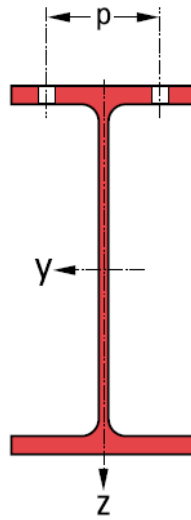
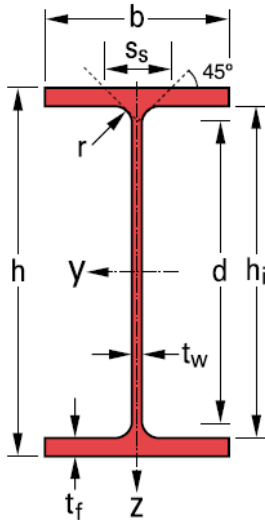


Verification of elements according to the EN 1993-1-1.

Color code: **Input - Blue**; **Result - green**; **Parameters for drawing - red**

1- Input

1.1 - Cross-section properties



$$h := 203.2 \text{ mm}$$

$$b := 266.7 \text{ mm}$$

$$t_w := 6.35 \text{ mm}$$

$$t_f := 9.5 \text{ mm}$$

$$r := 20 \text{ mm}$$

Area of section

$$A := 2 \cdot t_f \cdot b + (h - 2 \cdot t_f) \cdot t_w + (4 - \pi) \cdot r^2 = 65.803 \text{ cm}^2$$

Shear area parallel to web

$$A_{vz} := A - 2 \cdot b \cdot t_f + (t_w + 2 \cdot r) \cdot t_f = 19.534 \text{ cm}^2$$

Shear area parallel to flange

$$A_{vy} := A - (h - 2 \cdot t_f) \cdot t_w = 54.107 \text{ cm}^2$$

Depth of straight portion of web

$$d := h - 2 \cdot t_f - 2 \cdot r = 144.2 \text{ mm}$$

Inner depth between flanges

$$h_i := h - 2 \cdot t_f = 184.2 \text{ mm}$$

Depth of the web

$$h_w := h - 2 \cdot t_f = 184.2 \text{ mm}$$

Inertias

$$I_y := \frac{1}{12} \cdot (b \cdot h^3 - (b - t_w) \cdot (h - 2 \cdot t_f)^3) + 0.03 \cdot r^4 + 0.2146 \cdot r^2 \cdot (h - 2 \cdot t_f - 0.4468 \cdot r)^2 = 5351.779 \text{ cm}^4$$

$$I_z := \frac{1}{12} \cdot (2 \cdot t_f \cdot b^3 + (h - 2 \cdot t_f) \cdot t_w^3) + 0.03 \cdot r^4 + 0.2146 \cdot r^2 \cdot (t_w - 0.4468 \cdot r)^2 = 3004.526 \text{ cm}^4$$

Radius of giration

$$i_y := \sqrt{\frac{I_y}{A}} = 9.018 \text{ cm}$$

$$i_z := \sqrt{\frac{I_z}{A}} = 6.757 \text{ cm}$$

Polar radius of gyration

$$i_o := \sqrt{\frac{(I_y + I_z)}{A}} = 11.269 \text{ cm}$$

Torsion constant

$$I_t := \frac{2}{3} \cdot (b - 0.63 \cdot t_f) \cdot t_f^3 + \frac{1}{3} \cdot (h - 2 \cdot t_f) \cdot t_w^3 + 2 \cdot \left(\frac{t_w}{t_f}\right) \cdot \left(0.145 + 0.1 \cdot \frac{r}{t_f}\right) \cdot \left(\frac{\left(r + \frac{t_w}{2}\right)^2 + (r + t_f)^2 - r^2}{2 \cdot r + t_f}\right)^4 = 24.625 \text{ cm}^4$$

Warping constant

$$I_w := \frac{t_f \cdot b^3}{24} \cdot (h - t_f)^2 = (2.817 \cdot 10^5) \text{ cm}^6$$

Length of stiff bearing

$$s_s := t_w + 2 \cdot t_f + (4 - 2 \cdot \sqrt{2}) \cdot r = 48.781 \text{ mm}$$

Elastic section modulus

$$W_{el.y} := \frac{2 \cdot I_y}{h} = 526.75 \text{ cm}^3$$

$$W_{el.z} := \frac{2 \cdot I_z}{b} = 225.311 \text{ cm}^3$$

Plastic section modulus

$$W_{pl.y} := \frac{t_w \cdot h^2}{4} + (b - t_w) \cdot (h - t_f) \cdot t_f + \frac{4 - \pi}{2} \cdot r^2 \cdot (h - 2 \cdot t_f) + \frac{3 \cdot \pi - 10}{3} \cdot r^3 = 574.721 \text{ cm}^3$$

$$W_{pl.z} := \frac{b^2 \cdot t_f}{2} + \frac{h - 2 \cdot t_f}{4} \cdot t_w^2 + r^3 \cdot \left(\frac{10}{3} - \pi\right) + \left(2 - \frac{\pi}{2}\right) \cdot t_w \cdot r^2 = 342.343 \text{ cm}^3$$

1.2 - Material properties

Yield strength

$$f_y := 340 \text{ MPa}$$

Ultimate strength

$$f_u := 445 \text{ MPa}$$

Modulus of elasticity

$$E := 210000 \text{ MPa}$$

Poisson's ratio

$$\nu := 0.3$$

Shear modulus

$$G := \frac{E}{2 \cdot (1 + \nu)} = 80769.231 \text{ MPa}$$

Shear modulus (for computation)

$$G := 81000 \text{ MPa}$$

$$\varepsilon := \sqrt{\frac{235 \text{ MPa}}{f_y}} = 0.831$$

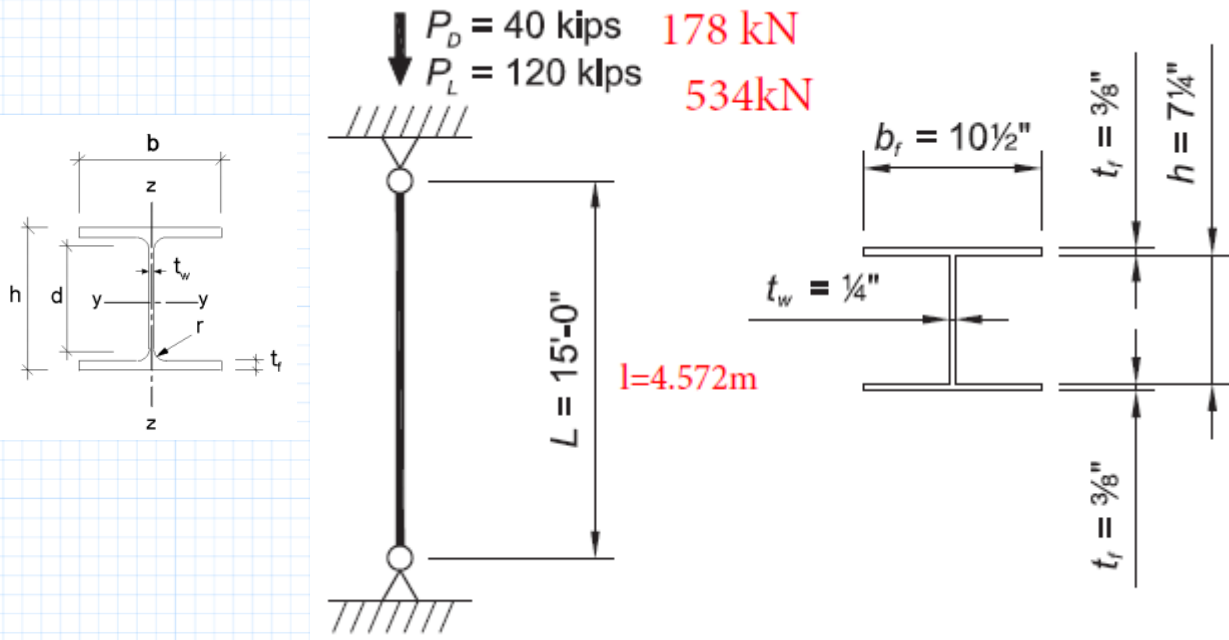
1.3 - Partial factors (6.1)

$$\gamma_{M0} := 1.0$$

$$\gamma_{M1} := 1.0$$

$$\gamma_{M2} := 1.25$$

2- Load conditions



2.1- Shear

$$V_{y.Ed} := 0 \text{ kN}$$

$$V_{z.Ed} := 0 \text{ kN}$$

2.2- Bending moment

$$M_{y.Ed} := 0 \text{ kN} \cdot \text{m}$$

$$M_{z.Ed} := 0 \text{ kN} \cdot \text{m}$$

2.3- Axial load

Loads corresponding to the LRFD & ASD load combinations:

$$N_{Ed} := \begin{bmatrix} 1067 \text{ kN} \\ 712 \text{ kN} \end{bmatrix}$$

3- Cross-section classification

3.1- Web classification

$$c := d = 144.2 \text{ mm} \quad t := t_w = 6.35 \text{ mm}$$

$$\begin{aligned}
 \text{Class}_{web.bending} &:= \left\| \begin{array}{l} \text{if } \frac{c}{t} \leq 72 \cdot \varepsilon \\ \quad \left\| \begin{array}{l} 1 \\ \text{else if } \frac{c}{t} \leq 83 \cdot \varepsilon \\ \quad \left\| \begin{array}{l} 2 \\ \text{else if } \frac{c}{t} \leq 124 \cdot \varepsilon \\ \quad \left\| \begin{array}{l} 3 \\ \text{else} \\ \quad \left\| \begin{array}{l} 4 \end{array} \right. \end{array} \right. \end{array} \right. \end{array} \right. \\
 \end{array} \right\| = 1
 \end{aligned}
 \quad
 \begin{aligned}
 \text{Class}_{web.compression} &:= \left\| \begin{array}{l} \text{if } \frac{c}{t} \leq 33 \cdot \varepsilon \\ \quad \left\| \begin{array}{l} 1 \\ \text{else if } \frac{c}{t} \leq 38 \cdot \varepsilon \\ \quad \left\| \begin{array}{l} 2 \\ \text{else if } \frac{c}{t} \leq 42 \cdot \varepsilon \\ \quad \left\| \begin{array}{l} 3 \\ \text{else} \\ \quad \left\| \begin{array}{l} 4 \end{array} \right. \end{array} \right. \end{array} \right. \end{array} \right. \\
 \end{array} \right\| = 1
 \end{aligned}$$

The worst case scenario is to consider that the entire web is in compression and therefore:

$$\text{Class}_{web} := \text{Class}_{web.compression} = 1$$

3.2- Flange classification

$$c := \frac{b}{2} - \frac{t_w}{2} - r = 110.175 \text{ mm} \quad t := t_f = 9.5 \text{ mm}$$

$$\begin{aligned}
 \text{Class}_{flange.compression} &:= \left\| \begin{array}{l} \text{if } \frac{c}{t} \leq 9 \cdot \varepsilon \\ \quad \left\| \begin{array}{l} 1 \\ \text{else if } \frac{c}{t} \leq 10 \cdot \varepsilon \\ \quad \left\| \begin{array}{l} 2 \\ \text{else if } \frac{c}{t} \leq 14 \cdot \varepsilon \\ \quad \left\| \begin{array}{l} 3 \\ \text{else} \\ \quad \left\| \begin{array}{l} 4 \end{array} \right. \end{array} \right. \end{array} \right. \end{array} \right. \\
 \end{array} \right\| = 3
 \end{aligned}$$

$$\text{Class}_{flange} := \text{Class}_{flange.compression} = 3$$

3.3- Cross-section classification

The worst case scenario is to consider the entire cross-section in compression and therefore:

$$\text{Class} := \max(\text{Class}_{web}, \text{Class}_{flange}) = 3$$

4- RESISTANCE OF CROSS-SECTION (6.2)

4.1- Tension resistance (6.2.3)

$$N_{t.Rd} := \frac{A \cdot f_y}{\gamma_{M0}} = 2237.313 \text{ kN}$$

4.2- Compression resistance (6.2.4)

$$N_{c.Rd} := \begin{cases} \text{if } Class = 4 \\ \quad \left| \frac{A \cdot f_y}{\gamma_{M0}} \right| \\ \text{else} \\ \quad \left| \text{"N.A."} \right| \end{cases} = 2237.313 \text{ kN}$$

4.3- Bending moment resistance (6.2.5)

$$M_{c.y.Rd} := \begin{cases} \text{if } Class = 1 \vee Class = 2 \\ \quad \left| \frac{W_{pl.y} \cdot f_y}{\gamma_{M0}} \right| \\ \text{else if } Class = 3 \\ \quad \left| \frac{W_{el.y} \cdot f_y}{\gamma_{M0}} \right| \\ \text{else} \\ \quad \left| \text{"N.A."} \right| \end{cases} = 179.095 \text{ kN} \cdot \text{m}$$

$$M_{c.z.Rd} := \begin{cases} \text{if } Class = 1 \vee Class = 2 \\ \quad \left| \frac{W_{pl.z} \cdot f_y}{\gamma_{M0}} \right| \\ \text{else if } Class = 3 \\ \quad \left| \frac{W_{el.z} \cdot f_y}{\gamma_{M0}} \right| \\ \text{else} \\ \quad \left| \text{"N.A."} \right| \end{cases} = 76.606 \text{ kN} \cdot \text{m}$$

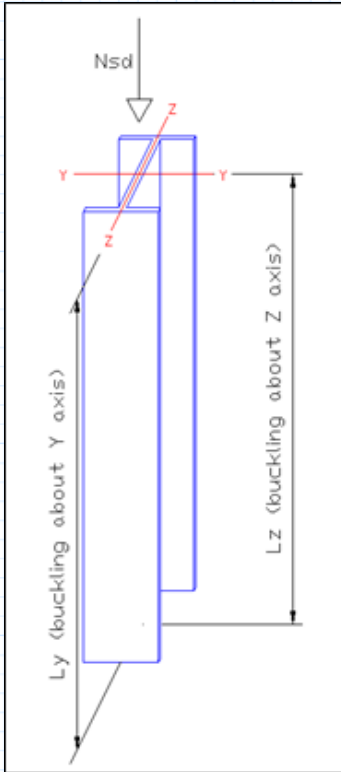
4.4- Shear resistance (6.2.6)

$$V_{pl.y.Rd} := \frac{A_{vy} \cdot \left(\frac{f_y}{\sqrt{3}} \right)}{\gamma_{M0}} = 1062.108 \text{ kN}$$

$$V_{pl.z.Rd} := \frac{A_{vz} \cdot \left(\frac{f_y}{\sqrt{3}} \right)}{\gamma_{M0}} = 383.442 \text{ kN}$$

5- Buckling resistance of members (6.3)

5.1- Uniform members in compression (6.3.1)



Buckled shape of column shown by dashed line					
Theoretical K value	0.5	0.7	1.0	1.0	2.0
Recommended design value K	0.65	0.80	1.2	1.0	2.0

5.1.1- Buckling input

Column unsupported length

$$L_y := 4.572 \text{ m}$$

Column effective length factor

$$k_y := 1$$

Column unsupported length

$$L_z := 4.572 \text{ m}$$

Column effective length factor

$$k_z := 1$$

5.1.2- Buckling resistance (6.3.1)

- Buckling about z-z axis - weak axis governs*

Column buckling length

$$L_{cr,z} := L_z \cdot k_z = 4.572 \text{ m}$$

Flexural elastic critical force (Euler)

$$N_{cr,z,F} := \frac{\pi^2 \cdot E \cdot I_z}{L_{cr,z}^2} = 2979.082 \text{ kN}$$

$$\sigma_{cr,z,F} := \frac{N_{cr,z,F}}{A} = 452.725 \text{ MPa}$$

Note: Open section shapes may additionally experience buckling by torsion or torsion combined with flexural buckling. These modes are more relevant in U and L shapes. As generally the rotation of the member is avoided by the boundary conditions, the flexural buckling is the limiting effect.

Elastic torsional buckling force
$$N_{cr.z.T} := \frac{1}{i_o^2} \cdot \left(G \cdot I_t + \frac{\pi^2 \cdot E \cdot I_w}{L_{cr.z}^2} \right) = 3770.515 \text{ kN}$$

Elastic torsional-flexural buckling force

$$N_{cr.z.TF} := \frac{1}{2} \cdot \left((N_{cr.z.F} + N_{cr.z.T}) - \sqrt{(N_{cr.z.F} + N_{cr.z.T})^2 - 4 \cdot N_{cr.z.F} \cdot N_{cr.z.T}} \right) = 2979.082 \text{ kN}$$

Imperfection factor

$$\alpha_z = 0.49$$

0 - for beam without torsion buckling modes,
 1 - for torsion buckling mode

$$torsion_{mode} := 1$$

Elastic critical force

$$N_{cr.z} := \begin{cases} \text{if } torsion_{mode} = 0 \\ \quad N_{cr.z.F} \\ \text{else} \\ \quad \min(N_{cr.z.F}, N_{cr.z.T}, N_{cr.z.TF}) \end{cases} = 2979.082 \text{ kN}$$

Non-dimensional slenderness

$$\lambda_z := \sqrt{\frac{A \cdot f_y}{N_{cr.z}}} = 0.867$$

Reduction factor

$$\Phi_z := 0.5 \cdot \left(1 + \alpha_z \cdot (\lambda_z - 0.2) + \lambda_z^2 \right) = 1.039$$

$$\chi_z := \min \left(\frac{1}{\Phi_z + \sqrt{\Phi_z^2 - \lambda_z^2}}, 1 \right) = 0.62$$

Design buckling resistance

$$N_{b.z.Rd} := \begin{cases} \text{if } Class = 4 \\ \quad \text{"N.A."} \\ \text{else} \\ \quad \frac{\chi_z \cdot A \cdot f_y}{\gamma_{M1}} \end{cases} = 1388.203 \text{ kN}$$