

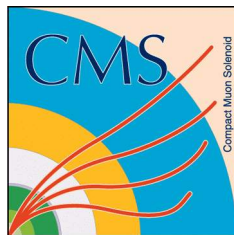
Thermal Measurements and FEA of the 2S Module for the CMS Phase-2 Tracker Upgrade

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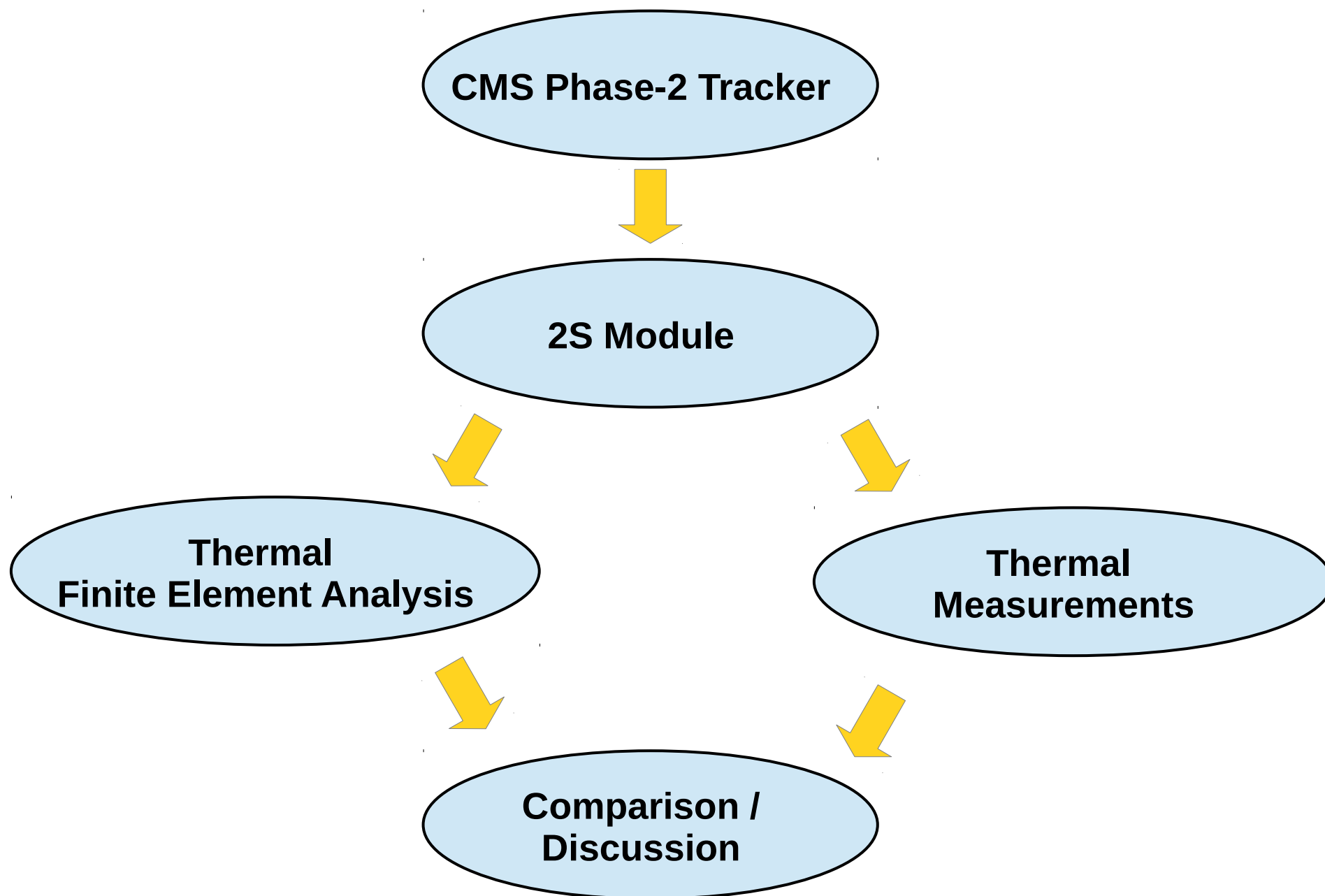
Forum on Tracking Detector Mechanics 2017, Marseille

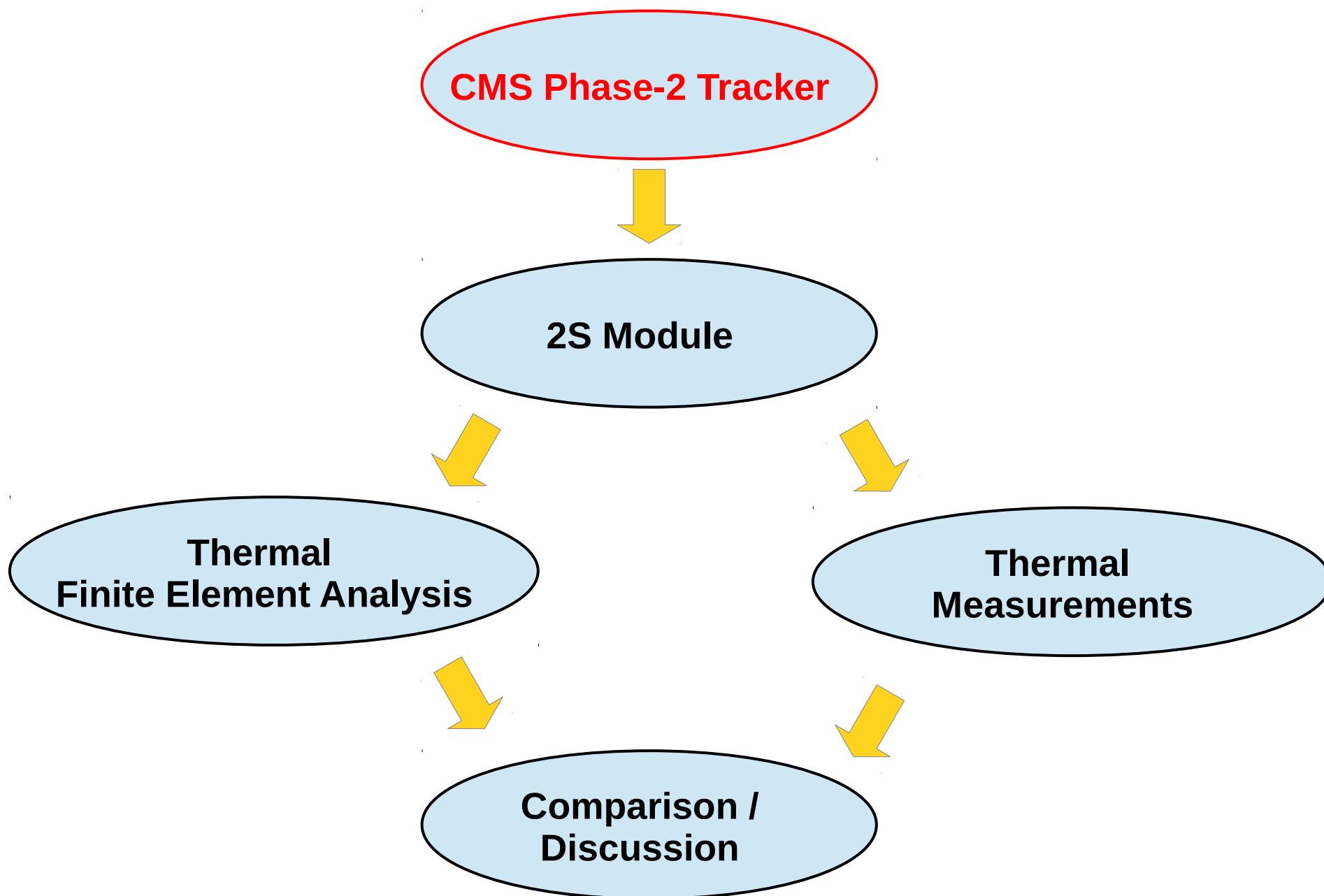


GEFÖRDERT VOM

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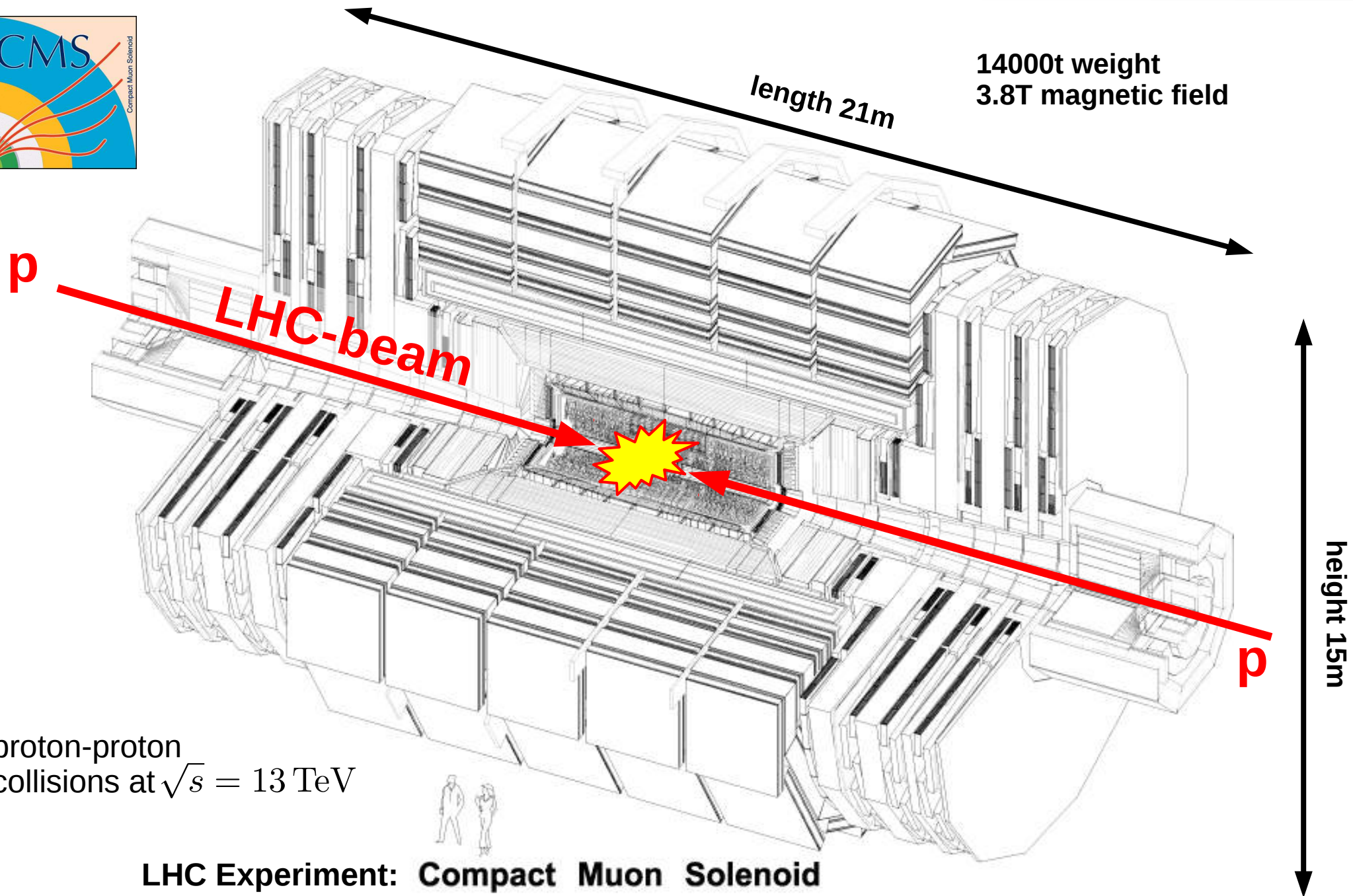
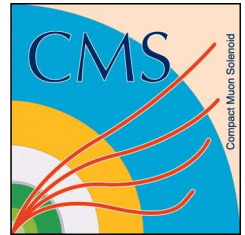


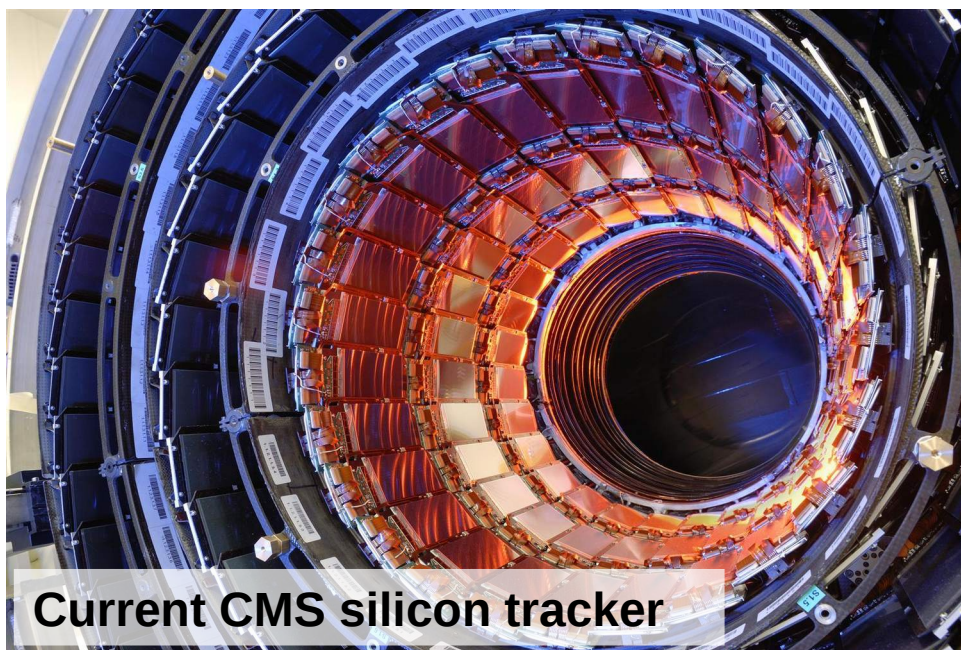




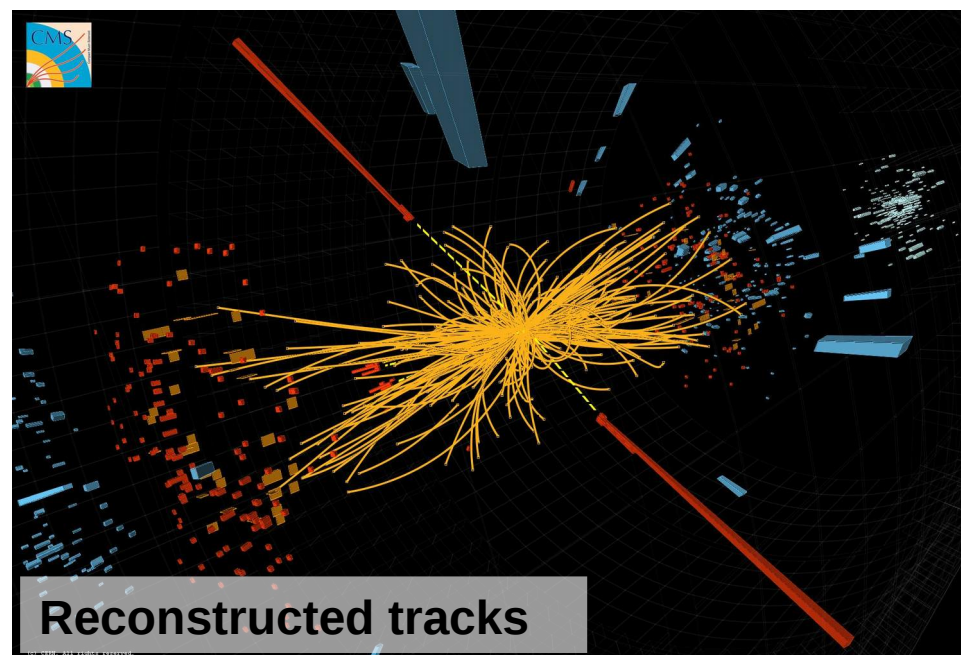


Compact Muon Solenoid



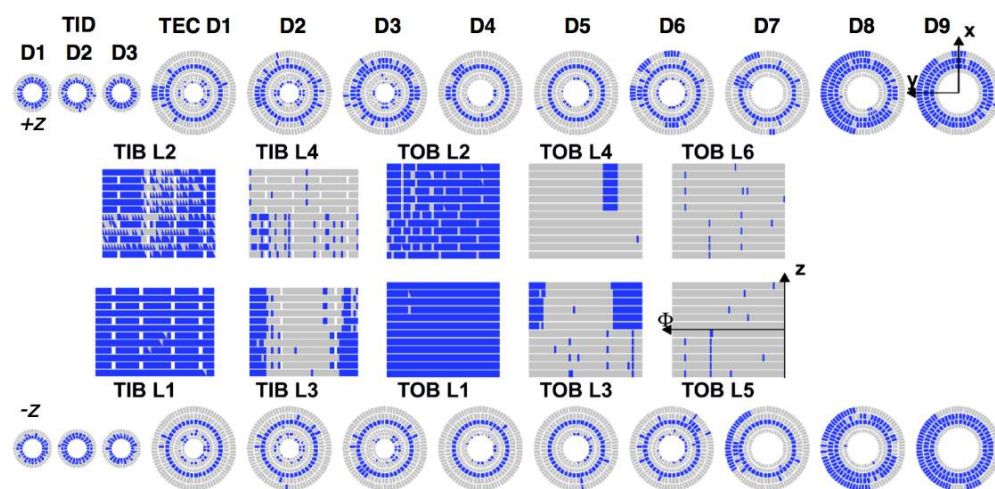


Current CMS silicon tracker

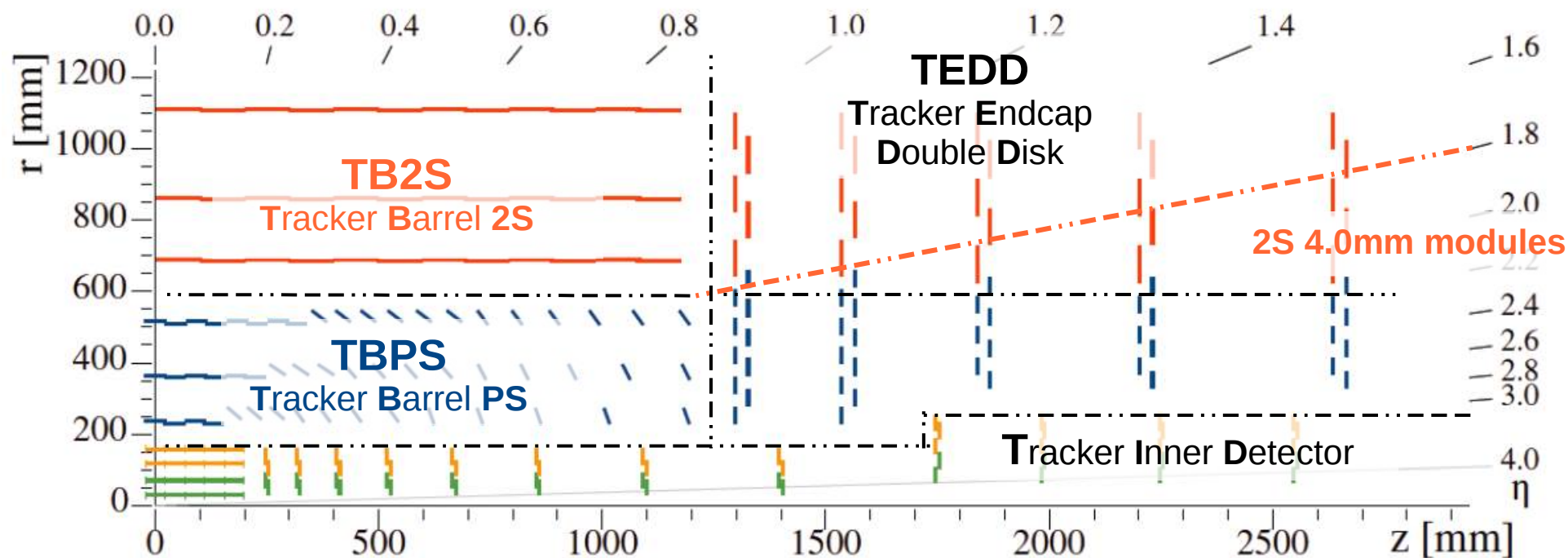


Reconstructed tracks

- upgrade from LHC to HL-LHC
 - up to 5 times higher luminosity
 - up to 10 times more integrated luminosity
- **insufficient cooling** of irradiated silicon sensors at HL-LHC
- full replacement of the current tracker detector with an **upgrade in 2024** planned



non-functional modules
functional modules



Inner Tracker

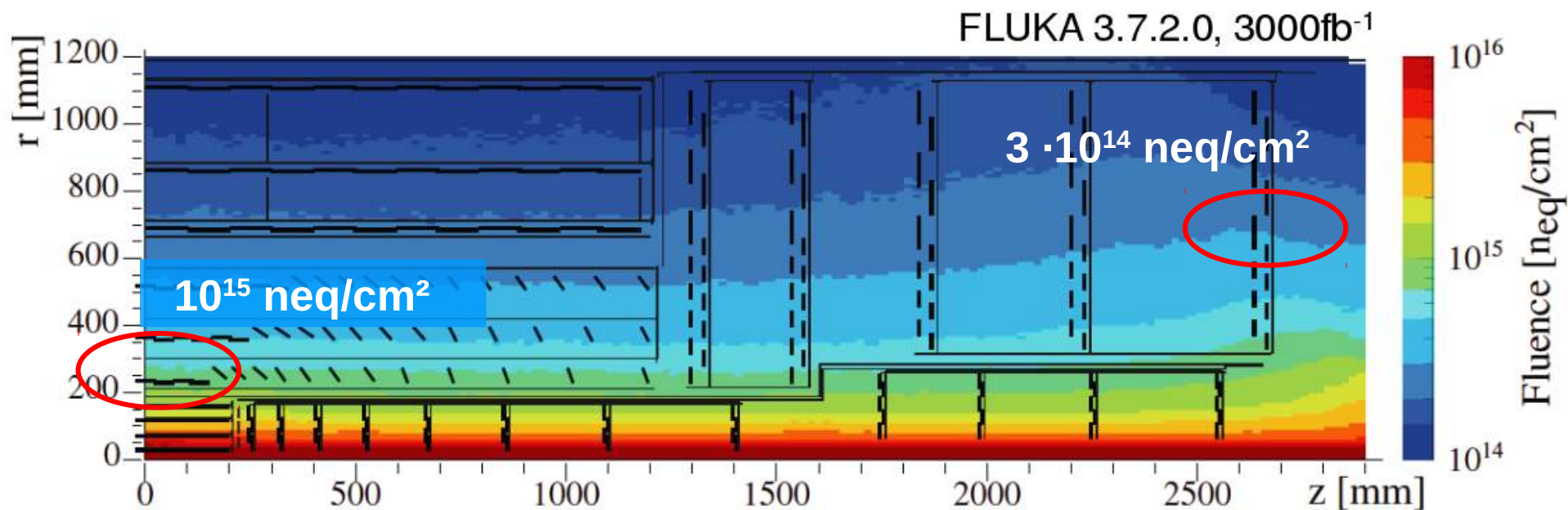
two types of hybrid pixel modules (1x2 and 2x2 chips), 4.9m², 2G channels

Macro-Pixel Sensor Modules:

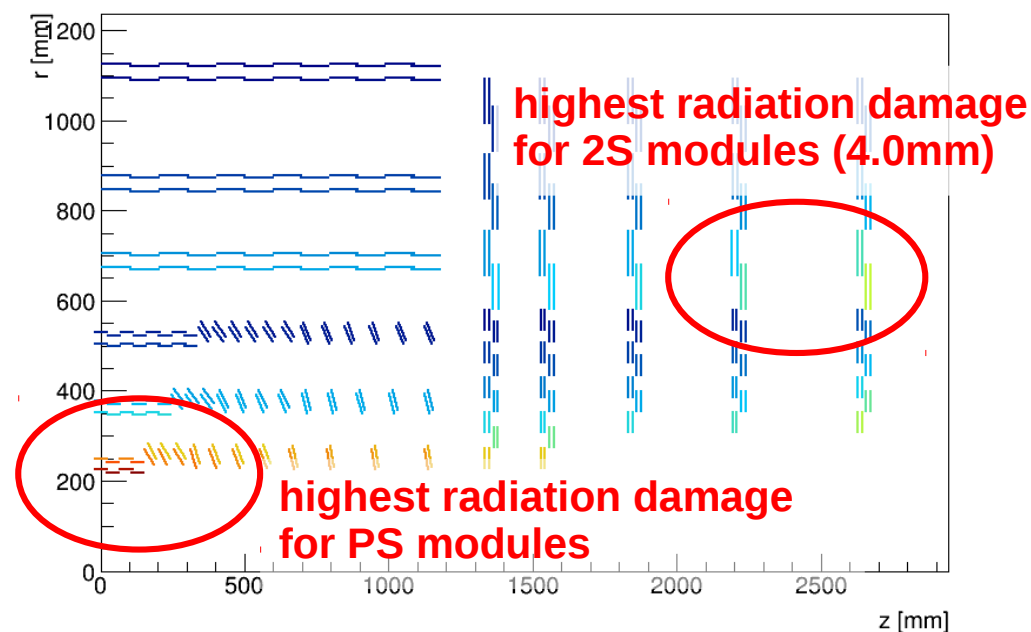
5500 modules, 11M strip channels, 170M macro-pixels, different sensor spacings, upto 10^{15} neq/cm²

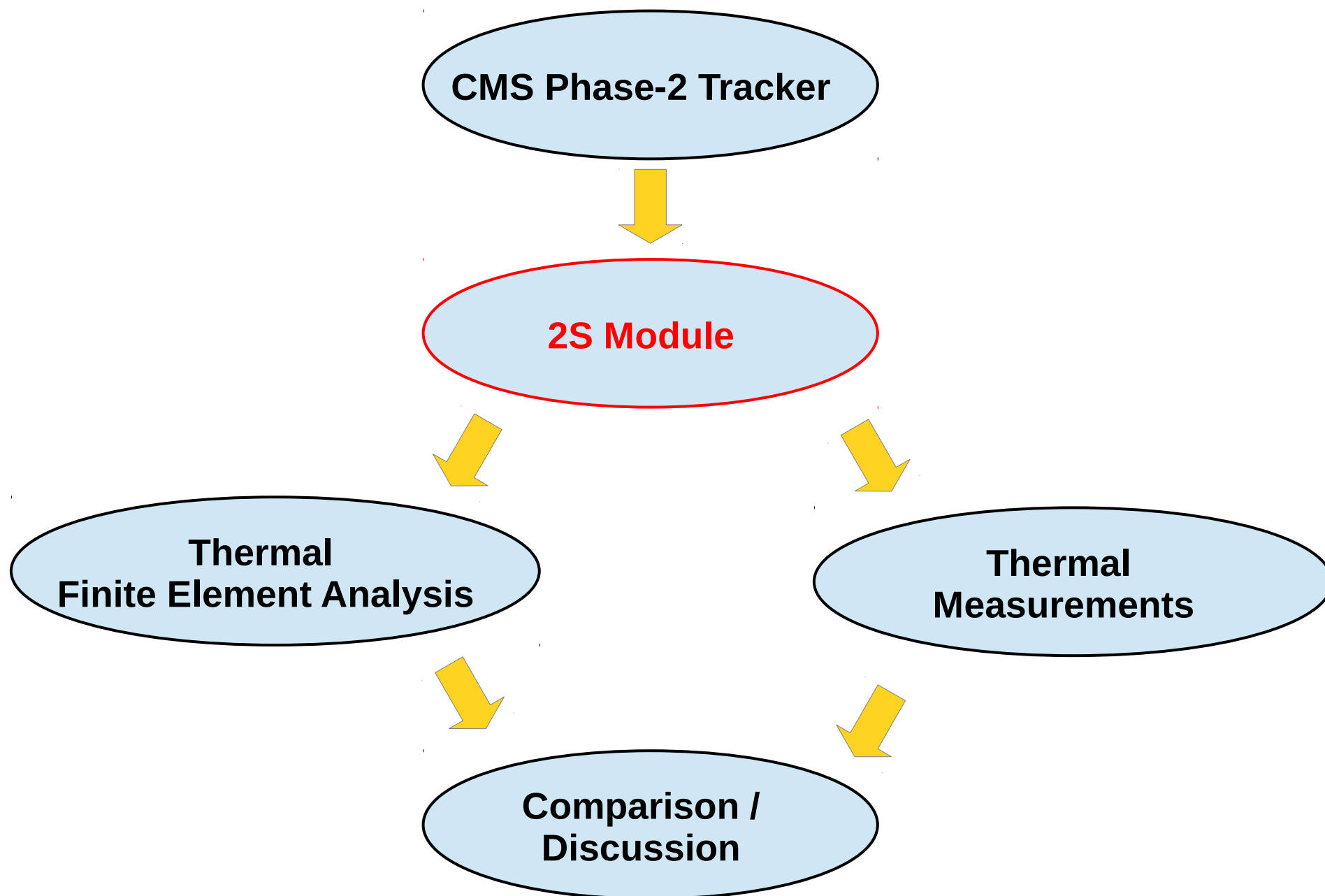
2 Strip Modules:

7500 modules, 30M strip channels, 1.8mm and 4.0mm sensor spacing, up to $3 \cdot 10^{14}$ neq/cm²



- silicon sensors will suffer from damage due to irradiation at HL-LHC conditions
 - instantaneous luminosities of $1 \cdot 10^{35} \text{ cm}^{-2}\text{s}^{-1}$
- leakage current (power) increases
 - with radiation damage
 - with temperature
- **goal:** keep the silicon sensor temperatures at lowest possible temperatures
- **challenge:** Outer Tracker will have an overall power of 100kW





2 Front-end flex-hybrids with read-out chips

- 2032 channel
- chips receive signals from both bottom and top sensor

Service hybrid with DC-DC-converters and opto-module

- 2 stage DC-DC-converter (from 11V to 2.5V and from 2.5V to 1.2V)
→ power efficiency about 50%

Aluminum carbon-fiber stiffeners

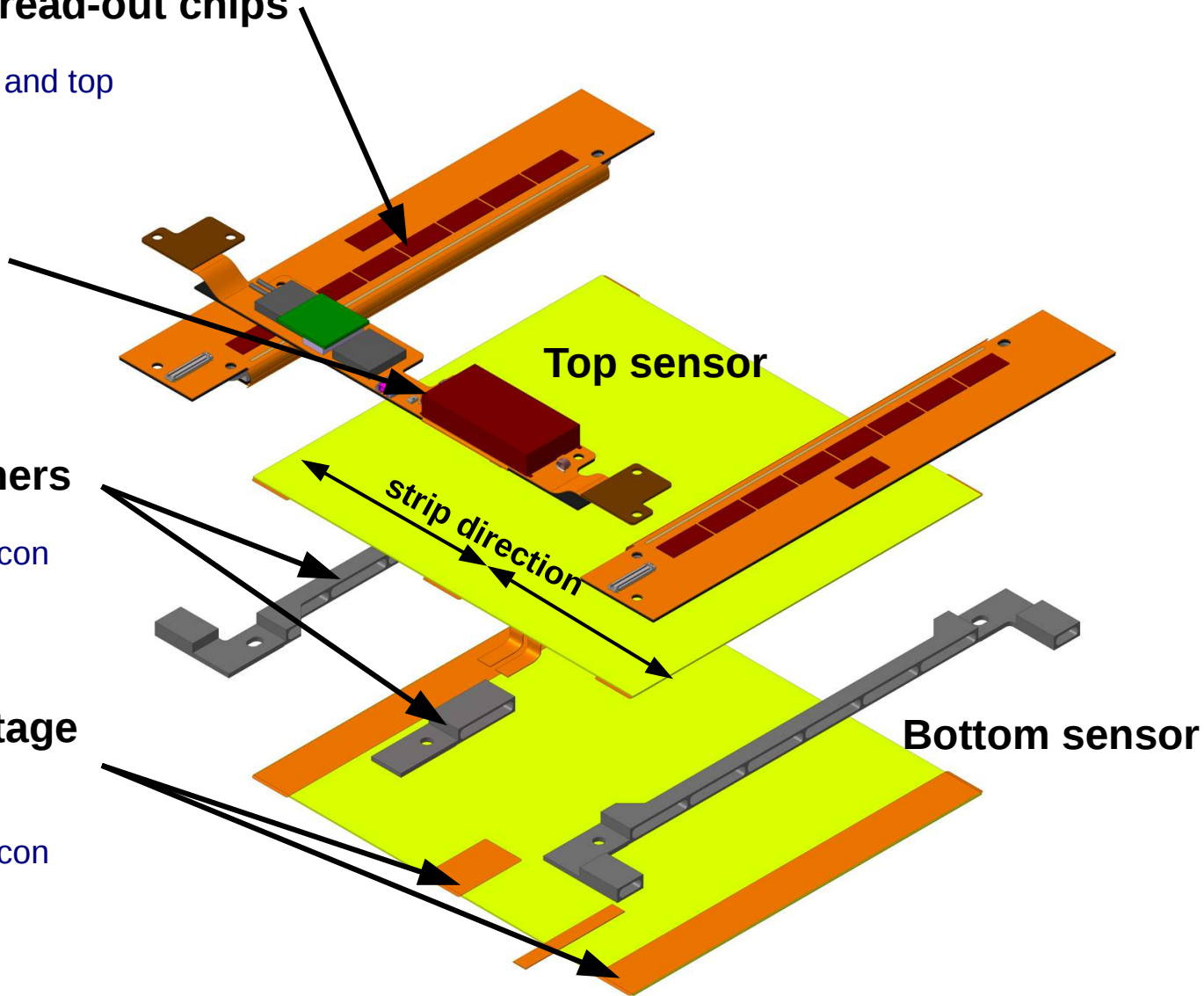
- carbon-fiber re-enforced aluminum for the CTE ($\approx 4\text{ppm/K}$) match between silicon sensors and spacers
- screwing holes for fixture and cooling

Kapton strips for the high-voltage isolation of the sensor

- carbon-fiber re-enforced aluminum for the CTE ($\approx 4\text{ppm/K}$) match between silicon sensors and spacers

2 n-in-p silicon strip sensors

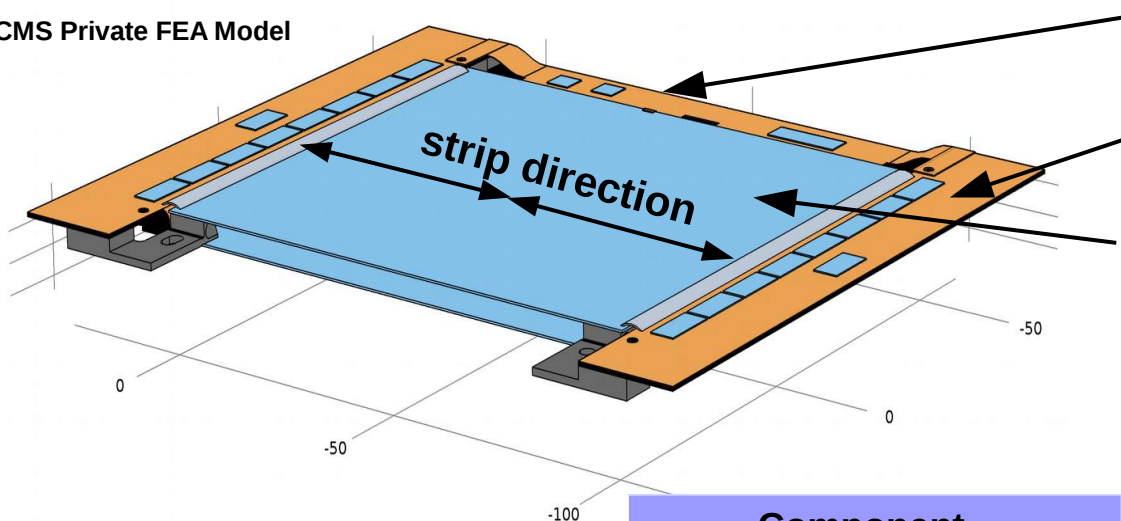
- 10cm x 10cm, 2 x 5cm strip length,
- 90 μm pitch
- 320 μm thickness, 200 μm active thickness



mass ca. 40g

size roughly 14cm x 12cm

CMS Private FEA Model



service hybrid: **2.6 W**

2 front-end hybrids: **2.8 W**

2 silicon strip sensors
after 3000fb⁻¹: **1.2 W**

Component	Part	Power [W]
Service hybrid	DC-DC-converters	1.8
	opto-components	0.8
Front-end hybrids	2 x 8 CBC chips	2.2
	2 CIC chips	0.6
Sensors at -20°C	top	0.6
	bottom	0.6
	Sum	6.6

6.6W per module!

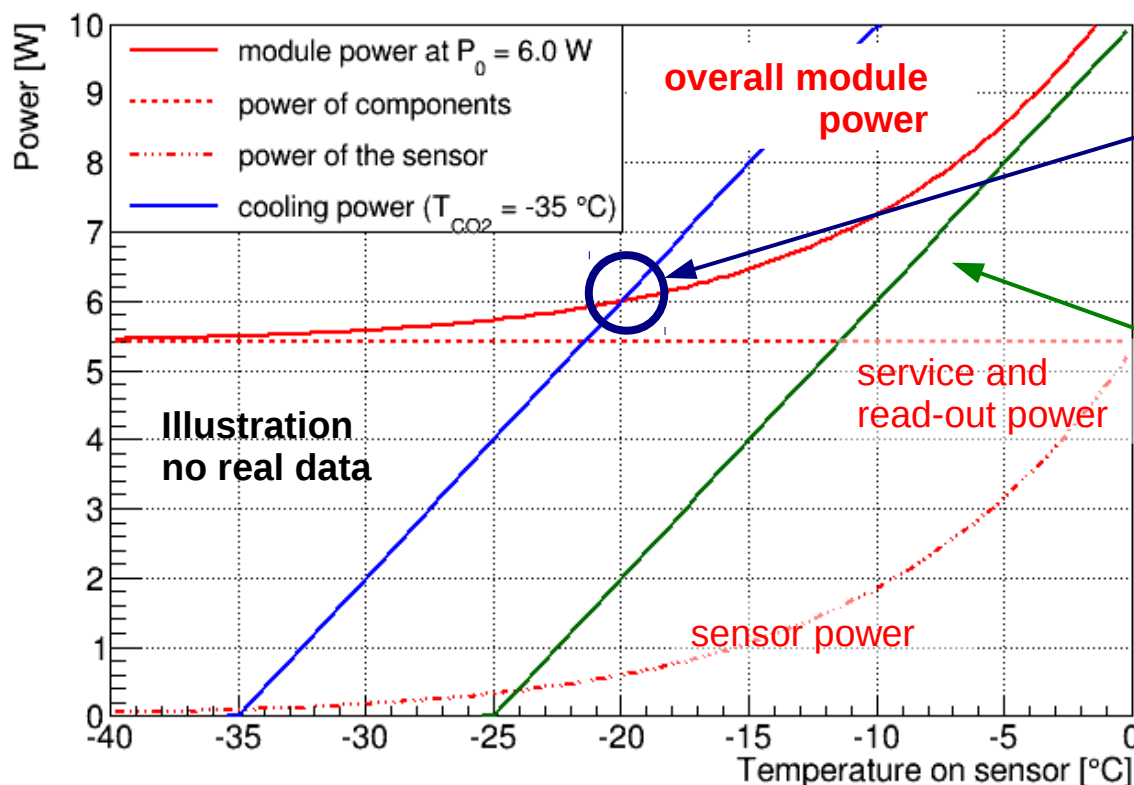
- Some components (read-out chips, DC-DC converters, ...) produce a constant power

$$P_{\text{components}} = \text{const.}$$

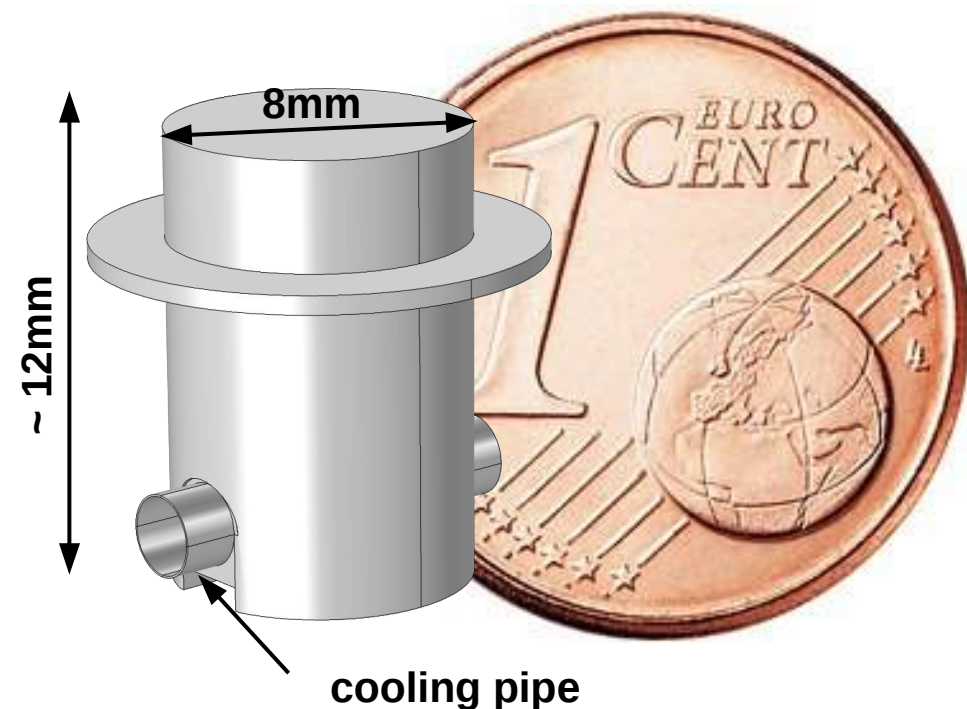
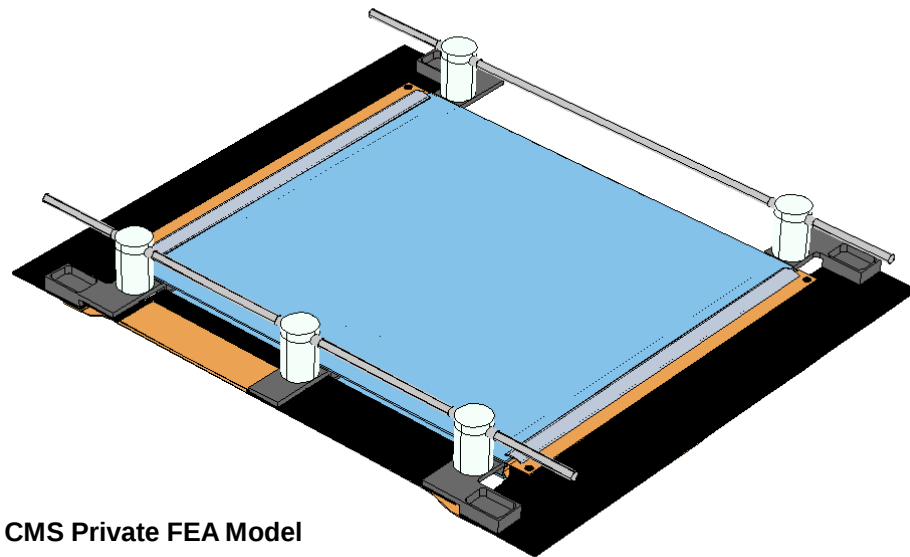
- Irradiated silicon sensors generate temperature dependent leakage current

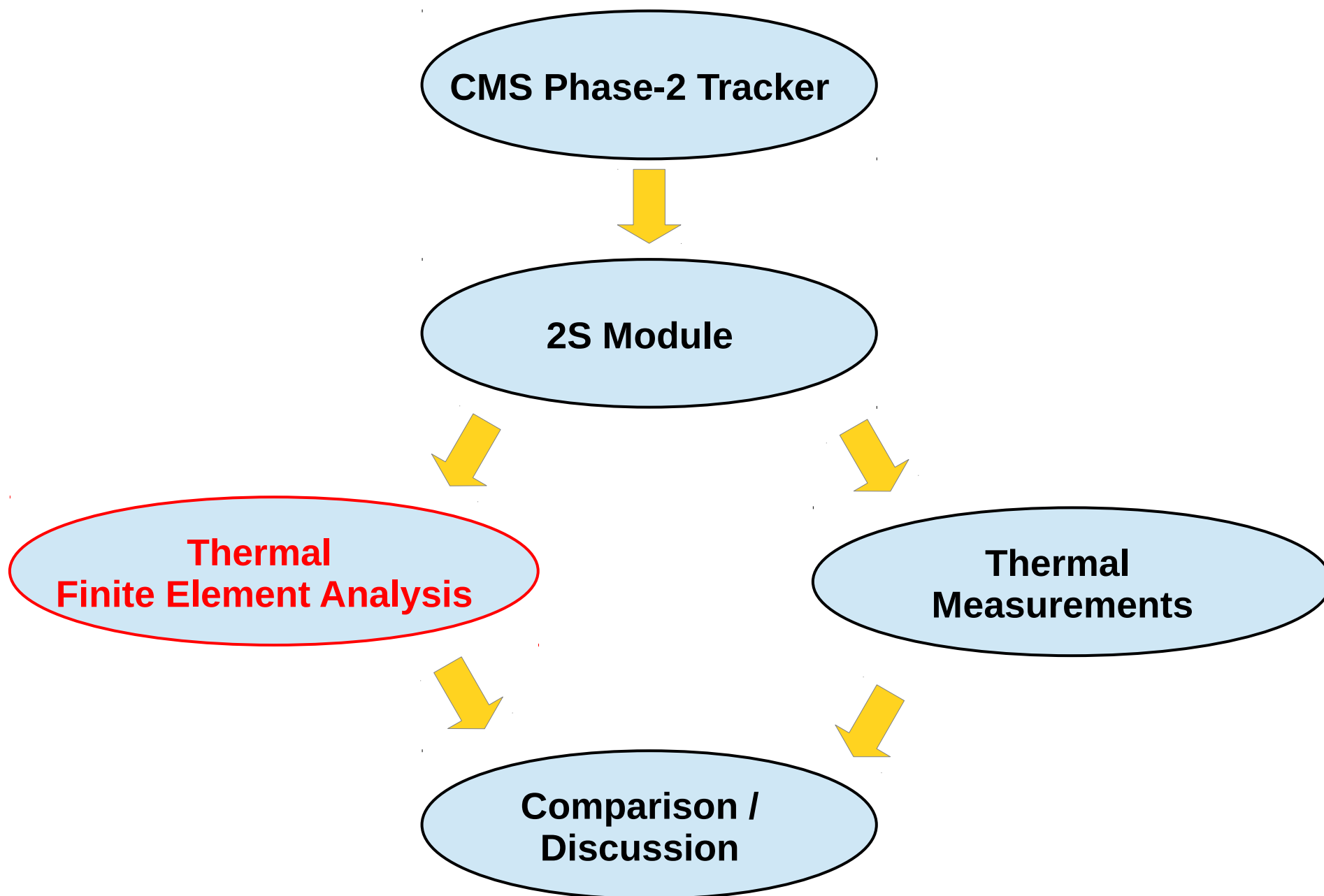
$$P_{\text{sensor}} \propto P_0 \frac{T^2}{T_0^2} \exp \left[- \frac{\Delta E}{2 k_B} \left(\frac{1}{T} - \frac{1}{T_0} \right) \right]$$

- Linear dependence of the cooling power $P_{\text{cooling}} \approx \frac{1}{\alpha} (T_{\text{module}} - T_{\text{coolant}})$



- cooling fluid CO₂ enters the detector in the 2-phase state (2PACL system)
 - 12 bar at a temperature of -35°C
 - CO₂ is evaporated at constant temperature → heat removal
- many advantages with respect to low-material tracking detectors
 - cooling pipes: stainless steel, inner diameter 2.0mm, 100µm wall thickness
 - high heat transfer coefficients in boiling CO₂ (5000 W/m²/K or higher)
- 2S modules in TEDD mounted on five cooling contacts

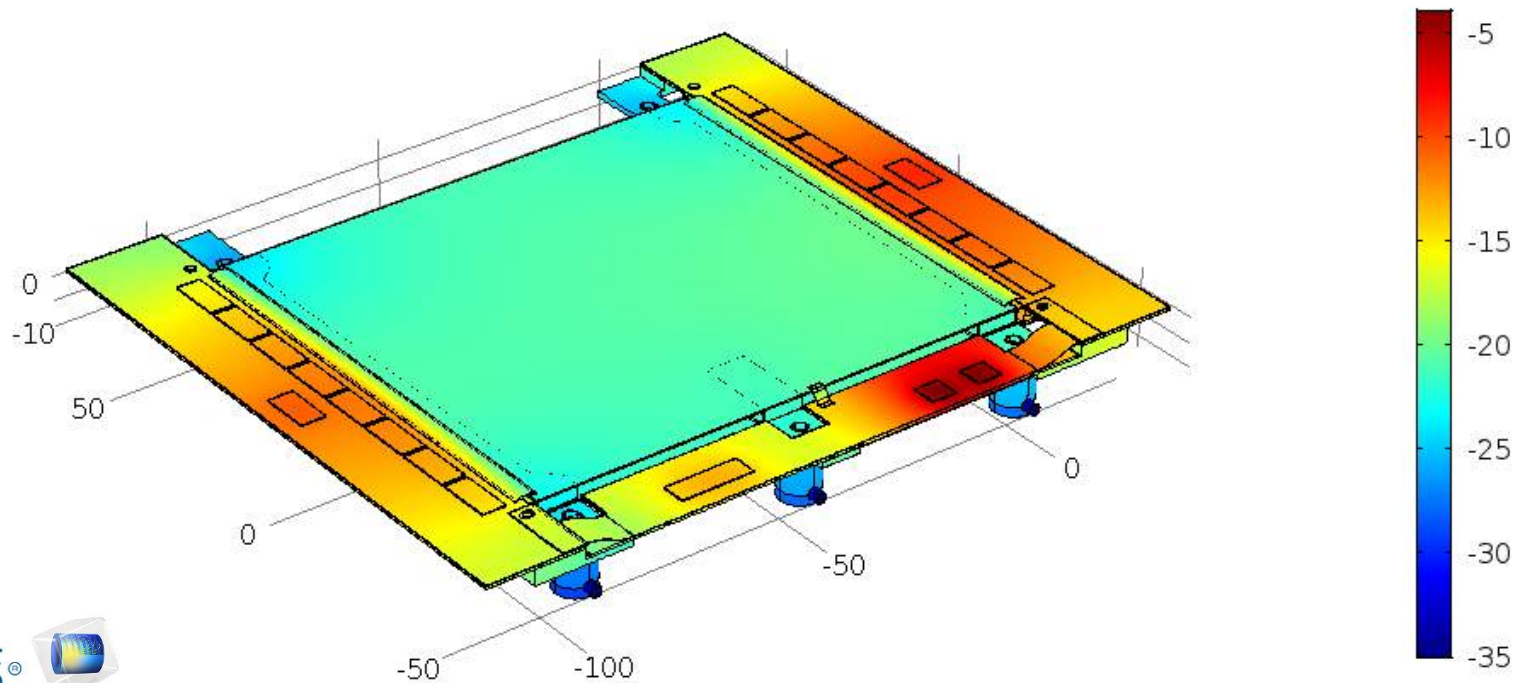


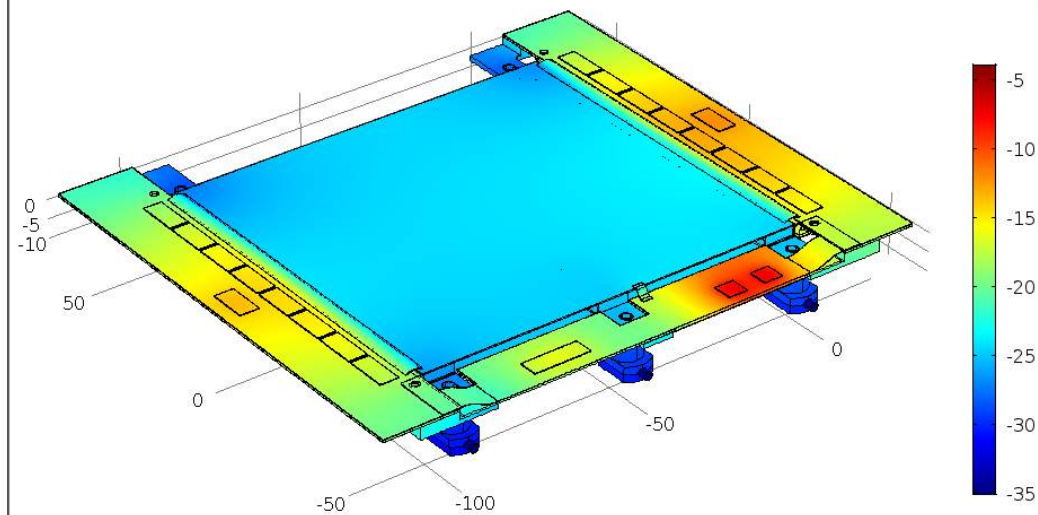
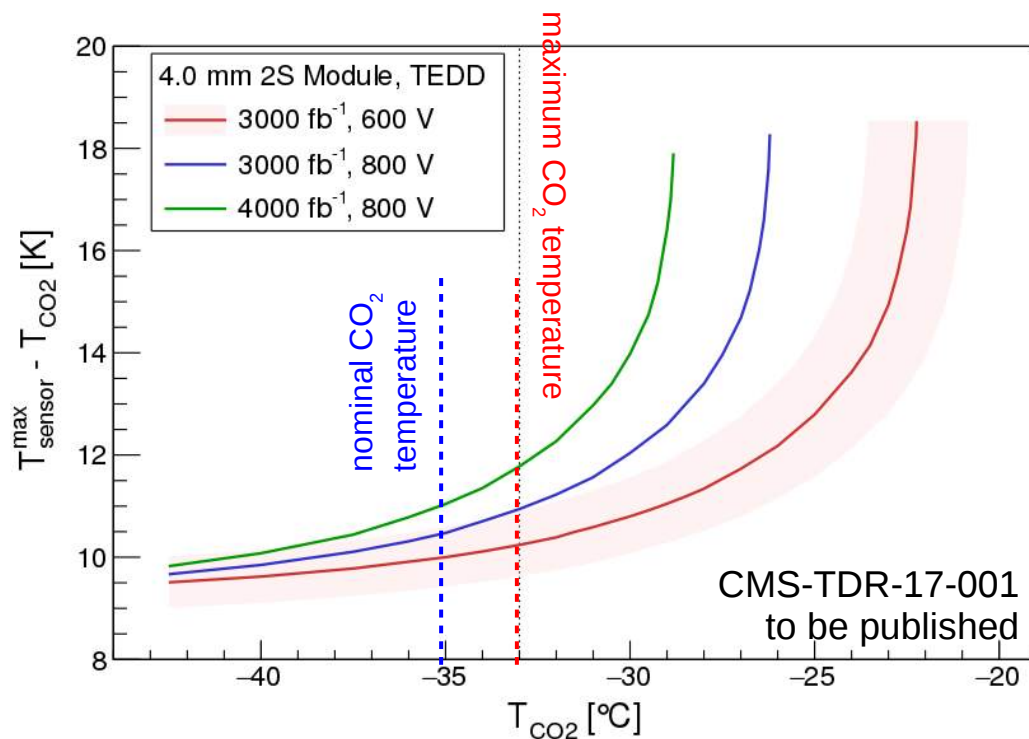


- Thermal FE analyses are made with COMSOL Multiphysics → Heat Transfer Module
- CO₂ temperature of -35°C is assumed in the cooling pipes
- Heat load is generated in the specific volumes
 - constant heat load for ASICs: 5.4W overall
 - temperature dependent leakage current of irradiated silicon sensors (Boltzmann distribution)

- $P_0 = 0.6W$ at $-20^\circ C$ after 3000 fb^{-1}

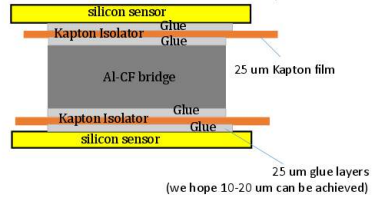
- maximum U_{bias} assumed is $-800V$

$$U_{\text{bias}} \cdot I(T) = U_{\text{bias}} I_0 \left(\frac{T}{T_0} \right)^2 \exp \left(- \frac{\Delta E}{2k_B} \left(\frac{1}{T} - \frac{1}{T_0} \right) \right)$$




- Maximum sensor temperature is computed as a function of the CO₂ temperature
- Difference between maximum sensor temperature and CO₂ temperature is calculated
- Sensor temperature stays in safe regimes
 - for the nominal CO₂ temperature -35°C → sensor temperature -25°C
 - for the maximum CO₂ temperature -33°C → sensor temperature -23°C
- Good margin between thermal runaway and the maximum CO₂ temperature
 - 4K for the scenario at 4000fb⁻¹ and a bias voltage of 800V

Between the sensors: Al-CF + 6 layers.

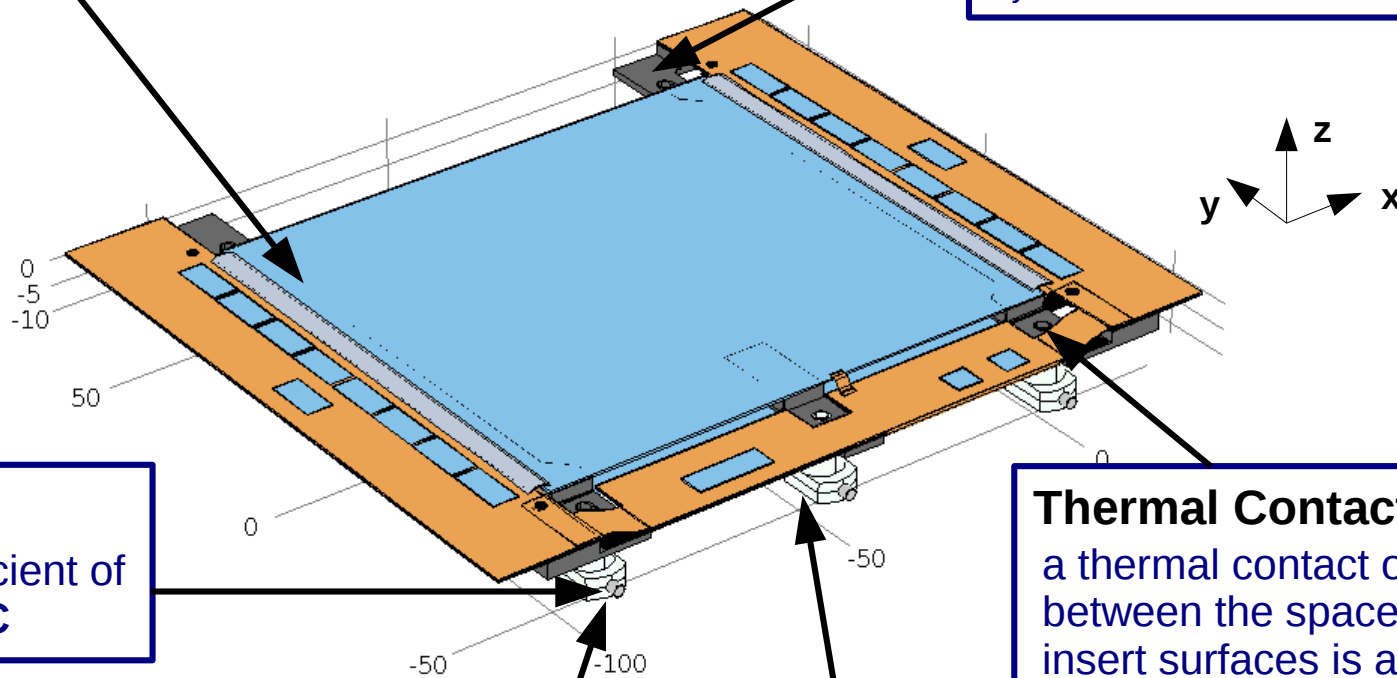


Epoxy-Kapton-Epoxy Layer

described with effective thermal through-plane conductivity 0.3 W/m/K and with **$65\mu\text{m}$** thickness

AlCF-Spacer

anisotropic thermal conductivity due to carbon fiber orientation:
 $k_{yz} = 230 \text{ W/m/K}$; $k_x = 120 \text{ W/m/K}$



CO₂ Heat Flux

heat transfer coefficient of **$5 \text{ kW/m}^2/\text{K}$** at **$-35^\circ\text{C}$**

Thermal Contact

a thermal contact of **$10 \text{ kW/m}^2/\text{K}$** between the spacers and in the insert surfaces is assumed

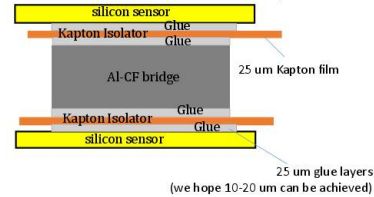
Cooling Pipe

stainless steel (**12 W/m/K**)
 2 mm inner diameter
 $100\mu\text{m}$ wall thickness

Cooling Pipe Contact

epoxy layer with **$20\mu\text{m}$** thickness and **0.2 W/m/K**

Between the sensors: Al-CF + 6 layers.



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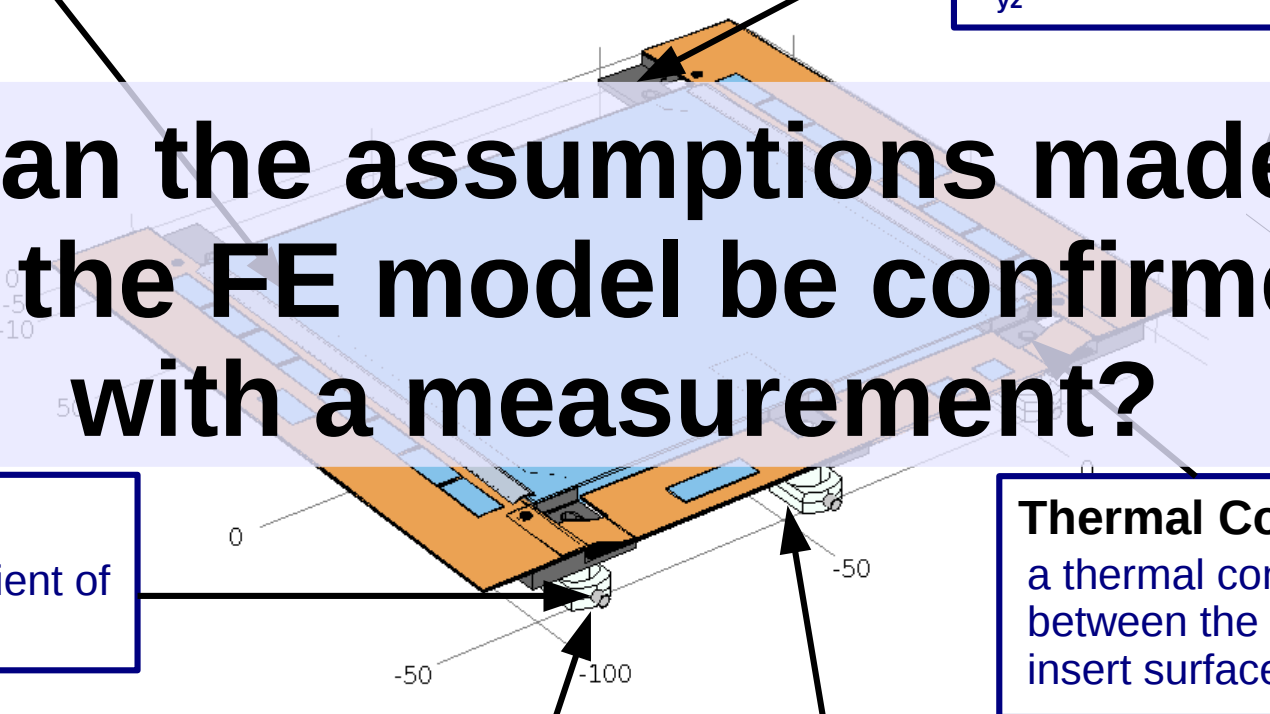
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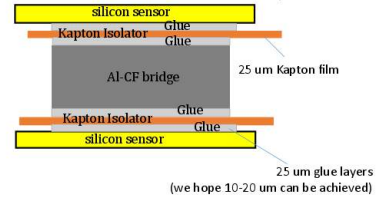
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Cooling Pipe Contact

epoxy layer with **$20\mu\text{m}$** thickness and **0.2 W/m/K**



Between the sensors: Al-CF + 6 layers.

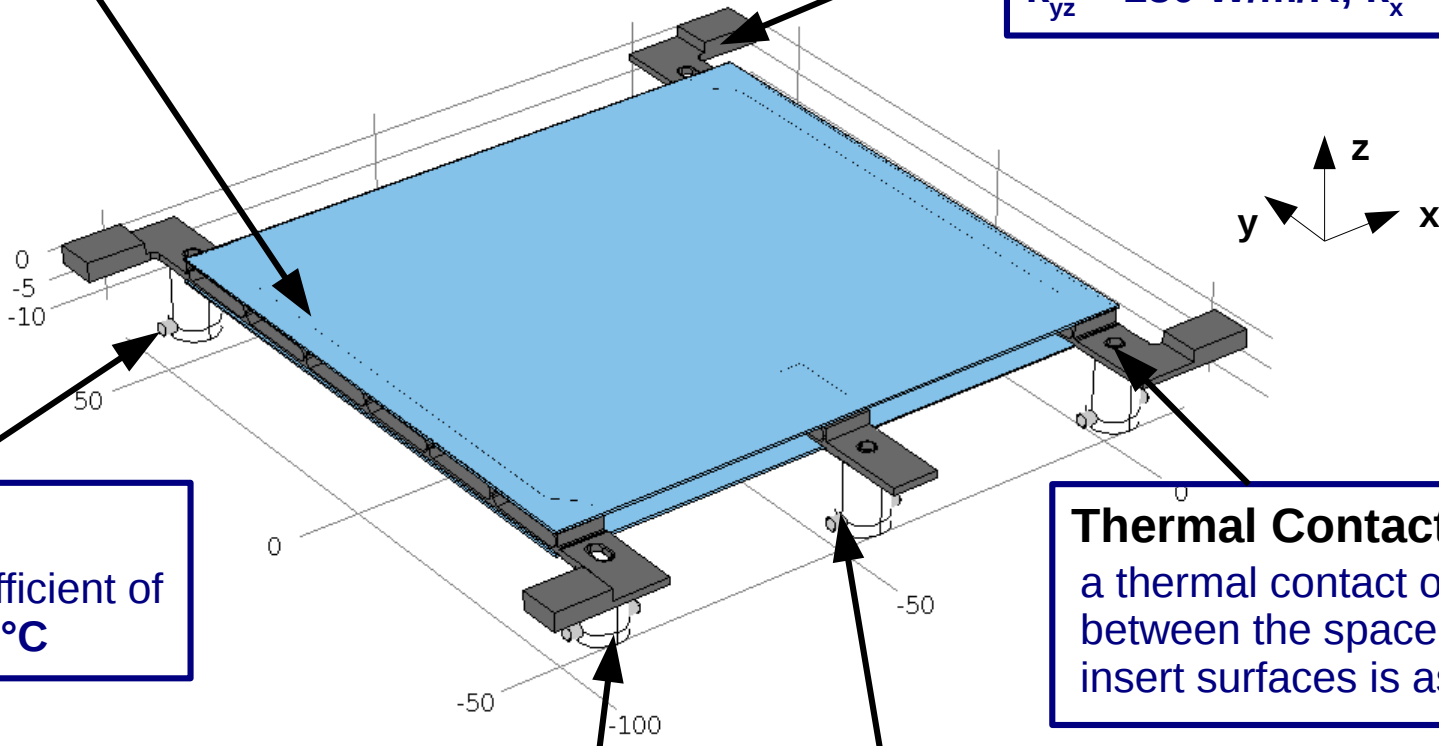


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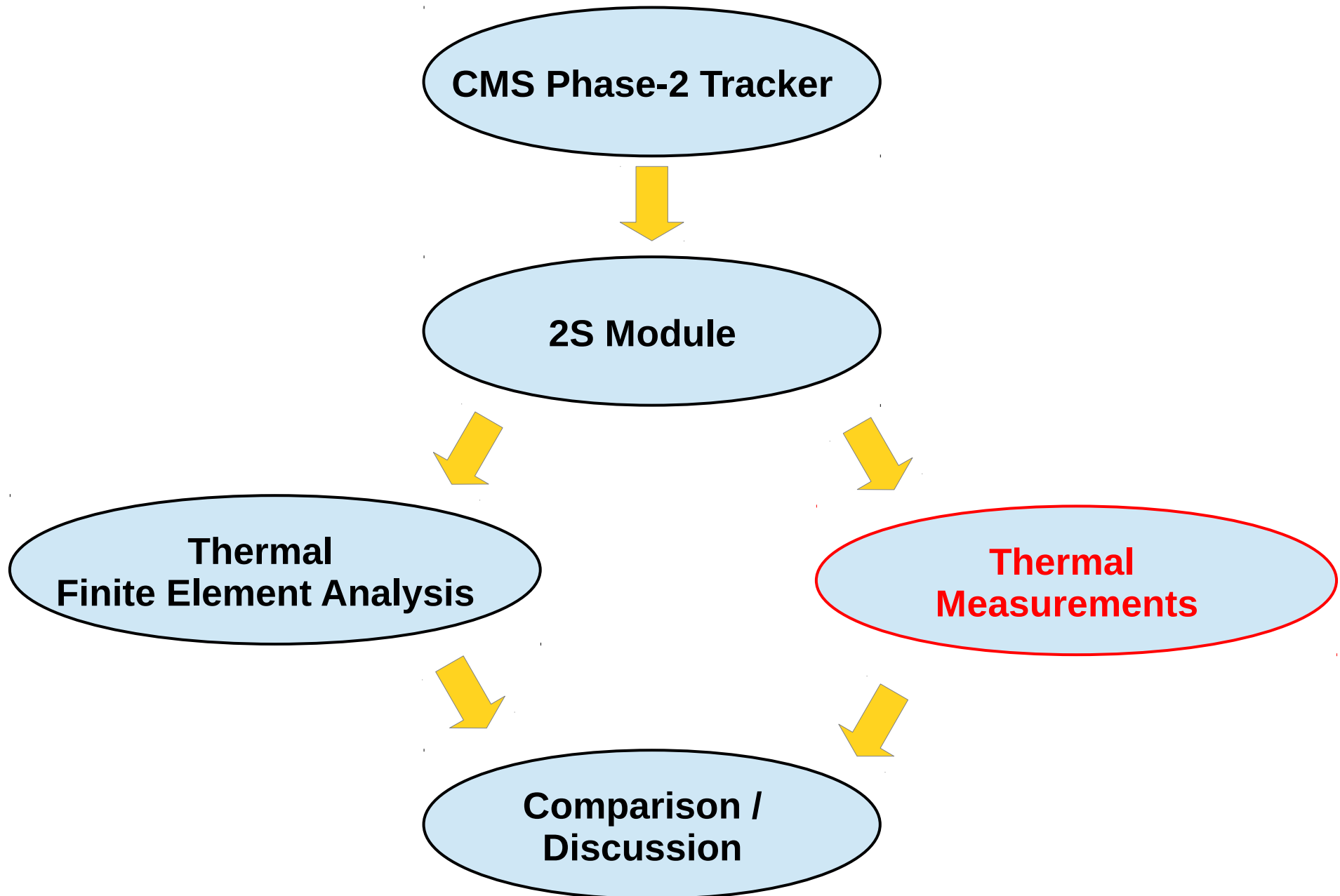
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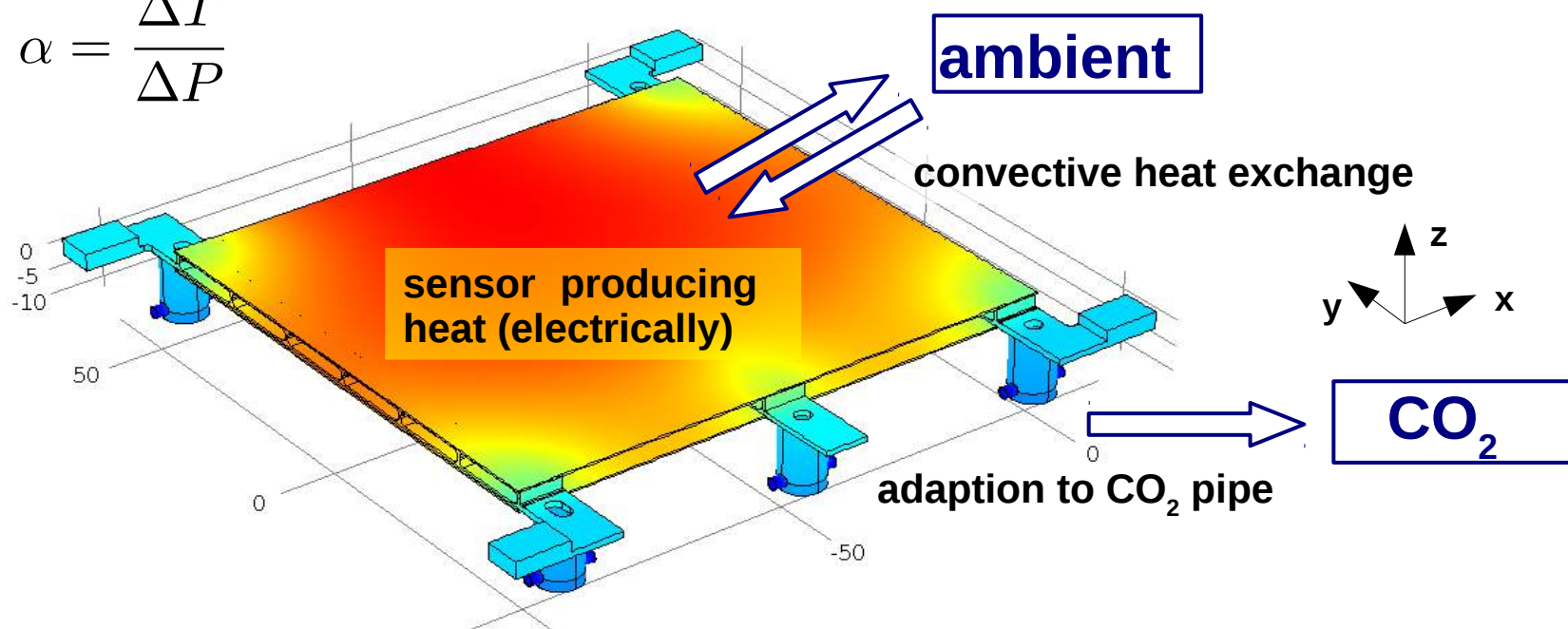
Cooling Pipe Contact

epoxy layer with $20\mu\text{m}$ thickness and 0.2 W/m/K

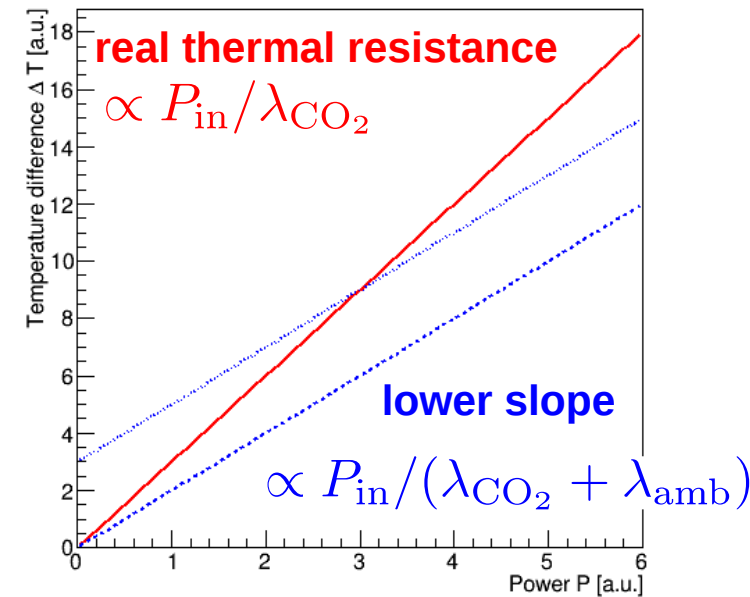
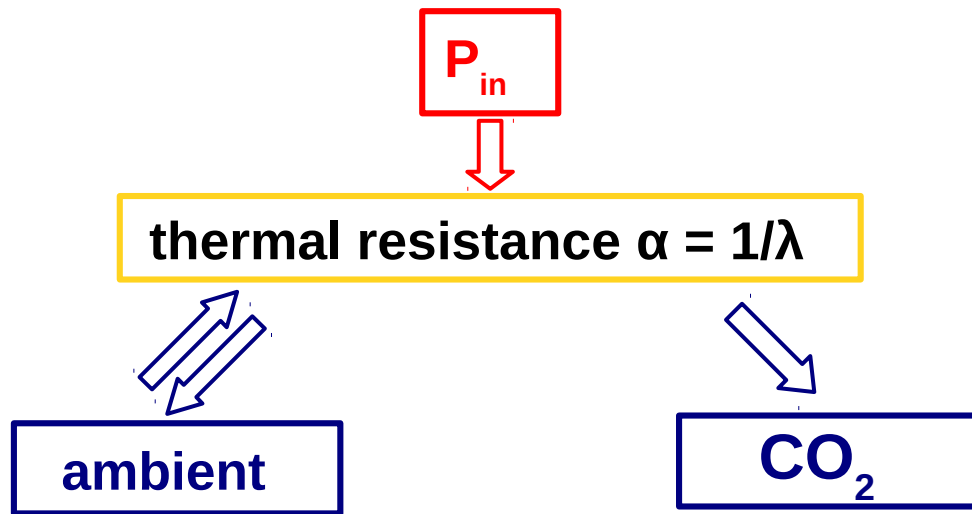


Thermal resistance

$$\alpha = \frac{\Delta T}{\Delta P}$$



- Requirements on a reliable thermal test setup
 - Feed the measured heating power fully into the sensor/module
 - difficult with heating resistors, heating foils, ...
 - Control the heat exchange with the ambient
- Generate thermal equilibrium between sensor and the ambient
 - FE simulations do not take into account the heat exchange with the ambient
 - 200m² of silicon in the detector → silicon dominates the entire volume
 - silicon sensors and ambient will be at the same temperature



- Mathematical model (equation of continuity)

$$P_{in} = \underbrace{\lambda_{amb}(T_i)(T_i - T_{amb})}_{\text{=heat exchange with the ambient}} + \underbrace{\lambda_{CO_2}(T_i - T_{CO_2})}_{\text{=heat removed by CO}_2, \text{ linear}}$$

- Solution of the equation

$$T_i = \frac{P_{in} + \lambda_{amb}T_{amb} + \lambda_{CO_2}T_{CO_2}}{\lambda_{amb} + \lambda_{CO_2}} \Rightarrow \frac{1}{\lambda_{amb} + \lambda_{CO_2}} P_{in} < \frac{1}{\lambda_{CO_2}} P_{in}$$

Find thermal equilibrium state where

$$P_{in} = \lambda_{CO_2}(T_i - T_{CO_2}) \Rightarrow T_i - T_{amb} = 0 \text{ K}$$

Illustration of the Measurement Method

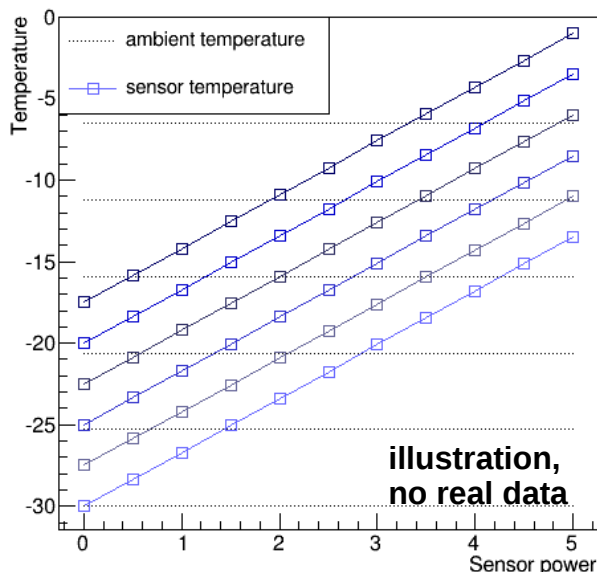


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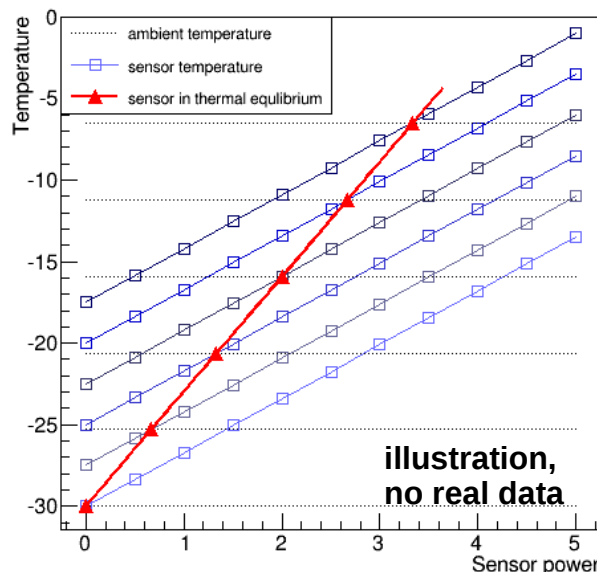
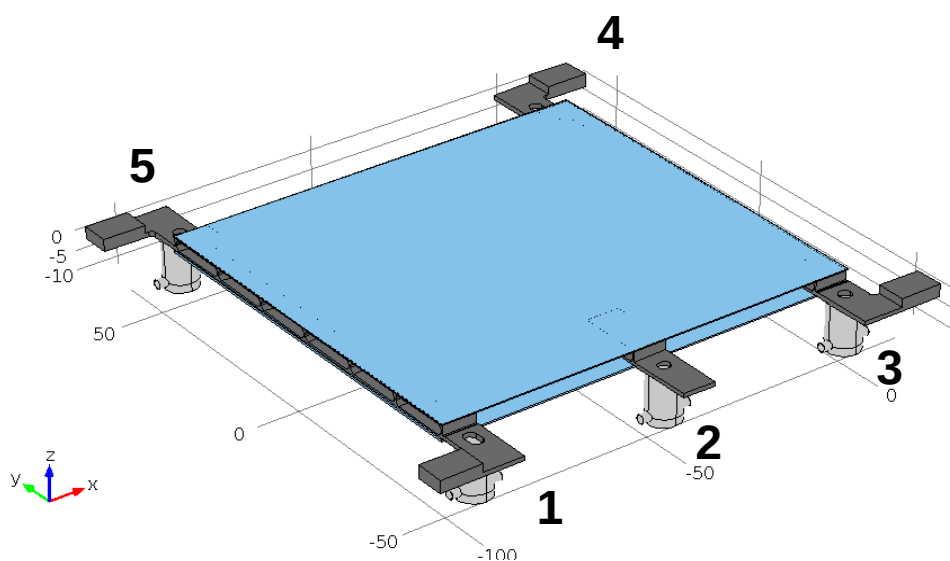
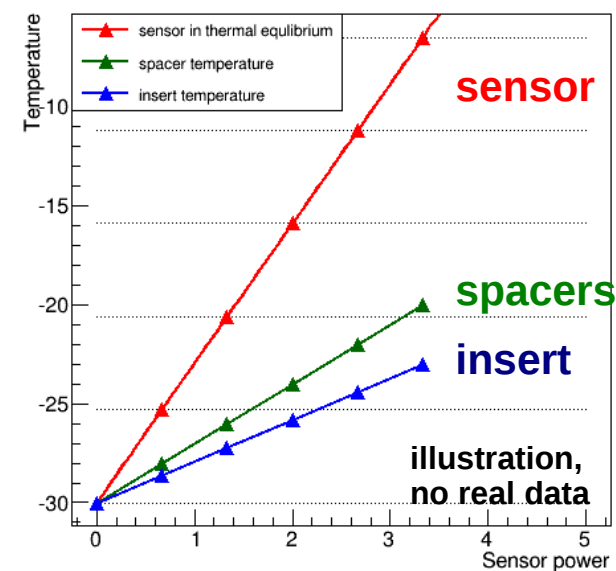


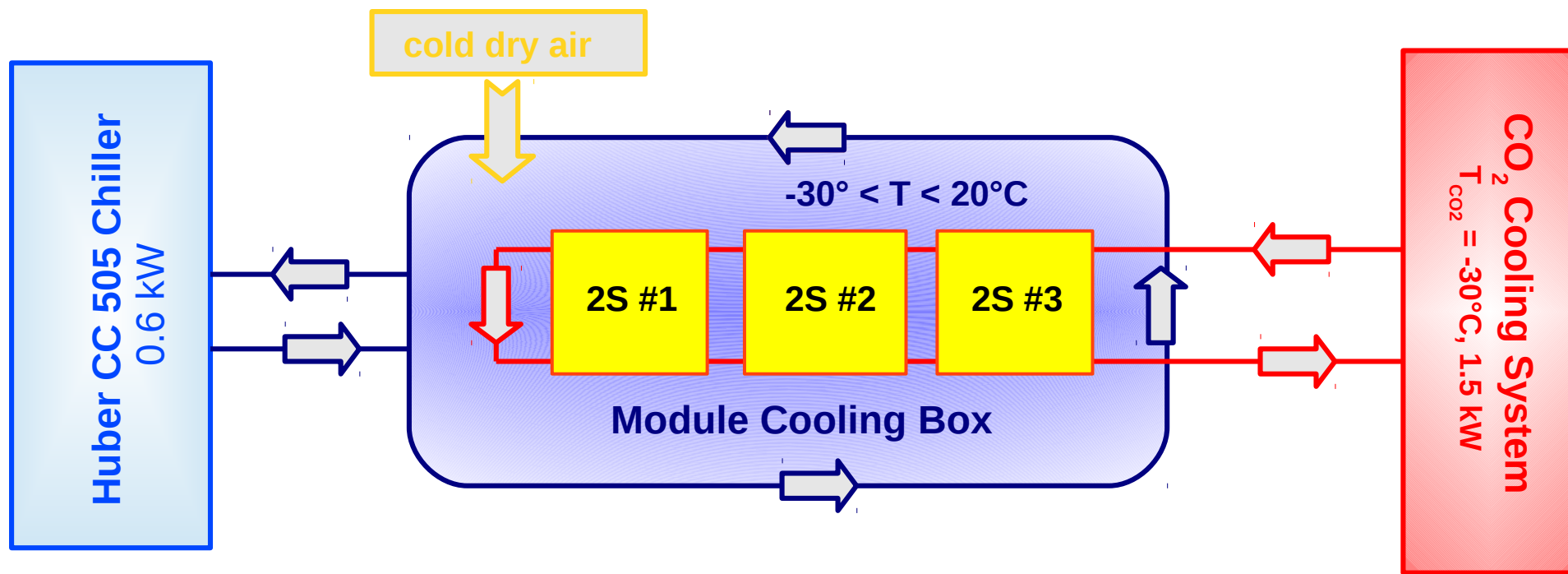
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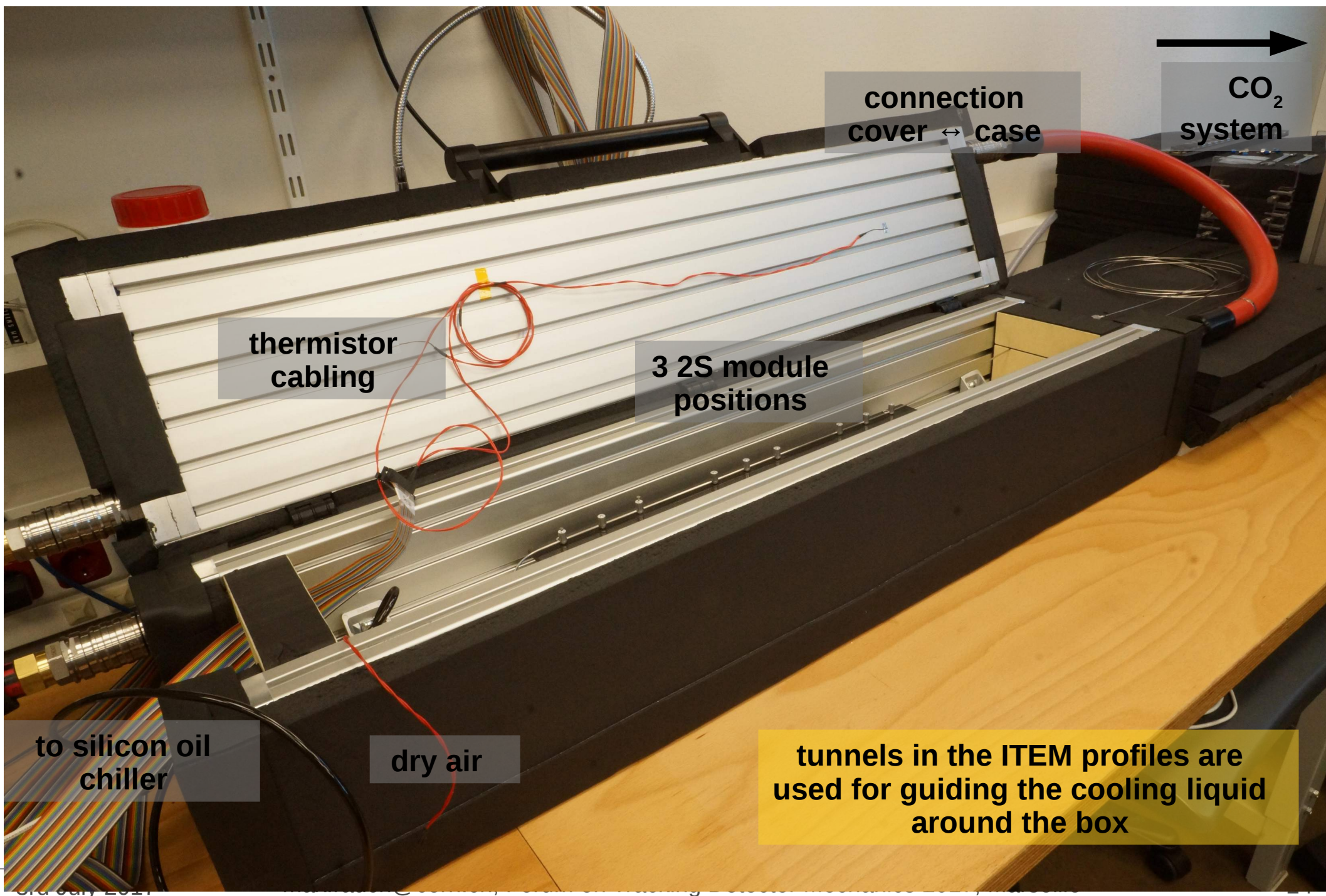
Thermal resistance

$$\alpha_{\text{sensor}} = \frac{\Delta T_{\text{equilibrium}}}{\Delta P_{\text{equilibrium}}}$$

- is used for characterization of the thermal properties
- comparison of measured and simulated data
- **goal: confirm, validate or improve parameters assumed in the FE model**



- Three positions for the 2S modules in a cooling box
- A CO₂ cooling system is available for cooling the modules: $T_{\text{CO}_2} = -30^\circ\text{C}$
- The ambient temperature of the box is controlled by a silicon oil chiller: $-30^\circ\text{C} \leq T_{\text{ambient}}$
- Cold dry air with the same temperature as the cooling box is injected



connection
cover ↔ case

CO₂
system

thermistor
cabling

3 2S module
positions

to silicon oil
chiller

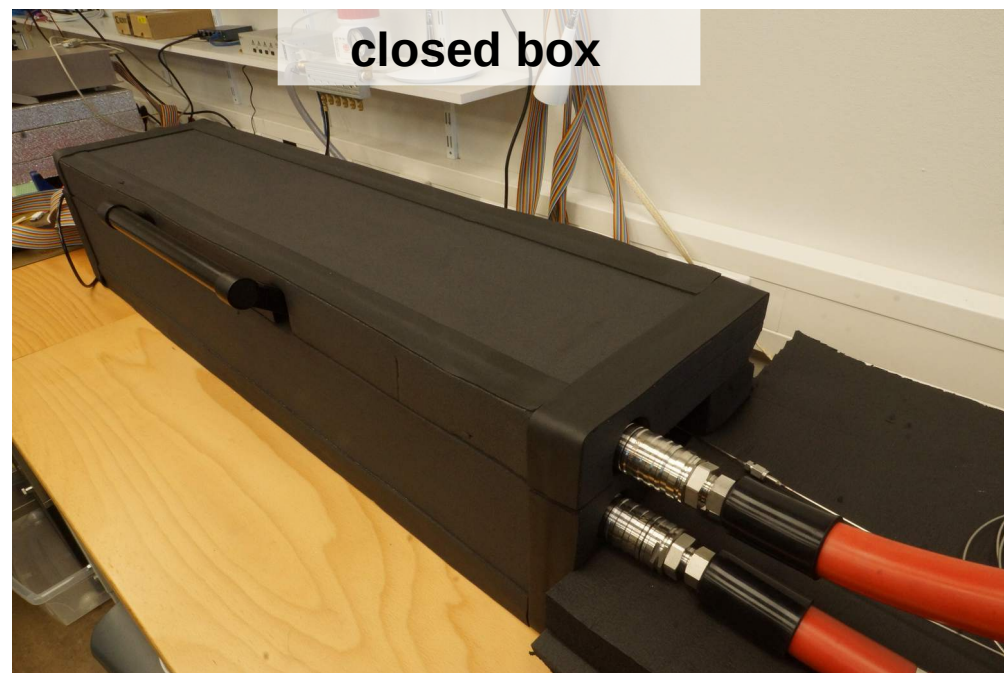
dry air

tunnels in the ITEM profiles are
used for guiding the cooling liquid
around the box

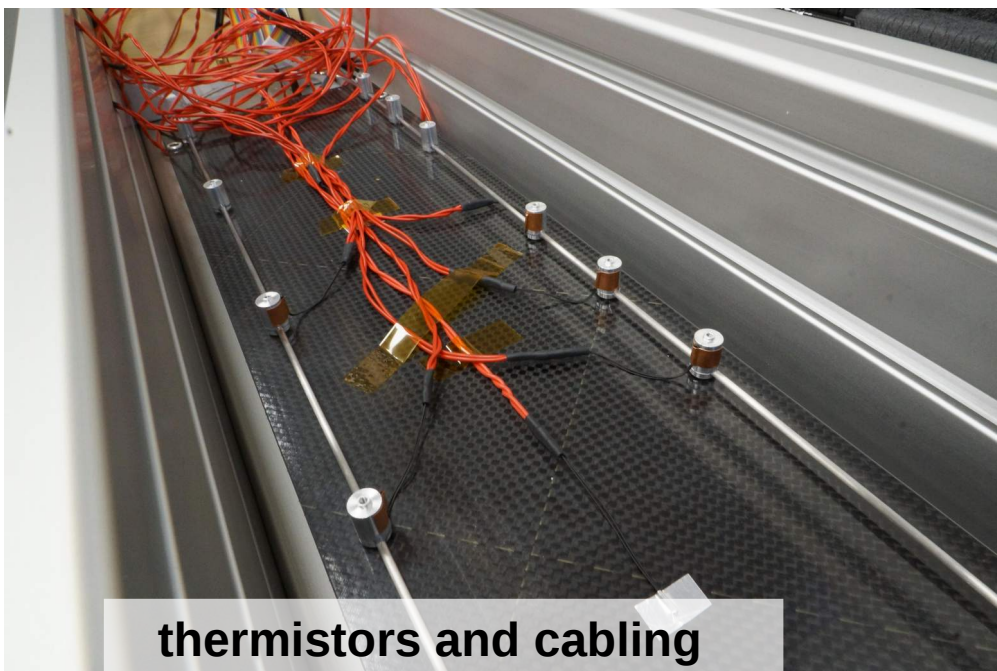
view in the inner of the box



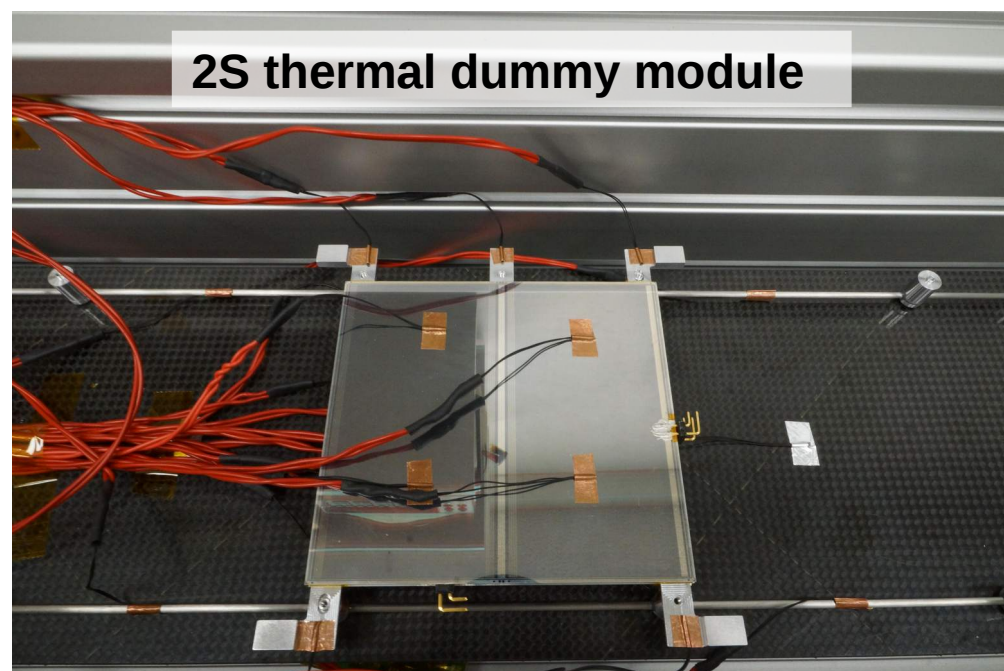
closed box

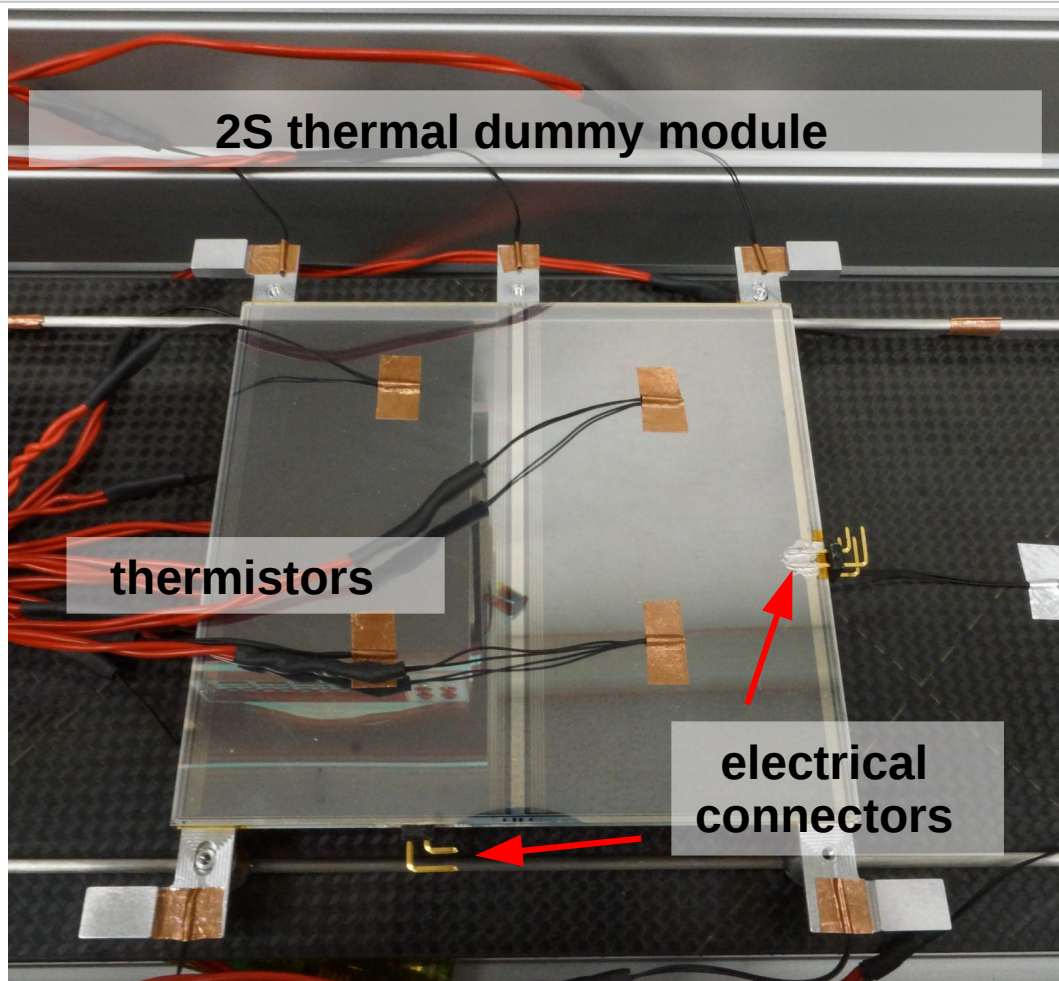


thermistors and cabling

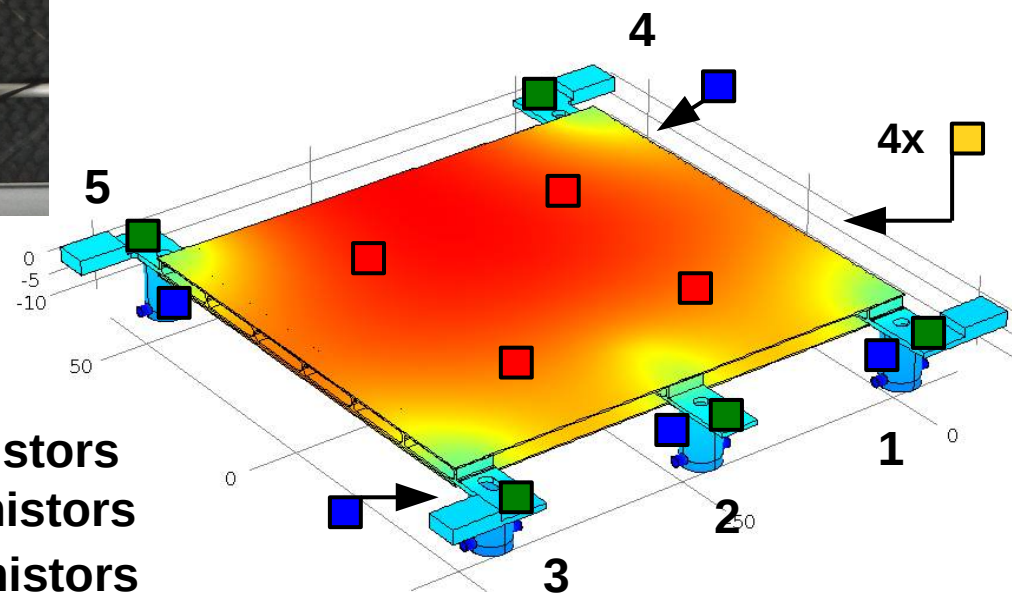


2S thermal dummy module



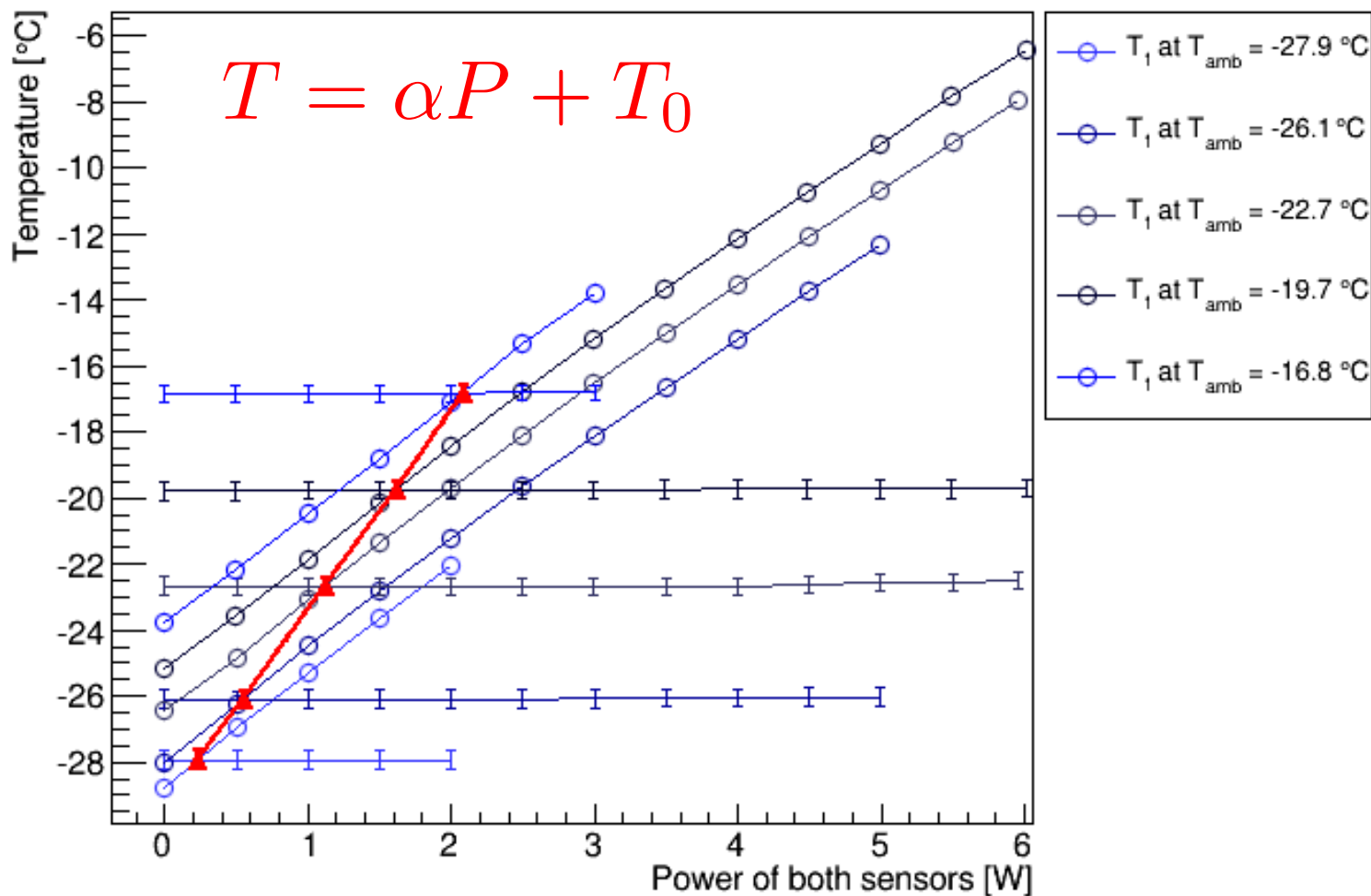


- Thermal dummy module
 - strips of the sensors not doped
 - resistance between strip side and back plane in the order of 4 Ohm
- Electrically connected with silver-based, electrically conducting glue
- Voltage sensing
 - the power is completely produced in the sensor → current through the bulk
 - power ramp up to 3W per sensor

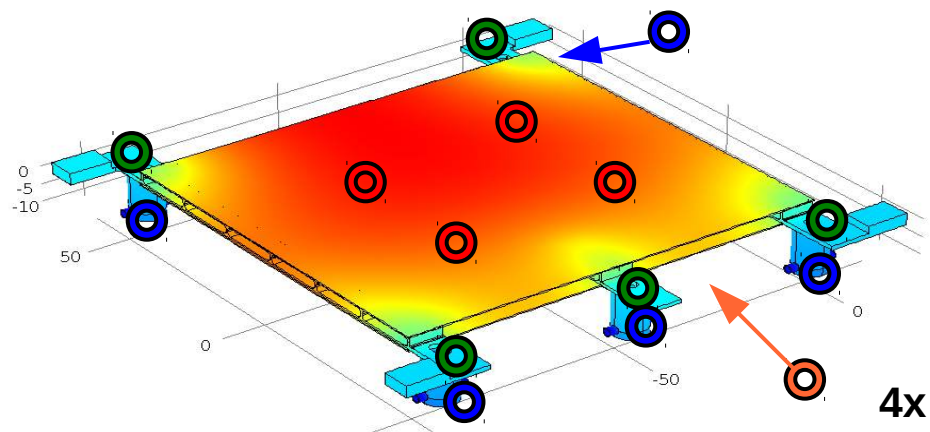


- insert thermistors
- spacer thermistors
- sensor thermistors

Measurement Series at five different T_{amb} – Sensor Temperature

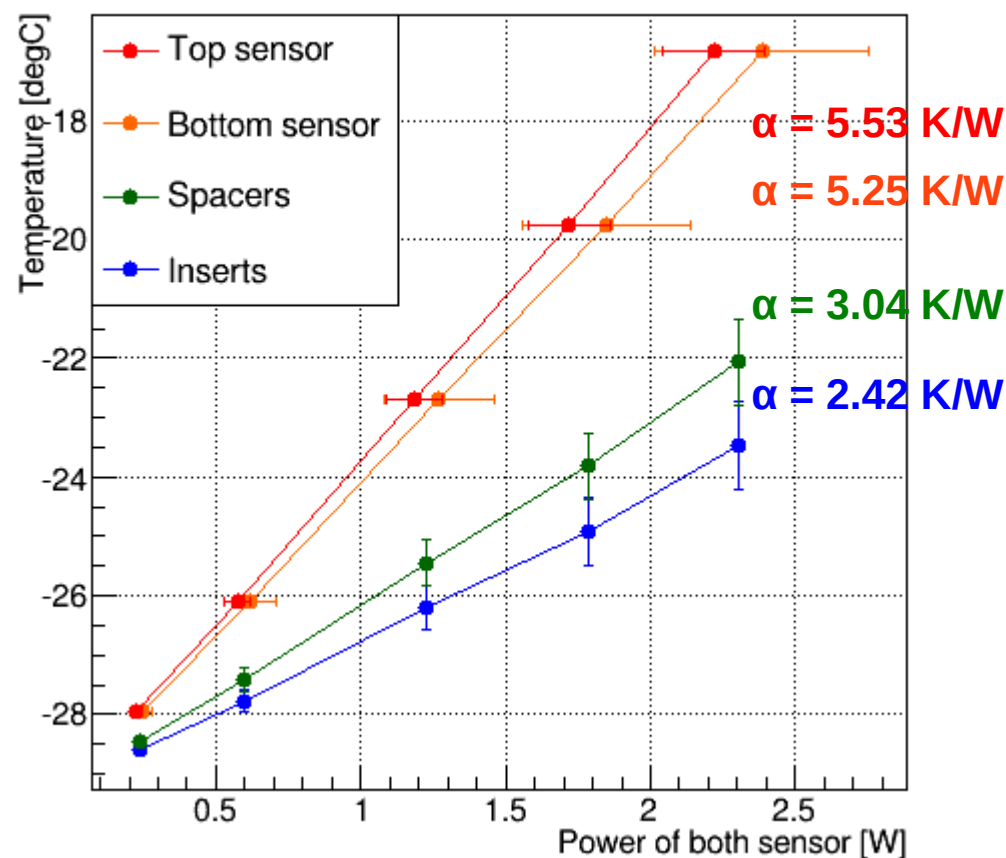


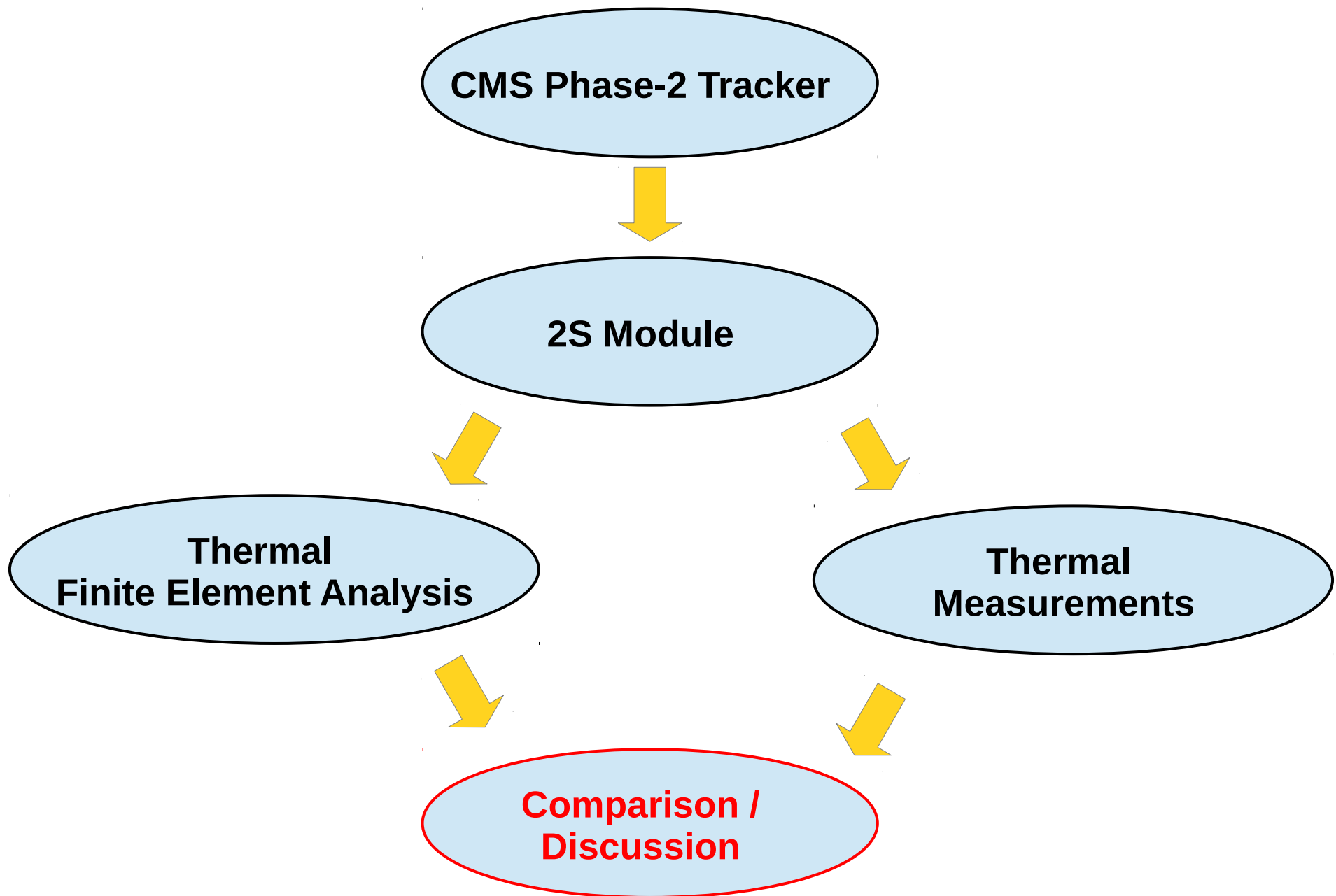
- sensor temperature differs for same power values, but different ambient temperatures
- thermal equilibrium is determined by the intersection of T_{amb} and T_{sensor}
- much higher linear slope is obtained for the thermal resistance

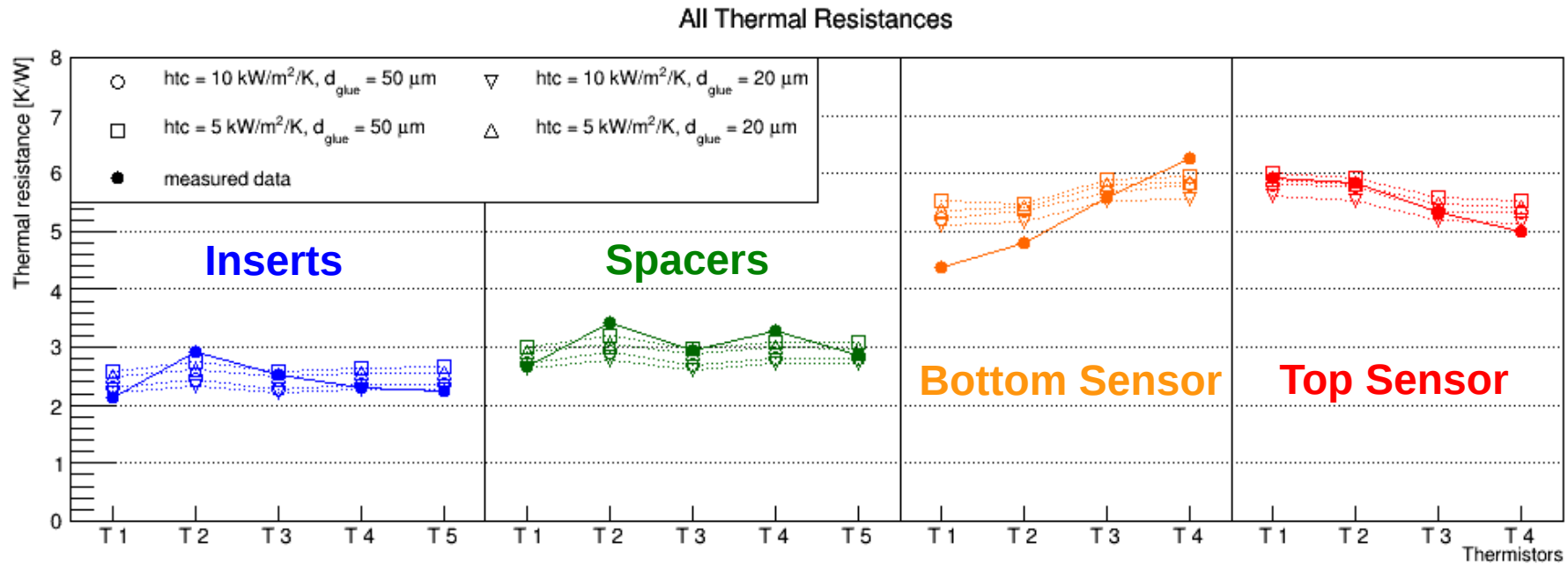


- mean values of thermal equilibrium of the single thermistors is calculated
- temperature for every single spacer and insert thermistor is evaluated
 - linear interpolation between real data points
 - mean values are plotted
- thermal resistances $\alpha = \Delta T / \Delta P$ are determined by a linear fit
- comparison with FE simulated temperatures

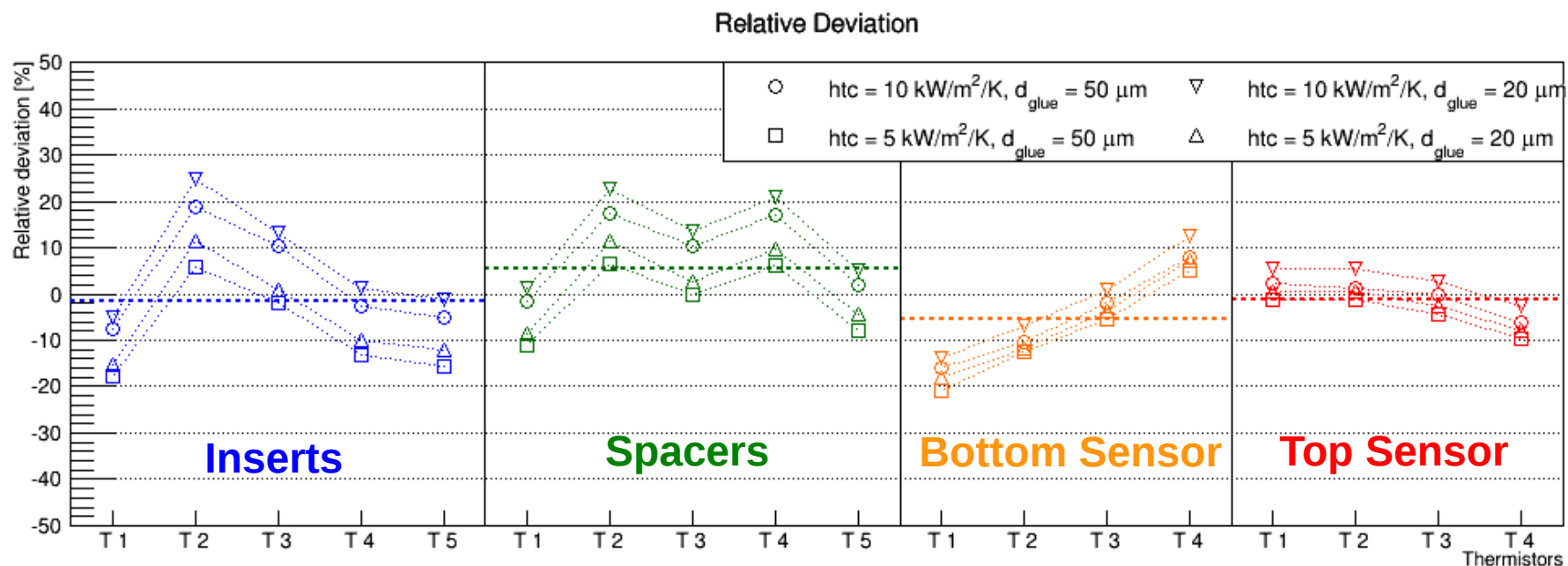
Mean Temperature Data



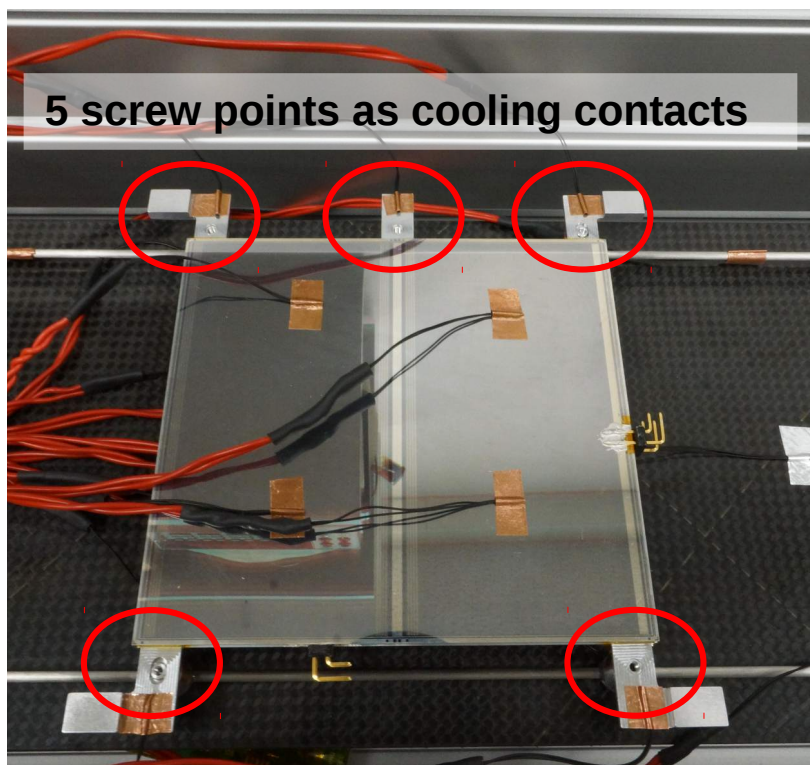




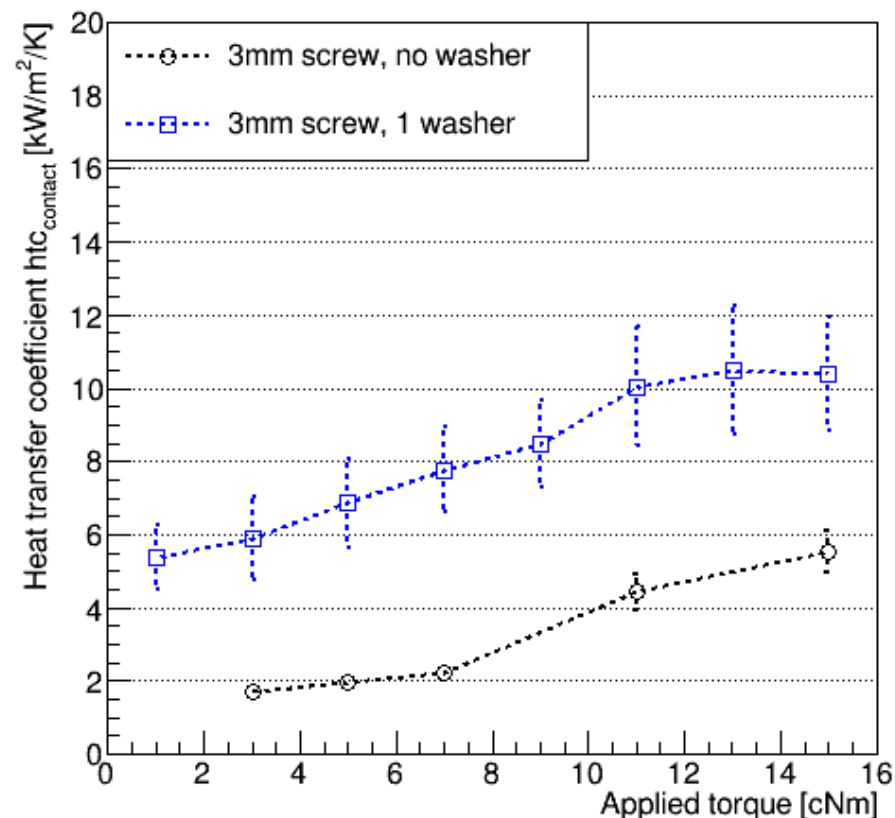
- Thermal resistances are extracted from data and compared to simulation $\alpha = \frac{\Delta T_i}{P_{\text{module}}}$
- Configuration of the FE model
 - different CO₂ heat transfer coefficients
 - different pipe glue layer thicknesses
- Absolute deviation between data and FE simulation is maximum 1.2 K/W, mostly smaller
- Average deviation per component is smaller than 0.4 K/W



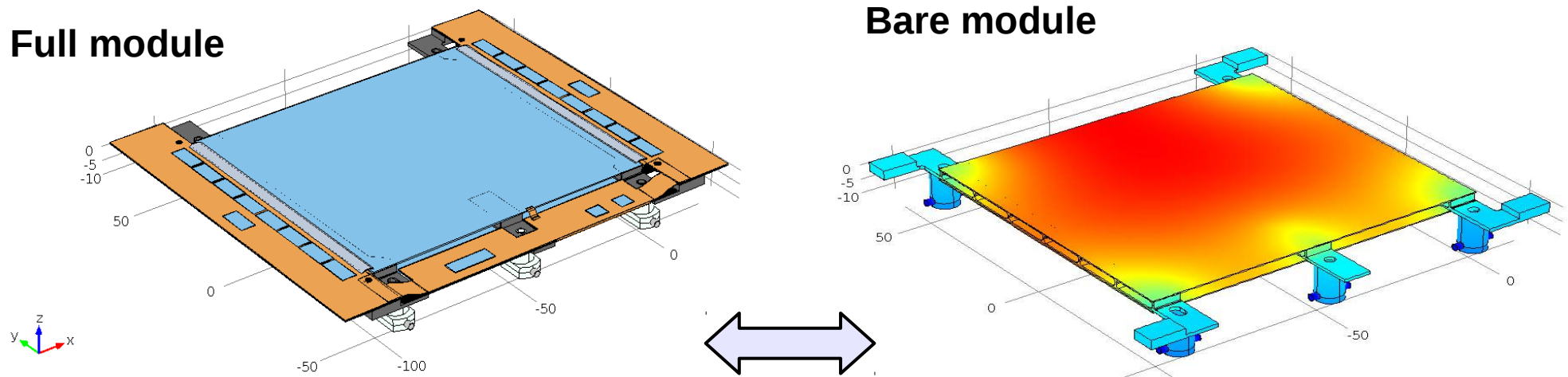
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- Absolute deviation between data and FE simulation is maximum 1.2 K/W, mostly smaller
- Average deviation per component is smaller than 0.4 K/W
- On average the FE simulated data can be confirmed with data better than 10%
- Maximum about 20% deviation for certain measurement points



Heat Transfer Coefficient vs. Applied Torque



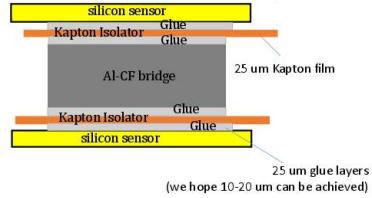
- M2 screws, 3mm long, are tested with an without washer
- heat transfer coefficient is measured
 - **significant dependence on torque**
- in FE simulation, 10kW/m²/K is assumed
 - plateau of 10kW/m²/K reached with 11cNm
 - **confirmation by measurement**



- not a full module was tested, but a bare module
 - bare module contains all thermal interfaces relevant for cooling of the sensor
 - in full modules, the hybrids are glued on other parts of the spacers
- critical parameter is the silicon sensor power ↔ temperature of the silicon sensor
- good understanding of the adaption to the cooling system is important
 - prediction of thermal runaway with FE simulations
- thermal interfaces between sensor and cooling system could be measured
- good compatibility with values assumed in FE simulations for all measured data
 - reliability of FE simulations

Thermal Interfaces

Between the sensors: Al-CF + 6 layers.



Epoxy-Kapton-Epoxy Layer

described with effective thermal through-plane conductivity 0.3 W/m/K and with **$65\mu\text{m}$** thickness

AlCF-Spacer

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Can the assumptions made for the FE model be confirmed with a measurement?

CO₂ Heat Flux

heat transfer coefficient of **$5 \text{ kW/m}^2/\text{K}$** at **$-35^\circ\text{C}$**

Thermal Contact

a thermal contact of **$10 \text{ kW/m}^2/\text{K}$** between the spacers and in the insert surfaces is assumed

Cooling Pipe

stainless steel (**12 W/m/K**)
 2 mm inner diameter
 $100\mu\text{m}$ wall thickness

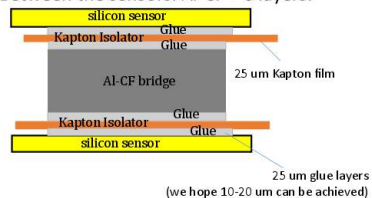
Cooling Pipe Contact

epoxy layer with **$20\mu\text{m}$** thickness and **0.2 W/m/K**

Thermal Interfaces



Between the sensors: Al-CF + 6 layers.



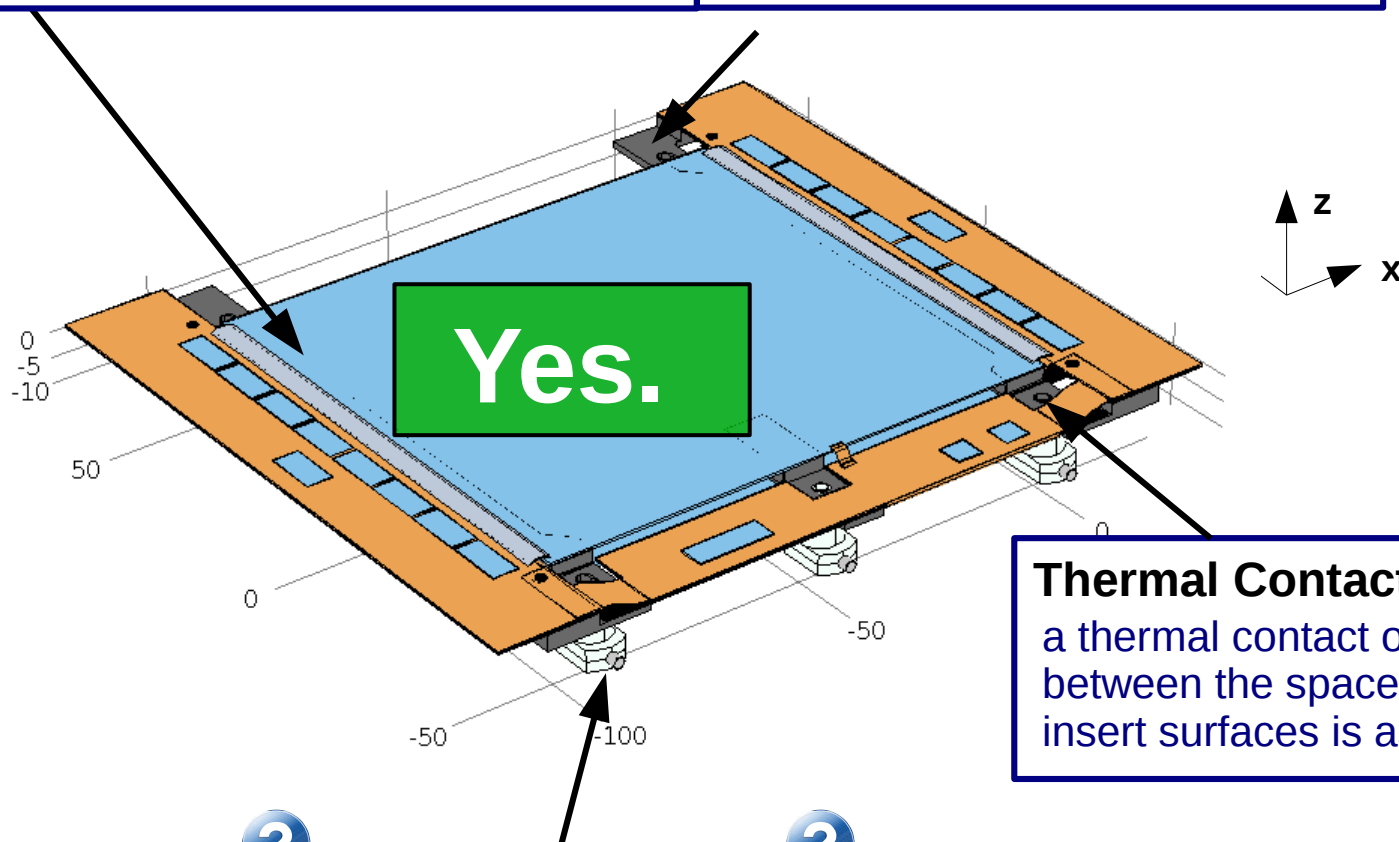
Epoxy-Kapton-Epoxy Layer

described with effective thermal through-plane conductivity 0.3 W/m/K and with **$65\mu\text{m}$** thickness



AlCF-Spacer

anisotropic thermal conductivity due to carbon fiber orientation:
 $k_{yz} = 230 \text{ W/m/K}$; $k_x = 120 \text{ W/m/K}$



Thermal Contact

a thermal contact of **$10\text{kW/m}^2/\text{K}$** between the spacers and in the insert surfaces is assumed



CO₂ Heat Flux

heat transfer coefficient of **$5 \text{ kW/m}^2/\text{K}$** at **$-35^\circ\text{C}$**



Cooling Pipe

stainless steel (**12 W/m/K**)
 2mm inner diameter
 $100\mu\text{m}$ wall thickness

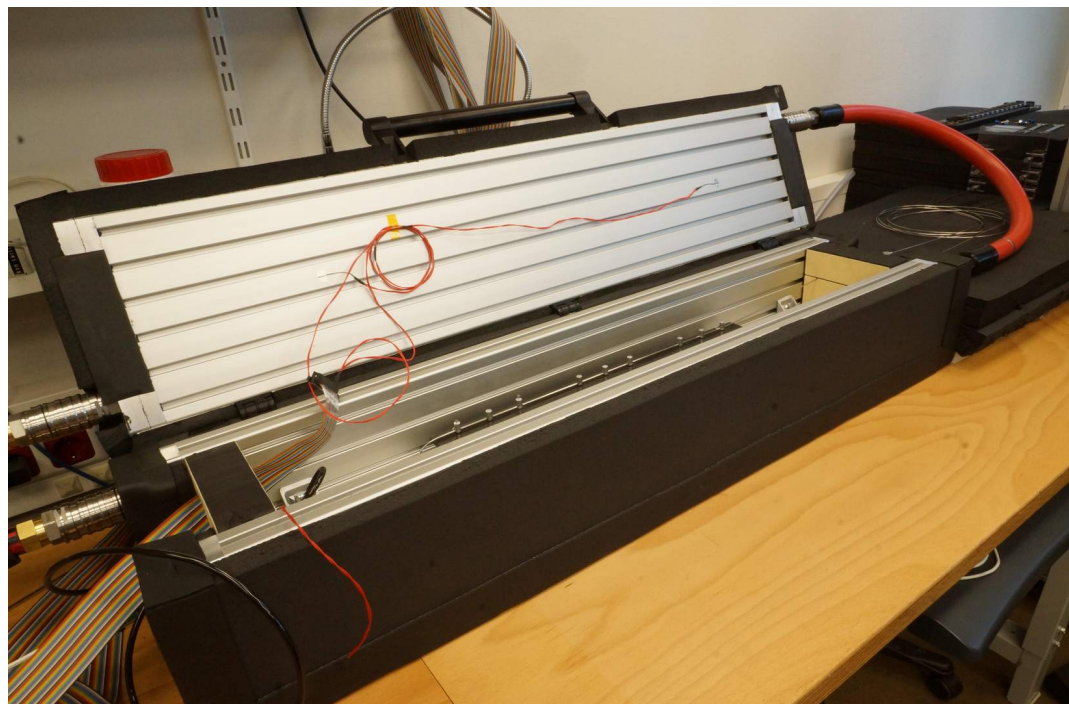


Cooling Pipe Contact

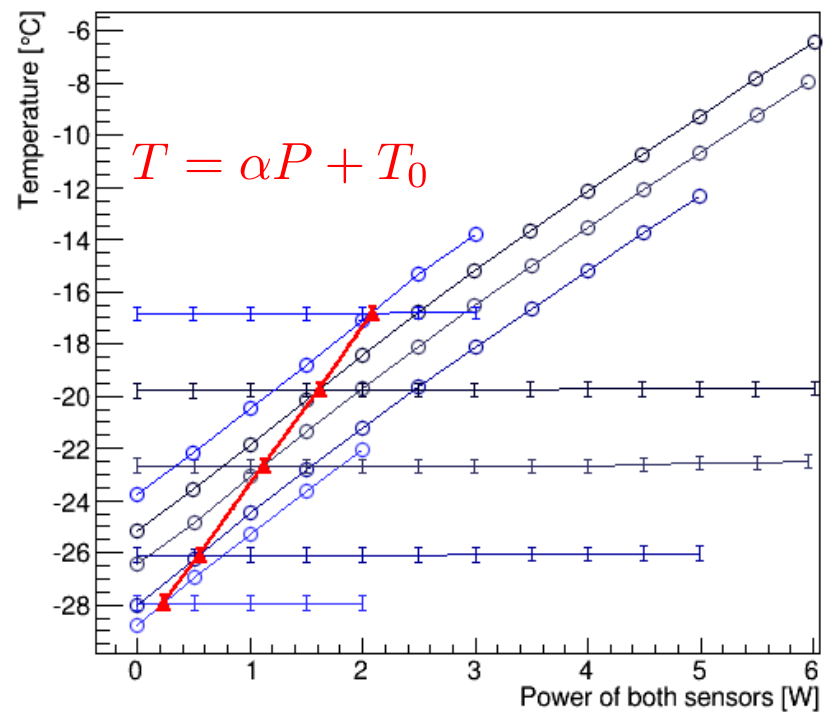
epoxy layer with **$20\mu\text{m}$** thickness and **0.2W/m/K**



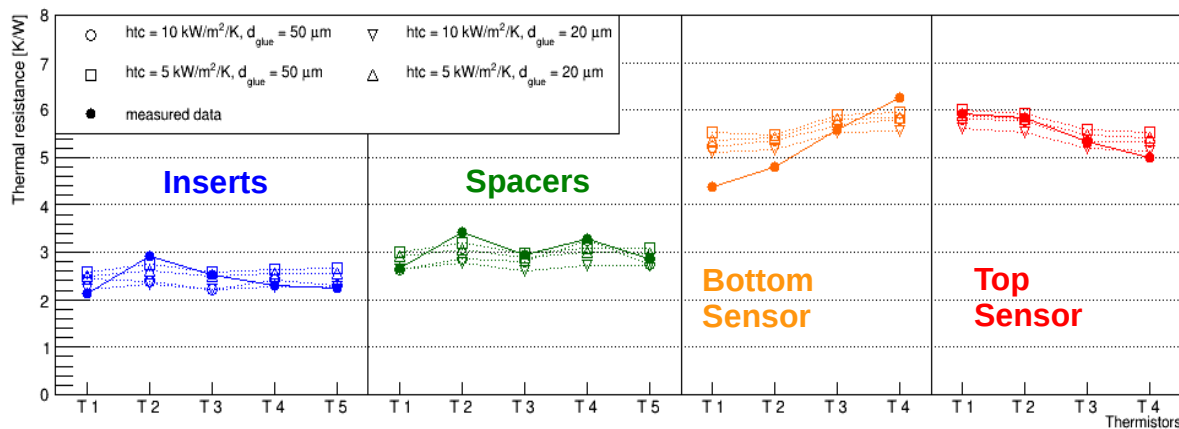
- description and understanding of the thermal properties of silicon sensors is important for detector development in the HL-LHC detector environment
 - radiation damage, sensor power, thermal runaway, ...
- FE simulations are used to estimate the thermal performance of the detector modules
 - assumptions are made for thermal interfaces like glue layer, heat transfer coefficients, ...
- thermal measurements are necessary to confirm or adjust assumptions made in the FEA
 - many systematic uncertainties and non linearities: convection, heat input from the ambient, ...
- a setup was developed and built which allows to understand and avoid a lot of systematic uncertainties
 - control of the ambient temperature
 - controlled heat input into the silicon sensors
- linear behavior of temperature in dependence on heat load can be measured
- thermal resistances assumed in FE simulations could be confirmed with good accuracy



Measurement Series at six Different T_{amb}



All Thermal Resistances

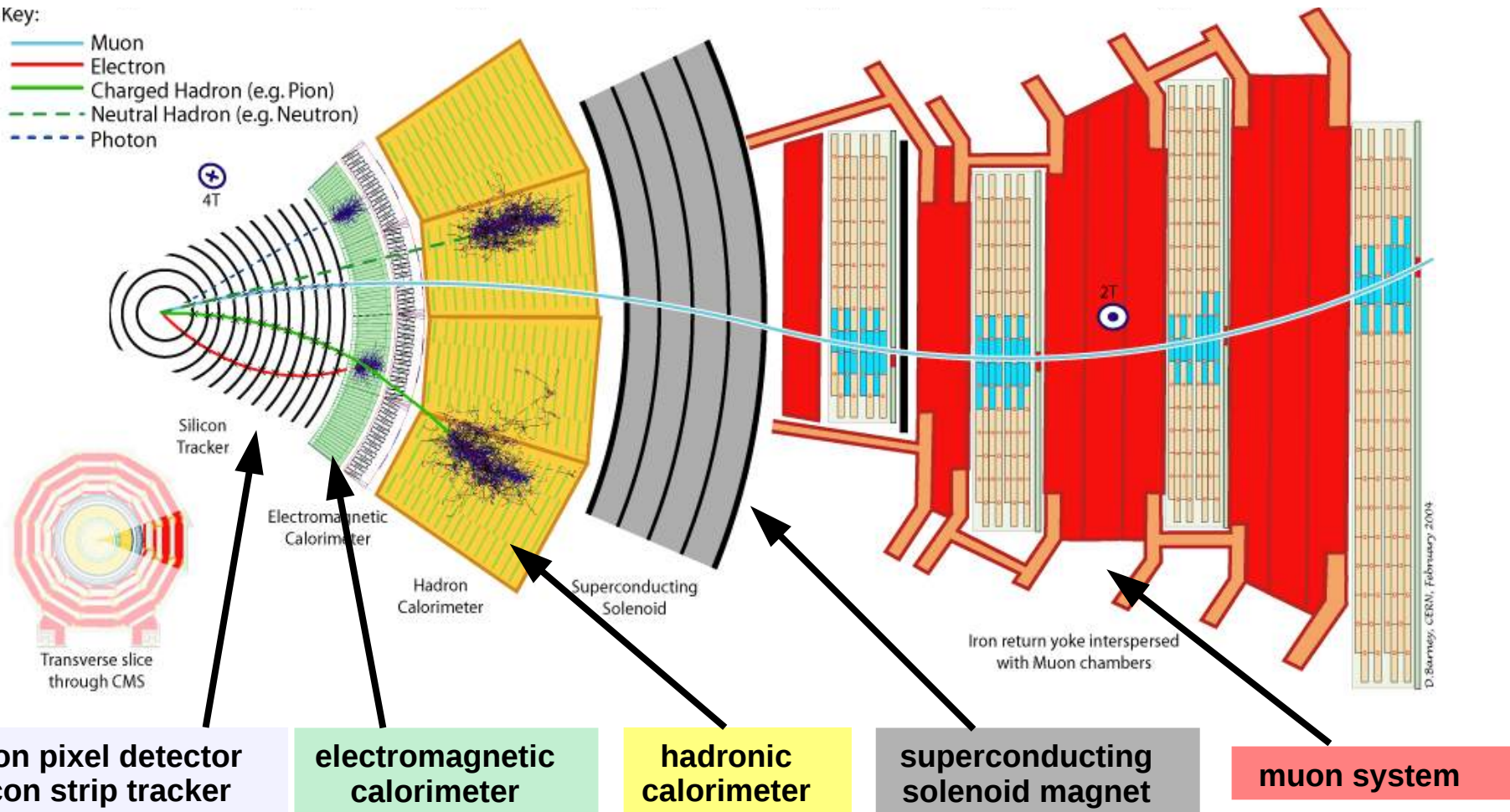


Additional Material

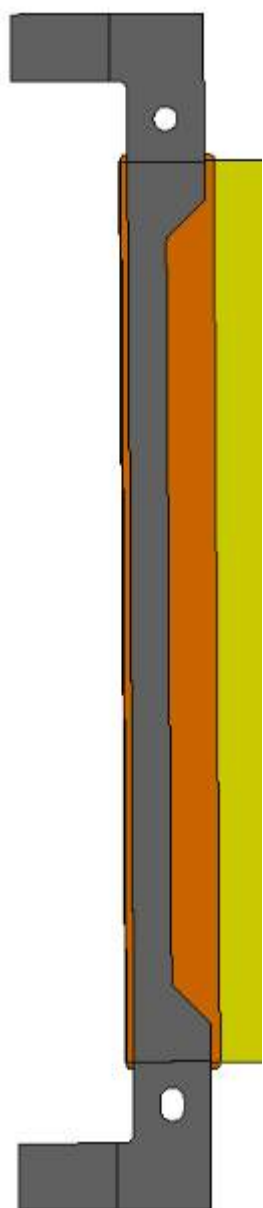
radius 7.5m

Key:

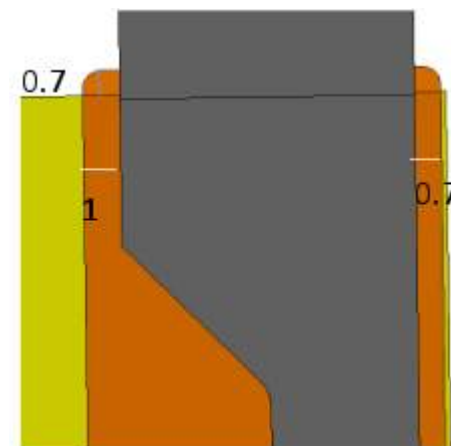
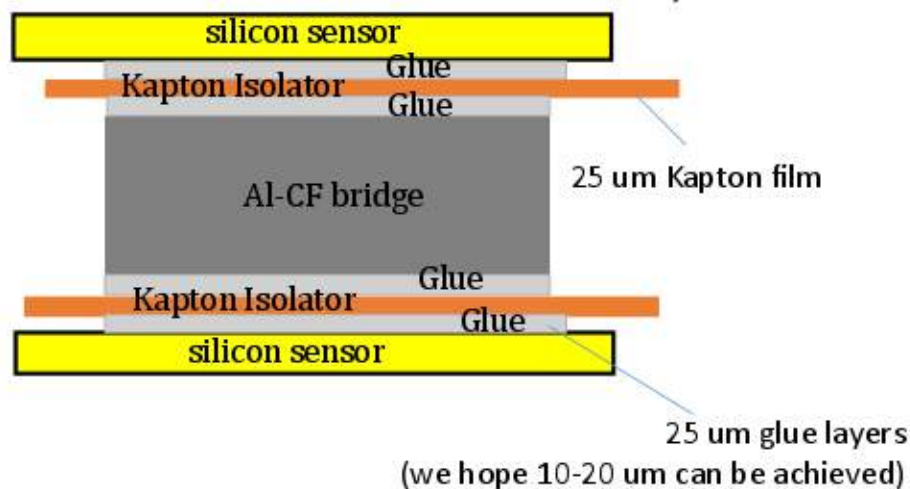
- Muon
- Electron
- Charged Hadron (e.g. Pion)
- - - Neutral Hadron (e.g. Neutron)
- - - Photon



- multipurpose detector with various detector components
- silicon strip tracker is an important part for the particle and event reconstruction



Between the sensors: Al-CF + 6 layers.



- Back plane of the sensors (inner plane) is on high voltage: -600V
- Electrical isolation of the sensor is necessary
 - Solution is gluing 25 μ m thick Kapton strips between sensor and spacer
 - Kapton MT+ with 0.8 W/m/K, thickness 25 μ m
 - thermally worse contact between sensor and cooling system
 - two layers of glue
 - Epoxy (Polytec 601-LV) with 0.2W/m/K and 20 μ m thickness
- Effective thermal through-plane-conductivity of the glue-Kapton-glue sandwich: 0.3 W/m/K

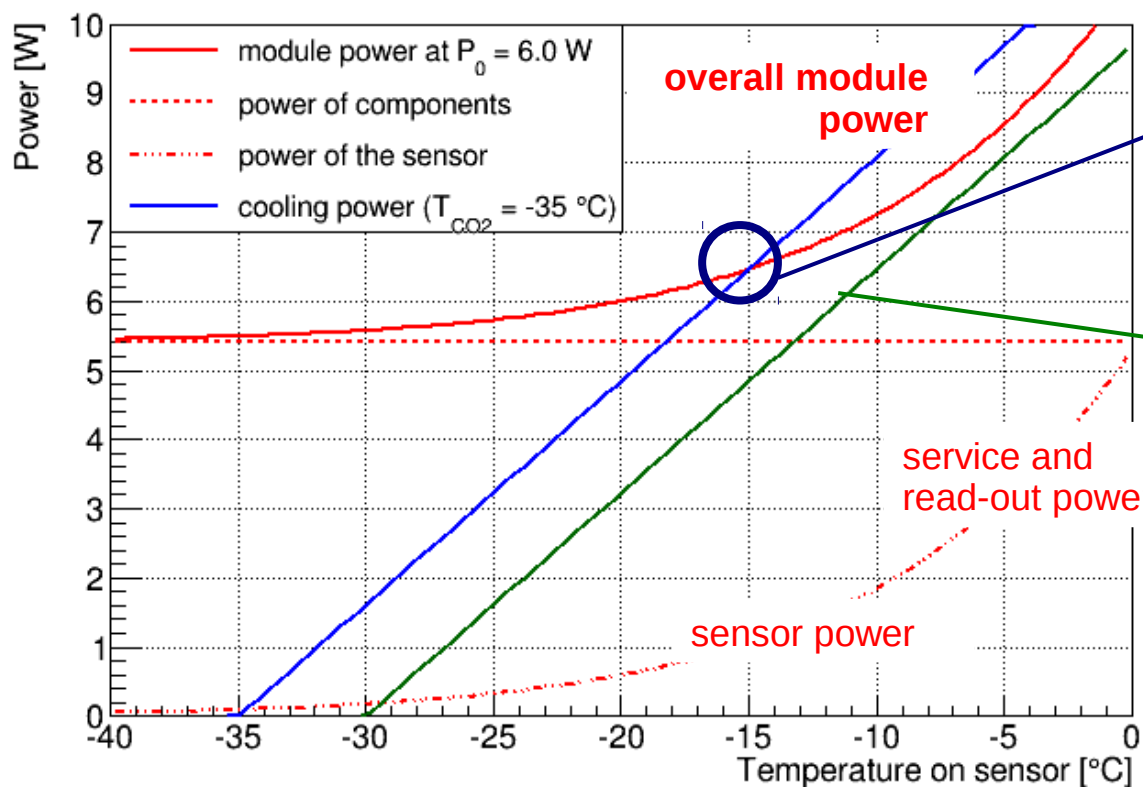
- Irradiated silicon sensors generate temperature dependent leakage current and thus power

$$P_{\text{sensor}} \propto P_0 \frac{T^2}{T_0^2} \exp \left[- \frac{\Delta E}{2 k_B} \left(\frac{1}{T} - \frac{1}{T_0} \right) \right]$$

- Other components (read-out chips, DC-DC converters, ...) produce a constant power

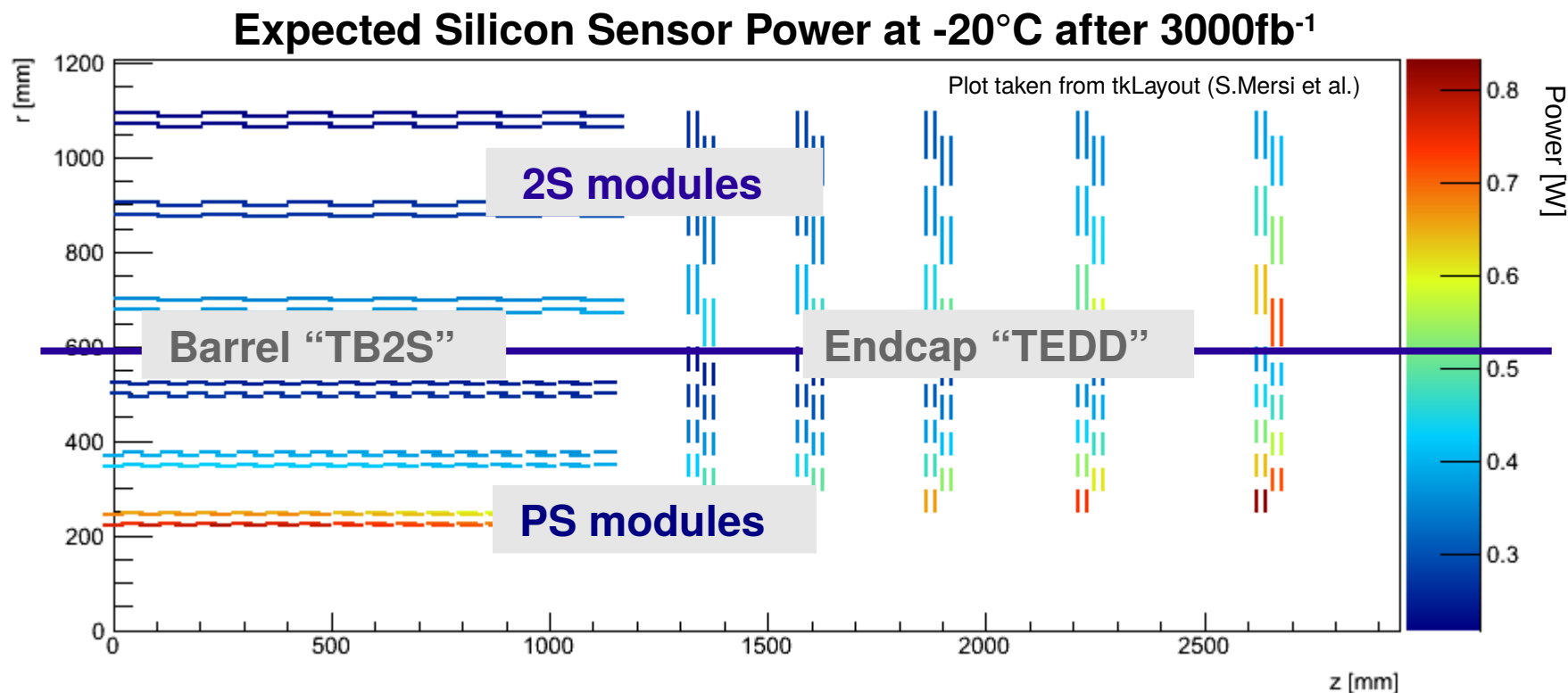
$$P_{\text{components}} = \text{const.}$$

- Linear dependence of the cooling power $P_{\text{cooling}} \approx \alpha(T_{\text{module}} - T_{\text{coolant}})$

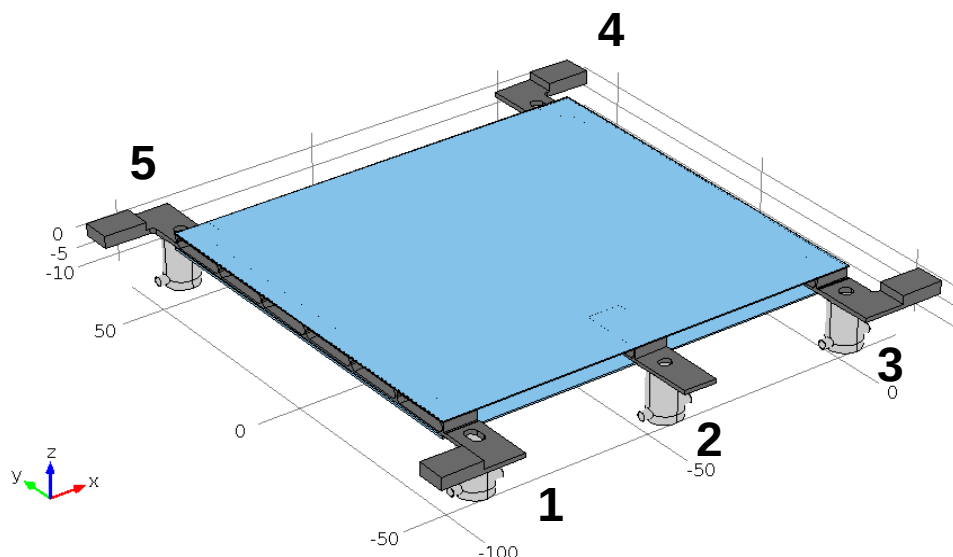


cooling possible at $T_{\text{CO}_2} = -30$ °C

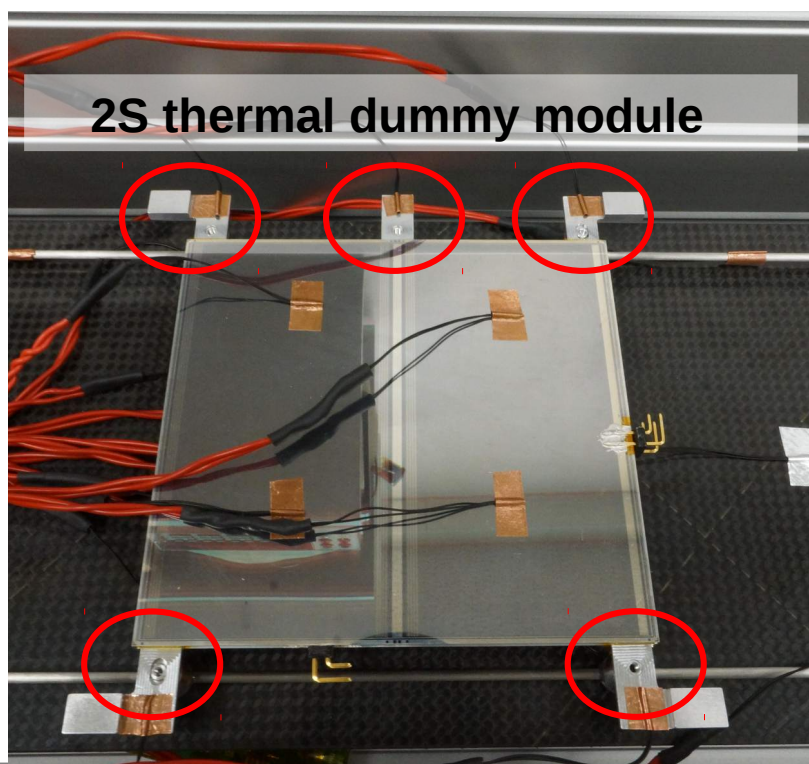
no cooling possible at $T_{\text{CO}_2} = -25$ °C:
“Thermal Runaway”



- Sensor power increases linearly with radiation damage
- highest irradiation for the 2S modules in the forward regions
- Technical Design Report value for -20°C sensor temperature:
 - maximum expected power is 570mW for 2S modules in the endcap



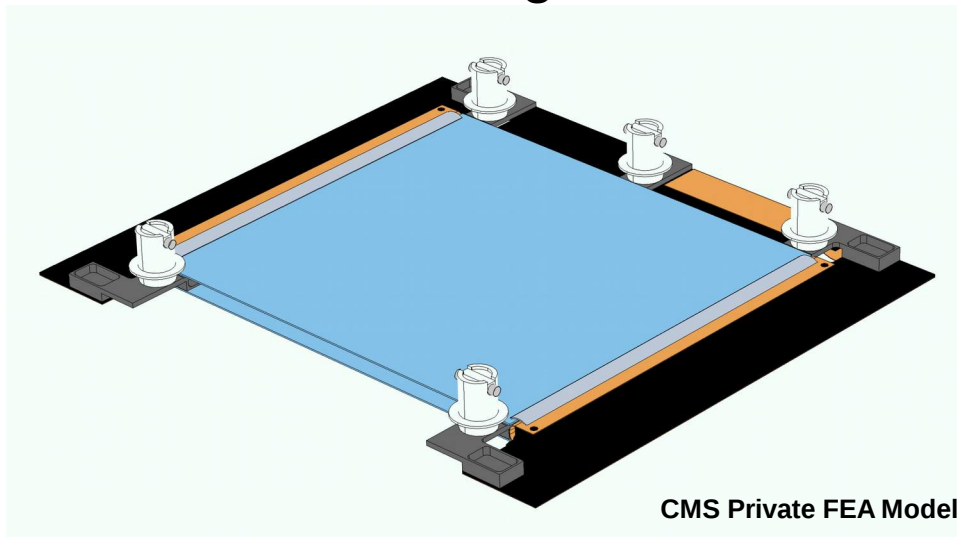
- For the five screwing junctions a heat transfer coefficient of $10 \text{ kW/m}^2/\text{K}$ is assumed
 - confirmed with measurements
- M2 screws fastened with a torque of 9 cNm were used
- Question: How does the thermal contact depend on the applied torque?
 - Answer: Measurement!



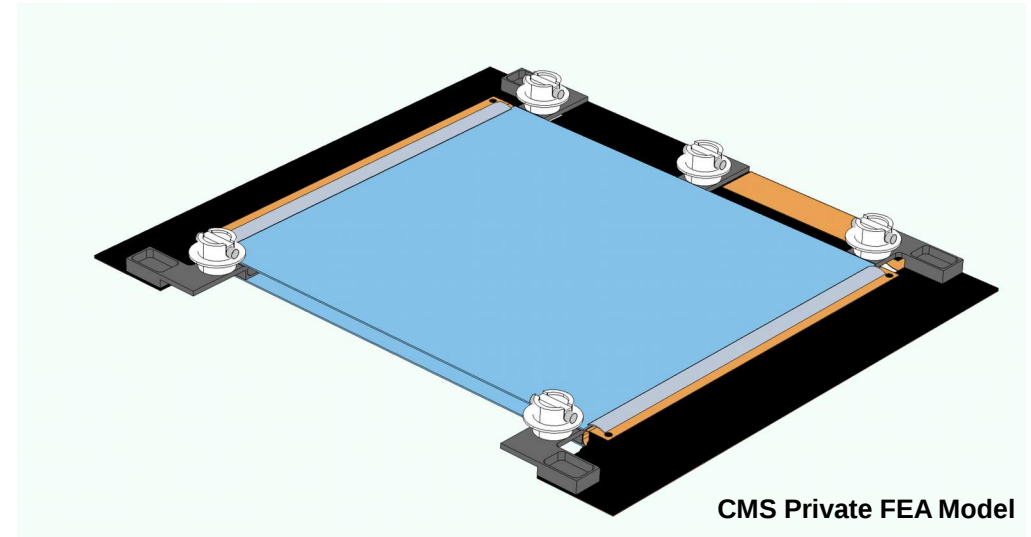
Three Measurement Series

- Serie #1:
 - M2 screws, 4mm long thread, 2 washers
- Serie #2:
 - M2 screws, 3mm long thread, 1 washer
- Serie #3:
 - M2 screws, 3mm long thread, no washer

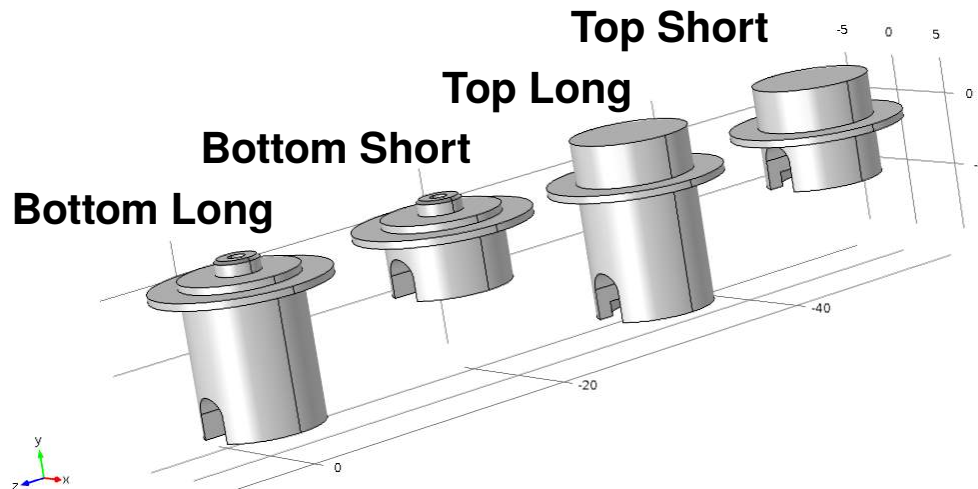
2S Module with “Long Inserts”



2S Module with “Short Inserts”

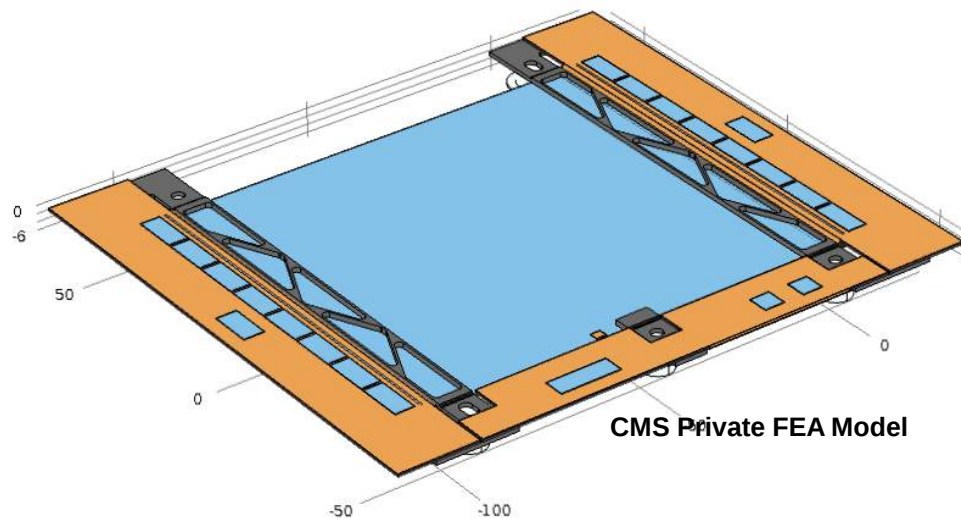


- two types of cooling inserts “Long” and “Short” due to the cooling pipe routing

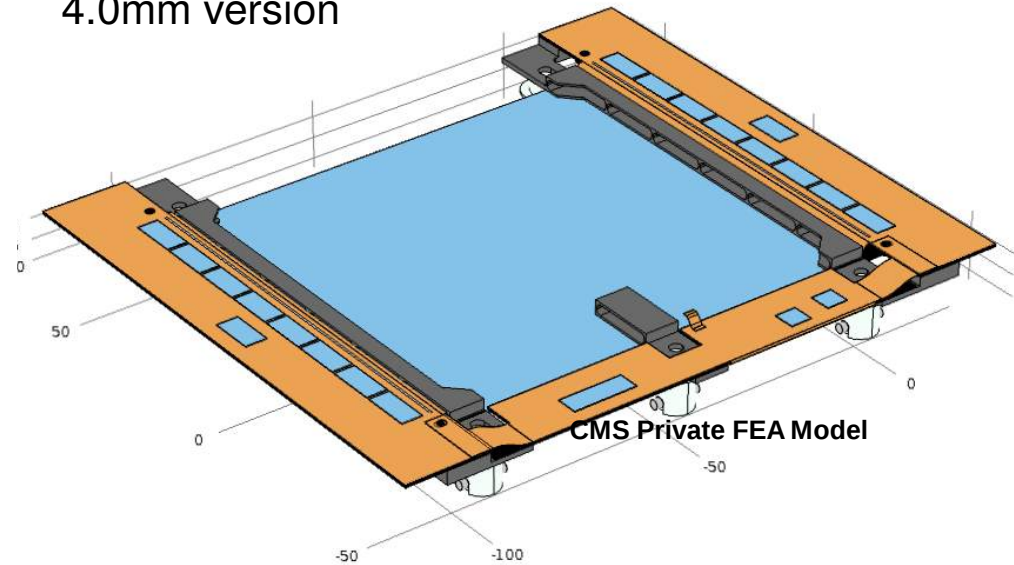


Two versions of the module for different sensor spacings

1.8mm version

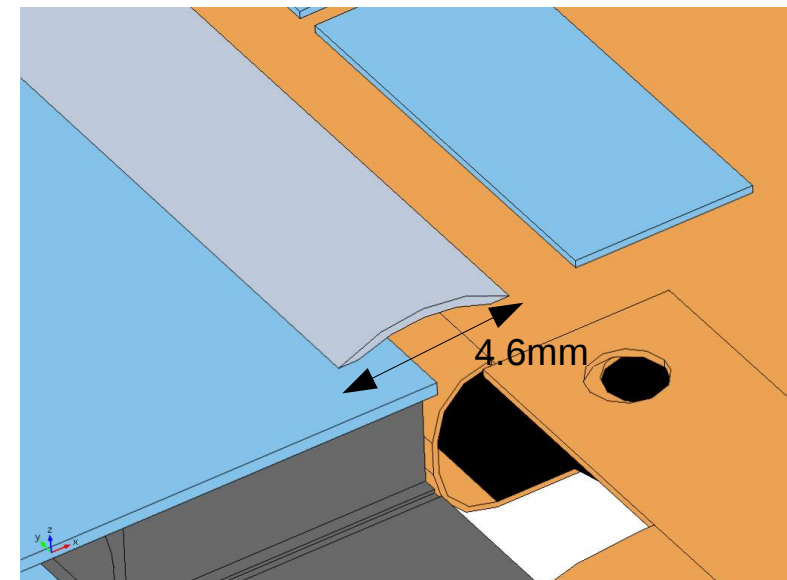


4.0mm version



Material	Component	Therm. conductivity [W/m/K]
Silicon	sensor, chips, opto dummy	140
Aluminum-Carbon Fiber (AICF)	sensor spacer, TB2S Ladder cooling contacts	$k_{xy} = 240 ; k_z = 120$
K13D2U carbon fiber	carbon fiber stiffener	$k_{xy} = 250 ; k_z = 0.5$
PCB-Material	service hybrid, read-out hybrid	50
Stainless steel	cooling pipes	12
Aluminum alloy	TEDD cooling inserts	130

- All **glue layers** are simulated with
 - the COMSOL function “Thin Layer”
 - a thickness of 20 μ m
 - a thermal conductivity of 0.2 W/m/K
- For the **wire bonds** an effective model is created
 - length 4.6mm, thickness 250 μ m
 - thermal conductivity 5 W/m/K
- The model corresponds to 1016 aluminium wire bonds
 - diameter per wire bond 12.5 μ m



- Temperature difference between CO₂ and inner cooling pipe

$$\Delta T_{\text{pipe}} \approx \frac{P}{h \cdot A_{\text{contact}}} = \frac{1.1 \text{ W}}{5000 \text{ W/m}^2/\text{K} \cdot \pi \cdot 2.2 \text{ mm} \cdot 8 \text{ mm}} = 4.0 \text{ K}$$

- Temperature gradient across cooling block:

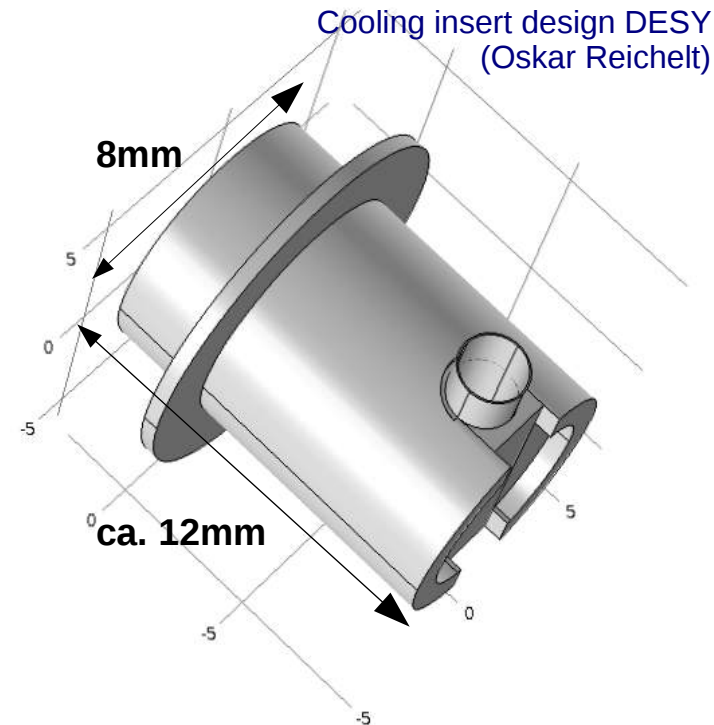
$$\Delta T_{\text{block}} \approx \frac{P \cdot h_z}{\lambda \cdot A_{\text{x-section}}} = \frac{1.1 \text{ W} \cdot 10 \text{ mm}}{240 \text{ W/m/K} \cdot \pi (4 \text{ mm})^2} = 0.9 \text{ K}$$

- Overall temperature difference, ΔT_{glue} is not negligible:

$$\Delta T_{\text{overall}} \approx \Delta T_{\text{block}} + \Delta T_{\text{pipe}} + \Delta T_{\text{glue}}$$

- Simulated $\Delta T_{\text{overall}}$ is

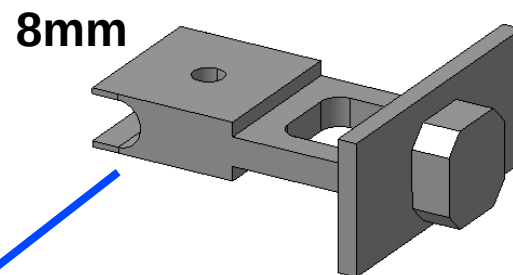
- 5.0K without glue layers
- 7.9K with 20 μm glue layers (0.2 W/m/K)



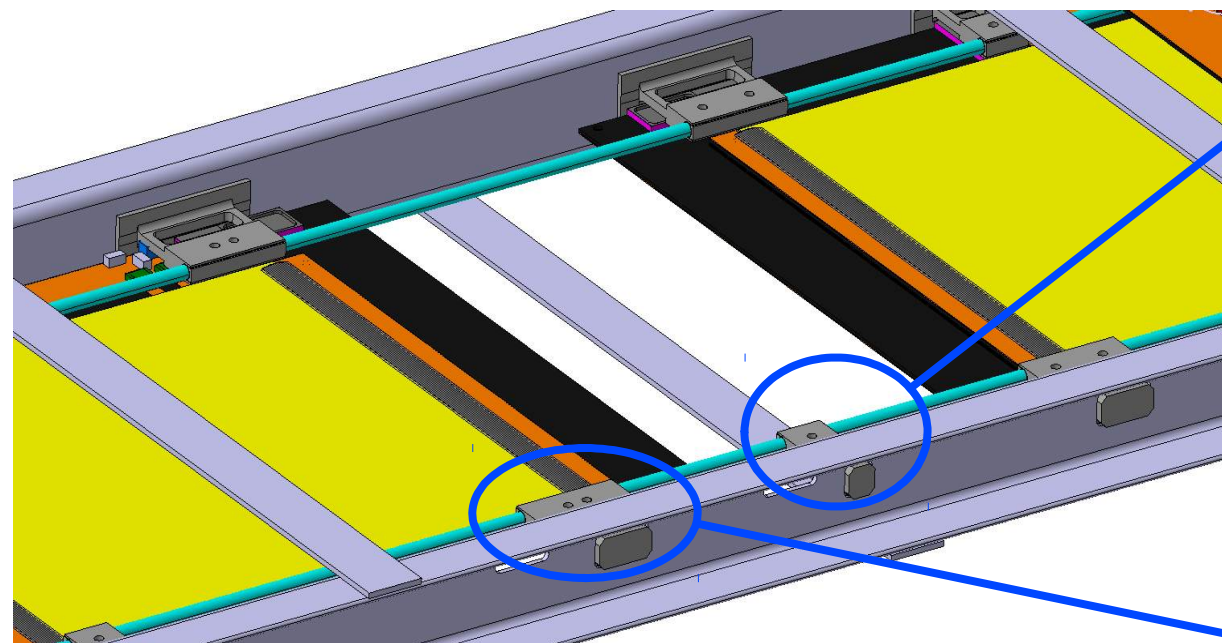
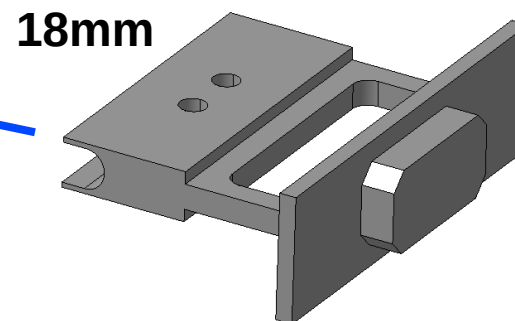
- 2S 1.8mm modules are mounted in the TB2S Ladder structure
- Modules are mounted back-to-back onto the AICF (?) cooling contacts

- cooling support for the 5th thermal contact
- cooling pipe contact length 8mm

2S 1.8mm Modules on the ladder



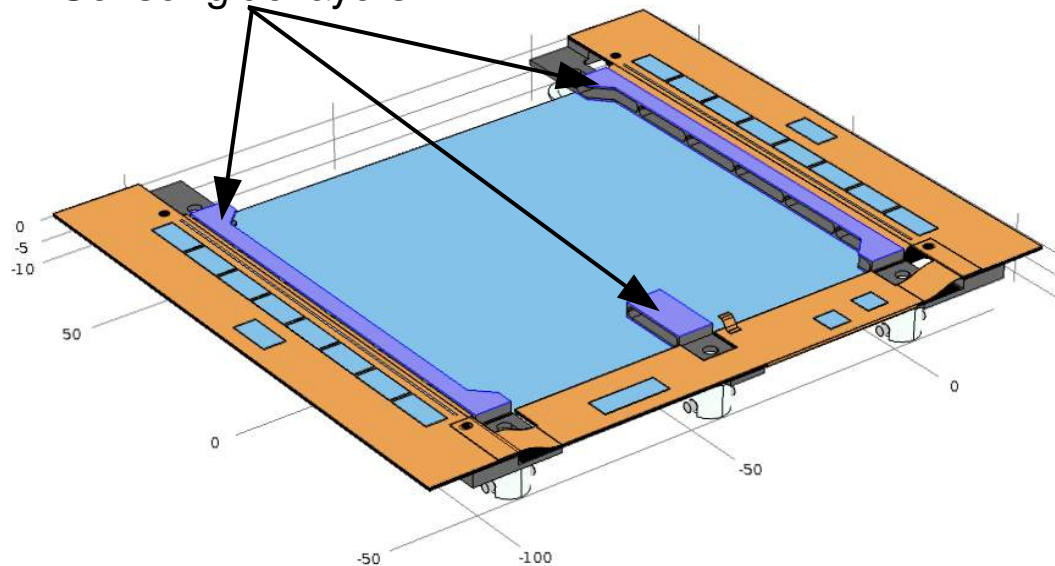
- cooling support for the four corners of the 2S modules
- cooling pipe contact length 18mm
- heat input on top and bottom side from two modules mounted to the cooling contact



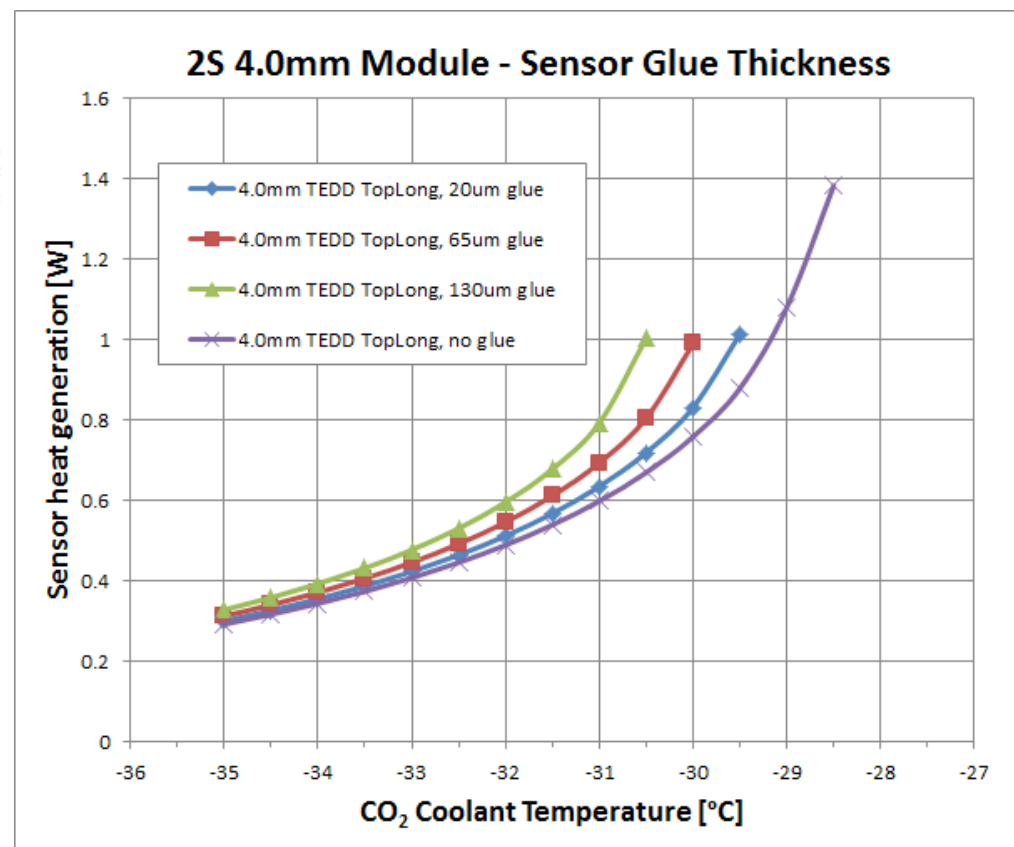
Pierre Rose

→ The factor 2 of heat input of two modules has to be taken into account in the FE simulation

Sensor glue layers

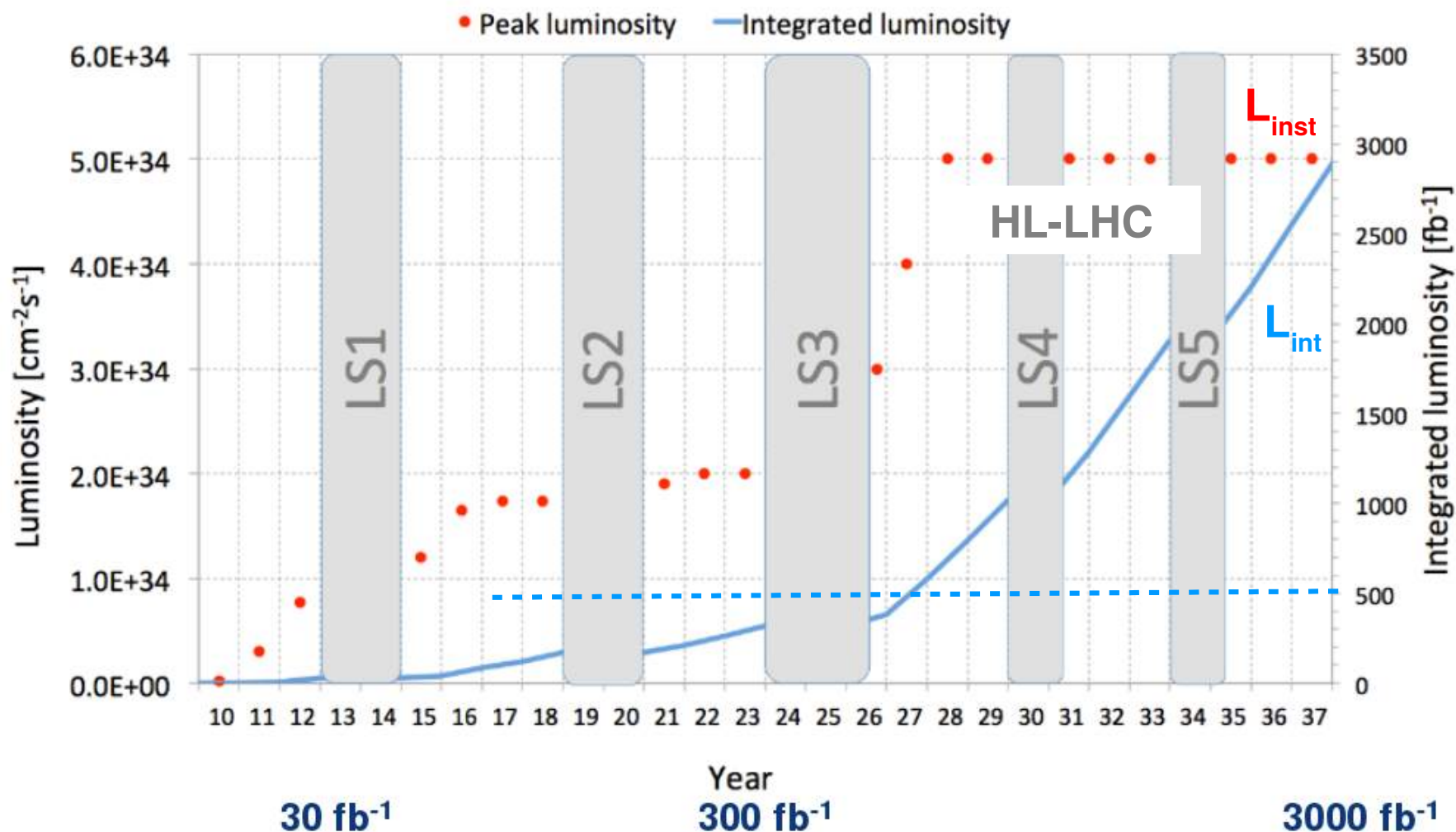


- Glue layer thicknesses of $0\mu\text{m}$, $20\mu\text{m}$, $65\mu\text{m}$, and $130\mu\text{m}$ are simulated
- Thermal conductivity 0.2 W/m/K
- Thermal runaway at 1-2K lower temperatures $130\mu\text{m}$ thick glue layer



- High-luminosity LHC (HL-LHC) will operate
 - at an instantaneous luminosity of $5 \cdot 10^{34} \text{ cm}^{-2}\text{s}^{-1}$ (pileup up to **200**)
 - deliver an integrated luminosity of **3000fb⁻¹**
- Current CMS Outer Tracker is designed to operate up to **500fb⁻¹** and pileup of **50**

Plot from Frederick Bordry (June 2015)



- backside heat inputs into cooling blocks have to be applied diagonally due to module arrangement in the TB2S Ladder

- no additional heat input necessary for the 5th cooling contact

→ can be implemented with COMSOL functions

