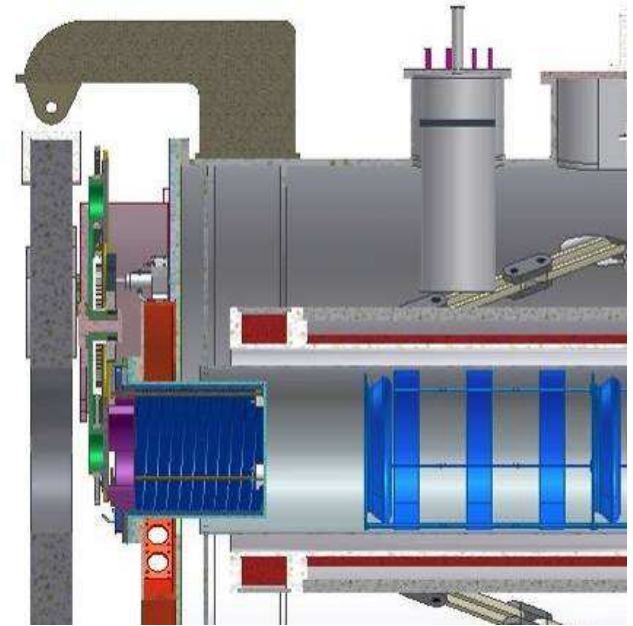
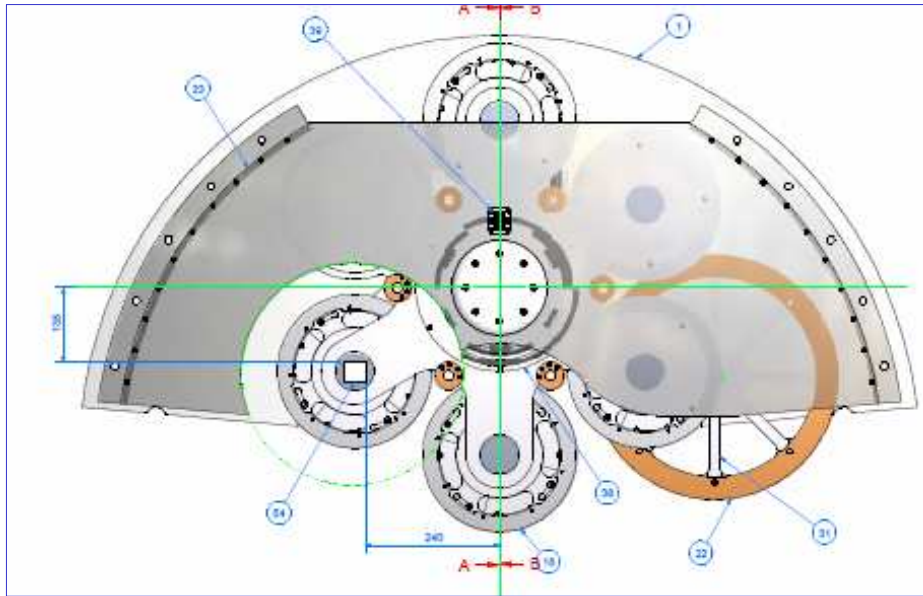


THE DIFFUSER

- **The Progress**
- **The Problems**
- **The Proposal**

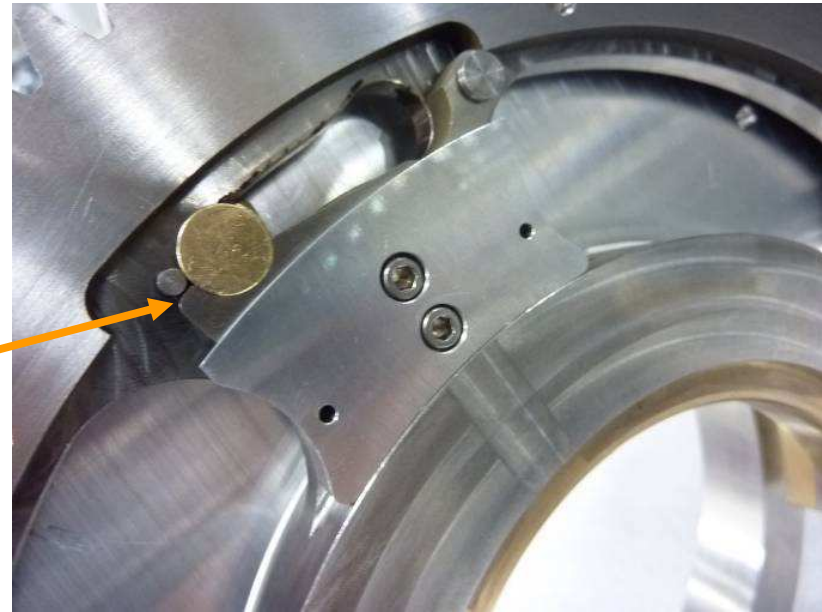
DIFFUSER



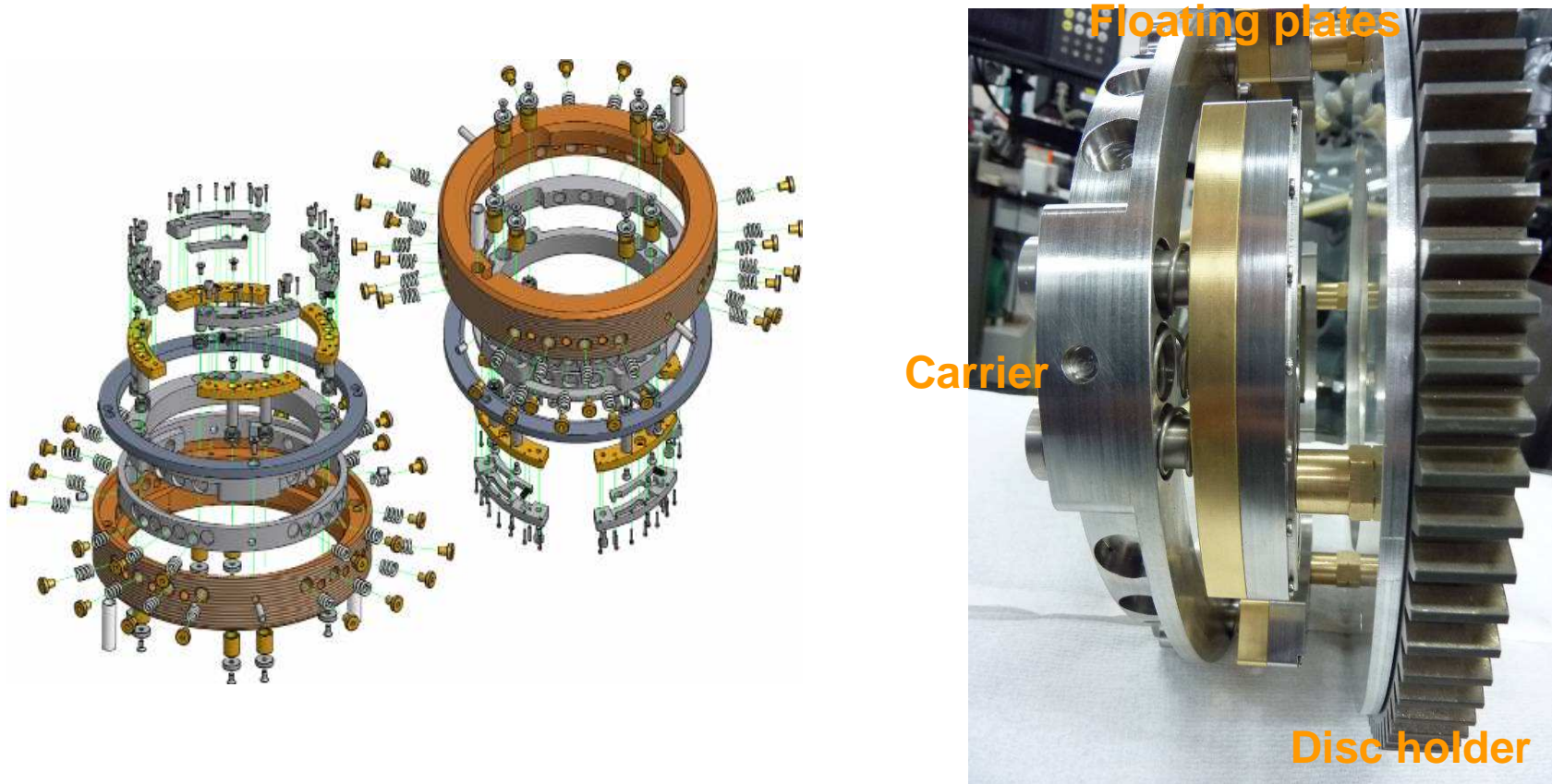
- Lead discs carried on Disc holders on a five-armed Carousel
- Selected disc transferred to or from Carrier and carried into solenoid bore
- Magnetic field precludes many normal engineering components
- Driven by three air motors
- Space is very limited – must fit in ~20cm or less

PROGRESS SINCE CM26

- Design modifications made, in particular:
 - New disc-holders in SS
 - New pins for the discs
 - Locking catches for discs
- Floating carrier front plate to accommodate misalignments



FLOATING CARRIER FRONT PLATES

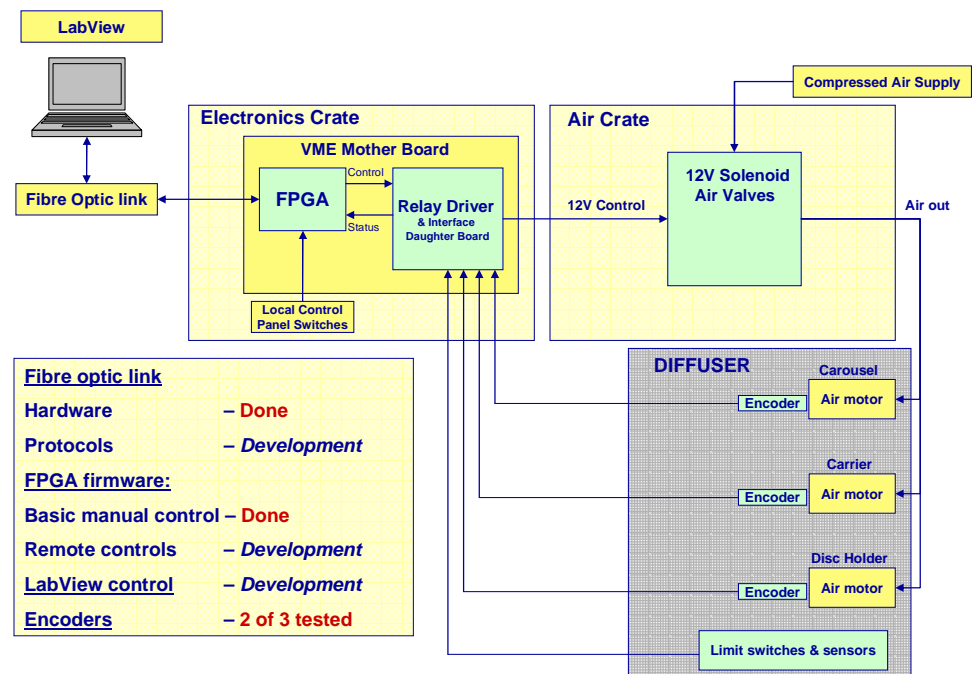
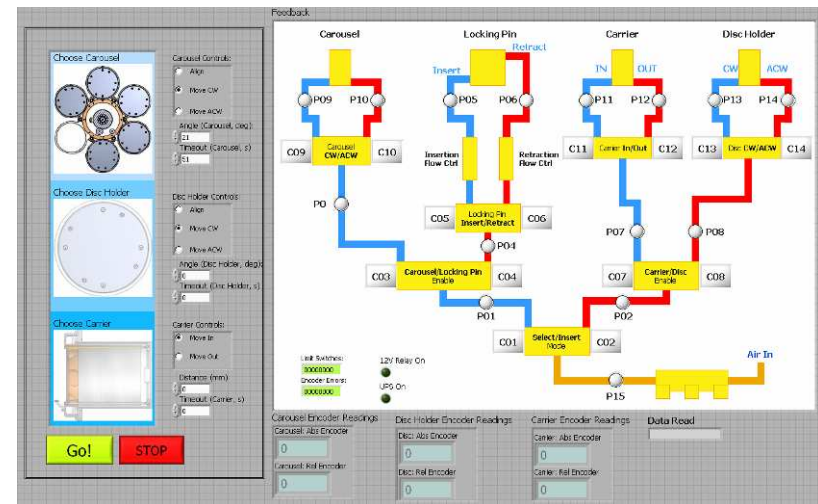


(about) 40 springs allow Left/Right, Up/Down & Fore/Aft floats of 0.5 – 1mm to accommodate alignment errors between disc-holders & carrier

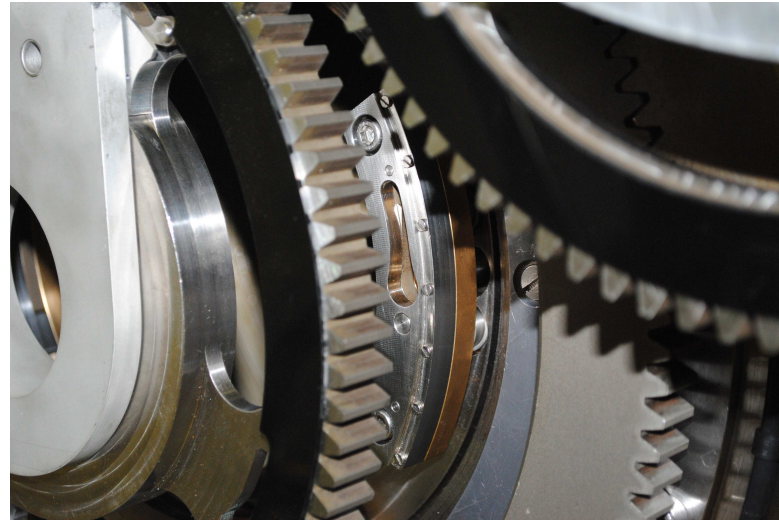
Works on bench – nice piece of engineering

CONTROLS

- FPGA in electronics crate controls air motors via valves in air crate and receives signals from encoders and microswitches
- FPGA controlled by LabView
- Air & electronics crates complete
- Firmware and software being developed
 - Versions exist for tests



REASSEMBLY – END OF MAY

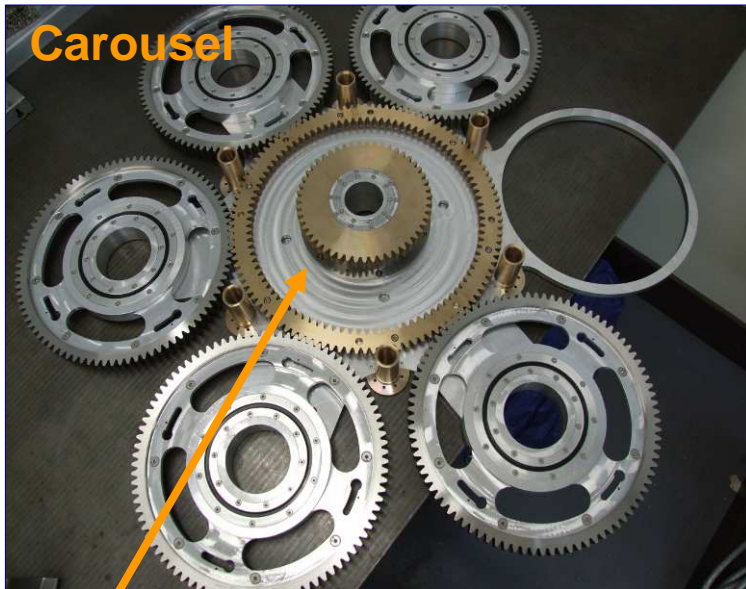


- **Beginning of June:**

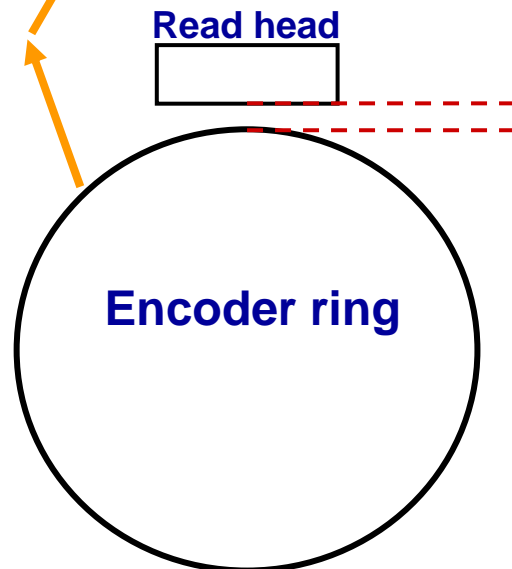
Started to test the ability to drive the carousel and disc-holders to the correct positions



- **Accurate alignment of disc-holders with carrier is crucial to allow transfer of disc**
- **Controlled by high-precision *shaft-encoders***
 - *Require fractional mm alignment themselves*
 - **Difficult to access**
 - **Buried inside mechanism →**



Carousel

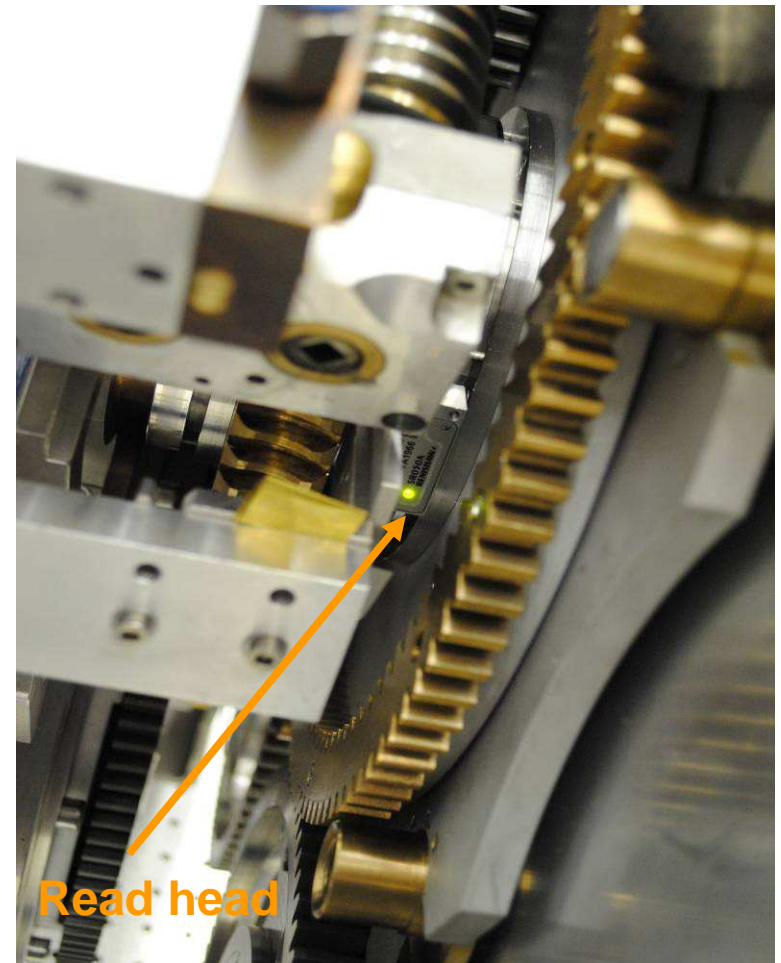


Read head

Encoder ring

**Ride height =
 0.8 ± 0.08 mm**

**Pitch, roll, yaw
 0.0 ± 0.5 degrees**

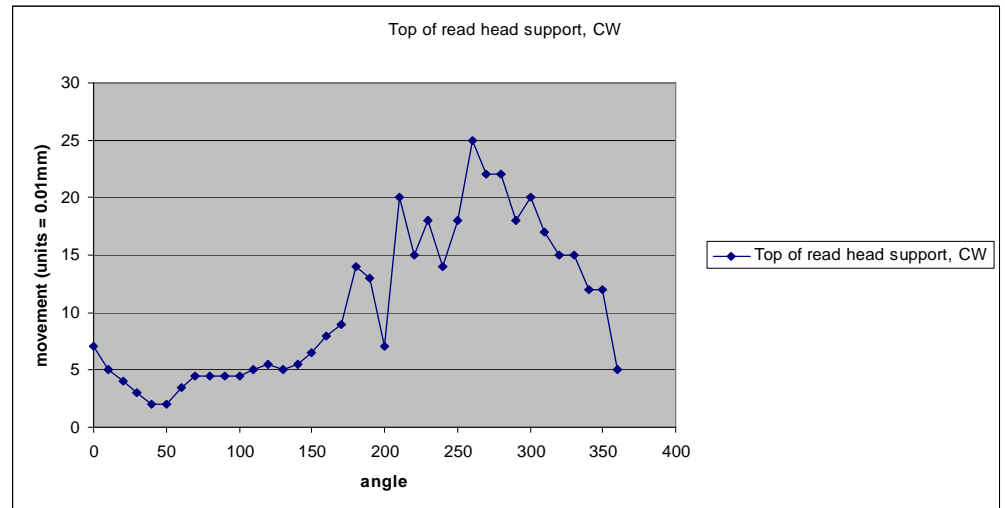
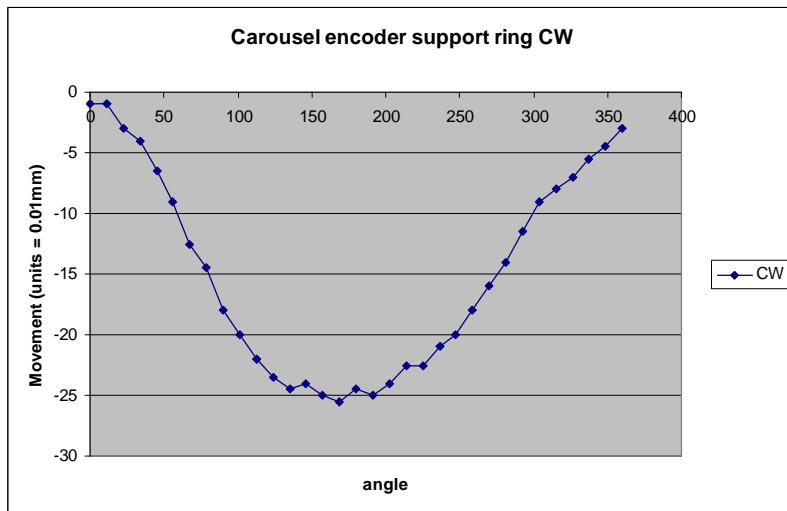
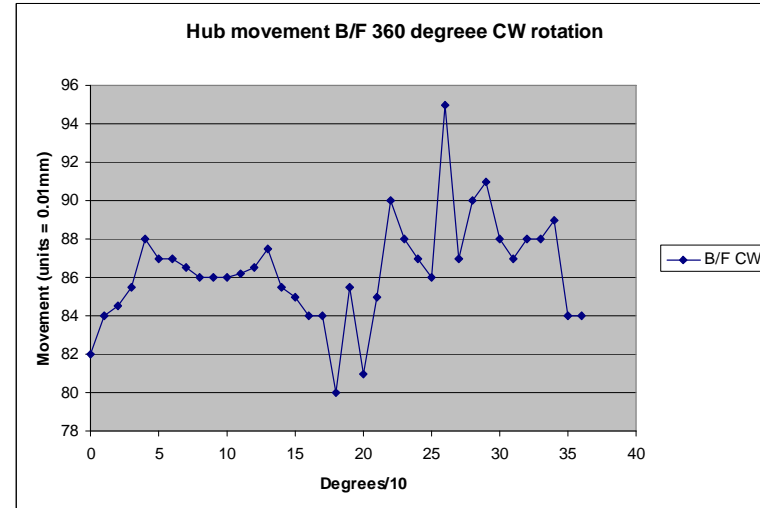
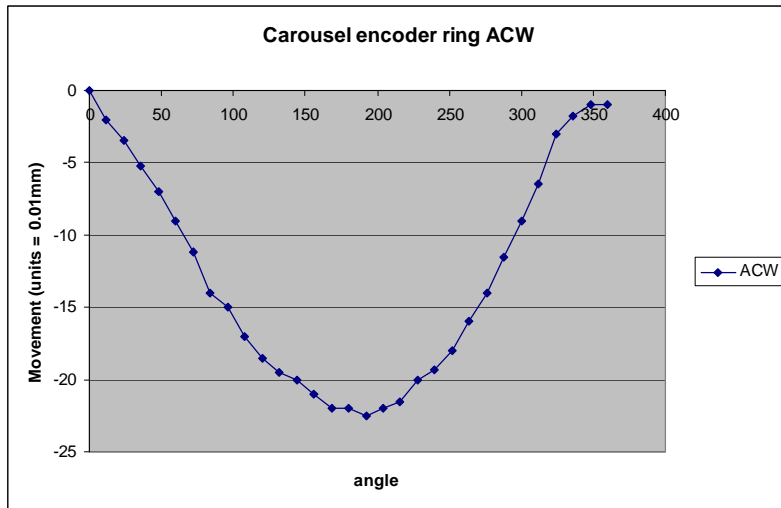


Read head

THE PROBLEMS (1.i)

- **Proved impossible to get stable signals from encoders**
 - **Obtained software from Renishaw (mfrs) giving signal size**
 - **Signal varies from < 30 % to > 100% of acceptable**
 - **Signal depends on position of carousel and movement**
 - **Made series of measurements at various places**
 - ***Hard to get to where we really want to measure***

SOME MEASUREMENTS



Components move by ~250 microns during one rotation of Carousel – too much

THE PROBLEMS (1.ii)

- **Movements of 0 – 0.25 mm during 360 degree rotation**
 - Compare with **+/- 80 microns** in encoder spec.
 - Also some visible roll >> 0.5 degrees
- **Carousel is unbalanced load**
 - **Whole assembly flexes**
 - Significant forces (~500N) generated during rotation
 - **Central 'shaft' not running true**
 - Actually several pieces
 - Independent support each end
 - 'Hub' on front plate & 'Inner hub' supported by drum
 - » Not tied together – unit comprises two pieces

THE PROBLEMS (2)

- Tried Disc-Transfer by hand
 - Floating plate tends to jam
 - Some misalignment forces pins outwards (??)
 - Possibly fixable by modifying pin profiles (??)
 - Carousel arms flex by ~1.5 mm when compressing springs of floating plate on carrier
 - Quite moderate force (few x 10N)
 - Much too much flex
 - Flexing of arms changes encoder signals !
- Too add to our woes...
 - Vital pin in carrier came loose
 - *Would require complete strip-down & rebuild to fix*
 - *2 weeks for that!*
- Moderate despair....

POSSIBLE SOLUTIONS

- **Strengthen mechanics**
 - But space very limited
 - One (~30mm) SS shaft for Carousel &c.
 - Beef-up Carousel
 - Use disc rather than 'pentapus'
 - Little (~2mm) space to make it thicker
 - *How thick is thick enough?*
 - *What material ?*
 - Would still be difficult to align the encoders
 - Space limited
 - Access difficult
- **Some ground-up re-design needed**
 - *Not guaranteed to work*
 - *A lot of time, £££ and effort*

OTHER CONSIDERATIONS

- Current design could (perhaps) be made to work in lab
 - Critical alignments $\leq 0.1\text{mm}$
- **Would have to repeat on Spectr. Solenoid eventually**
 - Magnetic or quench forces could upset alignments
- Magnetic field
 - **Unknown risks**
 - Behaviour of components in fringe field unknown
 - Even with SS there will be some magnetic forces
 - There are a few unavoidable small ferromagnetic components
 - Cannot guarantee (e.g.) that encoders would work
- ***Current design not converging***
 - *Haven't passed first base*
- **Need a Plan B**
 - Head scratching (*Why are we doing it like this?*)
 - 'Plan T' (Tacon & Tunnell)

PLAN T

Q. What do we really want to do ?

A. Place up to $\sim 3 X_0$ of dense material in beam to cover a diameter of up to 300mm depending on ε, p

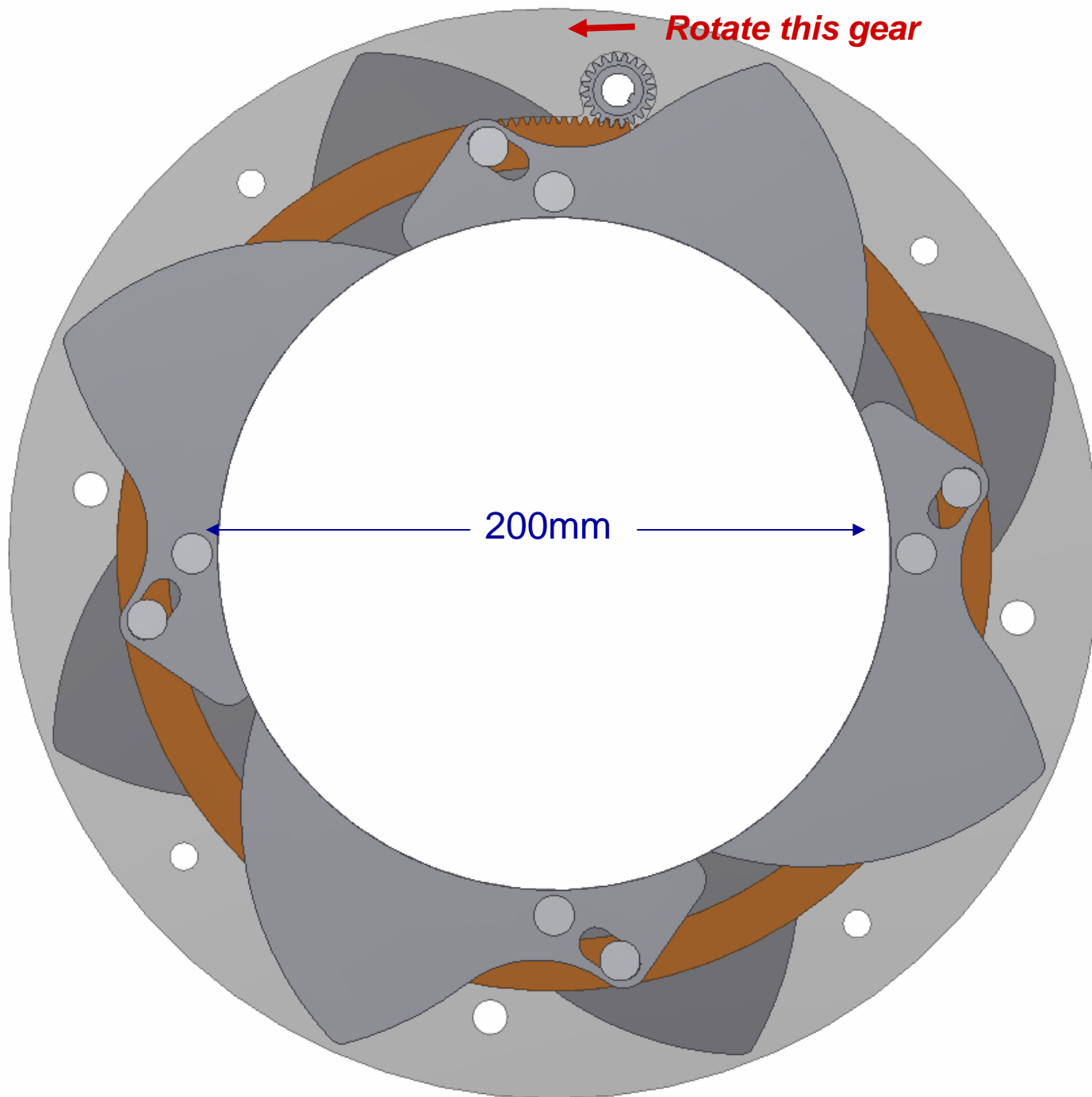
Do this as simply and reliably as possible

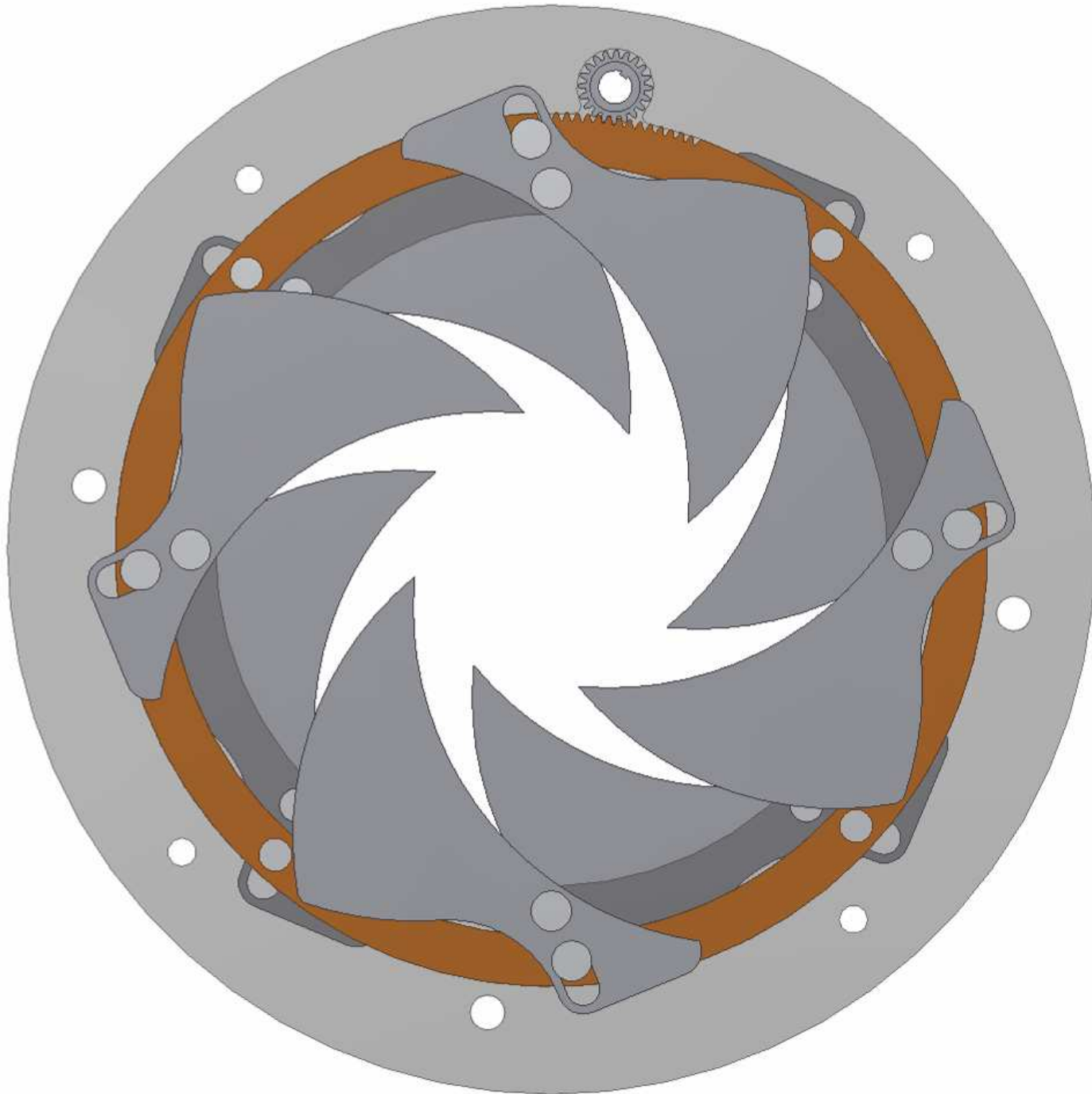
No need for complicated machinery – better without it

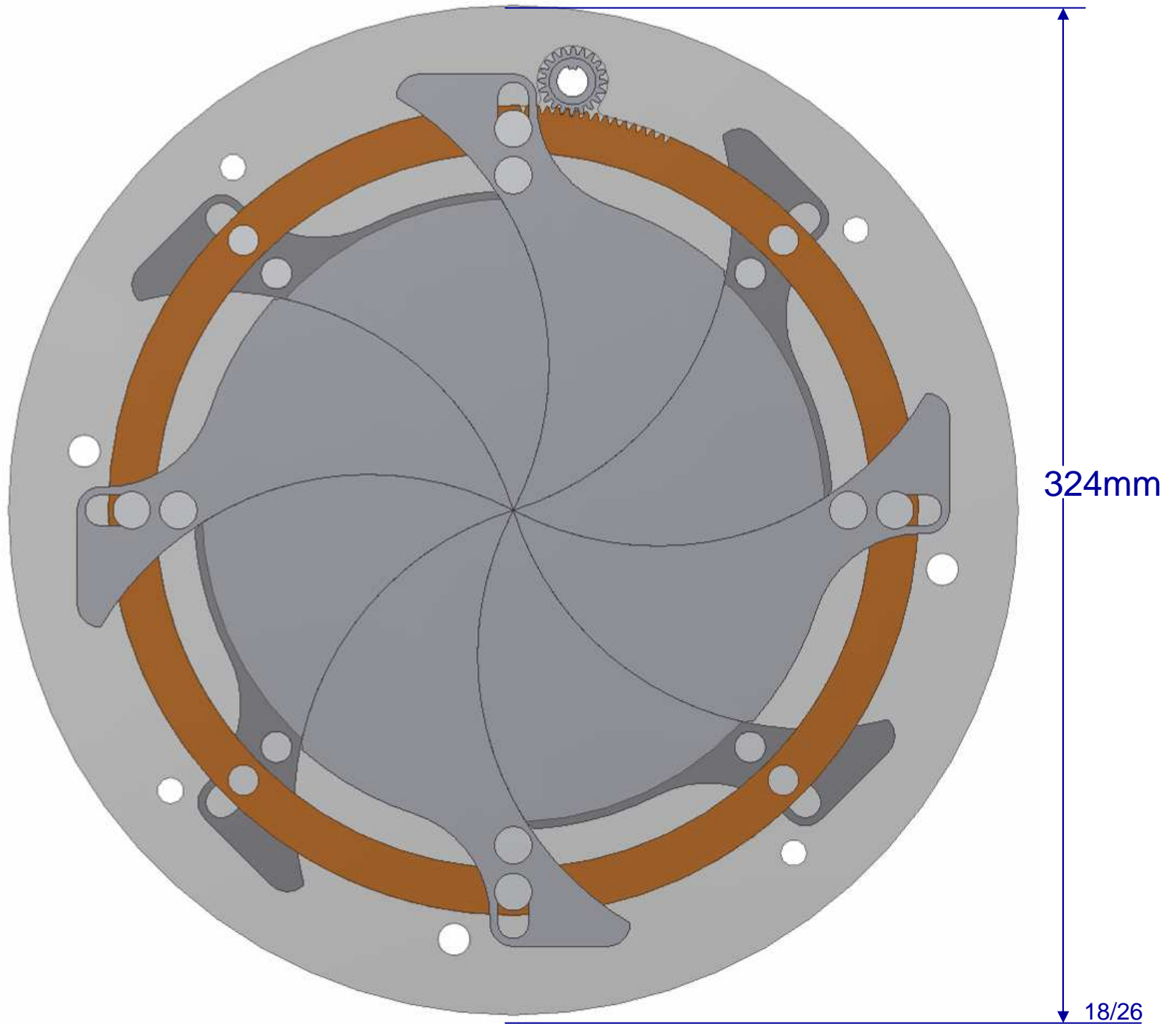
→ Use simple iris, or set of irises

Suggested by J. Tacon ~ 2 weeks ago

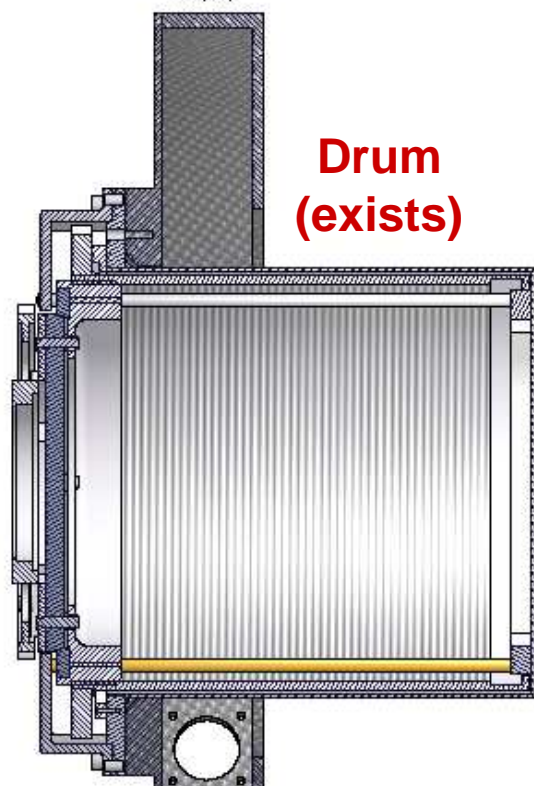
(also C. Tunnell)



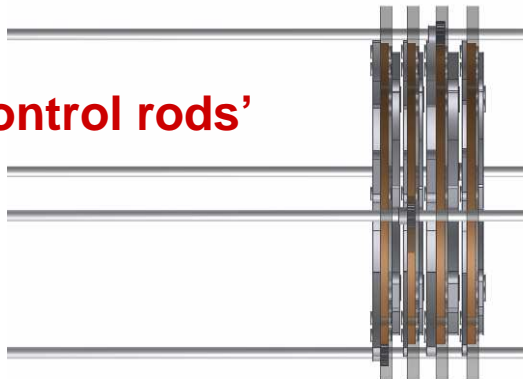




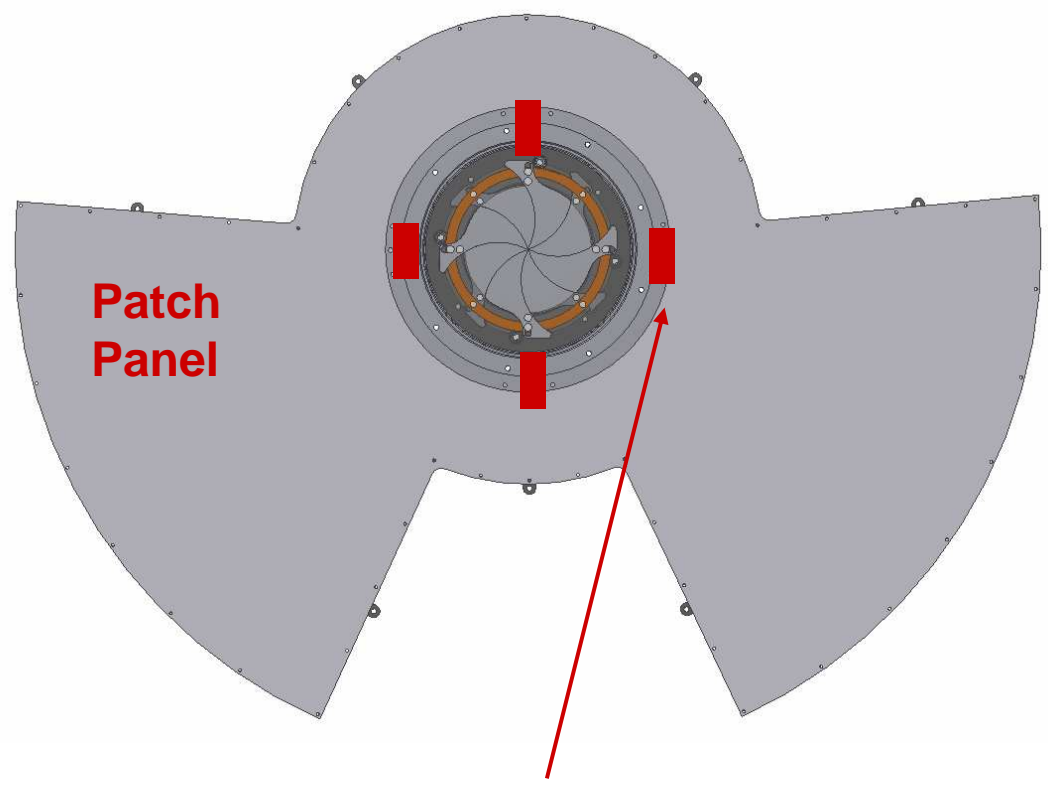
CM27



4 'Control rods'



4 Irises



Pneumatic actuators (4)

Details being worked out

Must be non-magnetic

4 Irises:

Double-sided with 4 petals / side
(max is 6 /side)

16 combinations of thickness

Provisionally (needs bit of work):

3.14 mm SS

6.28 mm SS

2.35 mm tungsten

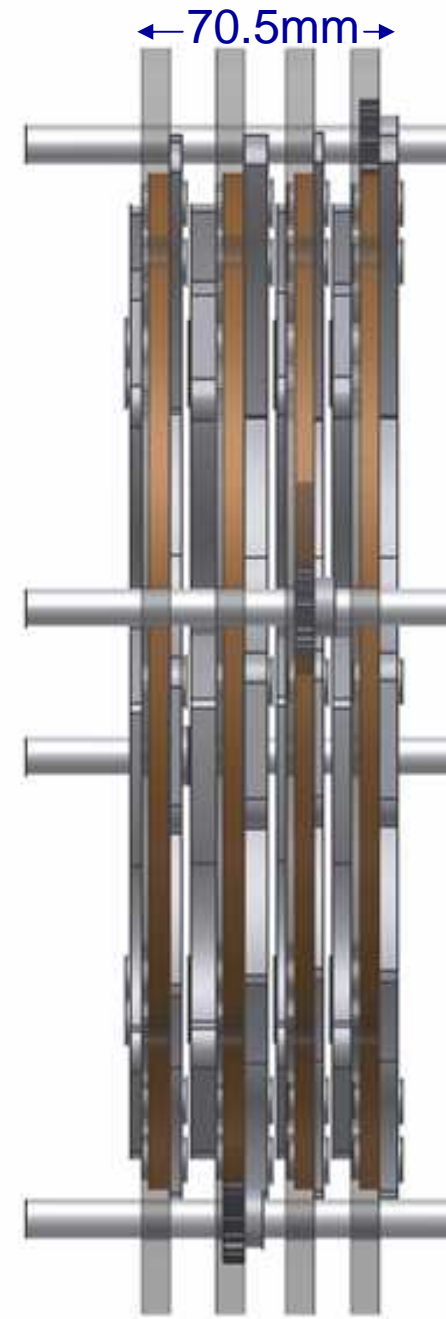
4.70 mm tungsten (or SS/Pb/SS sandwich)

Total of $2.7 X_0$ in 15 steps of $0.18 X_0$

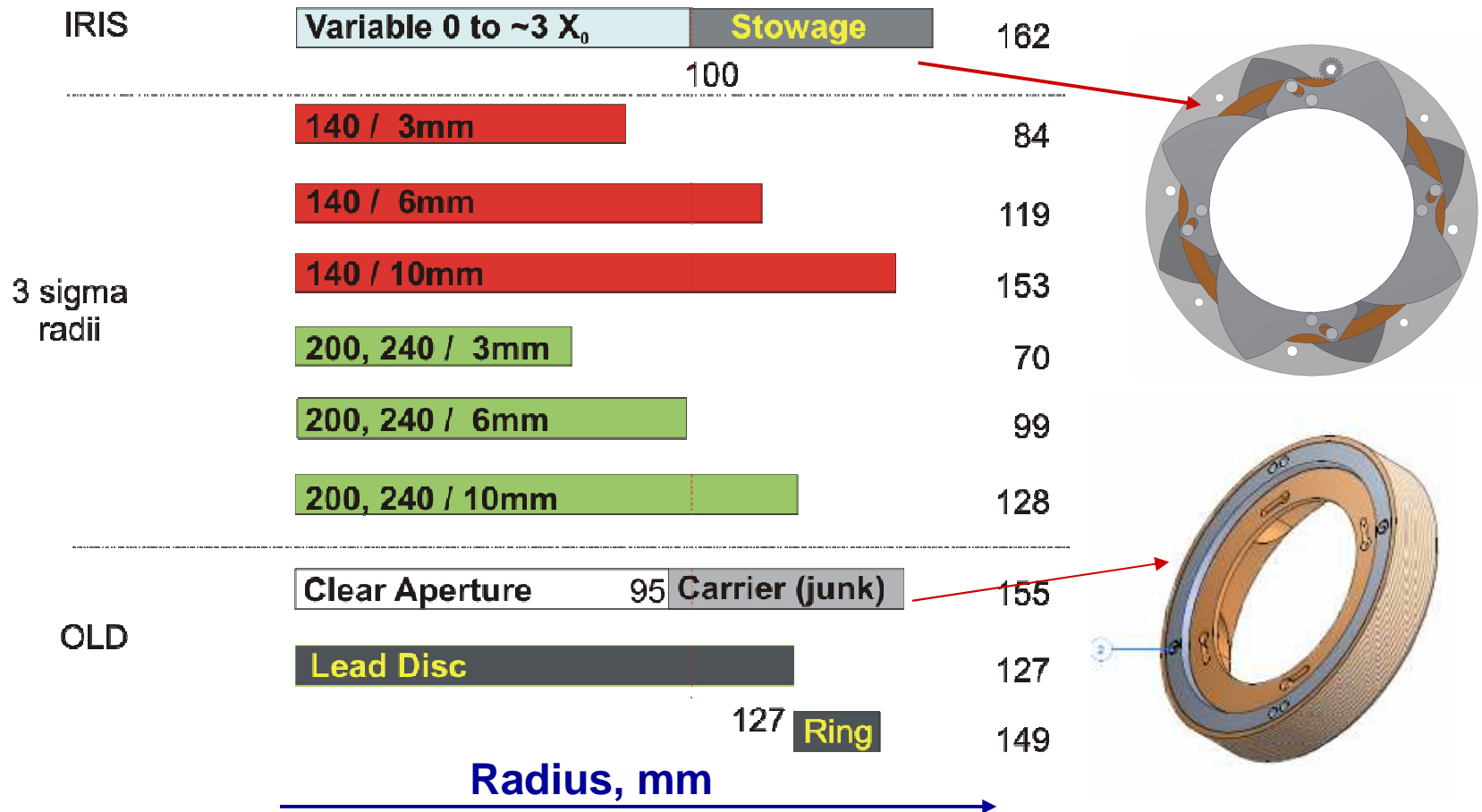
Operated by rotating 4 'control rods' 120 degrees

Will fit in existing 'Drum'

Somewhat longer than old version but initial MC says OK



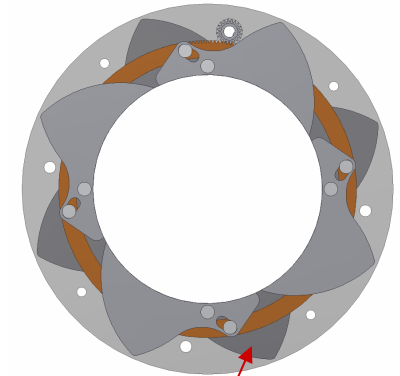
APERTURES & BEAM RADII



Increasing variable area → more petals → 3 or 4 layers per iris
 – may be possible but would make it longer by 50 – 100 %

LONGITUDINAL MATERIAL BUDGET

Emittance (mm)	Momentum (MeV/c)	Thickness required (X0)	3 sigma radius (mm)	Setting	In Aperture	Outside r = 100 mm	Outside + Supports
					(Radiation Lengths)		
				0	0	1.73	3.72
				1	0.18	1.62	3.60
				2	0.36	1.50	3.49
				3	0.54	1.38	3.37
				4	0.72	1.27	3.26
6	140	0.88	119	5	0.9	1.15	3.14
				6	1.08	1.04	3.03
				7	1.26	0.92	2.91
				8	1.44	0.81	2.80
				9	1.62	0.69	2.68
10	140	1.66	153	10	1.8	0.58	2.57
				11	1.98	0.46	2.45
				12	2.16	0.35	2.33
				13	2.34	0.23	2.22
				14	2.52	0.12	2.10
10	200 & 240	2.71	128	15	2.7	0.00	1.99



**Last column = Average material outside r = 100mm aperture
= Average over area of retracted petals
+ 5 x 7mm SS supports (to check)**

Not obviously worse than old design – which was accumulating various bits of junk on Carrier, r > 95mm

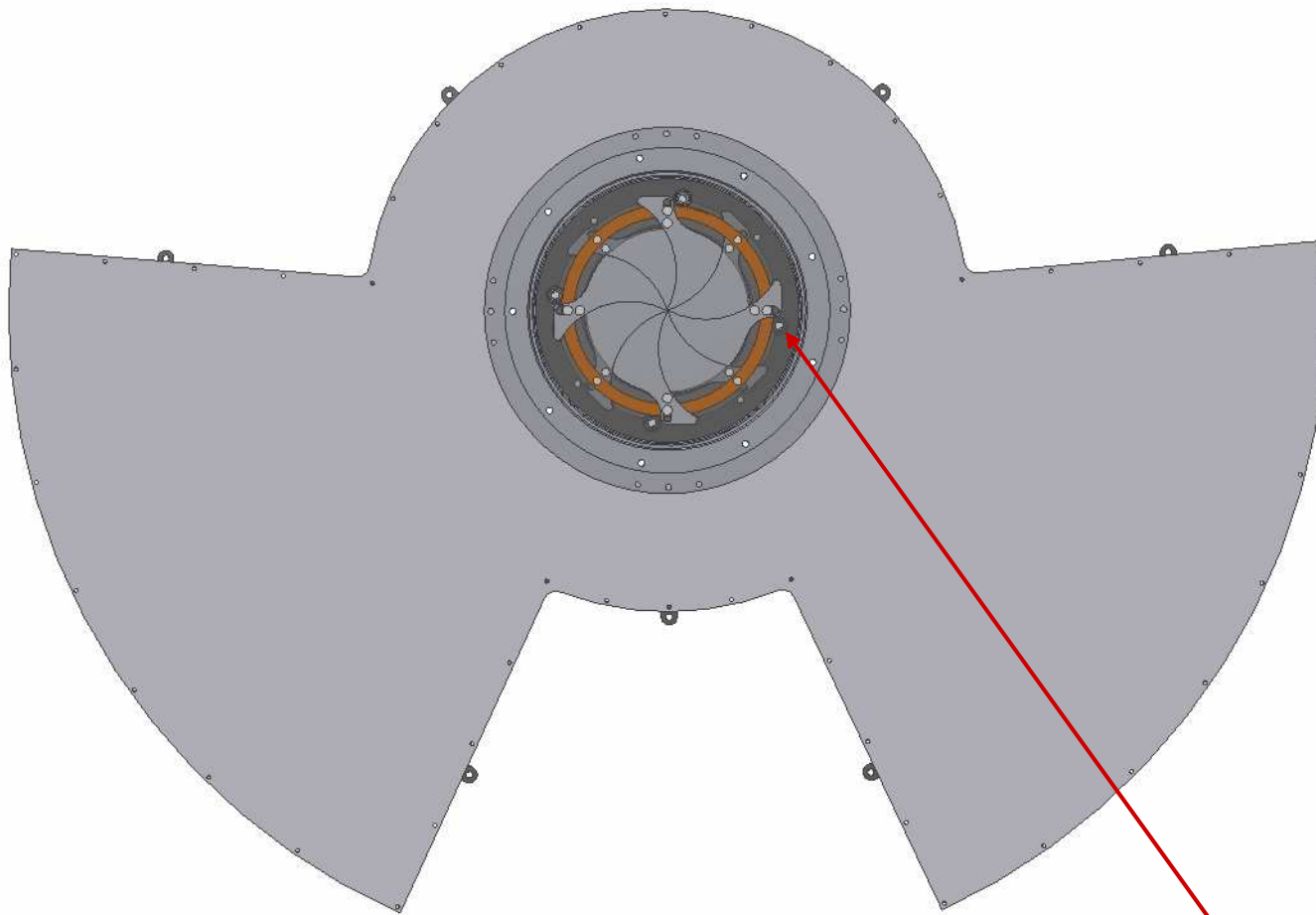
WHAT WOULD IT TAKE ?

- **Some consideration of Physics requirements**
 - Choice of 15 thicknesses probably advantageous
 - Can't easily do wedge (for dispersion)
 - But never in original plan
- **Mechanics**
 - Very simple
 - A few weeks to design & manufacture
 - Say 12 weeks
- **Controls**
 - Modify (rebuild) air crate
 - Re-use components (solenoid valves)
 - Modify electronics crate
 - Re-program FPGA
 - No big deal
- **Approval by MICE TB (presumably)**

CONCLUSION

- Hoped to demonstrate a working diffuser by now but *'The best laid plans of MICE and Men oft gang agla'*.
- Have seen enough of the problems of the 'flying wheel' design to be convinced that it will be very difficult to make it work, even on the bench
 - Have not passed first base after (too) much time & effort.
- Would be very nervous about the design in high field
 - Fixing problems would be a nightmare for everyone
- **It's been fun but.... *Tant pis, il faut en finir!***
- Though I don't like tempting providence...
 - **New design is simple**
 - **Quick to manufacture**
 - **Quick to install**
 - **Rugged in operation**

THE END



Occupies very little space on front of SS
Use simple pneumatic actuators to rotate 'control rods'
(~120 degrees to open / close)