

# SFGD Box Update

A. Gendotti 26.10.2020

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## Model Simplification

- All the materials properties are tuned according to the INR tests and includes holes
- Carbon Fiber is modeled as a stackup of 8 Layers according to CompositeDesign prototipe (2,3mm thickness of 8 Layers at different angles)
- Bolts conections between Panels-Panels and Bracket-Panels are not implemented (Bonded contact is used)
- Pads for the PCBs simplified by taking the thinner thickness of the Panel (coservative)
- Weight of the cubes is simulated as a Force in order to be conservative.
- For the Vibration Study attempt to caracterize the cubes

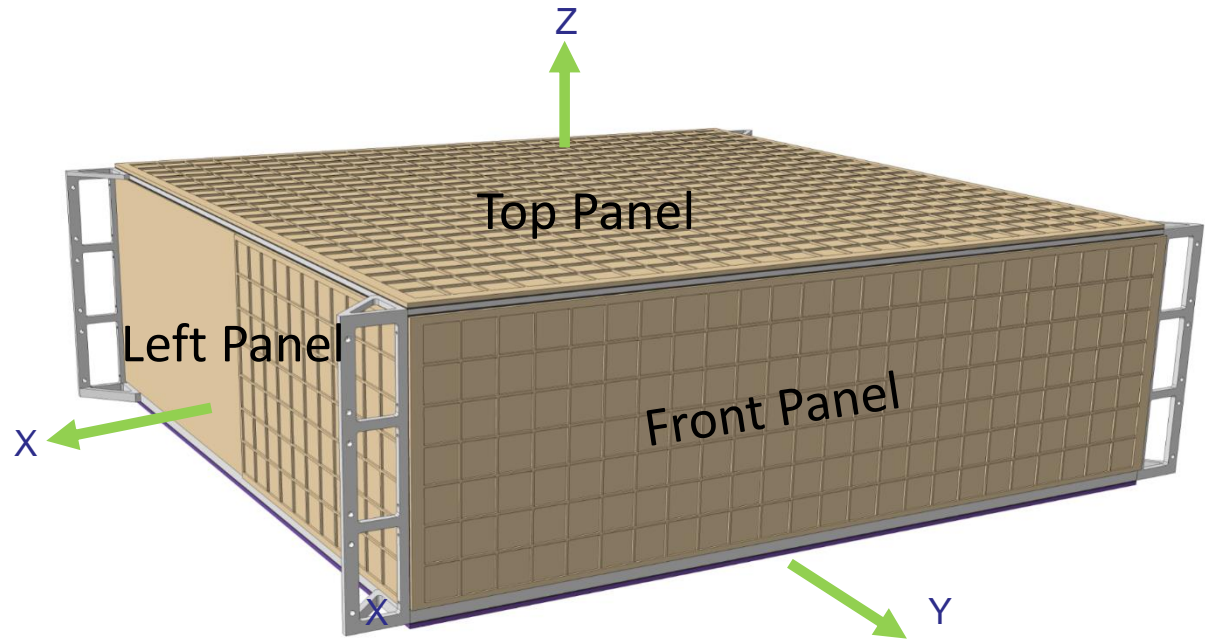
## FEA Studies

- **First Static study** is considered as the normal operation load: Earth Gravity at the Box and Cube weight.
- **Second Static Study** is cosidering Earth Gravity and 0.65g aceleration at each direction. This in order to spot critical «situation» at the level of stresses and deformations
- **Third dinamic study** is considering the PSD spectrum provided by KEK and simulate the PSD response at each direction. In this study the cubes have to be implemented. Since is very difficult to caracterize the Cubes behaviour, a simpification was needed. Anyway the study, to my point of you, can give a good idea of the situation. In PSD study all connection are bonded (linear)

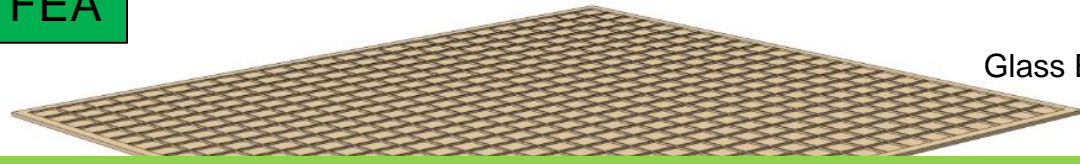
## Sandwich Materials and Thickness

• According to the design at the current status the Material used for the Box are below listed:

- Bottom Panel → Carbon Fiber – Acrylic – Carbon Fiber Thickness 33.6 mm
- Top Panel: → Carbon Fiber – Acrylic – Carbon Fiber Thickness 16.6mm
- Front and Back Panels: → Carbon Fiber – Divinycel H250 – Carbon Fiber Thickness 19.6mm
- Left and Right Panels: → Carbon Fiber – Divinycell H250 – Carbon Fiber Thickness 24.6mm

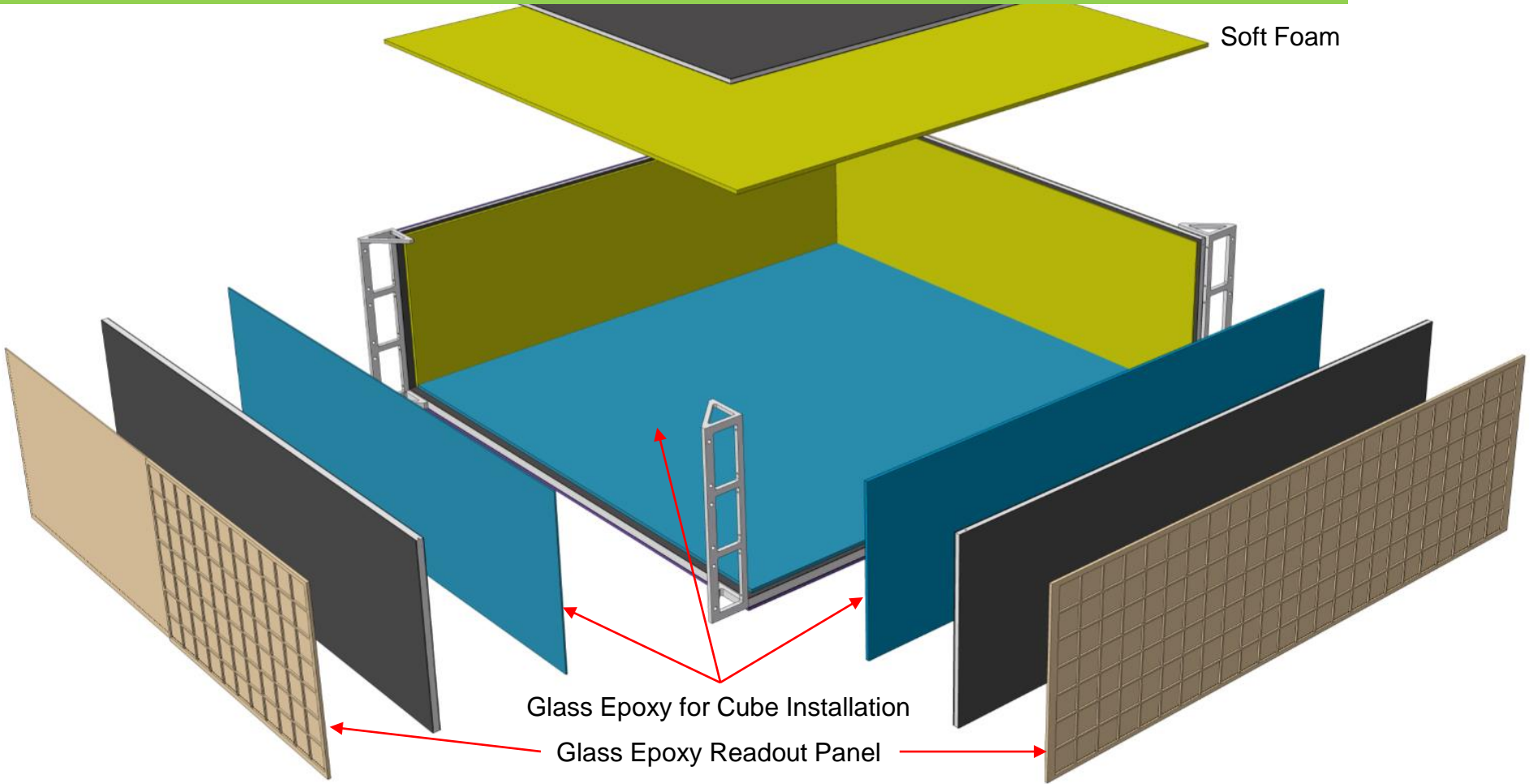


Glass Epoxy Readout Panel



- Readout panels and hard panels for Cubes Installation (blue) are G10 (glass epoxy)
- Soft Foam Panels (yellow) consists in Polipropilene Foam (used only when cubes are implemented)
- Brackets material is Aluminum

Soft Foam



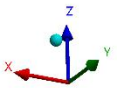
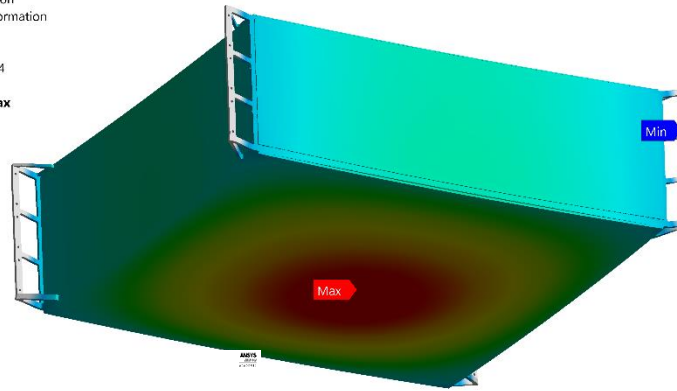
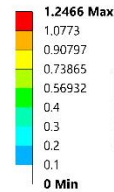
Glass Epoxy for Cube Installation

Glass Epoxy Readout Panel

### Z- direction:

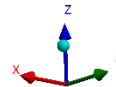
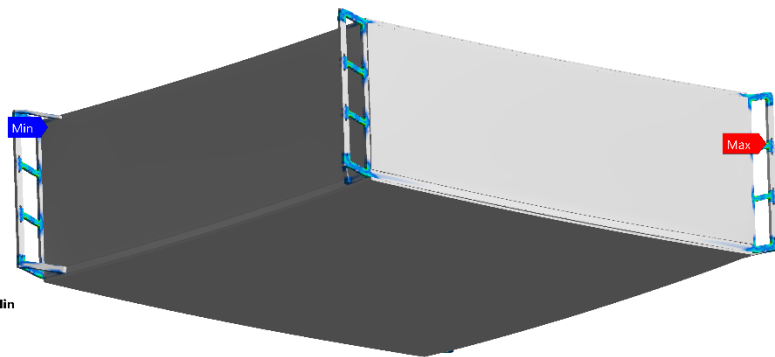
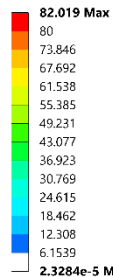
- Maximum deformation at the Bottom Panel: 1.5 mm
- Maximum deformation at the Top Panel: 0.85 mm
- Max Stress at the Brackets: 82 Mpa
- Max Stress at Alu Frames (Bracket connect.): 35.5 Mpa
- Max Stress at Acrylic Bottom panel ~1.26 Mpa
- Failure Safety Factor at Carbon Fiber SF: 4

**C: Box Preliminary Static**  
Total Deformation  
Type: Total Deformation  
Unit: mm  
Time: 2  
19.10.2020 10:54



**C: Box Preliminary Static**

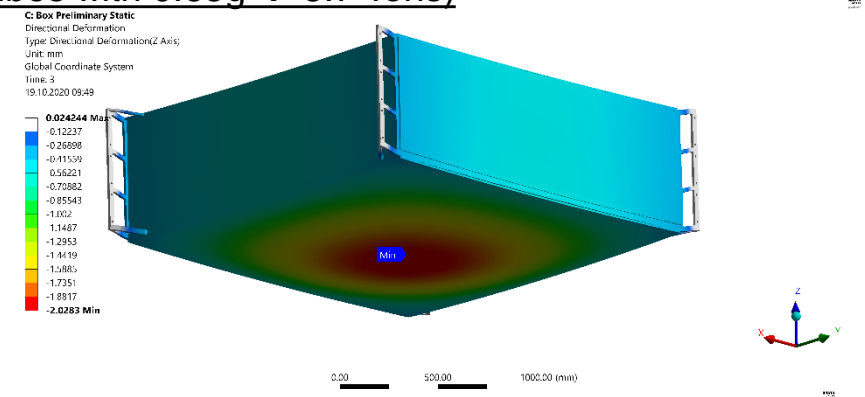
Equivalent Stress  
Type: Equivalent (von-Mises) Stress - Top/Bottom - Layer 0  
Unit: MPa  
Time: 2  
19.10.2020 11:04



Gravity on z axis + 0.65g in every direction (Weight of the cubes with 0.65g → 3.7 Tons)

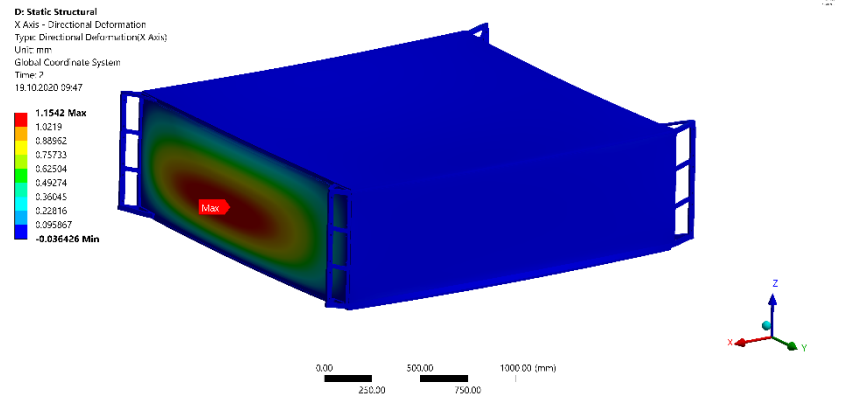
Z- direction:

- Maximum deformation at the Bottom Panel: 2.03 mm
- Maximum deformation at the Top Panel: ~1.4 mm
- Max Stress at the Brackets: 135.11 Mpa
- Max Stress at Alu Frames (Bracket connect.): ~50Mpa
- Max Stress at Acrylic Bottom panel ~2Mpa
- Failure Safety Factor at Carbon Fiber SF: 3.8



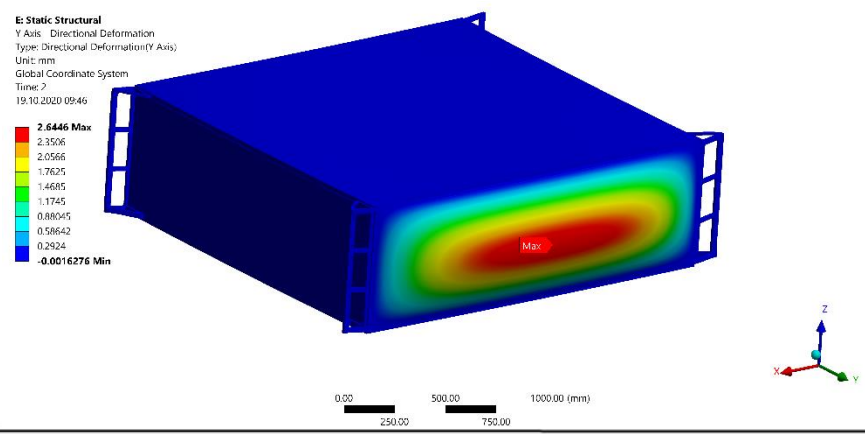
X- direction:

- Maximum deformation at the Left/Right Panel: 1.16 mm
- Max Stress at the Brackets: 86.26 Mpa
- Max Stress at Alu Frame <10Mpa
- Max Stress at Foam core <10Mpa



Y- direction:

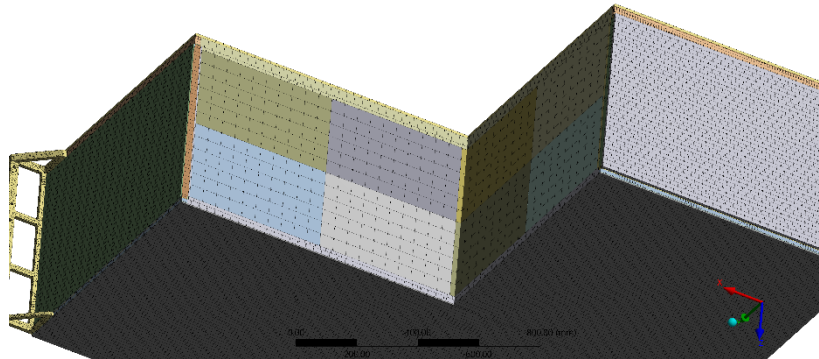
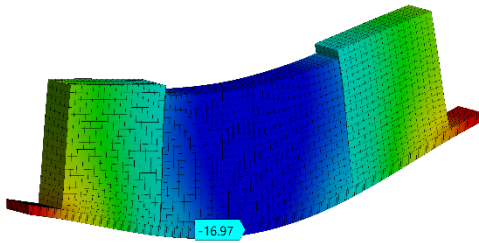
- Maximum deformation at the Front/Back Panel: 2.65 mm
- Max Stress at the Brackets: 85.8 Mpa
- Max Stress at Alu Frame 46 Mpa
- Max Stress at Foam core 10Mpa
- Failure Safety Factor at Carbon Fiber SF: 4



- Cubes characterized as a solid ( best approximation up to now). Cubes Envelope divided in 32 Solid and Young's modulus lowered (jelly kind solid)

**B: IRN without Holes Center Weight**  
 Directional Deformation  
 Type: Directional Deformation(Z Axis)  
 Unit: mm  
 Global Coordinate System  
 Time: 1  
 06.10.2020 08:25

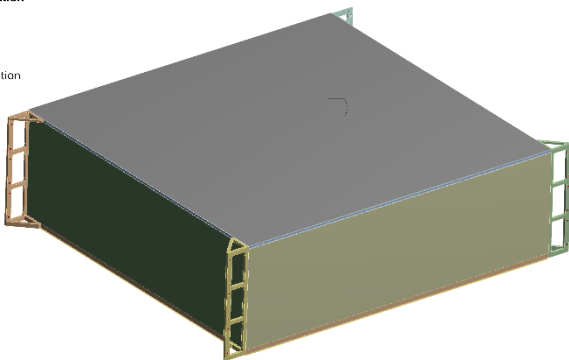
**3.1568 Max**  
 0.8136  
 -1.5296  
 -3.8729  
 -6.2161  
 -8.5593  
 -10.903  
 -13.246  
 -15.589  
**-17.932 Min**



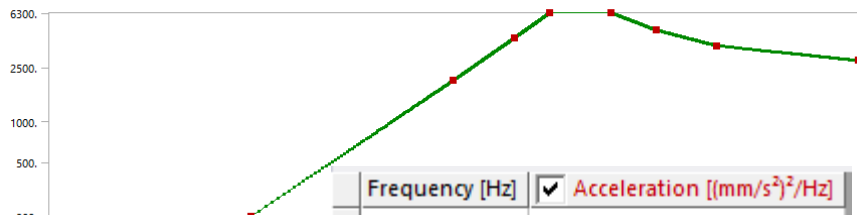
- Static calculation (non linear to verify if results are close to the first study)
- Modal Analysis → automatically considered as linear (all the contacts are bonded)
- PSD response from acceleration spectrum given by KEK  
 → Random Vibration applied at the Bracket fixed supports and calculated at all 3 directions with preload from static

**E: Random Vibration**  
 PSD Acceleration  
 Time: 1 s  
 19.10.2020 11:23

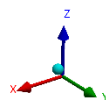
PSD Acceleration



0.00 250.00 500.00 750.00 1000.00 (mm)



	Frequency [Hz]	<input checked="" type="checkbox"/> Acceleration [(mm/s <sup>2</sup> ) <sup>2</sup> /Hz]
1	0.1	200.
2	1.	2000.
3	2.	4100.
4	3.	6300.
5	6.	6300.
6	10.	4700.
7	20.	3600.
8	100.	2800.
*		





Maximum deformation at the Bottom Panel:  
Maximum deformation at the Top Panel:  
Max Stress at the Brackets:  
Max Stress at Alu Frames (Bracket connect.):  
Max Stress at Acrylic Bottom panel  
Failure Safety Factor at Carbon Fiber

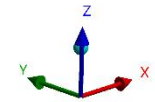
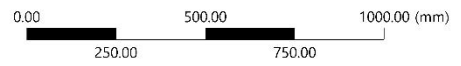
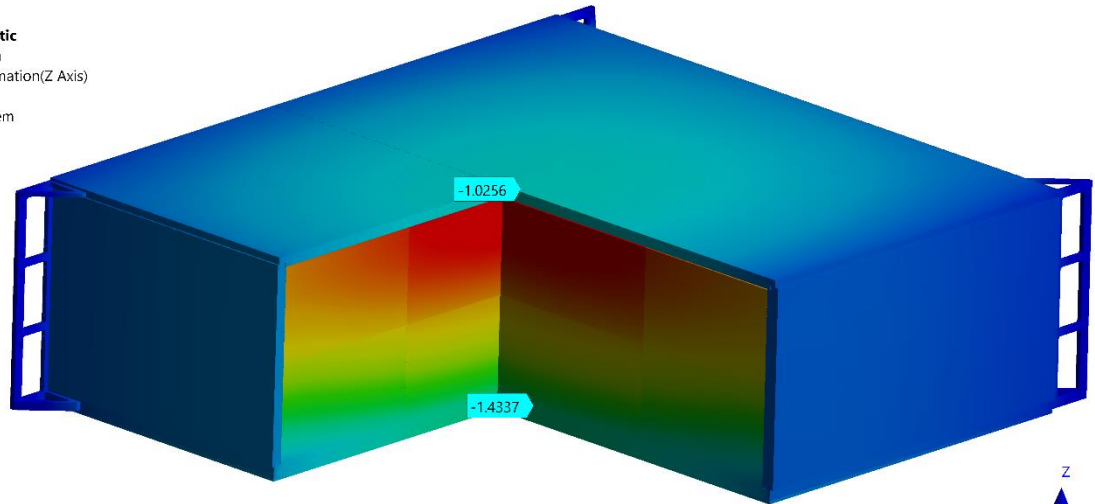
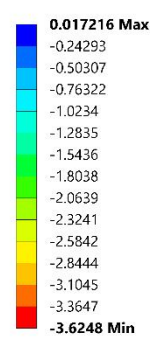
1.44 mm  
1 mm  
82 Mpa  
27 Mpa  
~1.26 Mpa  
SF: ~3.8

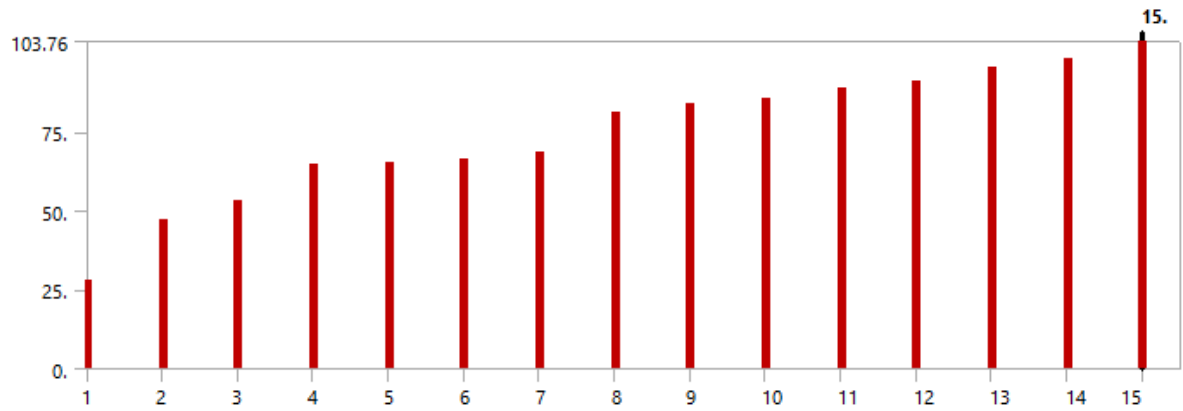
**Very similar to the first static Study**

Maximum deformation at the Bottom Panel:	1.5 mm
Maximum deformation at the Top Panel:	0.85 mm
Max Stress at the Brackets:	82 Mpa
Max Stress at Alu Frames (Bracket connect.):	35.5 Mpa
Max Stress at Acrylic Bottom panel	~1.26 Mpa
Failure Safety Factor at Carbon Fiber	SF 4

Cubes behave almost as pressure due to low Young's module

**C: Box Preliminary Static**  
Directional Deformation  
Type: Directional Deformation(Z Axis)  
Unit: mm  
Global Coordinate System  
Time: 2  
19.10.2020 11:54

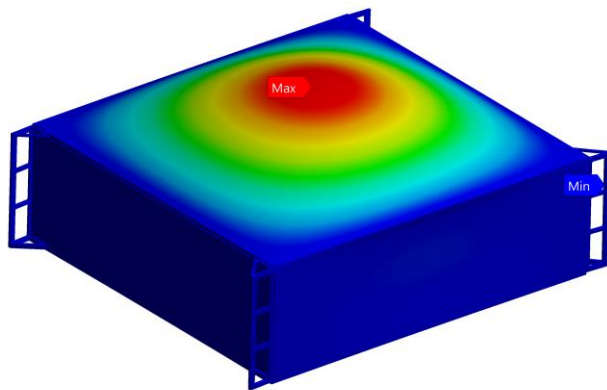
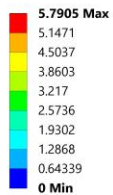




Mode	Frequency [Hz]
1	27.927
2	46.948
3	53.353
4	64.682
5	65.171
6	66.616
7	68.485
8	81.17
9	83.826
10	85.41
11	89.126
12	91.311
13	95.384
14	98.156
15	103.76

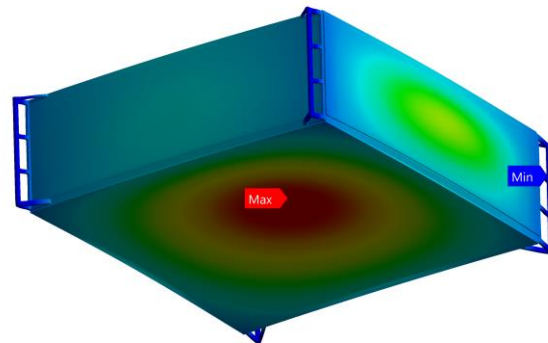
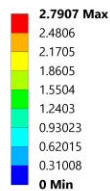
## First mode at ~28 Hz (Top Panel)

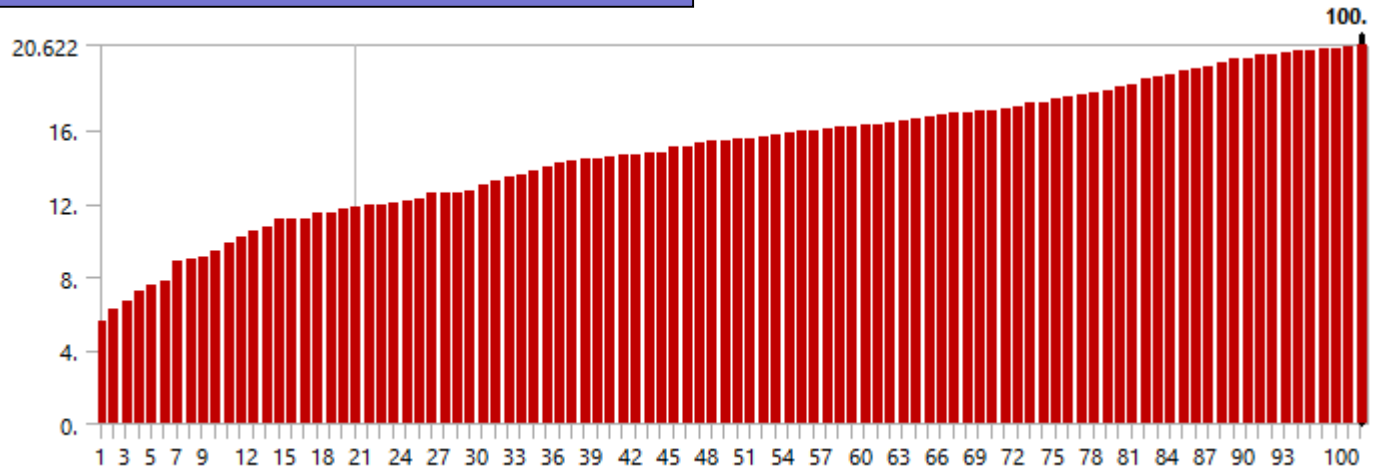
F: Modal Pre step 2  
 Total Deformation  
 Type: Total Deformation  
 Frequency: 27.927 Hz  
 Unit: mm  
 26.10.2020 12:40



## Second mode at ~47 Hz (Bottom Panel)

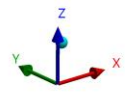
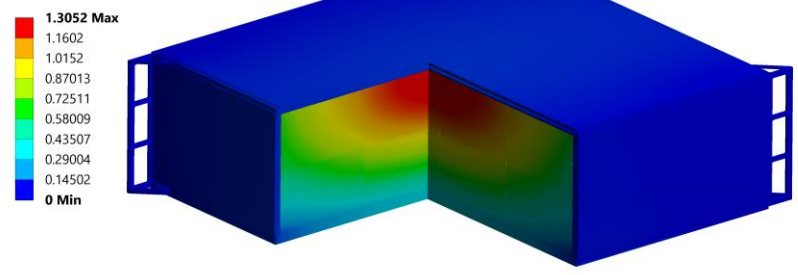
F: Modal Pre step 2  
 Total Deformation 2  
 Type: Total Deformation  
 Frequency: 46.948 Hz  
 Unit: mm  
 26.10.2020 12:42





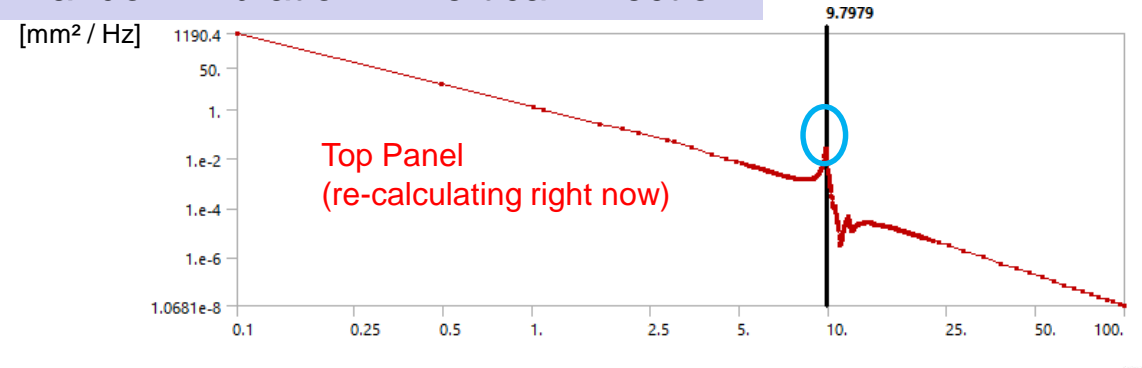
Modal Analysis at preloaded static model with cubes implemented: first 100 modes are at the cubes.

D: Modal  
Total Deformation 11  
Type: Total Deformation  
Frequency: 9.8004 Hz  
Unit: mm  
26.10.2020 12:17



# THIRD STUDY – PSD Response

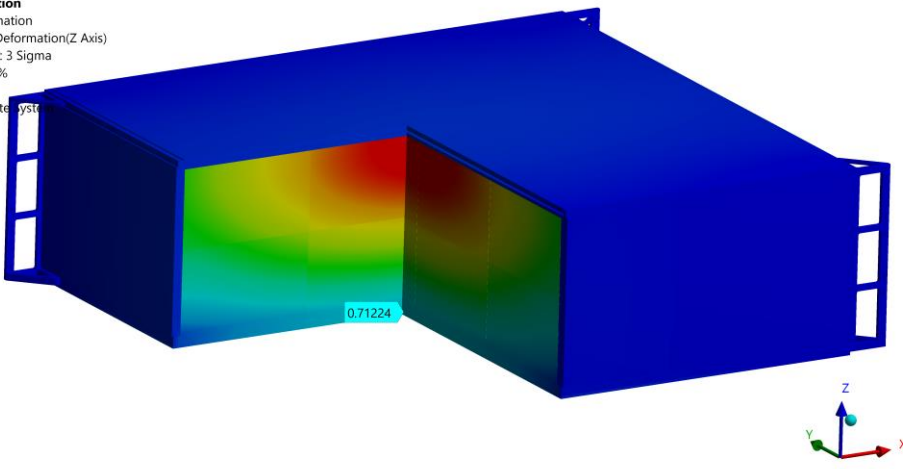
## Random Vibration in Vertical Direction



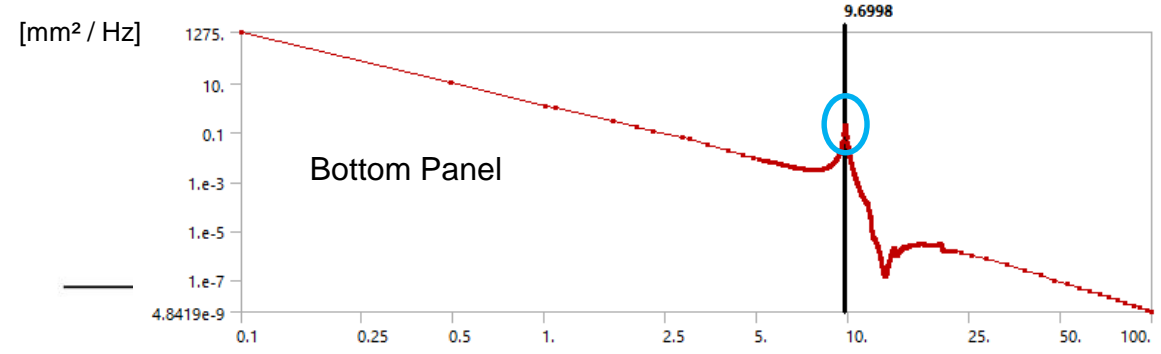
!Checked all the responses in all directions and the critical case is in the vertical direction at the Top Panel!

**E: Random Vibration**  
 Directional Deformation  
 Type: Directional Deformation(Z Axis)  
 Scale Factor Value: 3 Sigma  
 Probability: 99.73 %  
 Unit: mm  
 Solution Coordinate system  
 Time: 0  
 26.10.2020 12:22

**2.7372 Max**  
 2.4331  
 2.1289  
 1.8248  
 1.5207  
 1.2165  
 0.9124  
 0.60827  
 0.30413  
**0 Min**



Scale Factor	3 sigma
Probability:	99,73 %
Max Def Top Panel	<b>0.25mm</b>
Max Def. Bottom	0.72mm
Max Strees:	11Mpa (@Bracket)

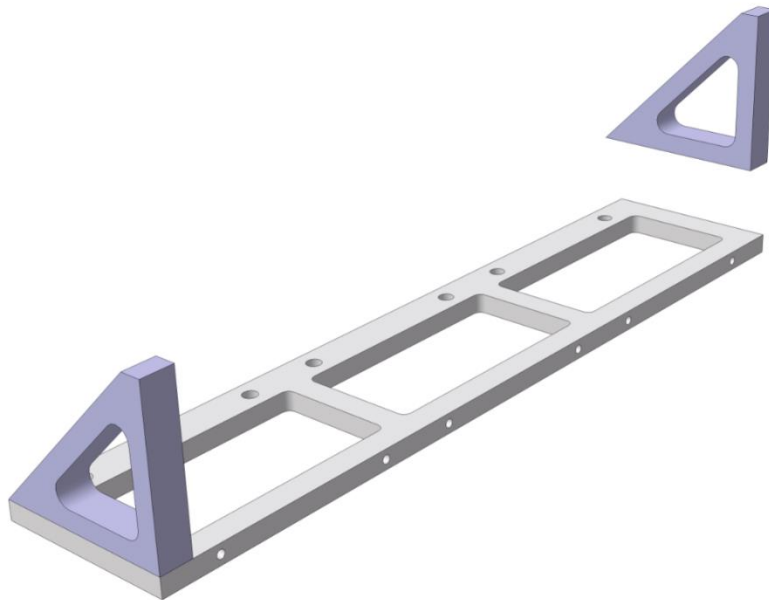


## Conclusion

- Good characterization of the Cubes behaviour is almost impossible.
- Deformations and stresses are considered mostly in order to get an idea of the Box behaviour under Random vibrations.
- Stresses and Deformations at the panels are always very low.
- Considering the real case of having all the cubes with frictional contact (to each other and to the Panels) it would reduce significantly the deformation at the panels due to vibrations. The single cubes would damp a lot the vibrations. → **!!My Opinion!!**
- Top Panel would be actually not connected to the Cubes and free to vibrate. Increasing the thickness of the Top panel would anyway increase the first mode vibration and reduce the deformation.

## Modification at the Box

- **Increase** top Panel thickness same as the Bottom Panel and use Divinycell H250 as a core instead of Acrylic.  
→ This would give less deformation in case of an Earthquake and would slightly increase the Resonance frequency.  
→ Reduce the possibility to hit the cubes in case of shaking
- **Use 316L** (non magnetic stainless steel) for the Bracket → Giving an extra safety. In case of using Aluminum, it would be necessary to use special/reinforced alloy.



Divide the Bracket in parts bolted or welded together.  
The shape could be lasered (company I'm in contact can laser up to 30mm Stainless Steel)

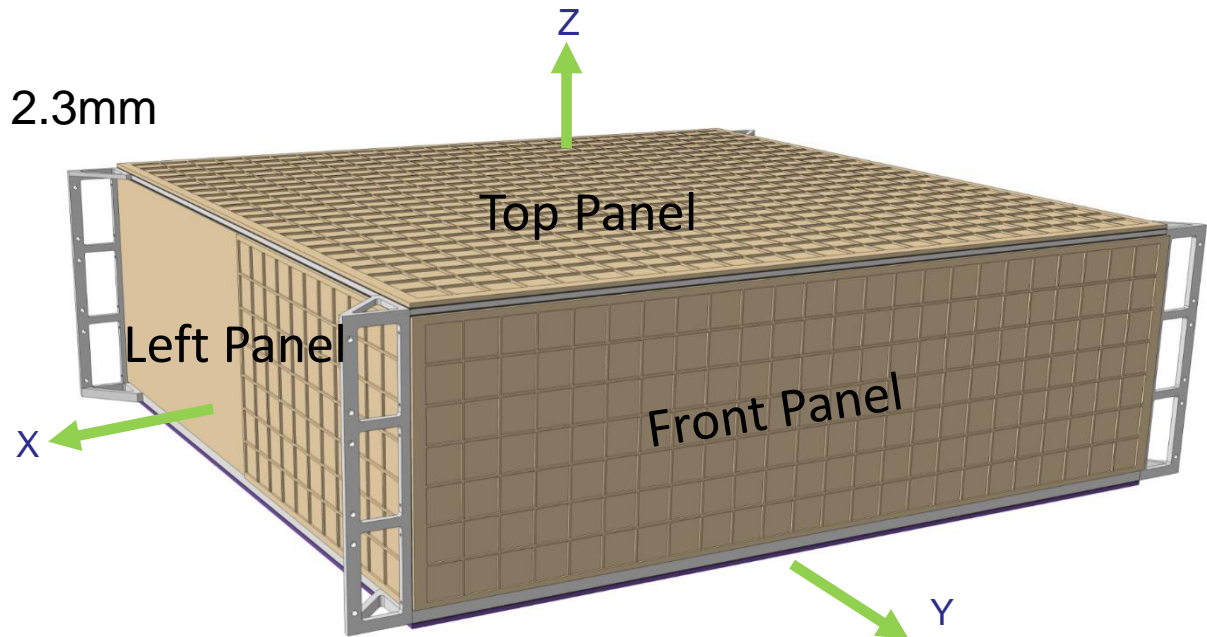
## EMDE Laser Company

Carbon steel	■ Stahl bis 25mm
Stainless steel	■ Edelstahl bis 30 mm
Alu	■ Aluminium bis 25 mm
Copper	■ Kupfer bis 10 mm
Brass	■ Messing bis 10 mm

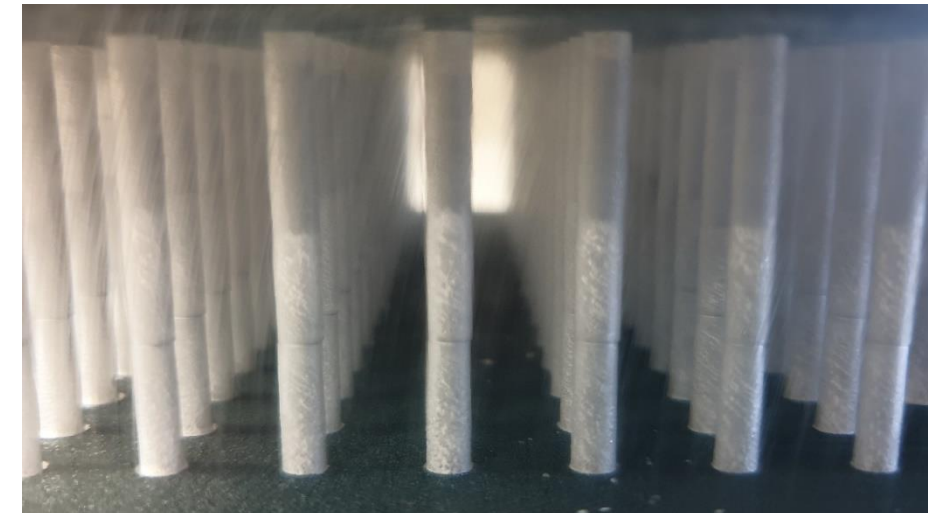
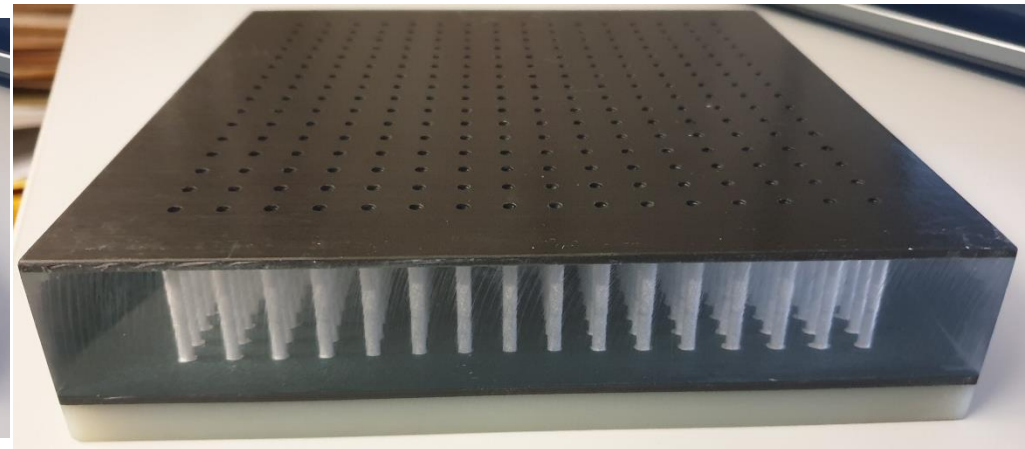
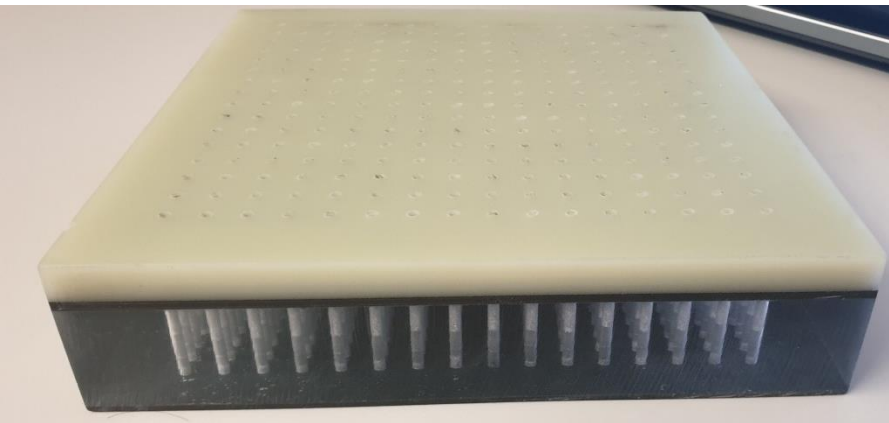
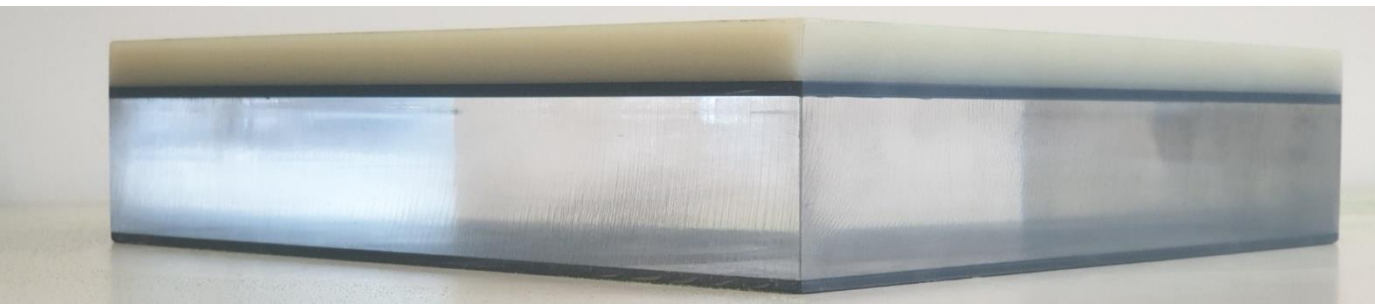
## Final Sandwich thickness Materials

- According to the design at the current status the Material used for the Box are below listed:
  - Bottom Panel → Carbon Fiber – Acrylic – Carbon Fiber Thickness 33.6 mm → waiting
    - Waiting for composite design to know the exact core dimension between 29mm and 30mm
  - Top Panel: → Carbon Fiber – Acrylic – Carbon Fiber Thickness 16.6mm → Change
    - Increase the thickness at 33.6mm (check with Franck if there is enough clearance) and Divinycell Core
  - Front and Back Panels: → Carbon Fiber – Divinycel H250 – Carbon Fiber Thickness 19.6mm
  - Left and Right Panels: → Carbon Fiber – Divinycell H250 – Carbon Fiber Thickness 24.6mm

Carbon Fiber Layer is always 2.3mm  
Glue is 0.1mm







- 200x200mm prototype received from CompositeDesign
- 3mm Holes drilled through by CIMFORM company
- Visual Check doesn't show any crack in the acrylic and glued parts looks good.
- Contacted CERN to perform stress test to verify bending etc..

- Purchasing New Prototype for INR:
  - Black G10
  - Black Acrylic
  - Size of 2000mmx150mm