



Wrocław University of Technology

Impact of high design pressures of the cryogenic transfer lines on heat inleaks

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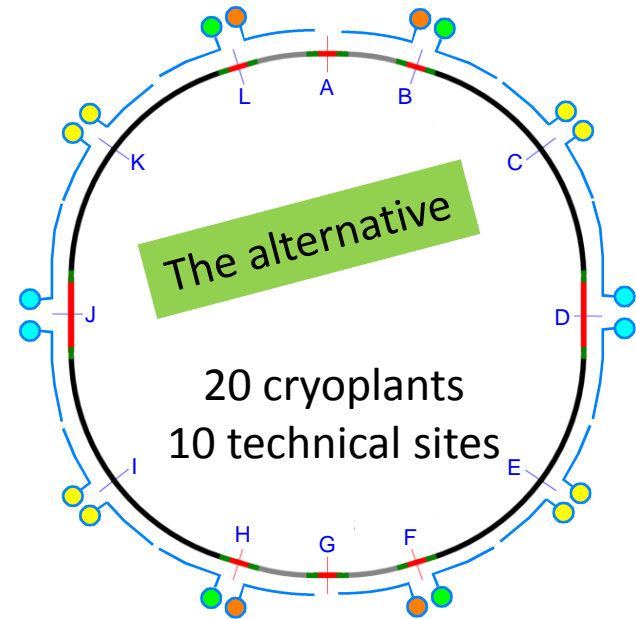
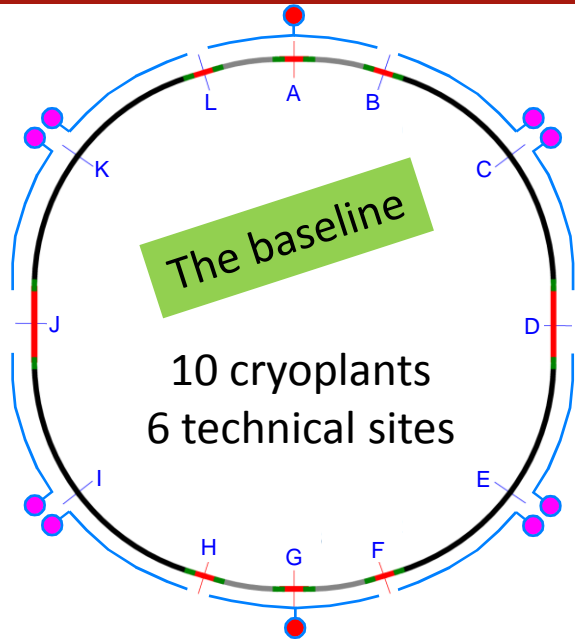


Contents

1. Introduction - FCC cryogenic system
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4. Comparison on the heat fluxes and failure rates
5. Summary



FCC Cryogenic system

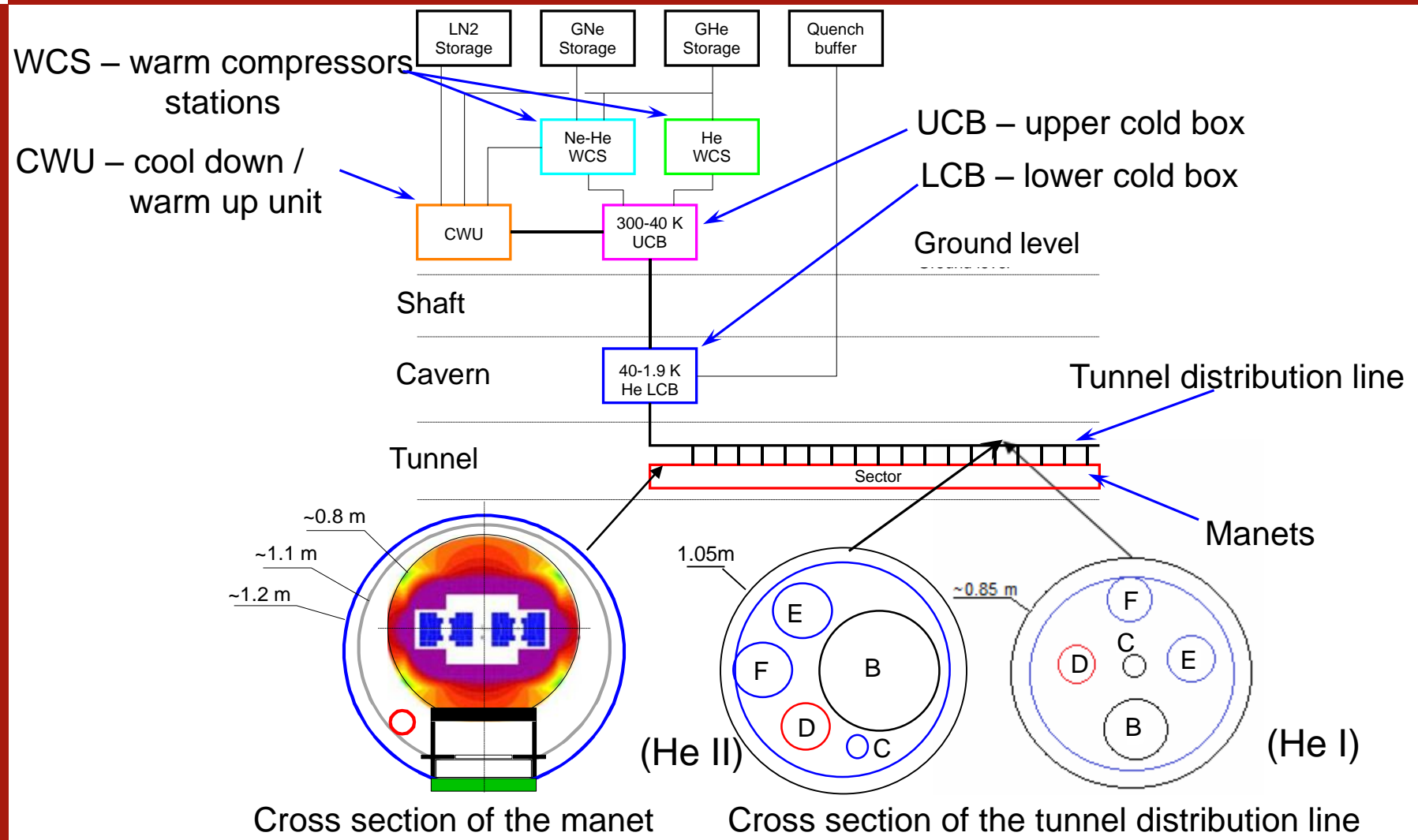


Cryoplant	L Arc+DS km	L distribution km
	$2 \times 4 = 8$	$2 \times 4.7 = 9.4$
	8.4	8.4

Cryoplant	L Arc+DS km	L distribution km
	4	4.7
	4.4	5.1
	4	4
	4.4	6.5

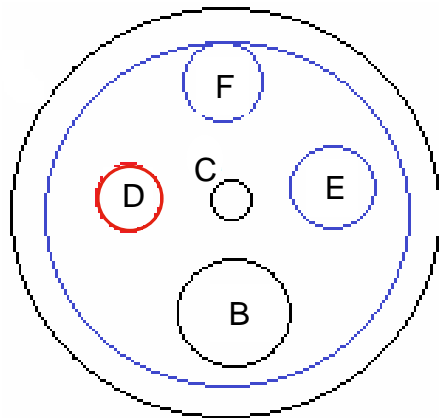


FCC Cryogenic system

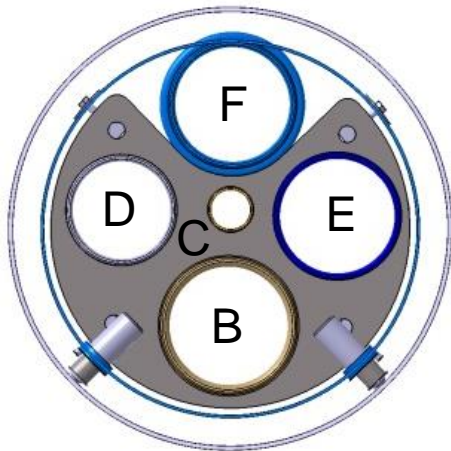




Specification of the process pipes of tunnel distribution line of FCC



(He I)



Header	Function	DN mm	Nom. T K	Nom. P _N bar	Design P _D bar	Test P _T bar
Header B	Pumping line	250	4	0.5	4	6
Header C	SHe supply	80	4.6	3	20	29
Header D	Quench line and current lead He supply	200	40	1.3	20	29
Header E	Thermal shield and beam screen He supply	240	40	20 (50)	20 (50)	29 (71.5)
Header F	Thermal shield and beam screen He return	240	60	15 (45)	20 (50)	29 (71.5)
Vacuum jacket	Insulation vacuum enclosure	850	300	0	0 -1.05	1.5

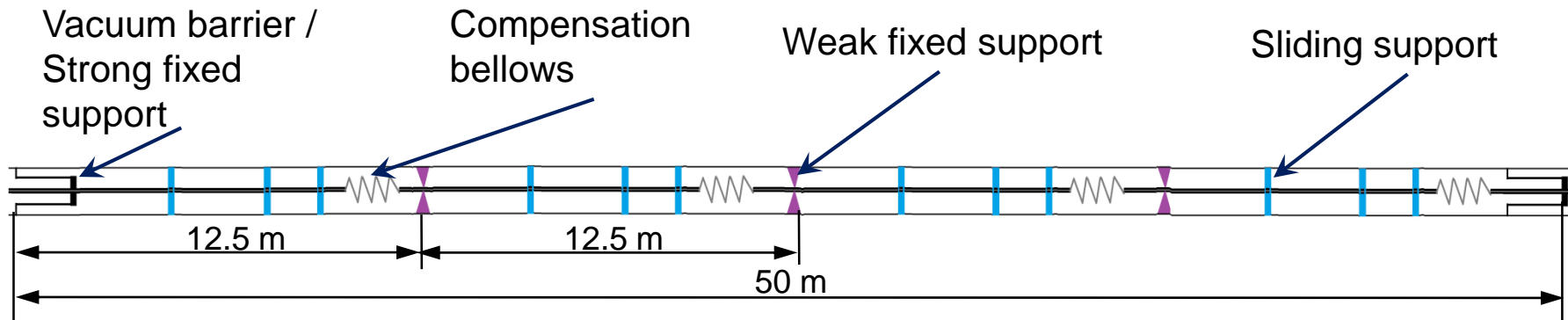


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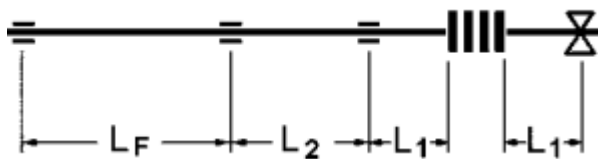
Scheme of the transfer line with stainless steel process pipes





Transfer line compensation and internal support system

Determination of the distance between supports



$$L_1 = 3 \cdot DN - \text{for axial expansion joints}$$

$$L_2 = 0.5 \cdot L_F$$

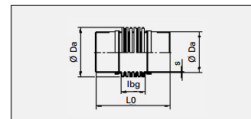
L_F - depends on the accepted deflection and the risk of pipe buckling

The forces from pressure and thermal shrinkage

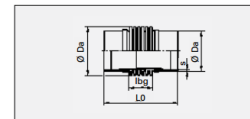
$$F_{NP} = F_P - F_{TS} - \text{for nominal parameters}$$

$$F_{TS} = \Delta L \cdot c_{\delta} - \text{depends on nominal temperature}$$

$$F_P = A \cdot P - \text{the highest value for the pressure test}$$



Type ARN without inner sleeve



Type ARN with inner sleeve

Nominal diameter	Nominal axial movement absorption	Type ARN 25 ...	Order No. standard version		Overall length	Weight approx.		Weld ends	
			without inner sleeve	with inner sleeve		without inner sleeve	with inner sleeve	outside diameter	wall thickness
mm	mm		mm	mm	mm	kg	kg	mm	mm
50	17	.0050.017.0	417821	417850	210	1.2	1.4	60.3	4
50	32	.0050.032.0	417822	417851	270	1.8	2	60.3	4
65	21	.0065.021.0	417823	417852	215	1.8	2	76.1	4
65	40	.0065.040.0	417824	417853	292	3.2	3.6	76.1	4
80	23	.0080.023.0	417825	417854	220	2.3	2.6	88.9	4
80	42	.0080.042.0	417826	417855	290	3.6	4	88.9	4
100	23	.0100.023.0	417827	417856	212	2.8	3.1	114.3	4
100	48	.0100.048.0	417829	417857	286	4.6	5.2	114.3	4
125	26	.0125.026.0	417830	417858	240	3.9	4.4	139.7	4
125	52	.0125.052.0	417831	417859	304	5.3	6.1	139.7	4
150	29	.0150.029.0	417832	417860	240	4.9	5.5	168.3	4.5
150	58	.0150.058.0	417833	417861	304	6.8	7.7	168.3	4.5
200	26	.0200.026.0	417835	417862	252	8.5	9.4	219.1	6.3
200	52	.0200.052.0	417836	417863	324	11.3	12.6	219.1	6.3
200	71	.0200.071.0	417837	417864	378	15.2	17.1	219.1	6.3
250	24	.0250.024.0	417838	417865	240	11.5	12.5	273	7.1
250	48	.0250.048.0	417839	417866	300	15.1	16.5	273	7.1
250	79	.0250.079.0	417840	417867	380	19.8	22	273	7.1

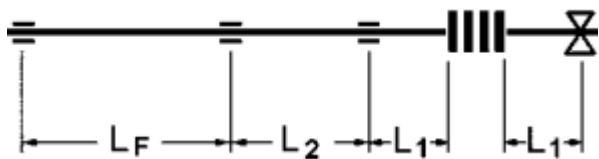
Bellows		effective cross-section	Nominal movement absorption ¹⁾ for 1000 loading cycles		Adjusting force rate		
outside diameter	corrugated length		angular ²⁾	lateral ³⁾	axial	angular	lateral
mm	mm	cm ²	degrees	mm	N/mm	Nm/degrees	N/mm
D _a	l _b	A	2α _N	2λ _N	c ₁	c ₂	c ₃
90	50	46.6	22	3.5	321	4.2	1113
91	110	47.2	33	15	199	2.6	144
109	55	70.1	23	4.1	272	5.3	1182
111	132	71.6	33	18	218	4.3	166
123	60	90.8	21	4.1	329	8.3	1555
125	130	92.5	32	17	222	5.7	227
151	52	140	18	3	340	13	3302
152	126	141	30	15	218	8.5	361
174	64	187	18	3.6	450	23	3864
174	128	187	29	14	225	12	483
205	64	267	17	3.4	440	33	5410
205	128	267	27	13	220	16	676
261	72	443	12	2.6	855	105	13759
261	144	443	20	11	428	53	1722
262	198	445	23	19	376	46	802
320	60	679	8.7	1.6	1298	245	46135
320	120	679	16	6.4	649	122	5762
320	200	679	21	18	390	74	1245



Transfer line compensation and internal support system

Increases the number of sliding supports

Determination of the distance between supports



$$L_1 = 3 \cdot DN - \text{for axial expansion joints}$$

$$L_2 = 0.5 \cdot L_F$$

L_F - depends on the accepted deflection and the risk of pipe buckling

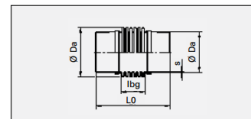
The forces from pressure and thermal shrinkage

$F_{NP} = F_P - F_{TS}$ - for nominal parameters

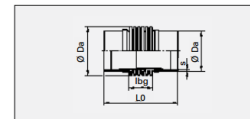
$F_{TS} = \Delta L \cdot c_{\delta}$ - depends on nominal temperature

$F_P = A \cdot P$ - the highest value for the pressure test

The most critical - determines the dimensions of the strong fixed support



Type ARN without inner sleeve



Type ARN with inner sleeve

Nominal diameter	Nominal axial movement absorption	Type	Order No. standard version		Overall length	Weight approx.		Weld ends	
			without inner sleeve	with inner sleeve		without inner sleeve	with inner sleeve	outside diameter	wall thickness
mm	mm	ARN 25 ...	mm	mm	mm	kg	kg	mm	mm
50	17	.0050.017.0	417821	417850	210	1.2	1.4	60.3	4
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250	79	.0250.079.0	417840	417867	380	19.8	22	273	7.1

Bellows			Nominal movement absorption ¹⁾ for 1000 loading cycles		Adjusting force rate		
outside diameter	corrugated length	effective cross-section	angular ²⁾	lateral ²⁾	axial	angular	lateral
Da	lbq	A	2α _N	2λ _N	c ₁	c _α	c _λ
mm	mm	cm ²	degrees	mm	N/mm	Nm/degrees	N/mm
90	50	46.6	22	3.5	321	4.2	1113
91	110	47.2	33	15	199	2.6	144
109	55	70.1	23	4.1	272	5.3	1182
111	132	71.6	33	18	218	4.3	166
123	60	90.8	21	4.1	329	8.3	1555
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151	52	140	18	3	340	13	3302
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320	120	679	16	6.4	649	122	5762
320	200	679	21	18	390	74	1245



Mechanical loads on the vacuum barriers

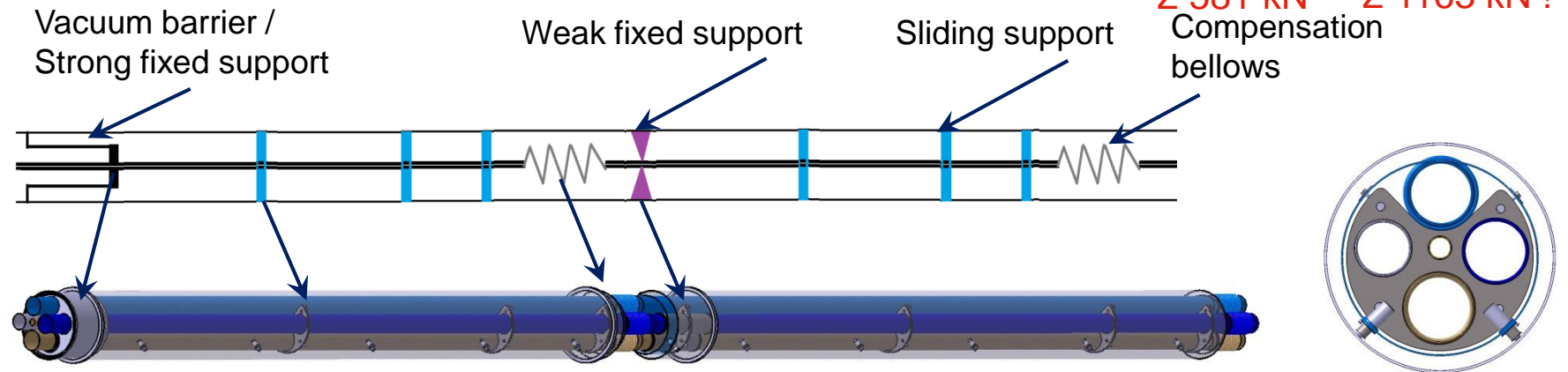
Parameters of bellows

Parameters of process pipes

Determination of forces

	2δ	A	c _δ	c _α	c _λ	Nom. p	Design p	Test p	ΔL	Prestr.	F _p	F _{ΔL}	F _{NP}	F _{P20bar}	F _{P50bar}
	mm	cm ²	N/mm	N/deg	N/mm	bar	bar	bar	mm	mm	kN	kN	kN	kN	kN
DN80	42	92.5	222	5.7	227	3	20	28.6	37.5	20	-2.8	3.9	1.1	-26	-26
DN200	52	443	428	53	1722	1.3	20	28.6	37.5	20	-5.8	7.5	1.7	-127	-127
DN240F	79	679	390	74	1245	20 (50)	20 (50)	28.6 (71.5)	37.5	20	-135.8	6.8	-129.0	-194	-485,5
DN240R	79	679	390	74	1245	15 (45)	20 (50)	28.6 (71.5)	37.5	20	-101.9	6.8	-95.0	-194	-485,5
DN250	79	695	370	78	1350	0.5	4	5.72	37.5	20	-3.5	6.5	3.0	-40	-40

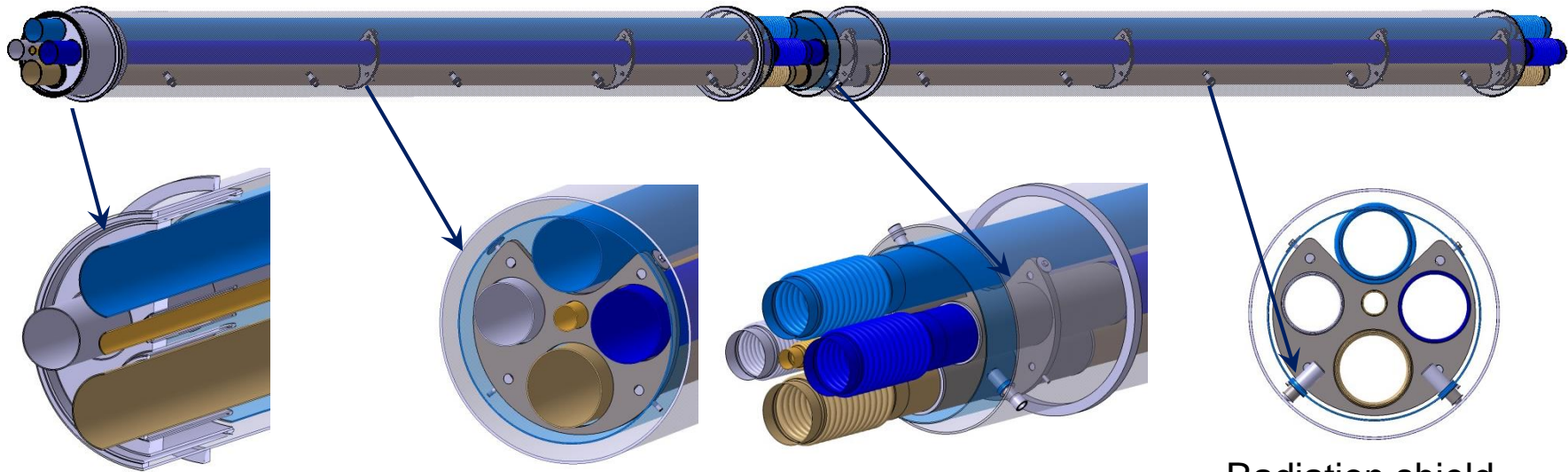
The proposal of design



Cross section



Heat transfer through the internal supports



Vacuum barrier x 2

Sliding support x 12

Weak fixed support x 3

Radiation shield sliding support x 20

	$Q_{VB\ 20bar}$ W	$Q_{VB\ 50bar}$ W	Q_{SS} W	Q_{WFS} W
DN80	0.2	0.3	0.33	0.07
DN200	1.28	1.45	0.03	0.1
DN240 F	1.41	2.13	0.03	0.1
DN240 R	33.71	39.65	-0.63	7.9
DN250	1.21	1.53	0.24	0.5

	Q_{RSSS} W
DN240 R	3.1



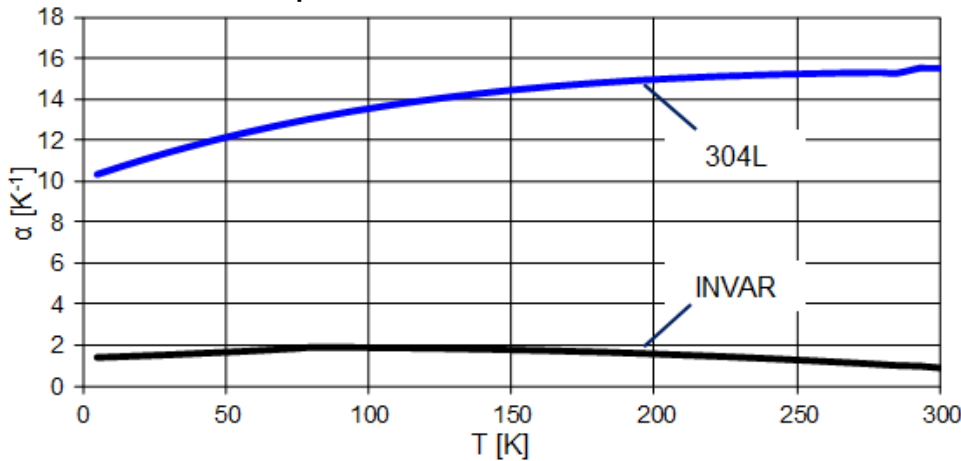
Contents

1. Introduction - FCC cryogenic system
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- 3. Design of the transfer line with invar process pipes**
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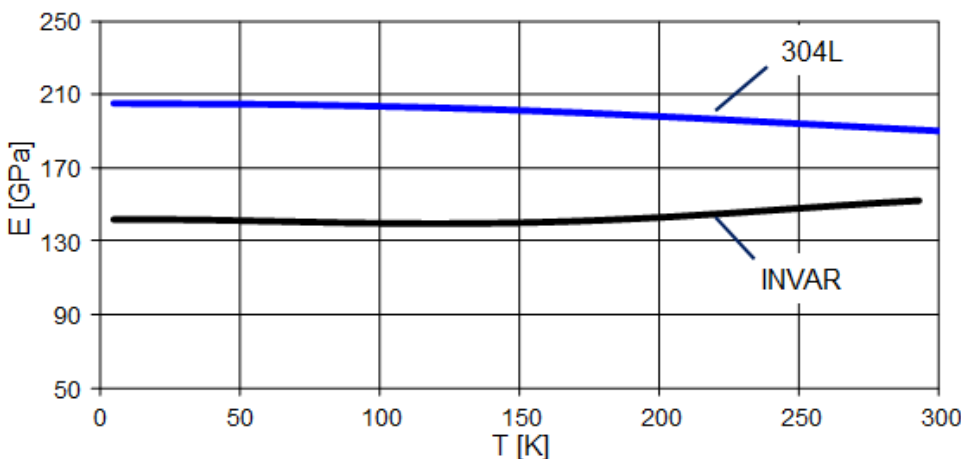


INVAR vs. stainless steel - comparison of material properties

Linear expansion



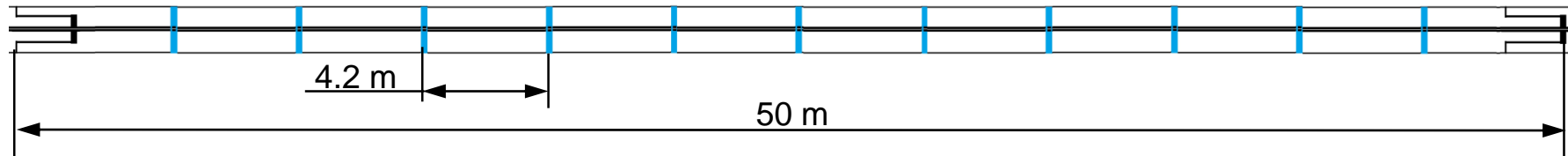
Young's Modulus



Material	Units	Invar 36	304 Stainless Steel
Density	g/cm ³	8.05	8.00
Young's Modulus	GPa	141	193
Poisson's Ratio	-	0.26	0.27
Yield Strength	MPa	248	230-260
Thermal Expansion Coefficient	x10 ⁻⁶ K ⁻¹	1	14.7
Thermal Conductivity	W/m K	10.4	16.2
Specific Heat	J/kg K	515	500
Specific Stiffness	-	17.5	24.1
Thermal Diffusivity	10 ⁻⁶ m ² /s	2.6	4.1
Thermal Distortion (Steady State)	μm/W	0.10	0.91
Thermal Distortion (Transient)	s/m ² K	0.38	3.68



Mechanical loads on the vacuum barriers for INVAR process pipes



Determination of the distance between supports on the basis of process pipes permissible deflection $f = 3 \text{ mm}$

$$x = \min \left[\sqrt[4]{\frac{87 f E (D_i^4 - d_i^4)}{20 g (\rho_{INV} (D_i^2 - d_i^2) + \rho_{He} d_i^2)}} \right]_i \quad i - \text{number of process pipe}$$

Determination of forces on the vacuum barriers based on Hooke's law

$$F = AE \frac{\delta l}{L}$$

A – cross section of process pipe

E - Young's Modulus

δl – thermal shrinkage

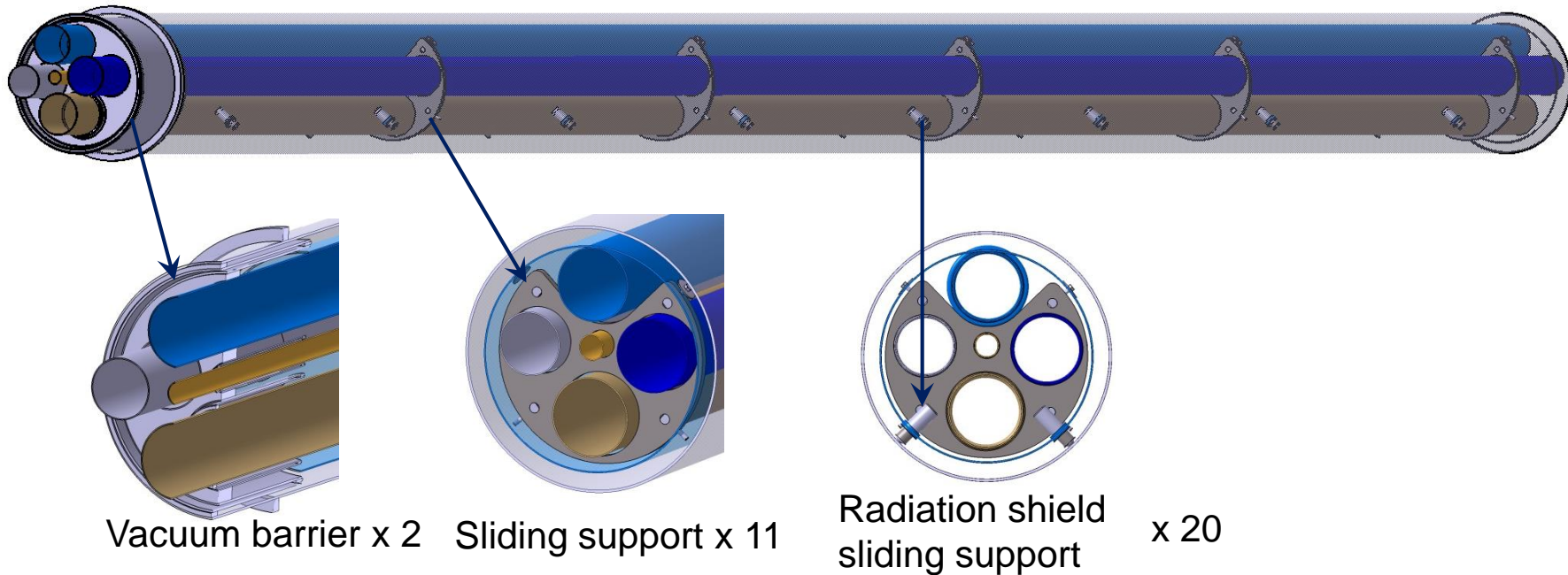
L - distance between the vacuum barriers

	T	δl	L	A	E	F
	K	mm	m	mm ²	GPa	kN
DN80	4.6	20	50	546	141.7	30.94
DN200	40	20	50	1836	141.7	104.0
DN240 F	40	20	50	2384	141.7	135.1
DN240 R	60	20	50	2384	140.8	134.3
DN250	4	20	50	1703	141.7	96.5

The forces acting of the supports slightly dependent on the preassure $\Sigma 501 \text{ kN}$



Heat transfer through the internal supports for INVAR process pipes



	Q_{VB} W	Q_{SS} W
DN80	0.19	0.33
DN200	1.21	0.03
DN240 F	1.33	0.03
DN240 R	27.42	-0.63
DN250	1.17	0.24

	Q_{RSSS} W
DN240 R	3.1

- lighter construction of the vacuum barrier
- sliding supports structure is the same as for stainless steel pipes



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Comparison of the supports system and heat fluxes

	Number of VB	Number of WFS	Number of SS	Number of SSRS	Number of welds	Number of bellows
Stainless steel	2	3	12	20	80	20
INVAR	2	0	11	20	25	0

	T	Q_{STELL} 20bar	Q_{STELL} 50bar	Q_{INV}
	K	W	W	W
DN80	4.6	4.6	4.6	4.0
DN200	40	3.2	3.7	2.8
DN240 F	40	3.5	4.9	3.0
DN240 R	60	146	158	110
DN250	4	6.8	7.4	5.0



Comparison of cumulative failure rates

Probabilities of defect occurrence (failure rates) of the most common process pipes defects

Stainless steel			INVAR	
Defect	Failure rate	Ref	Defect	Failure rate
FR ₁ Cold weld rupture	$2.53 \cdot 10^{-7} \text{ m}^{-1} \cdot \text{year}^{-1}$	[1]	FR ₁ Cold weld rupture	$5.06 \cdot 10^{-7} \text{ m}^{-1} \cdot \text{year}^{-1}$
FR ₂ Cold pipe leakage	$4.61 \cdot 10^{-6} \text{ m}^{-1} \cdot \text{year}^{-1}$	[2]	FR ₂ Cold pipe leakage	$4.61 \cdot 10^{-6} \text{ m}^{-1} \cdot \text{year}^{-1}$
FR ₃ Cold pipe rupture	$4.54 \cdot 10^{-7} \text{ m}^{-1} \cdot \text{year}^{-1}$	[2]	FR ₃ Cold pipe rupture	$4.54 \cdot 10^{-7} \text{ m}^{-1} \cdot \text{year}^{-1}$
FR ₄ Cold bellows rupture	$8.76 \cdot 10^{-5} \text{ year}^{-1}$	[3]		

Calculation of cumulative failure rate CFR

$$\text{CFR}_1 = \text{FR}_1 \cdot L_W \quad L_W - \text{length of welds}$$

$$\text{CFR}_2 = \text{FR}_2 \cdot L_P \quad L_P - \text{length of pipe}$$

$$\text{CFR}_3 = \text{FR}_3 \cdot L_P$$

$$\text{CFR}_4 = \text{FR}_4 \cdot n \quad n - \text{the number of bellows}$$

1. Cadwallader L.C.. Cryogenic System Operating Review for Fusion Application. Idaho National Engineering Laboratory. USA. 1992
2. Failure Frequency Guidance. Process Equipment Leak Frequency Data for Use in QRA.
http://www.dnv.com/services/software/products/phast_safeti/safeti/leak_frequency_guidance.asp
3. Cadwallader L. Vacuum Bellows. Vacuum Piping. Cryogenic Break and Copper Joint Failure Rate Estimates for ITER Design Use. Idaho National Laboratory. USA. 2010



Comparison of cumulative failure rates

Calculation of cumulative failure rate for stainless steel

	Circuit of pipe	Length of pipe	Number of welds	Length of welds	Number of bellows	CFR ₁	CFR ₂	CFR ₃	CFR ₄	CFR
	m	m	-	m	-	1/yer	1/yer	1/yer	1/yer	1/yer
DN80	0.279	50	16	14.0	4.0	2.3E-06	2.3E-04	2.3E-05	3.5E-04	6.1E-04
DN200	0.688	50	16	34.4	4.0	5.6E-06	2.3E-04	2.3E-05	3.5E-04	6.1E-04
DN240F	0.804	50	16	40.2	4.0	6.5E-06	2.3E-04	2.3E-05	3.5E-04	6.1E-04
DN240R	0.804	50	16	40.2	4.0	6.5E-06	2.3E-04	2.3E-05	3.5E-04	6.1E-04
DN250	0.858	50	16	42.9	4.0	6.9E-06	2.3E-04	2.3E-05	3.5E-04	6.1E-04

Calculation of cumulative failure rate for INVVAR process pipes $\Sigma 3.0E-3$ 1/yer

	Circuit of pipe	Length of pipe	Number of welds	Length of welds	CFR ₁	CFR ₂	CFR ₃	CFR
	m	m	-	m	1/yer	1/yer	1/yer	1/yer
DN80	0.279	50	5	1.40	7.1E-07	2.3E-04	2.3E-05	2.5E-04
DN200	0.688	50	5	3.44	1.7E-06	2.3E-04	2.3E-05	2.5E-04
DN240F	0.804	50	5	4.02	2.0E-06	2.3E-04	2.3E-05	2.6E-04
DN240R	0.804	50	5	4.02	2.0E-06	2.3E-04	2.3E-05	2.6E-04
DN250	0.858	50	5	4.29	2.2E-06	2.3E-04	2.3E-05	2.6E-04

$\Sigma 1.3E-3$ 1/yer



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Summary

INVAR

- Less types and numbers of supports
- Lower heat fluxes
- No compensation bellows
- Lower numbers of welds
- Lower forces on the vacuum barriers
- Lower probability of failure

Stainless steel

- Conventional design
- A well-known method of welding
- Pipes are commonly available

Using of invar process pipes seems to be very attractive alternative for FCC



Acknowledgment

The analysis was realized within the CERN-
WrUT agreement FCC-GOV-CC-0036