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# **SRF Development for PIP-II**

## **With focus on 325 MHz Spoke Resonators**

### **(for the common aspects with crab cavities)**

Leonardo Ristori

HL-LHC Crab Cavity Cryomodule Review

Cern - 11 November 2015

# Proton Improvement Plan II (PIP-II) mission

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## ☐ Particle Physics Project Prioritization Panel (P5) Science Drivers:

- Use the Higgs Boson as a New Tool for Discovery
- Pursue the Physics Associated with Neutrino Mass
- Identify the New Physics of Dark Matter
- Understand Cosmic Acceleration : Dark Energy and Inflation
- Explore the Unknown : New Particles, Interactions, and Physical Principles



## ☐ Proton Improvement Plan II (PIP-II):

The PIP-II goal is to support long-term physics research goals by providing increased beam power to neutrino experiments, while providing a platform for the future.

# Proton Improvement Plan II (PIP-II) mission and strategy

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- Increase Main Injector power
  - from 700 kW (NOvA) to >1 MW (LBNF)
  - in the energy range 60 – 120 GeV
- Increase Booster power from 80 to 160 kW
  - 8 GeV program: SBNE, ...

⇒ Roadmap for CD-3 in FY19/20

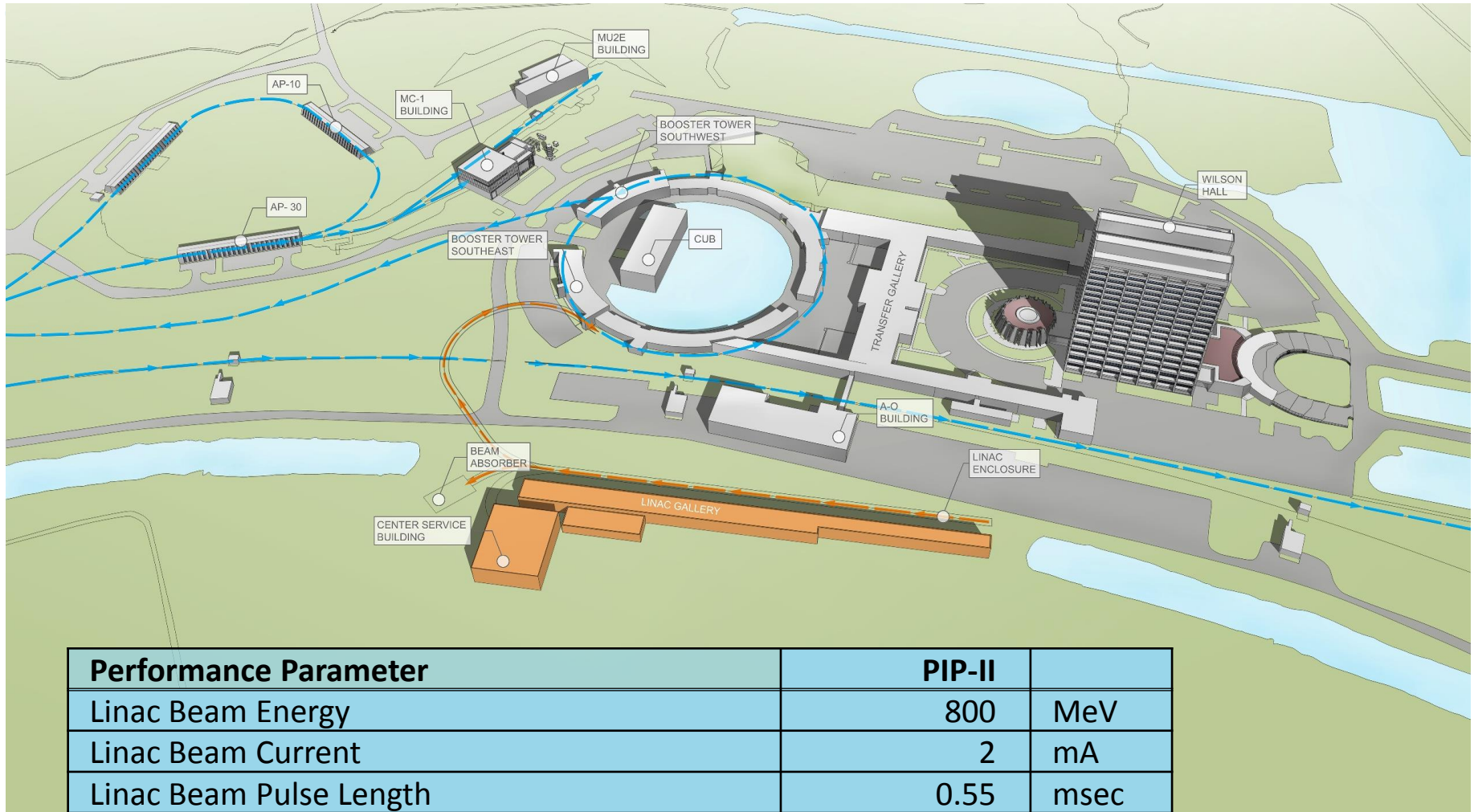
⇒ Construction phase is 5 years: 2019-23

⇒ Goal is 1 MW in 2024

**The diamond of PIP-II is a state-of-the-art SRF Linac**

Diamonds are beautiful but difficult to make

# PIP II SC Linac Requirements



Performance Parameter	PIP-II	
Linac Beam Energy	800	MeV
Linac Beam Current	2	mA
Linac Beam Pulse Length	0.55	msec
Linac Pulse Repetition Rate	20	Hz
Linac Beam Power to Booster	18	kW

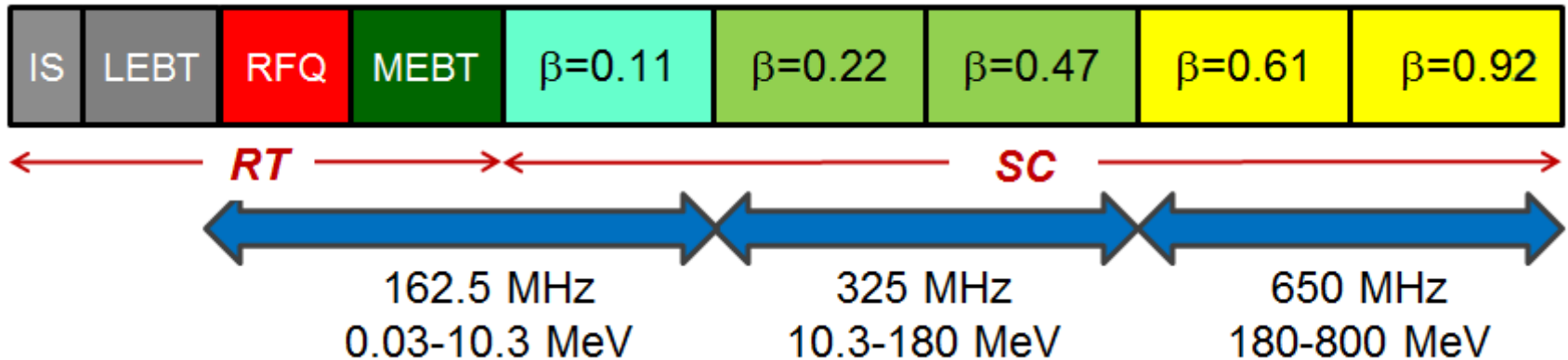


# The Linac Reference Design

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- The reference design is ready:
- Frequency choice: sub-harmonics of 1.3 GHz
  - 162.5 MHz, 325 MHz and 650 MHz;
- RF cavity types and betas:
  - one section of 162.5 MHz HWR type,  $\beta = 0.11$  cavity,
  - two sections of 325 MHz spoke-cavity type, SSR1 and SSR2 with  $\beta = 0.22$  and  $\beta = 0.47$ ; and
  - two sections of elliptical 650 MHz cavities with  $\beta = 0.61$  and  $\beta = 0.92$ ;
- Break points are optimized in order to minimize the number of the cavities;
- CM concept:
  - separate CMs,
  - solenoids for HWR and SSR,
  - no focusing elements for elliptical.
- Operating regimes – both pulsed and CW;
- No HOM dampers.

# The Linac Reference Design



Section	Freq	Energy (MeV)	Cav/mag/CM	Type
RFQ	162.5	0.03-2.1		
HWR ( $\beta_{opt}=0.11$ )	162.5	2.1-10.3	8/8/1	HWR, solenoid
SSR1 ( $\beta_{opt}=0.22$ )	325	10.3-35	16/8/ 2	SSR, solenoid
SSR2 ( $\beta_{opt}=0.47$ )	325	35-185	35/21/7	SSR, solenoid
LB 650 ( $\beta_g=0.61$ )	650	185-500	33/22/11	5-cell elliptical, doublet*
HB 650 ( $\beta_g=0.92$ )	650	500-800	24/8/4	5-cell elliptical, doublet*

\*Warm doublets external to cryomodules

**All components CW-capable**

# The Linac Reference Design

Name	beta	Freq (MHz)	Type of cavity	$B_{\text{peak}}$ (mT)	$E_{\text{peak}}$ (MV/m)	$E_{\text{acc}}$ (MV/m)	Gain (MeV)
HWR	0.11	162.5	Half wave resonator	48.3	44.9	9.7	2.0
SSR1	0.22	325	Single-spoke resonator	58.1	38.4	10	2.0
SSR2	0.47	325	Single-spoke resonator	64.5	40	11.4	5.0
LB650	0.61	650	Elliptic 5-cell	72	38.5	15.9	11.9
HB650	0.92	650	Elliptic 5-cell	72	38.3	17.8	19.9

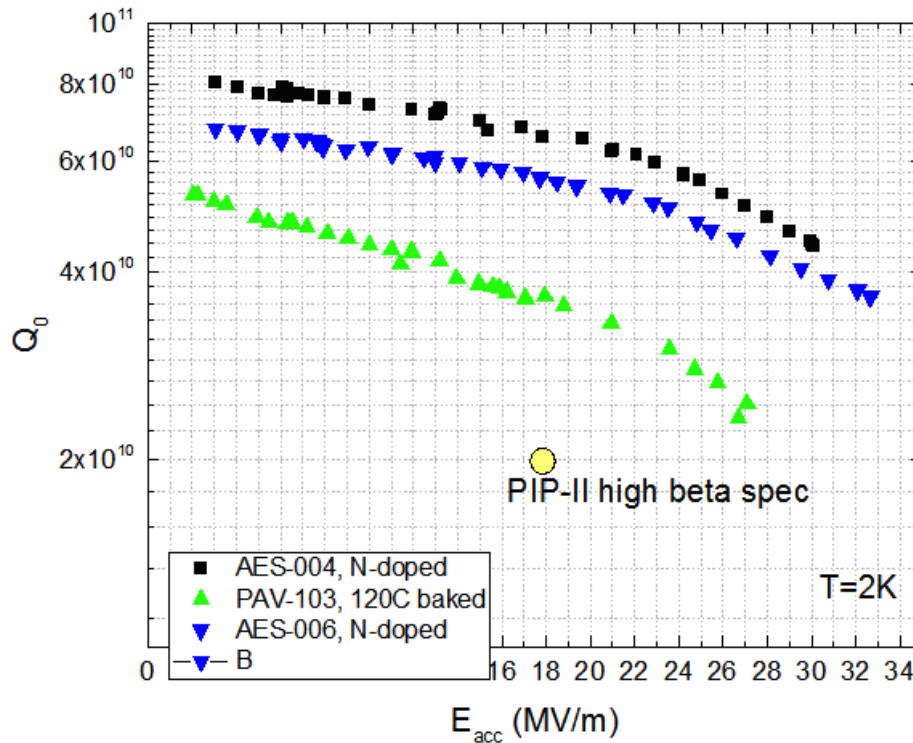
# The main challenges and technical risks

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- Future CW operation → cryo-losses → high  $Q_0$  is desired;
- Low beam loading → narrow bandwidth;
  - Pulsed regime → Lorentz Force Detune (LFD);
  - CW regime → microphonics;
- High-Order Modes → “to damp, or not to damp?”
  
- High- $Q_0$  program was initiated and is running successfully;
- Resonance Control program is underway in order to mitigate both microphonics and LFD;
- “Passive” mitigation of the cavity detune – improvement of cavity mechanical properties is underway;
- Detailed HOM analysis is performed.

# High Q0 R&D program

- Results – highlights – 120C bake versus N doping  
Q~ 7e10 at 2K, 17 MV/m – world record at this frequency!
- Applying N doping to 650 MHz (beta=0.9) leads to double Q compared to 120C bake (standard surface treatment ILC/XFEL)



A. Grassellino, MOYGB2, IPAC15, Richmond; **MOBA06, MOPB029, MOPB091**

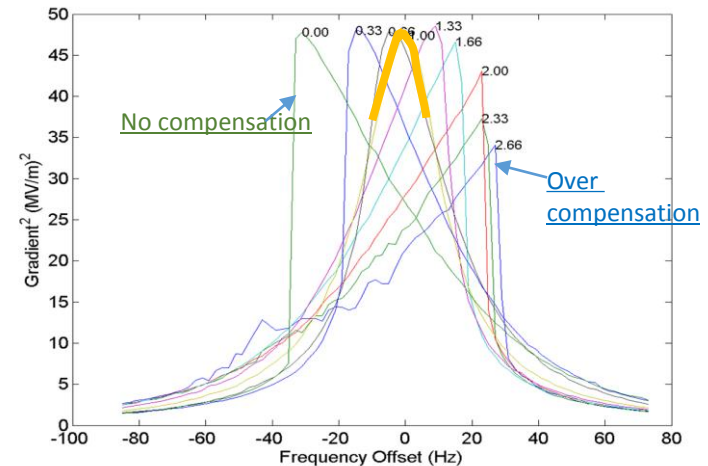
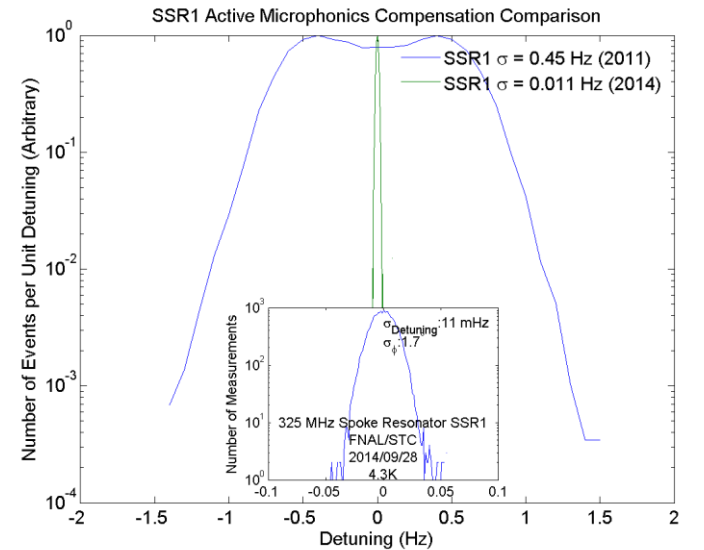
# Resonance Control R&D program

- Piezo feedback has successfully stabilized the resonance with high precision in CW to negligible levels (11 mHz RMS)
- Ponderomotive instability has been successfully mitigated using piezo feedforward tied to the square of the gradient during both CW and pulsed operation
- Adaptive feedforward has successfully suppressed detuning from deterministic sources of detuning
- Techniques for fully characterizing the tuner-cavity-waveguide system automatically have been developed and used successfully

Poster TUPB095

"Resonance Control for Narrow Bandwidth SRF Cavities"

(W. Schappert, Yu. Pischalnikov, J. Holzbauer)



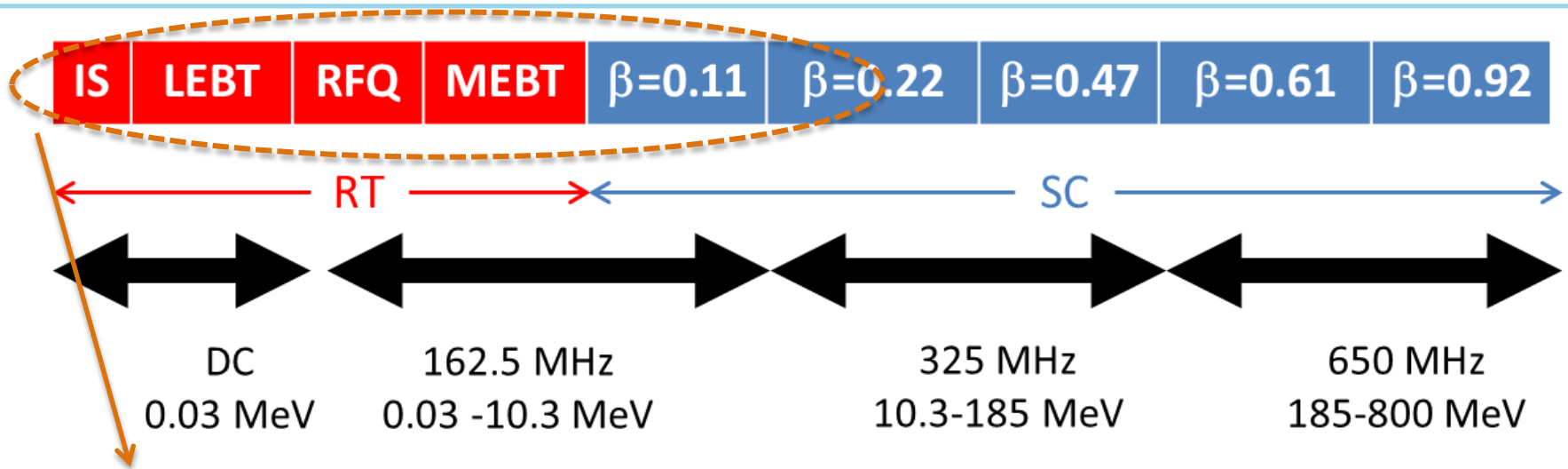
# General design approach

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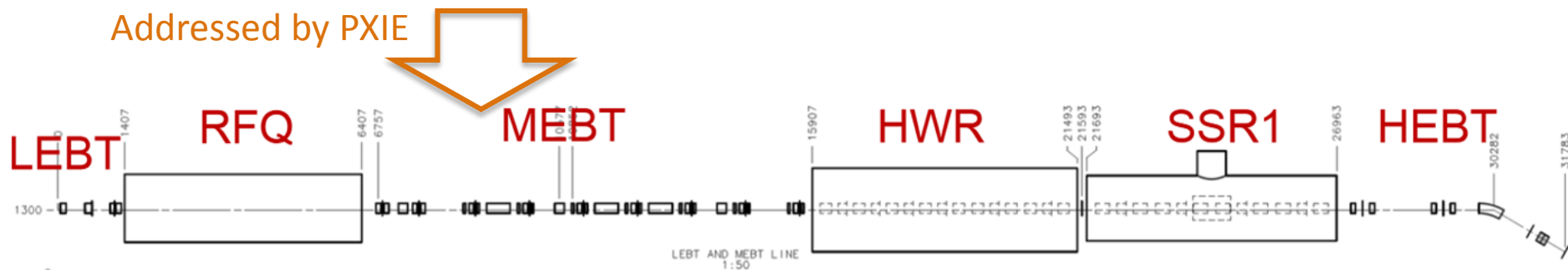
- Most components (couplers, tuners, etc.) should be of the same or similar type;
- Cryomodules should be preferably of the same type and contain mostly the same parts;
- Two types of CMs are to be prototyped,
  - spoke-cavity CM for SSR1 and
  - elliptical cavity CM for HB 650.
- Other CMs will be developed basing on the lessons learned for these CMs.



# PXIE: R&D program for PIP-II front end



Addressed by PXIE



30 keV  
H-, 5 mA  
from 1  $\mu$ s  
to DC

2.1 MeV, CW 4-  
vane, 4-  
module.  
Designed and  
manufactured at  
LBNL.

~10 m, with bunch-  
by-bunch selection.  
Magnets are being  
made by BARC,  
India.

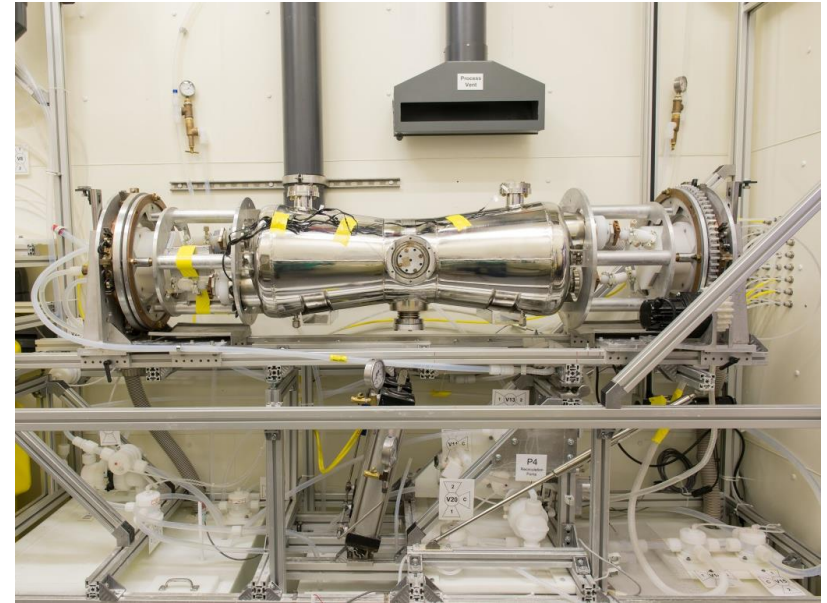
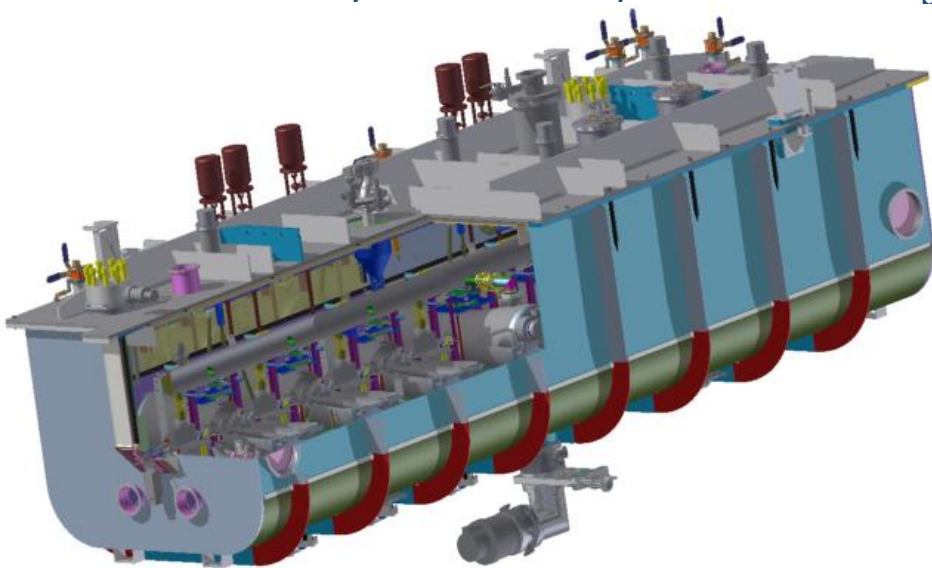
162.5 MHz, 10  
MeV. Designed  
and being  
manufactured  
at ANL.

325 MHz,  
25 MeV.

Diagnostics  
and 2 mA  
CW, 25 MeV  
dump.

# Status of development of critical components. HWR (ANL)

- 2 HWRs were tested with very high performance:
  - residual resistance is  $<2.7$  nOhm at 15 MV/m accelerating field ( $E_{\text{peak}}=70$  MV/m,  $B_{\text{peak}}=75$  mT and voltage = 3.2 MV with the cavity length=0.206m)
  - No X-rays observed up to 70 MV/m  $E_{\text{peak}}$



**Z. Conway, WEBA05 SRF2015 – Whistler Canada**  
“Achieving high peak fields and low residual resistance in half-wave cavities”

# Status of development of critical components, 650 MHz

## 650 MHz section:



Currently Available Cavities:

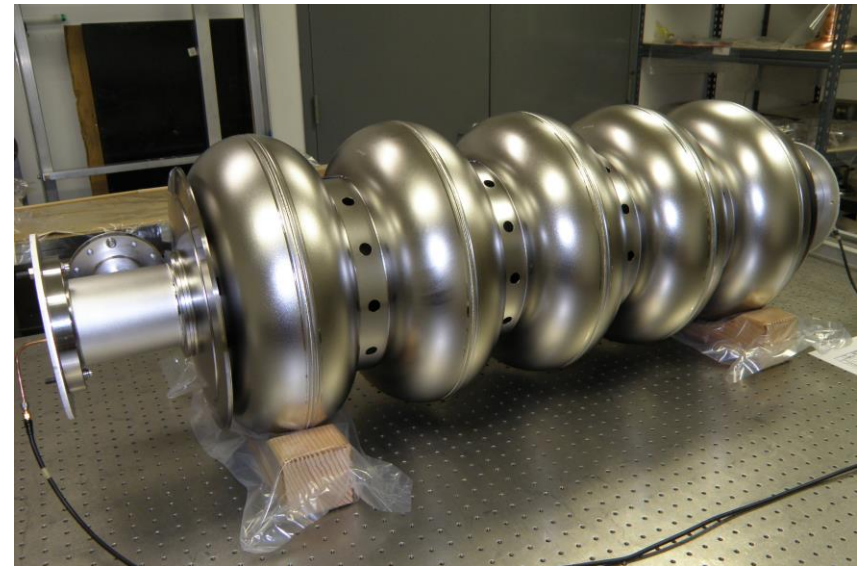
### 1-Cell 650 MHz\*

1. B9AS-AES-001
2. B9AS-AES-002
3. B9AS-AES-003
4. B9AS-AES-004
5. B9AS-AES-005
6. B9AS-AES-006

### 5-Cell 650 MHz

1. B9A-AES-007
2. B9A-AES-008
3. B9A-AES-009
4. B9A-AES-010

\*VTS Tested



Expected Cavities:

### 1-Cell 650 MHz

Pavac, Inc.

Three are delivered  
and VTS-tested

Two to be delivered  
in the end of 2015.

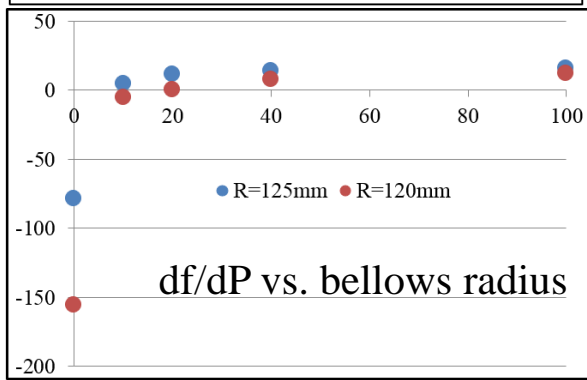
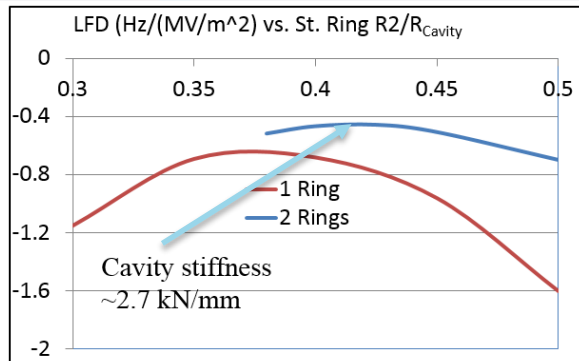
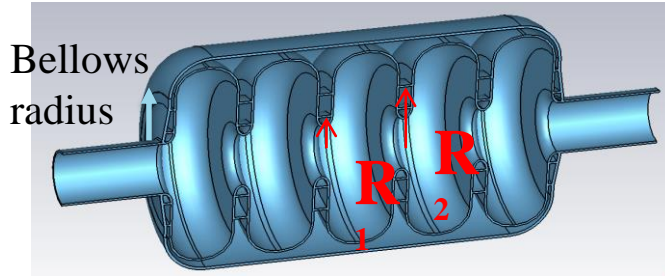
### 5-Cell 650 MHz

Pavac, Inc.

Five to be delivered  
in 2015.

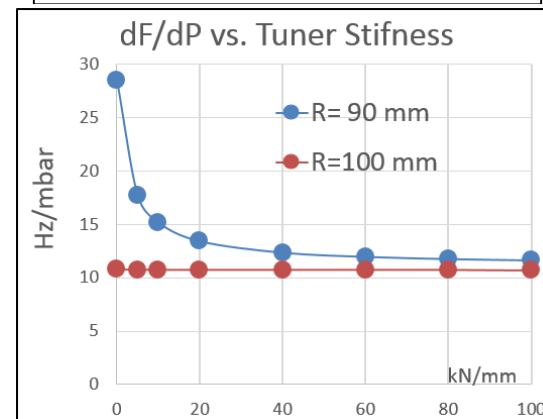
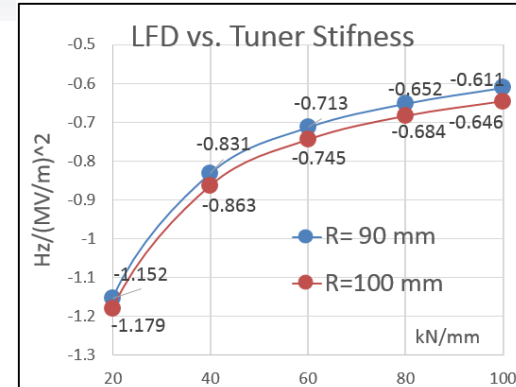
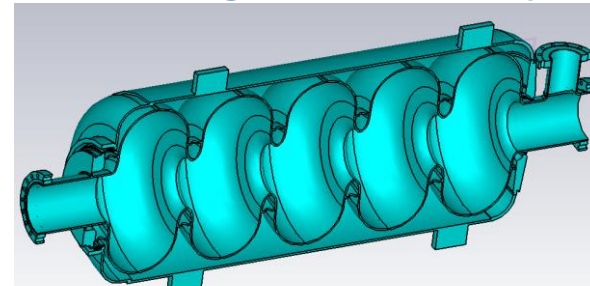
# Status of development of critical components, 650 MHz

## Low-Beta Cavity



Poster THPB014

## High-Beta Cavity

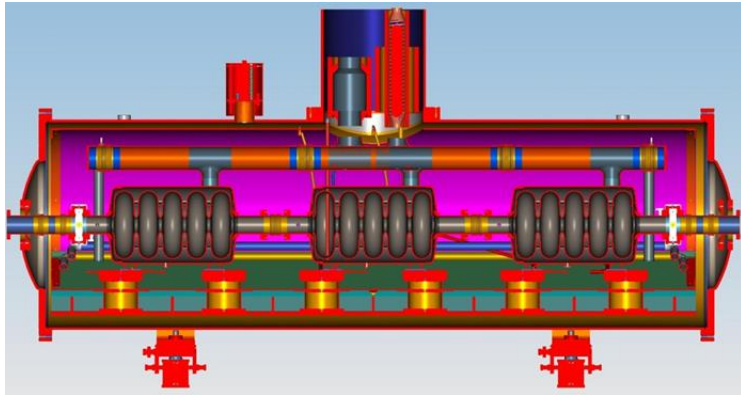




# Status of development of critical components, 650 MHz CMs

## Low-Beta Cryomodule

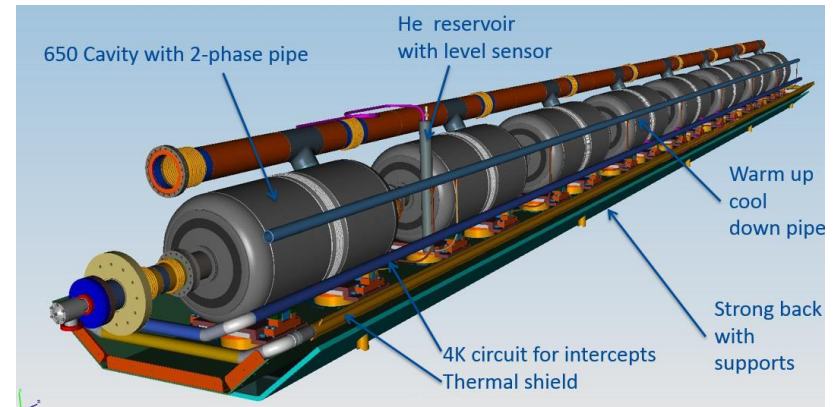
- 11 total cryomodules
- 3 cavities each (650 MHz, 5-cell)
- 33 total cavities
- No magnets internal to the cryomodule
- Approximate length = 3.9 m



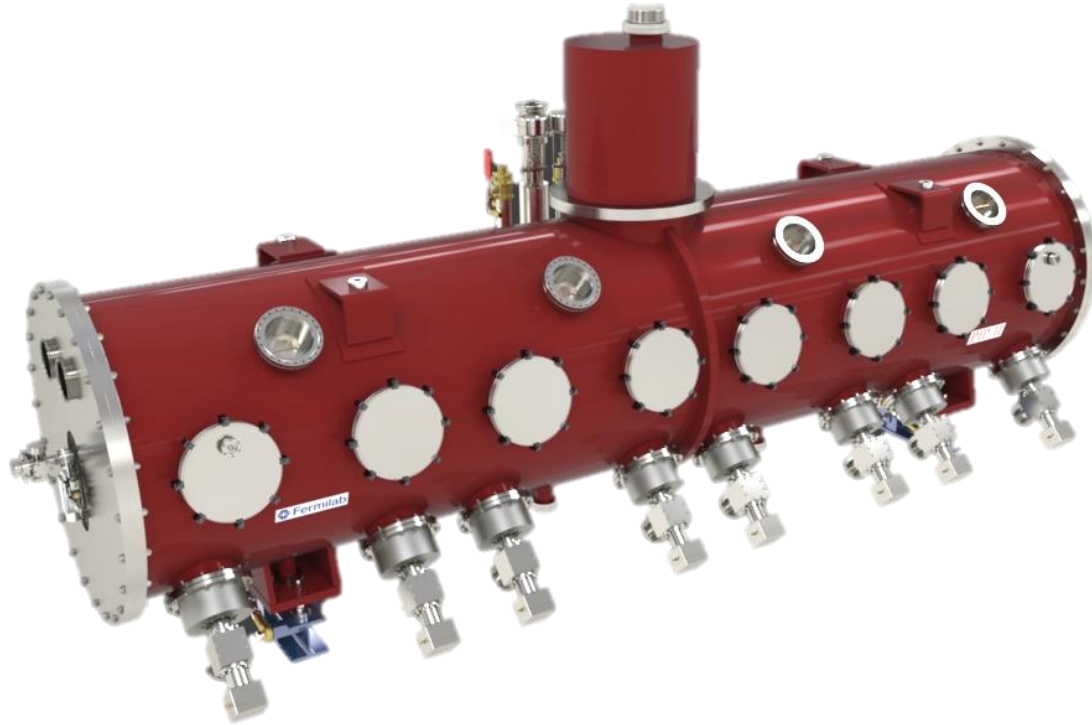
- Many design features common with the current SSR1 cryomodule design
- Coupler port locations are fixed with respect to the vacuum vessel
- Support system not subject to thermal distortions during cooldown
- To date, unproven

## High-Beta Cryomodule

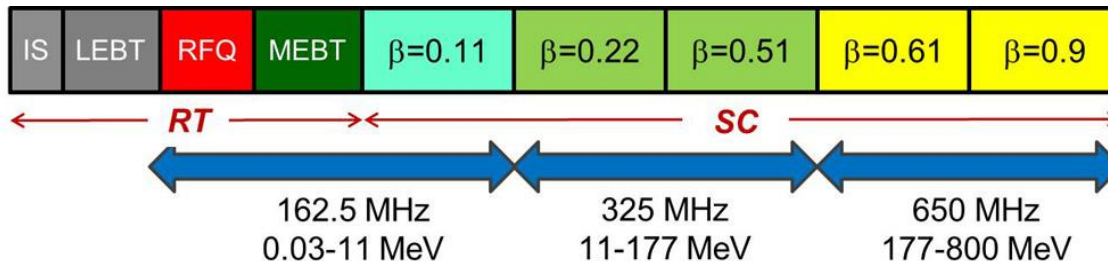
- 4 total cryomodules
- 6 cavities each (650 MHz, 5-cell)
- 24 total cavities
- No magnets internal to the cryomodule
- Approximate length = 9.5 m



# SSR1 Cryomodule for PIP-II



Two **SSR1 cryomodules** will be part of the 800 MeV linear accelerator complex that Fermilab is planning to build in the framework of the Proton Improvement Plan-II (PIP-II) project. The cavity string assembly of this cryomodule which constitutes the beam-line volume, contains *eight superconducting Single Spoke Resonators type 1 (SSR1)* with cold-end input couplers and four solenoids.



# SSR1 Cryomodule: Functional Requirements Specification

This engineering document “325 MHz SSR1 CRYOMODULE, FRS, ED0001316” (available in Teamcenter) addresses the functional requirements of the PIP-II SSR1 cryomodule. It includes physical size limitations, cryogenic system requirements and operating temperature, instrumentation, cavity and lens sequence and alignment requirements, magnet current leads, and interfaces to interconnecting equipment and adjacent modules.

General		SRF Cavities	
Physical beam aperture, mm	30	Number, total	8
Overall length (flange-to-flange), m	$\leq 5.4$	Frequency, MHz	325
Overall width, m	$\leq 1.6$	$\beta$ geometric	0.22
Beamline height from the floor, m	1.3	Operating temperature, K	2
Cryomodule height (from floor), m	$\leq 2.00$	Operating mode	CW
Ceiling height in the tunnel, m	3.20	Operating energy gain at $\beta=0.22$ , MV/cavity	2.05
Maximum allowed heat load to 70 K, W	250	Coupler type – standard coaxial with cold part impedance, $\Omega$	105
Maximum allowed heat load to 5 K, W	80	Coupler power rating, KW	$>20$
Maximum allowed heat load to 2 K, W	50	Solenoids	
Maximum number of lifetime thermal cycles	50	Number, total	4
Intermediate thermal shield temperature, K	45-80	Operating temperature, K	2
Thermal intercept temperatures, K	5 and 45-80	Current at maximum strength, A	$\leq 100$
Cryo-system pressure stability at 2 K (RMS), mbar	$\sim 0.1$	$\int B^2 dL$ , T <sup>2</sup> m	4.0
Environmental contribution to internal field	$<15$ mG	BPMs	
Transverse cavity alignment error, mm RMS	$<1$	Number, total	4
Angular cavity alignment error, mrad RMS	$\leq 10$	Number of plates per BPM	4
Transverse solenoid alignment error, mm RMS	$<0.5$	Accuracy of electrical center with respect to the geometric center, mm	$\leq \pm 0.5$
Angular solenoid alignment error, mrad RMS	$<1$		



# Prototype SSR1 Cryomodule for PXIE

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A prototype cryomodule is under design and development to achieve the goals of:

- Validate design concepts of new spoke cavity cryomodule
  - Room temperature strongback
  - Individual support posts to control axial motion
  - Conduction cooled magnet current leads
  - Single-window coaxial input coupler
  - Integral beam instrumentation
  - Determine the practicality of tuner access ports
  - Estimate heat loads
- Gain experience with the required alignment tolerances and check alignment stability during cooldown.
- Minimize risk ahead of full PIP-II design and production effort by gaining experience with strings of spoke cavities, solenoids, and beam instrumentation, e.g. cleanroom operations, final assembly in the vacuum vessel, shipping and handling, etc.

# SSR1 Cryomodule Heat Load Estimates

## SSR1 Cryomodule Heat Load Estimates

T. Nicol -February 9, 2012

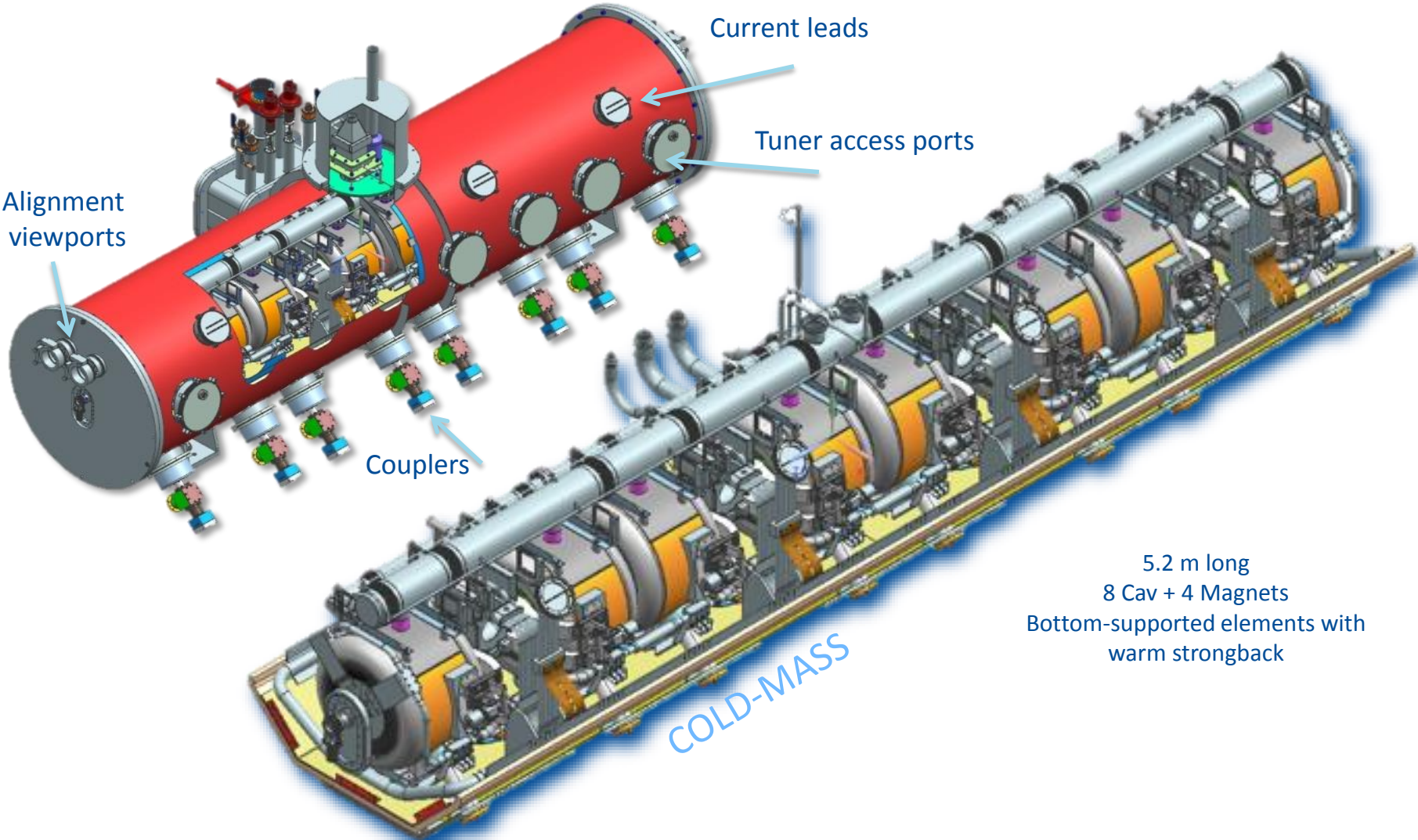
SSR1 8 cavities, 4 solenoids	Each unit			Mult	Total		
	80 K	4.5 K	2 K		70 K	4.5 K	2 K
Input coupler static	5.36	2.82	0.50	8	42.88	22.56	4.00
Input coupler dynamic	0.00	0.00	0.25	8	0.00	0.00	2.00
Cavity dynamic load	0.00	0.00	1.78	8	0.00	0.00	14.24
Support post	2.76	0.36	0.05	12	33.12	4.32	0.60
Conduction lead assembly	36.80	13.20	1.24	4	147.20	52.80	4.96
MLI (total 70 K + 2 K)	30.54	0.00	1.42	1	30.54	0.00	1.42
Cold to warm transition	0.72	0.08	0.01	2	1.44	0.16	0.02
<b>Total</b>					<b>255.2</b>	<b>79.8</b>	<b>27.2</b>

### Notes:

1. Cavity dynamic loads from Nikolai Solyak, February 2012.
2. Input coupler static loads from S. Kazakov, February 2012, no copper plating on outer conductor, intercepts at 15 K and 125 K.
3. Current lead heat loads assume 2 coils at 50 A, 1 coil at 200 A.

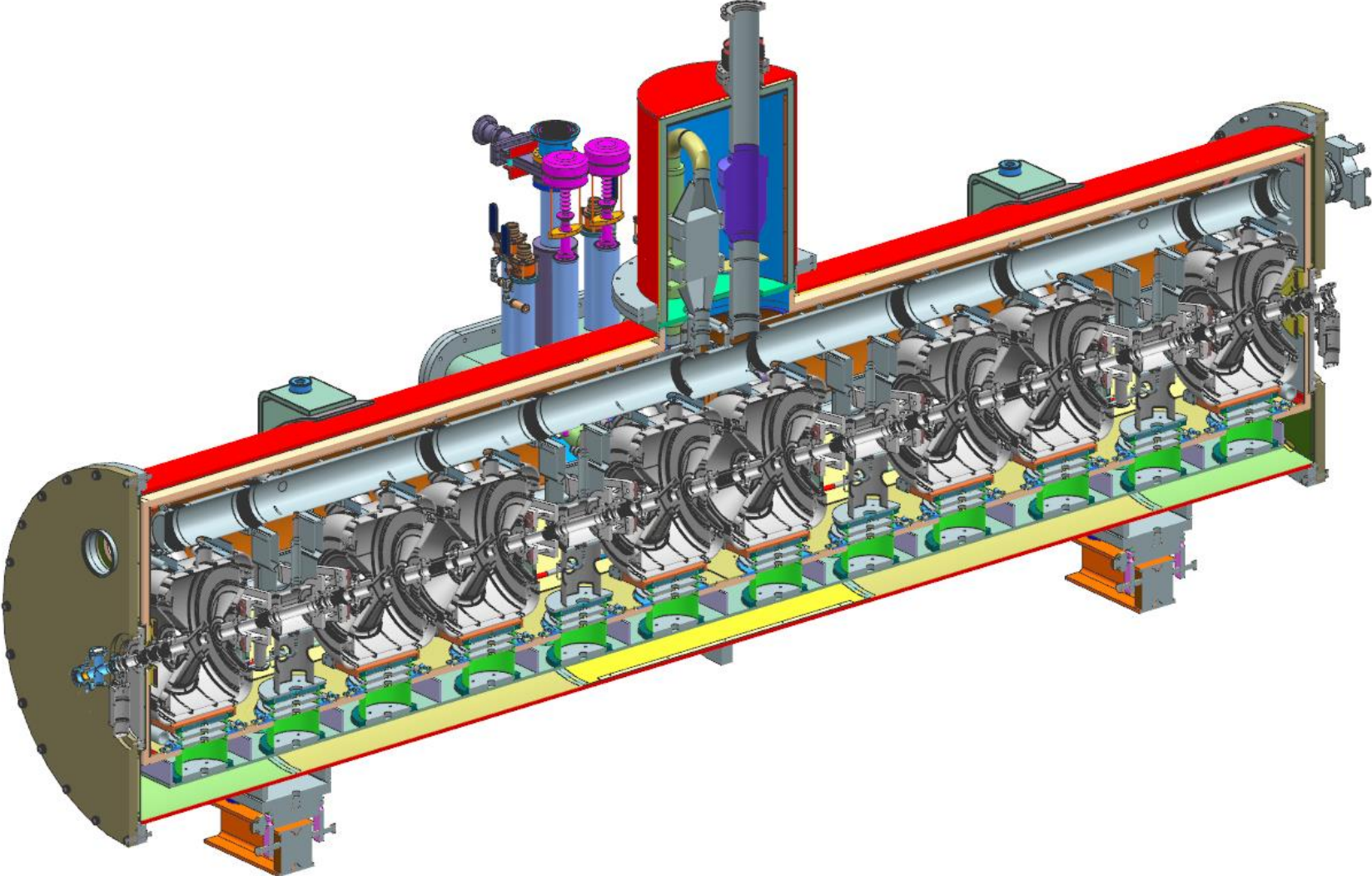
# SSR1 Cryomodule

L. Ristori, T. Nicol, Y. Orlov, D. Passarelli, M. Parise  
<http://accelconf.web.cern.ch/AccelConf/PAC2013/papers/thpma09.pdf>



5.2 m long  
8 Cav + 4 Magnets  
Bottom-supported elements with warm strongback

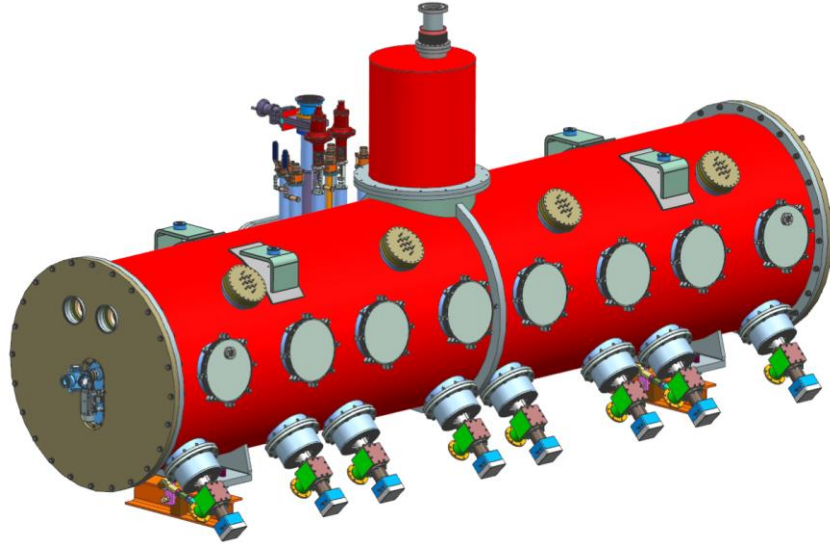
# Cutview of the prototype SSR1 cryomodule



*SSR1 Cryomodule Design for PXIE, T. Nicol et al., Proceedings of PAC2013, Pasadena, CA USA*



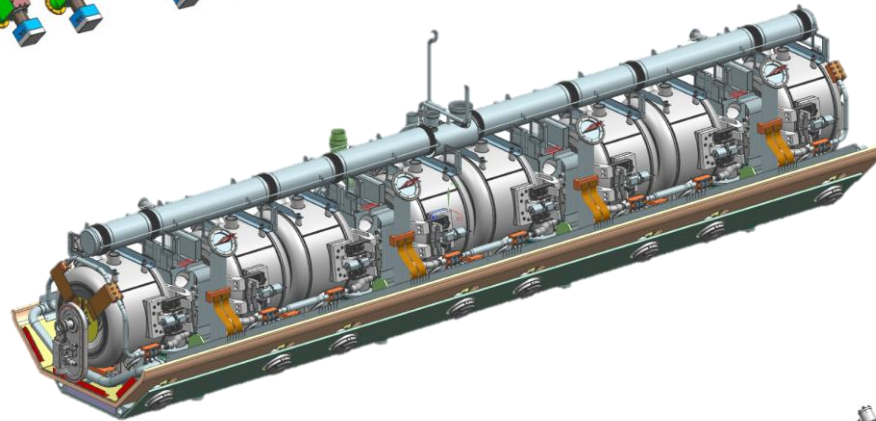
# Design of the prototype SSR1 Cryomodule



## SSR1 cryomodule – Top assembly

Conceptual design: completed

Final design: ~80% completed

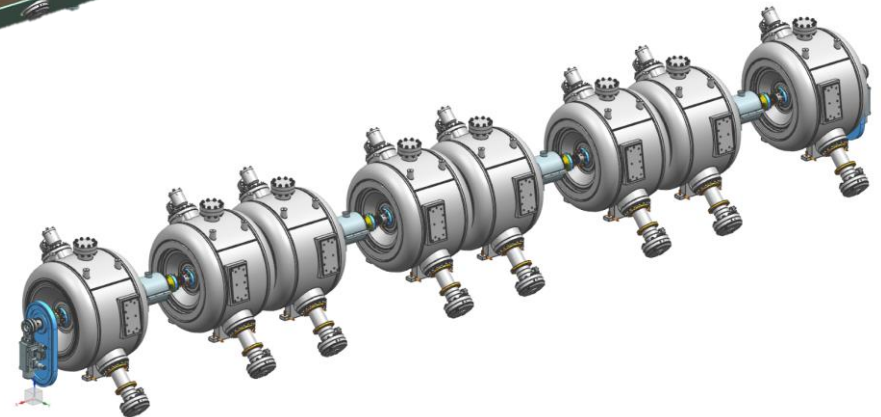


## SSR1 coldmass

Conceptual design: completed

Final design: ~80% completed

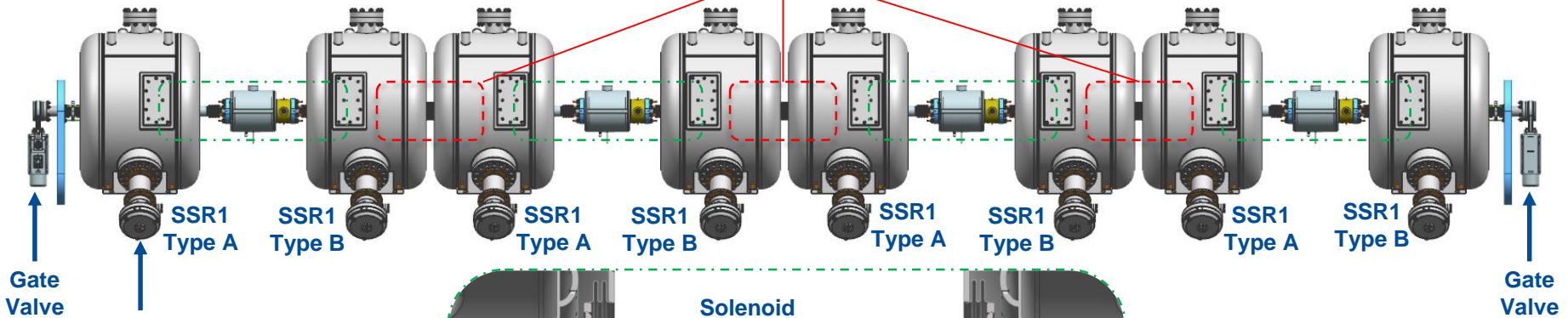
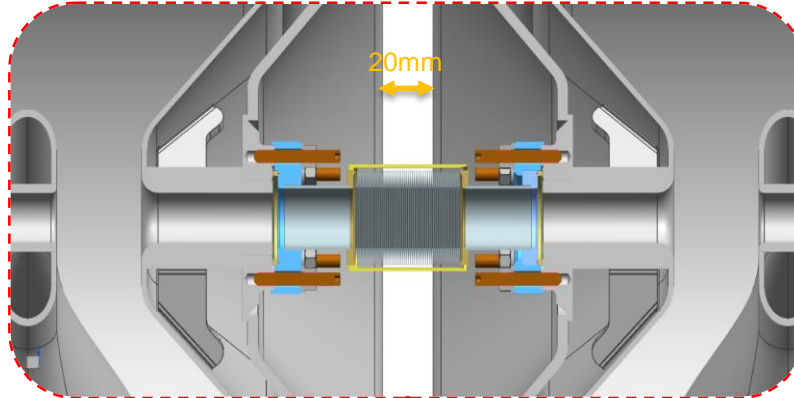
**SSR1 string assembly**  
Final design: 100% completed  
Procurement: 100% completed  
Assembling: completed in FY16



# SSR1 String Assembly: design features

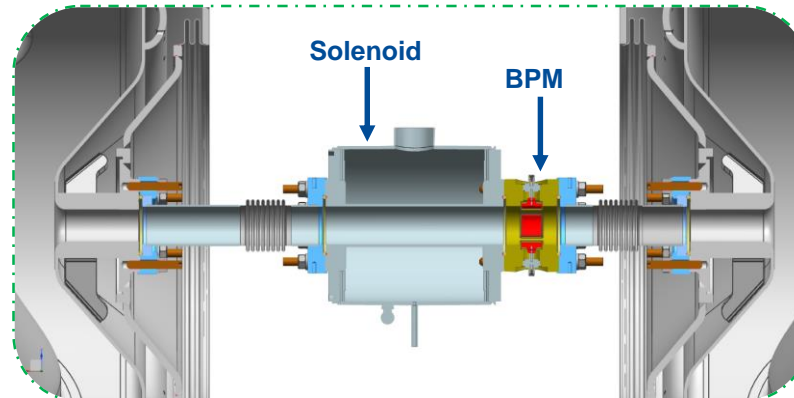
## Interconnection cavity-cavity

Edge-welded bellows assembly  
Al-diamond seals  
SiBr set screws  
316L stainless nuts and washers



## Interconnection cavity-magnet-BPM-cavity

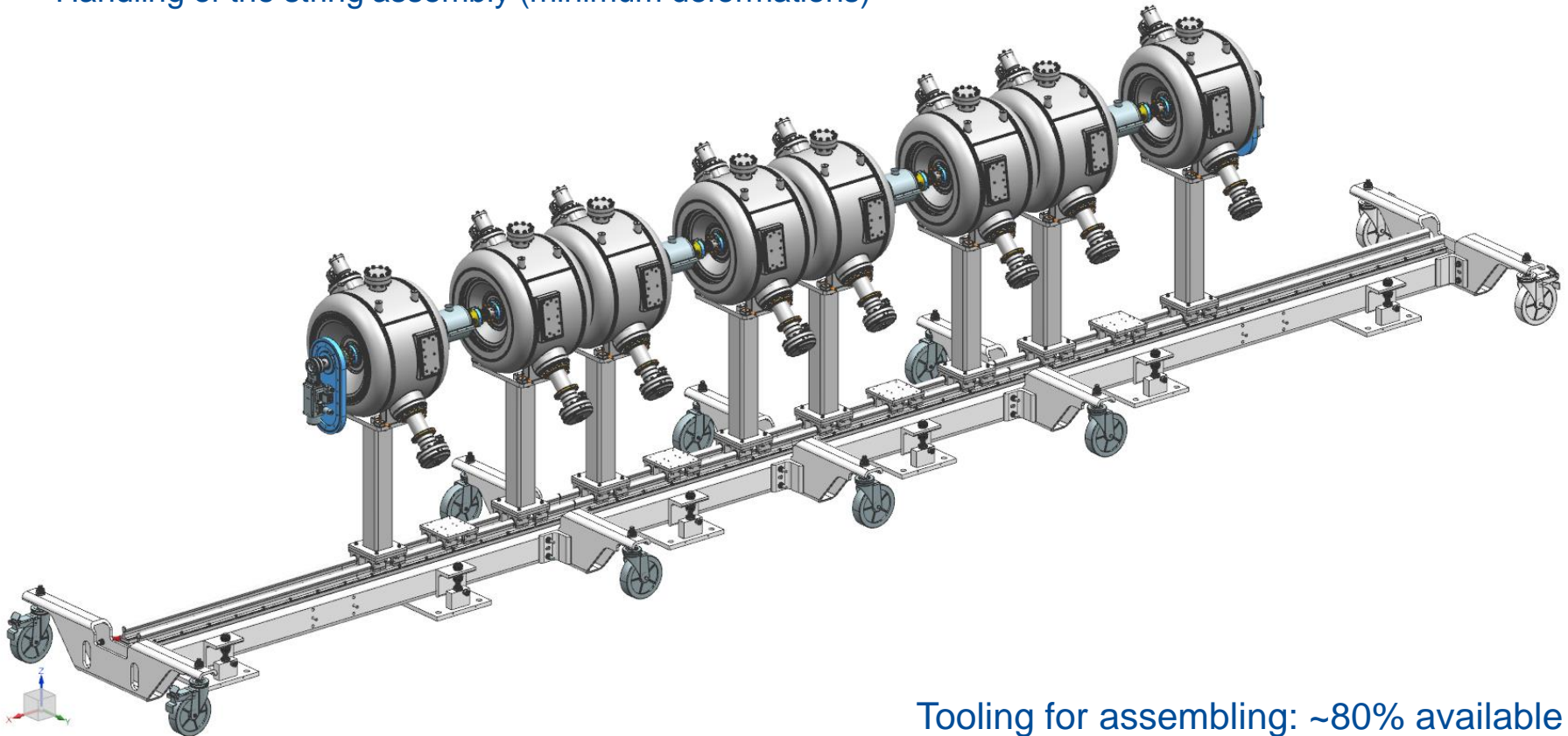
Hydro-formed bellows assemblies  
Al-diamond seals  
SiBr set screws  
316L stainless nuts and washers



# SSR1 String Assembly in cleanroom

## Key aspects:

- All cavities qualified bare and then jacketed with power coupler before string assembly
- Particle-free assembly in cleanroom class 10 (ISO 4)
- Alignment of cavities (electric axis) and magnets (magnetic axis) with a datum.
- Handling of the string assembly (minimum deformations)



Tooling for assembling: ~80% available



# SSR1 Cavities: requirements

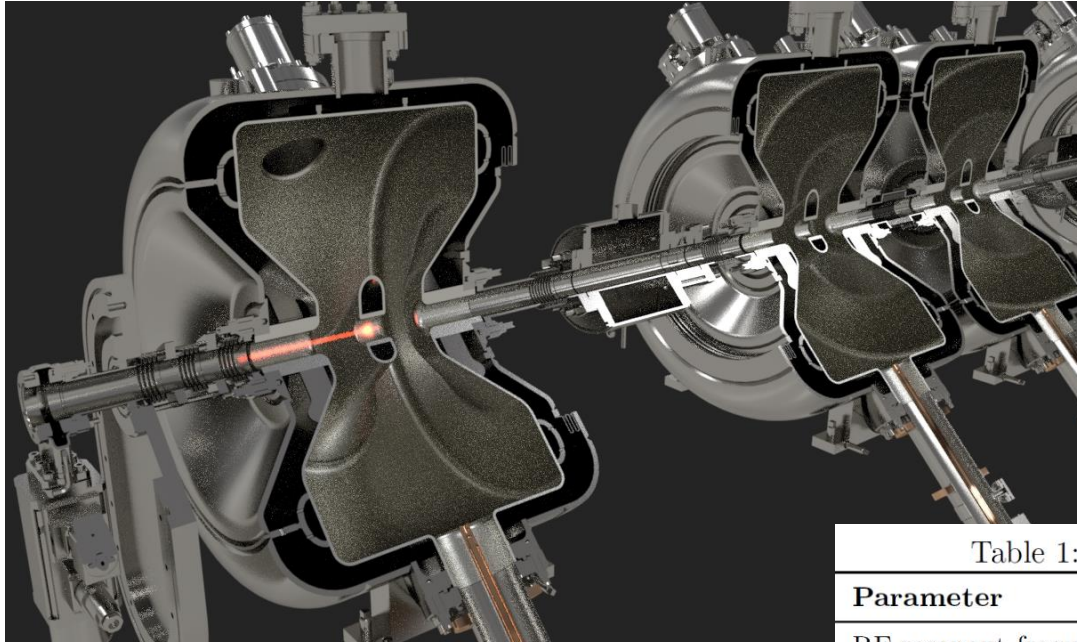
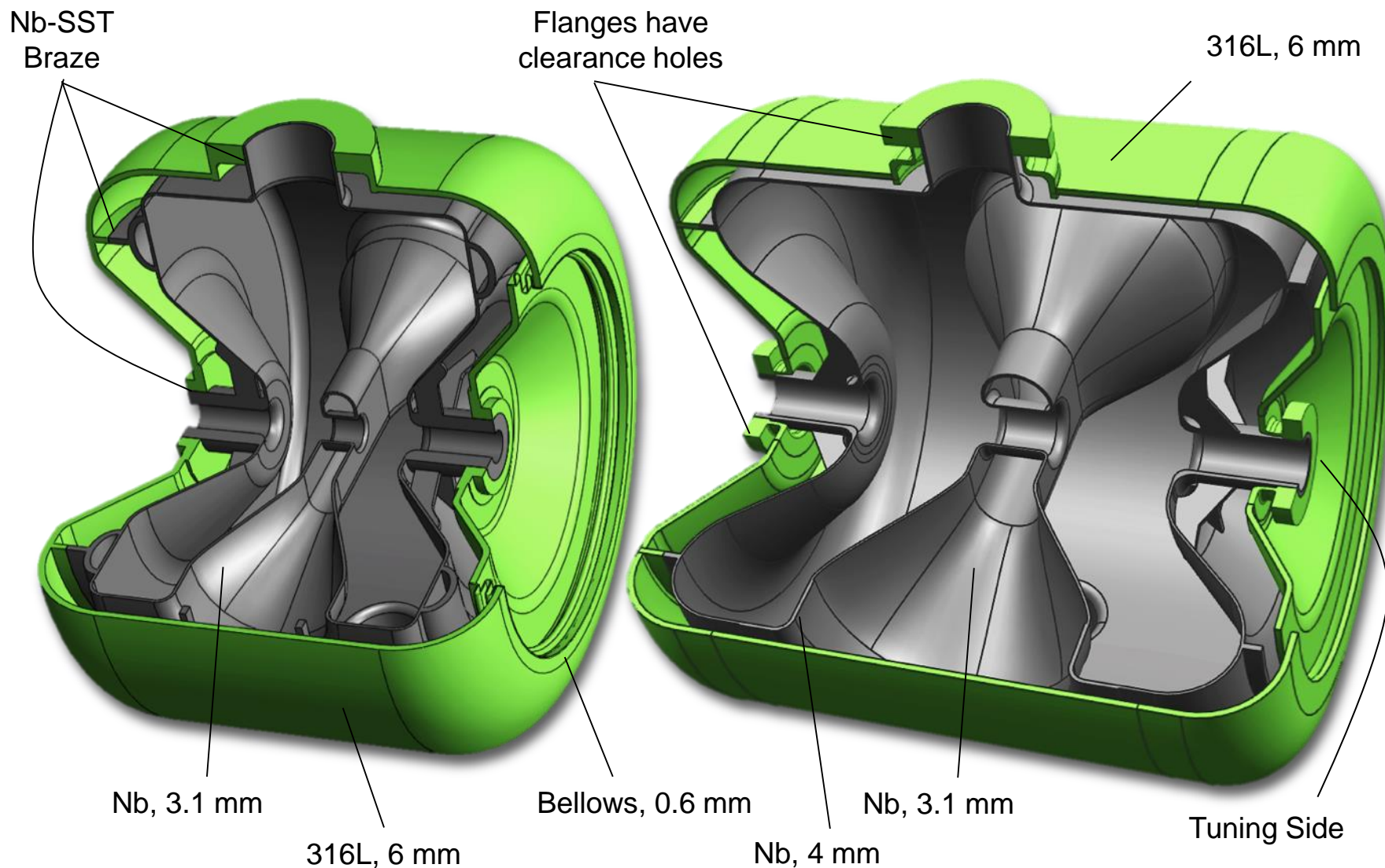


Table 1: Cavity operational and test requirements

Parameter	Value
RF resonant frequency	325 MHz
Bandwidth	$\pm 20$ Hz
Operating accelerating gradient ( $E_{acc}$ )	12 MV/m
Quality factor ( $Q_0$ ) at $E_{acc}$	$> 5 \cdot 10^9$
Operating gain per cavity	2 MeV
Maximum power dissipation at 2 K	5 W
Sensitivity to He pressure fluctuations	$< 25$ Hz/Torr
Field flatness	$\pm 10\%$
Operating temperature	1.8 $\div$ 2.1 K
Operating pressure	16 $\div$ 41 mbar (differential)
Maximum allowable working pressure	2 bar at 293 K, 4 bar at 2 K
RF power input per cavity	6 kW (CW, operating)
Max Leak Rate (room temp)	$< 10^{-10}$ atm $\cdot$ cc/s

# SSR1 and SSR2 – a closer look..

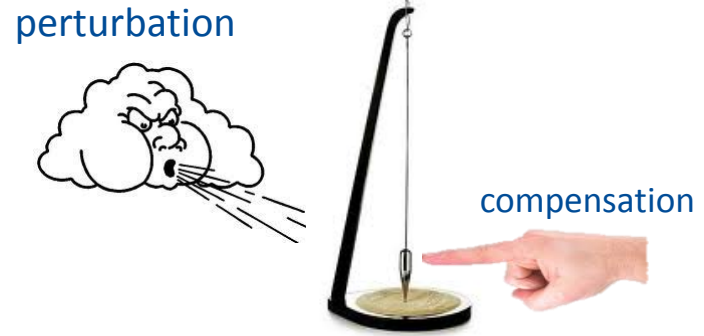


# Active and Passive control for SRF cavities



## Active only:

Constant compensation even without large perturbations



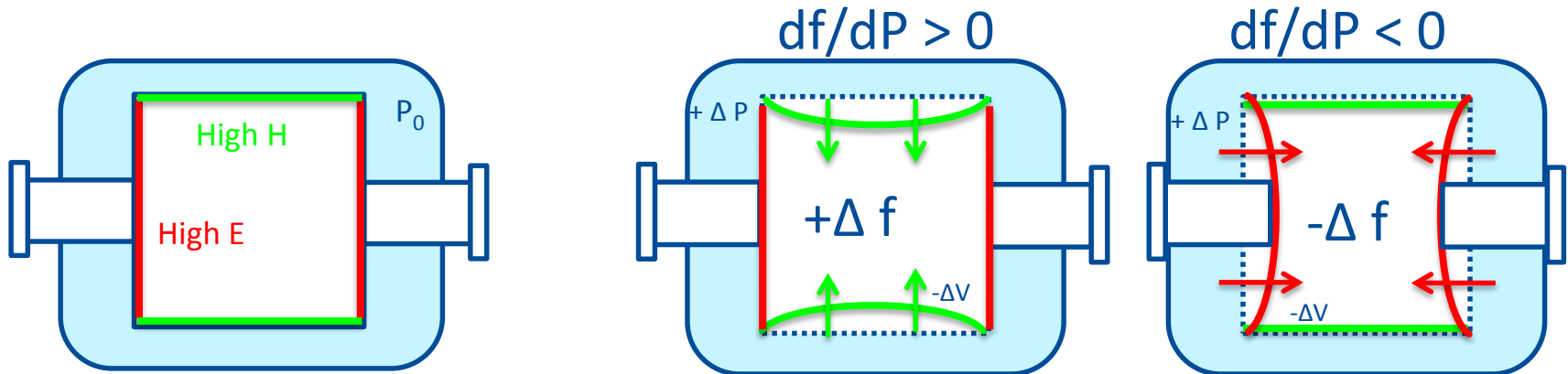
## Passive + Active:

Compensation only during large perturbations

- **Active controls**
  - Compensating the effects of perturbations
- **Passive controls (Environment)**
  - Reducing perturbations
- **Passive controls (Cavity Design)**
  - Reducing sensitivity of cavity to perturbations (LFD, Mechanical Resonances,  $df/dP$ )

# Passive Control: Sensitivity to He bath pressure (df/dP)

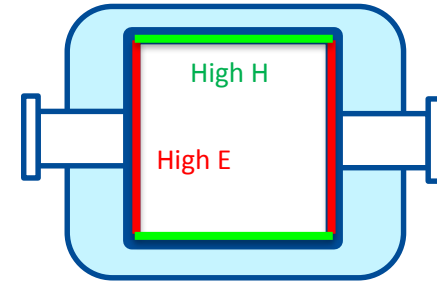
- Cavity walls deform proportionally to the increment of pressure in the liquid Helium.
- Changes in the cavity RF volume produce frequency shifts that can be positive or negative depending on the shape of deformation
- Slater's rule:  $\Delta f = k \int (\epsilon_0 E^2 - \mu_0 H^2) dV$



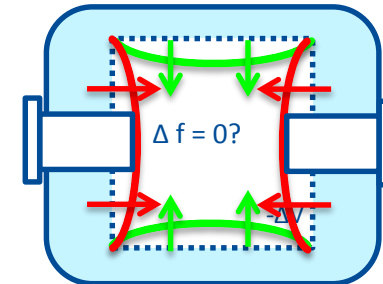
# Minimizing He pressure sensitivity

## Options for reducing $df/dP$

- Reduce deformations of cavity walls
  - This approach is widely utilized and easy to implement (increase cavity wall thickness, increase number/size of stiffeners)
  - Drawback is a general increase in cavity stiffness and in most cases an increase in tuning resistance ( $dF/df$ )
  - Sometimes impractical to achieve  $df/dP=0$
- Balance deformations of cavity walls
  - This method is less intuitive and requires a systematic approach
  - The specific behavior of the cavity needs to be understood
  - Opportunity to achieve  $df/dP=0$
  - Keep tuning resistance under control



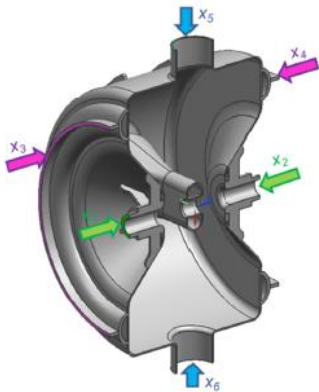
Reduce deformations by  
Increasing  $K_{cav}$



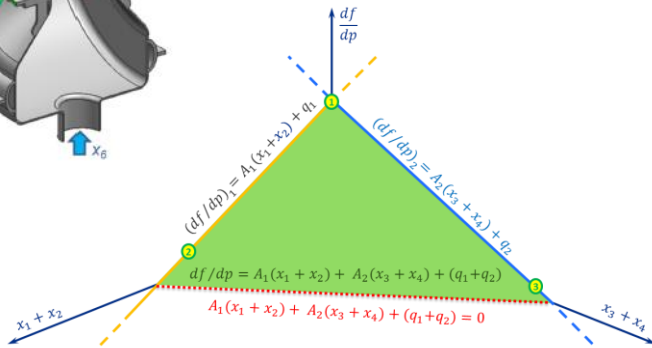
Balance deformations



# SSR1 Cavities: pressure sensitivity



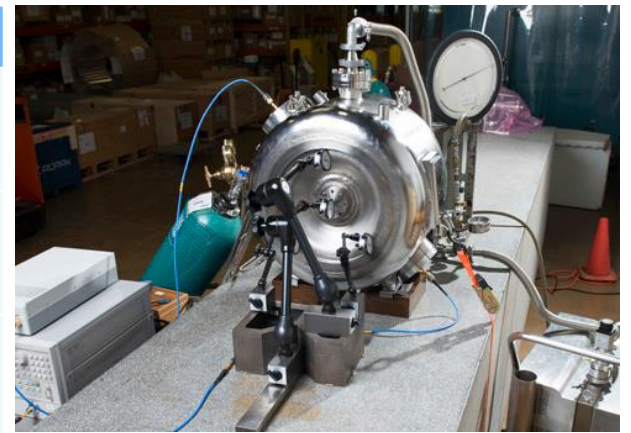
The jacketed SSR1 cavities were designed to have very low sensitivity to helium pressure fluctuations (microphonics). We physically coupled the Nb cavity and the helium vessel such that we obtain a combination of cavity walls deformations ( $x_1 + x_2$ ) and ( $x_3 + x_4$ ) giving a  $df/dp = 0$ .



*Self-compensated system --> Passive compensation  
No active control to mitigate the pressure fluctuations*

*PIP-II requirements:  $-25 \leq df/dp \leq 25$  Hz/Torr*

	df/dp [Hz/Torr]	S106	S107	S108	S109	S110	S111	S112	S113	S114
Measured	Bare cavity (with transition ring)	-564	-561	-553.5	-555.1	-568.8	-525.8	-524.6	-544.7	-557.2
	With He Vessel (without Tuner)	8	8	-1.2	5.4	7.9	2.7	9.0	6.3	10
	Fully integrated	4*	4	0*	2*	4*	2*	5*	3*	5*

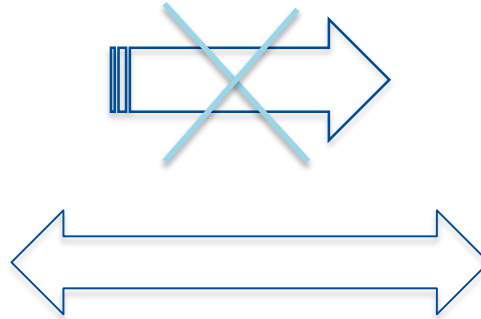


\* Not measured yet (best guess)

# SSR1 Cavities: design features

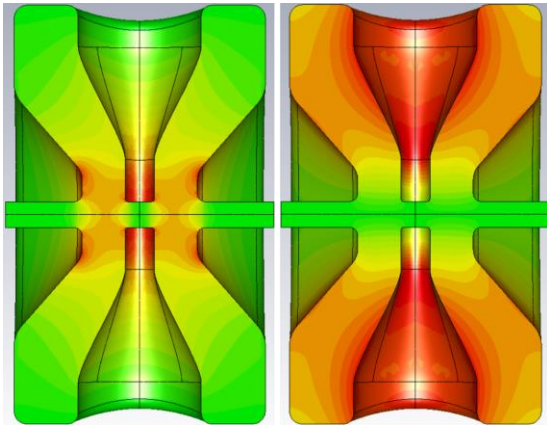
## Electromagnetic Design

- Shape optimization
- Multipacting analysis
- Higher order modes
- Kick analysis
- Multipole effect
- Pressure Sensitivity
- Lorentz force detuning

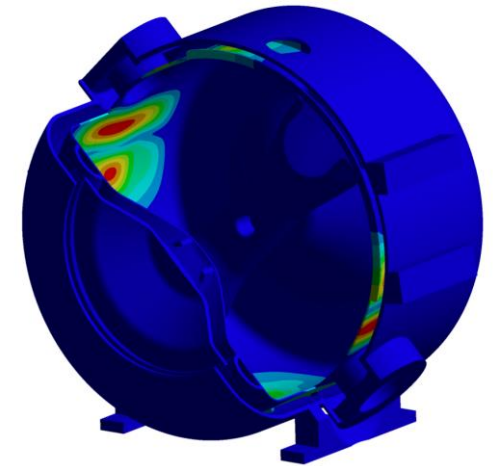


## Mechanical Design

- Niobium shell design
- Vessel design
- Shape optimization
- Pressure Rating
- Stiffening and detuning
- Modal analysis
- Tuner Design



The parallel approach in performing RF/Mech analyses benefits the final design...

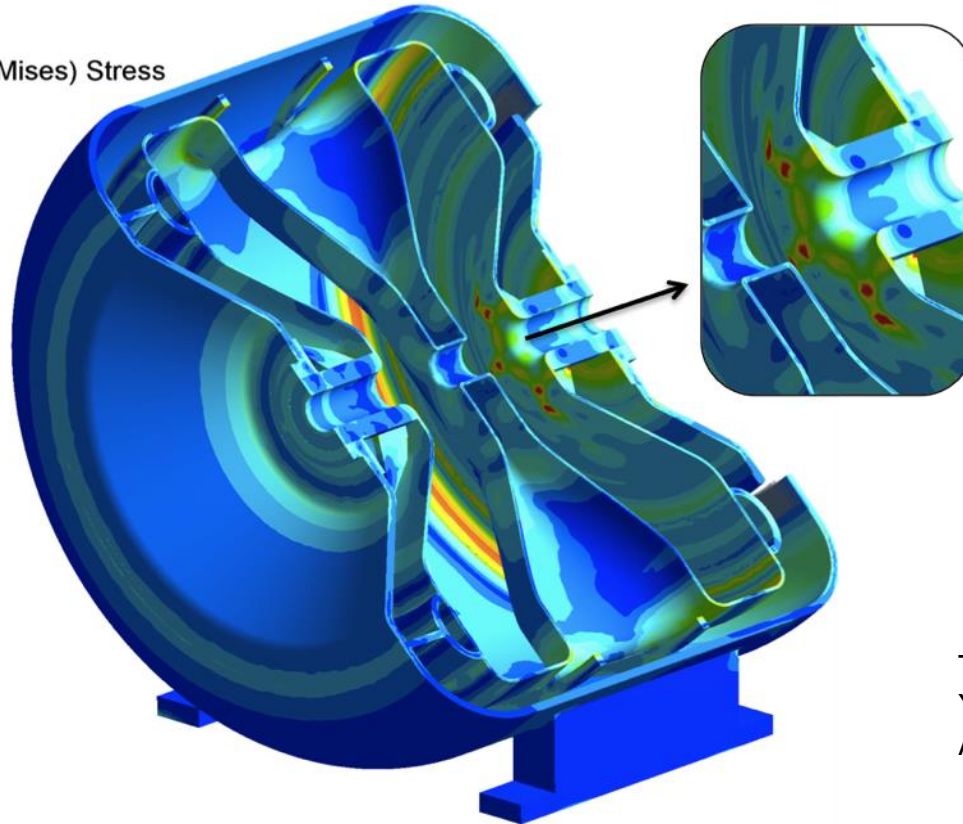
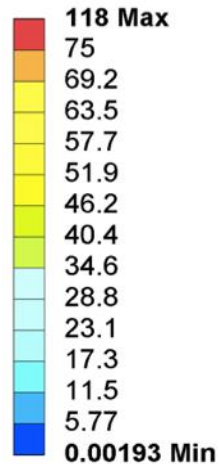






- We designed spoke resonators to reach **2 bar** (RT) and **4 bar** (CT) meeting applicable US safety codes, ASME, B&PV, for complex shapes extensive FEA necessary
- Von Mises stresses may appear higher than yield in certain locations (see image), **don't be scared**, depending on the specific case, **it may be OK!**

C: 1)\_P+D\_@RT  
Equivalent Stress 4  
Type: Equivalent (von-Mises) Stress  
Unit: MPa  
Time: 1  
9/18/2013 10:55 AM



Tensile tests on Nb batch  
Yield limit 75 MPa  
Allowable 47 MPa



We must comply with the ASME Boiler and Pressure vessel code and determine the pressure rating of each SRF resonator

## Division 1 vs. Division 2 of Chapter VIII:

Division 2 allows utilizing complex shapes without limitations in principle, it generally results also in thinner walls of the vessels. We decided to follow this approach for the production cavities.

The **Design-by-Analysis methodology** utilizes the results from *finite element analysis* to assure:

### 1. Protection against **plastic collapse**

avoid unbounded displacement in each cross-section of the structure due to the plastic hinge

- Elastic stress analysis method
- Elastic-plastic stress analysis method

### 2. Protection against **collapse from buckling**

buckling is characterized by a sudden failure of a structural member subjected to high compressive stress, where the actual compressive stress at the point of failure is less than the ultimate compressive stresses that the material is capable of withstanding.

- Elastic stress analysis (Linear buckling)

### 3. Protection against **failure from cyclic loading**

- Elastic ratcheting analysis method

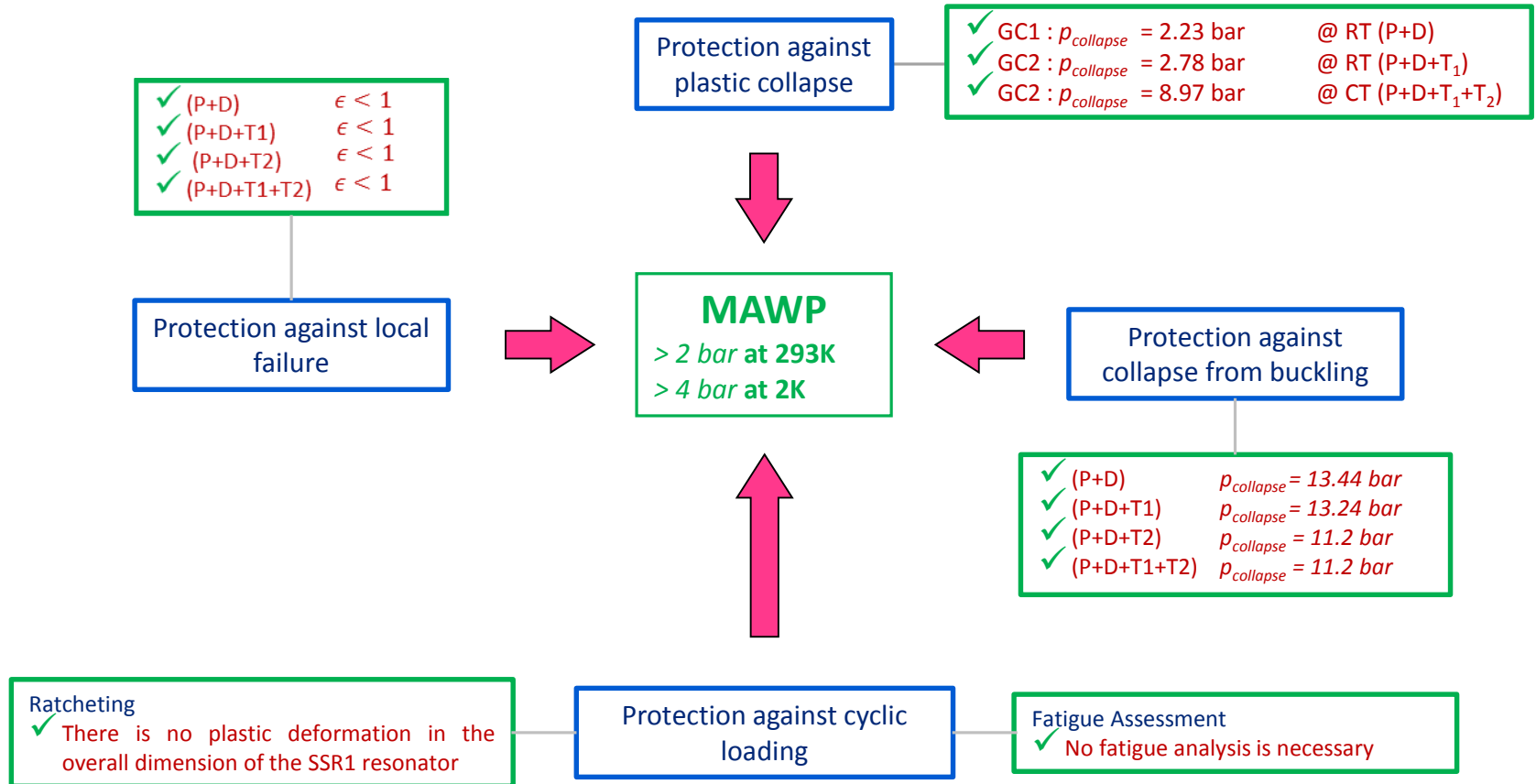
### 4. Protection against **local failure** (i.e. joints)

- Elastic-plastic analysis under the achieved MAWP

# ASME Pressure Rating



The table summarizes the results obtained by simulations performed following the Div 2, Part 5 directions. It **shows** that the desired MAWP is achieved both at RT and CT.

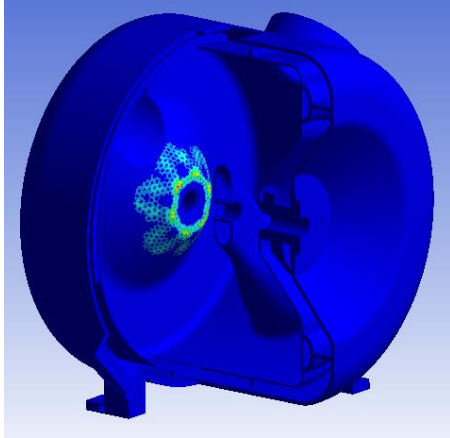


# ASME Pressure Rating - Results



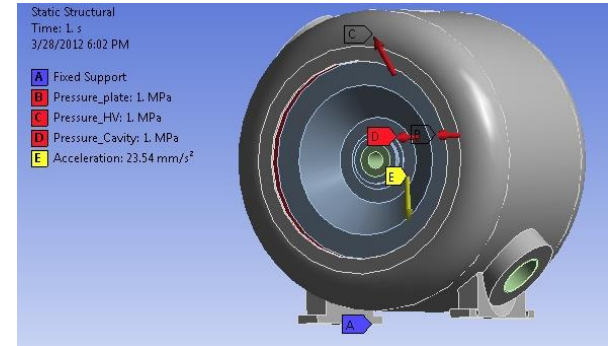
## Protection Against Plastic Collapse:

Load Combination: 2.4 (P+D)



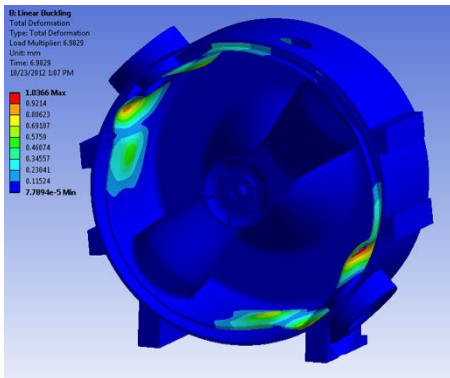
The elastic plastic stress analysis at 293K shows that the plastic collapse occurs on the *area of the Endwall* (bellows side), connected to the *Daisy ribs*, under a pressure of 5.35 bar (77.6 psi)

$$MAWP_{RT} = \frac{5.35bar}{2.4} = 2.23bar (32.3 psi)$$



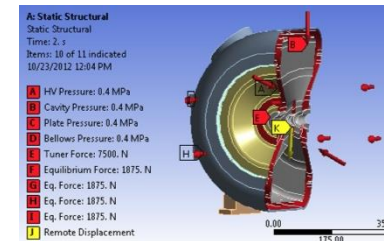
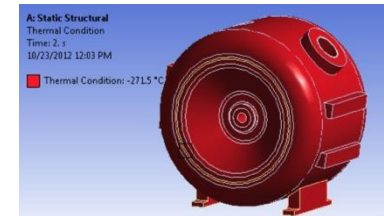
## Protection Against Collapse from buckling:

Load Combination: (P+D+T<sub>1</sub>+T<sub>2</sub>)



The load is applied in two steps, first the cooldown and then all other loads. The element that buckle first determines the MAWP for this failure mode. In the case under analysis the buckling occurs at the cavity

$$MAWP_{RT} = \frac{27.9bar}{2.5} = 11.2bar (162.4 psi)$$

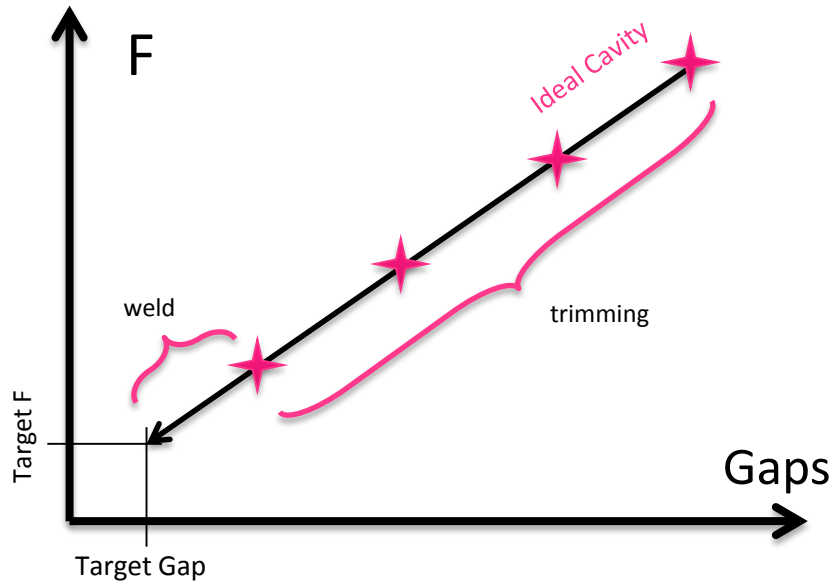


# Trimming before final weld



- Frequency is adjusted by trimming the outer conductor incrementally, before final equator welds.
- Ideal cavity will hit target frequency and gap size

Operation	Shift (kHz)	Freq. (MHz)
End-wall Welding	Negligible	323.975
BCP (120-150 $\mu\text{m}$ )	+ 160	324.135
BCP (20-30 $\mu\text{m}$ )	+ 40	324.175
Ring + Jacketing	+ 500	324.675
BCP (20-30 $\mu\text{m}$ )	+ 40	324.715
Cool-down	+ 385	325.100
Tuner Engaged	- 100	325.000





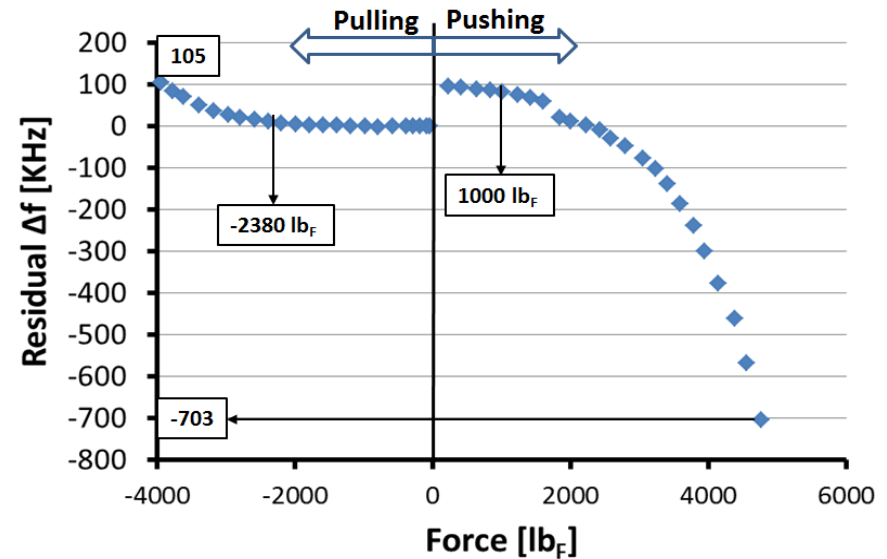
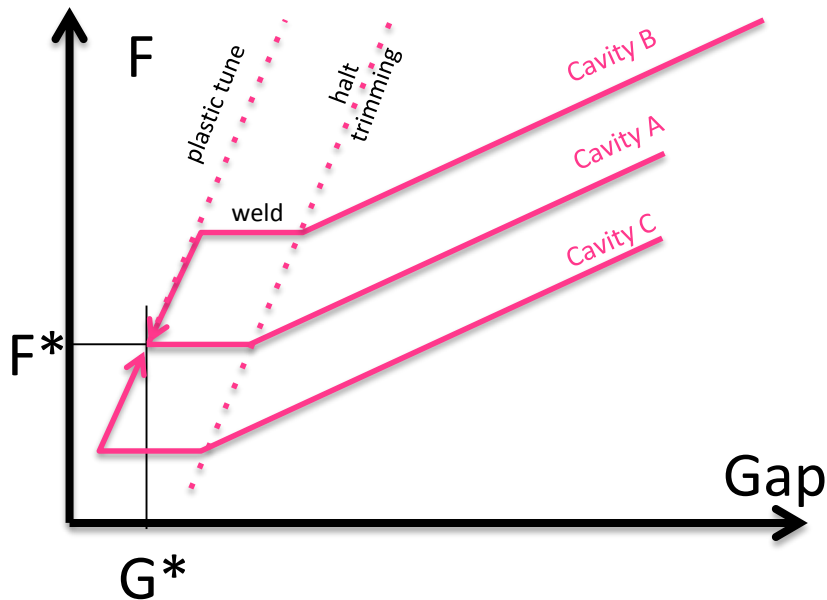
# Trimming and Plastic Tuning



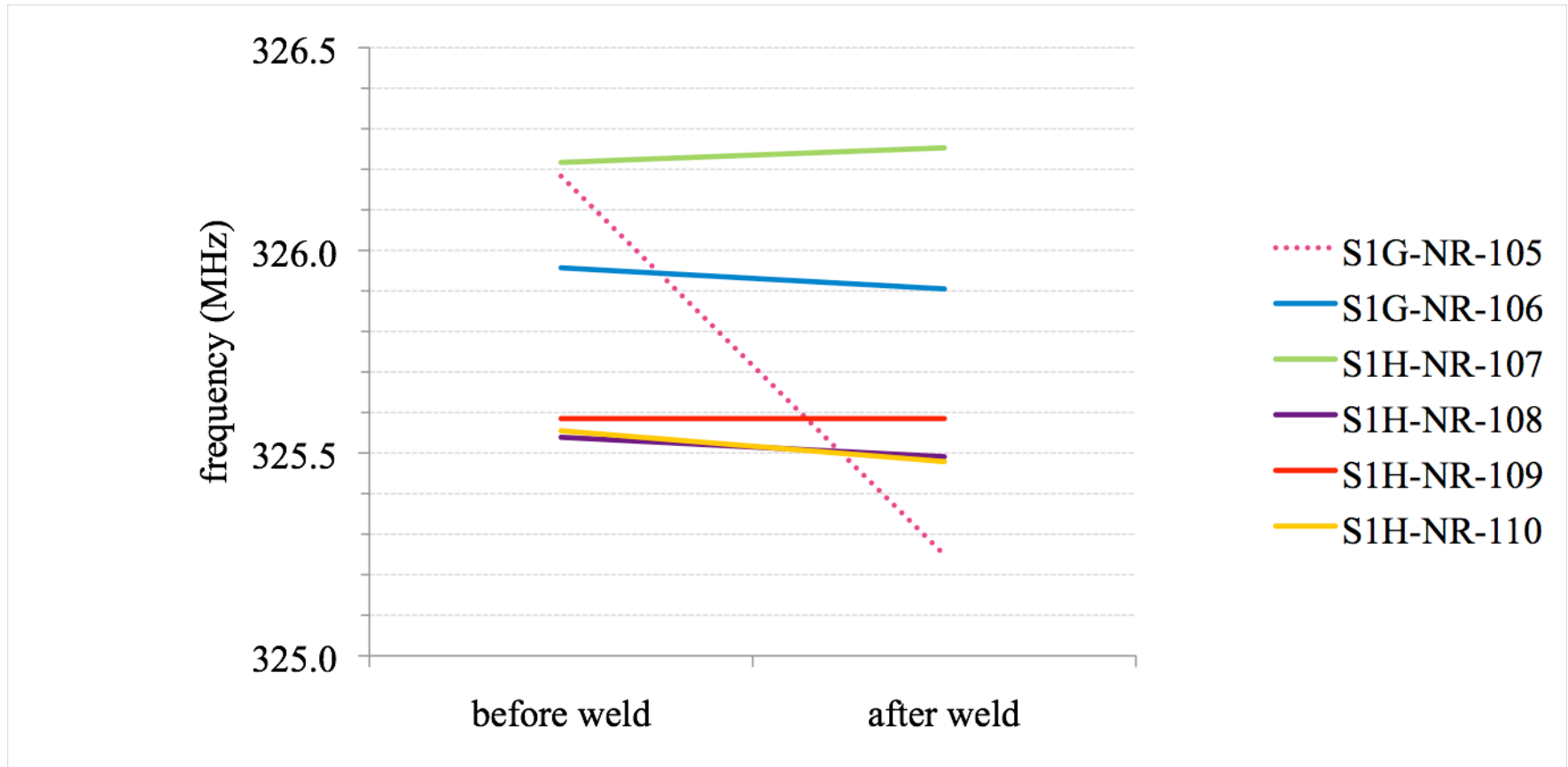
- Frequency AND gap size can be achieved by using wisely the plastic tuning process.
- Trimming should be interrupted at the crossing of the “halt trimming” line, not at a fixed frequency

kHz/mm	measured (avg)
trimming	345 (320-370)
plastic tune	465
elastic tune	585

Elastic and Plastic sensitivities are different due to different deformed shapes



# Shifts due to final welding (Roark)

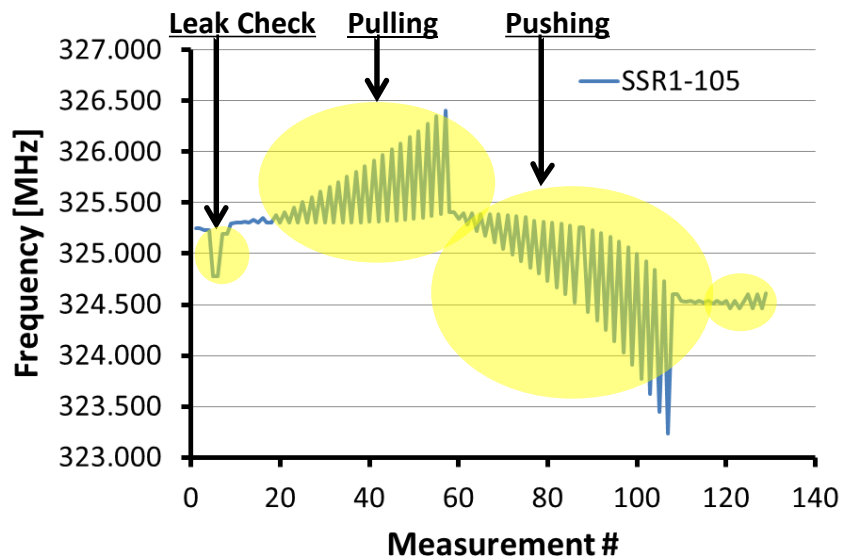


- Cavities so far showed negligible shifts (E and B shifts compensate)
- Large shift of #105 due to repeated welds on one side

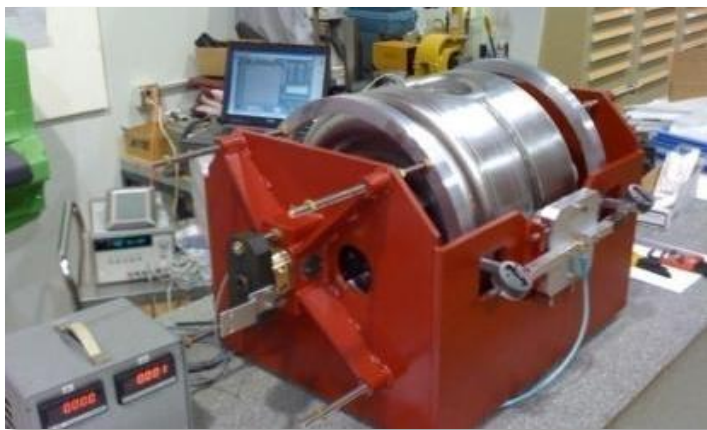
# Permanent tuning (FNAL)



(T. Khabiboulline, M. Hassan, P. Berrutti)



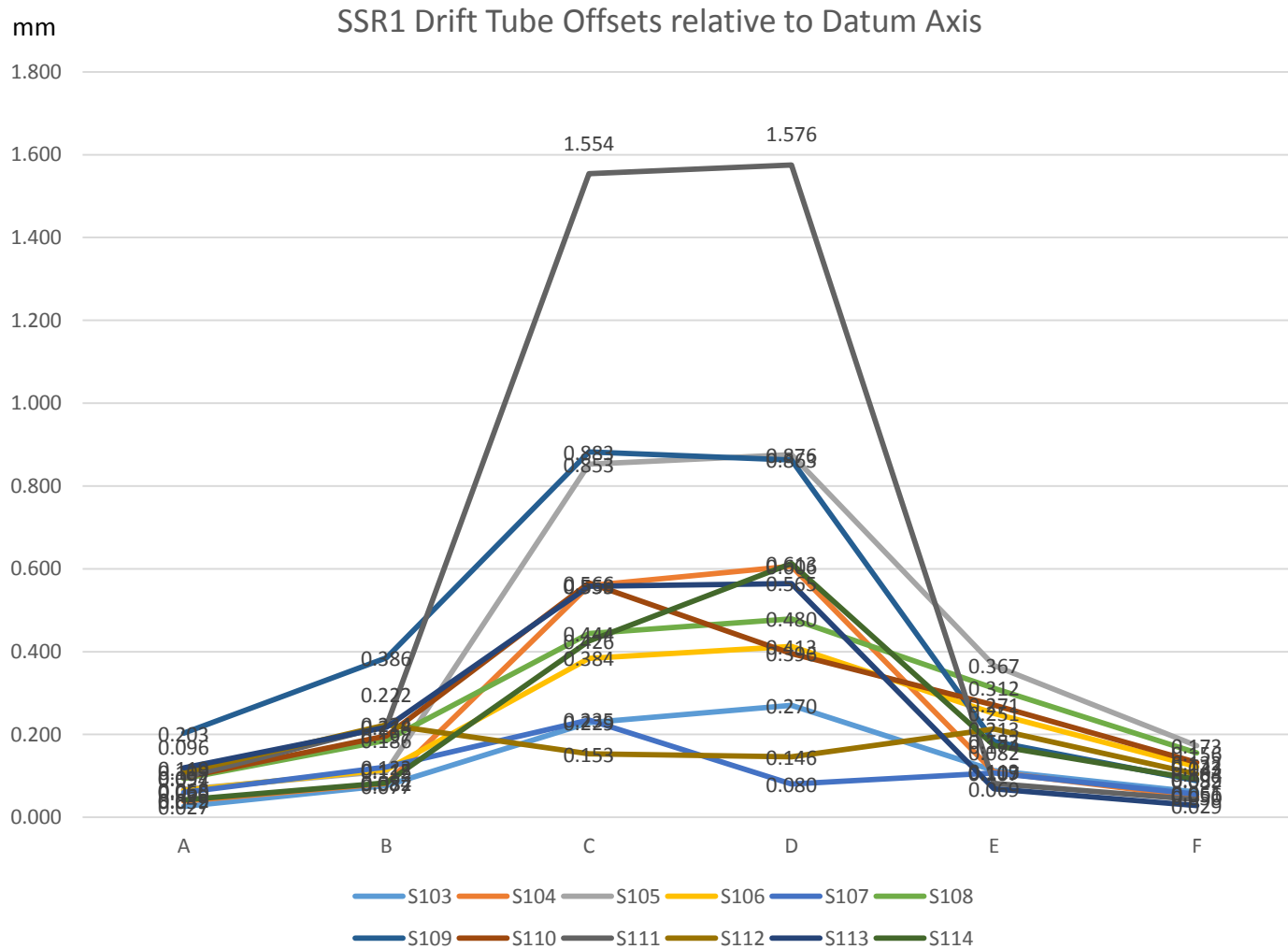
Importance of work hardening before installation in helium vessel.  
Elastic tuning range is increased by several times.  
Work hardening is directional.







# Cordax Measurements on 12 SSR1 Resonators



# Jacketing operations

D. Passarelli, M. Hassan, P. Berrutti



Thermal cameras insertion tubes (above)

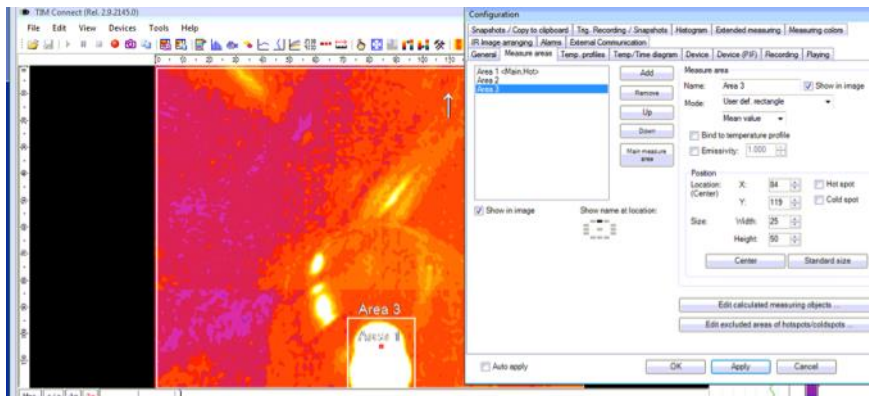


Welding inch-by-inch on 6 mm thick steel

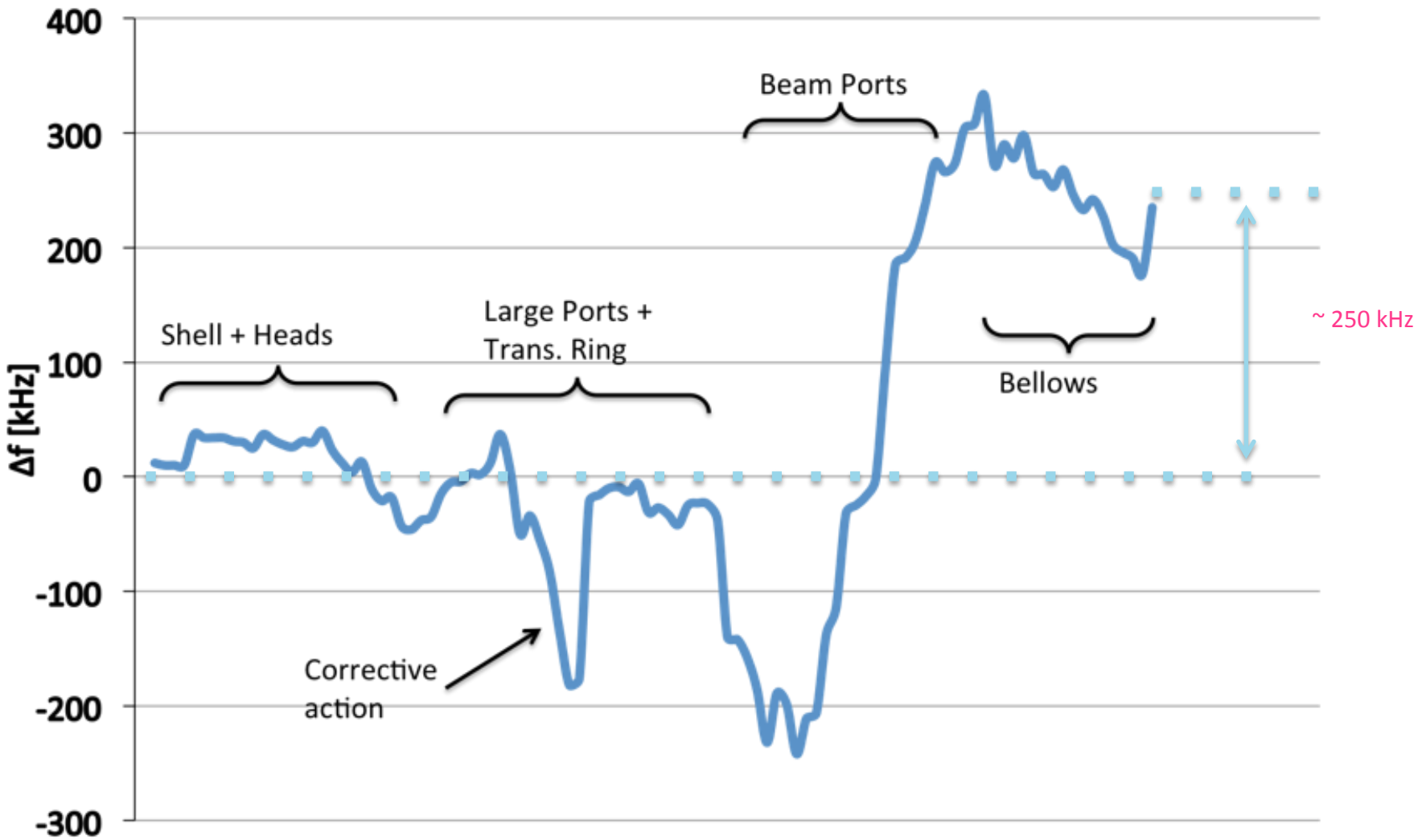
Large fixtures needed to control warping



Screenshot of remote connection from FNAL



# Shifts caused by jacking



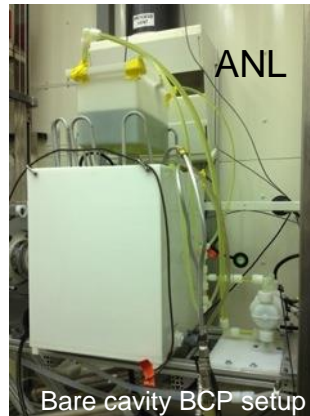
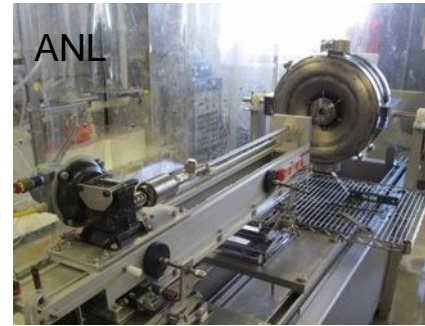
courtesy:  
P. Berrutti



# Processing/Testing steps (ANL, FNAL)



1. Inspection – RF & Optical
2. BCP 120-150 $\mu\text{m}$ (flip half-way)
3. HPR
4. 600 $^{\circ}\text{C}$ , 10 h (< 5 $^{\circ}\text{C}/\text{min}$ ramp rate)
5. RF Tuning
6. BCP 20-30 $\mu\text{m}$
7. HPR (horiz + vert)
8. Assemble
9. Evacuate + 120 $^{\circ}\text{C}$ , 48 h
10. Vertical Test
11. Helium Vessel Dressing
12. HPR
13. BCP 20-30 $\mu\text{m}$
14. HPR
13. Assemble
14. Evacuate + 120 $^{\circ}\text{C}$ , 48 h
15. Horizontal Test
16. Ready for String



Low-Temp Ovens (<300 $^{\circ}\text{C}$ )

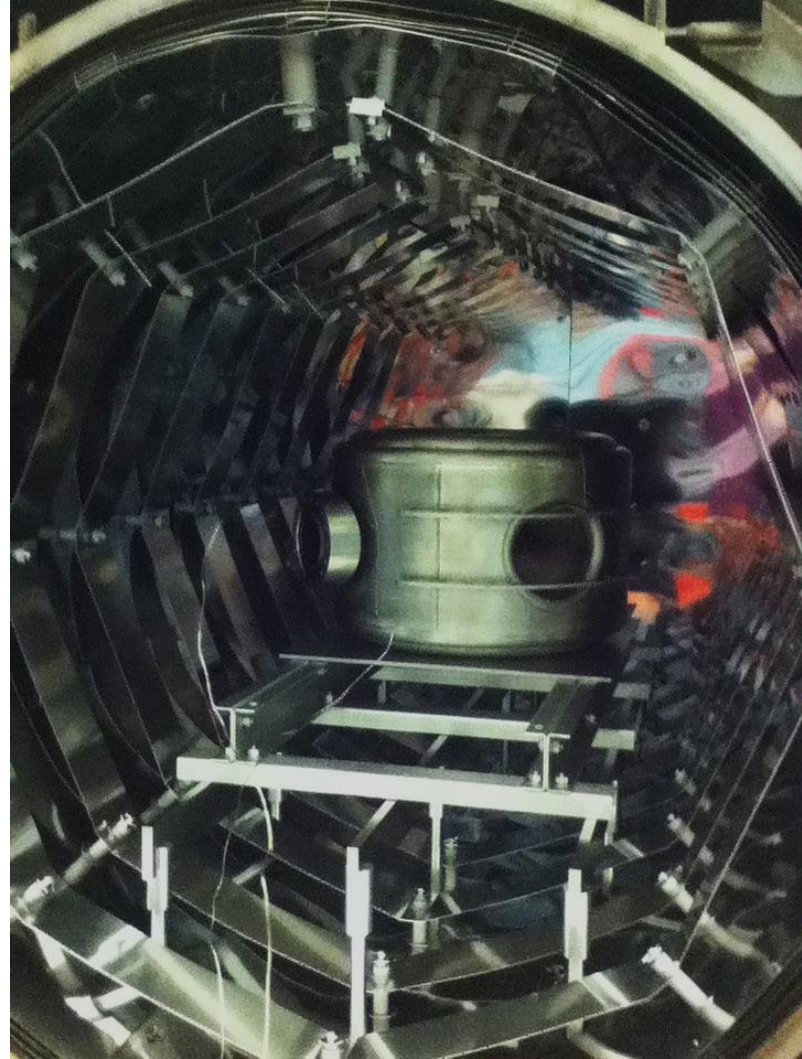




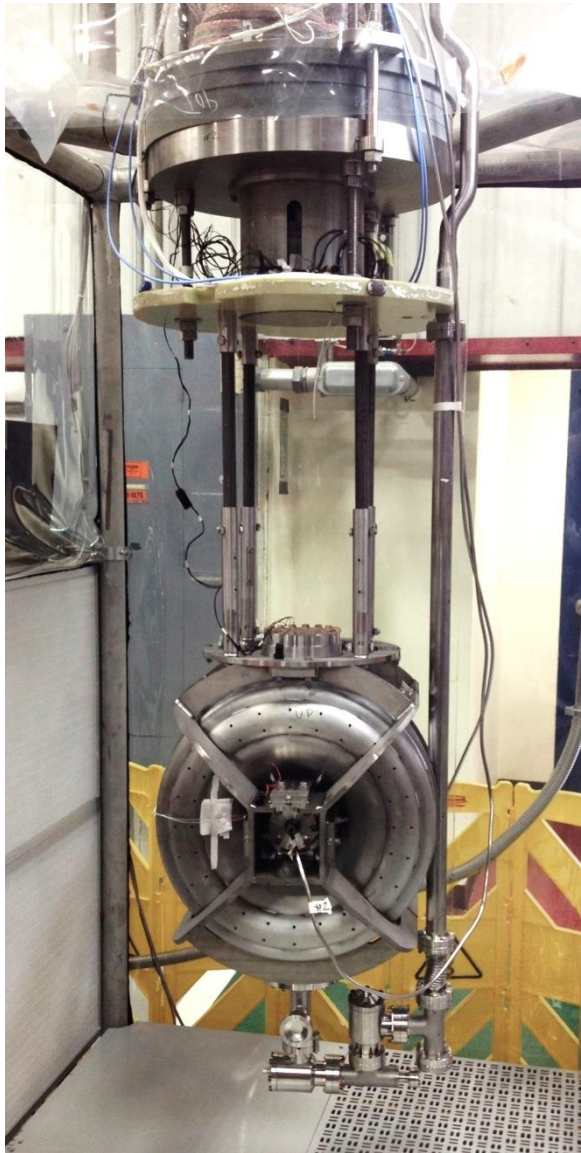
# Hydrogen Degassing



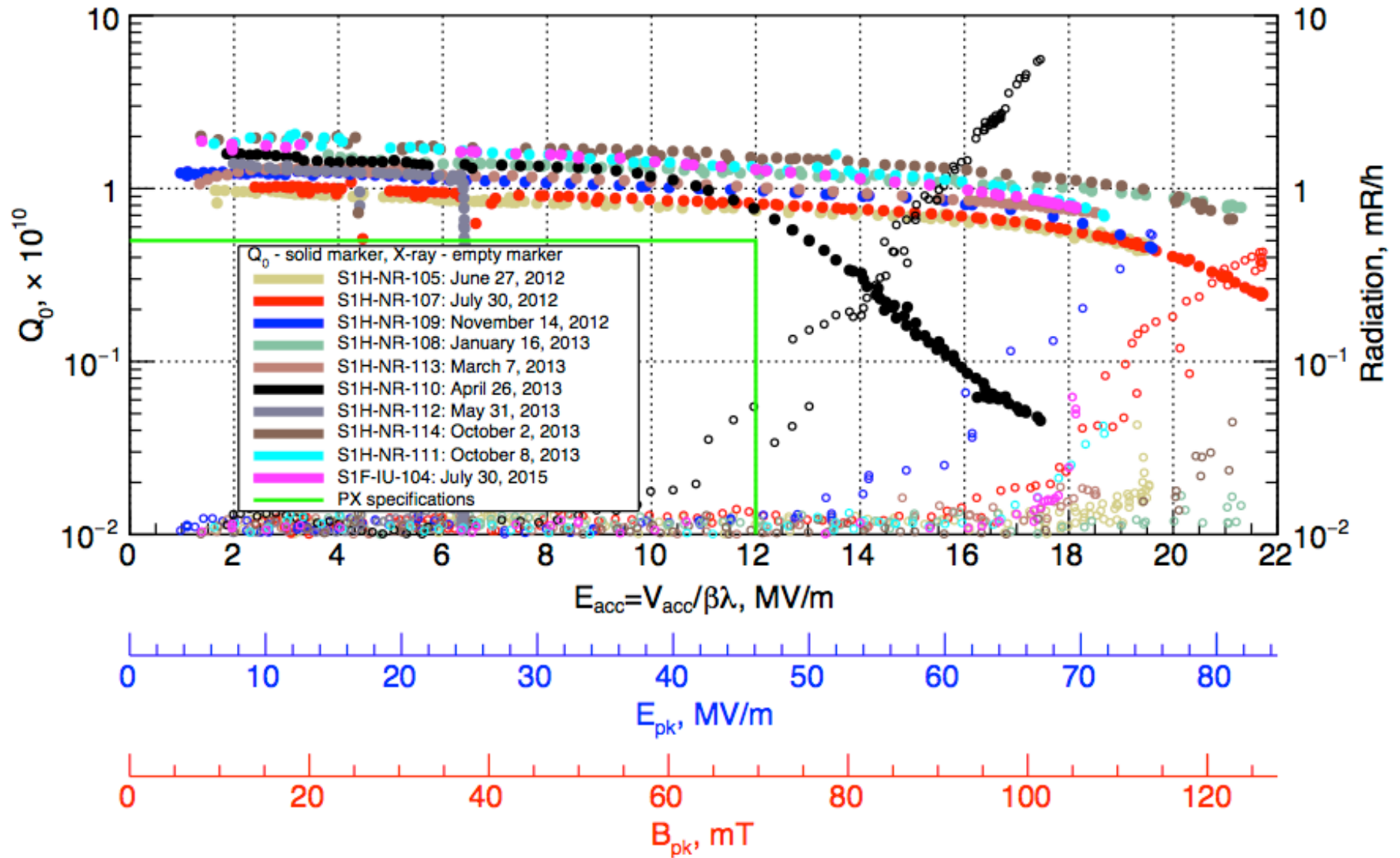
- 600 C
- 10 h
- $< 5$  C/min
  
- Concern about baking vacuum flanges at 800 C due to Cu-Braze joints



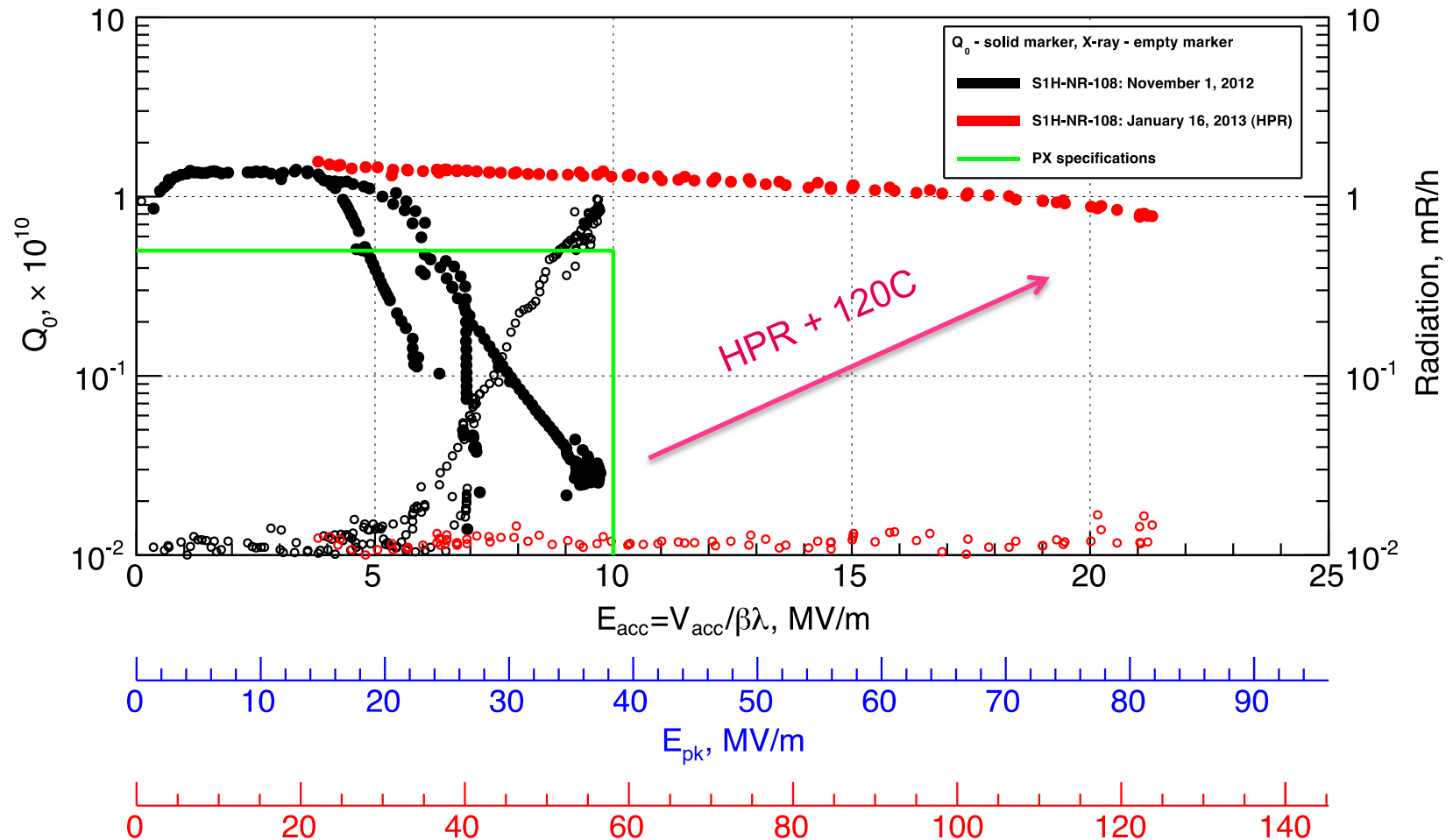
# Installation on Top-Plate



# Vertical Tests of 10 SSR1 Cavities (bare)



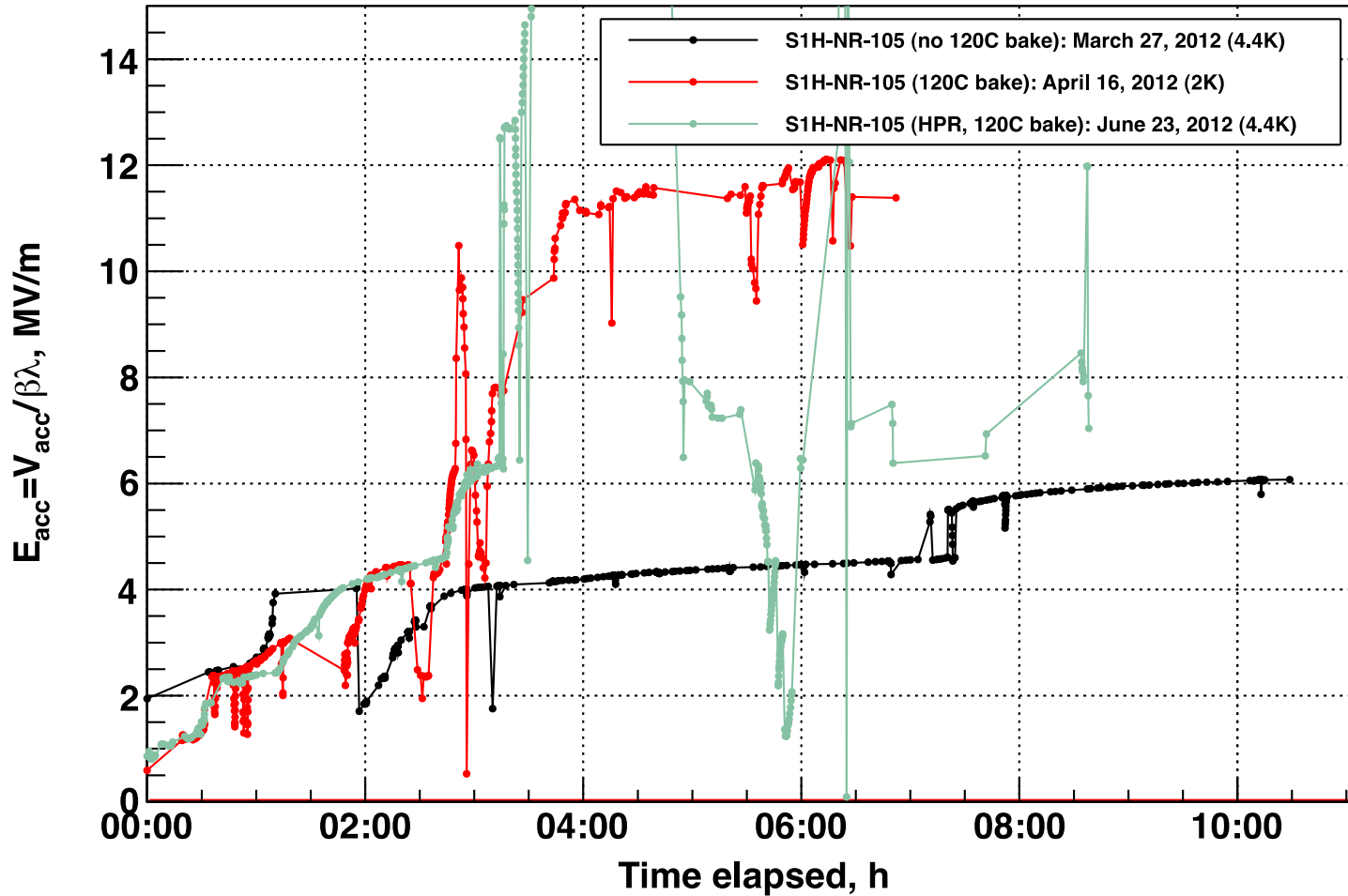
# Example of S108 – 1<sup>st</sup> and 2<sup>nd</sup> pass



courtesy:  
A. Sukhanov



# Multipacting Processing vs. 120C Bake



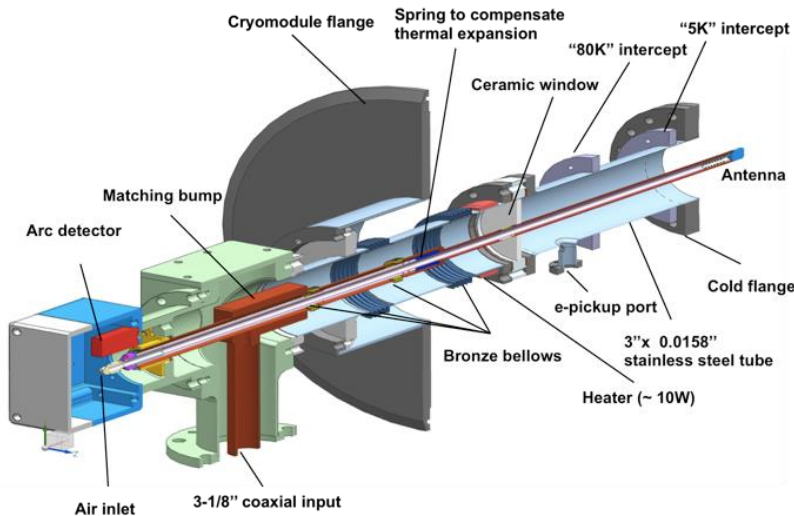


# SSR1 325MHz Coupler: Status

## Design specifications

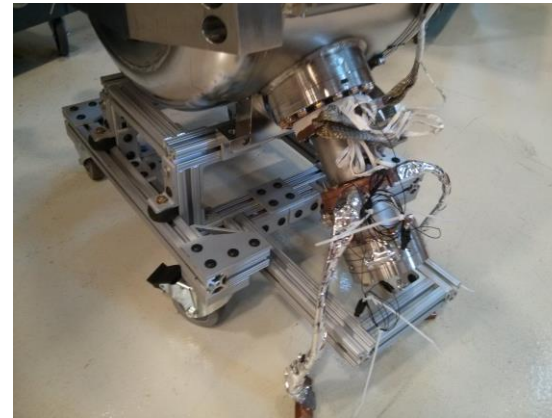
- Beam power gain per cavity (CW): ~2 kW.
- Maximum design power (PIP-II, 5 mA): ~30 kW.
- One ceramic window at room temperature.
- No external adjustment.
- Air cooled center conductor.

## 325 MHz coupler anatomy



## Prototype Couplers

Three prototype couplers successfully tested to 8.5 kW at room temperature.  
One prototype coupler tested in STC at the maximum design power of 30 kW.

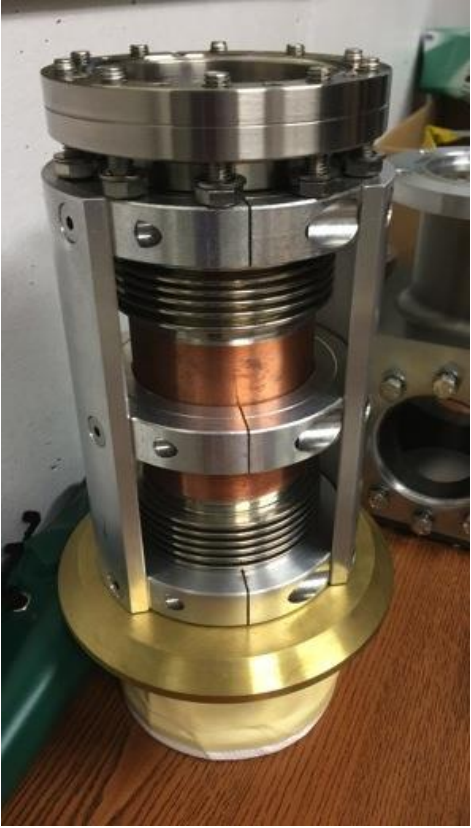


## Production couplers

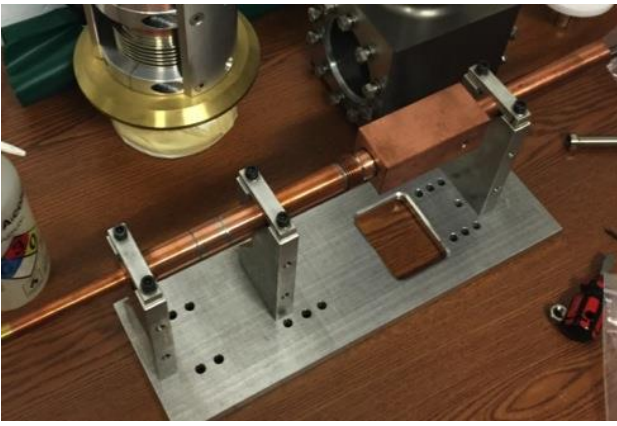
There is some delay on the production couplers because the design was changed to address several issues.  
All 10 production couplers (cold-ends) will arrive at Fermilab in mid-December.  
They are needed for qualification of cavities in STC.

*Status of 325 MHz Main Couplers for PXIE, S. Kazakov, Proceedings of LINAC2014, Geneva, Switzerland*

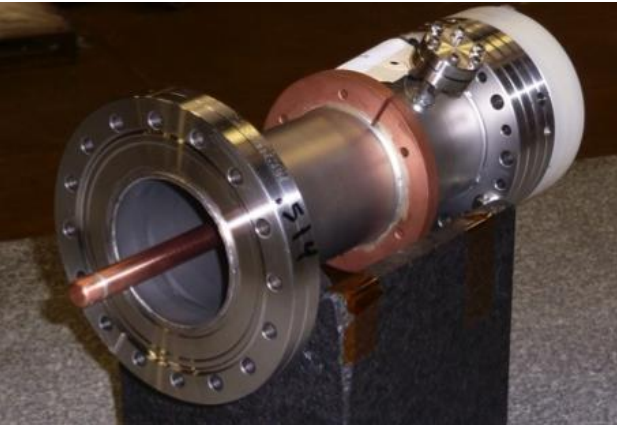
# SSR1 325MHz Coupler: prototypes



Warm Outer conductor  
Electro-deposited bellows  
(Cu-Ni layers)



Warm Inner conductor



Cold-end assembly

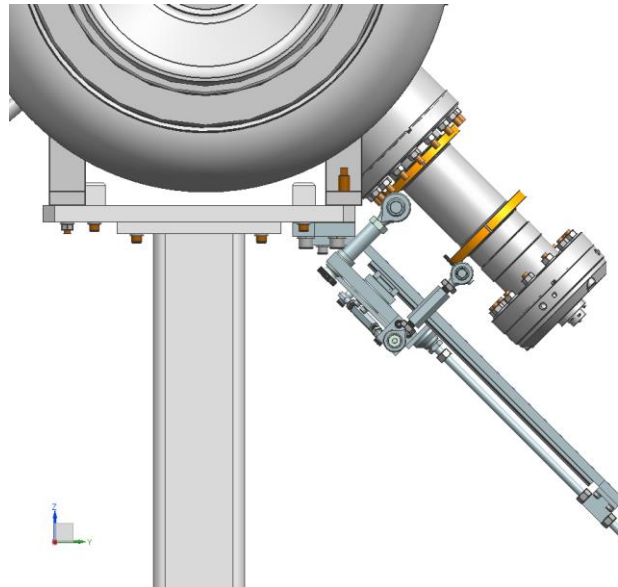
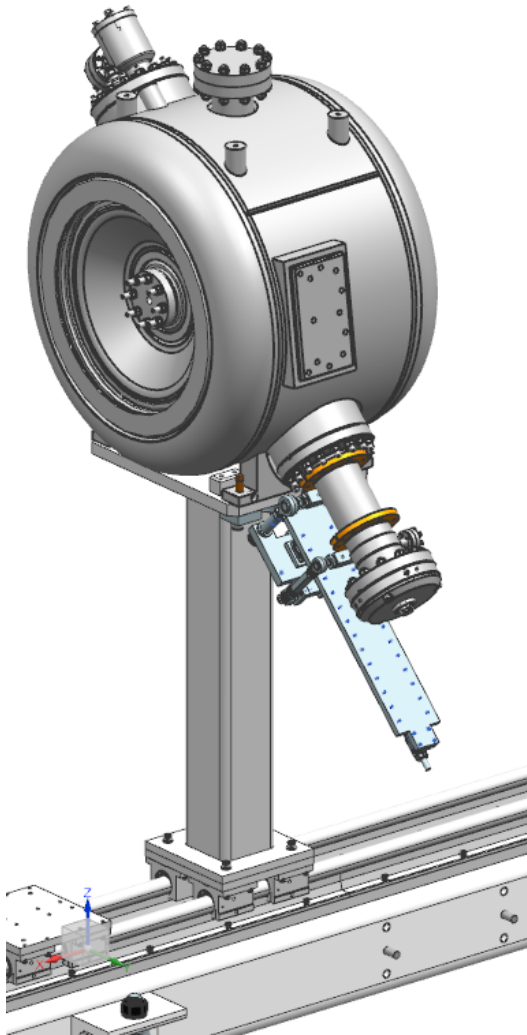


RF test stand

*Test Stand for 325 MHz Power Couplers, S. Kazakov, Proceedings of LINAC2014, Geneva, Switzerland*

# SSR1 325MHz Coupler: installation

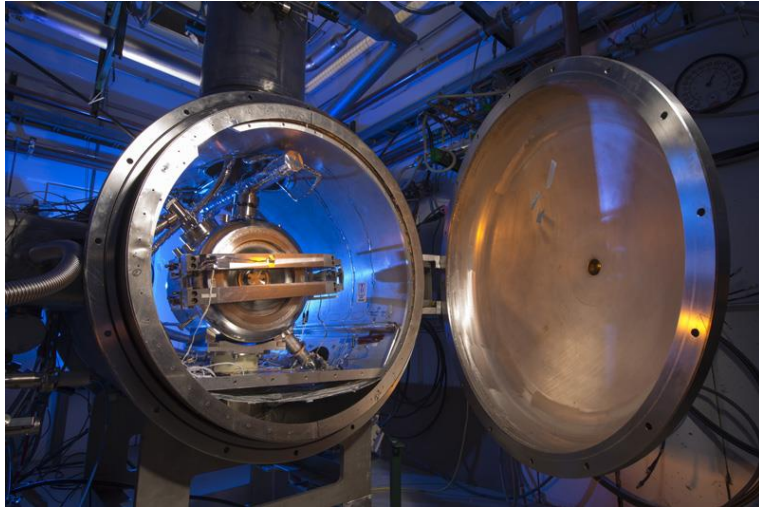
The cold-end of the coupler will be assembled on the cavity in a cleanroom (Class 10) using a specific tool and procedure for a “particle-free installation”.



Unity coupler was replaced with prototype high-power coupler



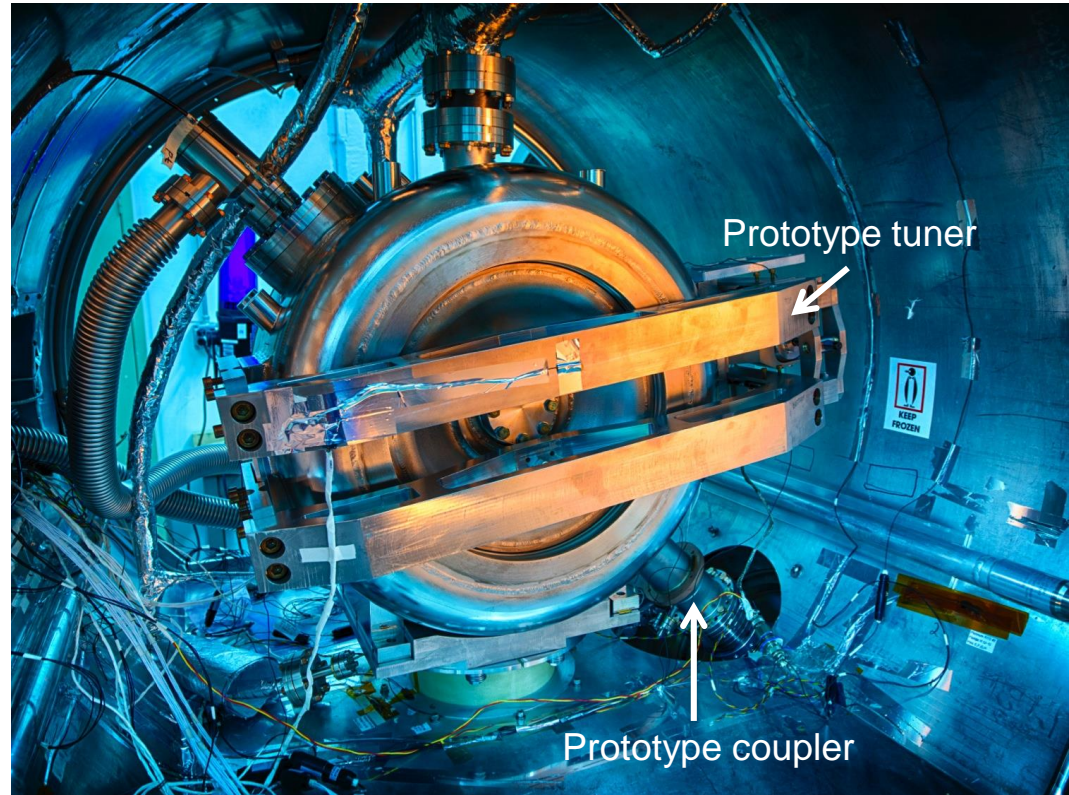
# Fully Integrated Test in the Spoke Test Cryostat



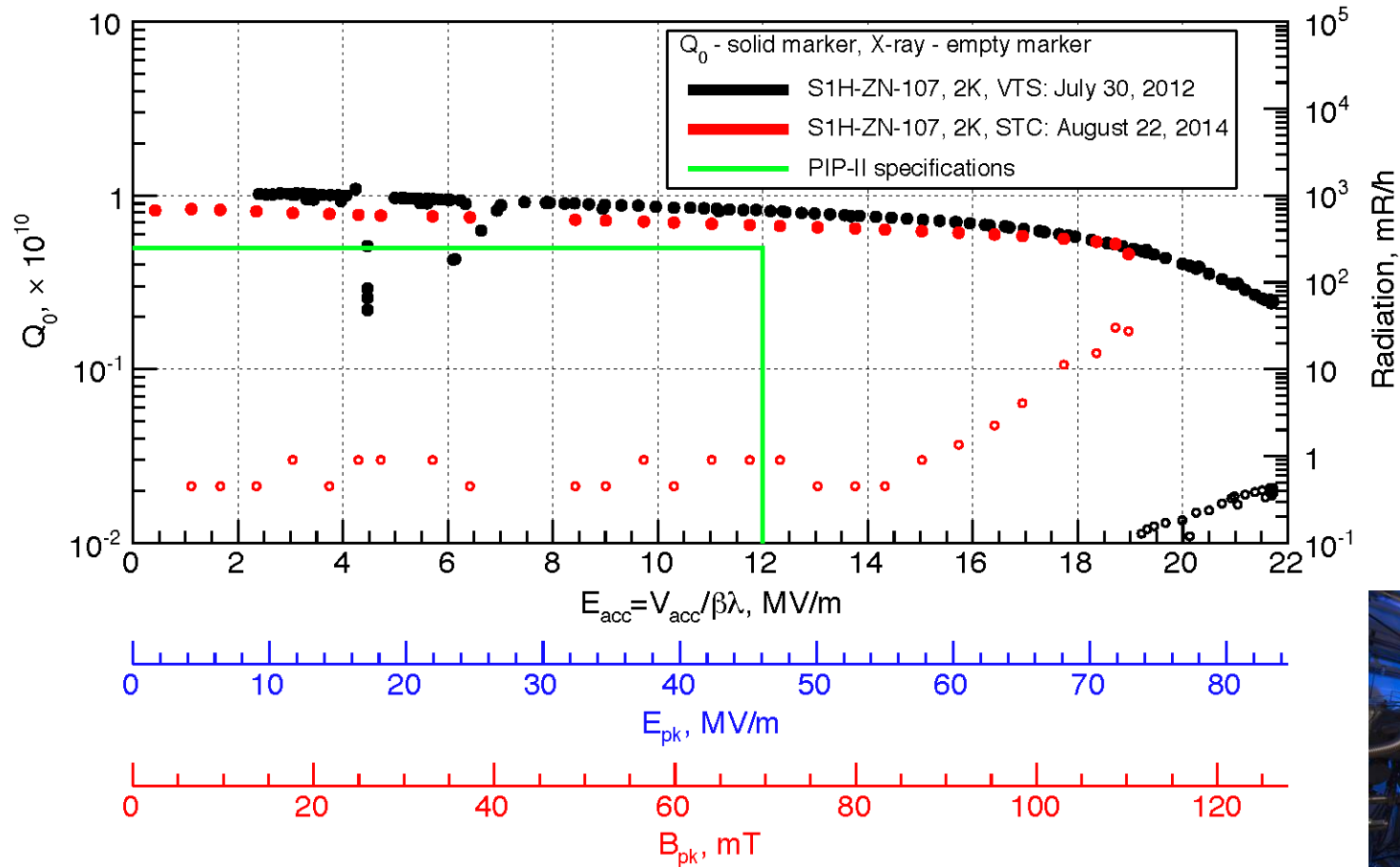
The jacketed SSR1 cavity (S1H-NR-107) dressed with prototype coupler and tuner was tested in the Spoke Test Cryostat at 2K. The performance of the power coupler and the frequency-tuning system were tested making sure they didn't interfere or degrade the performance of the cavity.

## Results:

- Design of coupler validated: Prototype successfully tested to maximum power 30 kW.
- Design of Tuner validated...
- Jacketed cavity exceeds the specifications...

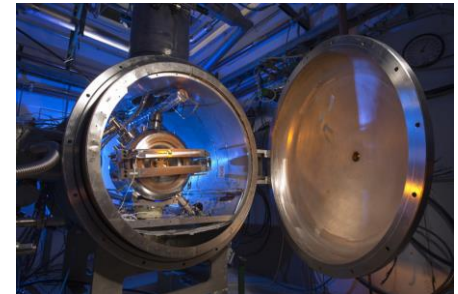


# Testing at STC: $Q_0$ vs. $E_{acc}$ curve (S1H-NR-107)



No  $Q_0$  degradation compared to VTS results  $\rightarrow$   
 Manufacturing process of the jacketed cavity validated.

PIP-II specification:  
 $Q_0 > 5 E9$   
 $E_{acc} = 12 \text{ MV/m}$

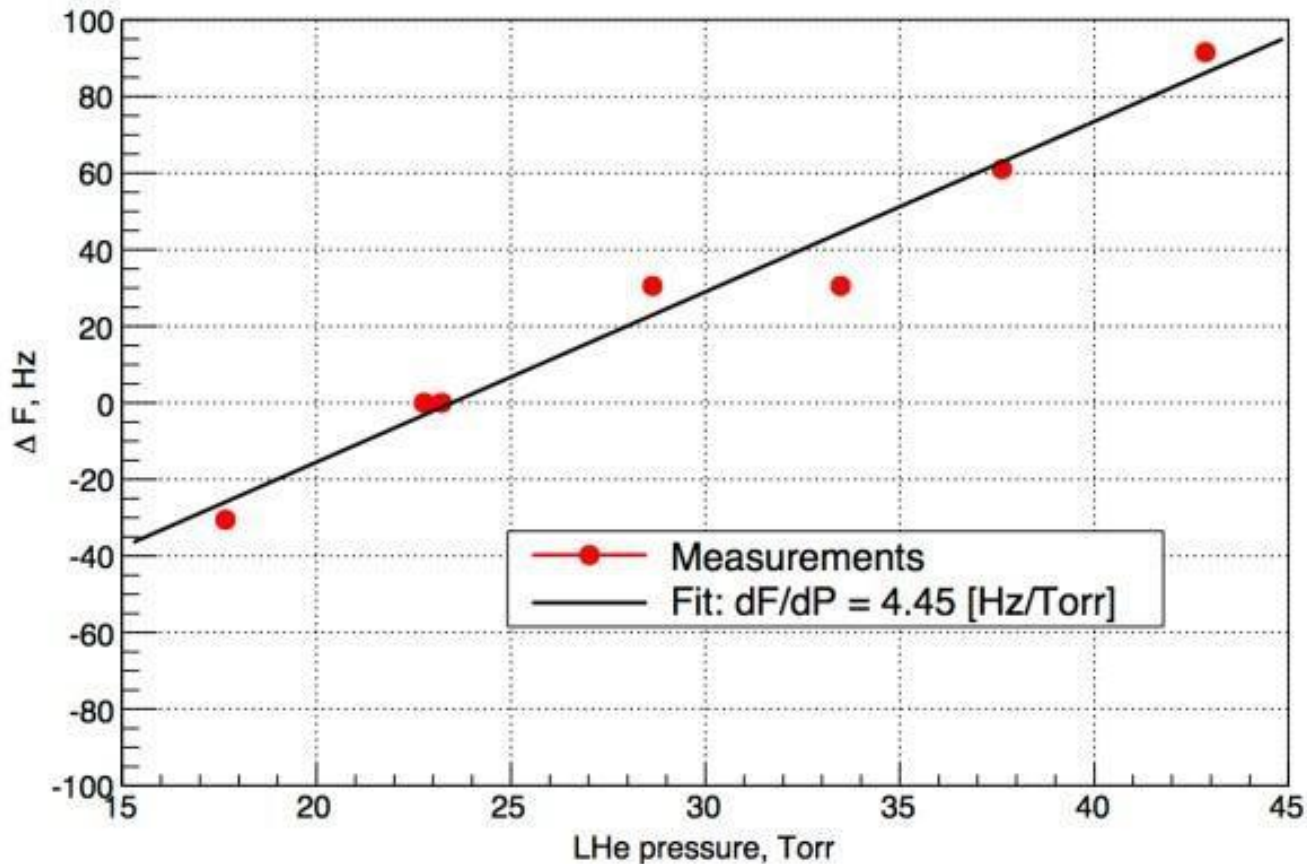


[Result of Cold Tests of the Fermilab SSR1 Cavities](#), A. Sukhanov et al., Proceedings of LINAC2014, Geneva, Switzerland



# Testing at STC: Cavity Pressure Sensitivity (@ 2K)

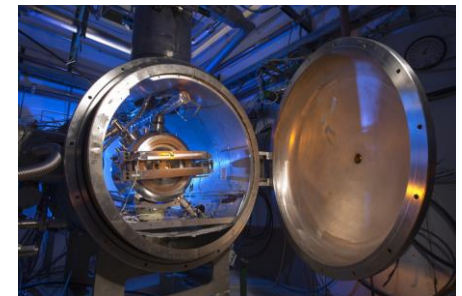
Pressure sensitivity measured with Tuner engaged (as in operating condition)



PIP-II specification  
 $df/dp \leq 25$  Hz/Torr

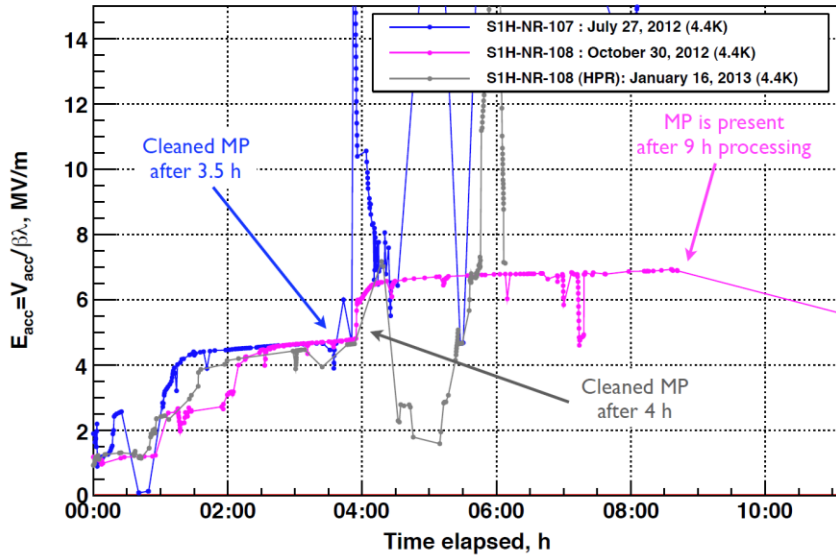
Design procedure to minimize the pressure sensitivity of the jacketed cavity with tuner is validated.

Estimated at room temperature:  
 $df/dp = 4$  Hz/Torr

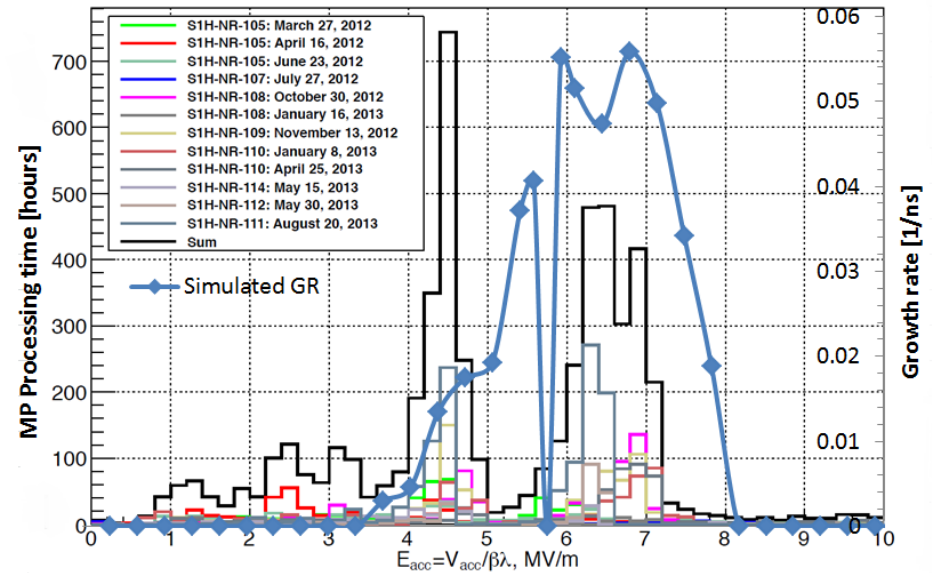
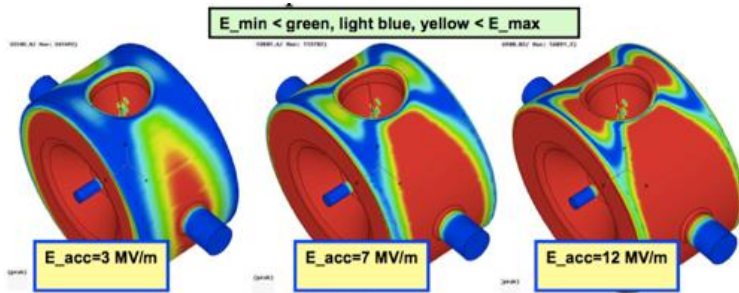


[RF Tests of Dressed 325 MHz Single-Spoke Resonators at 2K](#), A. Hocker et al., Proceedings of LINAC2014, Geneva, Switzerland

# Testing at STC: Multipacting



Multipacting was simulated in SSR1 cavities for PIP-II, the results of SSR1 simulations have been compared with experimental data and the agreement seems very good.



[Simulation of Multipacting in SC Low Beta Cavities at FNAL](#), G. Romanov et al., Proceedings of IPAC 2014, Richmond, VA, USA  
[Multipacting Simulations of SSR2 Cavity at FNAL](#), P. Berrutti et al., Proceedings of NA-PAC 2013, Pasadena, CA, USA

# SSR1 Tuner: Requirements and data

<i>Parameter</i>	<i>Value</i>	<i>Notes</i>
<b>Total Frequency Range</b>	> 135 kHz	From FRS
<b>Frequency Resolution of stepper motor</b>	< 20 Hz	From FRS
<b>Piezo Frequency Range</b>	> 1 kHz	From FRS
<b>Tuner Passive spring constant</b>	30 kN/mm	Derives from df/dP requirement
<b>Sensitivity of end-wall</b>	540 kHz/mm	Simulation/Experimental
<b>Cavity wall spring constant (<math>K_{cav}</math>)</b>	30 kN/mm	Simulation/Experimental

<i>Parameter</i>	<i>Value</i>	<i>Notes</i>
<b>Stepper motor max force</b>	$\pm 1300$ N	Symmetrical
<b>Stepper motor resolution*</b>	0.1 $\mu$ m (100 nm)	At interface with 2 <sup>nd</sup> lever
<b>Piezo stroke @ RT</b>	64 $\mu$ m $\pm$ 2%	Measured
<b>Piezo stroke @ operating T</b>	15 $\mu$ m (25% of RT)	
<b>Piezo max rated force</b>	3360-5040 N	4200 N $\pm$ 20% (blocking force)
<b>Piezo max operating force</b>	2688 N	3360 $\cdot$ 80%

<i>Parameter</i>	<i>Value</i>	<i>Notes</i>
<b>Motor Travel at beam pipe</b>	> 0.25 mm	135/540 kHz
<b>Piezo Travel at beam pipe</b>	> 1.85 $\mu$ m	1/540 kHz
<b>Maximum Force at beam pipe</b>	7500 N	0.25 mm $\cdot$ 30000 N/mm
<b>Motor Resolution at beam pipe</b>	< 37 nm	20/540000 mm
<b>Motor Tuning Efficiency (<math>T_e</math>)</b>	< 37 %	37/100 nm
<b>Motor Mechanical Advantage (<math>M</math>)</b>	> 5.8	7500/1300 N, <b>picked 6</b>
<b>Piezo Tuning Efficiency (<math>T_e</math>)</b>	> 12 %	1.85/15 $\mu$ m
<b>Piezo Mechanical Advantage (<math>M</math>)</b>	> 1.4	0.5* $\cdot$ 7500/2688 N, <b>picked 2</b>
<b>Piezo Elastic Efficiency (<math>E</math>)</b>	> 24 %	2 $\cdot$ 12 % ( $T_e \cdot M$ )

# SSR1 Tuner

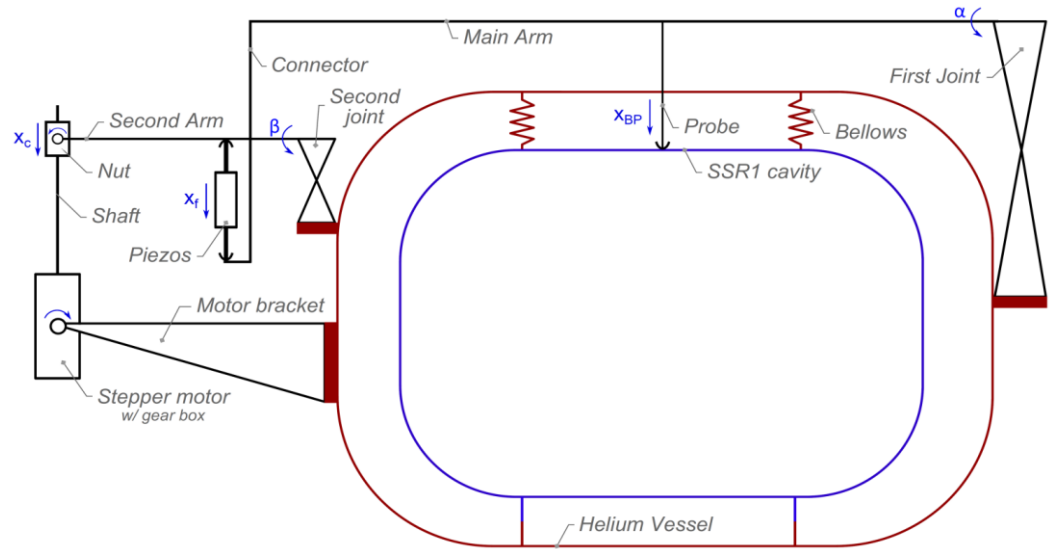
## Specification

- Fine tuning >1 kHz
- Coarse tuning > 135 kHz
- Made of SS316L

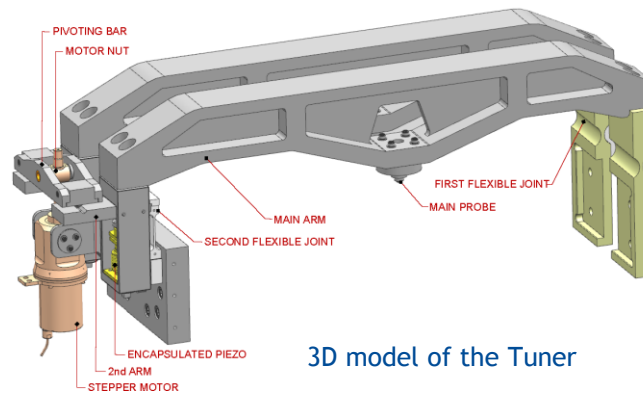
## Double lever Tuner

- Well known technology
- Adjustable mechanical advantage
- Piezos and motor in series
- Piezos away from the beam
- Estimated efficiency and stiffness
- Respect of the following maximum forces at the actuating components:

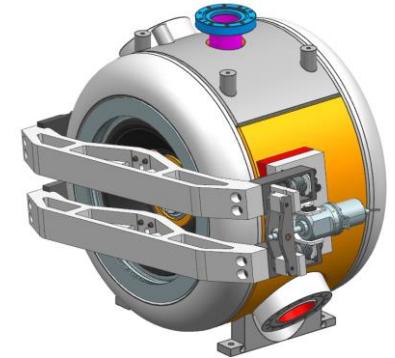
Parameter	Value
<i>Stepper motor with gear box</i>	
Max force	1300 N
Resolution	0.1 $\mu\text{m}$
<i>Piezo</i>	
Stroke ( $x_f$ ) at 293 K	68 $\mu\text{m}$
Stroke ( $x_f$ ) at 20 K	15 $\mu\text{m}$
Max operating force	2700 N
Min operating force	840 N



Schematic representation of the working principle of the SSR1 tuner.



3D model of the Tuner



3D model of the Tuner assembled on the SSR1-G3 cavity

[SSR1 Tuner Mechanism: Passive and Active Device](#), D. Passarelli et al., Proceedings of LINAC2014, Geneva, Switzerland.

# SSR1 Tuner: Testing

## Operating mode

**Coarse tuning** *135 kHz measured*  
 $\Delta f_c \geq 135 \text{ kHz} \rightarrow x_{BPC} \geq 250 \mu\text{m}$

Active compensation of uncertainty due to **cooldown, preload the system**

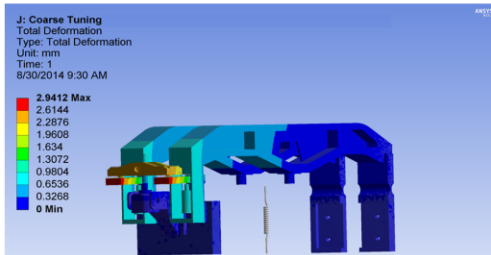
**Fine tuning** *2.5 kHz measured*  
 $\Delta f_f \geq 1 \text{ kHz} \rightarrow x_{BPF} \geq 1.85 \mu\text{m}$

Actively compensate the frequency shifts due to **microphonics**

**Passive tuning** *+4.5 Hz/Torr measured*  
 $k_{pass} \geq 30 \text{ N}/\mu\text{m} \rightarrow \frac{df}{dp} \leq 25 \text{ Hz/Torr}$

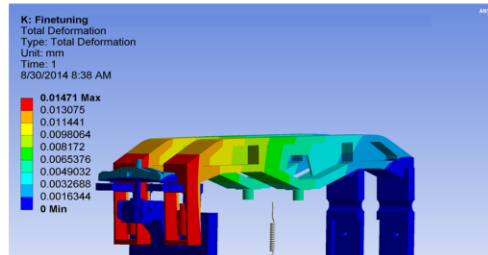
Passively minimize the **pressure sensitivity of the cavity**

- FE analyses to simulate the three operating conditions and verify the value of stiffness and efficiency *...and Testing Results*



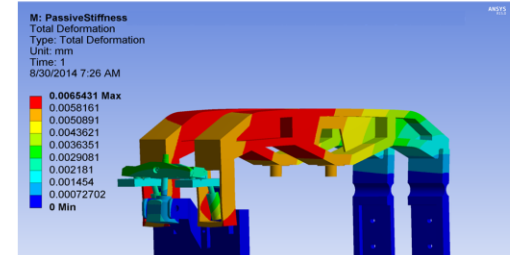
$$E_{ct} = \frac{x_{BPC}}{x_c} = \frac{260}{2500} = 10.4\% \leq 37\% \quad \text{5.5\% measured}$$

$$x_{BPC} = \frac{F_m^{max}}{k_c} = 260 \mu\text{m} \geq 250 \mu\text{m}$$



$$E_{ft} = \frac{x_{BPF}}{x_f} = \frac{5.30 \mu\text{m}}{15 \mu\text{m}} = 35\% \geq 17\% \quad \text{44\% measured}$$

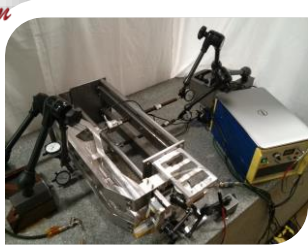
$$x_{BPF} = \frac{F_p^{max}}{k_f} = 5.30 \mu\text{m} \geq 1.85 \mu\text{m}$$



$$k_{pass} = \frac{F_{BP}}{x_{BP}} = 40 \text{ N}/\mu\text{m} \geq 30 \text{ N}/\mu\text{m}$$

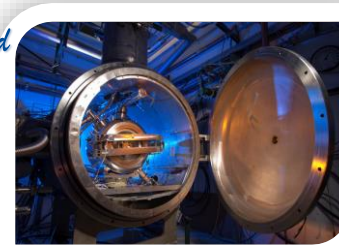
$$\frac{df}{dp} = \frac{F_{BP}}{x_{BP}} = +4 \text{ Hz/Torr} \leq 25 \text{ Hz/Torr} \quad \text{+4.5 Hz/Torr measured}$$

*Tests at Room Temperature*



Tuner mounted on a test stand for initial measurement at room temperature.

*Tests at Cold Temperature*

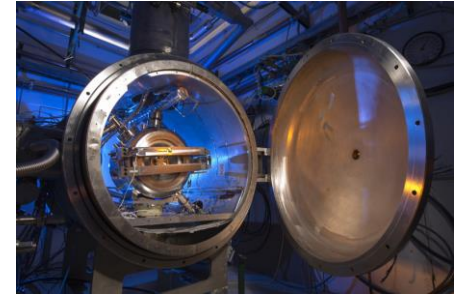
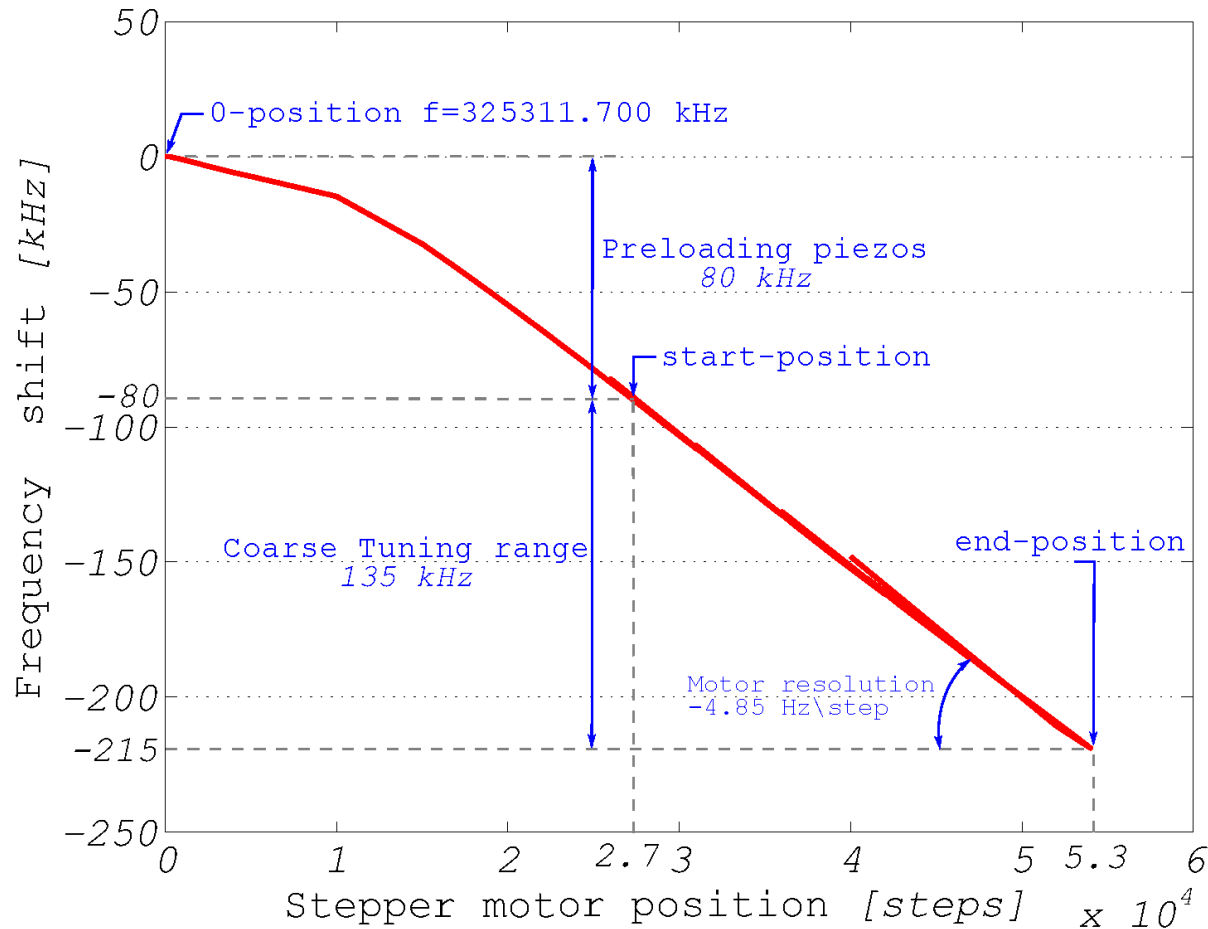


Installation of SSR1 cavity (S1H-NR-107) with tuner and coupler into the Spoke cavity Test Cryostat (STC) at Fermilab.

[SSR1 Tuner Mechanism: Passive and Active Device](#), D. Passarelli et al., Proceedings of LINAC2014, Geneva, Switzerland.



# SSR1 Tuner: Testing

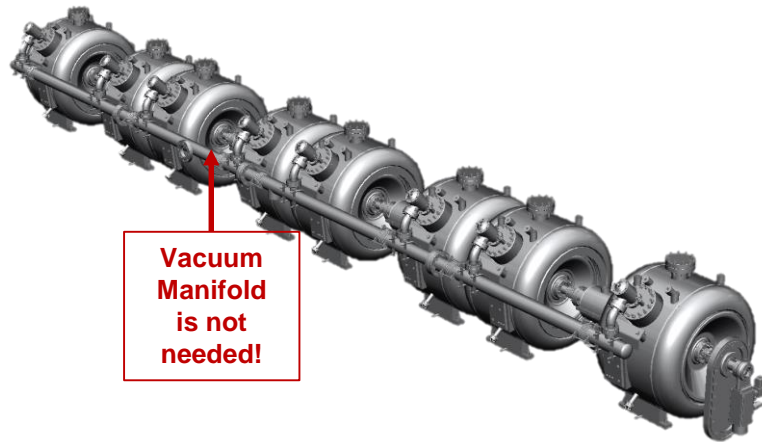


Studies of microphonics control of the SSR1 cavity have been carried out using this tuner mechanism.

Performance of the Tuner mechanism for SSR1 Resonators During Fully Integrated Tests at Fermilab, D. Passarelli et. al., Proceedings of SRF2015, Whistler, Canada, THPB061

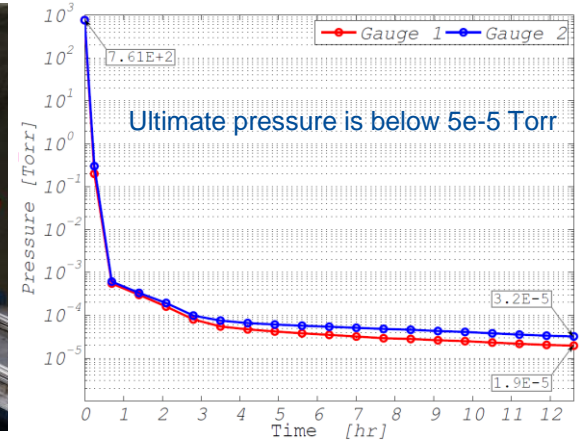
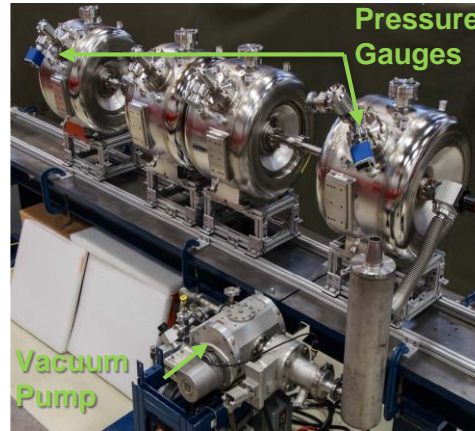
# SSR1 String Assembly: Beam-line vacuum

High vacuum level ( $< 5E-5$  Torr) is needed inside the beam line volume before the introduction of liquid helium in less than 12 hours.

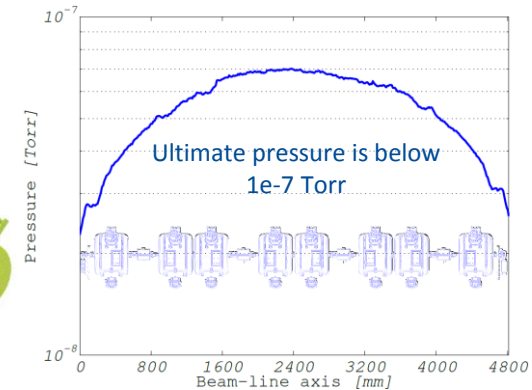
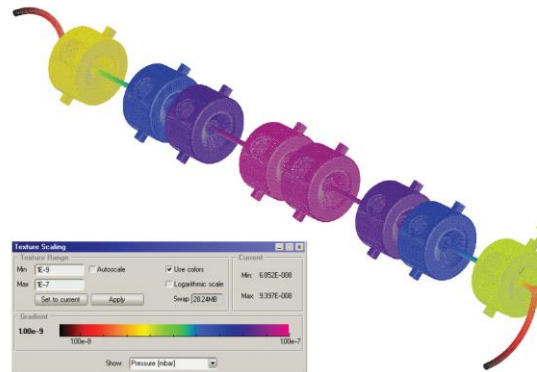


The high-vacuum level at room temperature can be achieved pumping down by the beam ports only. Furthermore, simulations performed on the entire string with clean components show that the achievable pressure would be of  $7E-8$  Torr pumping from both ends.

## ➤ Measurements (very conservative conditions)



## ➤ Vacuum simulation (best scenario)



[High-vacuum Simulations and Measurements on the SSR1 Cryomodule Beam-line](#), D. Passarelli et al., Proceedings of SRF2015, Whistler, BC, Canada

# SSR1 String Assembly: Alignment (in progress...)

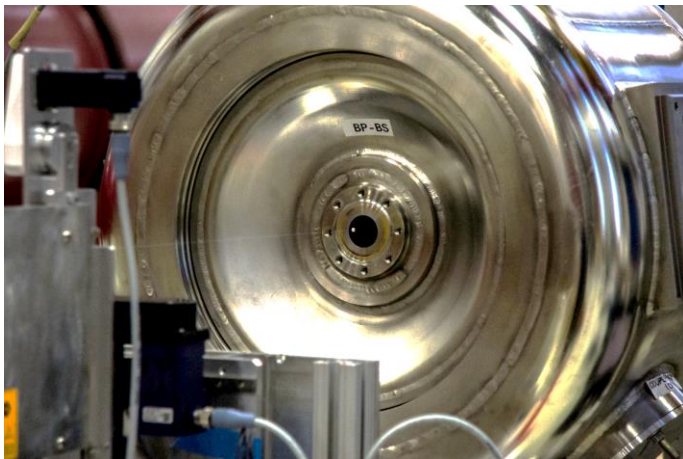
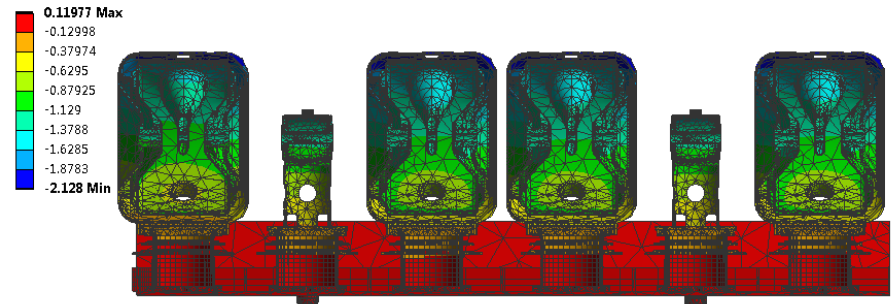
## Alignment specifications (ED0001316)

Transverse cavity alignment error, mm RMS	<1
Angular cavity alignment error, mrad RMS	≤10
Transverse solenoid alignment error, mm RMS	<0.5
Angular solenoid alignment error, mrad RMS	<1

Referencing the electric axis of the cavity to external fiducials (beadpull + optical measurements + laser tracker). Error:  $\sim \pm 150\mu\text{m}$

Estimation of cavities and solenoids misalignments due to the cooldown (293K --> 2K)

C: Static Structural  
Directional Deformation  
Type: Directional Deformation(Y Axis)  
Unit: mm  
Global Coordinate System  
Time: 1  
9/23/2015 2:17 PM



The positioning of the components in the string assembly will be defined starting from these results...

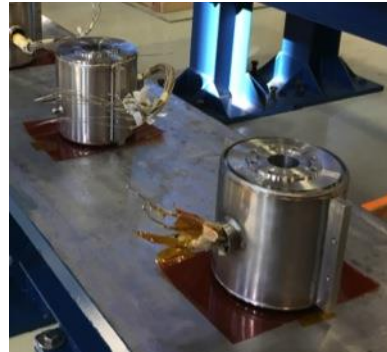


# SSR1 String Assembly: components available

All components of the SSR1 string assembly were ordered and most of them are received.



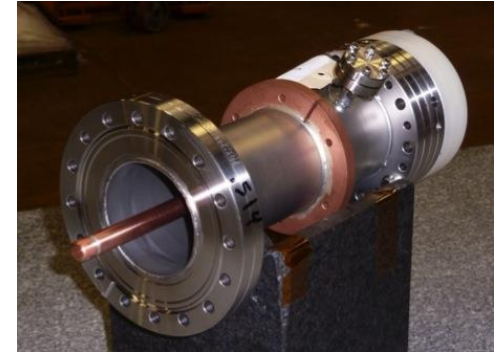
Jacketed SSR1 cavities  
(4 Type-A, 4 Type-B)



4 Solenoids



4 BPMs



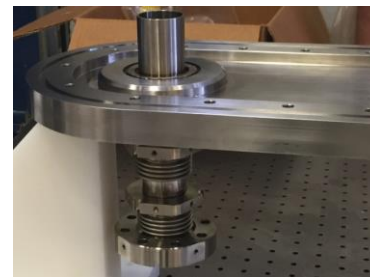
Production Couplers will  
arrive in mid-December



All Hardware and Seals



Interconnecting Bellows



Warm-end transitions  
and gate valves



Tooling

# Other cryomodule components received as of today..



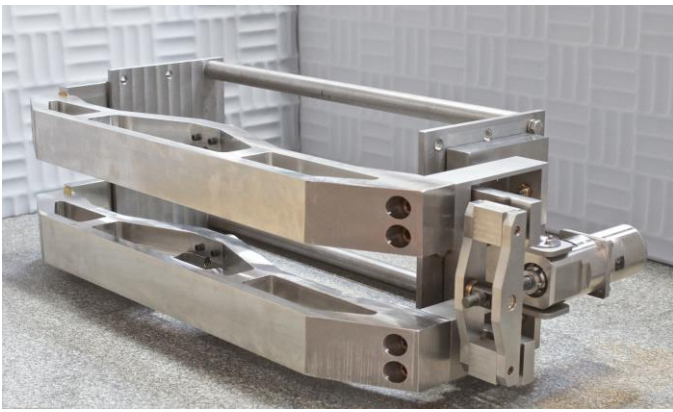
Vacuum vessel



Strong-back



Support Posts



Prototype Tuner mechanism