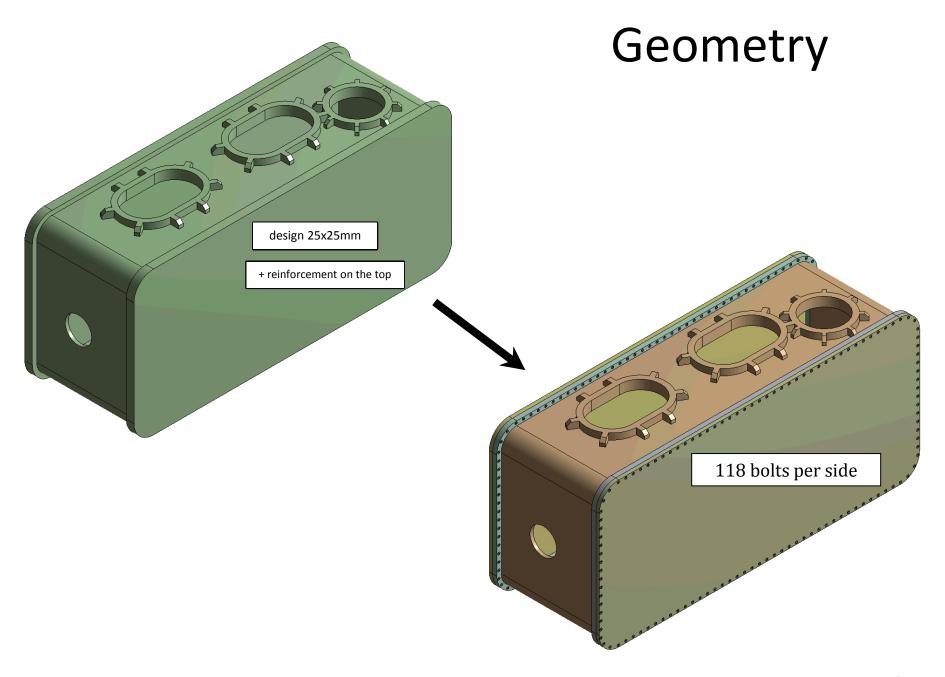


FE calculations for the vacuum vessel with bolts

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Norbert Kuder, EN/MME



Material

| Physical properties at temperature 20°C [1] Material: Stainless Steel 316LN | | | | | |
|--|--------------------|---------|-------------------|-------------------|---|
| Poisson's ratio | Elastic modulus | Density | R _{p0.2} | R _{p1.0} | Max allowable stress R _{p0.2} /1.5 |
| - | (GPa) | (kg/m³) | (MPa) | (MPa) | (MPa) |
| 0.3 | 195 | 7950 | 280 | 320 | 186 |

^[1] EN 10088-2, Stainless steels. Part 2: Technical delivery conditions for sheet/plate and strip of corrosion resisting steels for general purposes. September 2005.

D = 20d = 10M₁₀ L = 62

Bolt stiffness

$$K = \frac{\pi d^2 E}{4L} \approx 250 \text{ kN/mm}$$

Bolt modeling washer face 118 springs per side pretensioned with 35 kN each Definition Type Longitudinal Spring Behavior Both 2.5e+005 N/mm Longitudinal Stiffness Longitudinal Damping | 0. N-s/mm Preload Load Load 35000 N Suppressed No Spring Length 62. mm frictional contact $\mu = 0.5$ fixed joint face-face edge-edge

A Pressure: 0.1 MPa

B Vacuum 1: 8245. N

C Vacuum2: 16143 N

D Vacuum3: 16143 N

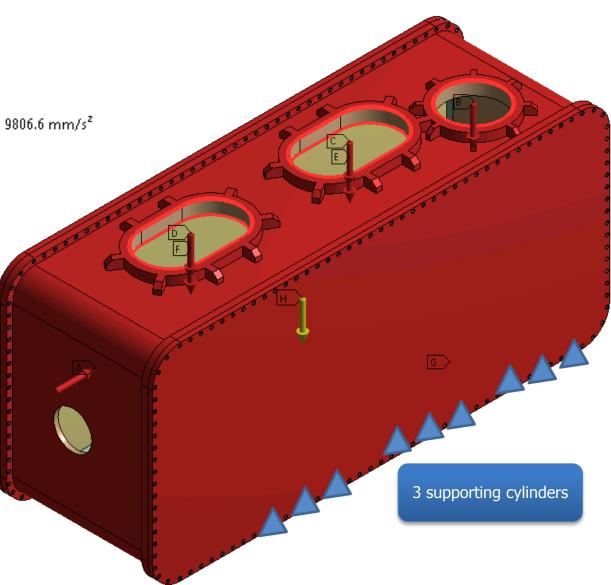
E Weight1: 1500. N

F Weight2: 1500. N

G Fixed Support

H Standard Earth Gravity: 9806.6 mm/s²

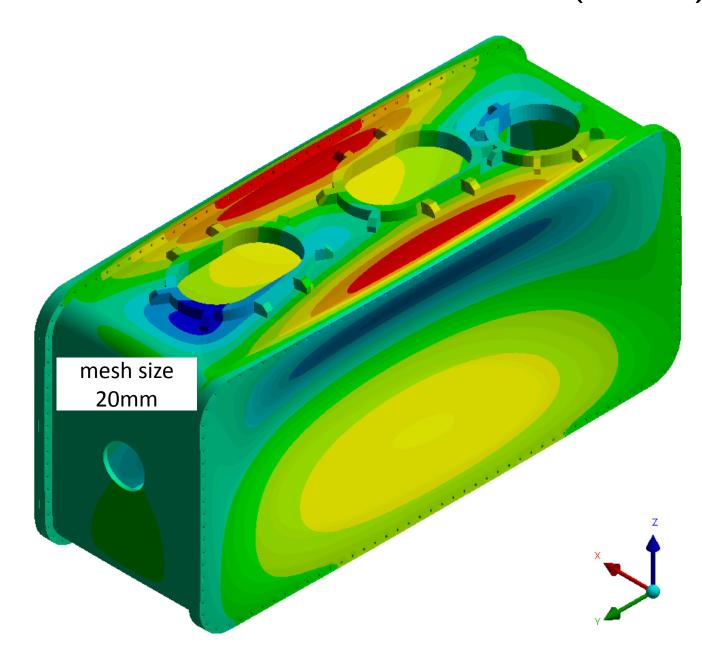
BCs & Loads



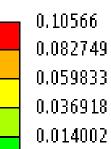
Z-directional deformation (bolts)



- -0.0086967
- -0.030157
- -0.051617
- -0.073078
- -0.094538
- -0.116
- -0.13746
- -0.15892



Z-directional deformation (w/o bolts)

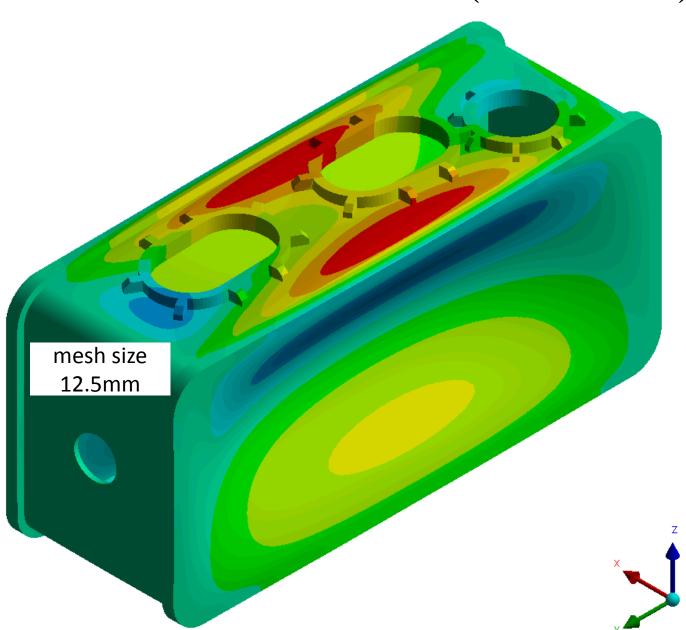


-0.008913 -0.031828

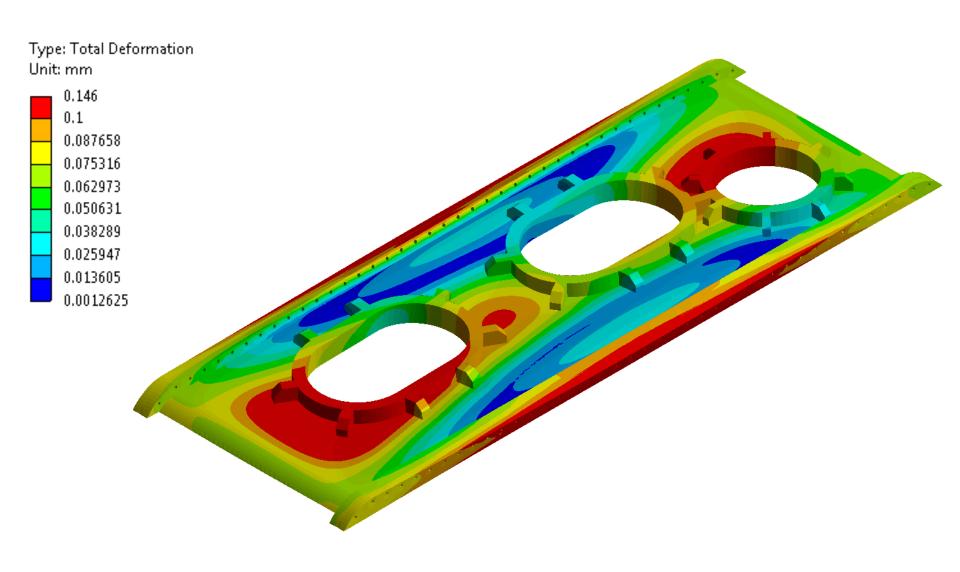
-0.054744

-0.077659

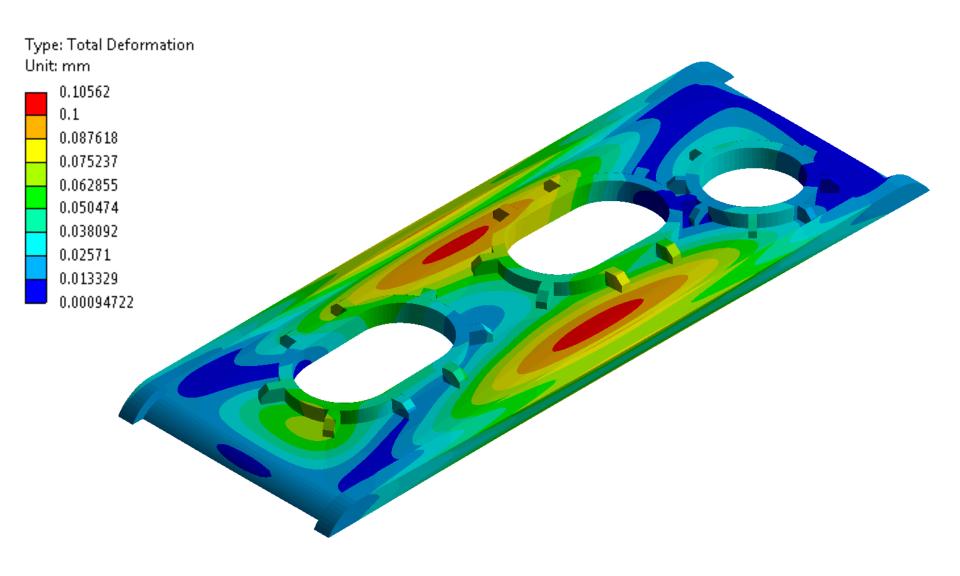
-0.10057



Total deformation (bolts)



Total deformation (w/o bolts)



Conclusions

- 1. Calculations for the vacuum vessel with bolts showed that the deformation of the top plate is greater than for the vacuum vessel without bolts.
- 2. The lateral caps do not stiffen the structure and therefore increase the deformation of the main plate.
- 3. Fixed joints and the frictional contact can highly affect the results. Modeling of solid bolts?
- 4. Mesh size of the main plate for the compared models is different. Mesh quality for the model with holes is not so good.
- 5. Calculations for the bolted model are time-consuming (over 10hrs/simulation) and for the further design development problematic.