

Thermosiphon pre-design

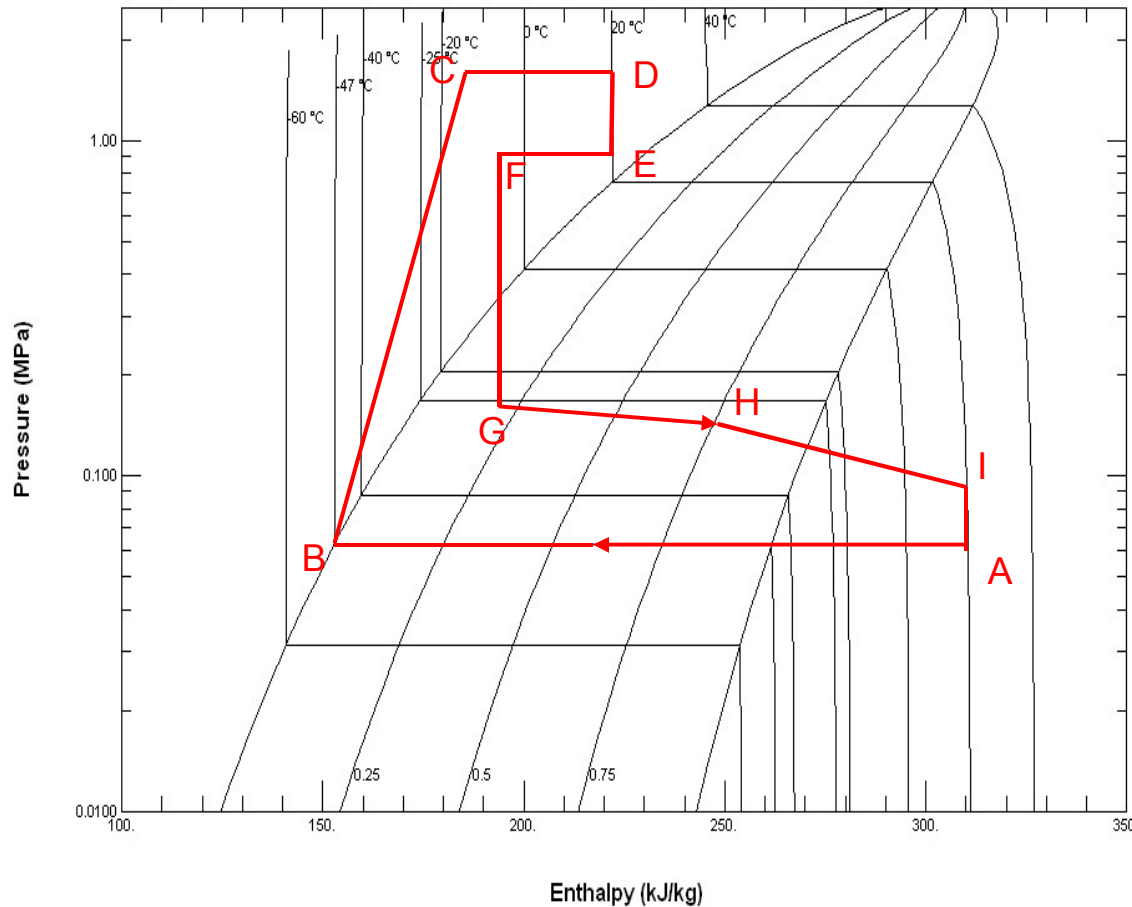
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Gravity-driven Cooling Concept

- Natural circulation of the fluid:
 - Condensation temperature/pressure must lower than evaporation temperature/pressure
- No working components in the main circuit
 - Less probability of occurrence of leaks
 - No significant vibrations on the system
 - Low maintenance operations
- Access to refrigeration units at the surface
- No limit for the Evaporation Temperature:
 - Suction pressure of the compressors is not a limit anymore
-> possibility of reaching lower evaporation temperatures

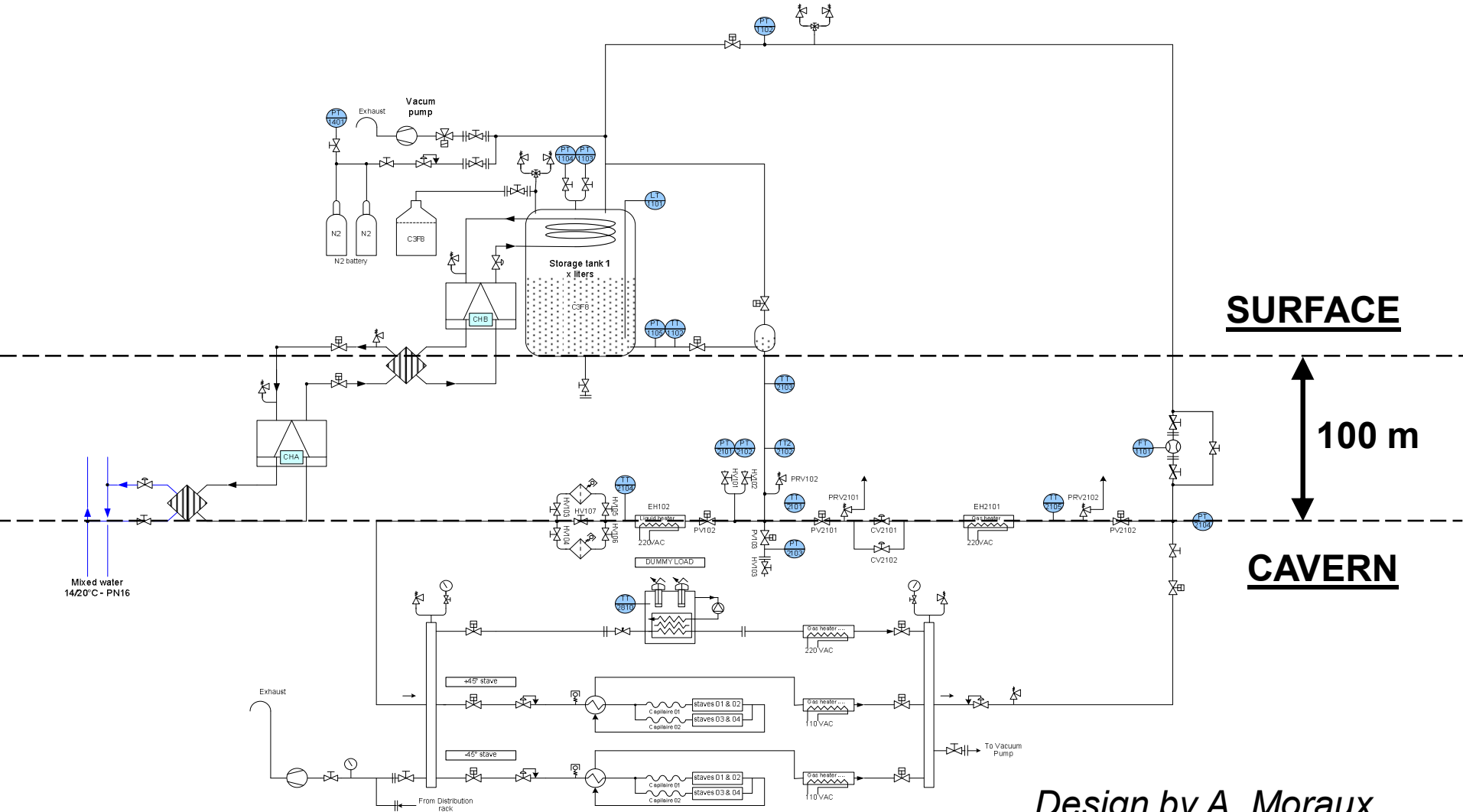
Preliminary design and parameters

- Thermodynamic Cycle Example for C3F8 with Sub-Cooling



- A-B : Condensation
- B-C : Hydrostatic dP
- C-D : Heat Exchanger
- D-E : Pressure Regulation
- E-F : Sub-Cooling
- F-G : Capillary Expansion
- G-H : Evaporation
- H-I : Heater
- I-A : Back-Pressure regulation

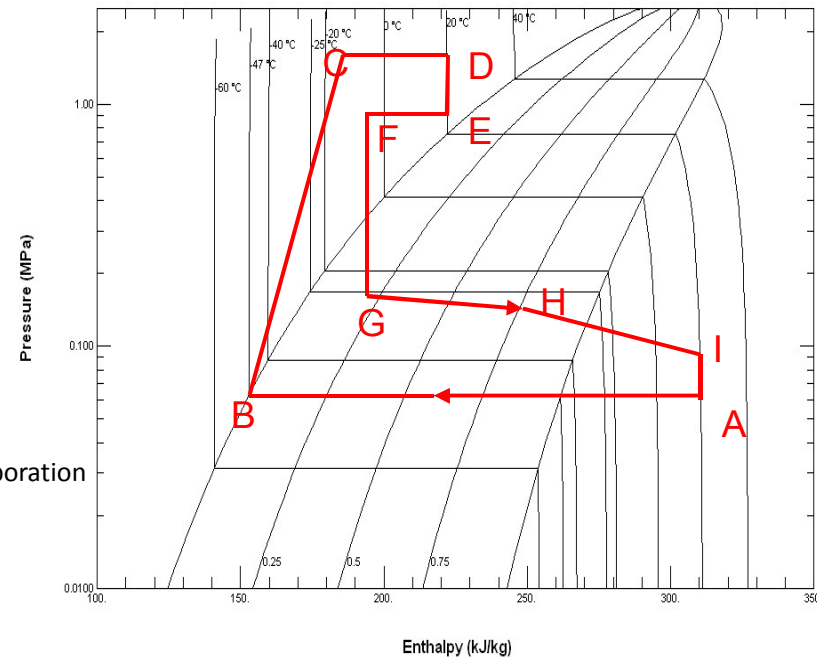
Preliminary design and parameters



Design by A. Moraux

Preliminary design and parameters

- **A-B : Condensation** – $dh = h(\text{at } T_{\text{condensation}}) - h(\text{at } T_{\text{return lines}})$
 - Definition of the Condensation Power
- **B-C : Hydrostatic** – $dP > P_{\text{transfer lines}}(T_{\text{transfer lines}}) - P_{\text{return lines}}(T_{\text{return lines}})$
 - Definition of minimum height
- **C-D : Heat Exchanger** – $dh = h(\text{at } T_{\text{in}}) - h(T_{\text{supply lines}})$
 - Definition of HX power
- **D-E : Pressure Regulation** – Pressure Regulator Cv
 - Regulation of mass flow
- **E-F : Sub-Cooling** – $dh = h(T_{\text{evap}} \text{ at } T_{\text{cond}}) - h(X_{\text{in}})$
 - Definition of HX Power for sub-cooling
- **F-G : Capillary pressure drop** – $dP = P_{\text{pressure regulator}} - P_{\text{evaporation}}$
 - Definition of capillary length
- **G-H : Evaporation** – dP_{stave}
 - Definition of dP at the stave
- **H-I : Heater** – $dh = h(X_{\text{out}}) - h(T_{\text{transfer lines}})$
 - Definition of Heater power
- **I-A : Back-Pressure regulation** – Pressure Regulator Cv
 - Regulation of Evaporation Temperature



Preliminary design and parameters

- Application example for the IBL detector and Distribution Racks at USA 15:
 - Required mass flow = 36g/s
 - Evaporation temperature = -40C
 - Supply and return lines Temperature = 20C
 - Altitude difference \approx 100m
- Definition of the main components:
 - Minimum height O.K.: 16 bar > dP ($P_{\text{cond}} = 7.57$ bar ; $P_{\text{evap}} = 0.87$ bar)
 - liquid line = DN25 -> dP= 16 bar ; Vapor line = DN50 -> dP = 90 mbar
 - Condensation Temperature < -42.5 C ($P_{\text{cond}} < 0.87$ bar – 90mbar)
 - Condenser Power > 5.94kW ($dh = 165$ kJ/kg; $m = 36$ g/s)
 - Heat Exchanger Power > 1.44kW ($dh = 40$ kJ/kg; $m = 36$ g/s)