

# ALICE ITS UPGRADE Mechanics and Cooling



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On behalf of Alice collaboration

CLIC Tracker Technology Meeting

28 May 2015

# Outline

ALICE NEW INNER TRACKER SYSTEM

OBJECTIVES

LAYOUT

STAVE MECHANICS AND COOLING

MATERIALS

PRODUCTION PROCESSES

MATERIAL BUDGET

CHARACTERIZATION

thermal, mechanical, thermoelastic

STAVE ASSEMBLY

LAYERS, BARRELS

INSTALLATION IN ALICE

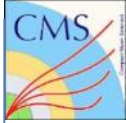


+IBL PIXEL,  
CO<sub>2</sub>, ~1.5 KW  
<BEAMPIPE DIAM, OD=50mm

NEW TRACKER



<BEAMPIPE DIAM, OD=45mm



+PIXEL REPLACEMENT,  
CO<sub>2</sub>, ~ 10 kW

NEW TRACKER



**LS2**



**LS1**

**LS3**

VELO,  
SILICON MICRO-CHANNEL,  
CO<sub>2</sub>, ~ 1.8 kW



UT TRIGGER  
(STRIPS)  
CO<sub>2</sub>, ~ 4 kW



**ALICE**

**NEW TRACKER,  
ALL SILICON,  
WATER LEAK-LESS, ~ 11kW  
<BEAMPIPE DIAM, OD=38mm**

# tracker mechanics and cooling



Towards HL-LHC: new tracker systems based on light thermomechanical structures

## Boundary conditions

- high radiation levels
- external vibrations
- temperature and humidity variations

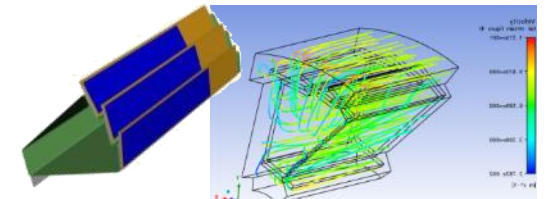
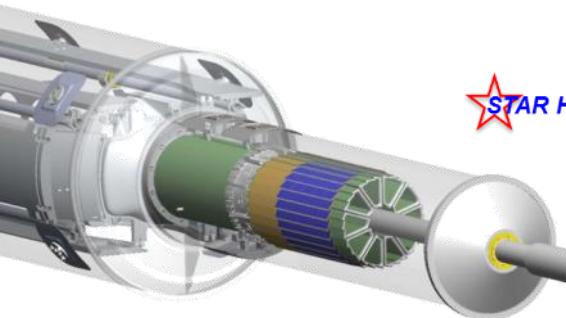
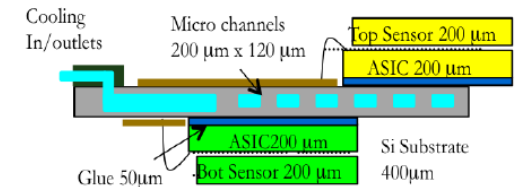
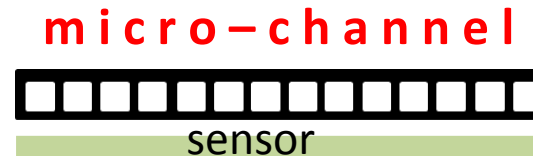
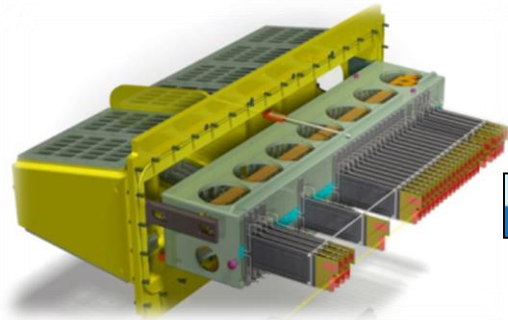
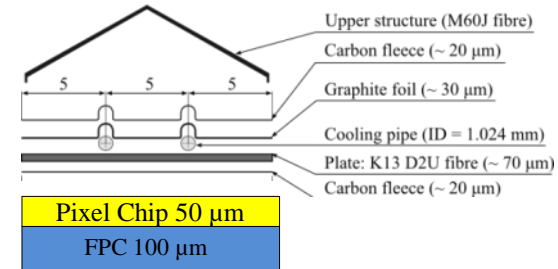
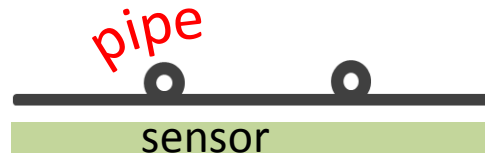
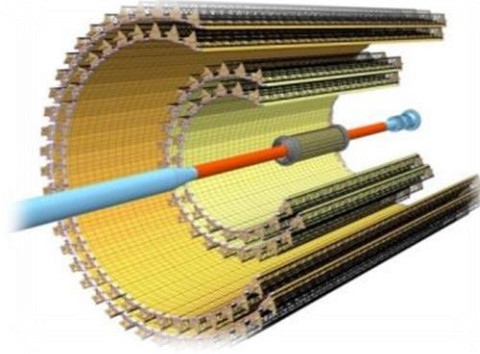
## Requirements

- support the sensor in position, with minimum mass, low  $x/X_0$
- provide high dimensional and dynamic stability
- provide thermal control and stability

**The design of the mechanical structure is inherently linked to the cooling requirements**

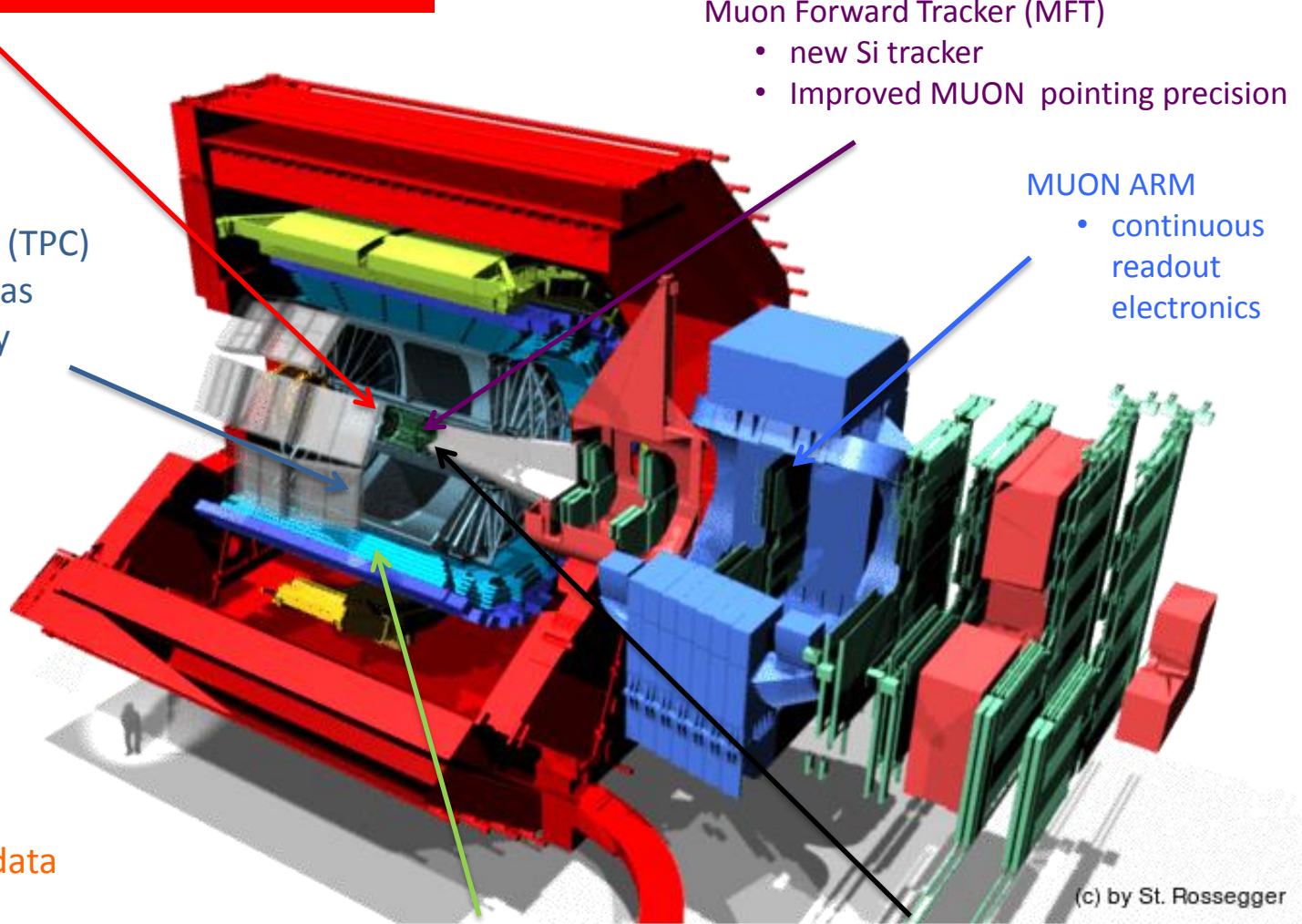
# Trackers design baselines

Different designs under study or already implemented to meet structural and thermal requirements



# ALICE upgrade

## New Inner Tracking System (ITS)



### Muon Forward Tracker (MFT)

- new Si tracker
- Improved MUON pointing precision

### MUON ARM

- continuous readout electronics

### Time Projection Chamber (TPC)

- New Micropattern gas detector technology
- continuous readout

### New Central Trigger Processor (CTP)

### Data Acquisition (DAQ)/ High Level Trigger (HLT)

- new architecture
- on line tracking & data compression
- 50kHz Pbb event rate

### TOF, TRD

- Faster readout

### New Trigger Detectors (FIT)

(c) by St. Rossegger

## 1. Improve impact parameter resolution by a factor of $\sim 3$

- Get closer to IP (position of first layer): 39mm  $\rightarrow$  23mm
- Reduce  $x/X_0/\text{layer}$ :  $\sim 1.14\%$   $\rightarrow$   $\sim 0.3\%$  (for inner layers)
- Reduce pixel size: currently  $50\mu\text{m} \times 425\mu\text{m}$   $\rightarrow$   $\text{O}(30\mu\text{m} \times 30\mu\text{m})$

## 2. Improve tracking efficiency and $p_T$ resolution at low $p_T$

- Increase granularity:
  - 6 layers  $\rightarrow$  7 layers
  - silicon drift and strips  $\rightarrow$  pixels

## 3. Fast readout

- readout Pb-Pb interactions at  $> 100$  kHz and pp interactions at  $\sim$  several  $10^5$  Hz (currently limited at 1kHz with full ITS)

## 4. Fast insertion/removal for yearly maintenance

- possibility to replace non functioning detector modules during yearly shutdown

Install detector during LHCC LS2 (2018-19)

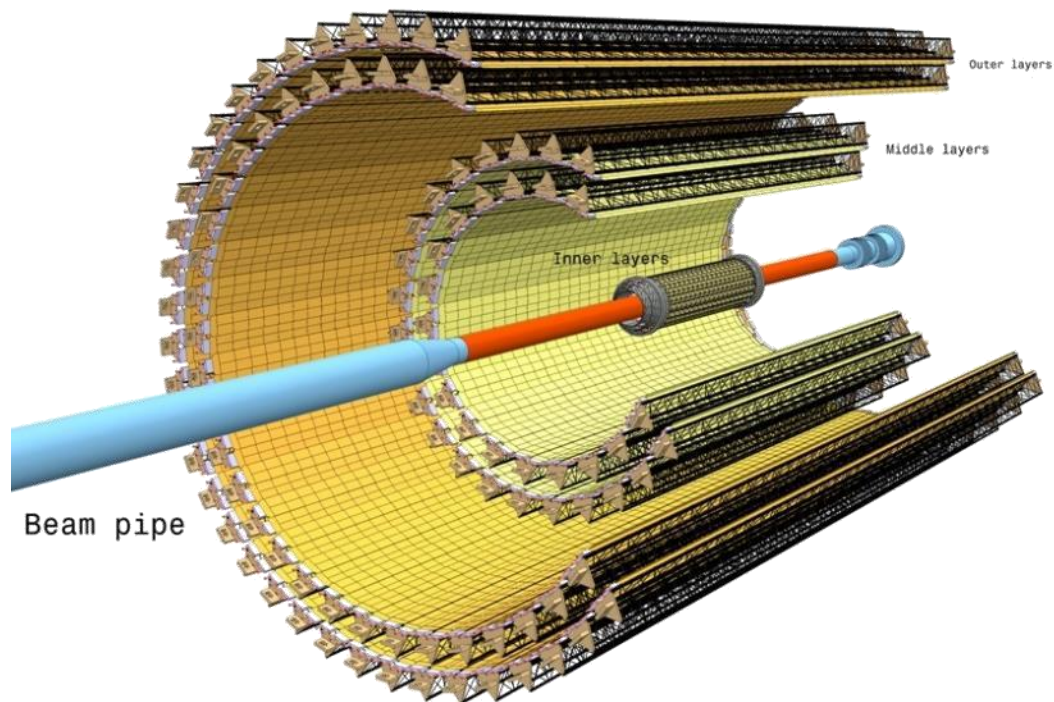


CERN-LHCC-2013-24

# ITS layout



# ITS layout



Beam pipe

**3 Inner Barrel layers (IB)**

**4 Outer Barrel layers (OB)**

Material /layer : 0.3%  $X_0$  (IB), 1%  $X_0$  (OB)

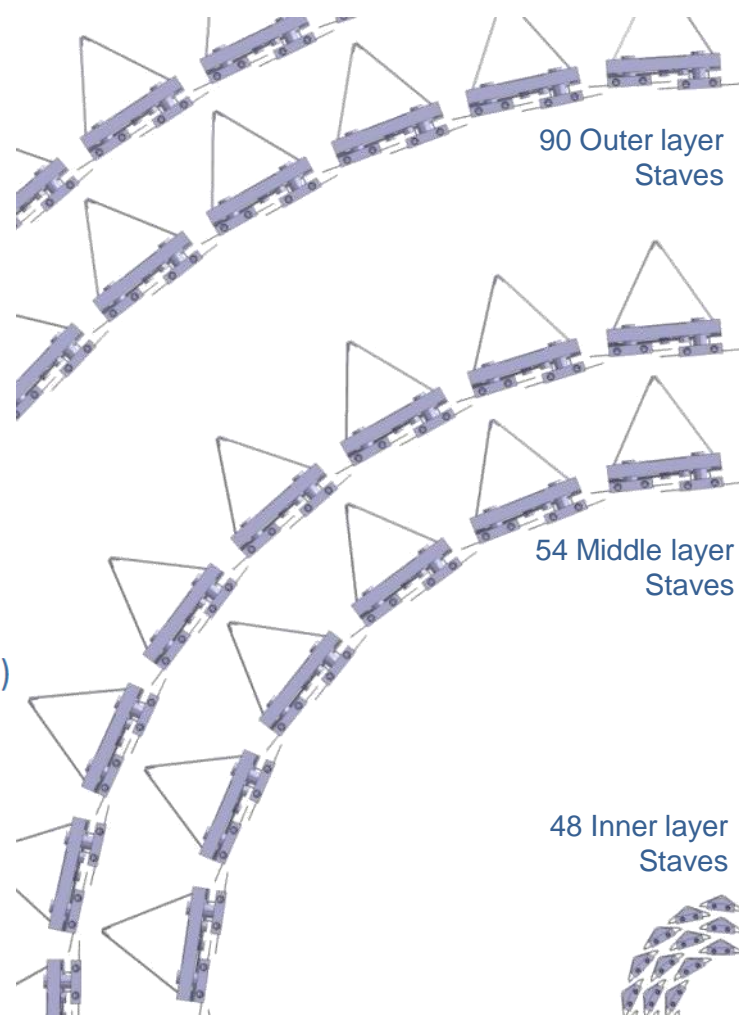
12.5 G-pixel camera ( $\sim 10\text{m}^2$ )

7-layer barrel geometry based on MAPS

r coverage: 23 – 400 mm

$\eta$  coverage:  $|\eta| \leq 1.22$

for tracks from 90% most luminous region



90 Outer layer Staves

54 Middle layer Staves

48 Inner layer Staves

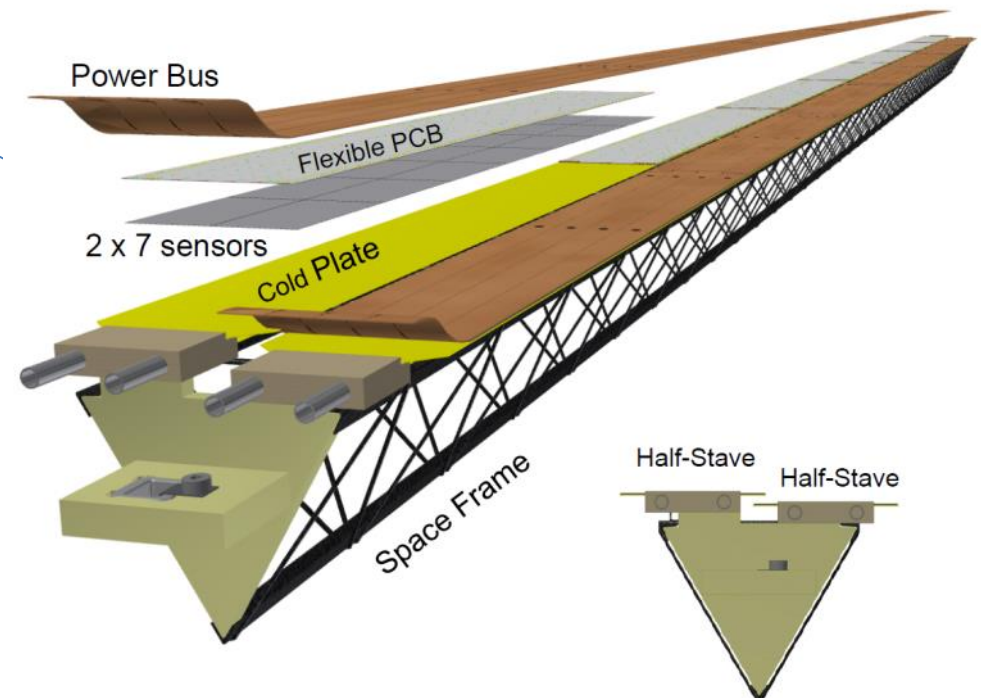
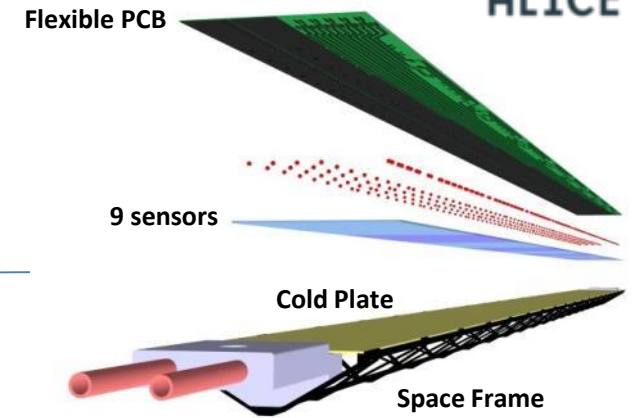
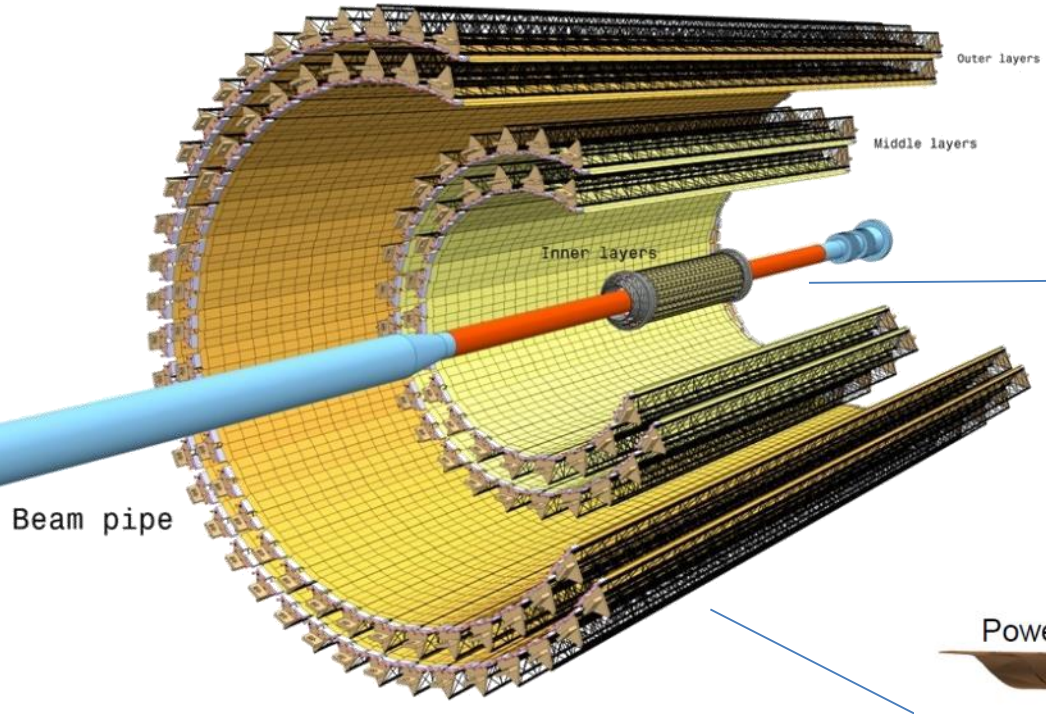
	Inner Barrel	Outer Barrel
--	--------------	--------------

TID radiation hardness (*)	2700 krad	100 krad
NIEL radiation hardness (*)	$1.7 \times 10^{13}$ 1MeV $n_{\text{eq}}/\text{cm}^2$	$10^{12}$ 1MeV $n_{\text{eq}}/\text{cm}^2$

(\*) 10 x radiation load integrated over approved programme ( $\sim 6$  years of operation)



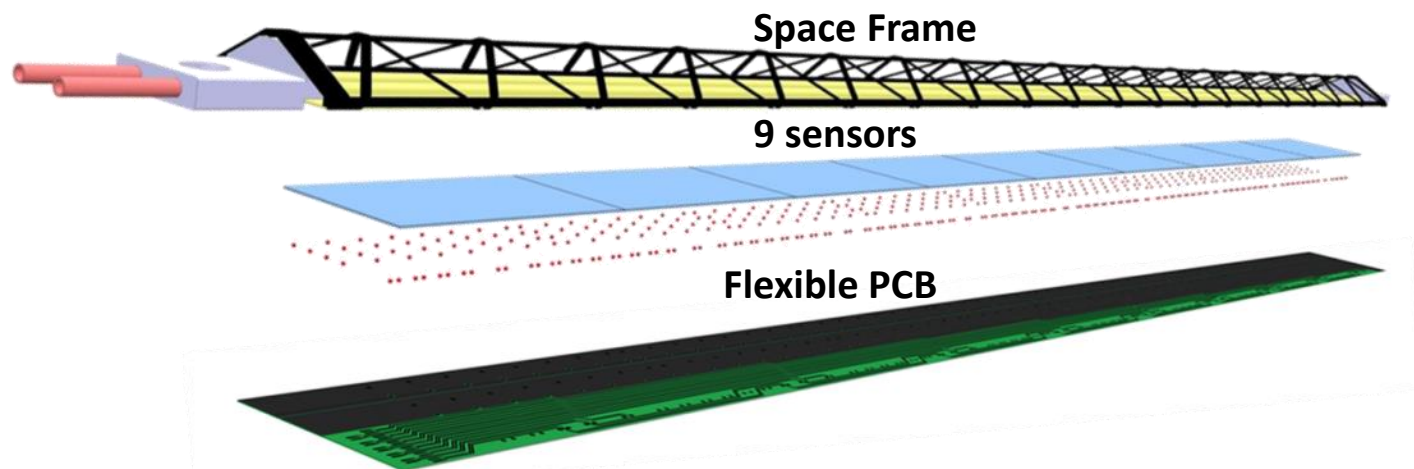
# ITS staves



## The ITS Staves length

- 290mm Inner Layers,
- 900mm Middle Layers,
- 1500mm Outer Layers

# ITS Inner Barrel (IB) staves



<Radius> (mm): 23,31,39

Nr. of staves: 12, 16, 20

Nr. of chips/layer: 108, 144, 180

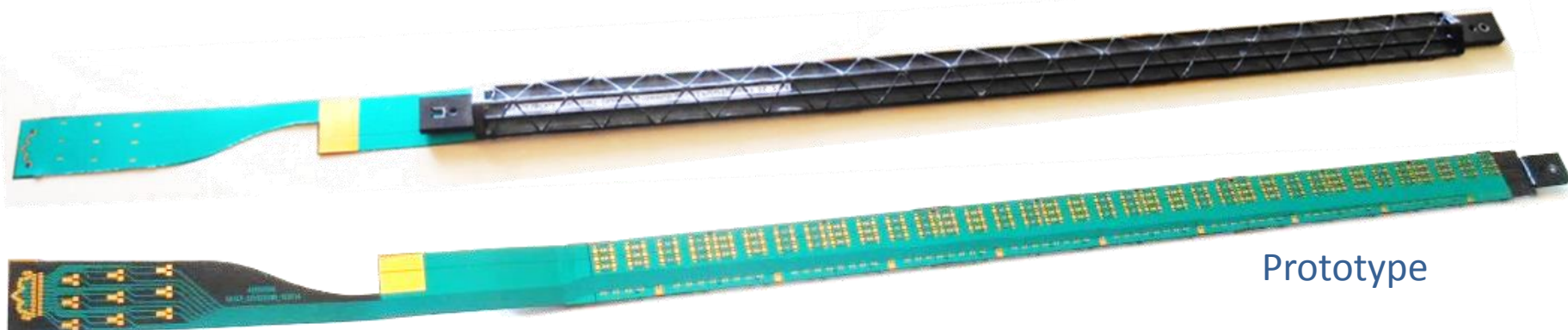
Power density: < 100 mW/cm<sup>2</sup>

Length in z (mm): 290

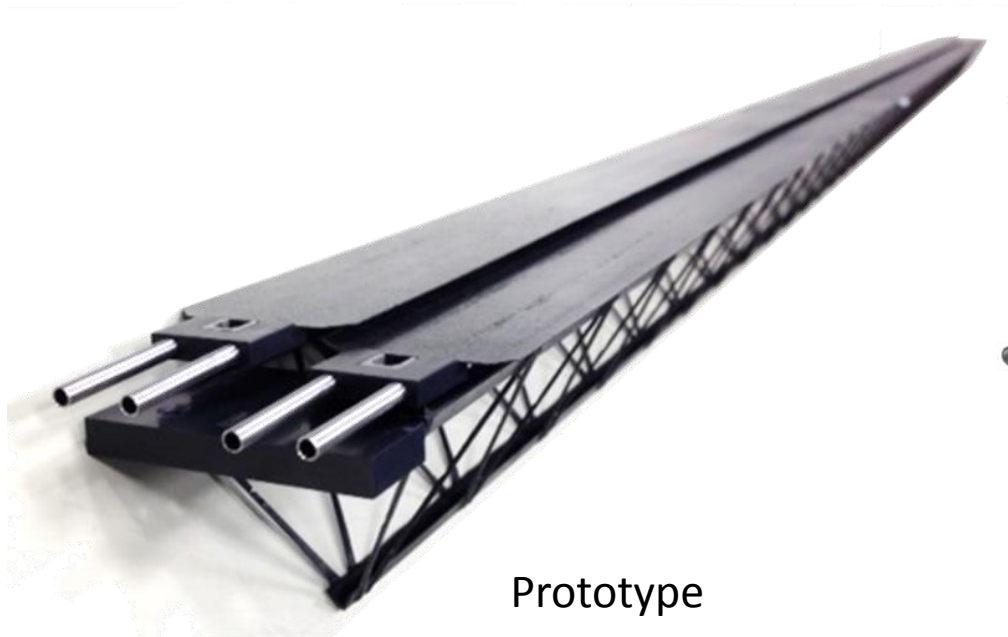
Nr. of chips/stave: 9

Material thickness: ~ 0.3% X<sub>0</sub>

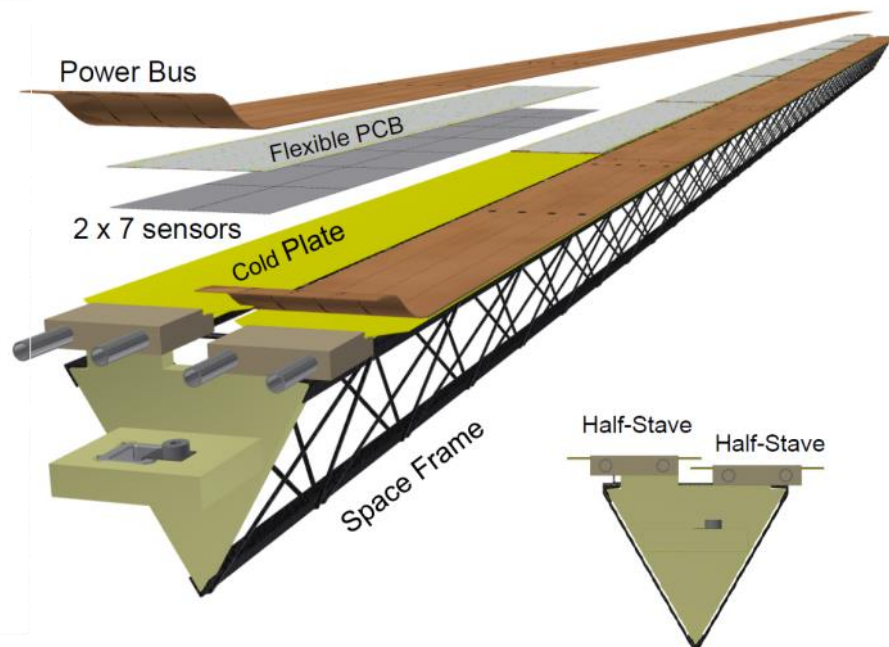
Throughput (@100kHz): < 80 Mb/s × cm<sup>-2</sup>



# ITS Outer Barrel (OB) staves



Prototype



## Outer Barrel (OB)

<radius> (mm): 194, 247, 353, 405

Nr. staves: 24, 30, 42, 48

Nr. Chips/layer: 6048 (ML), 17740(OL)

Power density < 100 mW / cm<sup>2</sup>

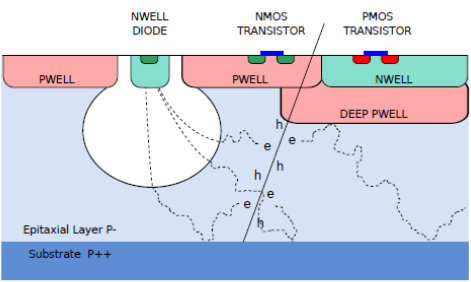
Length (mm): 900 (ML), 1500 (OL)

Nr. modules/stave: 4 (ML), 7 (OL)

Material thickness: ~ 1% X<sub>0</sub>

Throughput (@100kHz): < 3Mb/s × cm<sup>-2</sup>

# Pixel chip

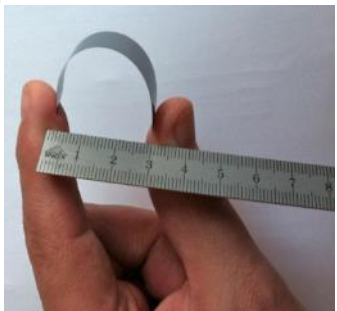


Schematic cross-section of CMOS pixel sensor (ALICE ITS Upgrade TDR)

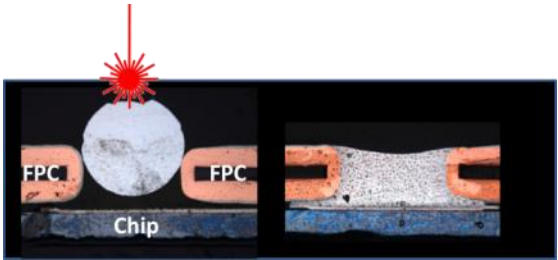
## Schematic cross section of a Monolithic Active Pixel Sensor

### Pixel Chip: CMOS TowerJazz 0.18 $\mu\text{m}$

- Chip size: 15 mm x 30 mm
- Pixel pitch  $\sim 30 \mu\text{m}$
- Si thickness: 50  $\mu\text{m}$
- Spatial resolution  $\sim 5 \mu\text{m}$
- Power density  $< 100 \text{ mW/cm}^2$



### 9 chips



**Laser soldering:** Interconnect of Pixel chip on flexible printed circuit

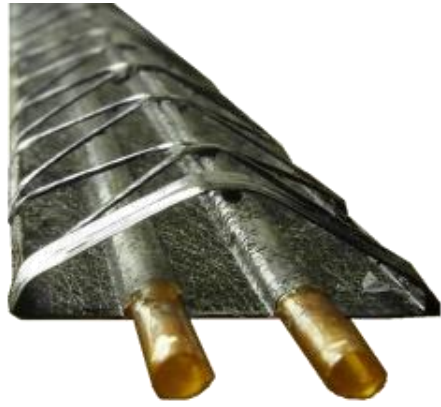
FPC

*Layout for Inner Barrel*

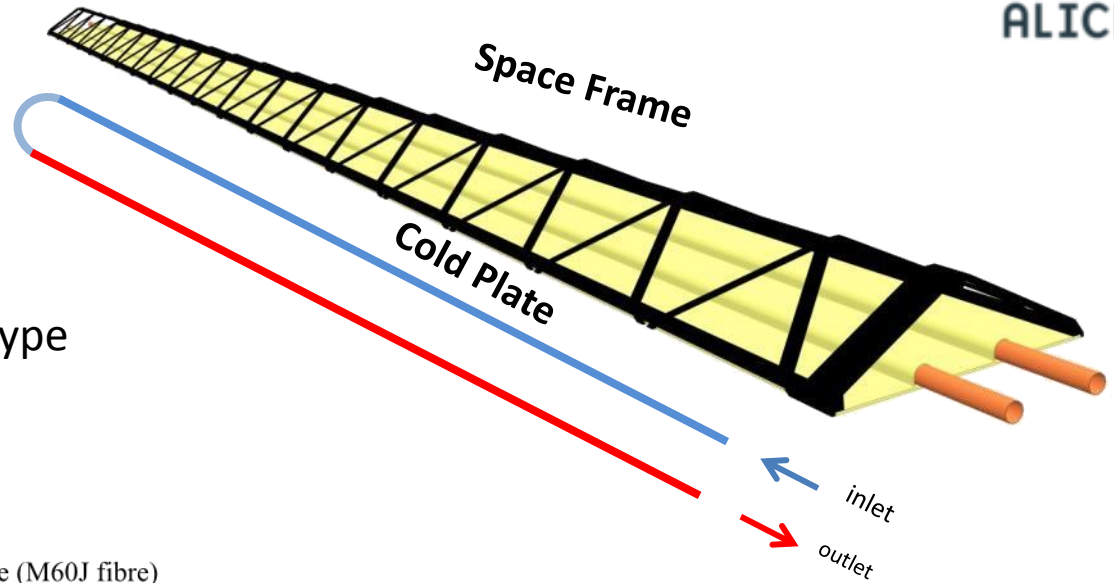


# Flex Printed Circuit FPC

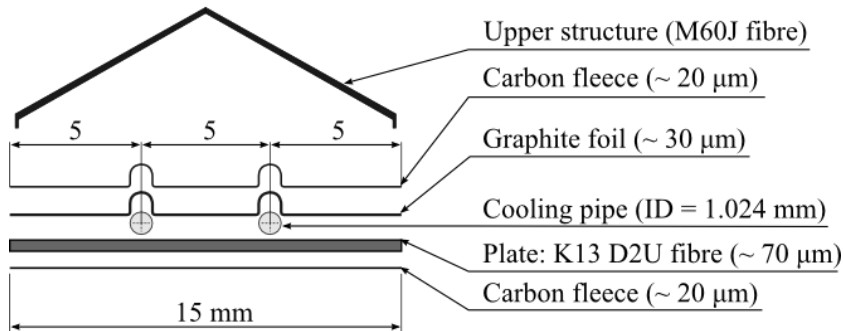
# IB stave mechanics and cooling



Prototype



Transversal section:



Coolant Single-phase  $\text{H}_2\text{O}$  leak-less  
Pixel operational temperature  $< 30^\circ\text{C}$   
Pixel max temperature non-uniformity  $< 5^\circ\text{C}$   
Chip Power dissipation  $< 100\text{mW}/\text{cm}^2$



Prototype

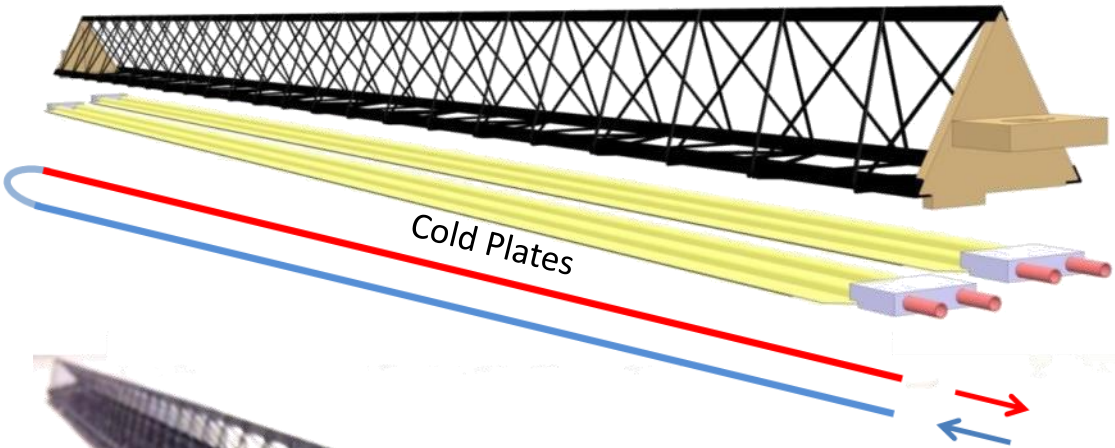
290mm length, 1.5gram weight

# OB stave mechanics and cooling

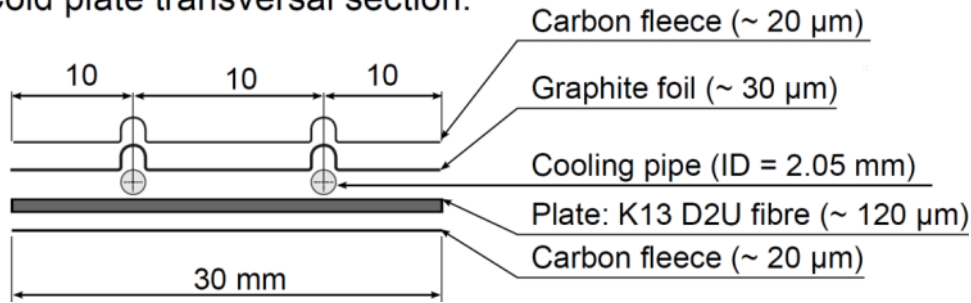
Cold Plate



Spaceframe

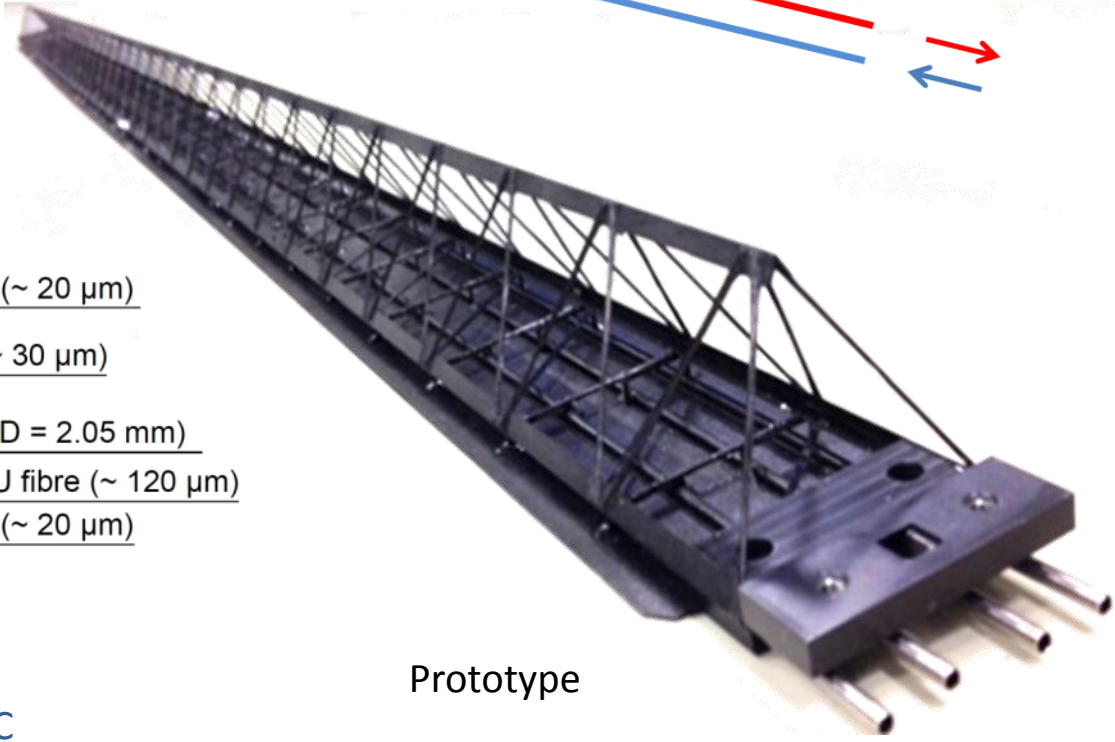


Cold plate transversal section:



- Coolant Single-phase  $\text{H}_2\text{O}$  leak-less
- Pixel operational temperature < 30°C
- Pixel max temperature non-uniformity < 5°C
- Chip Power dissipation < 100mW/cm<sup>2</sup>

Prototype



1500mm length, 80gram weight

# STAVE Mechanics and Cooling

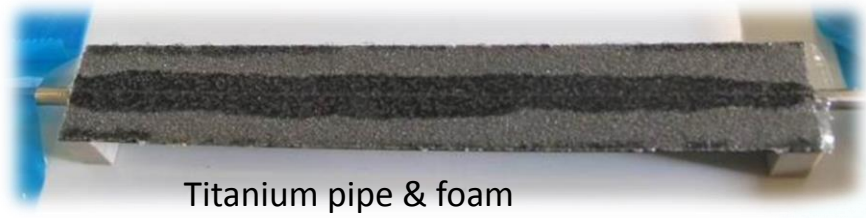
✓ **materials**



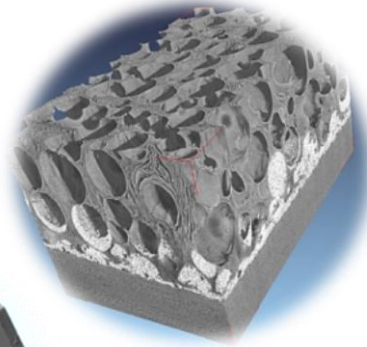
# ATLAS IBL stave operated @ $-20^{\circ}\text{C}$ , $\text{CO}_2$ evaporative



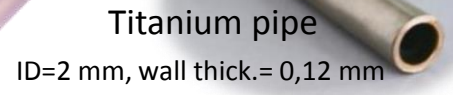
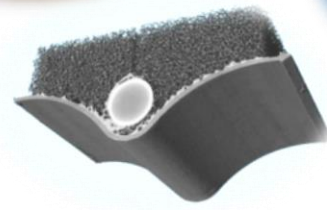
K13C2U



Titanium pipe & foam



Tomography of K9 carbon foam  
40 W/mK



Titanium pipe

ID=2 mm, wall thick.= 0,12 mm

## «pipe» materials

- High Thermal Conductive (HTC) material carries the heat to a pipe with coolant.
- Pipe embedded in carbon foam or graphite foil
- Pipe can be either metallic or plastic

# ALICE ITS stave upgrade, operated @ $<30^{\circ}\text{C}$ , water leakless



1.5 grams  
(290mm length)

D=1.02 mm  
wall thick.=24  $\mu\text{m}$

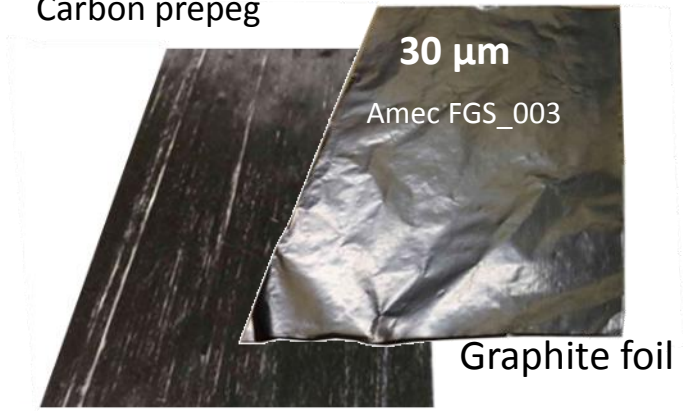


Polyimide pipes

Pyre M.L.

High thermal conductivity  
1500 W/mK

Carbon prepeg



30  $\mu\text{m}$

Amec FGS\_003

Graphite foil

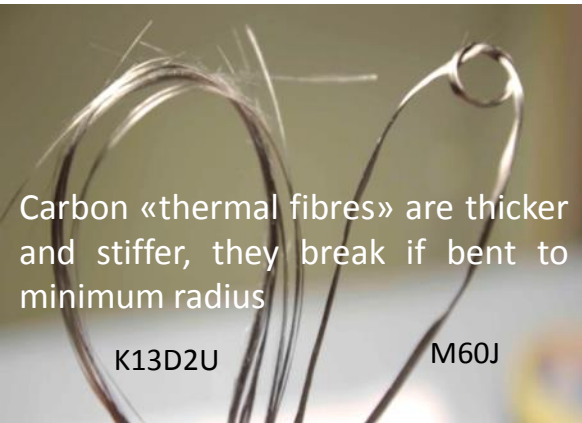


Carbon fibre filament wound

## ...carbon fibres

available in different shapes, offer a large range of properties

- ↑ High stiffness and strength, fibre  $E \sim 900\text{GPa}$ ,  $X \sim 4\text{GPa}$
- ↑ High thermal conductivity, fibre  $K \sim 1000\text{W/mK}$
- ↑ Minimum thickness, prepreg  $\sim 45\ \mu\text{m}$
- ↓ Limitation on fibres minimum bending radius
- ↓ Poor mechanical and thermal properties  $\perp$  fibres



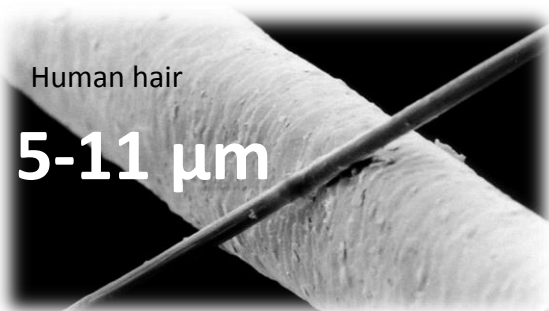
Carbon «thermal fibres» are thicker and stiffer, they break if bent to minimum radius

K13D2U

M60J

Thermal/ Structural fibres

# materials



Human hair

5-11  $\mu\text{m}$

Carbon filament



EX-1515

Cyanate ester resin

Roving

3000 filaments



M60J, 3K



20  $\mu\text{m}$

Fleece

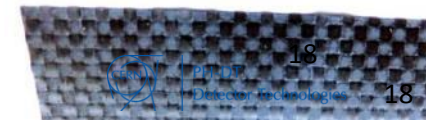


$\sim 45\ \mu\text{m}$

K13D2U

Unidirectional  
Prepreg

Fabric



# Carbon Structural

E [GPa]	$\chi_t$ [GPa]	
240	4,4	HT fiber



## Carbon Fleece

continuous-strand mat finished with a chemical binder to hold fibers in place  
 filament diameter=  $5\mu\text{m}$   
 $t=20\mu\text{m}$ ,  $8\text{g}/\text{m}^2$



	Filaments [K=1000]	Tex [g/km]	E [GPa]	$\chi_t$ [GPa]	K [W/mK]	CTE [K <sup>-1</sup> ]
<b>M60j</b>	3K	110	588	3,9	140	$-1,1 \times 10^{-6}$
<b>M55j</b>	6K	220	540	4,2	150	$-1,1 \times 10^{-6}$

## Carbon Roving

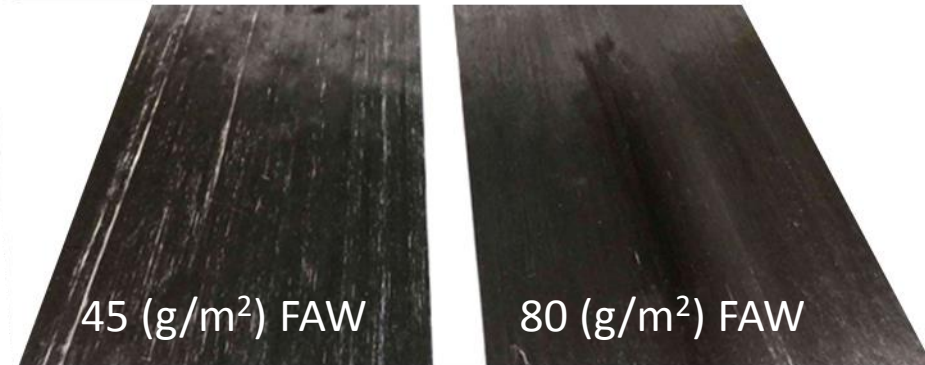
Bunches of filament parallel to each other  
 filament diameter=  $5\mu\text{m}$

# Carbon Thermal

Prepreg: Ready to mold or cure material in sheet form which contains fiber all aligned in one direction

filament diameter= 11 $\mu$ m

## Carbon Unidirectional Prepreg



	Filaments [K=1000]	E1 [GPa]	X <sub>t</sub> /X <sub>c</sub> [MPa]	E2 [GPa]	Y <sub>t</sub> [MPa]	K [W/mK]	CTE [10 <sup>-6</sup> K <sup>-1</sup> ]
<b>K13D2U [0] fibre</b>	2K	935	3600			800	-1,2
<b>K13D2U [0] prepreg</b>	2K	560	1800/340	5,1	25	~450	-1 / 61

## Carbon Paper



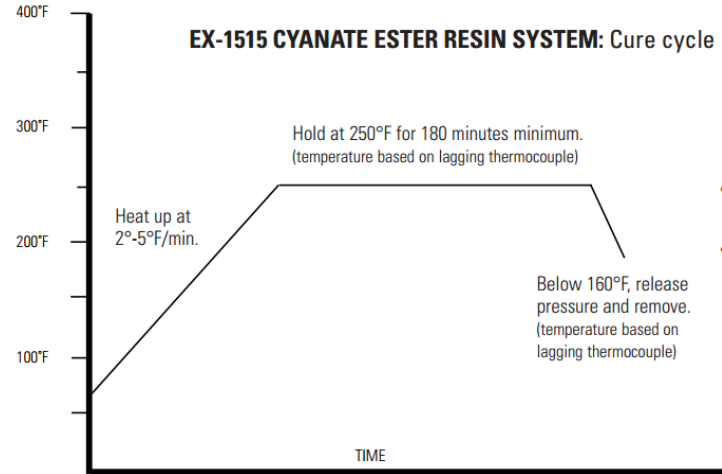
Thermal management material with very high thermal conductivity and flexibility

Thick. [ $\mu$ m]	Density [g/cm <sup>3</sup> ]	K pl [W/mK]	K th [W/mK]
30	1.6	1500	15

Amec FGS\_003

# Resin system: Cyanate Ester

Cyanate Ester (CE) resin to be preferred to epoxy  
Optimal mechanical properties, high radiation resistance, low moisture absorption



- Apply 25 inches Hg vacuum minimum.
- Apply 40 - 100 psig pressure to autoclave (optional).

## TECHNICAL DATA



TENCATE ADVANCED COMPOSITES USA, INC.

### EX-1515 Resin System

#### PRODUCT TYPE

225°F/121°C (Without Post Cure)  
Cure Toughened Cyanate Ester

#### SERVICE TEMPERATURE

250°F/121°C (Without Post Cure)  
325°F/163°C (With Post Cure)

#### TYPICAL APPLICATIONS

- High Dimensional Stability Space Structures
- Optical Benches
- Reflectors
- Radomes and Antennae
- Low Observables
- Radar Transparent Structures

#### FEATURES

- High radiation resistance
- Low microcracking under severe thermocycling
- Low moisture absorption
- Low dielectric constant & dissipation factors
- Low stress-free cure temperature with high level of cure
- Optional mechanical properties
- Compatible EX-1516 adhesive

#### SHELF LIFE

7 days @ 77°F/25°C  
6 months @ < 0°F/-18°C

#### PRODUCT DESCRIPTION

TenCate's EX-1515 cyanate ester resin system is very unique among its peers in that it is able to achieve an extremely high level of conversion cure after a 250°F/121°C cure. This level of conversion provides optimal mechanical properties, high radiation resistance, low moisture absorption/low outgassing while retaining unparalleled toughness, a low 244°F/118°C, stress free temperature and long out time. The resin system excels in its ability to resist microcracking, even when subjected to thermal cycling and high levels of radiation exposure.

EX-1515 also displays low dielectric/low loss values similar to other cyanate esters which allows its use in radome and antenna applications as well. TenCate's EX-1515 can be post cured, free standing, to increase its thermal performance for temperature critical structures.

#### NEAT RESIN PHYSICAL PROPERTIES

Moisture Absorption..... 2.1% after 313 days at 160°F/71°C, 85% RH  
Outgassing .....TML: 0.179%, VCM: 0.007%  
Density ..... 1.17 gm/cc  
Tg by DMA ..... 249°F/121°C cured @250°F/121°C  
345°F/174°C post cured @350°F/177°C

CTE34 ppm/°F ( 61 ppm/°C)

Thermal Conductivity..... 0.169 W/m\*K

#### NEAT RESIN ELECTRICAL PROPERTIES

Dielectric Constant ..... 2.8 @10 GHz  
Loss Tangent ..... 0.004 @10 GHz

#### LAMINATE ELECTRICAL PROPERTIES ON 4581 AQIII QUARTZ

	X-Band	X-Band	O-Band	W-Band
	8-12.6 GHz	18-26.5 GHz	33-50 GHz	75-110 GHz
Dielectric Constant	3.32	3.30	3.30	3.30
Loss Tangent	0.0035	0.0035	0.0052	0.0065

	4581 AQ III / EX-1515	7781 Fg / EX-1515
	8 HS FAW 300 gsm	
Tensile Strength	109.8 Ksi (757 MPa)	61.5 Ksi (424 MPa)
Tensile Modulus	3.45 Msi (23.8 GPa)	3.65 Msi (25.3 GPa)
Compression Strength	78.8 Ksi (543.3 MPa)	57.0 Ksi (393 MPa)
Compression Modulus	4.06 Msi (28.0 GPa)	3.7 Msi (25.5 GPa)
Flexural Strength	107.0 Ksi (737.7 MPa)	71.0 Ksi (489.5 MPa)
Flexural Modulus	3.16 Msi (21.8 GPa)	3.15 Msi (21.8 GPa)
ILSS	9.86 Ksi (68.0 MPa)	6.7 Ksi (46.2 MPa)

\* Normalized to 55% fiber volume

## EX-1515 TENCATE (127°C)

-radiation resistance

-extensive use at CERN

- Low curing temperature, 127°C

## RS-3 TENCATE (177°C)

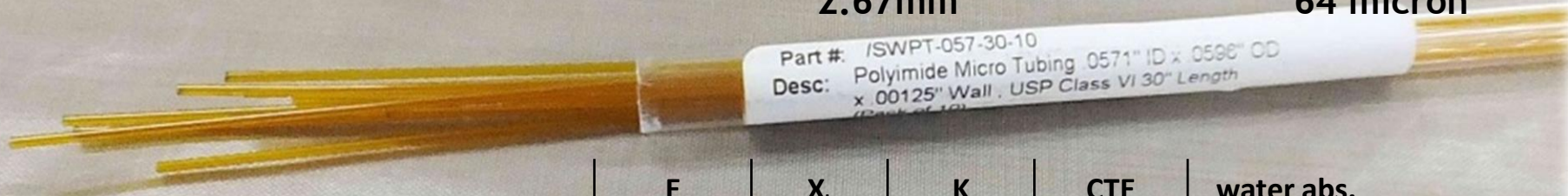
-the other predominant Cyanate Ester (CE) resin used at CERN.

## Hexcel 954 (177°C)

-moderate use at CERN

# polyimide tubes

Inner Diameter	1.02 mm	wall thickness	24 micron
	1.45mm		32 micron
	2.05mm		32 micron
	2.67mm		64 micron



Medical application

	E [GPa]	$X_t$ [MPa]	K [W/mK]	CTE [ $10^{-6} K^{-1}$ ]	water abs. [%]
<b>PI</b>	2,5	305	0,205	40	0,841

Pyre M.L.

**High radiation hardness:** according to **CERN-98-01** report, polyimide:

No problem below  $10^7$  Gy, Mild damage between  $10^7$  to  $5 \cdot 10^7$  Gy

1<sup>st</sup> layer of ITS Inner Barrel will be exposed to 700 krad/yr.=7000 Gy/yr.

**Ageing:** physical and chemical stability over time.

Plastic Pipe Institute states corrosion is not an issue in plastic pipes.

**Fire Safety:** Comply to LHC Fire Safety Instruction (IS-41)

Polyimide is allowed.

**High radiation length material**

Polyimide:  $X_0 = 29$  cm, minimum wall thickness is 0.025 mm.

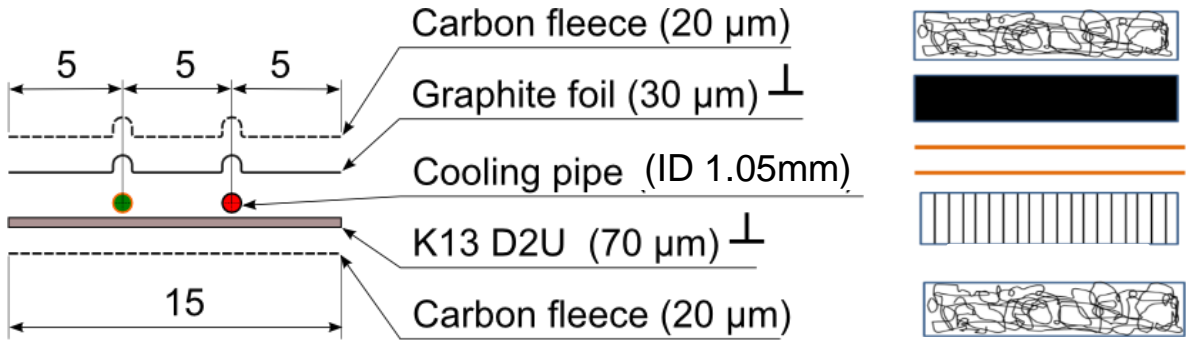
PEEK:  $X_0 = 31.45$  cm, minimum wall thickness is 0.25 mm.

# STAVE Mechanics and Cooling

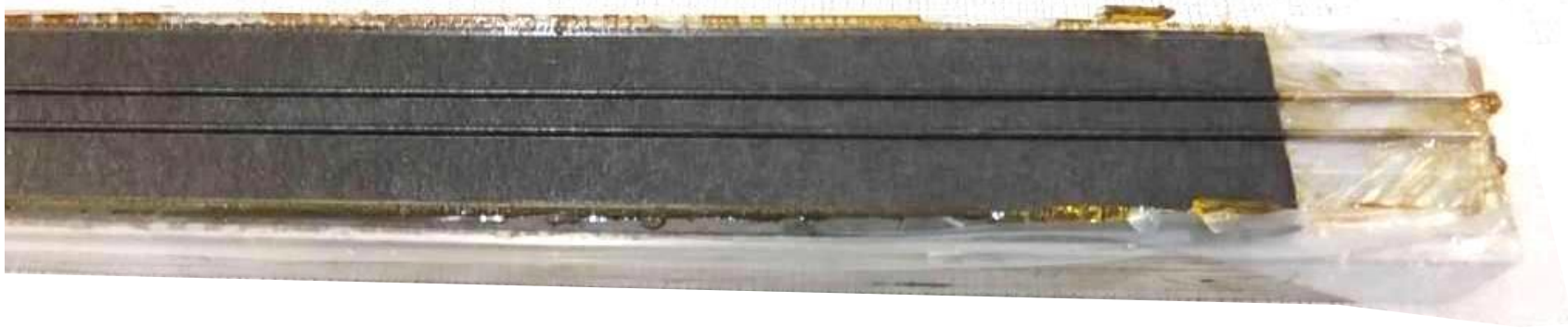
✓ **production process**

# IB Production Process: Manual Lay-up

New innovative design developed fo ALICE ITS Upgrade







**IB Production Process: Co-curing process**

# IB Production Process: Co-curing process



Inner mould extraction



Cut and mould separation

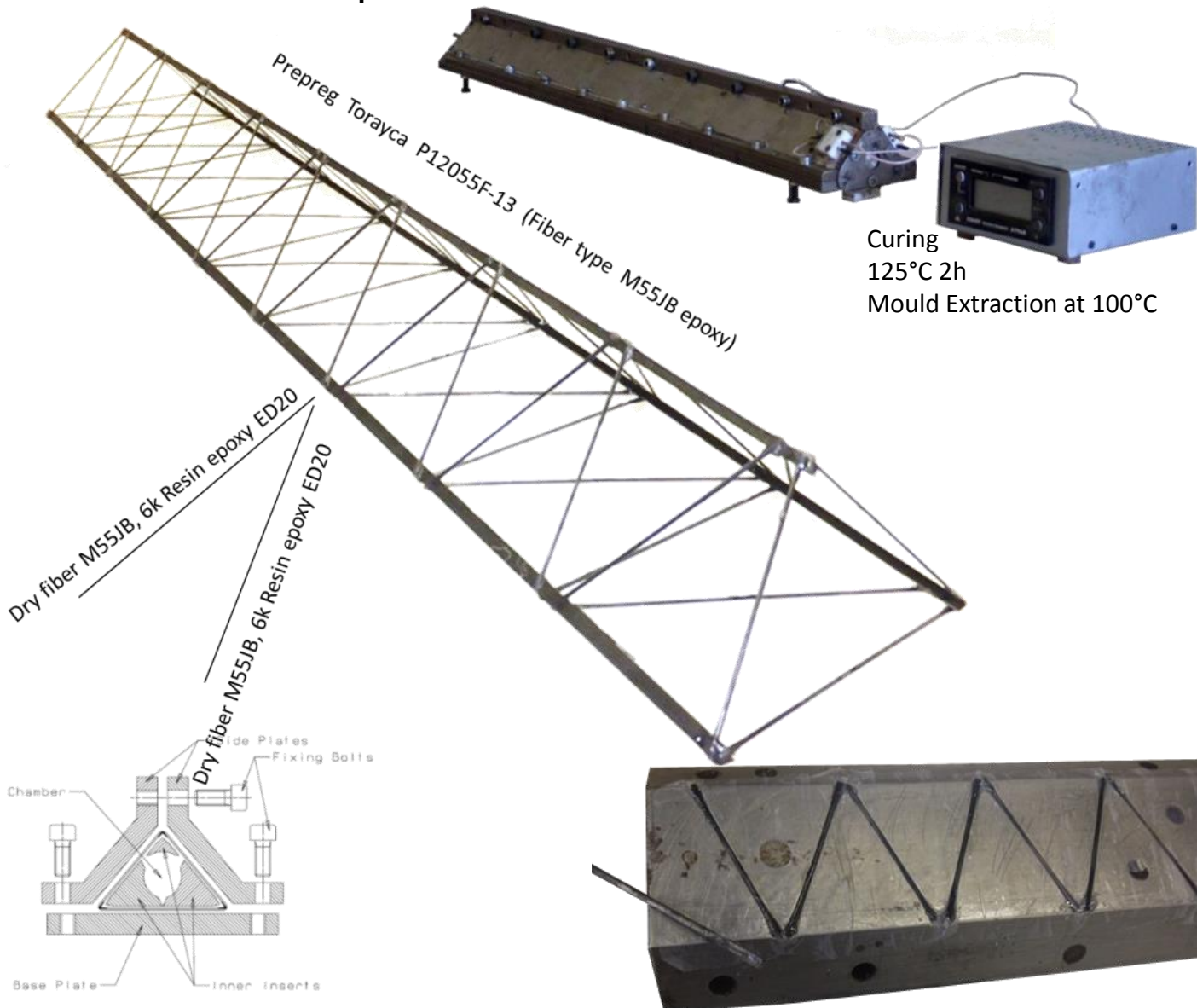


290mm length, 1.5gram weight

# OB Production Process: Spaceframe

Based on original design developed by St Petersburg University, Utrecht Univeristy and INFN Torino for the ALICE tracker now in operation.

Spaceframe <1m

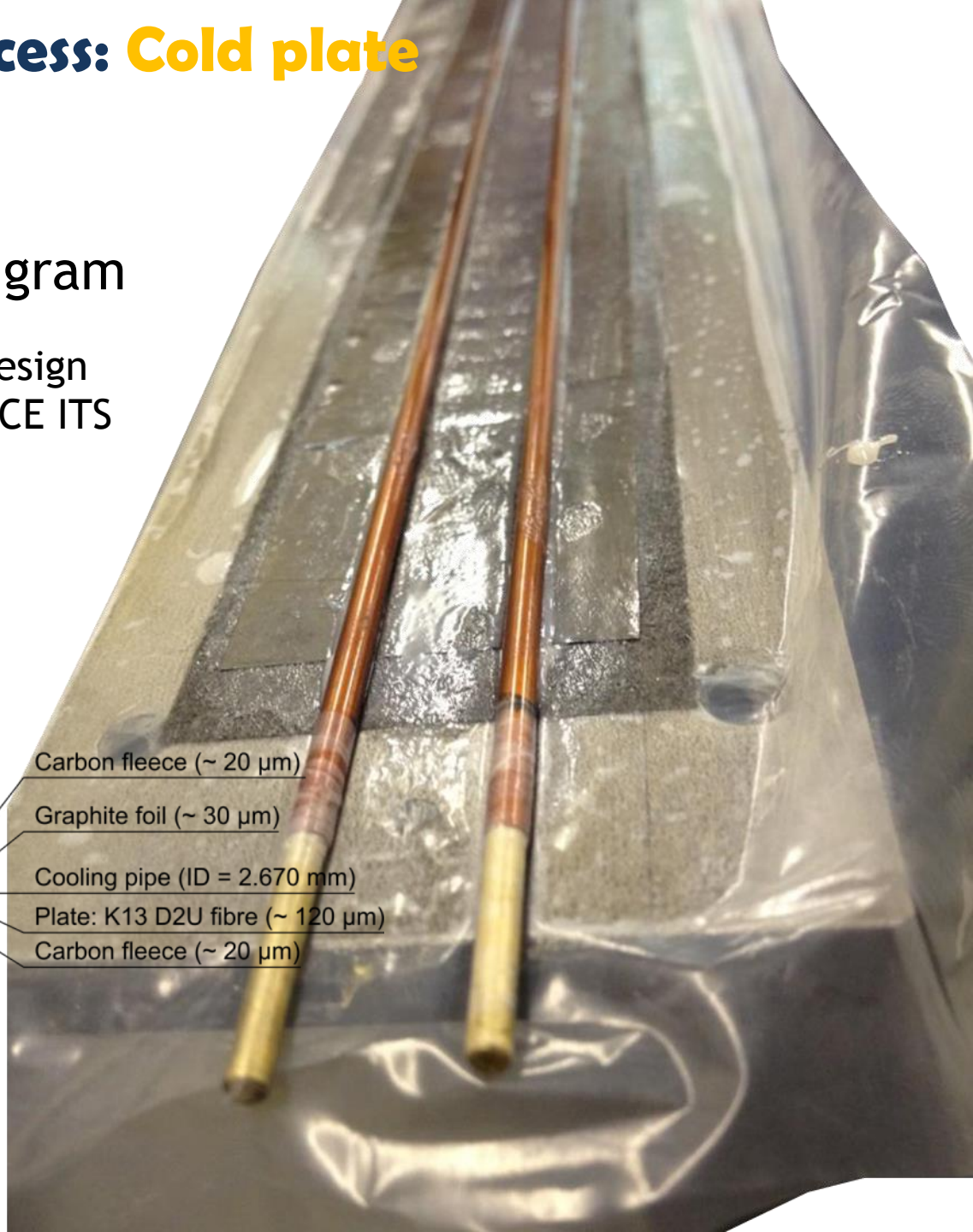
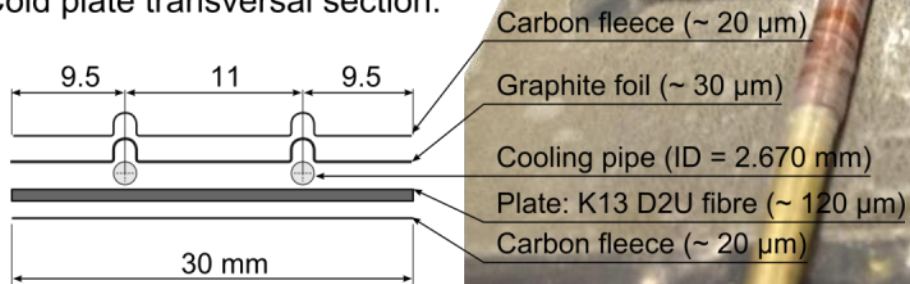


# OB Production Process: Cold plate

1500mm ~22 gram

New innovative design  
developed for ALICE ITS  
Upgrade

Cold plate transversal section:



# **STAVE Mechanics and Cooling**

✓ **material budget**

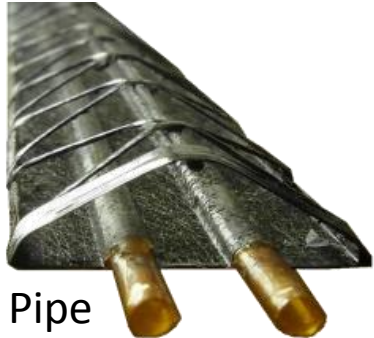
# material budget

...very stringent requirements

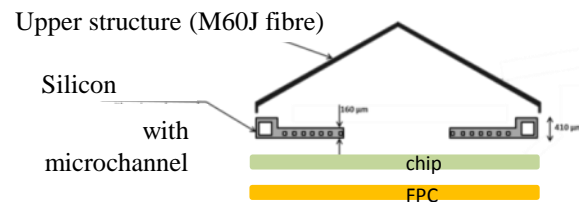
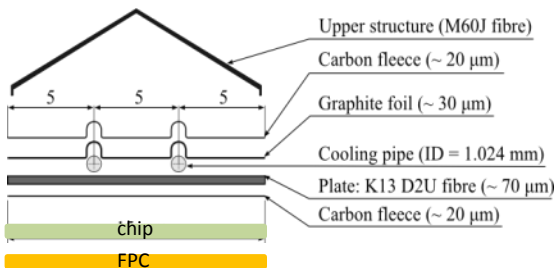
$X_0 E$  is used as figure of merit for all the structural material of a detector. With this criterion, the best materials are beryllium and carbon fibre.

Material	Be	CFC	Al-Be	Al	Ti	Fe
$E$ (GPa)	290	200	193	70	110	210
$X_0$ (m)	0.353	0.271	0.253	0.089	0.036	0.018

Pipe



microchannel



ALICE

mean

$X/X_0 \sim 0.3\%$

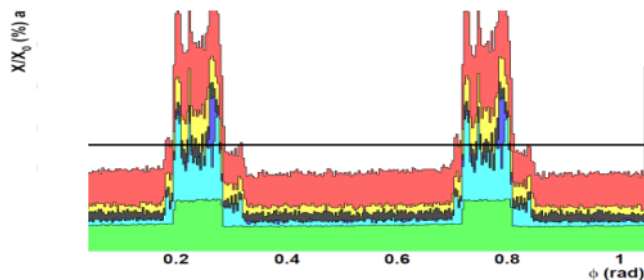
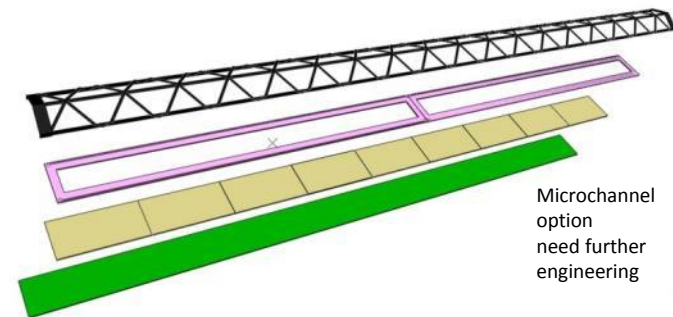
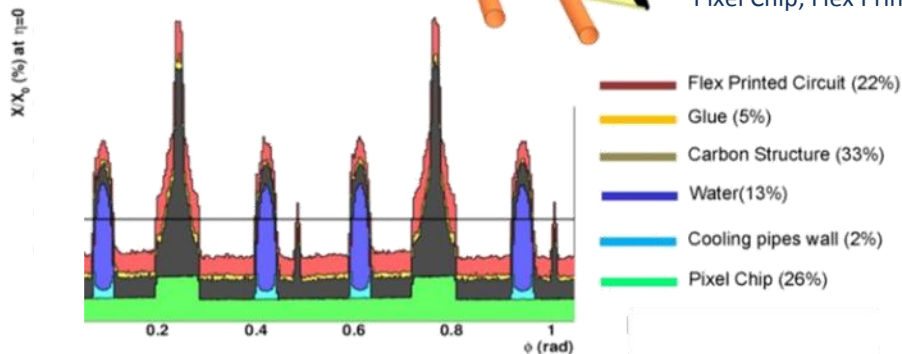
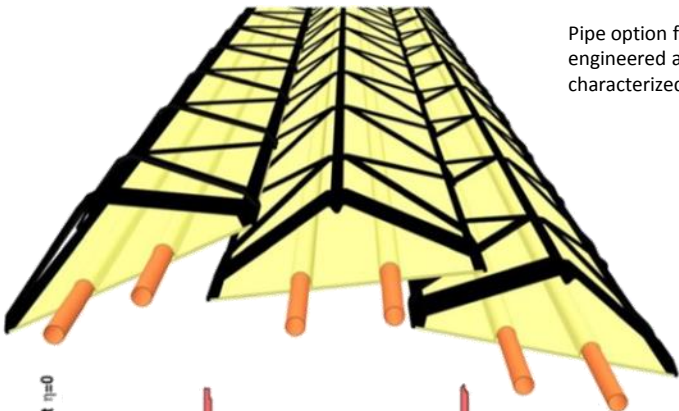
per layer

Includes:  
Structure,

Pixel Chip, Flex Printed Circuit, Coolant

Pipe option fully engineered and characterized

Microchannel option need further engineering



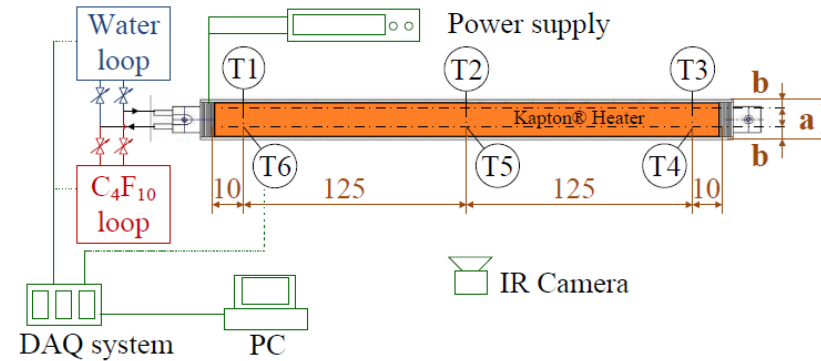
# **STAVE Mechanics and Cooling**

✓ **thermal characterization**

# Thermal characterization



$q$ [ $\text{W cm}^{-2}$ ]	$G$ [ $\text{L h}^{-1}$ ]	$\Delta T_{\text{CHIP-H}_2\text{O}}$ [K]	$\Delta T_{\text{H}_2\text{O}}$ [K]	$\Delta p$ [bar]	$v_{\text{H}_2\text{O}}$ [ $\text{m s}^{-1}$ ]
<b>0.15</b>	3.0	2.4	1.4	0.3	1.0



## Water leakless (<1bar) baseline

Water in  $15^\circ\text{C}$  --->  $T_{\text{chip}} < 30^\circ\text{C}$

Pixel max temperature non-uniformity  $< 5^\circ\text{C}$

Pressure drop  $\Delta P$  below 0.3 bar

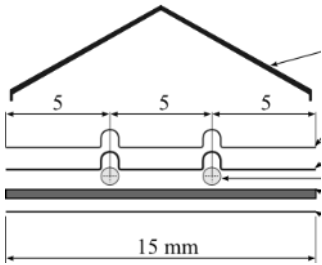


$q$ [ $\text{W cm}^{-2}$ ]	$G$ [ $\text{L h}^{-1}$ ]	$\Delta T_{\text{CHIP-H}_2\text{O}}$ [K]	$\Delta T_{\text{H}_2\text{O}}$ [K]	$\Delta p$ [bar]	$\Delta T_{\text{HEATERS}}$ [K]	$v_{\text{H}_2\text{O}}$ [ $\text{m s}^{-1}$ ]
<b>0.15</b>	6.3	6.7	6.9	0.08	4	0.31



# Chip non uniform power dissipation

Transversal section:



**dummy metalized chip acts as heater**

$P_{in} = 1 \text{ bar}$   
 $T_{in} = 15.8^\circ \text{ C}$

$T_{out} = 16.6^\circ \text{ C}$   
 $P_{out} = 0.7 \text{ bar}$

$Q = 3 \text{ L/h}$

15 mm

2 mm

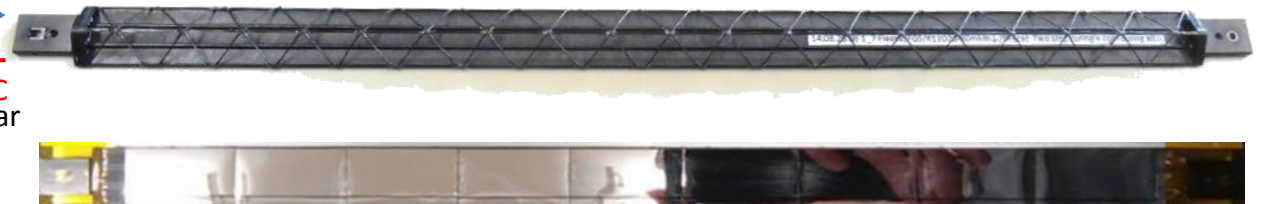
30 mm

Si 50  $\mu\text{m}$

Titanium 20 nm

Platinum 200 nm

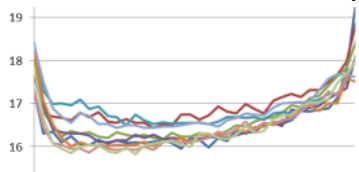
Chip Power dissipation concentrated at periphery: 83% of total power



Heating is provided by dummy metalized chip : thickness= 50  $\mu\text{m}$  chip + 20/200 nm Titanium /Platinum

$\sim 40 \text{ mW/cm}^2$

Periphery: 0.145 W/chip  
Pixels: 0.03 W/chip

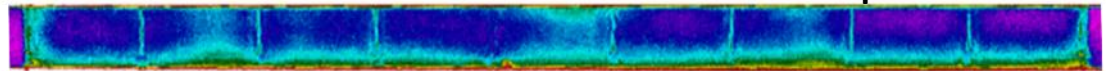


$\sim 60 \text{ mW/cm}^2$

Periphery: 0.217 W/chip  
Pixels: 0.045 W/chip



Min T pixel =  $16^\circ \text{ C}$

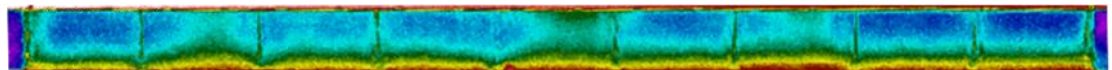


Max T periph =  $17.5^\circ \text{ C}$

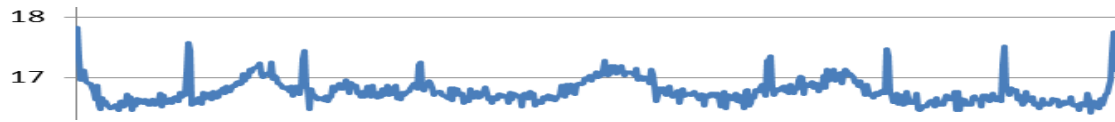


middle section

Min T pixel =  $16.5^\circ \text{ C}$



Max T periph =  $18.5^\circ \text{ C}$

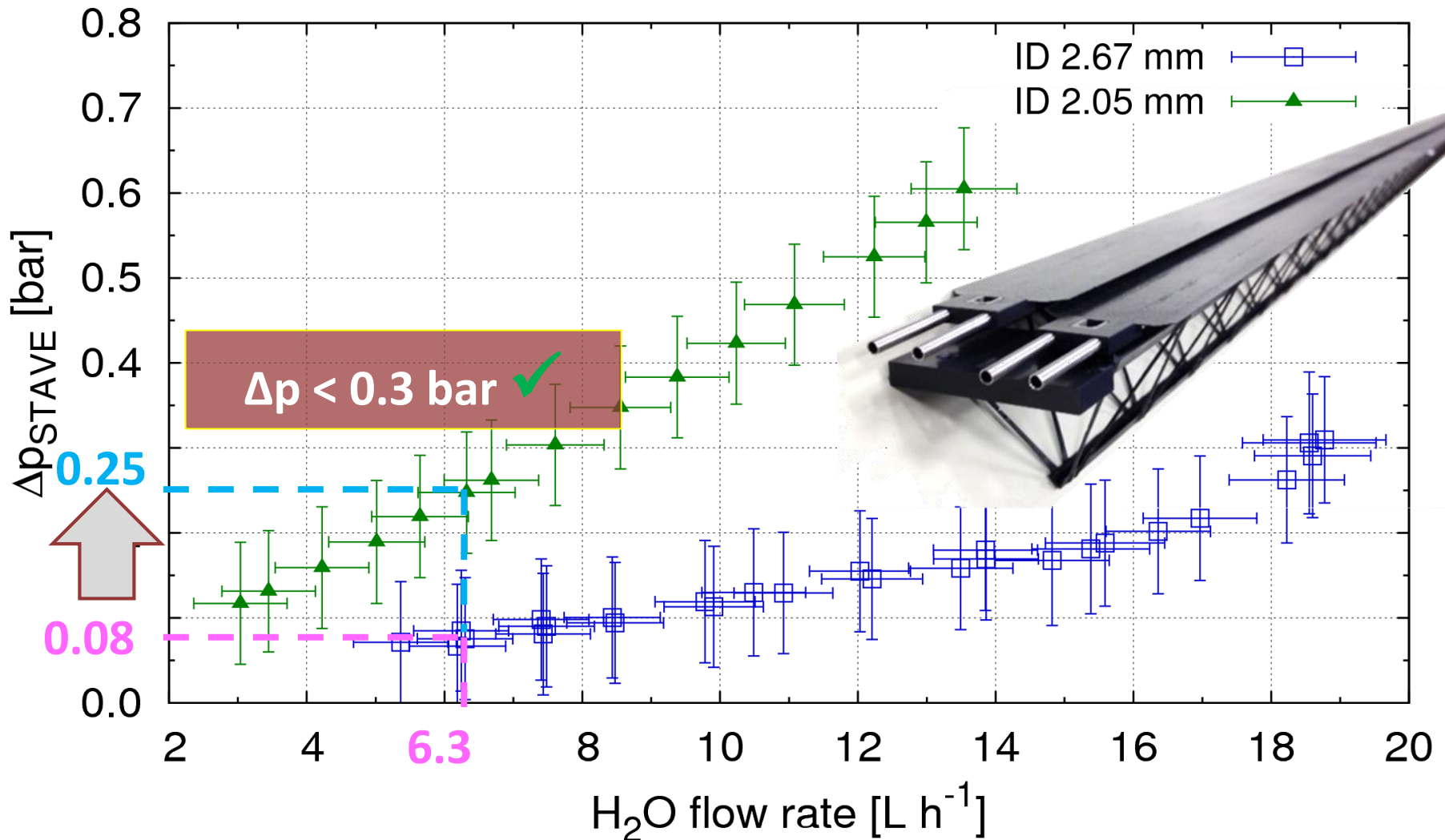


middle section

# pressure drop for 1,5m OB stave

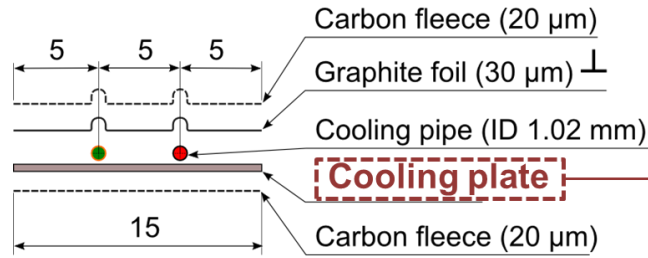
Pressure drop for leak less system to be below 0,3 bar

Pressure drop vs. H<sub>2</sub>O flow rate



# Studies: different carbon layup

## STAVE SECTION



1. K13 D2U 70  $\mu\text{m}$   $\perp$

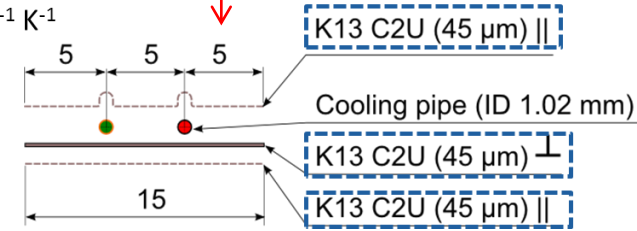
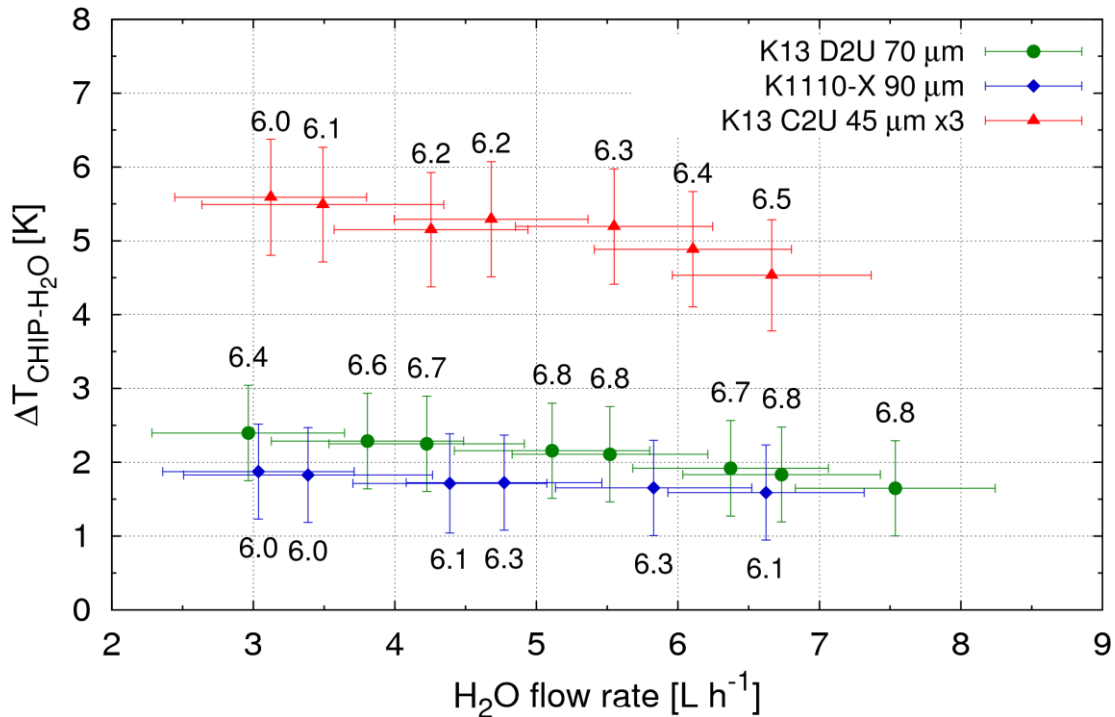
▪  $k_{K13D2U} \sim 800 \text{ W m}^{-1} \text{ K}^{-1}$

2. K1100-X 90  $\mu\text{m}$   $\perp$

▪  $k_{K1100} \sim 1000 \text{ W m}^{-1} \text{ K}^{-1}$

3 (alt). K13 C2U 45  $\mu\text{m}$  x3

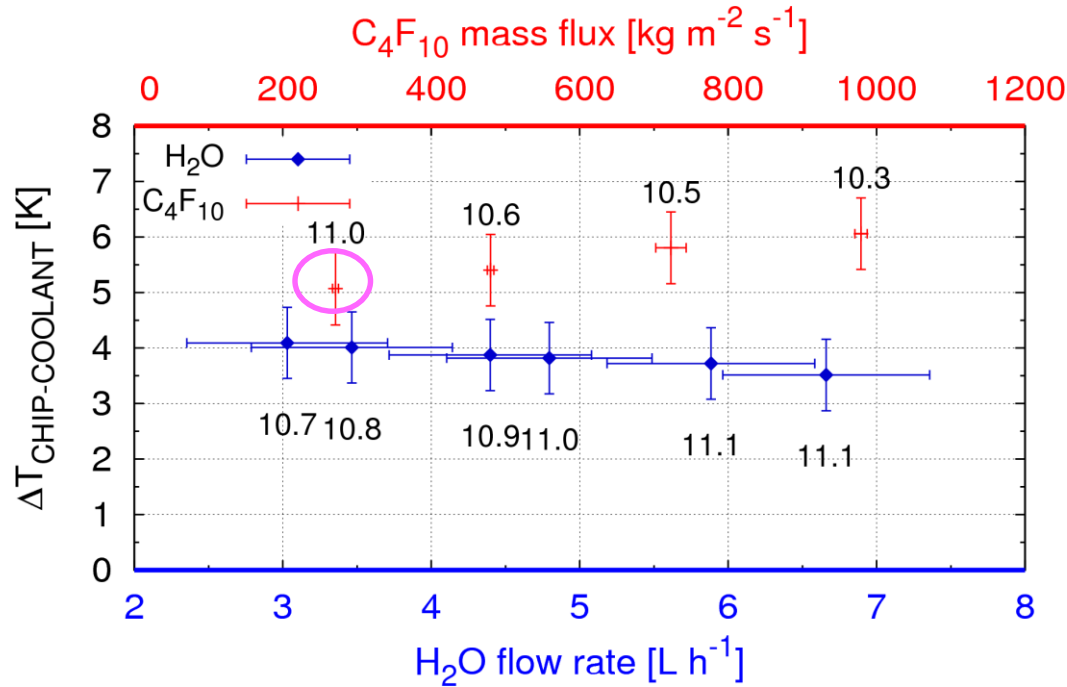
▪  $k_{K13C2U} \sim 620 \text{ W m}^{-1} \text{ K}^{-1}$



# Studies: evaporative C4F10

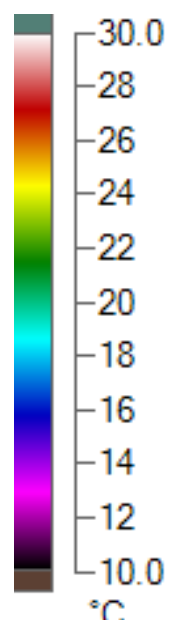
IB K1100-X 90  $\mu\text{m}$ ,  
50mW/cm<sup>2</sup>

Almost no difference in thermal performance H2O vs C4F10



Cold plate performance driven by cold plate minimum thickness

$G$ [kg m <sup>-2</sup> s <sup>-1</sup> ]	$\Delta p$ [bar]	$\Delta T_{\text{C4F10}}$ [K]	$T_{\text{C4F10}}$ [°C]	$T_{\text{Chip}}$ [°C]	$x_{\text{In}}$ [-]	$x_{\text{Out}}$ [-]
271	0.29	3.1	13.9	19.0	0.14	0.40



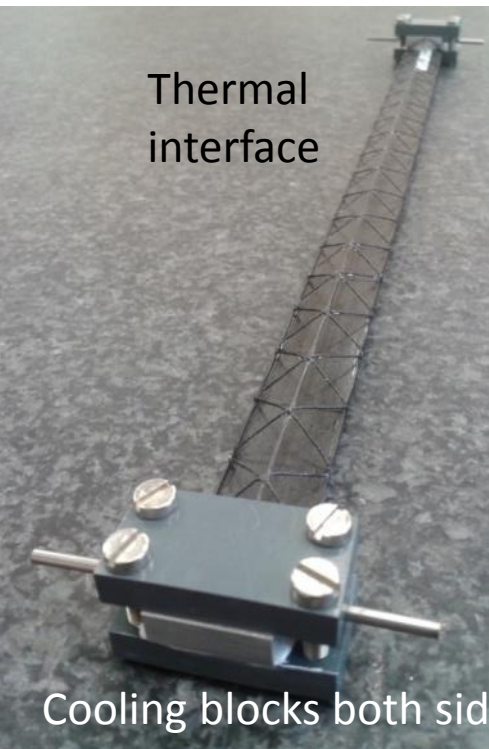
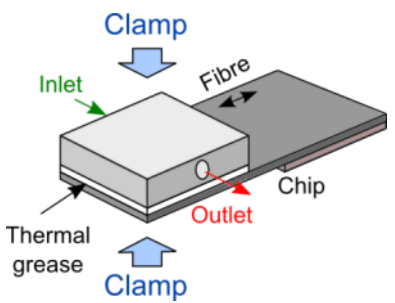
# studies: peripheral cooling, no pipes on the stove



←→  
**Carbon Fibre orientation**

**Cold plate section**

- C fleece (20)
- C Paper (30)
- k1100 (90)
- C Paper (30)
- C fleece (20)



$q$ [W cm <sup>-2</sup> ]	Power input [W]	Estim. power to flow [W]	T water [°C]	T map [°C]
0.02	0.82	0.56	12.6	<p>Max = 27.9 Avg = 23.2 Min = 12.5</p> <p>27.9 +</p>
0.03	1.25	0.71	12.7	<p>Max = 33.8 Avg = 27.8 Min = 15.3</p> <p>33.8 +</p>
0.04	1.64	0.80	12.9	<p>Max = 38.6 Avg = 32.2 Min = 18.4</p> <p>38.6 +</p>
0.05	2.02	0.86	12.9	<p>Max = 42.9 Avg = 34.6 Min = 19.3</p> <p>42.9 +</p>

# STAVE Mechanics and Cooling

- ✓ **mechanical characterization**

sandwich

I-Beam

spaceframe

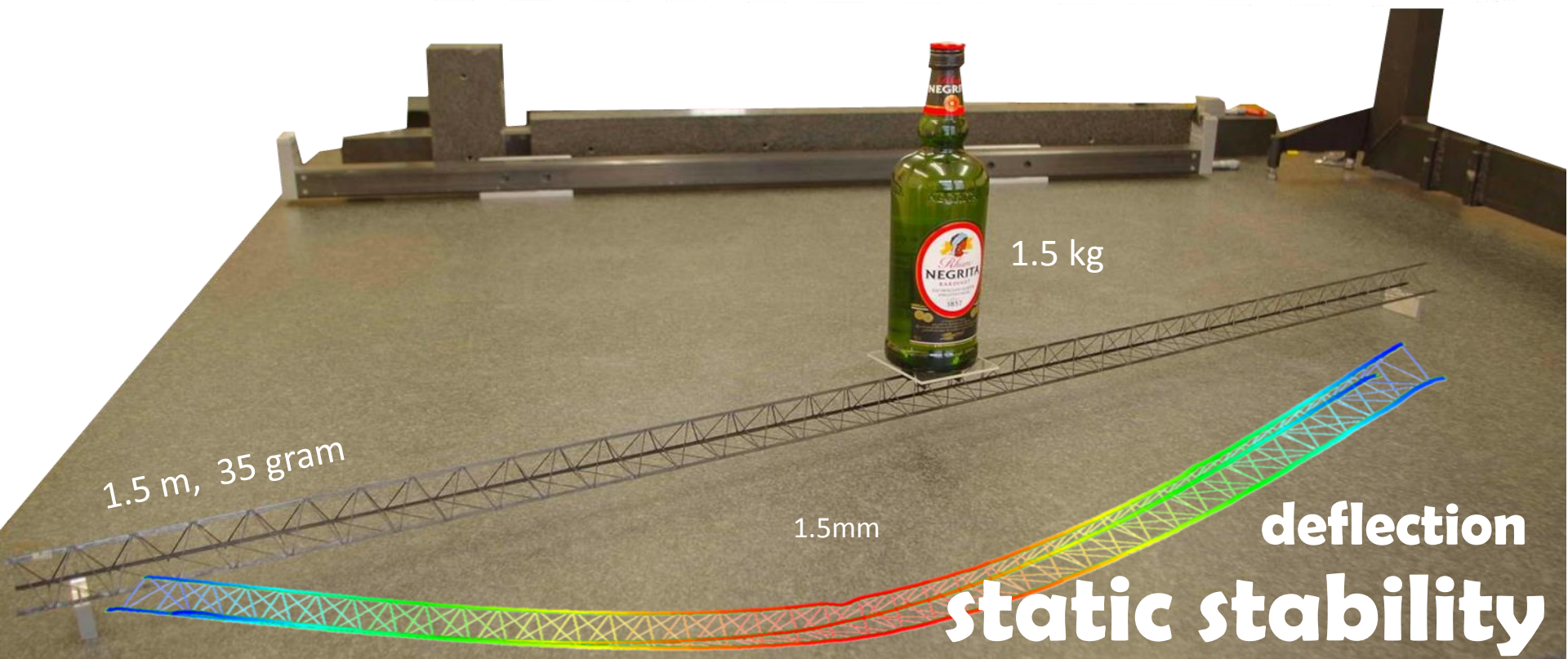


Phase II strip

Phase II pixel

ITS upgrade

Stiffness is provided not only by the choice of high modulus material but also by large inertia section, different options proposed



1.5 m, 35 gram

1.5 kg

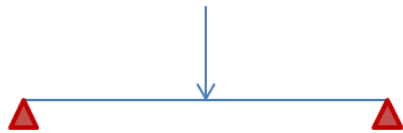
1.5mm

deflection  
static stability

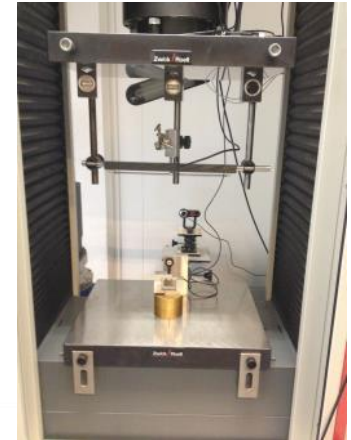
# static stability: deflection

Objective: evaluate stave sag under Chip+FPC load

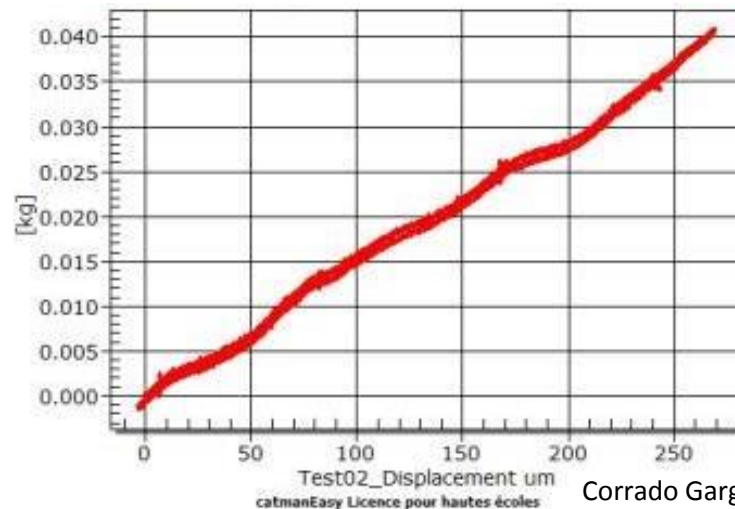
3 point bending test results



Displacement sensor:  
HEIDENHAIN (MT 1201), Accuracy:  $\pm 0.2 \mu\text{m}$   
Force transducer:  
HBM (S2M-10), Accuracy: 0.002 N



INNER Barrel stave  
HIC (chip + FPC) mass estimate = 2 gr  
**predicted sag 4-9  $\mu\text{m}$**



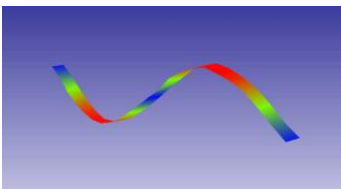
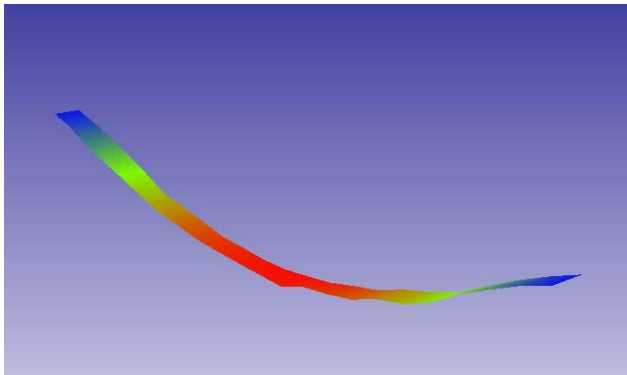


# static stability: natural frequencies

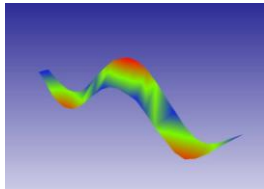
**Objective:** Determine the natural frequencies of the Inner tracker stave.

Test results

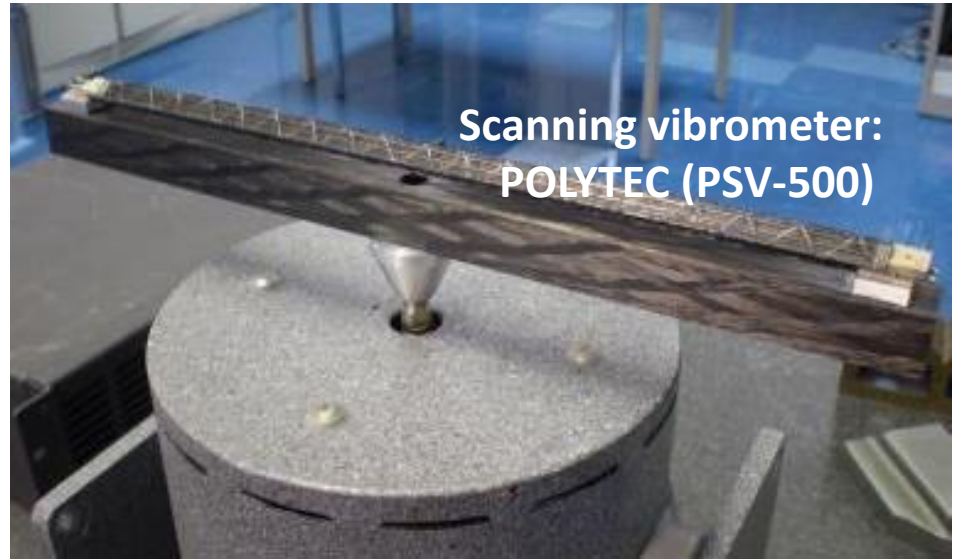
**167 Hz**



**461 Hz**

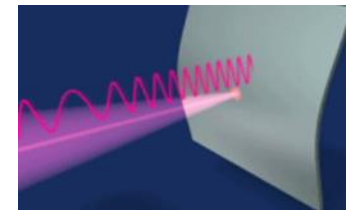
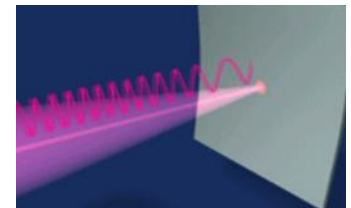


**764 Hz**



Laser non-contact vibration  
measurement

Mechanical Measurements Laboratory  
EN-MME-EDM

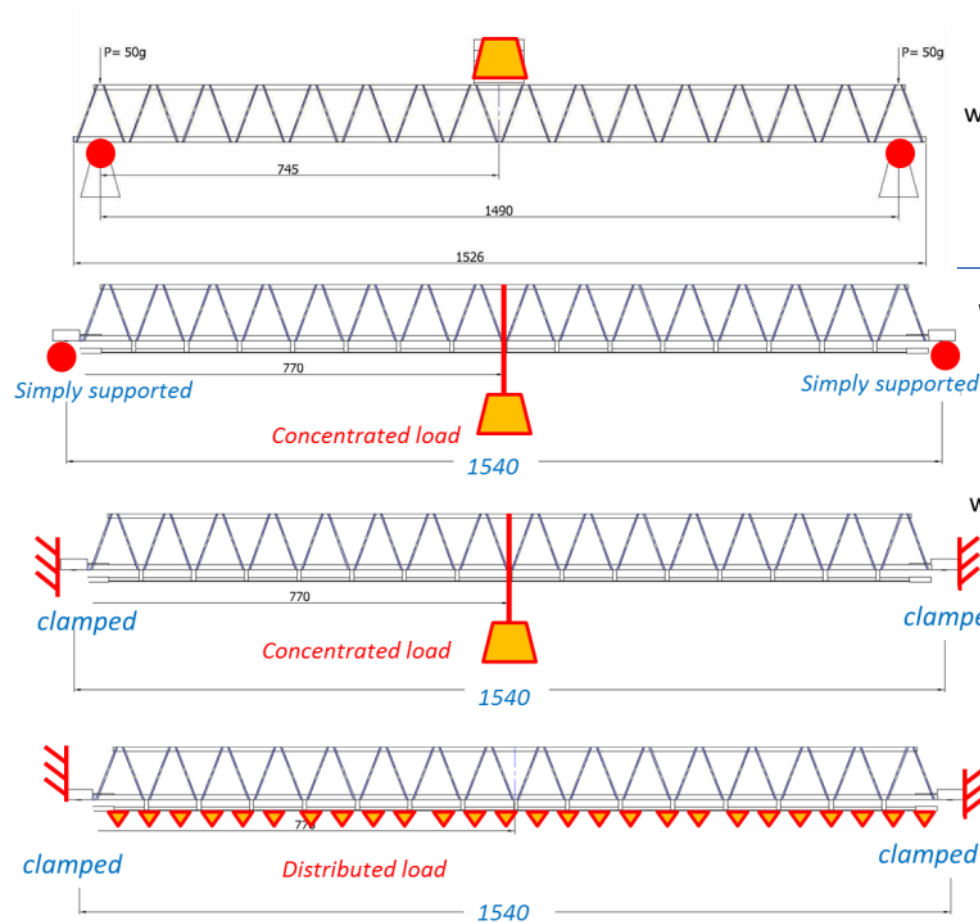


# static stability: deflection

Objective: evaluate stave sag under Chip+FPC+Bus load



OUTER Barrel stave  
HIC (chips + FPC)+Bus mass estimate =170 gr  
**predicted max sag 80  $\mu$ m**



**test**

weight 170g, sag=160 $\mu$ m\*

weight 170g, sag=153 $\mu$ m

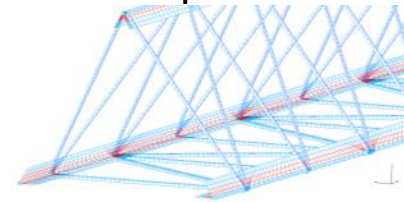
weight 170g, sag=123 $\mu$ m

weight 170g, sag=80 $\mu$ m

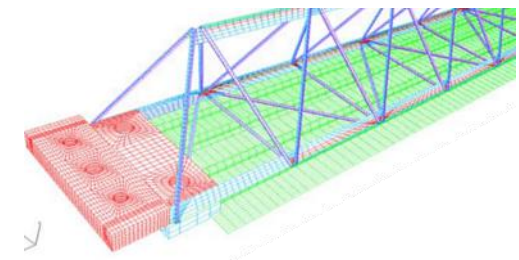
TEST do not account for HIC/PWBus stiffness

**FEA**

spaceframe

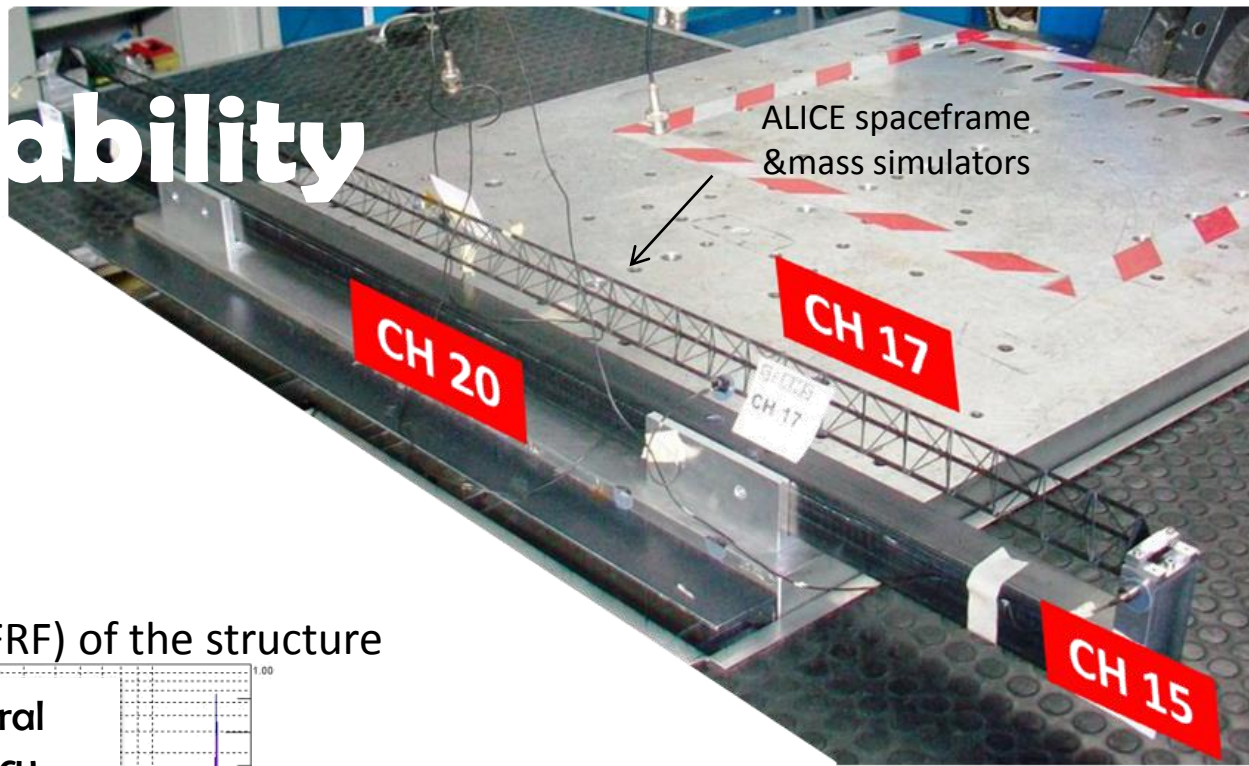


Spaceframe+ coldplate

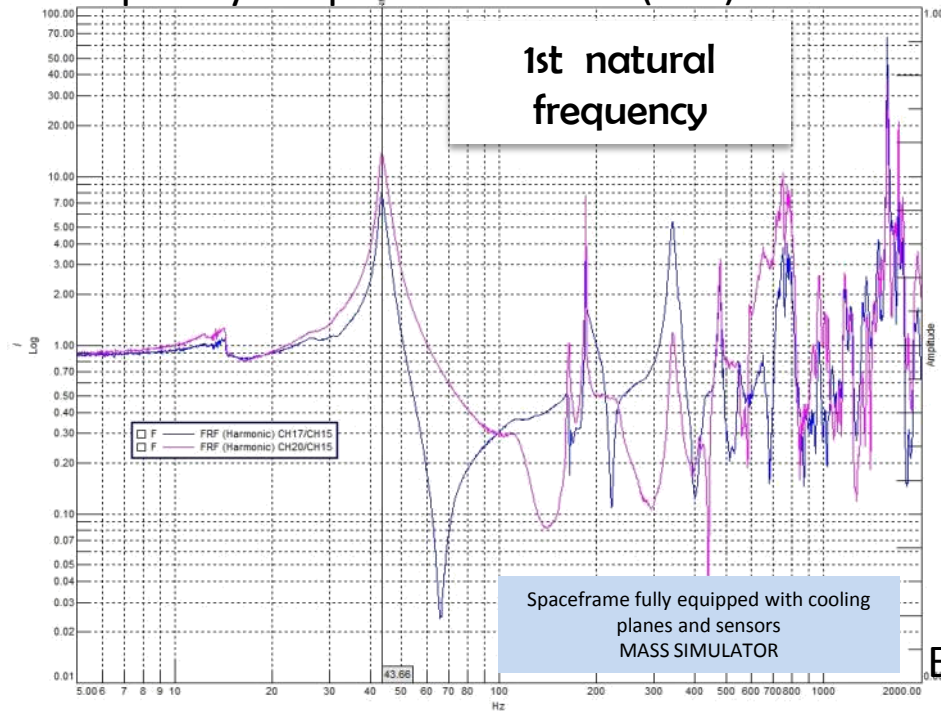


# dynamic stability

- natural frequencies
- modal **damping** coefficients
- modal shapes



Frequency Response Function (FRF) of the structure



Determine vibrational input



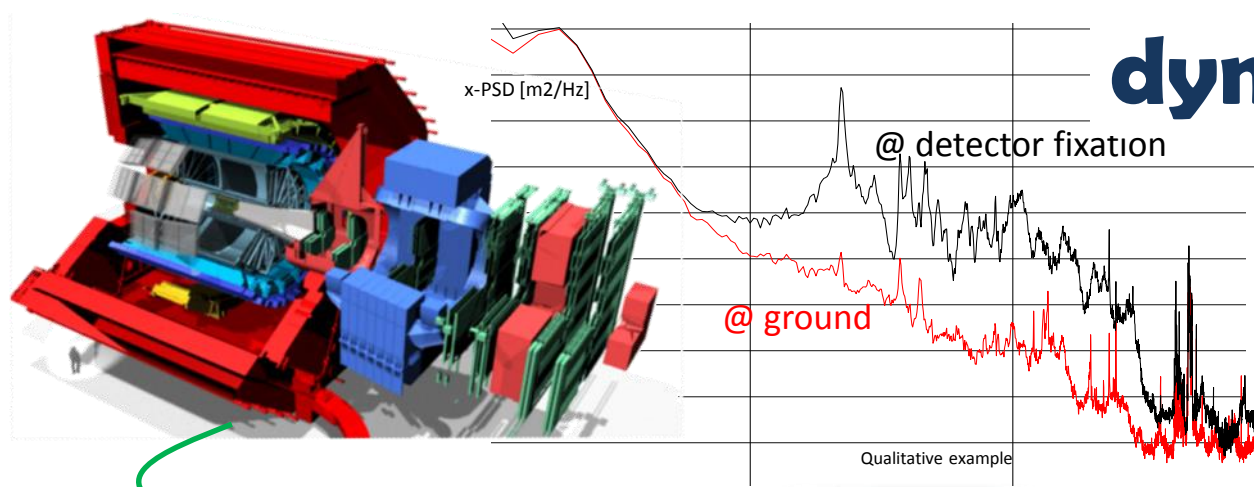
Determine structure FRF



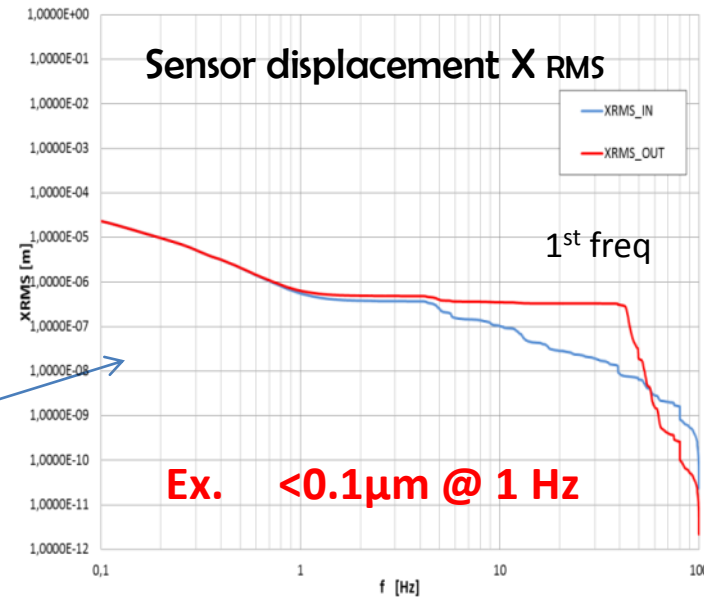
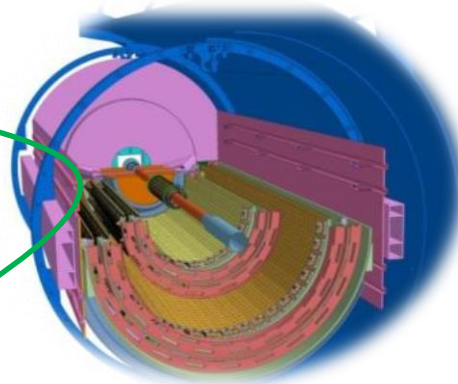
Evaluate output in term of sensors (chip) stability

# dynamic stability

The first source of vibration comes from the ground  
Seismic (<1Hz, cultural>1Hz)

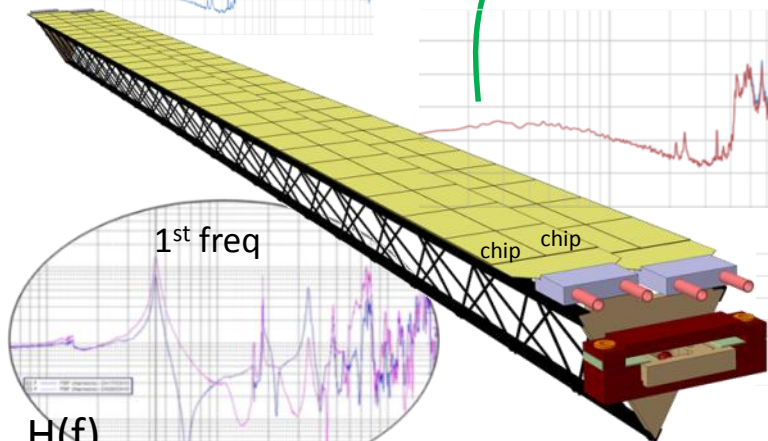
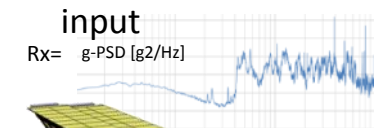


The Experiments amplify the ground motion at frequencies above 1Hz



$$R_Y(f) = |H(f)|^2 R_X(f)$$

Amplification @1st freq



input

$$R_X = g_{PSD} = \frac{(2\pi f)^4}{9.81^2} x_{PSD}$$

Vs  
pixel stability requirement  
5-10µm

# **STAVE Mechanics and Cooling**

✓ **mechanical characterization**

**erosion test**

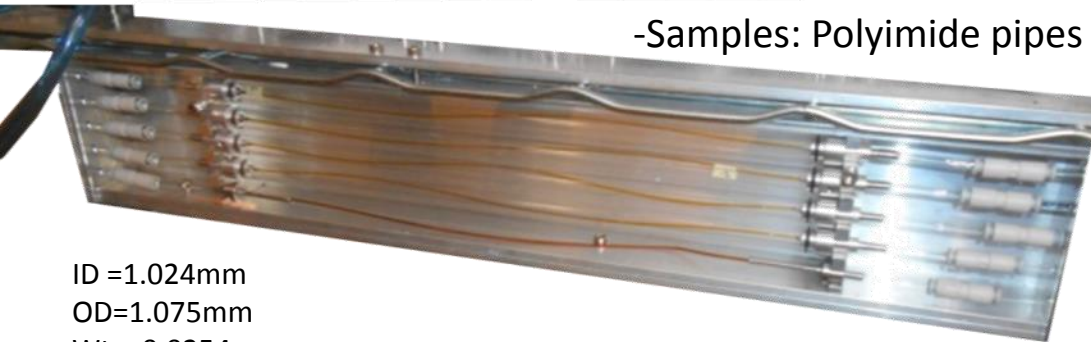
# erosion test



**Objective:** Long term test with water flow to evaluate erosion effects on polyimide pipes

SEM (surface damage)  
AFM (surface roughness change)

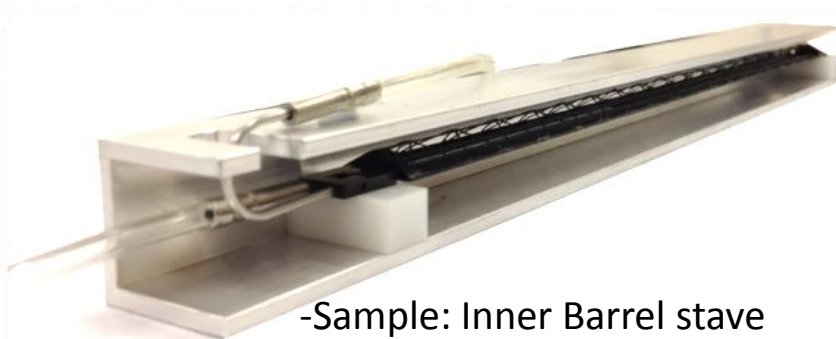
-Test set up in the ALICE cavern



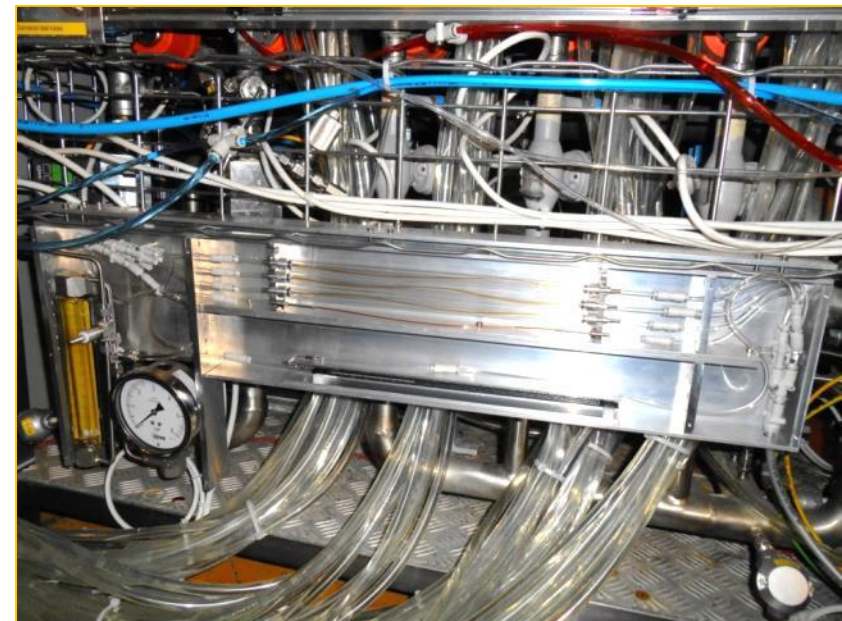
-Samples: Polyimide pipes

ID = 1.024mm  
OD = 1.075mm  
Wt = 0.0254mm  
L = 300mm

-Flow rate: 0.3 l/min  
-P inlet: 2.30 bar  
-T = 20°C



-Sample: Inner Barrel stave



# erosion test

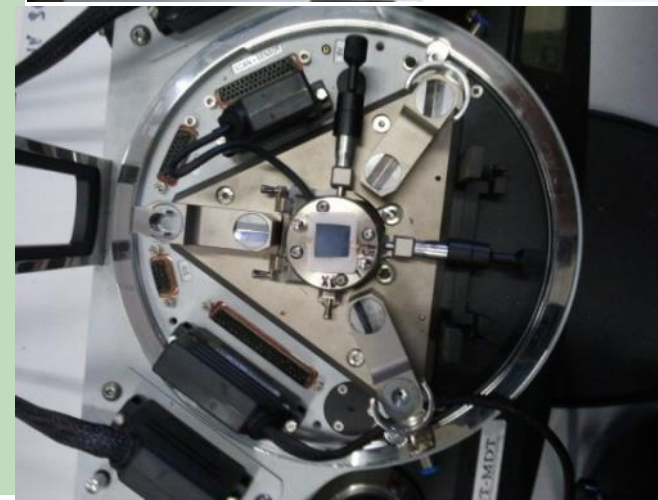
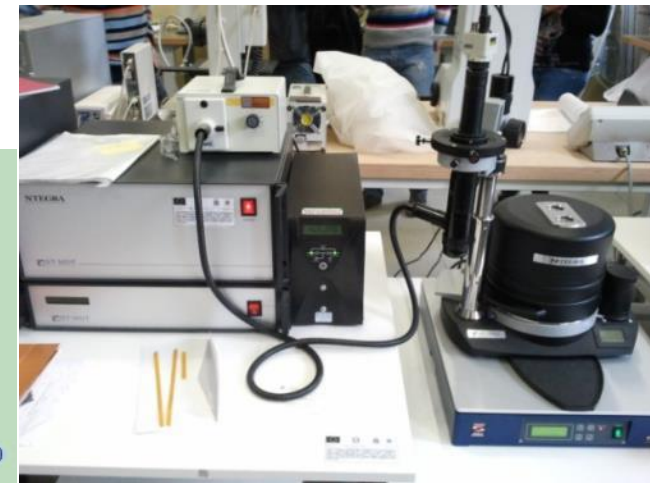
Water erosion effects: surface roughness measurement

A surface roughness measurement was carried out before and after the water erosion test on Polyimide:

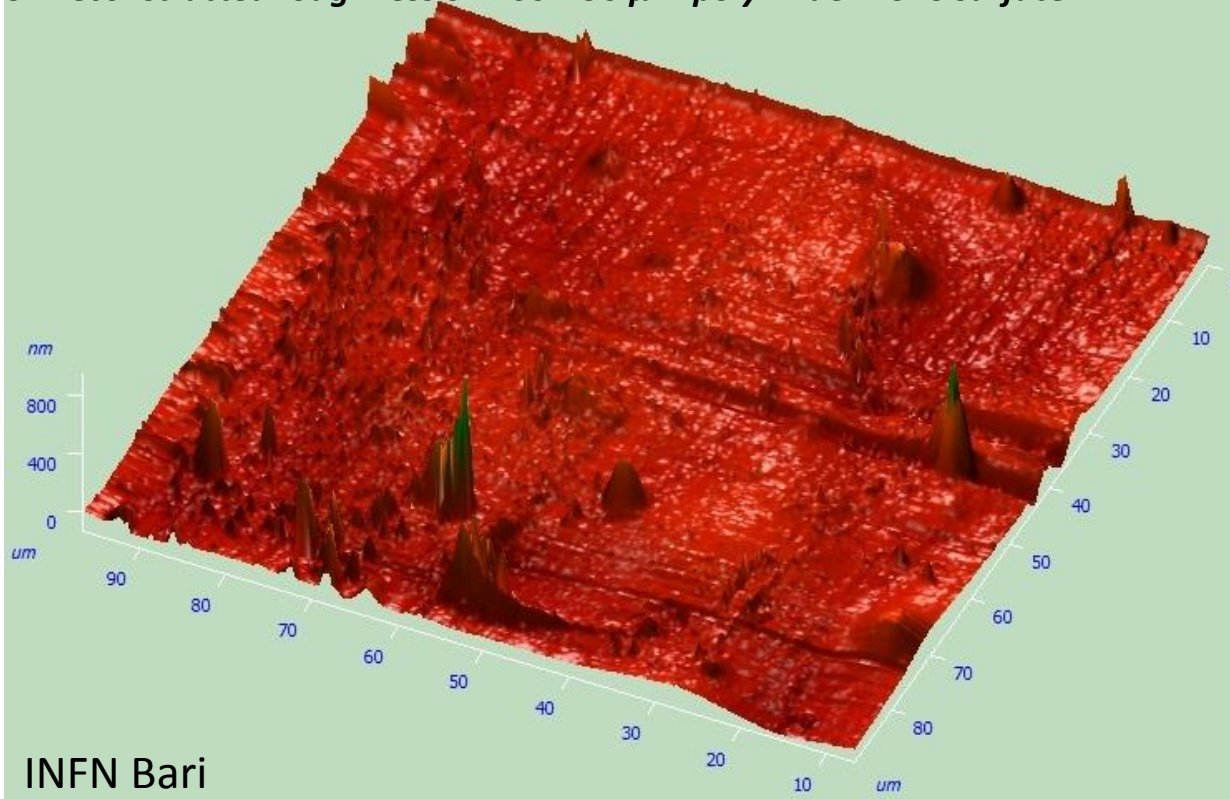
The statistical average roughness, on an area of  $100 \times 90 \mu\text{m}^2$  with  $256 \times 256$  points is **34.94 nm**.

Measure after 10 months of testing :  
**no difference**  
on the average value

Atomic Force Microscope



*3D reconstructed roughness on  $100 \times 90 \mu\text{m}^2$  polyimide MCHS surface*



# **STAVE Mechanics and Cooling**

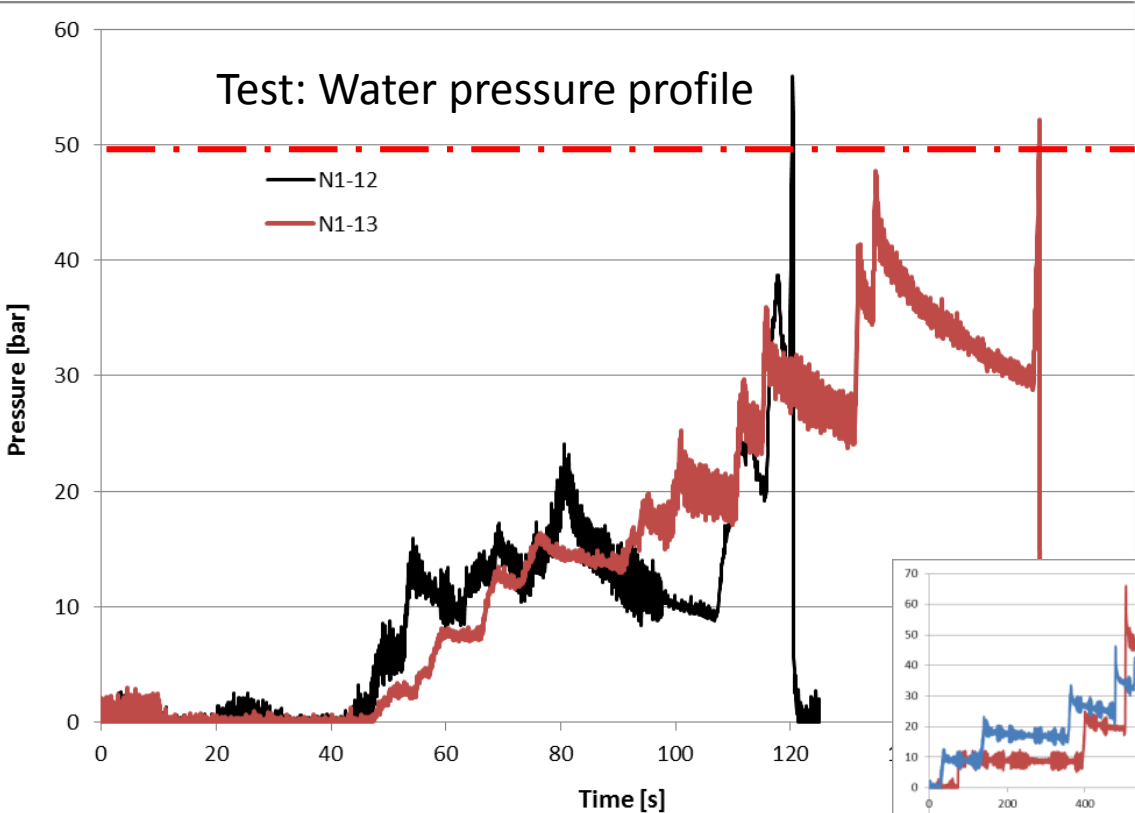
✓ **mechanical characterization**

**Pressure test**



# Pressure test

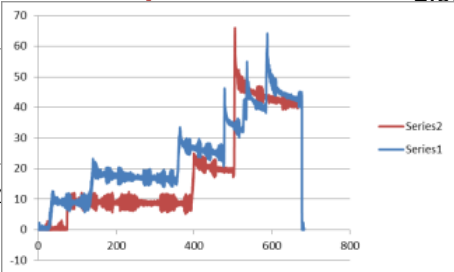
Tests result , Burst Pressure  **$p \geq 50$  bar** for both prototypes  
 Failure occurred on both samples at pipe, not at the connectors



by design

Polyimide Tubing Pressure resistance

ID [mm]	Wt [mm]	P [bar]
1,024	0,0254	40
1,450	0,032	35
2,050	0,032	25
2.667	0,0635	38

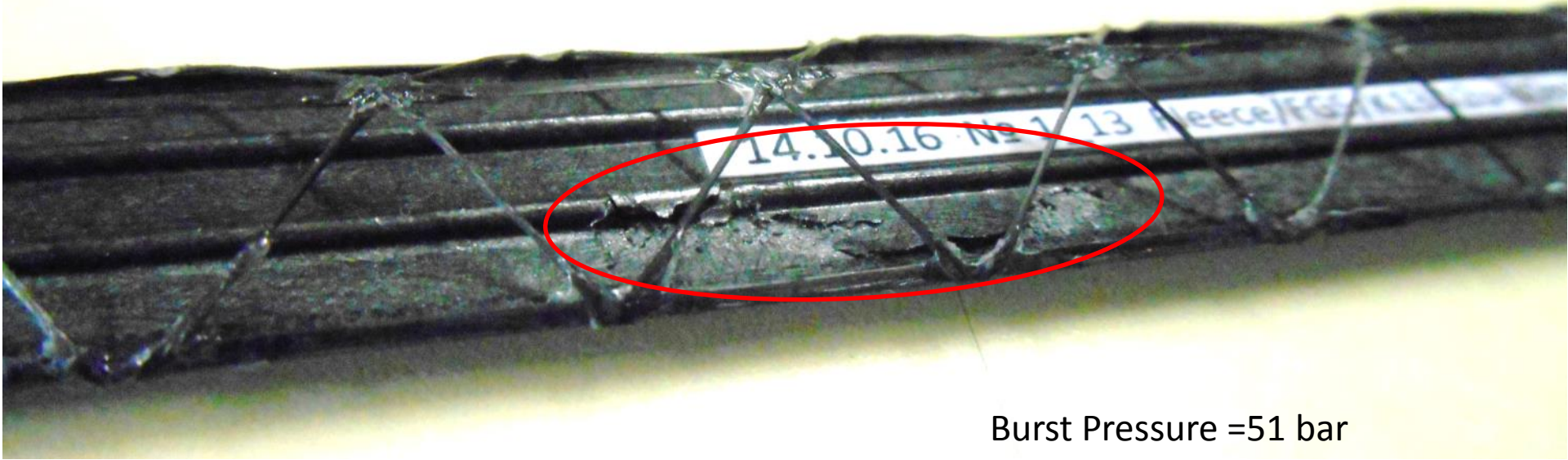


Closed circuit= pressure losses due to internal pump circuit

# Pressure test



failure modes



Burst Pressure =51 bar

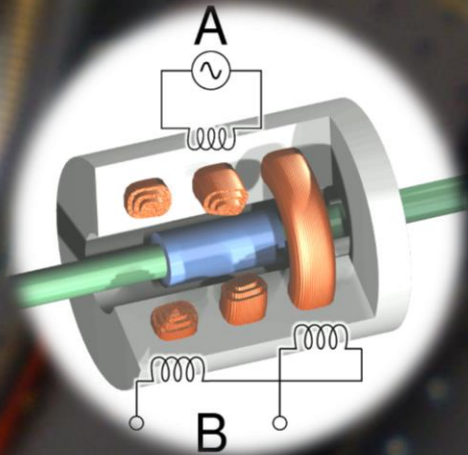
# STAVE Mechanics and Cooling

- ✓ **thermo elastic characterization**

# thermoelastic characterization

## measured by LVDT

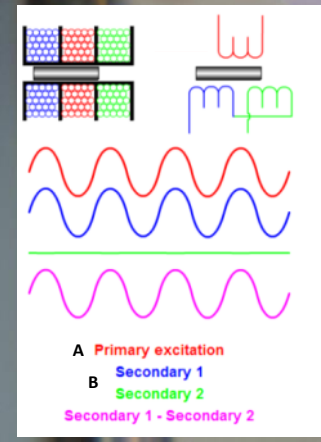
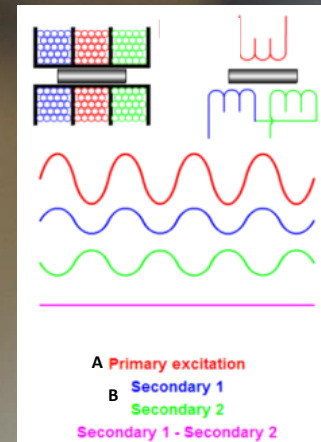
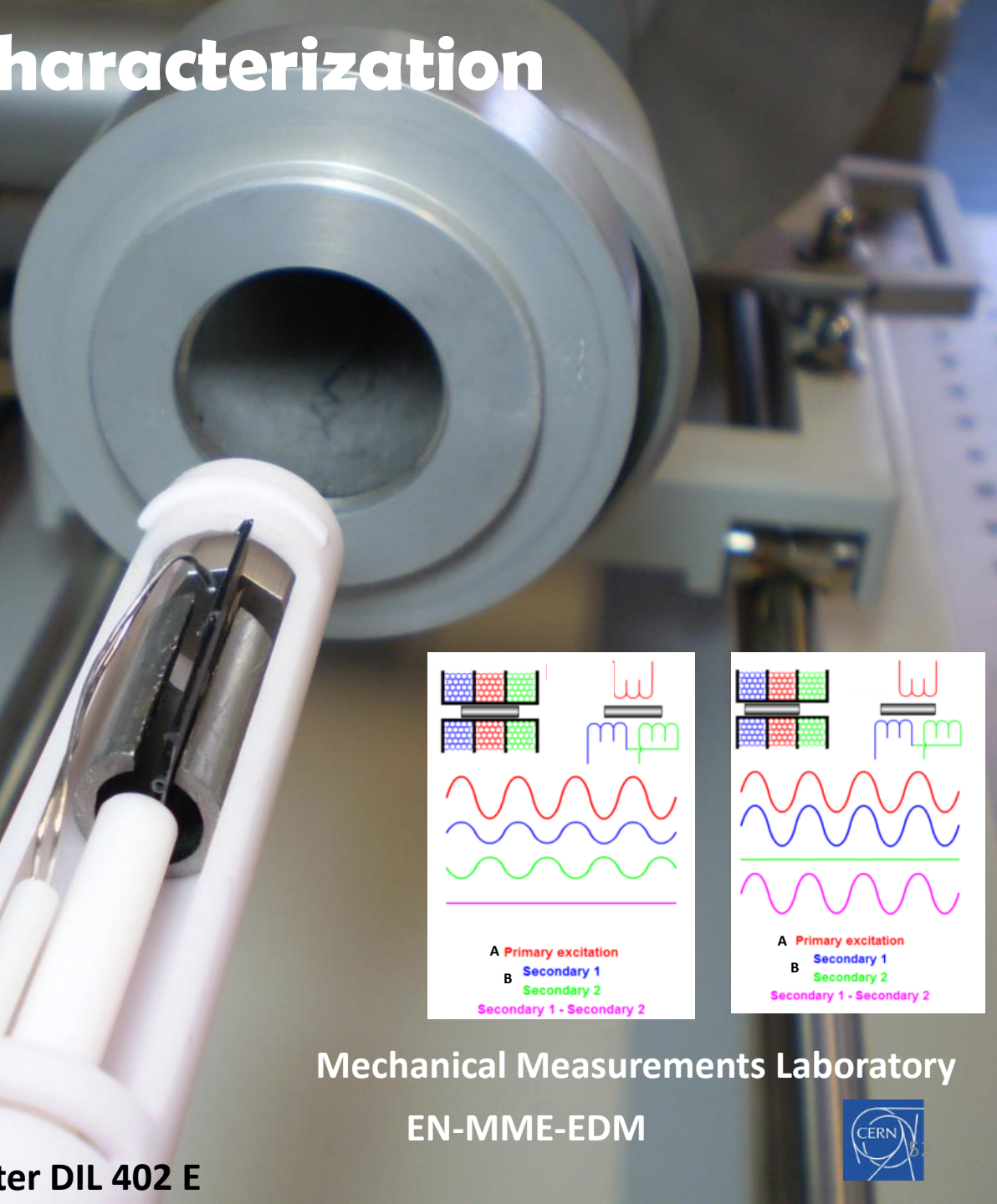
An LVDT Displacement Transducer comprises a primary (A) and two secondaries (B) coils



The transfer of current between the primary and the secondaries of the LVDT displacement transducer is controlled by the position of a magnetic core

The magnetic core is in contact with the specimen.

Specimen thermoelastic deformation induce a displacement of the magnetic core



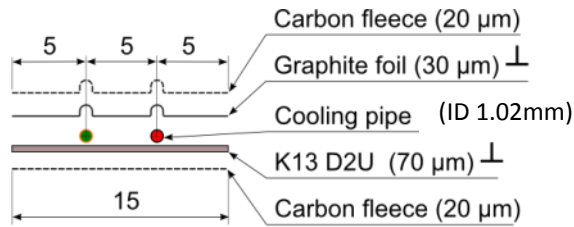
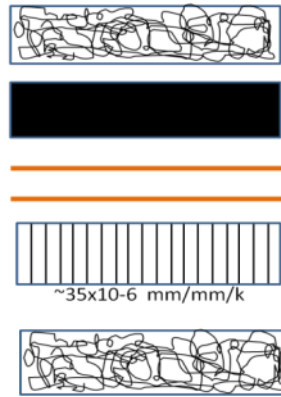
Mechanical Measurements Laboratory

EN-MME-EDM

Dilatometer DIL 402 E

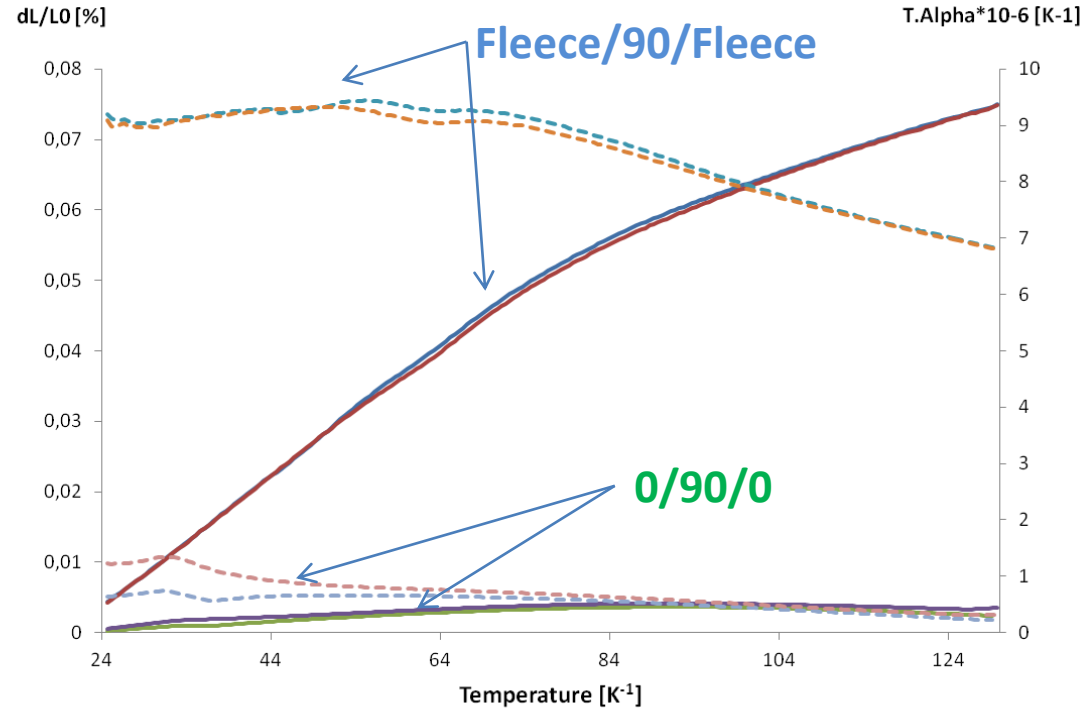


# thermoelastic characterization

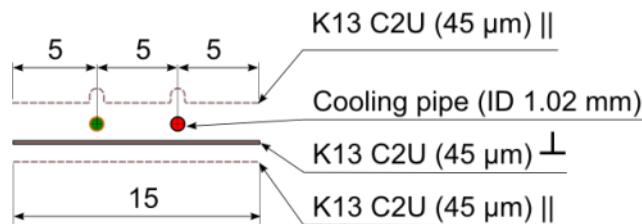
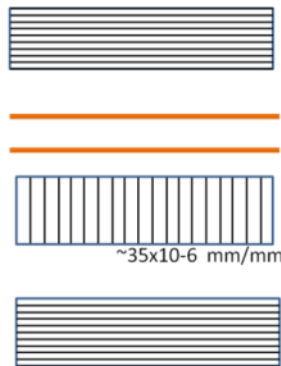


Baseline stave layout  
 $\text{CTE} < 10 \cdot 10^{-6} [\text{K}^{-1}]$

Fleece/90/Fleece



0/90/0



Alternative option to reduce CTE  
 $\text{CTE} = 0 \cdot 10^{-6} [\text{K}^{-1}]$

# **STAVE**

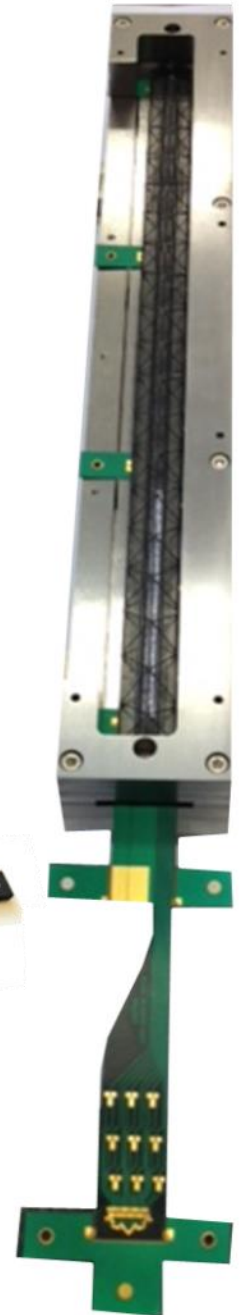
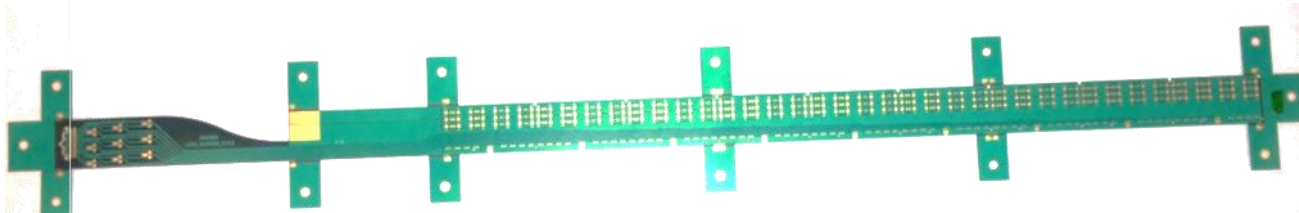
## **assembly**

# Stave assembly

## HIC gluing to spaceframe

*HIC=FPC+ soldered dummy CHiPs with pads*

- ✓ **Precise alignment** defined by accurate jig and assembly procedures

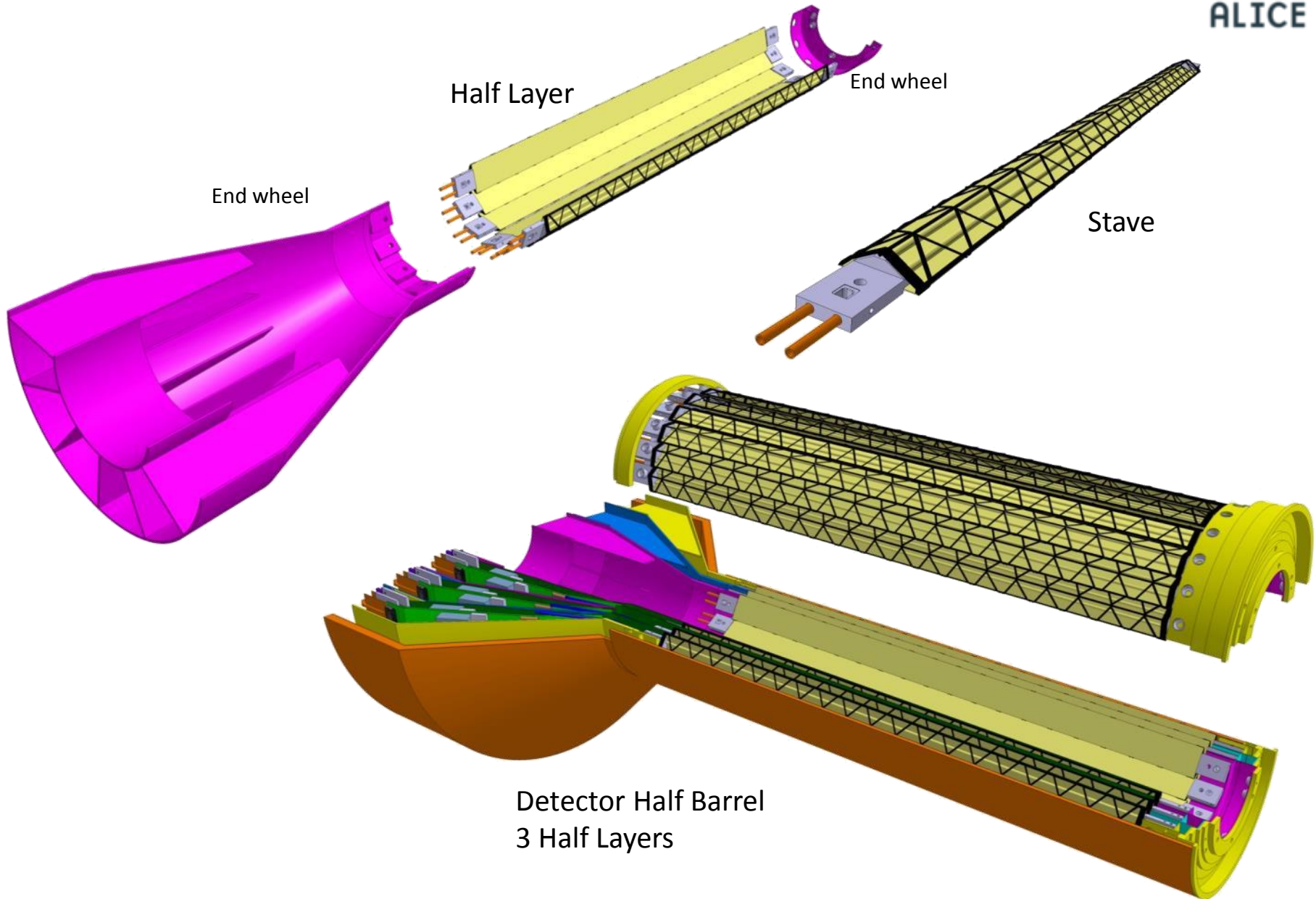


- ✓ **non-rigid bond** preferred to minimize possible structure bending deformation due to differential thermal deformation  
Eccobond 45/ catalyst 15 (100/100) semirigid formulation

# **LAYERS and BARRELS**



# Inner Barrel layers and barrel



Half Layer

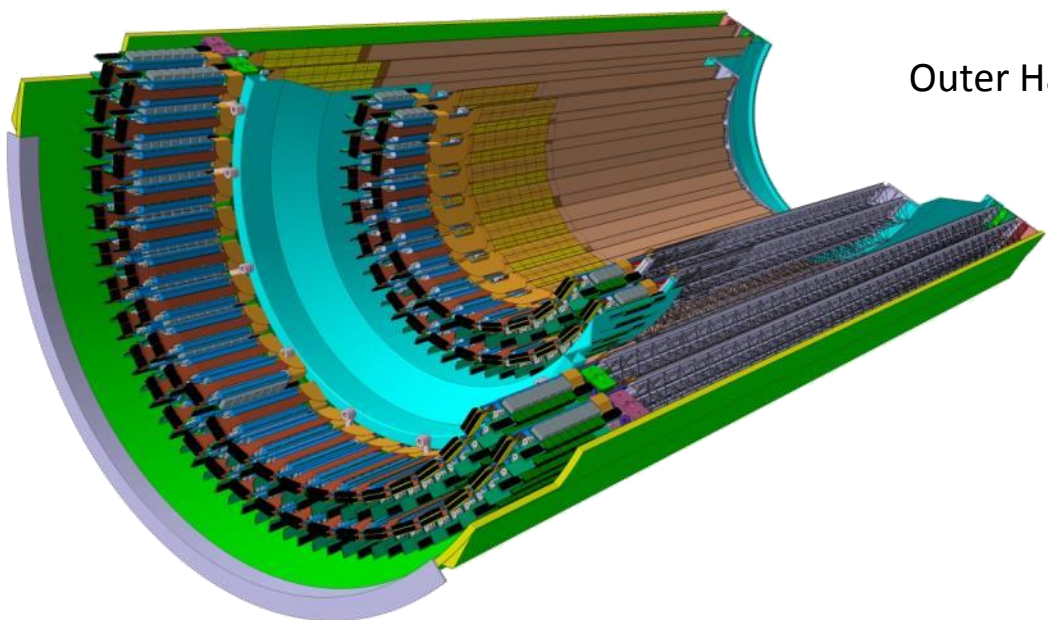
End wheel

End wheel

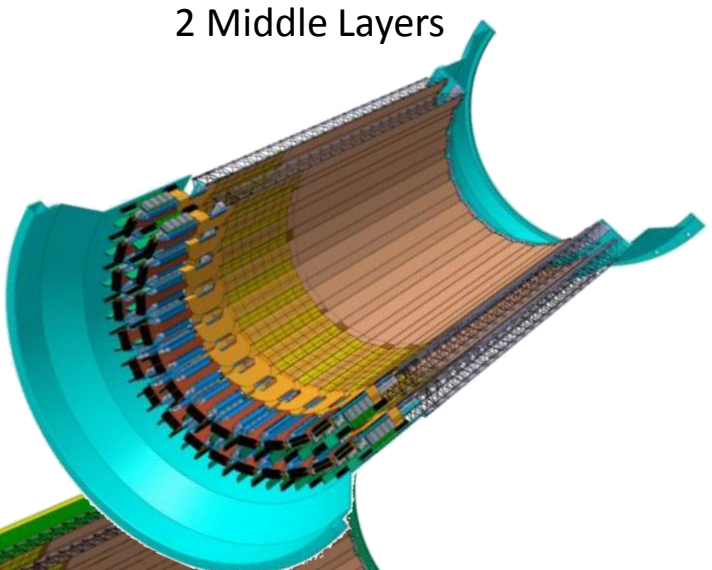
Stave

Detector Half Barrel  
3 Half Layers

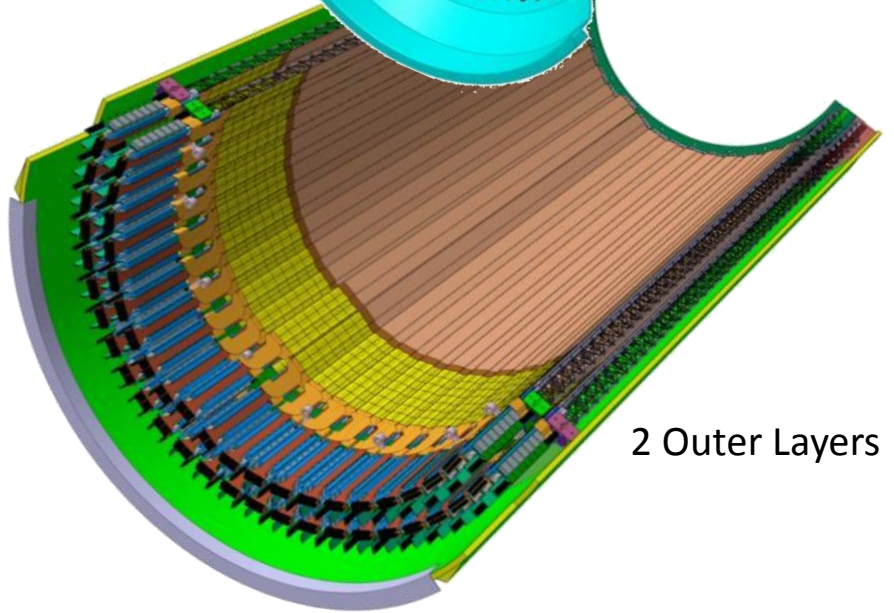
# Outer Barrel layers and barrel



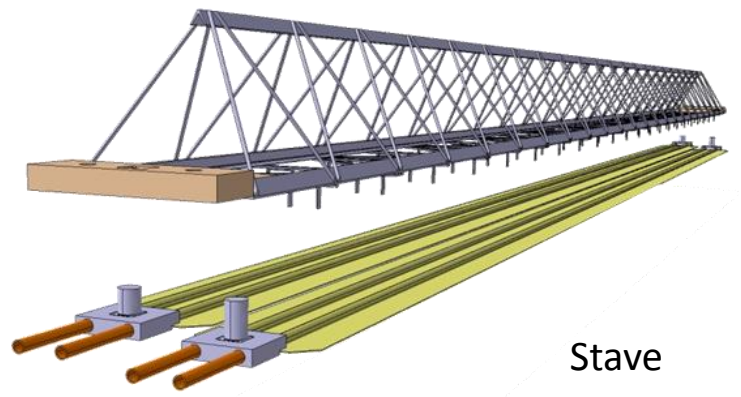
Outer Half Barrel



2 Middle Layers

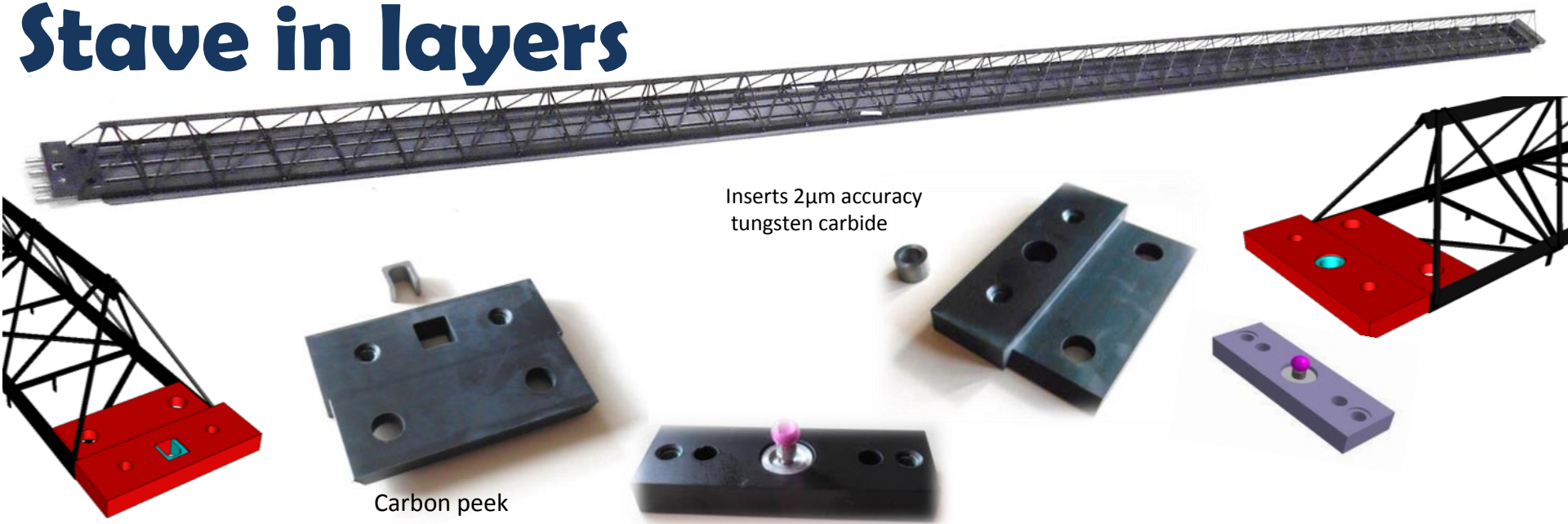


2 Outer Layers



Stave

# Stave in layers



Inserts 2 $\mu$ m accuracy tungsten carbide

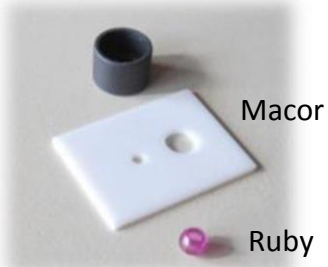
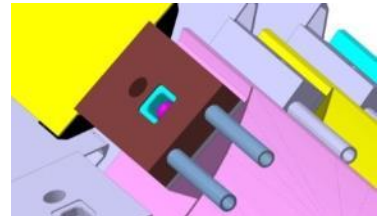
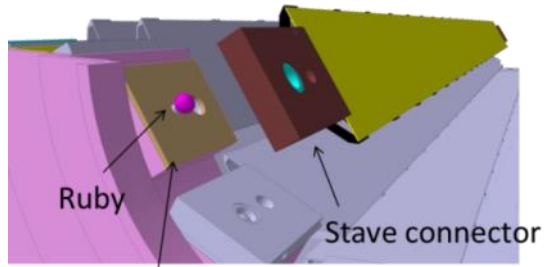
Carbon peek

## Outer Barrel stave

---

## Inner Barrel stave

## Precise locating ruby sphere and inserts



Macor

Ruby



Carbon peek

Inserts 2 $\mu$ m accuracy tungsten carbide



# IB layers and barrel prototype

Cylindrical Structural Shell



Layer 0



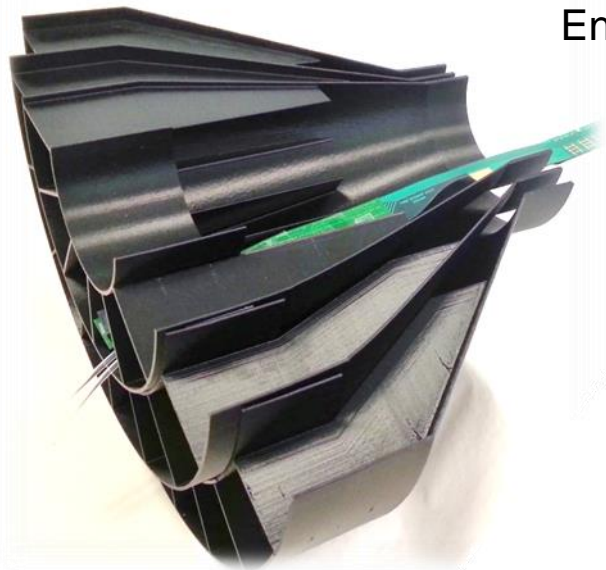
Layer 1



Layer 2



End wheel



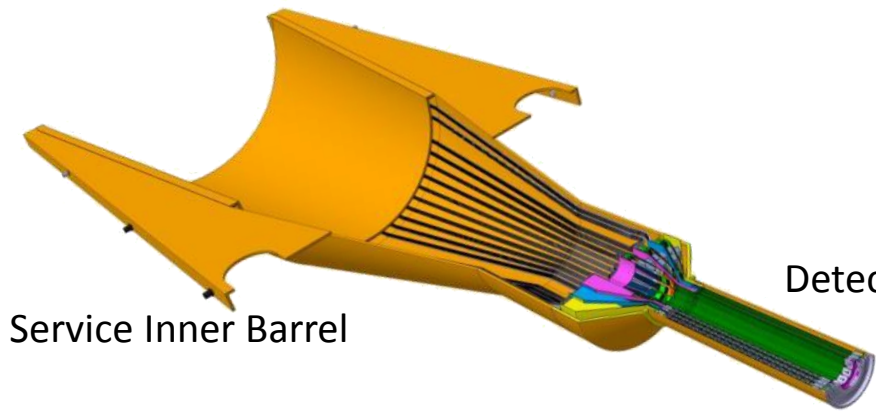
Half Barrel



Stave

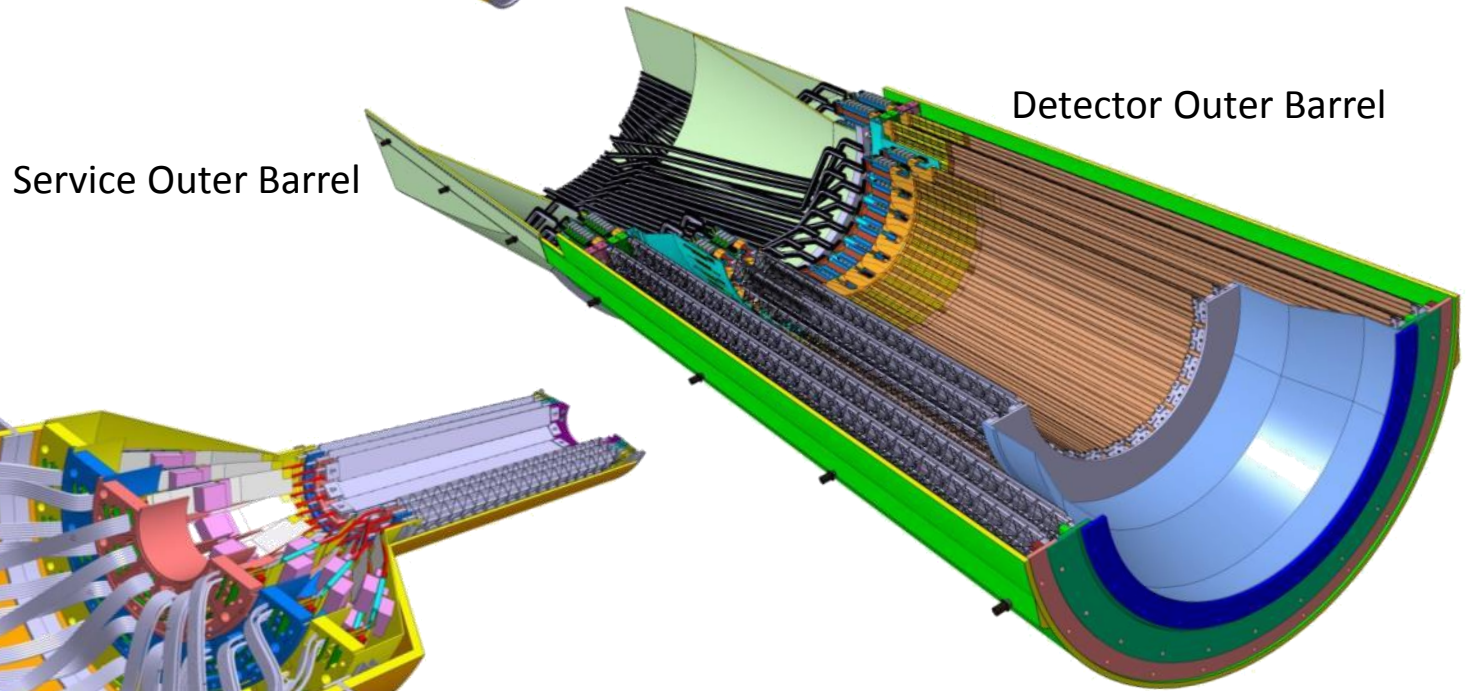


# Service barrels



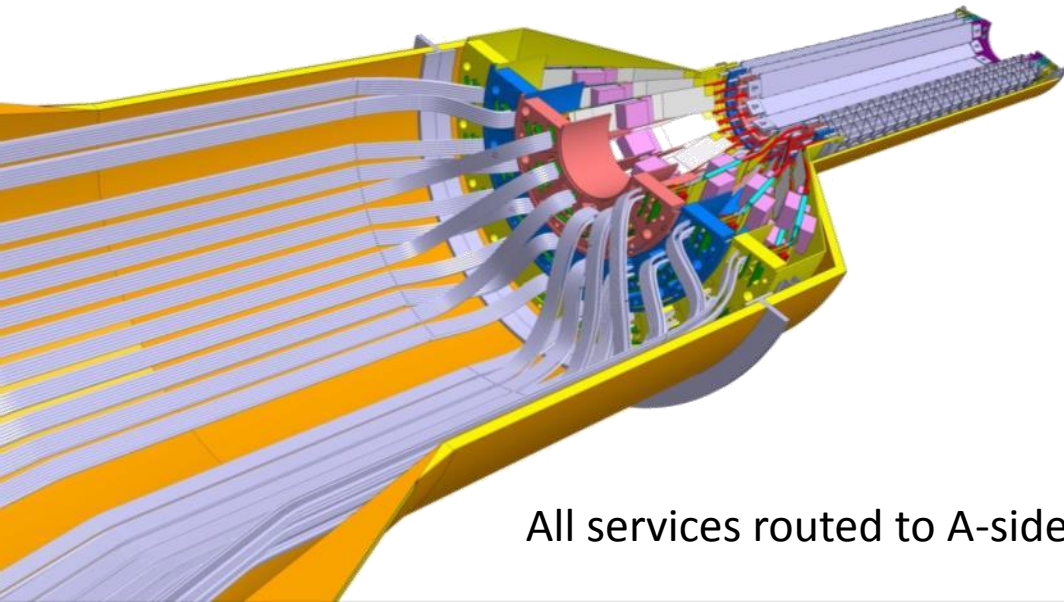
Service Inner Barrel

Detector Inner Barrel



Service Outer Barrel

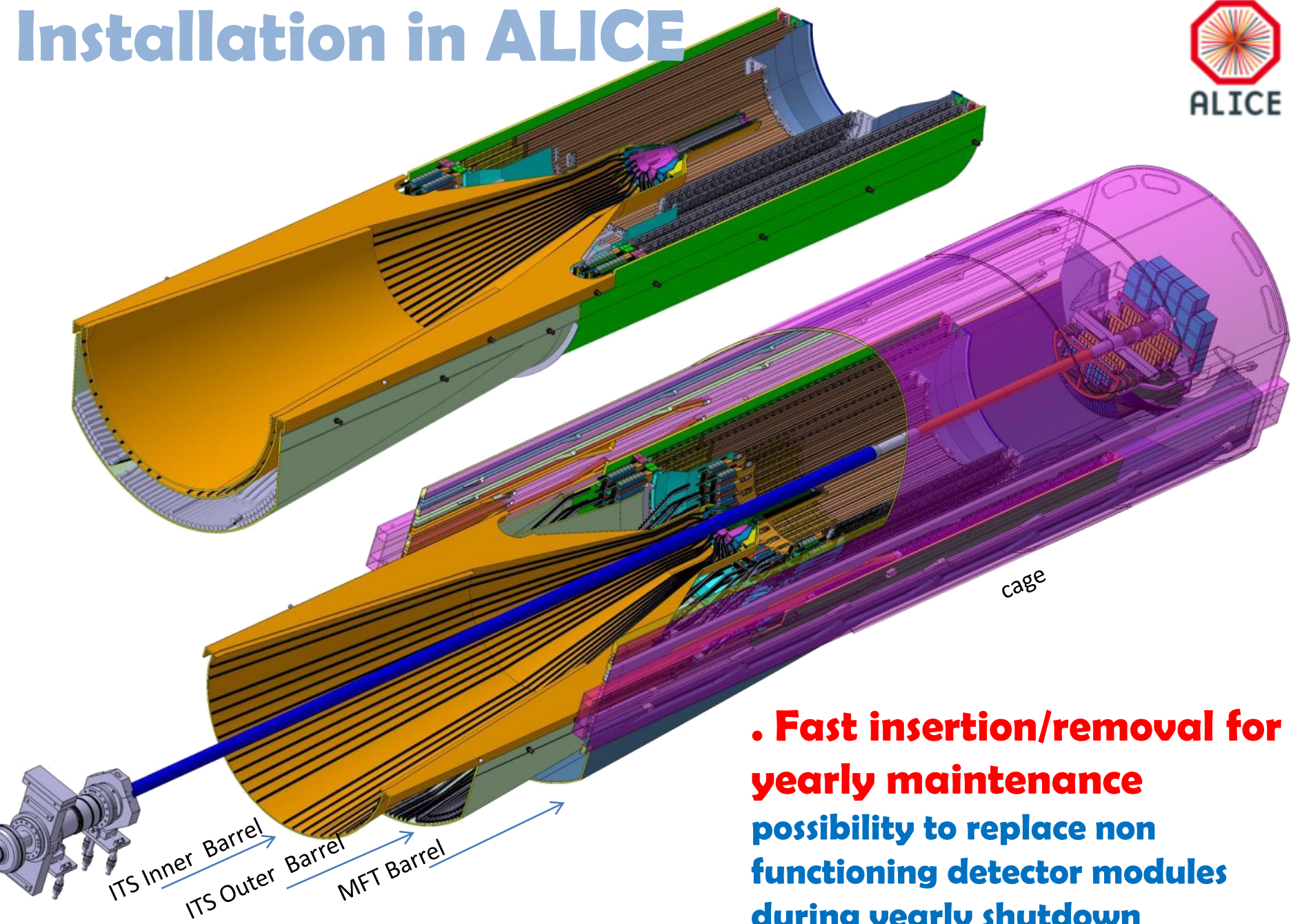
Detector Outer Barrel



All services routed to A-side

# **Installation in ALICE**

# Installation in ALICE



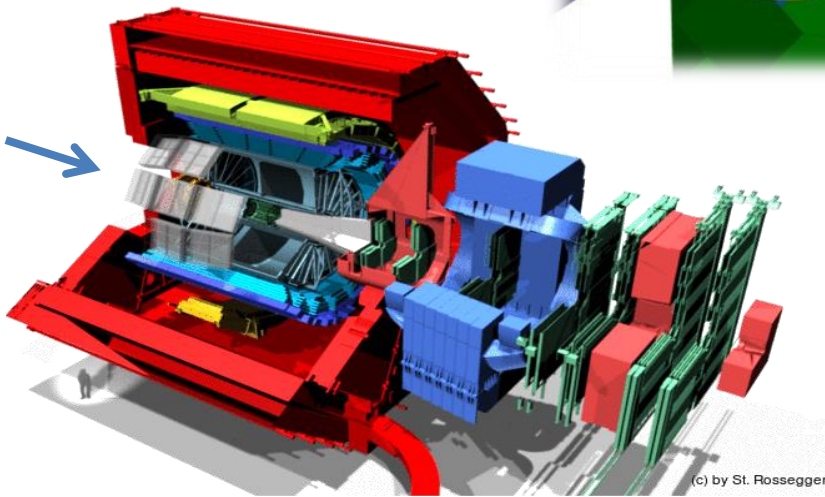
**. Fast insertion/removal for yearly maintenance**  
**possibility to replace non functioning detector modules during yearly shutdown**

# Installation in ALICE

- The cage is inserted in the TPC bore  
The cage support in position the beam pipe

- ITS Outer Barrel, and ITS Inner Barrel are inserted in the cage, one half barrel at the time

- The detector barrels and the BP relative position is guaranteed by the cage

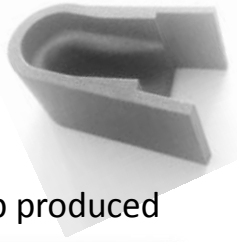
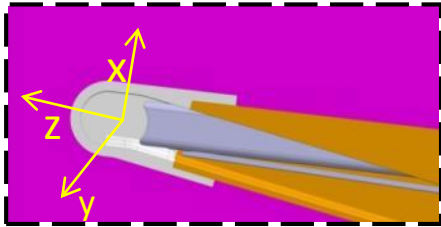
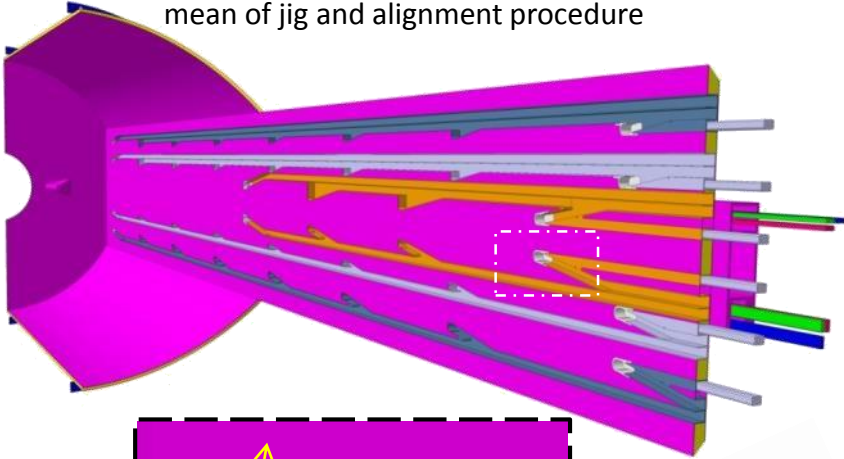


(c) by St. Rossegger

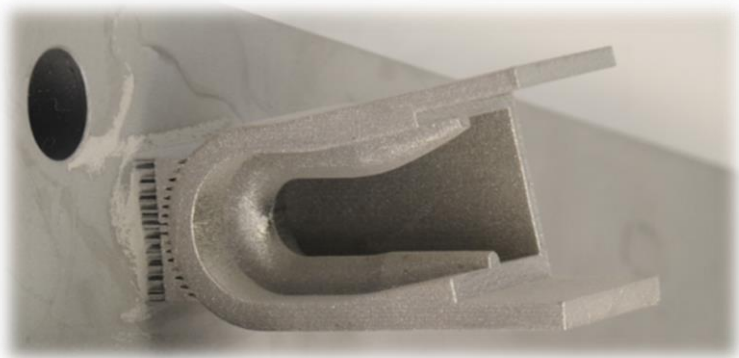


# Cage

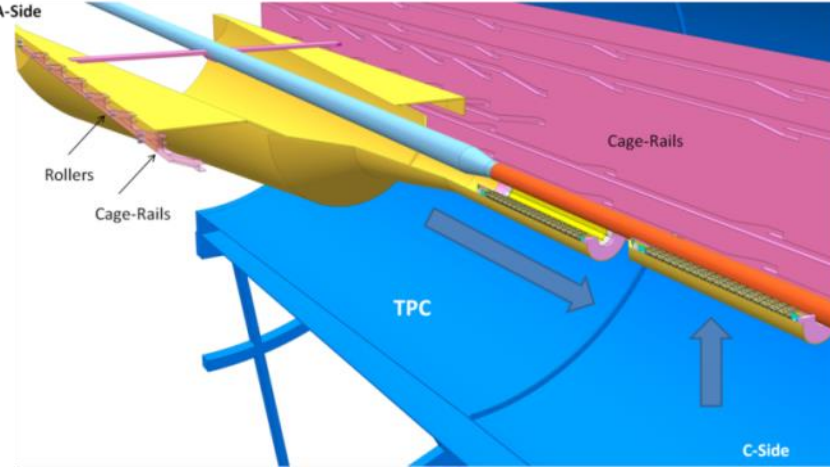
**rails** *Rails end-stop*, providing detector final position, are glued in the cage in a precise relative position by mean of jig and alignment procedure



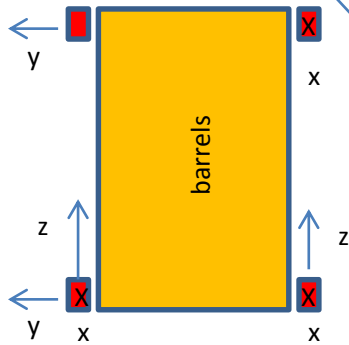
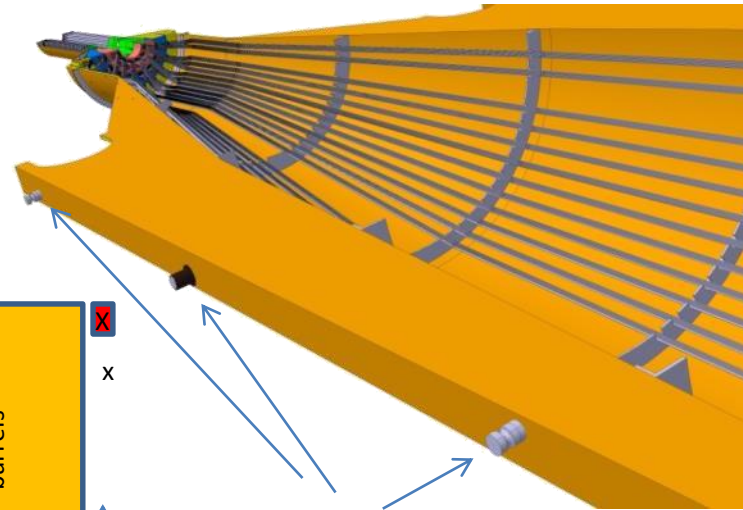
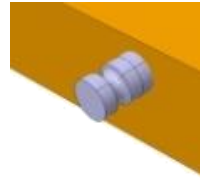
3D Printed prototype of rail end stop produced

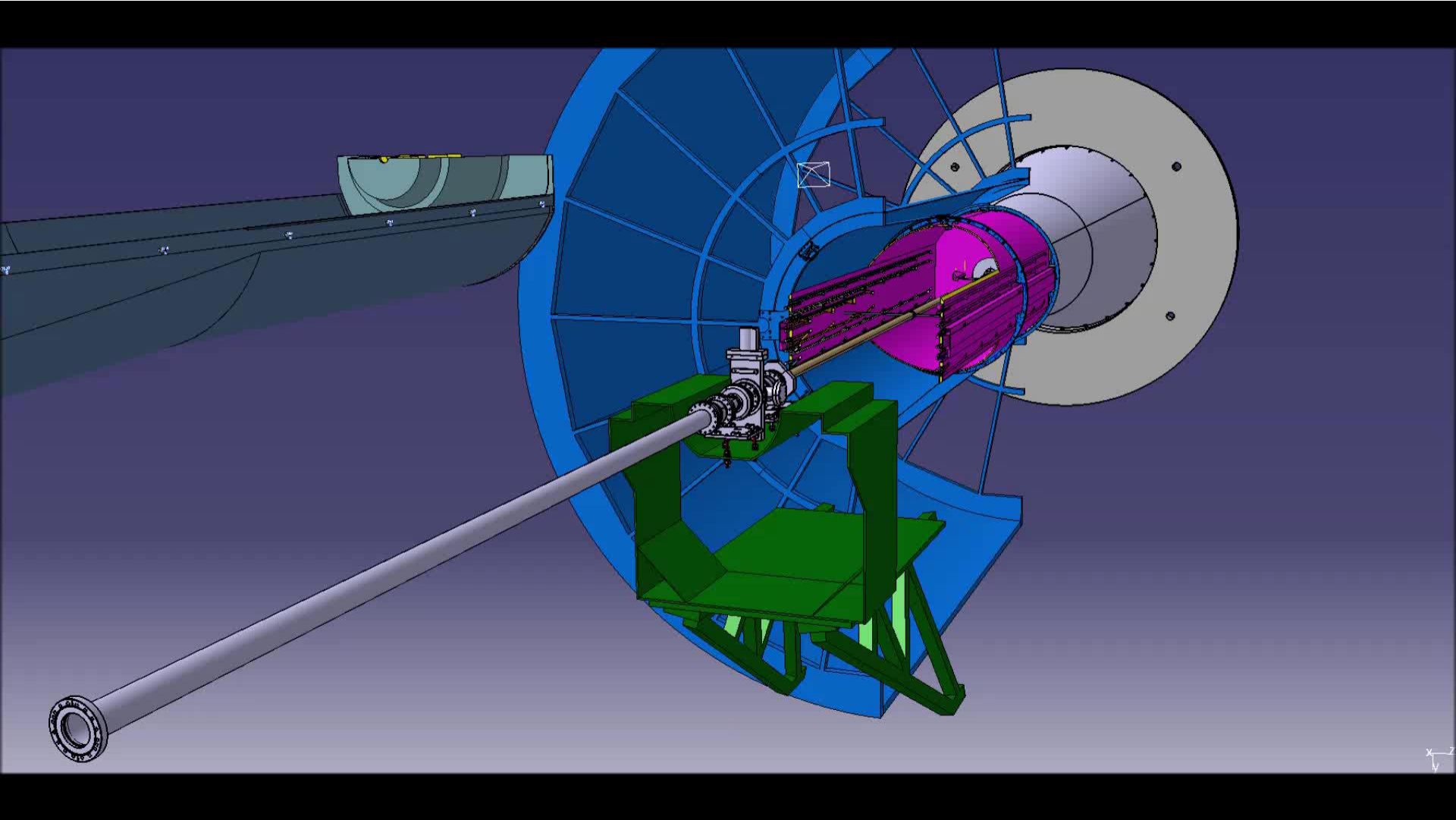


prototype: Titanium CL42Ti



**rollers** *Rollers*, engaging Cage rails, are aligned respect to the Detectors.  
4 primary rollers drive the detector final position.





# Insertion (and Removal)