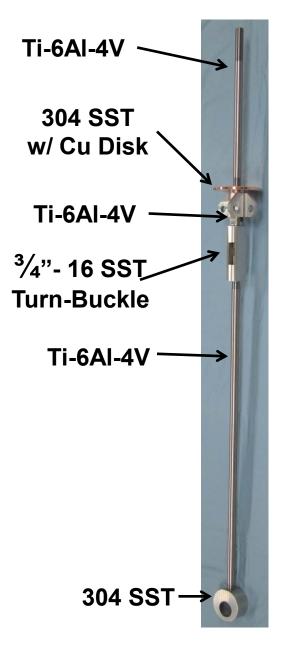
"Optimal design techniques for kinematic couplings," Precision Engineering **25**, Pg. 114 (2001).

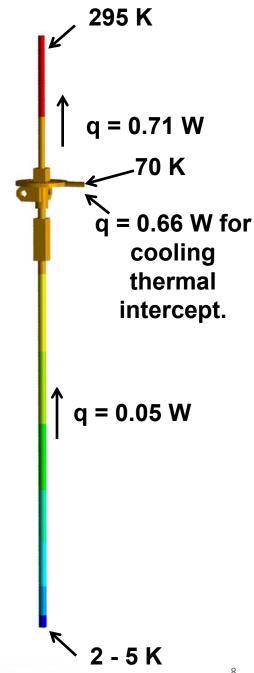




## **Cold-Mass Hangers**

- Hangers have to:
  - Support the 4 ton coldmass.
  - Allow for adjustment and alignment of the cold-mass.
  - Thermally isolate the ~2 K cold-mass from room temperature.
- We take advantage of:
  - Low thermal conductivity materials.
  - Relatively high thermal contact resistance for grease- and lubrication-free connections.

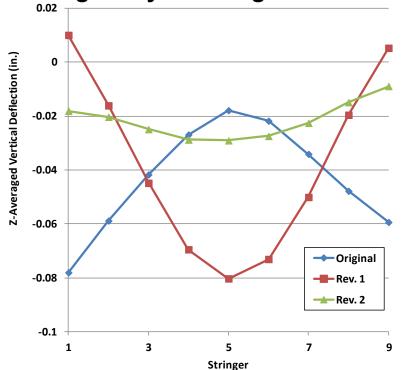




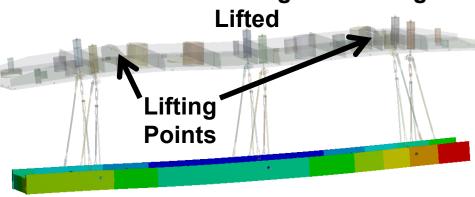
# Alignment - 2: Ti Strong-Back

- When lid is on the box the loaded strong-back rails are flat and parallel within 0.005".
- Lifting may perturb the alignment.
- Reduced lifting disturbance via design.

#### **Lifting Analysis Design Evolution**



#### Model of Lid/Strong-Back being Lifted



#### Strong-Back

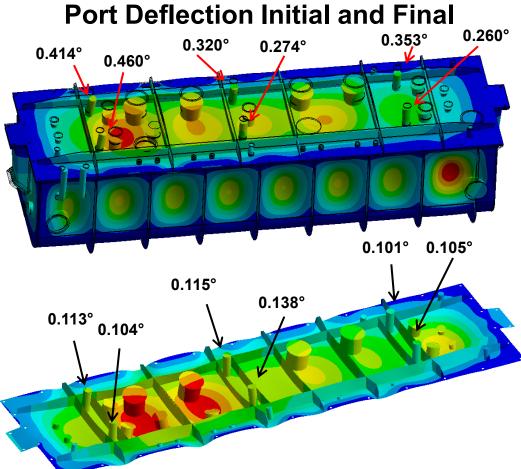




## **Design: Cavities and Cryomodules**

- Design must protect against:
  - Plastic Collapse.
  - Local Failure.
  - Buckling.
  - Failure with Cyclic Loading.
- Design must also:
  - Maintain alignment.
  - Not break penetrations.
- Not discussing solenoids.
   They receive an ASME Ustamp.

stamp.  20°C Material Pr				
Material	Young's Modulus (ksi)	Poisson's Ratio	Density (lbs/in³)	Maximum Allowable Stress (ksi)
304 Stainless Steel	29,000	0.270	0.286	20.0
Niobium	15,200	0.396	0.310	5.5



# Vessel Design: Cryomodule

Vacuum Vessel @ 14.7 psiv.

Used ASME BPVC code to demonstrate protect against:

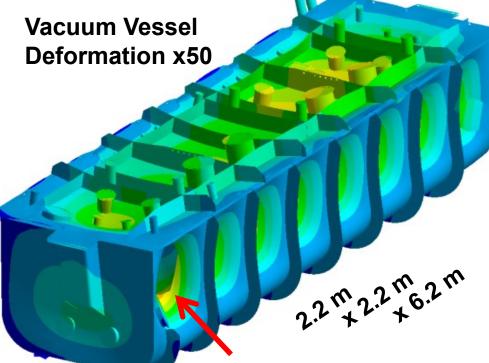
Plastic Collapse (Limit-Load).

Local Failure.

Buckling.

Ratcheting and Cyclic Loading.

Very safe vacuum vessel.



Max Deformation = 0.26"

 Magnetic shielding lines the inner surface of the vacuum vessel.

 70 K thermal shield inboard of magnetic shield.

- 32 layers MLI outside.
- 16 layers MLI inside.



## **Vessel Design: Cavities**

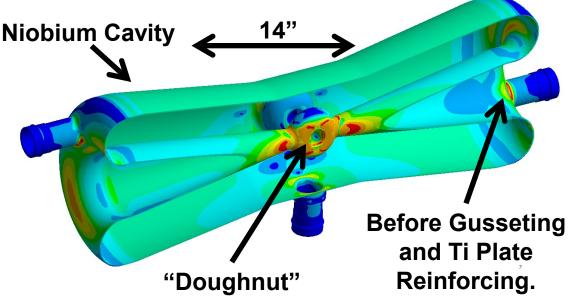
- **Design Loads:** 
  - 2 bar @ R.T.
  - 4 bar @ 2 K.

**Bracket** 

- Used the rules in the ASME **BPVC.** No code stamp.
- **Used material properties** for Nb in compliance with **FNAL** safety guidelines.

**Finished Cavity** 

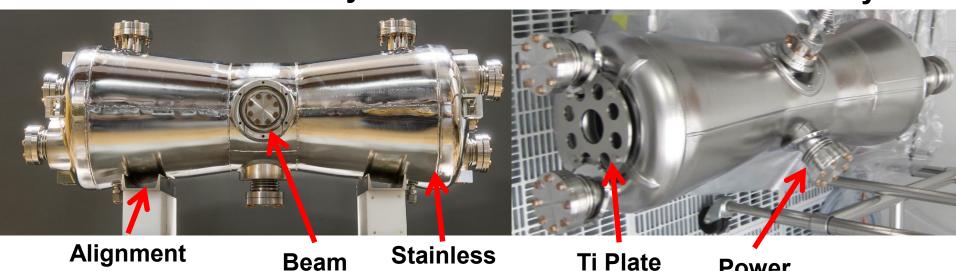
**Port** 



### **Bare Niobium Cavity**

**Power** 

**Coupler Port** 



**Steel Jacket** 

## **Summary**

- At ANL we are developing a 2K superconducting accelerator cavity cryomodule.
- Cryomodule assembly is starting now.
- Hope to test the system without cavities or solenoids late this year.

**Delivery of Cryomodule @ ANL** 



