

Detector cooling R&D with multi-micro-channels: *lessons learned & open issues*

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Presentation outline

Definition and advantages of micro-channel cooling

 \rightarrow Multi-micro-channel detector cooling R&D with Silicon-substrate cold plates:

- first generation channel layouts, design necessities and experimental setup
- **Problem**: unknown channel contamination after experimental testing
- **Explanation**: hydrodynamic erosion in micro-channels
- **Solution**: Proposal of a parametrical map for safe cold plate operation
- Conclusion & **lessons learned**
- Outlook & **open issues**

Definition and advantages of micro-channel cooling

micro-channel cooling =

- Network of submillimeter-sized channels embedded into a thin, planar substrate to form a micro-fluidic heat exchanger within which a certain coolant is circulated (cold plate)
- many channel materials, manufacturing techniques, geometries and working fluids possible
	- **→** Herein discussed:
	- **Silicon** (Si) **substrate cold plates** manufactured with micro-electromechanical systems (MEMS) techniques
	- $\,$ **Carbon dioxide** (CO $_2$) as working fluid
	- Examples from EP-DT-FS + collaborating institutes (\neq LHCb devices)

Advantages of Si-substrate cold plates for the detector cooling community =

- reduction of material
- direct planar contact with the sensor surface
- minimized thermal resistances
- reduced mismatch in thermal expansion coefficients
- large heat transfer surface involved
- homogeneous temperature distribution across sensor surface

Generic channel layout for boiling flows

Multi-micro-channel cold plates produced with MEMS techniques: etching

Generic channel layout for boiling flows

Multi-micro-channel cold plates produced with MEMS techniques: bonding: Si-Si / Si-glass

A-A: micro-channels after Si-glass bonding

B-B: hydraulic inter-connection: soldered variant

Example: Resulting R&D device

Actual channel layouts for boiling flows (1st generation)

ATLAS

overall dimension 45 x 20 mm year: ∼ 2012 intended for: ATLAS FEI4 institutes involved: CERN, University of Manchester, CSEM, EPFL Cmi

FRAME4

overall dimension: 85 x 10 mm year: ∼ 2012 intended for: NA62 GTK institutes involved: CERN, CSEM, EPFL Cmi

REFLECS

overall dimensions: 20 x 40 mm, 20 x 80 mm, 20 x 100 mm year: ∼ 2014 intended for: generic 20 x 20 mm chips institutes involved: CERN, FBK, LPNHE

Detailed channel design for boiling flows

↔ inappropriate restriction design allowing back flow and flow oscillations (video)

Appropriate inlet restrictions/capillaries:

1) Crucial requirement: for a correct functionality without flow instabilities during two-phase flow and flow boiling which otherwise would induce severe pressure fluctuations and local hot spots

2) Preferred side effect: Fluid-dynamic backward-facing step can trigger boiling onset due to recirculation areas causing low pressure zones in the near-wall region

← At this stage a complete analytical fluid dynamical evaluation was omitted and the technological manufacturing aspects were prioritised

Experimental setup for detector cooling R&D with mini- and micro-scale carbon dioxide evaporators at CERN

- **=** Centerpiece of our studies, described in <http://cds.cern.ch/record/2748428>
- **=** Extended version for flow visualisation: + Photron Fastcam Mini AX + software

High-speed camera with LED lights

REFLECS micro-channel cold plate mounted inside vacuum vessel

→ REFLECS image analysis

2 x 2 cm² metallized silicon heater glued onto the silicon side simulating a chip footprint

cross-section 200 μ m x 120 μ m, length ~ 55 - 80 mm channel: inlet-restriction: cross-section 60 um x 120 um, length 6 mm

ANTIQUES COM

inlet with backward-facing step 13600 fps (video)

All Control

►

main channel, detail 102000 fps (video)

cross-section 200 μ m x 120 μ m, length ~ 55 - 80 mm - channel: - inlet-restriction: cross-section 60 μ m x 120 μ m, length 6 mm

main channel, detail

Problem:

Some contamination was found inside the channels which appears as back- and foreground in the images and videos **→** glass and Si side (and probably walls)

200 μm

Examples: inlet, main channel, bend

- High-speed videos with ∼ 3000 frames or more
- Applying certain thresholds and summation of all pre-processed images reveals stationary contamination

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At that point remembering that the FRAME4 device tested in 2012 (G. Romagnoli) and sent to Twente University for further testing (A. Koutoulaki) in 2016 was reportedly "dirty" upon arrival → possibly the same contamination?

→ delayed device analysis on FRAME4 device

→ analysis reveals severe "contamination" of the channel internals after testing with fluid flow

"**before**" = untested device

"**after**" = tested device (note: no flow, no fluid inside)

outlet manifold **induction** main channels main channels inlet with backward-facing step

→ further analysis reveals patterns similar to pitting damage found in "literature"

→ CAVITATION induced local, crater-like EROSION may be the cause of the "contamination"

→ further analysis to confirm erosion damage

- **•** Confirmation of eroded "craters" required
- non-trivial due to small dimensions, layered and indirect optical access (outer glass, inner glass, inner Si) and no possibility to remove the glass

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How can fluid flow damage (erode) a surface in relatively short time ?

- \rightarrow hydrodynamic cavitation damage
- \rightarrow flash boiling/ flashing damage
- \rightarrow fast and highly erosive (solid) particles in the flow

Hydro-mechanic machinery damaged by hydrodynamic cavitation

Cavitation vs. flashing damage

- Both are the result of related, but rather different thermo-dynamical processes
- at a fluid flow restriction (control valves, orifice, venturi tube etc.) the pressure drops suddenly and may drop below the vapour pressure of the liquid and liquid-vapour phase change occurs: formation of bubbles
- downstream, the flow passage expands and **partial pressure recovery** occurs as the flow velocity decreases
- Then ...

cavitation occurs …

- if the local pressure recovers to above P_{vapor}
- the vapour bubbles will suddenly collapse and implode
- the implosion induces high-speed fluid micro-jets, shock waves and noise

Flash boiling/ flashing occurs …

- if the local pressure does not recover to above P_{vanour}
- the vapour bubbles remain part of two-phase flow
- Fast bubbles may impinge on surface and create damage

Bubble collapse: schematic and real life [3]

→ Cavitation and its related damage is being studied and can be quantified, flashing not so much

Cavitation – a very short overview

cavitation – studied

- external flow (propellers, impellers, projectiles/submarines or hydro-foils)
- internal flow: e.g. studies on orifices and venturis
- the pressure evolution is monitored before/after the restrictive device: $P_{upstream}$ and $P_{downstream}$
- pressure ports are implemented accordingly
- also for micro-scale devices

cavitation – quantified

With those pressure measurements one can calculate the

- Cavitation number
	- the smaller the more aggressive the form of cavitation
	- experimentally the exit pressure is lowered to lower σ until flow choking occurs
	- Mostly under constant fluid properties
	- downstream pressure P_2 : ideally measured directly at throat exit, but various locations in literature (10s x D_b downstream)

$$
\sigma = \frac{P_2 - P_v}{\frac{1}{2}\rho v^2}
$$

• Cavitation index

and relate it to an existing "safety range"

Cavitation induced by propeller [4]

Cavitation induced inside venturi tube [5]

Safety range of the cavitation index [6]

Cavitation – difference between EP-DT devices and devices studied in literature

Considering only the inlet geometry of *one* restriction

- Clearly no orifice or venturi geometry
- No pressure ports before and after restriction to estimate the cavitation potential

+ In addition

- multi-channels instead of single-channel
- No inlet plenum *per se*
- Unique inlet geometry, hence even less clear picture of upstream pressure
	- + Furthermore, for the same device
	- length of the multiple restrictions varies (FRAME4)
	- outlet geometry varies among the multiple restrictions (FRAME4)
	- directional change varies among the multiple restrictions (FRAME4, REFLECS, ATLAS)

- **= rather complex layout to study any fundamental flow behaviour**
- **= note: technological manufacturing aspects of earliest final cold plates was prioritised**
- **→ Cavitation number cannot be calculated in a meaningful way**
- **→ Other means to estimate the cavitation potential required**

→ by means of the theoretical pressure evolution along one restrictive inlet section

→ analytical combination of flow contraction and expansion with capillary in between

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→ evaluation of theoretical bulk static pressure in relation to the local vapour pressure

- macro-scale textbook formulations for duct flow used
- actual complex inlet geometry simplified to simple textbook contraction
- sub-cooled single-phase flow at inlet
- no meta-stability effect accounted for
- development of vena contracta and flow separation at contraction
- isenthalpic pressure drop
- even distribution of the flow into n channels
- straight capillary; in reality curved

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Using very basic criterion: sudden ΔP below vapour pressure + recovery = possibility of formation and collapse of bubbles = 3 theoretical locations

- **1** sudden pressure reduction due to flow separation and vena contracta **→** bubble formation **→** collapse after flow reattachment
- **2** low system pressure in core flow at the end of the restriction **→** bubble formation **→** collapse after expansion
- **3** sudden very low pressure due to recirculation area at the walls **→** bubble formation **→** collapse upon re-entry into core flow
- **= theoretical danger zones**

A₂ **Estimation of the cavitation potential of our devices** A_1 A_2 A_3 A_4

- → the combination of *fluid properties* and *working parameters* (sub-cooling level, flow rate,
- T_{sat}) within a fixed geometry
	- influences the outcome of this analysis
	- however those were chosen rather randomly so far for the actual experiments (within reason)

20

10

 $\mathbf 0$

 -10

 -20

 -30

 0.1

 $[°C]$

 $T_{\rm sat}$

→ creation of a parametrical map

- in function of total flow rate and T_{sat}
- fixed sub-cooling level, fixed geometry
- example: REFLECS @ 0.15°C sub-cooling

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→ indication of 3 theoretical danger zones using basic criterion mentioned before (T → S)

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Note: no true flash boiling occurs: S-S-S-T-S-T

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← reverse-tuning the map by changing the geometrical inputs:

EXAMPLE: include actual flash boiling by reducing the main channel area = reduce ratio between restriction and main channel

(Similar outcome can be achieved with fine-tuning the sub-cooling level)

Solution:

Use the map as a preliminary design guide for future micro-channel cooling devices

= Actively seeking out or avoiding certain flow regimes

 A_1

subcooled single-phase

 $A₂$

 A_3

??? Only a very rough estimate due to multiple issues not easily foreseen without further experiments:

- Cavitation at the micro-scale differs to macro-scale (surface tension forces at the micro-scale may delay fluid rupture)
- actual cavitation inception (gaseous and vapourous cavitation, depends on many factors, not studied at all with $CO₂$ at the micro-scale)
- uneven flow distribution in multi-channels might affect the outcome
- the peculiar geometry of the devices not fully accounted for in approach so far most likely alters the real outcome
- meta-stable flow might alter outcome drastically (= ability of the fluid to sustain tension before rupture, although already under apt conditions to form vapour cavities)

→ Clearly some CFD simulations would be useful, but still tricky for bi-phase CO₂

!!! Further emphasising the general cavitation hypothesis in our case:

- low surface tension of CO₂: low surface tension fluids rupture more easily (\to cavitate more easily, meta-stability less likely)
- Other reports from literature of bubbles collapsing far downstream of restriction/expansion (vs. immediately)
- Comparison of parametrical map with visual results reveals non-cavitating and cavitating flow patterns from literature in rough accordance with the map

- …

→ Comparison of map with visual results and results from literature

Experimental parameters

- total flow rates: 0.1 and 0.3 g/s
- T_{sat} : +15 to -25°C, examples here +15 and -25°C
- No heat flux!
- **= 4 points (I, II, III, IV) in map ready for comparison**

II 0.1 g/s, -25 °C

Prediction:

single-phase flow adjoining danger zone 2 & 3 = cavitation potential at restriction outlet

Visual observation:

- Single phase flow alternating between cavitating and non-cavitating two-phase flows with "normal" bubbles
- Cavitating flows form single stationary super-cavity
- Higher surface tension at lower T_{sat} favours larger cavities

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I 0.1 g/s, +15 °C

Prediction: deep in single-phase flow regime

Visual observation:

- Not so deep in single-phase flow
- minor boiling observed, initiated due to heat leaks
- No cavitation flow pattern observed

III 0.3 g/s, +15 °C

Prediction:

Normal two-phase flow establishing in restriction adjoining danger zone 1 = cavitation potential at restriction inlet

Visual observation:

- non-cavitating bubbly flow with bubble bursts of cavitating nature already observed in restriction
- = periodical bubble shedding typical to cavitating flows
- bubble collapse observable

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Visual observation:

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IV 0.3 g/s, -25 °C

Prediction: danger zone 1 = cavitation potential at restriction inlet

Visual observation:

- Restrictions filled with cavitation bubbles or single cavities
- change into other regime not too frequent, as predicted
- Pulsating flow structure at the outlet
- bubble collapse and pressure waves observable
- Similar to report on cavitation in macro-venturi tube

[10]

Conclusion & lessons learned

= existing and future two-phase micro-channel cooling devices require an inlet restrictive inlet section to allow for a proper functionality without flow oscillations

= if not chosen carefully in tandem the geometry and flow parameters used so far induce cavitating flows and the associated erosion damage

= to avoid this destructive flow regime in hindsight for existing micro-channel geometries and in foresight for future microchannel geometries a parametrical map can be created, indicating the "safe" and "dangerous" flow parameters

= the true - *initially intended* - flash boiling regime can be sought out with reverse-tuning the channel geometry

= the approximate validity of this map could be confirmed by means of flow visualisation and comparison to examples from literature

→ in view of the new era of micro-channel detector cooling with 3D-printed light-weight realisations in metals or ceramics any potential pitting damage may prove much more disastrous, compared to those produced by MEMS techniques in monocrystalline silicon or glass

→ the proposed map may be used as guidance for designers and experimentalists, designing and investigating future multi-micro-channel cold plate layouts, tuning the geometry and the flow parameters for a safe and efficient operation

Outlook & open issues

→ acknowledging that more investigations are needed to understand the issue of cavitating CO² at the micro-scale

_ further systematic high-speed camera recordings with existing devices

- **_** direct confirmation by means of cavitation number: new device geometries with pressure ports
- **_** further investigation: including data with heat flux, ATLAS device analysis

Thank you for your attention!

Quick-References

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