

Dynamic Modeling of the Pressure Increase in LHe Cryostats in Case of Incidents

C. Heidt, S. Grohmann

Cryogenic Safety HSE seminar, 21st September 2016, CERN

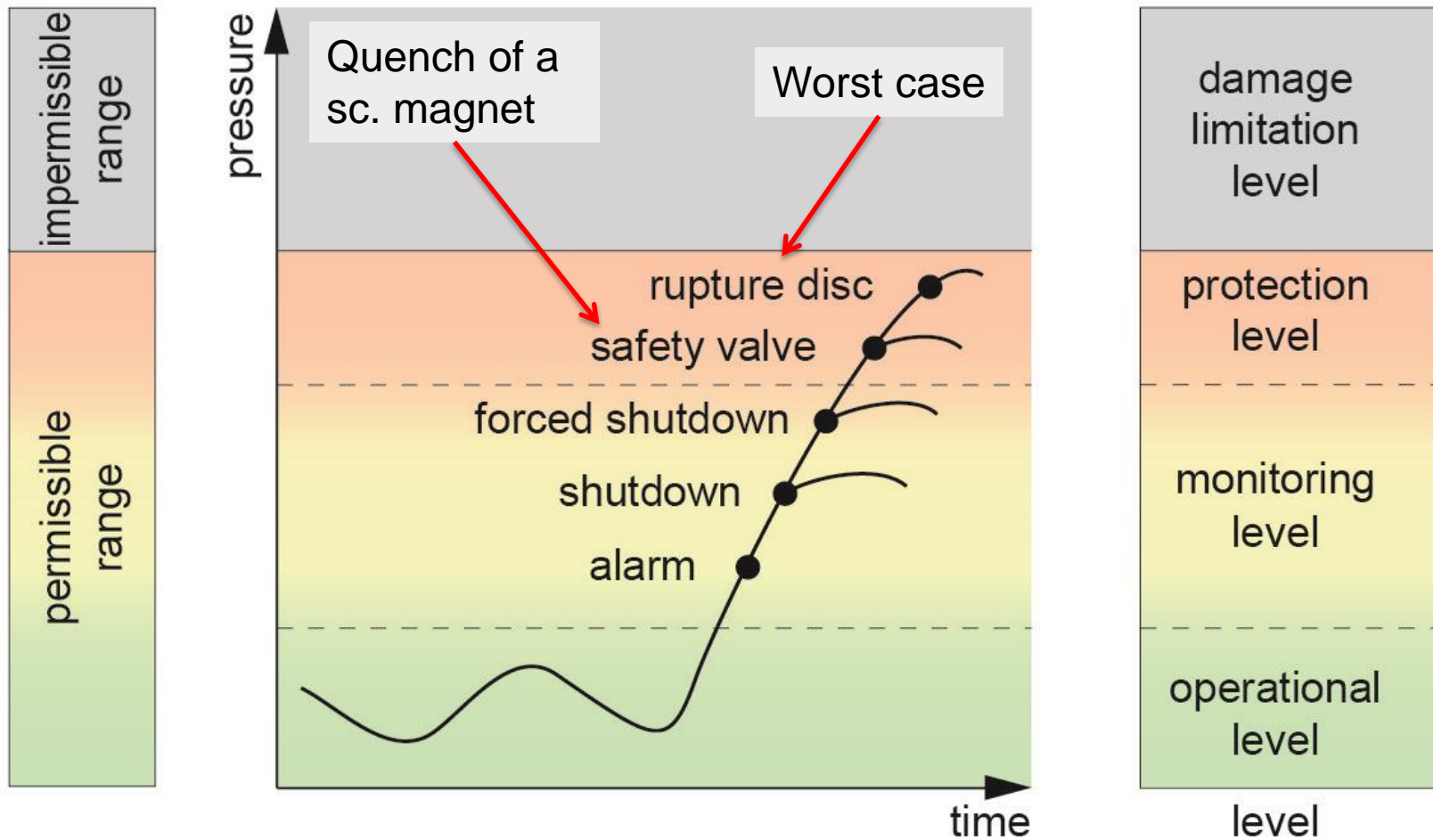
INSTITUTE FOR TECHNICAL PHYSICS
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Outline

- Motivation Dynamic Modeling
- Model Development & Solution
- Heat and Mass Transfer
- Comparison Model - Experiment
- Conclusion & Outlook

Safety Concept



[1] DIN SPEC 4683:2015-04: *Cryostats for liquefied helium – Safety devices for protection against excessive pressure*;

[2] S. Grohmann, M. Süßner: Conceptual Design of Pressure Relief Systems for Cryogenic Application, 2014 *AIP Conf. Proc.* **1573**, 1581-1585

State of the Art

- **Sizing** of safety valves [3]: $A_0 = f(\dot{m}_{\text{Out}})$, $\dot{m}_{\text{Out}} = f(\dot{q})$
- **Worst case** often venting of insulating vacuum with atm. air
- EN13648 [4]: not considering **process dynamics** → $\dot{q}_{\text{max}} = \text{const.}$
 - Lehmann/Zahn [5]: $\dot{q}_{\text{max}} = 3.8 \text{ W/cm}^2$
 - Cavallari et al. [6]: $\dot{q}_{\text{max}} = 4 \text{ W/cm}^2$
- **Possible oversizing** of safety valves
 - Implications on spending, space and helium leakage
 - Unstable operation → reduced relief flow capacity (*pumping, chattering*)
- **Objective:** Dynamic model linking all sub-processes

[3] ISO 4126-7 Safety devices for protection against excessive pressure –Part 7: Common data, German version pr EN ISO 4126-7:2011

[4] EN 13648-3 Cryogenic vessels - Safety devices for protection against excessive pressure - Part 3: Determination of required discharge Capacity and sizing, German version EN 13648-3:2002

[5] Lehmann W and Zahn G, Safety aspects for LHe cryostats and LHe containers, 1978 *Proc. Int. Cryog. Eng. Conf.* **7** 569-579

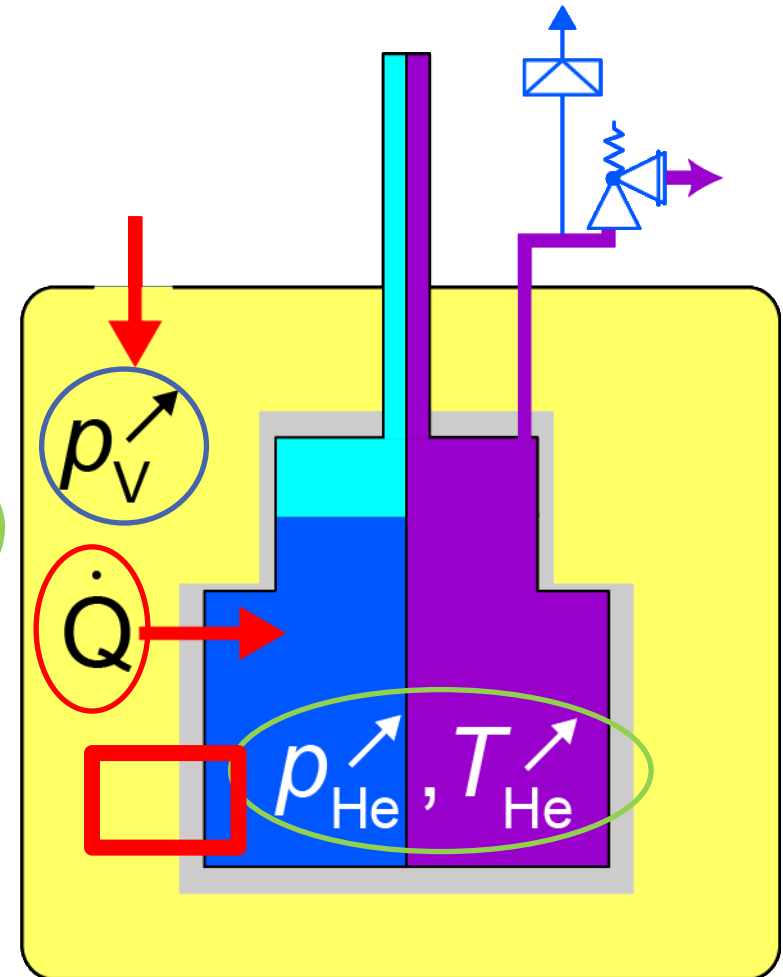
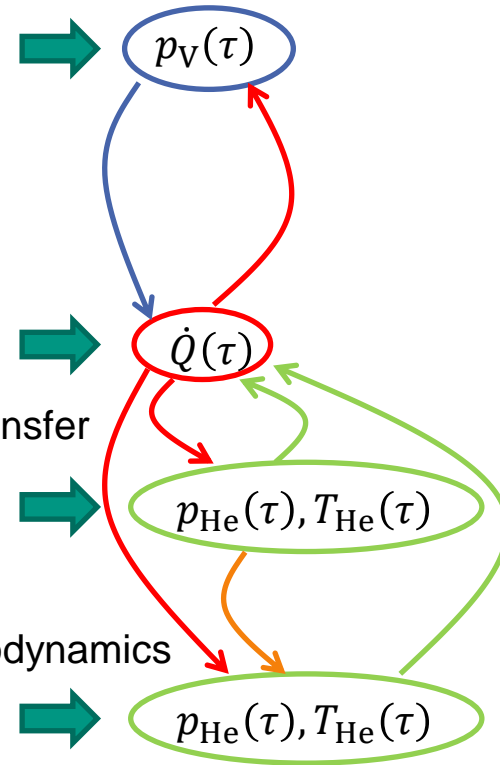
[6] Cavallari G, Gorin I, Güsewell D and Stierlin R, Pressure protection against vacuum failures on the cryostats for LEP SC cavities, 1989 *Proc. 4th Workshop on RF Superconductivity* **1** 781-803

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Model Development & Solution [4]

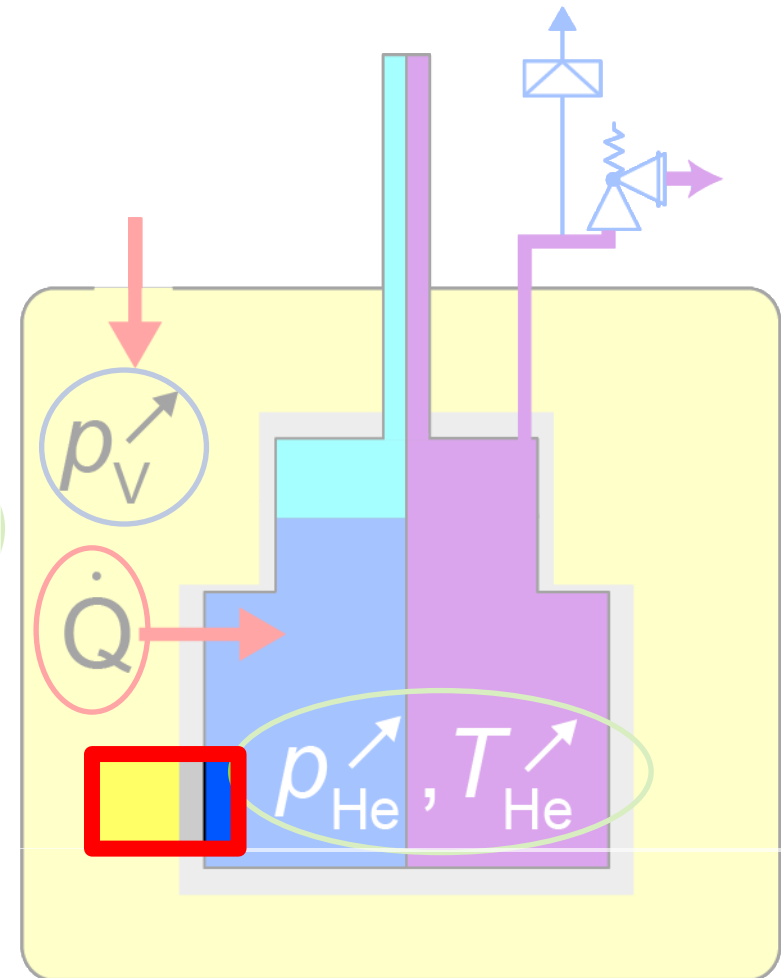
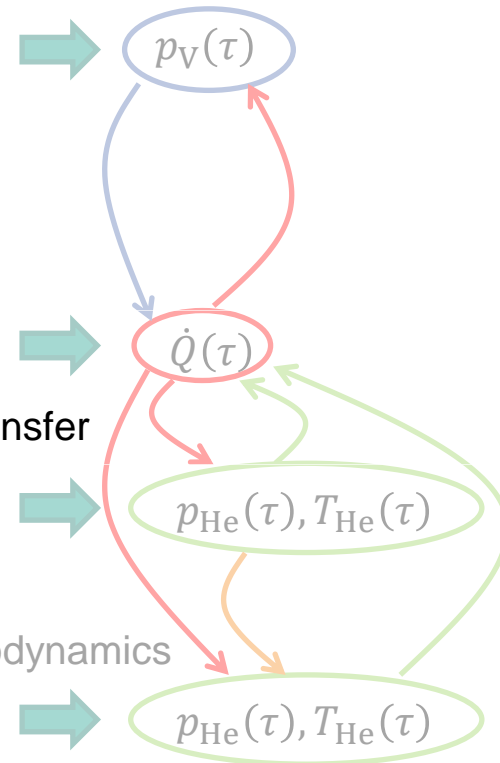
- Vacuum space
 - Ideal gas law
 - Ideal gas flow through orifice
 - Transition
 - Heat and mass transfer
 - Closed inner vessel
 - Isochoric process
 - First law of thermodynamics
 - Open inner vessel
 - Isobaric process
 - First law of thermodynamics
- Simultaneous solution of ODEs



[4] Heidt, C., Grohmann, S., Süßner, M., Modeling the Pressure Increase in Liquid Helium Cryostats after Failure of the Insulating Vacuum, 2014 AIP Conf. Proc. **1573** 1574-1580

Model Development & Solution [4]

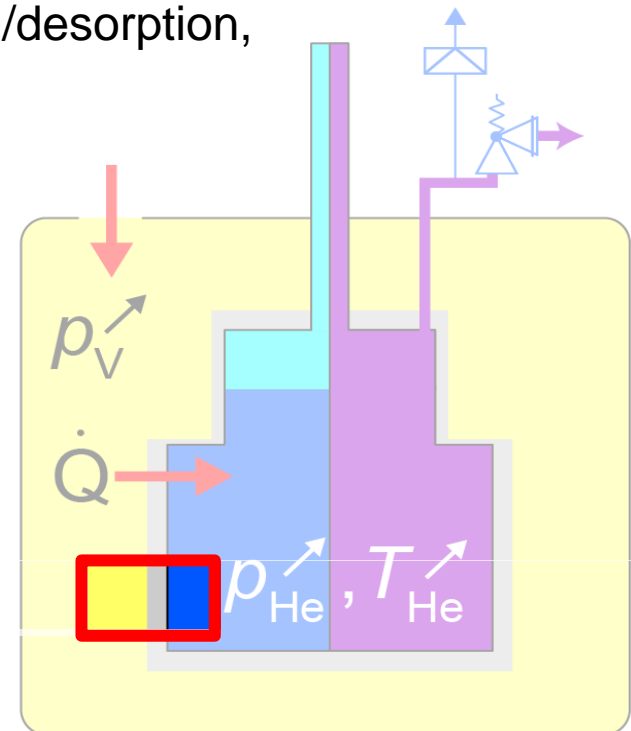
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Heat and Mass Transfer

- Purely analytical approach not feasible
 - Gas flow in vacuum space, pressure gradients $\rightarrow f(x, \tau)$
 - Temperature gradients on inner vessel surface $\rightarrow f(x, \tau)$
 - Simultaneous processes: diffusion, adsorption/desorption, VLS phase changes $\rightarrow f(x, \tau)$
 - Sticking of air on cold surface $\rightarrow f(x, \tau)$
 - Solid air density, heat conductivity, layer thickness, composition $\rightarrow f(x, \tau)$
 - ...
- Fitting necessary



Deposition of Air on Cold Surface [5]

■ Calculation based on kinetic theory: *Knudsen-Langmuir-Hertz*

- $\dot{m}_{\text{dep}} = A_{\text{Cr}} \cdot (I_{\text{C}} - I_{\text{E}})$

- Condensation: $I_{\text{C}} = \alpha_{\text{C}} \cdot \frac{p_{\text{V}}}{\sqrt{T_{\text{V}}}} \cdot \frac{1}{\sqrt{2\pi R_{\text{Air}}}}$

- Evaporation: $I_{\text{E}} = \alpha_{\text{E}} \cdot \frac{p_{\text{Sat}}(T_{\text{W}})}{\sqrt{T_{\text{W}}}} \cdot \frac{1}{\sqrt{2\pi R_{\text{Air}}}}$

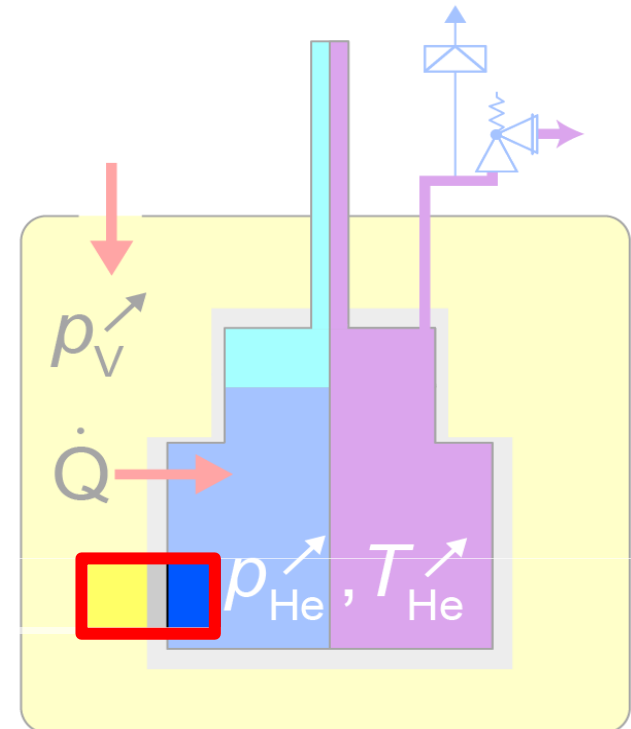
■ Coefficients $\alpha_{\text{C}}, \alpha_{\text{E}}$

- $\alpha_{\text{C}}, \alpha_{\text{E}} = f(T_{\text{V}}, T_{\text{W}}, p_{\text{V}}, p_{\text{Sat}}(T_{\text{W}}), \dots)$

- $\alpha_{\text{C}} = \alpha_{\text{E}}$ at equilibrium

- $0 \leq \alpha_{\text{C}}, \alpha_{\text{E}} \leq 1$

- $T_{\text{W}} \uparrow: \alpha_{\text{C}} \downarrow, \alpha_{\text{E}} \uparrow$

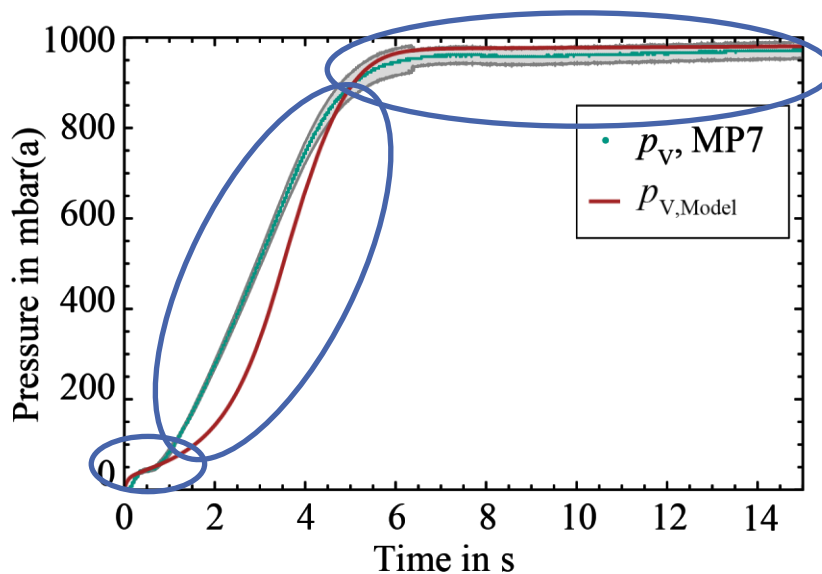


[5] Haefer, R, Cryopumping – Theory and Practice, 1989 *Monographs on Cryogenics 4* Clarendon Press, Oxford

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Comparison Model - Experiment



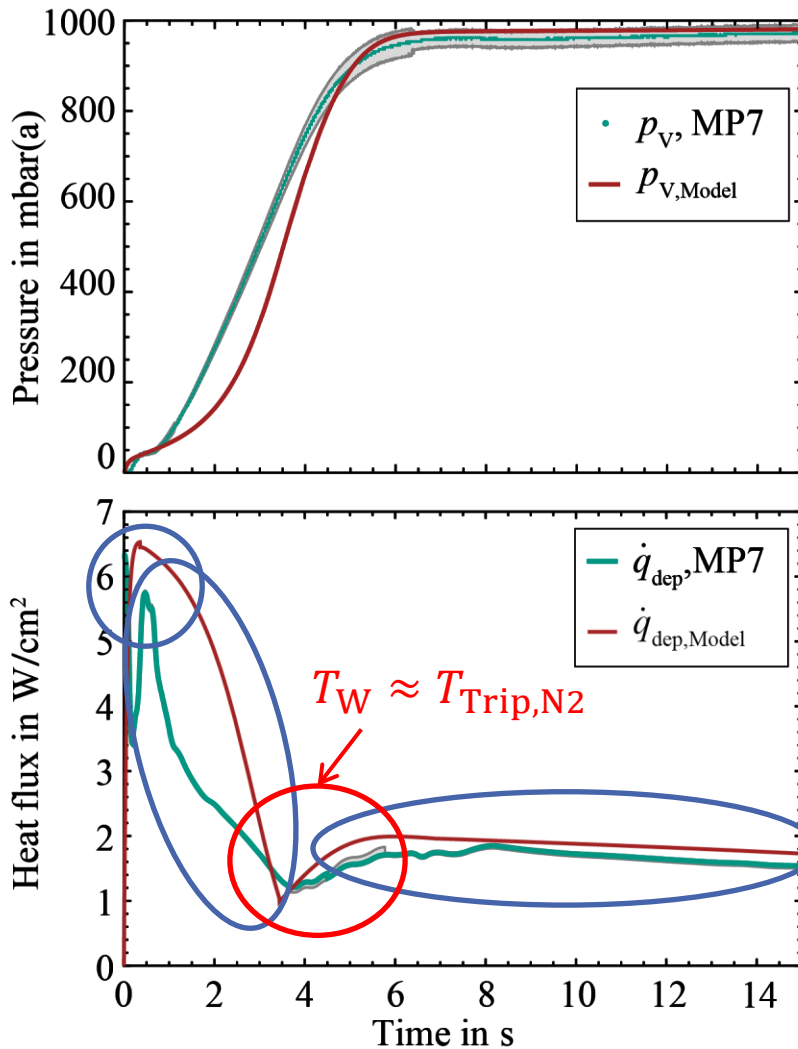
Experimental data MP7

- $d_{\text{Vent}} = 30 \text{ mm}$
- $p_0 = 6 \text{ bar(g)}$

Model

- $\dot{m}_{\text{dep}} \sim \left(\alpha_C \cdot \frac{p_V}{\sqrt{T_V}} - \alpha_E \cdot \frac{p_{\text{Sat}}(T_W)}{\sqrt{T_W}} \right)$
- $\alpha_C, \alpha_E = \begin{cases} \text{const.}, & T_W < 20 \text{ K} \\ \text{linear}, & \alpha_C \downarrow, \alpha_E \uparrow \\ \text{const.}, & T_W > T_{\text{Trip}, \text{N}_2} \end{cases}$
- Fit $T_W = f(T_{\text{Cr}})$

Comparison Model - Experiment



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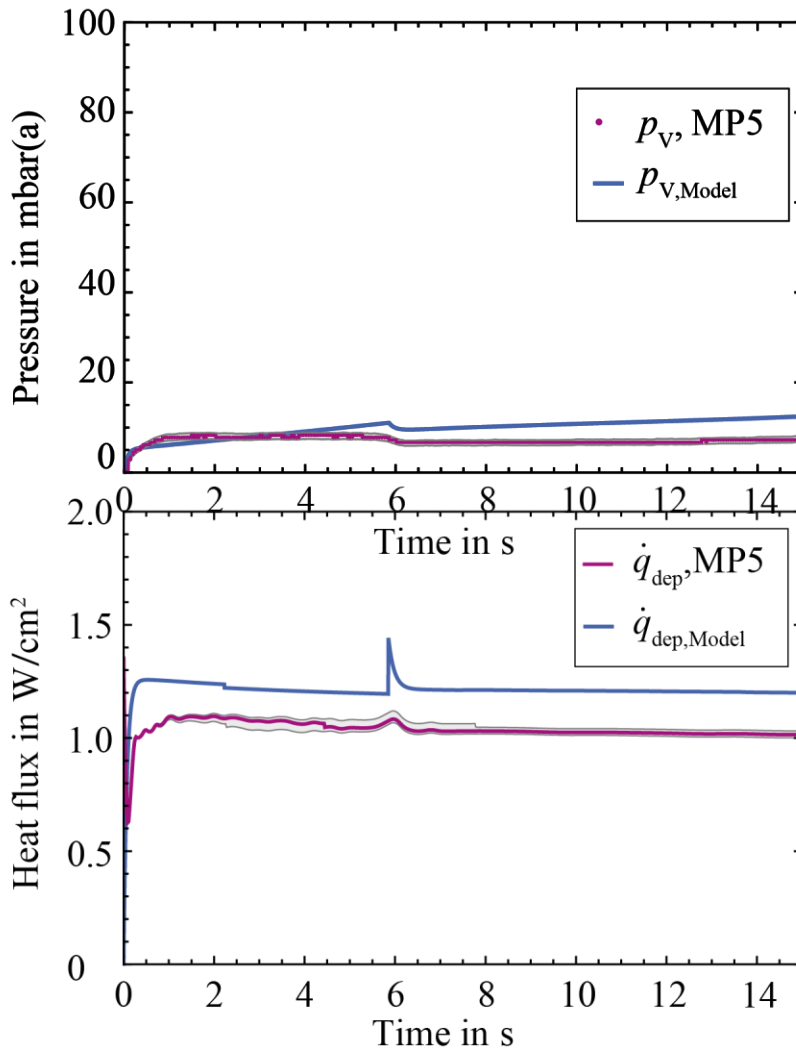
- Fit $T_W = f(T_{\text{Cr}})$

- $T_W \approx T_{\text{Trip}, \text{N}_2}: s \rightarrow l \text{ air}$

- $\alpha_C \downarrow, \alpha_E \uparrow, \frac{p_{\text{Sat}}(T_W)}{\sqrt{T_W}} \uparrow \Rightarrow \dot{m}_{\text{dep}} \downarrow$

- $\frac{p_V}{\sqrt{T_V}} \uparrow \Rightarrow \dot{m}_{\text{dep}} \uparrow$

Comparison Model - Experiment



Experimental data MP5

- $d_{Vent} = 12.5 \text{ mm}$
- $p_0 = 2 \text{ bar(g)}$

Model

- $\dot{m}_{dep} \sim \left(\alpha_C \cdot \frac{p_V}{\sqrt{T_V}} - \alpha_E \cdot \frac{p_{Sat}(T_W)}{\sqrt{T_W}} \right)$
- $\alpha_C, \alpha_E = \begin{cases} \text{const.}, & T_W < 20 \text{ K} \\ \text{linear}, & \alpha_C \downarrow, \alpha_E \uparrow \\ \text{const.}, & T_W > T_{Trip, N2} \end{cases}$
- Fit $T_W = f(T_{Cr})$

➤ Good correspondence

Conclusions & Outlook

- Inclusion of simple fit $T_W = f(T_{Cr}), \alpha_C(T_W), \alpha_E(T_W)$ from kinetic theory
- Good agreement between measured data and experiment for very different conditions
- Shape of heat flux and vacuum pressure increase can be explained

- Improvement of modeling desublimation through experiments
- More experiments for wall temperature fit

Thank you for your attention!



The PICARD Test Facility - KIT/CERN Collaboration on Cryogenic Pressure Relief Experiments

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